

# Oscillations.

1

In many cases we can write our dynamical system in the form

$$\dot{\theta} = f(\theta) \quad \text{where} \quad \text{[diagram of a circle with a point } \theta(t) \text{ and an arrow indicating motion]} \quad \text{i.e. we oscillate on the circle.}$$

So, for this to happen, we need

$$f(\theta + 2\pi) = f(\theta) \quad \text{i.e. } f \text{ is a } 2\pi \text{ periodic function.}$$

Note though,  $\theta = \theta(t)$ , and so  $\theta$  has its own "period", say  $T$  so that

$$\theta(t+T) = \theta(t) + 2\pi \quad ? \quad \text{So } \theta \text{ is not itself actually periodic.}$$

$$\hookrightarrow f(\theta(t+T)) = f(\theta(t) + 2\pi) = f(\theta(t))$$

To compute the period, we note that

$$\frac{d\theta}{dt} = f(\theta) \quad \rightarrow \quad \frac{d\theta}{f(\theta)} = dt \quad \rightarrow \quad \int_0^{2\pi} \frac{d\theta}{f(\theta)} = \int_0^T dt = T$$

Thus:

(2)

$$T = \int_0^{2\pi} \frac{d\theta}{f(\theta)}$$

Nice Example:

$$\dot{\theta} = \omega \longrightarrow \theta(t) = \omega t + \theta_0$$

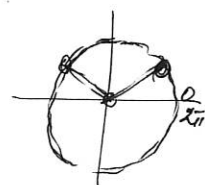
$$T = \int_0^{2\pi} \frac{d\theta}{\omega} = \frac{2\pi}{\omega}$$

Less Nice Example:

$$\dot{\theta} = \omega - a \sin(\theta), \quad \omega > 0, \quad a \geq 0$$

So before doing anything else:

$$FP's: \quad \omega - a \sin(\theta) = 0 \longrightarrow \sin(\theta) = \omega/a$$

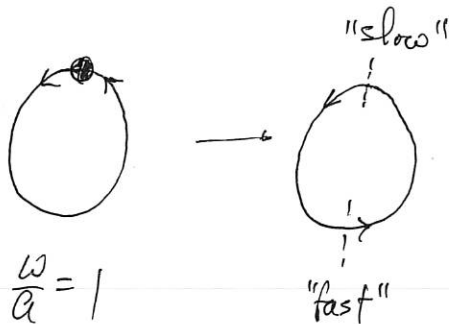
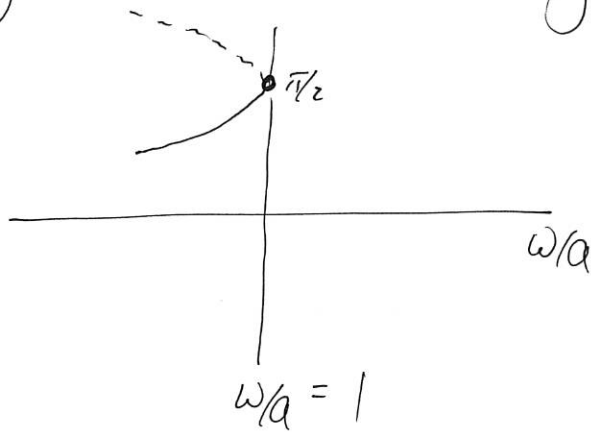


$$\hookrightarrow \left| \frac{\omega}{a} \right| < 1 \longrightarrow \theta_* = \begin{cases} \sin^{-1}\left(\frac{\omega}{a}\right) \\ \pi - \sin^{-1}\left(\frac{\omega}{a}\right) \end{cases}$$

$$sf6: \quad \partial_{\theta} f = -a \cos(\theta_*) = \begin{cases} -a(1 - \omega^2/a^2)^{1/2} \\ a(1 - \omega^2/a^2)^{1/2} \end{cases}$$

So, we get a Saddle-Node Bifurcation

(3)



$$\frac{\omega}{a} < 1$$

Too slow, and  
we come to a halt.

$$\frac{\omega}{a} > 1$$

$$\theta = \pi/2, \omega - a \sin(\theta) = \omega - a \rightarrow \text{slowest}$$

$$\theta = 3\pi/2, \omega - a \sin(\theta) = \omega + a \rightarrow \text{fastest}$$

Further, as  $\omega/a \gtrsim 1$ ,

we can make the "slow" portion take  
arbitrarily long.

So, honestly, Strogatz should have first said

(4)

$$\tau = t/t_s$$

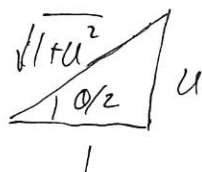
$$\hookrightarrow \frac{1}{t_s} \frac{d\theta}{d\tau} = \omega - a \sin(\theta)$$

$$\hookrightarrow \frac{d\theta}{d\tau} = t_s \omega - t_s a \sin(\theta)$$

If we let  $t_s a = 1 \rightarrow \frac{d\theta}{d\tau} = \tilde{\omega} - \sin(\theta)$ ,  $\tilde{\omega} = \frac{\omega}{a}$ .

$$\hookrightarrow \tilde{\tau} = \int_0^{2\pi} \frac{d\theta}{\tilde{\omega} - \sin(\theta)} = \int_{-\pi}^{\pi} \frac{d\theta}{\tilde{\omega} + \sin(\theta)}$$

$$\hookrightarrow u = \tan\left(\frac{\theta}{2}\right)$$



$$\hookrightarrow \sin(\theta/2) = \frac{u}{\sqrt{1+u^2}}; \quad \cos(\theta/2) = \frac{1}{\sqrt{1+u^2}}$$

$$\hookrightarrow \sin(\theta) = 2 \sin(\theta/2) \cos(\theta/2) = \frac{2u}{1+u^2}$$

$$du = \frac{1}{2} \sec^2\left(\frac{\theta}{2}\right) d\theta \quad \text{or} \quad d\theta = \frac{2}{(1+u^2)} du$$

and thus

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$$\frac{z}{1} = \int_{-\infty}^{\infty} \frac{\mathcal{L} du}{(1+u^2)(\tilde{\omega} + \mathcal{L}u/(1+u^2))}$$

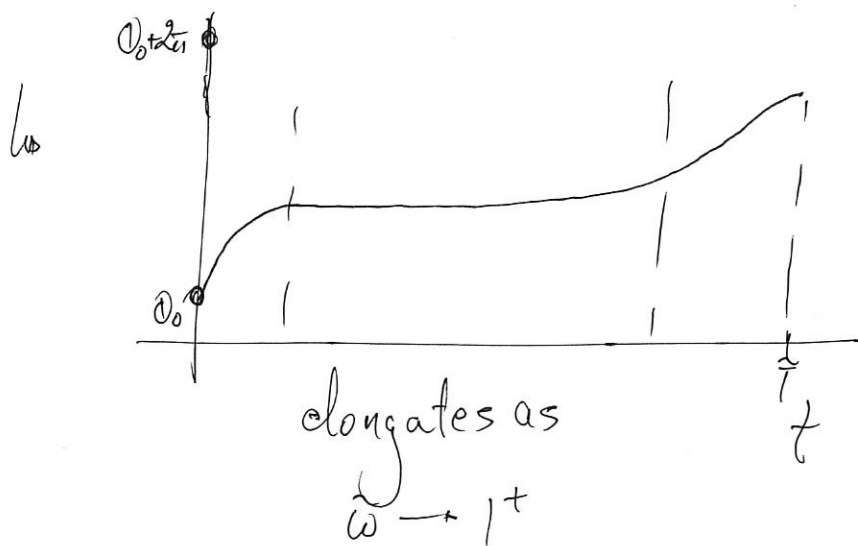
~~$= \int_{-\infty}^{\infty} \frac{\mathcal{L} du}{\tilde{\omega}(1+u^2) + \mathcal{L}u}$~~

$$= \int_{-\infty}^{\infty} \frac{\mathcal{L} du}{\tilde{\omega}(1+u^2) + \mathcal{L}u}$$

$$= \frac{2}{\sqrt{\tilde{\omega}}} \int_{-\infty}^{\infty} \frac{du}{u^2 + \tilde{\omega} - \frac{1}{\tilde{\omega}}} = \frac{2\pi}{\sqrt{\tilde{\omega}^2 - 1}}$$

So for  $\tilde{\omega} \sim 1$ :

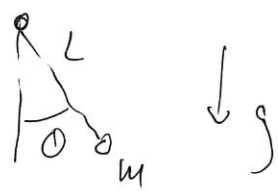
$$\frac{z}{1} \approx \frac{2\pi}{(1+\tilde{\omega})^{1/2}(-1+\tilde{\omega})^{1/2}} \sim \frac{\pi\sqrt{2}}{(-1+\tilde{\omega})^{1/2}} \text{ for } \tilde{\omega} \sim 1^+$$



Examples:

(6)

The Overdamped Pendulum:



damping

external torque

$$mL^2 \ddot{\theta} + b \dot{\theta} + mgL \sin(\theta) = \tau$$

$$\frac{L}{g} \ddot{\theta} + \frac{b}{mgL} \dot{\theta} + \sin(\theta) = \tilde{\tau}$$

$$\frac{L}{g\tau^2} \ddot{\theta} + \frac{b}{mgL\tau} \dot{\theta} + \sin(\theta) = \tilde{\tau} \quad (\tau = t/T)$$

$$\tau = \frac{b}{mgL} \quad \rightarrow \quad \frac{L}{g\tau^2} = mgL \left( \frac{mL^2}{b^2} \right) = \varepsilon \ll 1$$

$$\dot{\theta} \sim \tilde{\tau} - \sin(\theta)$$

So again, for  $\tilde{T} > 1$ , we get periodic motion, and (7)

for  $\tilde{T} < 1$ , the torque is not enough to overcome damping.

# Systems of Equations:

①



$$T = \frac{1}{2} m (l\dot{\theta})^2; \quad V = mgl \cos(\theta) (1 - \cos(\theta))$$

$$\hookrightarrow \ddot{\theta} + \frac{g}{l} \sin(\theta) = 0$$

$$\text{So if } \theta \approx 0 \Rightarrow \sin(\theta) \approx \theta$$

$$\hookrightarrow \ddot{\theta} + \omega^2 \theta = 0 \quad \text{where } \omega = \sqrt{\frac{g}{l}}$$

$$\text{or } \boxed{\text{mass } m} \xrightarrow{k} \ddot{x} + \frac{k}{m} x = 0 \rightarrow \omega = \sqrt{\frac{k}{m}}$$

$$\hookrightarrow \ddot{x} + \omega^2 x = 0$$

So, to make this 1<sup>st</sup> order, we let  $v = \dot{x}$

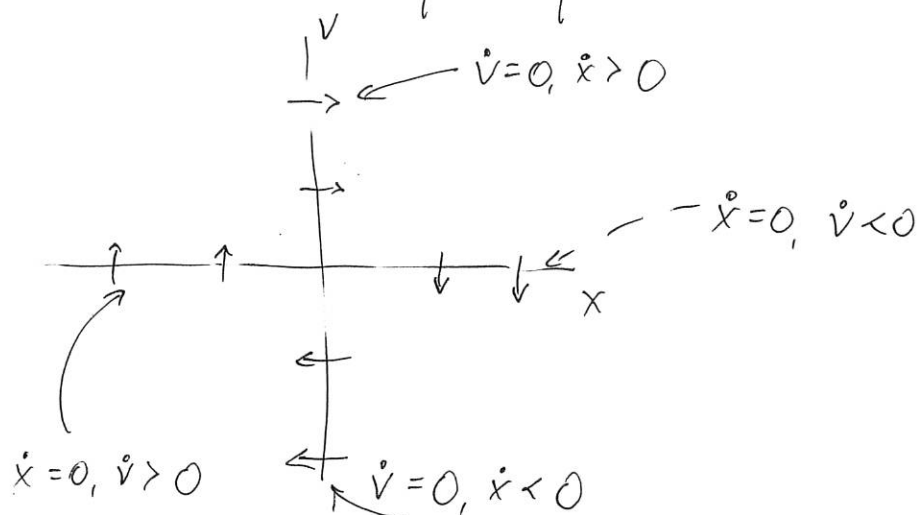
$$\hookrightarrow \begin{cases} \dot{x} = v \\ \dot{v} = -\omega^2 x \end{cases}$$

$$\hookrightarrow \frac{d}{dt} \begin{pmatrix} x \\ v \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ -\omega^2 & 0 \end{pmatrix} \begin{pmatrix} x \\ v \end{pmatrix}$$

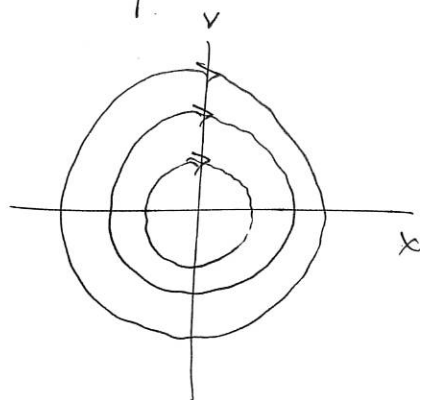


We track motion in the phase-plane:

(2)



As we will see, the paths  $(x(t), v(t))$  are closed circles



So, we in general want to be able to understand how to solve

$$\frac{d}{dt} \begin{pmatrix} x \\ v \end{pmatrix} = \begin{pmatrix} a & b \\ c & d \end{pmatrix} \begin{pmatrix} x \\ v \end{pmatrix} ; \begin{pmatrix} x(0) \\ v(0) \end{pmatrix} = \begin{pmatrix} x_0 \\ v_0 \end{pmatrix}$$

or

$$\frac{d}{dt} \bar{x}' = A \bar{x}', \quad \bar{x}' \in \mathbb{R}^2, \quad \bar{x}'(0) = \bar{x}'_0$$

To do this, we are going to "diagonalize"  $A$  i.e.

(3)

① Find eigenvectors/eigenvalues  $A\vec{v}_1 = \lambda_1 \vec{v}_1$ ,  $A\vec{v}_2 = \lambda_2 \vec{v}_2$

note: we can run into trouble when  $\lambda_1 = \lambda_2$ , will get to that.

$$\hookrightarrow \text{Let } V = (\vec{v}_1 | \vec{v}_2)$$

$$\begin{aligned} \hookrightarrow AV &= (A\vec{v}_1 | A\vec{v}_2) = (\lambda_1 \vec{v}_1 | \lambda_2 \vec{v}_2) \\ &= (\vec{v}_1 | \vec{v}_2) \begin{pmatrix} \lambda_1 & \\ & \lambda_2 \end{pmatrix} \end{aligned}$$

$$\hookrightarrow AV = V \underline{1} ; \underline{1} = \begin{pmatrix} \lambda_1 & \\ & \lambda_2 \end{pmatrix}$$

$$\hookrightarrow A = V \underline{1} V^{-1}$$

② in  $\dot{\vec{x}} = A\vec{x} = V \underline{1} V^{-1} \vec{x}$

$$\hookrightarrow V^{-1} \dot{\vec{x}} = \underline{1} V^{-1} \vec{x}$$

$$\hookrightarrow \frac{d}{dt} (V^{-1} \vec{x}) = \underline{1} (V^{-1} \vec{x})$$

$$\text{Let } \bar{y} = V^{-1} \bar{x}$$

(4)

$$\hookrightarrow \frac{d\bar{y}}{dt} = A \bar{y} \rightarrow \begin{pmatrix} \dot{y}_1 \\ \dot{y}_2 \end{pmatrix} = \begin{pmatrix} \lambda_1 & 0 \\ 0 & \lambda_2 \end{pmatrix} \begin{pmatrix} y_1 \\ y_2 \end{pmatrix} = \begin{pmatrix} \lambda_1 y_1 \\ \lambda_2 y_2 \end{pmatrix}$$

$$\hookrightarrow \dot{y}_1 = \lambda_1 y_1 ; \quad \dot{y}_2 = \lambda_2 y_2$$

So diagonalization reduces the problem to two very simple ODE's.

$$\hookrightarrow y_1(t) = y_{1,0} e^{\lambda_1 t} ; \quad y_2(t) = y_{2,0} e^{\lambda_2 t}$$

(5) So  $\bar{y}(t) = y_{1,0} \begin{pmatrix} 1 \\ 0 \end{pmatrix} e^{\lambda_1 t} + y_{2,0} \begin{pmatrix} 0 \\ 1 \end{pmatrix} e^{\lambda_2 t}$

To go back to  $\bar{x}(t)$  we note that

$$\bar{y}_0 = V^{-1} \bar{x}_0$$

$$\bar{x}(t) = V \bar{y}(t) = V \left\{ y_{1,0} \begin{pmatrix} 1 \\ 0 \end{pmatrix} e^{\lambda_1 t} + y_{2,0} \begin{pmatrix} 0 \\ 1 \end{pmatrix} e^{\lambda_2 t} \right\}$$

$$\cancel{V} V \begin{pmatrix} 1 \\ 0 \end{pmatrix} = (\vec{v}_1 | \vec{v}_2) \begin{pmatrix} 1 \\ 0 \end{pmatrix} = \vec{v}_1 ; \quad (\vec{v}_1 | \vec{v}_2) \begin{pmatrix} 0 \\ 1 \end{pmatrix} = \vec{v}_2$$

↳

$$\vec{x}(t) = \gamma_{1,0} \vec{v}_1 e^{\lambda_1 t} + \gamma_{2,0} \vec{v}_2 e^{\lambda_2 t}, \quad \vec{y}_0 = V^{-1} \vec{x}_0$$

(5)

↳ So all dynamics determined by eigenvectors, eigenvalues.

Tr / Det plane:

$$(A - \lambda I) \vec{v} = 0 \rightarrow \det \begin{pmatrix} a-\lambda & b \\ c & d-\lambda \end{pmatrix} = 0$$

$$\rightarrow \lambda^2 - \text{tr}(A)\lambda + \det(A) = 0$$

$$\text{tr}(A) = a+d; \det(A) = ad-bc$$

$$\begin{aligned} \rightarrow \lambda^* &= \frac{1}{2} \left[ \text{tr}(A) \pm \left( \text{tr}(A)^2 - 4\det(A) \right)^{1/2} \right] \\ &= \frac{1}{2} \left[ \tau \pm \left( \tau^2 - 4\Delta \right)^{1/2} \right] \end{aligned}$$

$\Delta < 0$  / saddle:

(6)

$$\Delta < 0 \rightarrow \tau^2 - 4\Delta > \tau^2 \rightarrow (\tau + (\tau^2 - 4\Delta)^{1/2})(\tau - (\tau^2 - 4\Delta)^{1/2}) < 0$$

i.e. opposite signs

$$\hookrightarrow \text{if } \vec{x}(t) = \gamma_{1,0} \vec{v}_1 e^{\lambda_1 t} + \gamma_{2,0} \vec{v}_2 e^{\lambda_2 t}$$

$\hookrightarrow$  one direction represents growth, the other decay

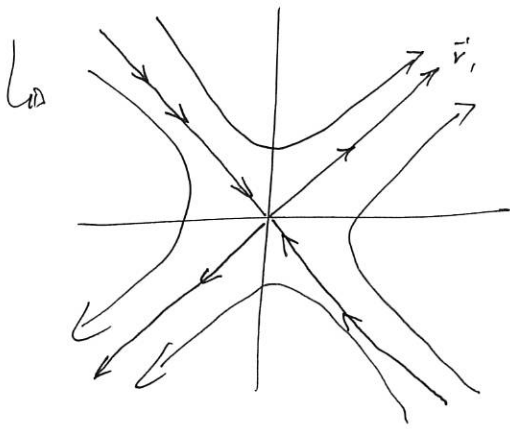
$$\begin{pmatrix} \dot{x} \\ \dot{y} \end{pmatrix} = \begin{pmatrix} 1 & 1 \\ 4 & -2 \end{pmatrix} \begin{pmatrix} x \\ y \end{pmatrix}$$

$$\hookrightarrow \tau = -1; \Delta = -6$$

$$\hookrightarrow \lambda = \frac{1}{2} (1 \pm 5) = 3, -2$$

$$\lambda = 3 \rightarrow \begin{pmatrix} -2 & 1 \\ 0 & 0 \end{pmatrix} \begin{pmatrix} 0 \\ 0 \end{pmatrix} \rightarrow \vec{v} = \begin{pmatrix} 1 \\ 2 \end{pmatrix}$$

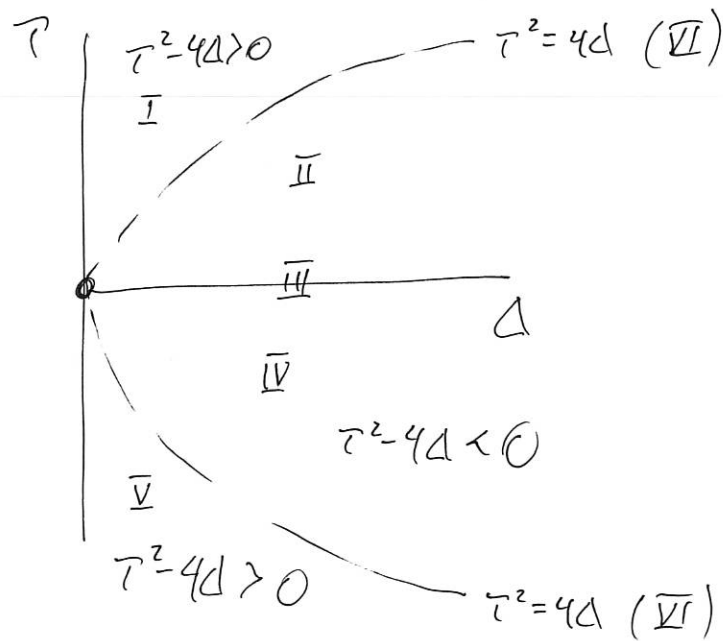
$$\lambda = -2 \rightarrow \begin{pmatrix} 3 & 1 \\ 0 & 0 \end{pmatrix} \begin{pmatrix} 0 \\ 0 \end{pmatrix} \rightarrow \vec{v} = \begin{pmatrix} 1 \\ -3 \end{pmatrix}$$



i.e. we have a "saddle"

$$\Delta > 0 : \quad \tau^2 - 4\Delta > 0, \quad \tau^2 - 4\Delta = 0, \quad \tau^2 - 4\Delta < 0$$

"saddle"



$$\text{I: } \tau^2 - 4\Delta > 0, \tau > 0$$

$$\text{II: } \tau^2 - 4\Delta < 0, \tau > 0$$

$$\text{III: } \tau^2 - 4\Delta < 0, \tau = 0$$

$$\text{IV: } \tau^2 - 4\Delta < 0, \tau < 0$$

$$\text{V: } \tau^2 - 4\Delta > 0, \tau < 0$$

Moving on: Phase-Plane for 2-D nonlinear systems

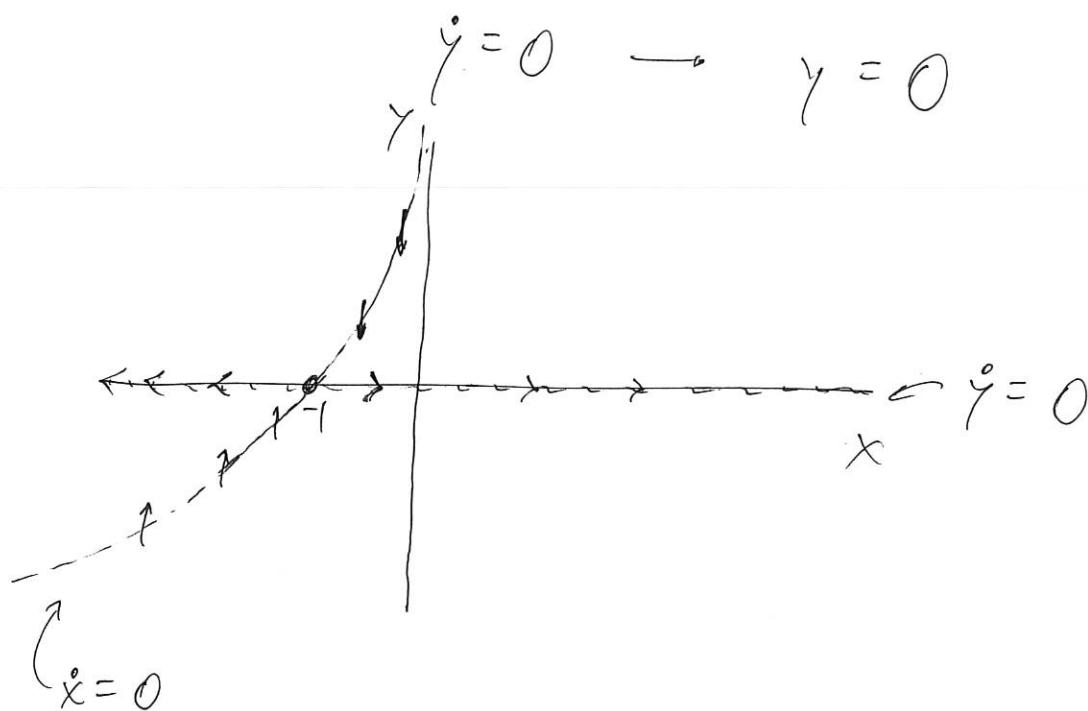
(1)

$$\dot{x} = x + e^{-y}$$

$$\dot{y} = -y$$

↳ Nullclines:  $\dot{x} = 0 \rightarrow x = -e^{-y}$

$$\dot{y} = 0 \rightarrow y = 0$$



So, from this we get our, the appearance of a fixed point at  $(-1, 0)$  which is clearly the intersection of null-clines. But we also get the appearance of what very much looks like a saddle.

To wit, for a non-linear system

(2)

$$\dot{x} = f(x, y)$$

$$\dot{y} = g(x, y)$$

Let  $(x_*, y_*)$  be a fixed point i.e.  $f(x_*, y_*) = g(x_*, y_*) = 0$

$$\hookrightarrow f(x, y) = f(x_*, y_*) + f_x(x_*, y_*)(x - x_*) + f_y(x_*, y_*)(y - y_*) + \dots$$

$$g(x, y) = g(x_*, y_*) + g_x(x_*, y_*)(x - x_*) + g_y(x_*, y_*)(y - y_*) + \dots$$

$$\hookrightarrow \text{if } \dot{x} = \frac{d}{dt}(x - x_*) ; \quad \dot{y} = \frac{d}{dt}(y - y_*)$$

$$\hookrightarrow \frac{d}{dt} \begin{pmatrix} x - x_* \\ y - y_* \end{pmatrix} \approx \begin{pmatrix} f_x(x_*, y_*) & f_y(x_*, y_*) \\ g_x(x_*, y_*) & g_y(x_*, y_*) \end{pmatrix} \begin{pmatrix} x - x_* \\ y - y_* \end{pmatrix}$$

$\hookrightarrow$  so by finding the Jacobian of our system, we can hopefully linearize the problem and determine the local dynamics around the fixed point.



So for  $f(x,y) = x + e^{-y}$ ;  $g(x,y) = -y$

(3)

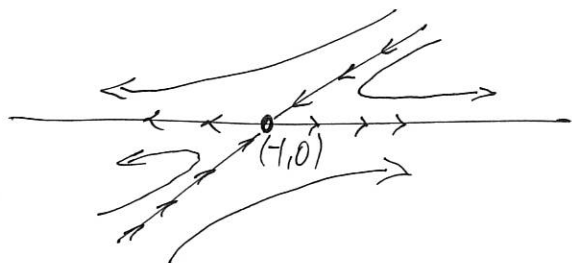
$$\hookrightarrow \begin{pmatrix} f_x & f_y \\ g_x & g_y \end{pmatrix} = \begin{pmatrix} 1 & -e^{-y} \\ 0 & -1 \end{pmatrix}$$

$$\hookrightarrow @ (-1, 0) \rightarrow \begin{pmatrix} 1 & -1 \\ 0 & -1 \end{pmatrix} \rightarrow \tau = 0; \Delta = -1$$

$$\hookrightarrow \lambda = \frac{1}{2} (0 \pm \sqrt{0+4})^{\frac{1}{2}} = \pm 1$$

$$\hookrightarrow \lambda = 1 \rightarrow \begin{pmatrix} 0 & -1 & | & 0 \\ 0 & 0 & | & 0 \end{pmatrix} \rightarrow \vec{v}' = \begin{pmatrix} 1 \\ 0 \end{pmatrix}$$

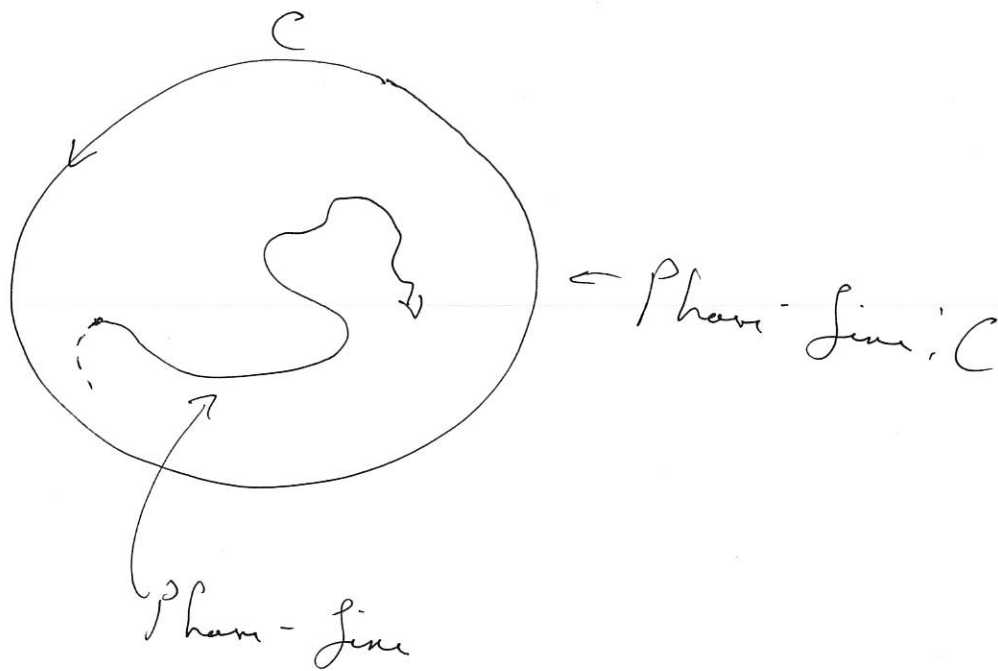
$$\lambda = -1 \rightarrow \begin{pmatrix} 2 & -1 & | & 0 \\ 0 & 0 & | & 0 \end{pmatrix} \rightarrow \vec{v}' = \begin{pmatrix} 1 \\ 2 \end{pmatrix}$$



rules of the Phase-Plane: For an autonomous system, existence and uniqueness of solutions imply: (4)

Phase Lines Never Cross

So:



If there is no fixed point within  $C$ , then the interior arc, in the limit, approach  $C$ . (Poincaré-Bendixson THM)

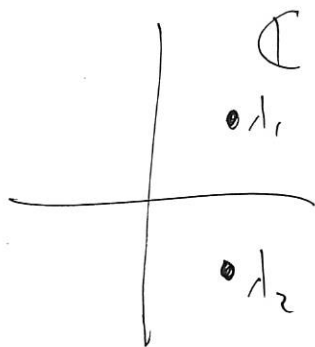
Hyperbolic or Non-Hyperbolic Fixed Points:

For  $\dot{x} = f(x, y)$ , let  $f(x_*, y_*) = g(x_*, y_*) = 0$ .  
 $\dot{y} = g(x, y)$

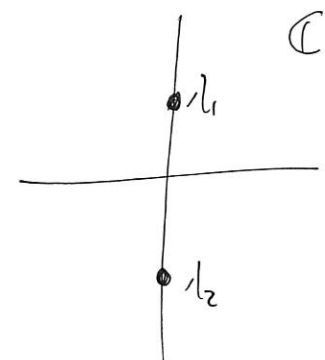
(5)

So, when we find the associated roots of the problem  
via  $\lambda = \frac{1}{2}(\tau \pm (\tau^2 - 4\Delta)^{1/2})$

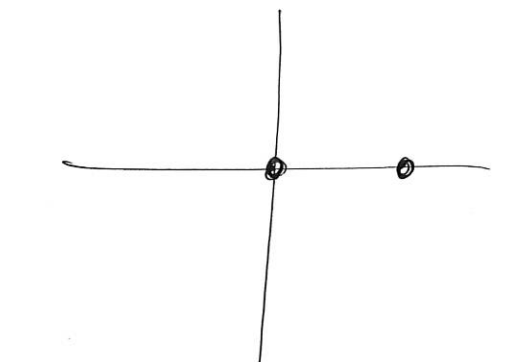
we say the fixed point is hyperbolic if:  $\operatorname{Re}(\lambda) \neq 0$   
non-hyperbolic if:  $\operatorname{Re}(\lambda) = 0$



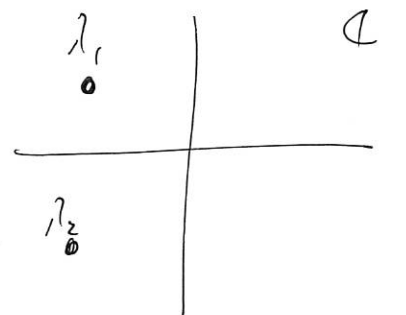
hyperbolic



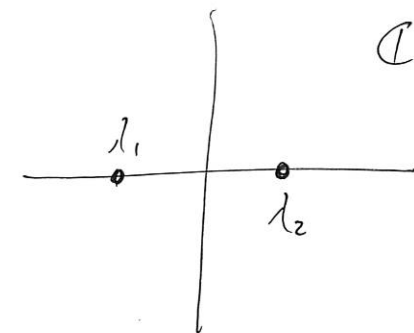
non-hyperbolic



non-hyperbolic



hyperbolic



hyperbolic

(6)  
Hyperbolicity means what happens in the linearization happens in a local neighborhood of the fixed point.

i.e. in  $\begin{cases} \dot{x} = x + e^{-y} \\ \dot{y} = -y \end{cases} \rightarrow (-1, 0) \text{ is a linearized saddle}$   
so the non-linear flow behaves like a saddle.

Pericious Example:

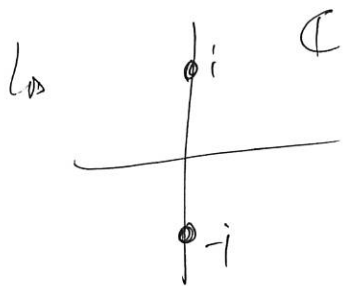
$$\dot{x} = -y + ax(x^2 + y^2)$$

$$\dot{y} = x + ay(x^2 + y^2)$$

Clearly  $(0, 0)$  is a fixed point

Jacobian: 
$$\begin{pmatrix} 3ax^2 + ay^2 & -1 + 2axy \\ 1 + 2axy & ax^2 + 3ay^2 \end{pmatrix}$$

$$@ (0, 0) \rightarrow \begin{pmatrix} 0 & -1 \\ 1 & 0 \end{pmatrix} \rightarrow \lambda^2 + 1 = 0 \rightarrow \lambda = \pm i$$



not hyperbolic ~~is~~ !

(7)

To analyze this, go to polar coordinates:

$$x = r \cos(\theta), \quad y = r \sin(\theta)$$

$$\dot{x} = \dot{r} \cos(\theta) - r \dot{\theta} \sin(\theta)$$

$$\dot{y} = \dot{r} \sin(\theta) + r \dot{\theta} \cos(\theta)$$

$\hookrightarrow$

$$\dot{r} = \dot{x} \cos(\theta) + \dot{y} \sin(\theta) = ar^3$$

$$r \dot{\theta} = \dot{y} \cos(\theta) - \dot{x} \sin(\theta) = r$$

$\hookrightarrow$

$$\dot{r} = ar^3; \quad \dot{\theta} = 1$$

$\hookrightarrow$

~~$$\int \frac{dr}{r^3} = \int a dt \rightarrow \left[ -\frac{1}{2} r^{-2} \right] = at + C$$~~