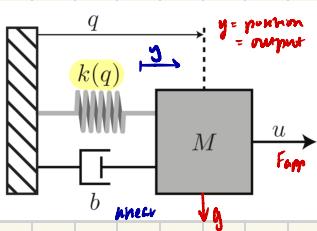
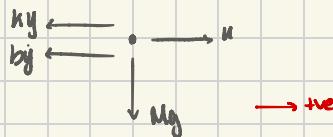


PSN #3

1) DE + Linearization + TF



spring restoring force: $k(q) = K(1 - \alpha^2 q^2)q$



$$F_{\text{net}} = ma$$

$$M\ddot{q} = u - kq - bq = u - K(1 - \alpha^2 q^2)q$$

$$u = M\ddot{q} + bq + K(1 - \alpha^2 q^2)q$$

$$u = M\ddot{q} + bq + Kq - Ka^2 q^3 \quad (\text{a})$$

linearize at $q_0 = 0.5m$ is controlled equilibrium

$$u = \cancel{M\ddot{q}} + \cancel{b\dot{q}} + Kq - Ka^2 q^3 \quad \leftarrow \text{system at rest}$$

$$u_0 = \cancel{Kq_0} - \cancel{Ka^2 q_0^3} \quad \leftarrow \dot{q}, \ddot{q} = 0$$

nonlinear term

$$\text{let } f(q) = Kq - Ka^2 q^3$$

$$f(q) = f(q_0) + \frac{df}{dq} \Big|_{q=q_0} (q - q_0) \rightarrow \text{Taylor Series (neglect higher order terms)}$$

$$f(q) - f(q_0) = \frac{df}{dq} \Big|_{q=q_0} (q - q_0) \quad \leftarrow \frac{df}{dq} \Big|_{q=q_0} = K - 3Ka^2 q^2 = K - 3Ka^2 (0.5)^2$$

$$\Delta f(q) = m \Big|_{q=q_0} \Delta q = K - 3Ka^2 (0.75) = K - Ka^2 (0.75) = (1 - 0.75a^2)K$$

$$\Delta u = Ma\ddot{q} + ba\dot{q} + K(1 - 0.75a^2)\Delta q \quad (\text{b})$$

\leftarrow linearized stiffness

transfer function: $\Delta Q(s)/\Delta U(s)$ using Laplace Transform

$$U(s) = M s^2 Q(s) - sQ(0) - \dot{Q}(0) + b s Q(s) - \cancel{\dot{Q}(0)} + K(1 - 0.75a^2) Q(s) \rightarrow 0 \text{ initial conditions}$$

$$U(s) = M s^2 Q(s) + b s Q(s) + K(1 - 0.75a^2) Q(s)$$

$$G(s) = \frac{Q(s)}{U(s)} = \frac{1}{Ms^2 + bs + K(1 - 0.75a^2)} \quad (\text{c})$$

$$2) \text{ seesaw: } (M_b + 3M_i)L\ddot{\theta}(t) = b(f(t) - M_i(g)) \cos(\theta(t))$$

linearity at $\theta = \pi/4$ & controlled equilibrium

$$\text{let } \theta = \Delta\theta + \pi/4 \rightarrow D = b(f_0 - M_i(g)) \cos(\pi/4)$$

$$f_0 = M_i(g)$$

$$\text{nonlinear part: } h(f, \theta) = b(f - M_i(g)) \cos(\theta) \quad \text{11th order Taylor expansion at } (f_0, \theta_0)$$

$$\begin{aligned} \Delta RMS &= \frac{\partial h}{\partial f} \Big|_{(f_0, \theta_0)} \Delta f + \frac{\partial h}{\partial \theta} \Big|_{(f_0, \theta_0)} \Delta \theta \\ &= b \cos(\theta_0) \Delta f + [-b(f_0 - M_i(g)) \sin(\theta_0)] \Delta \theta \\ &= 3\sqrt{2} \Delta f \end{aligned}$$

$$(M_b + 3M_i)L\ddot{\theta} = 3\sqrt{2} \Delta f \quad (\text{a})$$

transfer fn $\Delta\theta/\Delta f$

$$(M_b + 3M_i)L\ddot{\theta} = 3\sqrt{2} \Delta f \rightarrow \text{take LT, zero ICS}$$

$$(M_b + 3M_i)L^2 s^2 \Delta\theta(s) = 3\sqrt{2} \Delta f(s)$$

$$\frac{\Delta\theta(s)}{\Delta f(s)} = \frac{3\sqrt{2}}{(M_b + 3M_i)L^2 s^2} \quad (\text{b})$$

poles: double pole at $s=0$

zeroes: none

stable? NO, unstable (c)

BIBO Stability: if every bounded input \rightarrow bounded output

↳ since pole @ $s=0$, this yields an unbounded output

(system is BIBO unstable)

3) EM relay: $i_1 = V - R_1 i_1 - R_2 i_2 \quad \left. \begin{array}{l} \\ R_2 i_2 = \frac{1}{C} (i_1 - i_2) \\ m\ddot{x} = -\frac{K i_2^2}{x^2} - K(x - 0.1) - B\dot{x} \end{array} \right\}$ already linear

linearize at $x = -10m$ & controlled equilibrium

$$i_1 = i_2 = \dot{x} = \ddot{x} = 0 \rightarrow \text{equilibrium}$$

$$0 = V_0 - R_1 i_{1,0} - R_2 i_{2,0} \rightarrow V_0 = R_1 i_{1,0} + R_2 i_{2,0}$$

$$0 = \frac{1}{C} (i_{1,0} - i_{2,0}) \rightarrow i_{1,0} = i_{2,0} \equiv i_0$$

$$V_0 = i_0 (R_1 + R_2)$$

$$0 = -K \frac{i_0^2}{x_0^2} - K(x_0 - 0.1) \rightarrow \frac{K i_0^2}{x_0^2} = -K(x_0 - 0.1)$$

$$\text{for } x_0 = -10: \quad x_0^2 = 100, \quad x_0 - 0.1 = -10.1$$

$$i_0 = \frac{(10.1K)100}{K} = \frac{1010K}{K} \rightarrow i_0 = \pm \sqrt{1010K}$$

$$\Delta i_1 = i_1 - i_0, \quad \Delta i_2 = i_2 - i_0, \quad \Delta x = x - x_0, \quad \Delta V = V - V_0$$

linearize: $-K \frac{i_0^2}{x^2}$ about (i_0, x_0)

$$\left. \frac{\partial}{\partial i_2} \left(-K \frac{i_0^2}{x^2} \right) \right|_0 = \frac{-2K i_0}{x_0^2} = \frac{-2K i_0}{100} = \frac{-K i_0}{50}$$

$$\left. \frac{\partial}{\partial x} \left(-K \frac{i_0^2}{x^2} \right) \right|_0 = \frac{2K i_0^2}{x_0^3} = \frac{2K i_0^2}{1000} = -2.02K$$

$$\left. \frac{\partial [-K(x - 0.1)]}{\partial x} \right|_0 = -K \rightarrow \text{spring term derivative}$$

$$-2.02K + (-K) = -3.02K$$

$$m\ddot{x} = -\frac{K i_0}{50} A i_2 - B \dot{x} - 3.02K \Delta x \quad (a)$$

transfer in $X/V \rightarrow$ take LTS

$$(Ls + R_1) I_1 + R_2 I_2 = V$$

$$(R_2 s + 1/C) I_2 = (1/C) I_1 \rightarrow I_2 = \frac{1}{1 + CR_2 s} I_1$$

$$I_1 = \frac{V - R_2 I_2}{Ls + R_1} \rightarrow I_2 = \frac{V - R_2 I_1}{(Ls + R_1)(1 + CR_2 s)}$$

$$I_2 (Ls + R_1)(1 + CR_2 s) + R_2 I_2 = V$$

$$I_2 [(Ls + R_1)(1 + CR_2 s) + R_2] = V$$

$$(ms^2 + bs + 3.02k)X = -\frac{Ki_0}{50} I_2$$

$\hookrightarrow -\frac{Ki_0}{50} \left[\frac{V}{(Ls + R_1)(1 + CR_2 s) + R_2} \right]$

$$\frac{\Delta X(s)}{\Delta V(s)} = -\frac{Ki_0}{50} \left[\frac{1}{(Ls + R_1)(1 + CR_2 s) + R_2} \right] \frac{1}{(ms^2 + bs + 3.02k)} \quad (b)$$

4) Consider the feedback control system shown in Figure 2. Suppose that

$$P(s) = \frac{1}{s-1}, \quad C(s) = K_p$$

where $K_p > 0$ is a constant called the gain. This controller is called a proportional controller because the input $u(t)$ is proportional to the error $e(t)$.

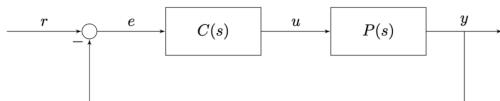


Figure 2: Feedback Control System for Problems 4-5

a. Is plant stable? ↗ pole in RHP

pole at $s=1 \rightarrow \text{Re}(p) > 0$, growth → unstable

$$U(s) = C(s) \cdot (R(s) - Y(s))$$

$$U(s) = K_p(R(s) - Y(s))$$

$$Y(s) = (R(s) - Y(s)) C(s) \cdot P(s)$$

$$Y(s) = \frac{K_p(R(s) - Y(s))}{s-1}$$

$$\hookrightarrow Y(s) = \frac{U(s)}{s-1} \quad // \text{sub into } T_{ry}$$

b. transfer $\rightarrow T_{ry}, T_{ru}$

$$T_{ry} = \frac{Y(s)}{R(s)} = \frac{K_p}{s-1 + K_p}$$

$$T_{ru} = \frac{U(s)}{R(s)}$$

$$\hookrightarrow Y(s) = \frac{U(s)}{s-1} \rightarrow U(s) = Y(s)(s-1)$$

$$T_{ru} = \frac{U(s)}{R(s)} = \frac{K_p(s-1)}{s-1 + K_p}$$

c. values of K_p for stability → stable if pole in LHP

$$T_{ry} = \frac{K_p}{s-1 + K_p} \quad s + K_p - 1 = 0$$

$$s = 1 - K_p$$

$$T_{ru} = \frac{K_p(s-1)}{s-1 + K_p} \quad 1 - K_p < 0 \rightarrow K_p > 1$$

d. tradeoff in choosing large K_p

settling time: $p = K_p - 1 \rightarrow$ as K_p increases, τ decreases

$$\tau = \frac{1}{K_p - 1} \quad \text{faster response, shorter settling time}$$

steady-state error: DC gain of T_{ry} is $\frac{K_p}{K_p - 1} \rightarrow$ as $K_p \rightarrow \infty$, gain approaches 1
output tracks reference correctly

tradeoff: larger K_p makes system faster and more accurate

requires larger initial control effort → could saturate/damage physical actuator

- 5) Consider the feedback control system shown in Figure 2. Suppose that

$$P(s) = \frac{1}{s-1}, \quad C(s) = K_p$$

where $K_p > 0$ is a constant called the gain. This controller is called a proportional controller because the input $u(t)$ is proportional to the error $e(t)$.

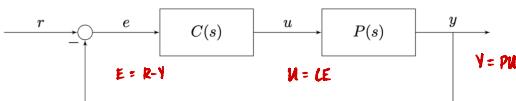


Figure 2: Feedback Control System for Problems 4-5

a. find transfer fn $T_{re}(s)$

$$T_{re}(s) \rightarrow E(s)/R(s)$$

$$\begin{aligned} E(s) &= R(s) - Y(s) \\ &= R(s) - P(s)U(s) \\ &= R(s) - P(s)C(s)E(s) \\ E(s)(1 + P(s)C(s)) &= R(s) \end{aligned}$$

$$\frac{E(s)}{R(s)} = \frac{1}{(1 + P(s)C(s))}$$

$$\text{characteristic equation: } 1 + P(s)C(s)$$

b. stability of PC combos

$$(i) P(s) = \frac{1}{s+1} \quad C(s) = \frac{-6}{s+2}$$

$$T_{re} = \frac{1}{1 + PC}$$

$$s = -1$$

stable

$$s = -2$$

stable

$$1 + \frac{1}{s+1} \cdot \frac{-6}{s+2} = 1 + \frac{-6}{(s+1)(s+2)} = 1 + \frac{-6}{s^2 + 3s + 2} = 0$$

$$s^2 + 3s + 2 = 0 \quad 1 = \frac{6}{s^2 + 3s + 2} \rightarrow s^2 + 3s + 2 = 6$$

$$(s+4)(s-1) = 0$$

$$s^2 + 3s - 4 = 0$$

$$s = -4, +1 \rightarrow \text{unstable}$$

$$(ii) P(s) = \frac{1}{s+1} \quad C(s) = \frac{1}{s+2}$$

$$T_{re} = \frac{1}{1 + PC}$$

$$s = -1$$

stable

$$s = -2$$

stable

$$1 + PC = 1 + \frac{1}{s^2 + 3s + 2} = 0$$

$$s^2 + 3s + 3 = 0 \quad \boxed{\text{Re}(s) < 0} \quad \text{stable}$$

$$s = \frac{-3 \pm \sqrt{9-12}}{2} = -1.5 \pm j\sqrt{3}$$

$$(iii) P(s) = \frac{1}{s-1} \quad C(s) = \frac{4s}{s-2}$$

$$T_{re} = \frac{1}{1 + PC}$$

$$s = 1$$

unstable

$$s = 2$$

unstable

$$1 + \frac{1}{s-1} \cdot \frac{4s}{s-2} = 1 + \frac{4s}{s^2 - 3s + 2} = 0$$

$$s^2 - 3s + 2 = -4s$$

$$s^2 + s + 2 = 0 \quad \boxed{\text{Re}(s) < 0} \quad \text{stable}$$

$$s = \frac{-1 \pm \sqrt{1-8}}{2} = -0.5 \pm j\sqrt{7}$$

$$\text{from P4: } \frac{Y(s)}{R(s)} = \frac{K_p}{s-1 + K_p}$$

$$Y(s) = (R(s) - Y(s))C(s)P(s)$$

$$Y(s) = E(s)C(s)P(s)$$

b) unit step response transfer fn matching

c) (i) $\frac{409}{6} \left(\frac{1}{s+3-20j} + \frac{1}{s+3+20j} \right)$ (i) $p = -3 \pm 20j$

$\text{Re}(p) = -3 \rightarrow \text{decay, stable}$

$\text{Im}(p) = \pm 20 \rightarrow \text{osc, } f = 20$

a) (ii) $\frac{2.5}{s+5} + \frac{5}{s+10}$

b) (iii) $\frac{3}{s+3}$ pole $s=-3$
 $\text{Re}(p) < 0 \rightarrow \text{decay}$

e) (iv) $\frac{109}{6} \left(\frac{1}{s+3-10j} + \frac{1}{s+3+10j} \right)$

d) (v) $\frac{-3}{s-3}$ pole $s=3$

(vi) $\text{Re}(p) > 0 \rightarrow \text{growth, unstable}$

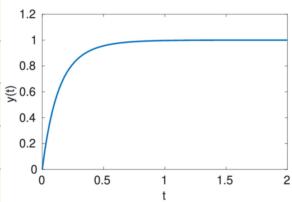


Figure 6: Option (d) for Problem 6

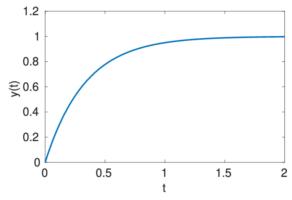


Figure 3: Option (a) for Problem 6

(i) $p = -3 \pm 20j$

$\text{Re}(p) = -3 \rightarrow \text{decay, stable}$

$\text{Im}(p) = \pm 20 \rightarrow \text{osc, } f = 20$

(iv) $p = -3 \pm 10j$

$\text{Re}(p) = -3 \rightarrow \text{decay, stable}$

$\text{Im}(p) = \pm 10 \rightarrow \text{osc, } f = 10$

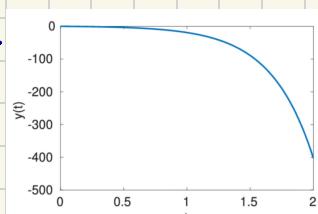


Figure 4: Option (b) for Problem 6

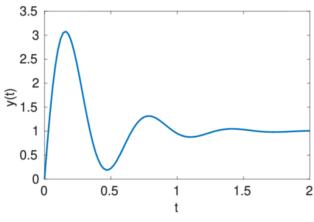


Figure 7: Option (e) for Problem 6

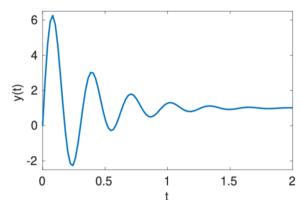


Figure 5: Option (c) for Problem 6