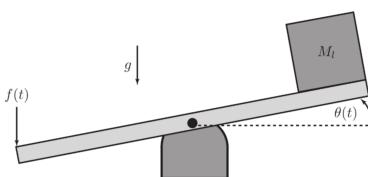


ANL H2

1) seesaw DEs



→ beam: length L

mass M_b

$$\text{moment of inertia: } \frac{1}{3}M_bL^2$$

$$\text{torque: } F_i = rF_i \sin\theta$$

\downarrow
distance from pivot

$$\Sigma T = I\ddot{\theta}$$

$$I_b = \frac{1}{3}M_bL^2$$

$$I_M = MR^2$$

$$T = rF_i \sin\theta$$

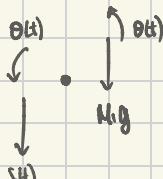
$$T_{\text{net}} = M_1g\cos\theta - M_2g\cos\theta$$

$$\Sigma T = I\ddot{\theta}$$

$$= r(F_i \sin\theta - M_2g)$$

$$= \frac{1}{2}(f(t) - M_2g) \cos\theta(t)$$

FBD:



$$I_{\text{tot}} = I_b + I_M$$

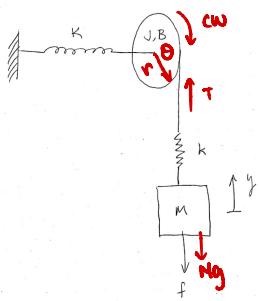
$$= \frac{1}{3}M_bL^2 + M_1(\frac{1}{2})^2$$

$$= \frac{1}{3}M_bL^2 + \frac{1}{4}M_1L^2$$

$\Sigma T = I\ddot{\theta}$ → Newton's 2nd Law for Rotational Motion

$$(\frac{1}{3}M_bL^2 + \frac{1}{4}M_1L^2)\ddot{\theta}(t) = \frac{1}{2}L(f(t) - M_2g) \cos\theta(t)$$

2) DE for mechanical system



r, θ = rad. angle of pulley

K = torsion spring constant, $T_K = K\theta$

D = damping force, $T_D = D\dot{\theta}$

moment of inertia = J

spring constant = K

M = mass, F = force (-y dir)

$$y = -r\theta$$

$$T = Ky$$

$$\dot{y} = -r\dot{\theta}$$

$$My = f + Mg - Ky \rightarrow f_{\text{net}} = ma$$

$$\ddot{y} = -r\ddot{\theta}$$

$$T = f + Mg - M\ddot{y}$$

$$\text{pulley torque: } \Sigma T = I\ddot{\theta} = J\ddot{\theta}$$

Angular acceleration = $\ddot{\theta}$

$$J = T_S + T_D + T_K$$

$$J\ddot{\theta} = rT - K\theta - D\dot{\theta}$$

$$J\ddot{\theta} = r(f + Mg - M\ddot{y}) - K\theta - D\dot{\theta}$$

3) electromagnetic relay

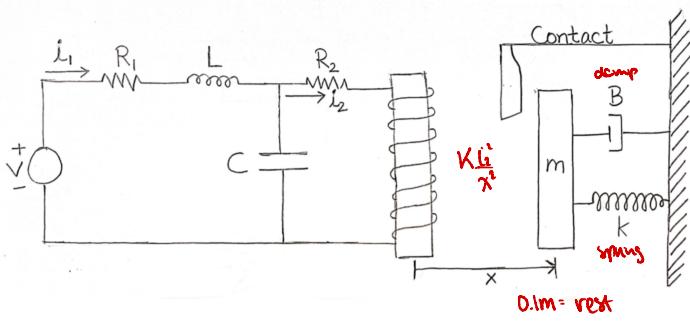


Figure 3: Electromagnetic Relay for Problem 3

mass:

$$\begin{array}{c} \leftarrow \\ F_m \end{array} \quad \begin{array}{c} \longrightarrow \\ B\dot{x} \end{array}$$

$$F_m = \frac{Kx^2}{x^2}, \quad K > 0$$

$$\Delta x = x - x_e$$

$$L x_e = 0.1m \quad F_0 = k(x_e - 0.1)$$

$$m\ddot{x} = F_m - B\dot{x} - k(x - 0.1)$$

$$L = \frac{Kx_e^2}{x_e^2}$$

$$m\ddot{x} = \frac{Kx_e^2}{x_e^2} - B\dot{x} - k(x - 0.1)$$

$$NVL: \quad V = R_1 i_1 + L \dot{i}_1 + R_2 i_2$$

$$0 = \frac{1}{C} \int (i_2 - i_1) dt + R_2 i_2$$

↓ diff

$$0 = \frac{1}{C} (i_2 - i_1) + R_2 i_2 \rightarrow i_1 = i_2 + C R_2 i_2$$

$$M_1 \ddot{y}_1 = B(y_2 - y_1) - K_1 y_1 \rightarrow \text{find transfer fn}$$

$$M_2 \ddot{y}_2 = f - K_2 y_2 - B(y_2 - y_1) \quad \leftarrow Y_1(s)/F(s)$$

Using Laplace Transforms: $\ddot{y} = s^2 Y(s)$ $\dot{y} = sY(s)$ $y = Y(s)$ $f = F(s)$

$$M_1(s^2 Y_1(s)) = B(sY_2(s) - sY_1(s)) - K_1 Y_1(s)$$

$$M_2(s^2 Y_2(s)) = F(s) - K_2 Y_2(s) - B(sY_2(s) - sY_1(s))$$

$$M_1 s^2 Y_1(s) = BSY_2(s) - BSY_1(s) - BK_1 Y_1(s)$$

$$BSY_2(s) = M_1 s^2 Y_1(s) + BSY_1(s) + BK_1 Y_1(s)$$

$$Y_2(s) = \frac{M_1 s^2 Y_1(s) + BSY_1(s) + BK_1 Y_1(s)}{BS} = \frac{Y_1(s)(M_1 s^2 + BS + K_1)}{BS}$$

$$F(s) = M_2 s^2 Y_2(s) + K_2 Y_2(s) + BS Y_2(s) - BS Y_1(s)$$

$$F(s) = Y_2(s)(M_2 s^2 + K_2 + BS) - BS Y_1(s)$$

$$\downarrow F(s) = \frac{[Y_1(s)(M_1 s^2 + BS + K_1)]}{BS} (M_2 s^2 + K_2 + BS) - BS Y_1(s)$$

$$= Y_1(s) \left\{ \frac{(M_1 s^2 + BS + K_1)(M_2 s^2 + K_2 + BS)}{BS} - \frac{BS}{BS} \right\}$$

$$\frac{Y_1(s)}{F(s)} = \frac{BS}{(M_1 s^2 + BS + K_1)(M_2 s^2 + K_2 + BS) - B^2 s^2}$$

5) $M\ddot{y}_2 = -k_s(y_2 - \hat{y}_1) - Mg - b_s(\dot{y}_2 - \dot{\hat{y}}_1)$
 $m\ddot{y}_1 = k_s(y_2 - y_1) + b_s(\dot{y}_2 - \dot{y}_1) - k_t(y_1 - \hat{y}_1) - mg - b_t(\dot{y}_1 - \dot{\hat{y}}_1) \rightarrow \text{find } \Delta y_2(s)/y_0(s)$

choose: $\Delta y_1 = y_1 - \hat{y}_1$, $\Delta y_2 = y_2 - \hat{y}_2$
 L const

$$\Delta y_1 = y_1, \quad \Delta y_2 = y_2$$

$$M\ddot{y}_2 = -k_s(y_2 - y_1) - Mg - b_s(\dot{y}_2 - \dot{y}_1) \quad \text{set } \bar{y}, \bar{y} = 0$$

$$0 = -k_s(y_2 - y_1) - Mg \quad y_0 = 0 \text{ (rest/equilibrium)}$$

$$\therefore y_2 = \Delta y_2 + \hat{y}_1$$

$$Mg = -k_s(\hat{y}_2 - \hat{y}_1)$$

$$\begin{aligned} m\ddot{y}_1 &= k_s(y_2 - y_1) + b_s(\dot{y}_2 - \dot{y}_1) - k_t(y_1 - \hat{y}_1) - mg - b_t(\dot{y}_1 - \dot{\hat{y}}_1) \\ mg &= k_s(\hat{y}_2 - \hat{y}_1) - k_t(\hat{y}_1) \\ mg &= -Mg - k_t\hat{y}_1 \end{aligned}$$

$$\hat{y}_1 = \frac{mg + Mg}{-k_t} = -\frac{(m+M)g}{k_t}$$

$$Mg = -k_s(\hat{y}_2 - \hat{y}_1)$$

$$\hat{y}_2 = -\frac{Mg + \hat{y}_1}{k_s} = -\frac{Mg}{k_s} - \frac{(m+M)g}{k_s} \quad \hat{y}_2 = \hat{y}_1 - \frac{Mg}{k_s}$$

time domain eqns: $M_2 \Delta \ddot{y}_2 = -k_s(\Delta y_2 - \Delta y_1) - b_s(\Delta \dot{y}_2 - \Delta \dot{y}_1)$
 $m \Delta \ddot{y}_1 = k_s(\Delta y_2 - \Delta y_1) + b_s(\Delta \dot{y}_2 - \Delta \dot{y}_1) - k_t \Delta y_1 - b_t \Delta \dot{y}_1 + k_t y_0 + b_t \dot{y}_0$

LTI model

transfer function: $A = k_s + b_s S$, $B = k_t + b_t S$ — dynamic stiffness terms
 $X_1 = \Delta y_1$, $X_2 = \Delta y_2$, $Y_0 = y_0$

$$MS^2 X_2 = -A(X_2 - X_1) \rightarrow (MS^2 + A)X_2 - AX_1 = 0$$

$$X_1 = \frac{(MS^2 + A)X_2}{A}$$

$$MS^2 X_1 = A(X_2 - X_1) - BX_1 + BY_0$$

$$(MS^2 + A + B)X_1 - AX_2 = BY_0 \quad \text{sum}$$

$$(MS^2 + A + B)(MS^2 + A)X_2 - AX_2 = BY_0$$

$$(MS^2 + A + B)(MS^2 + A)X_2 - A^2 X_2 = ABY_0$$

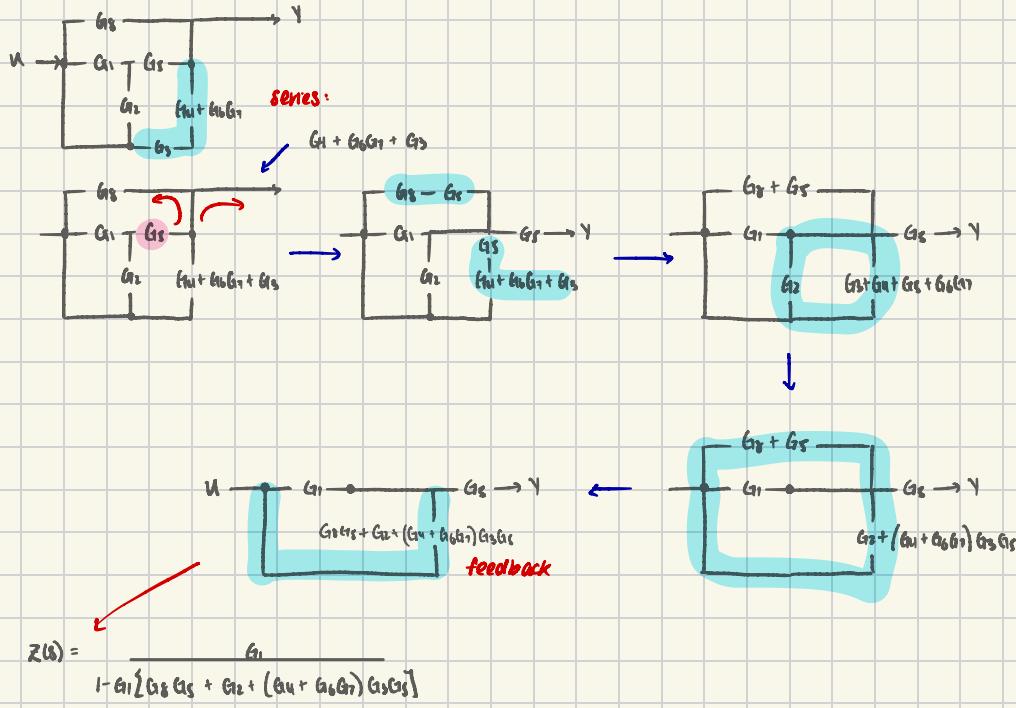
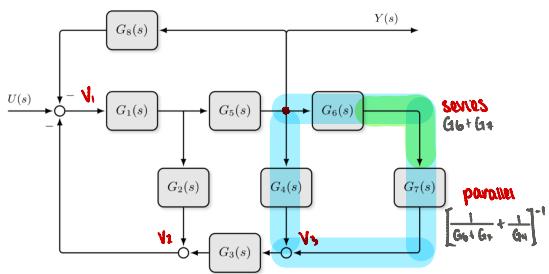
$$((MS^2 + A + B)(MS^2 + A) - A^2)X_2 = ABY_0 \rightarrow Y_0 = \frac{((MS^2 + A + B)(MS^2 + A) - A^2)X_2}{AB}$$

$X_2 = \Delta y_2$

$$\frac{\Delta y_2(s)}{Y_0(s)} = \frac{AB}{((MS^2 + A + B)(MS^2 + A) - A^2)} = \frac{(k_s + b_s s)(k_t + b_t s)}{(MS^2 + k_s + b_s s + k_t + b_t s)(MS^2 + k_s + b_s s) - (k_t + b_t s)^2}$$

Problem 6

Reduce the block diagram in Figure 4 to a single block relating the output $Y(s)$ to the input $U(s)$.
Do NOT use a system of algebraic equations to perform this reduction.

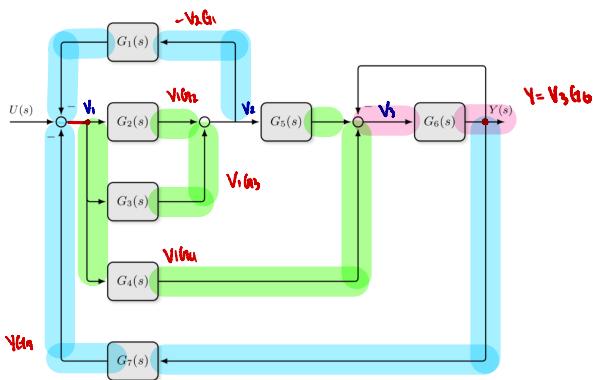


$$Z(s) = \frac{G_4}{1 - G_4 \left[G_3 G_4 + G_2 + (G_4 + G_6 G_3) G_3 G_4 \right]}$$

$$U(s) \rightarrow \frac{G_1 G_5}{1 - G_1 \left[G_3 G_4 + G_2 + (G_4 + G_6 G_3) G_3 G_4 \right]}$$

Problem 7

Reduce the block diagram in Figure 5 to a single block relating the output $Y(s)$ to the input $U(s)$.



$$V_1 = U - G_1 V_2 - G_3 Y = U - G_1 V_1 (G_2 + G_3) - G_3 Y$$

$$V_2 = V_1 (G_2 + G_3) = V_1 (G_2 + G_3)$$

$$Y = V_3 G_6 = (V_2 G_3 + V_1 G_1) G_6$$

$$= (V_1 G_2 G_3 + V_1 G_3 G_1 + V_1 G_1) G_6$$

$$Y = V_1 G_6 (G_2 G_3 + G_3 G_1 + G_1)$$

$$V_1 = \frac{Y}{G_6 (G_2 G_3 + G_3 G_1 + G_1)}$$

$$\text{L } V_1 = U - G_1 V_1 (G_2 + G_3) - G_3 Y$$

$$V_1 (1 + G_1 (G_2 + G_3)) = U - G_3 Y$$

$$V_1 = \frac{U - G_3 Y}{(1 + G_1 (G_2 + G_3))}$$

$$\frac{Y}{(G_2 G_3 + G_3 G_1 + G_1) G_6} (1 + G_1 (G_2 + G_3)) = U - G_3 Y$$

$$Y \cdot \frac{(1 + G_1 (G_2 + G_3)) + G_3 Y}{G_6 (G_2 G_3 + G_3 G_1 + G_1)} = U$$

$$Y \left[\frac{(1 + G_1 (G_2 + G_3)) + G_3 Y}{(G_2 G_3 + G_3 G_1 + G_1) G_6} \right] = U$$

$$Y = \left[\frac{(1 + G_1 (G_2 + G_3) + G_3 (G_2 G_3 + G_3 G_1 + G_1))}{(G_2 G_3 + G_3 G_1 + G_1) G_6} \right] U$$