

Aufgabe 1: Realiter:

1a) Wärme

$$b) \Rightarrow T_{HF} = \frac{T_2 - T_1}{\ln\left(\frac{T_2}{T_1}\right)} = \underline{\underline{293.12}}$$

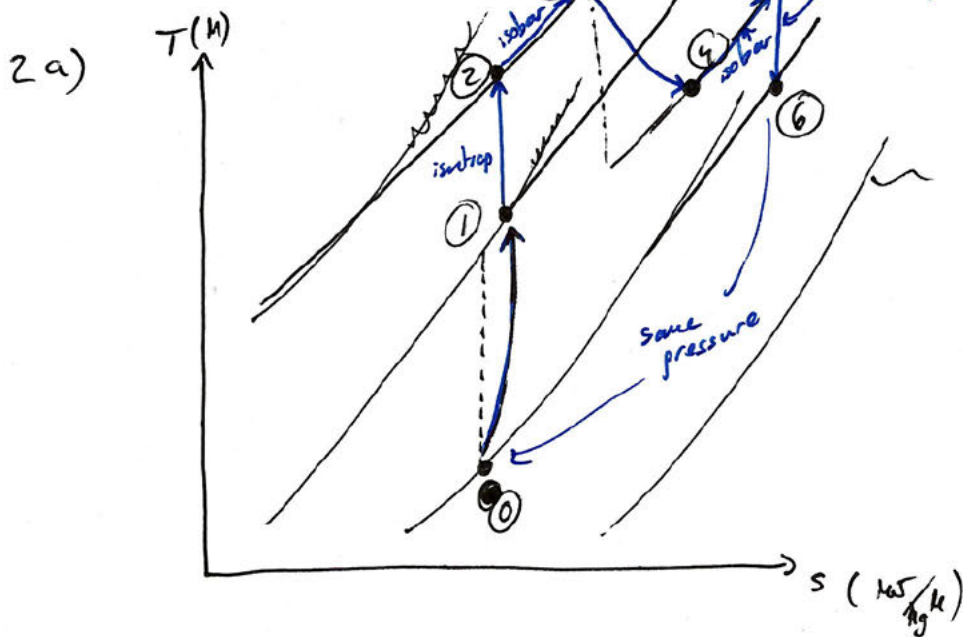
from $\int T ds \rightarrow$
from $s_{ex} - s_e$

$$c) \dot{S}_{erz} = \text{irrev. work} \\ =$$

d) stationär,

$$e) \Delta S_{12} = m(s_2 - s_1)$$

Aufgabe 2: Exergie am Turbinen:



isobaren are a little steep, they should probably be more like //

2 b)

$$w_5 = 220 \text{ m/s}$$

$$p_5 = 0.5 \text{ bar}$$

$$T_5 = 431.9 \text{ K}$$

$$w_6 = ?$$

$$p_6 = 0.191 \text{ bar}$$

$$T_6 = ?$$

Schubdüse adiab + reversibel \Rightarrow isentrop

$$\Rightarrow \text{ideales Gas: } \frac{T_6}{T_5} = \left(\frac{p_6}{p_5} \right)^{\frac{n-1}{n}} \quad \gamma = 1.4$$

$$\Rightarrow T_6 = 431.9 \cdot \left(\frac{0.191}{0.5} \right)^{\frac{1.4-1}{1.4}}$$

$$= \underline{\underline{328.075 \text{ K}}}$$

→ Energiebilanz:
(stationär)

$$0 = \dot{m} \left(h_e - h_a + \frac{w_e^2 - w_a^2}{2} \right) + \dot{Q} - \dot{W}$$

$\frac{c_p (T_6 - T_5)}{1.006 \cdot 431.9}$

Solve $f/w_a \Rightarrow w_a = \underline{\underline{220 \text{ m/s}}}$

cont'd
→

2c)

$$\Delta e_x = \dot{m}_{\text{ges}} \left[u - u_0 - T_0 (s - s_0) + p_0 (v - v_0) \right] - [u - u_0 - T_0 (s - s_0)]$$

2d)

$$\dot{E}_{x, \text{verl.}} = T_0 \cdot \dot{S}_{\text{erz}}$$

$$\Rightarrow e_{x, \text{verl.}} = T_0 \cdot S_{\text{erz}}$$

↳

Aufgabe 3: Perfektes Gas

3a)

Pressure consists of weight of fluid + weight + outside pressure:

$$p_{g,1} = \frac{mg}{A} + \frac{m_{EW}g}{A} + p_{amb}$$

$$= \frac{32 \cdot 9.81}{0.0314} + \frac{0.1 \cdot 9.81}{0.0314} + 1 \text{ bar}$$

$$= \underline{\underline{1.1002 \text{ bar}}}$$

$$\left[\begin{array}{l} \phi = 10 \text{ cm} \\ = 0.1 \text{ m} \end{array} \right]$$

$$\Rightarrow A = 0.0314 \text{ m}^2$$

Mass of gas: ideal Gas law $\Rightarrow pV = mRT$

$$\Rightarrow m = \frac{pV}{RT} = \frac{1.1002 \text{ bar} \cdot 0.00314 \text{ m}^3}{\frac{8.314 \cdot 10^3}{50} \cdot 773.15 \text{ K}} = 0.0027 \text{ kg} \Rightarrow \underline{\underline{\text{ca } 2.69 \text{ g}}}$$

3b)

Pressure is equal ^{to $p_{g,1}$}

$$\Rightarrow \underline{\underline{1.1002 \text{ bar} = p_{g,2}}}$$

(We still have the same weight pushing down and the system is closed so no gas escapes)

\rightarrow Zustand 2: Thermodyn. GGW \Rightarrow Equal temp. $T_{2,g} = T_{2,EW}$

~~\Rightarrow Pressure in EW $p_{EW,2}$~~

~~\rightarrow Perfect Gas \rightarrow w/ reversible process (reibungsfrei) \rightarrow polytropic state change~~

$$\frac{T_2}{T_1} = \dots$$

cat/d
 \rightarrow

3c) Übertragene Wärmeenergie:

→ Energiebilanz: Geschlossenes sys: $\underbrace{\Delta E}_2 = Q_{12} - \underbrace{W_{v,12}^{rev.}}_1$

→ ①: $W_{12}^{rev.} = \int_{V_1}^{V_2} p_{ausen} dV = p_{ausen} (V_2 - 0.00314 \text{ m}^3)$
 $= p_{g,1}$
 $= -223.41 \text{ J}$

V_2 : (using $T_{g,2} = 0.003^\circ\text{C}$)
 $pV = mRT$
 $= V_2 = \frac{m_g R_g T_{g,2}}{p_{ausen}}$
 $= 1.11 \text{ L} = 0.0011 \text{ m}^3$

②: $\Delta E = m(u_2 - u_1) = m(c_v(T_2 - T_1))$
 $= 0.00269 (0.633(273.153 - 773.15))$
 $= -851.38 \text{ J}$

$\Rightarrow \underline{Q_{12} = -1074.79 \text{ J}}$

2/2)

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3 d) Eisgehalt

Zustand ①: 0.6

Zustand ②: ?

+ $\approx 1075 \text{ J}$ Wärme from Gas
 ~~1075 J~~
~~Gas~~

Energiebilanz: $\Delta E = Q + W_{v,12}$ ^{incompressible}
(Geschlossenes Sys) $\hookrightarrow \approx 1075 \text{ J}$

$$\Rightarrow m(u_2 - u_1) = 1075 \text{ J}$$

0.1 kg

TAB 1: 0°C :

$$u_{\text{sl}} = -333.458$$

$$u_{\text{fl}} = -0.045$$

$$\Rightarrow u_2 = \frac{1.075 \text{ kJ}}{0.1 \text{ kg}} + (-200.0928)$$

$$= \cancel{10.75} - 189.3448$$

$$\Rightarrow u_1 = u_{\text{fl}} + x_{\text{Eis}}(u_{\text{sol}} - u_{\text{fl}})$$

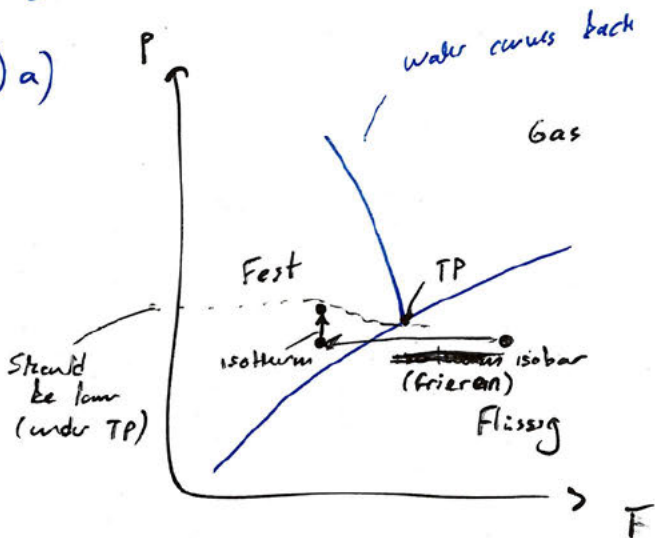
$$= \cancel{10.75} - 200.0928$$

$$\Rightarrow u_2 = u_{2,\text{fl}} + x_{2,\text{Eis}}(u_{2,\text{sol}} - u_{2,\text{fl}}) \quad @ \quad 0.003^\circ\text{C}$$

$\underbrace{-0.033} \quad \underbrace{-333.442} \quad \underbrace{-0.027} \rightarrow \text{all from TAB 1}$

$$\Rightarrow x_{2,\text{Eis}} = \frac{-189.245 + 0.033}{-333.442 + 0.033} = \underline{\underline{0.568}}$$

4) a)



b)

Energiebilanz um Verdichter: $0 = \dot{m} (h_e - h_a) + \dot{Q} - \dot{W}_k$

(stationär)

\Rightarrow Nur Dampf (vollständig verdampft)

Dampf-Flüssig Gemisch (8 bar)

28 W

↳ TAB AII: $h_g = 93.42 \text{ kJ/kg}$
 $h_g = 267.15 \text{ kJ/kg}$
 \Rightarrow

c)

$$x = 5$$

d)

$$\epsilon_k = \frac{|\dot{Q}_{zu}|}{|\dot{W}_t|} \rightarrow Q_H - Q_{ab}$$

e)

The temperature would go down to absolute 0 and stop there (theoretically, but it gets harder and harder to cool the ~~more~~ colder you get)