

A 1)

a) $\dot{Q}_{\text{aus}} = ?$

stationärer Fließprozess im Rotor

$$0 = \dot{m}_{\text{ein}} (h_{\text{ein}} - h_g) + \dot{Q}_R - \overset{20}{\dot{W}_t} - \dot{Q}_{\text{aus}}$$

$$\dot{Q}_{\text{aus}} = \dot{m}_{\text{ein}} (h_{\text{ein}} - h_g) + \dot{Q}_R = \underline{\underline{62,182 \text{ kW}}}$$

Tab A-2

$$h_c = h_f(70^\circ\text{C}) = 292,98 \text{ kJ/kg}$$

$$h_g = h_f(100^\circ\text{C}) = 419,04 \text{ kJ/kg}$$

$$b) \frac{\overline{T}}{T} = \frac{\int_1^2 T ds}{s_{g1} - s_{c1}} = \frac{h_g - h_c}{s_{g1} - s_{c1}} = \frac{126,06}{0,352} = \underline{\underline{358,125 \text{ K}}}$$

$$s_g = s_f(100^\circ\text{C}) \quad \text{Tab A-2}$$

$$s_{c1} = s_f(70^\circ\text{C}) = 0,5545 \text{ kJ/kgK}$$

$$s_{g1} = s_f(100^\circ\text{C}) = 1,3069 \text{ kJ/kgK}$$

c)

$$0 = m(s_c - s_g) + \frac{\dot{Q}}{T_g} + S_{\text{irr}}$$

~~m~~ q, b

$$S_{\text{irr}} = m(s_g - s_c) - \frac{\dot{Q}}{T_g} = 0,1056 - \frac{Q_{\text{irr}}}{T_g} = 0,1056 + \frac{61,182}{275}$$

auf zweites Blatt

$$= 0,3164 \text{ kJ/kgK}$$

$$= 316,39 \text{ J/kgK}$$

d) Halboffenes System

$$\Delta E = m_2 u_2 - m_1 u_1 = \Delta m(h)_{\text{in}} + Q$$

Tab A.2

$$(m_1 + \Delta m) u_2 - m_1 u_1 = \Delta m h_{\text{in}} + Q$$

$$m_1 u_2 - m_1 u_1 - Q = \Delta m(h_{\text{in}} - u_2)$$

$$\Delta m = \frac{m_1 u_2 - m_1 u_1 - Q}{h_{\text{in}} - u_2} = \underline{\underline{572,4 \text{ kg}}}$$

$$h_{\text{in}} = h_f(20^\circ\text{C}) = 83,96 \text{ kJ/kg}$$

$$u_1 = u(20^\circ\text{C}, x=0,005)$$

$$= u_f(20^\circ\text{C}) + x(u_g - u_f) = 425,38 \text{ kJ/kg}$$

$$u_2 = u_f(70^\circ) = 292,95 \text{ kJ/kg}$$

$$Q = 35 \cdot 10^3 \text{ kJ}$$

4)

e) $\Delta m_{12} = 3600 \text{ kg}$

Halboffizs S_1, S_2

$$\Delta S_{12} = m_2 S_2 - m_1 S_1 = (m_1 + m_{12}) S_2 - m_1 S_1 = \underline{\underline{1237,89 \text{ kJ}}}$$

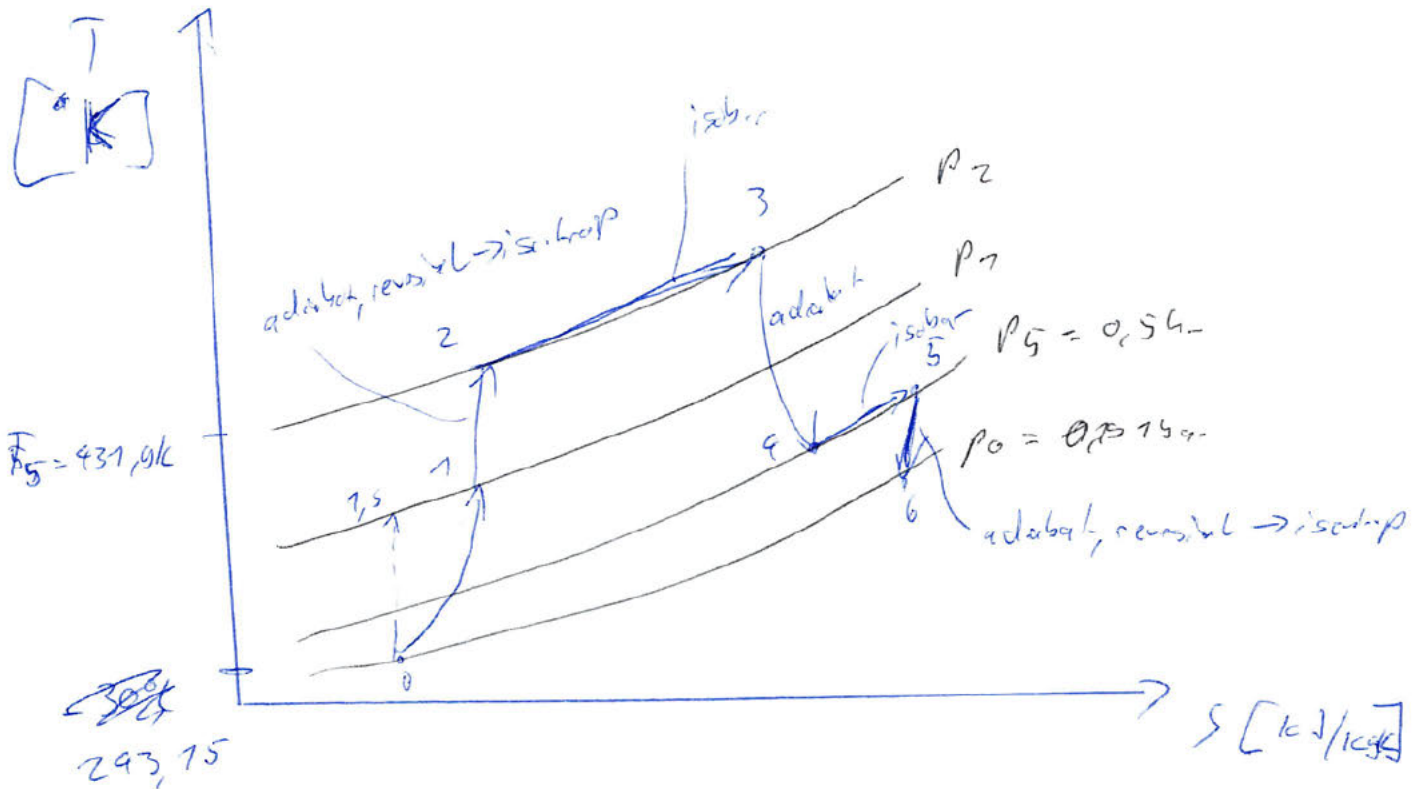
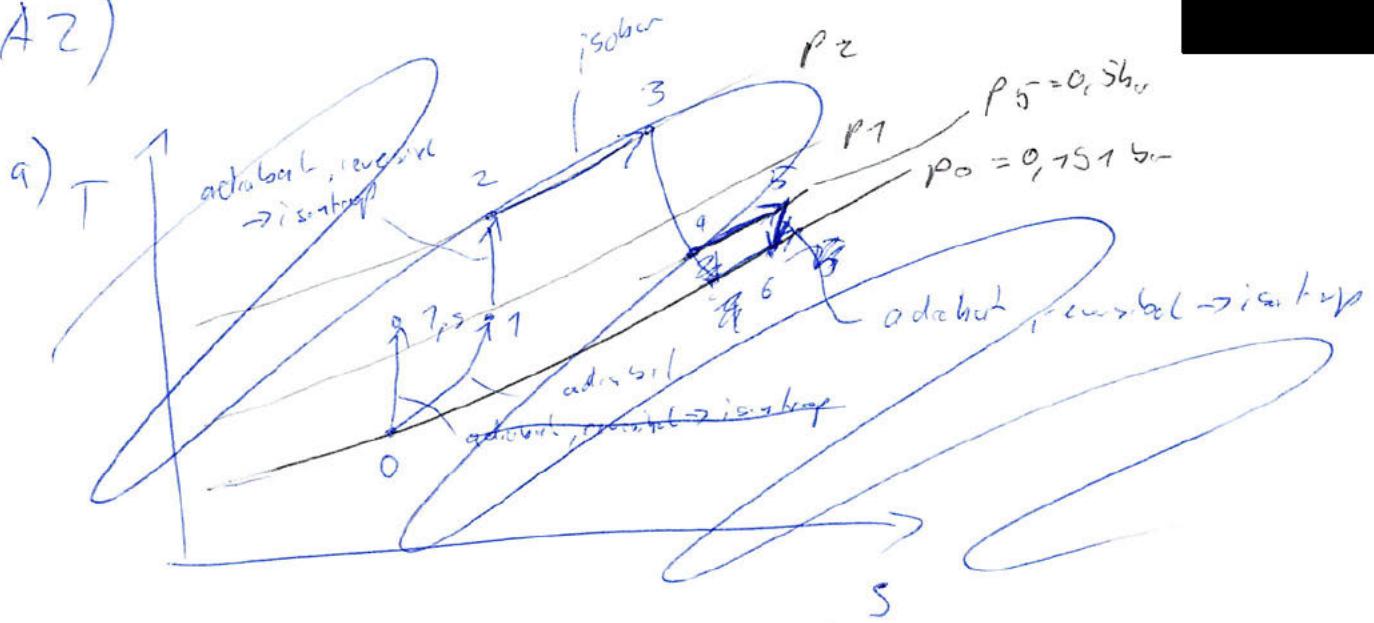
Tab A-2 bei 100°C

$$S_1 = S(100^\circ\text{C}, x=0,005) = S_f + x(S_g - S_f) = 7,33714 \text{ kJ/kg K}$$

$$S_2 = S_f(70^\circ) = 0,9545$$

$$c) \quad \bar{T} = \frac{\int_e^q T ds}{S_q - S_c} = \frac{\int_e^q T ds}{\int_{T_c}^{T_q} \frac{c}{T} dT} = \frac{\int_e^q T ds}{c \ln\left(\frac{T_q}{T_c}\right)}$$

$$= \frac{Q}{c \ln\left(\frac{T_q}{T_c}\right)} = \frac{c(\bar{T}_q - \bar{T}_c)}{c \ln\left(\frac{T_q}{T_c}\right)} = \frac{\bar{T}_q - \bar{T}_c}{\ln\left(\frac{T_q}{T_c}\right)}$$



$$b) \quad v_6 = ? \quad T_6 = ?$$

$$v_5 = 220 \text{ m/s}, \quad T_5 = 431,5 \text{ K}, \quad p_5 = 0,56 \text{ bar}$$

$$p_6 = p_0 = 0,1914 \text{ bar}$$

$$\text{isentrope} \quad s_5 = s_6$$

Stationärer Fließprozess

$$0 = \dot{m} \left(h_5 - h_6 + \frac{v_5^2 - v_6^2}{2} \right) + \dot{Q} - \dot{W}_r$$

↑ adiabatisch
↑ reversibel

$$h_5 - h_6 = c_p (T_5 - T_6) = +146,228 \text{ kJ/kg}$$

$$c_p = k \cdot c_v = n \cdot c_v = 7,4089 \text{ kJ/kgK}$$

$$\rightarrow R = c_p - c_v = 0,4024 \text{ kJ/kgK}$$

$$h_5 - h_6 + \frac{v_5^2 - v_6^2}{2} = 0$$

$$2(h_5 - h_6) + \frac{v_5^2}{\cancel{2}} = v_6^2$$

$$v_6 = \sqrt{\dots}$$

$$= 215,334 \text{ m/s}$$

$$= 1729,98 \text{ m/s}$$

$$\text{isentrope} \quad s_5 = s_6$$

$$0 = c_p \ln\left(\frac{T_6}{T_5}\right) - R \cdot \ln\left(\frac{p_6}{p_5}\right)$$

$$R \cdot \ln\left(\frac{p_6}{p_5}\right) = c_p \ln\left(\frac{T_6}{T_5}\right) \quad | \cdot e^{\dots}$$

$$e^{R \cdot \ln\left(\frac{p_6}{p_5}\right)} = e^{c_p \ln\left(\frac{T_6}{T_5}\right)}$$

$$\left(\frac{p_6}{p_5}\right)^R = \left(\frac{T_6}{T_5}\right)^{c_p} \rightarrow T_6 = T_5 \left(\frac{p_6}{p_5}\right)^{\frac{R}{c_p}} = \underline{\underline{328,075 \text{ K}}}$$

c)

$$\Delta e_{x, str} = e_{x, str 6} - e_{x, str 0} = h_6 - h_0 - T_0 (s_6 - s_0) + \frac{w_6^2}{2} - \frac{w_0^2}{2}$$

$$h_6 - h_0 = c_p (T_6 - T_0) = 119,608 \text{ kJ/kg}$$

$$s_6 - s_0 = c_p \ln\left(\frac{T_6}{T_0}\right) - R \ln\left(\frac{p_6}{p_0}\right) = 0,421905 \text{ kJ/kg}$$

$$\frac{w_0^2}{2} = 20 \text{ kJ/kg}$$

$$\frac{w_6^2}{2} = 1487,8 \text{ kJ/kg}$$

$$\Delta e_{x, str} = 1495,21 \text{ kW/kg}$$

d)

$$\dot{E}_{x, vol} = T_0 \cdot \dot{S}_{irr}$$

Stehender Flussprinzip

$$0 = \dot{m} (s_0 - s_6) + \frac{\dot{Q}}{T_G} + \dot{S}_{irr}$$

$$\dot{S}_{irr} = \overbrace{s_6 - s_0}^c - \frac{\dot{q}}{T_G} = 0,421905 \text{ kJ/kg} - \frac{1195}{1285} \text{ kJ/kg}$$

$$= 0,505 \text{ kJ/kg}$$

$$\dot{E}_{x, vol} = T_0 \cdot \dot{S}_{irr} = 243,15 \cdot 0,505 = 122,832 \text{ kW/kg}$$

A 3)

$$a) \quad R = \frac{\overline{R}}{M} = \frac{8319}{50} = 166,28 \text{ J/kg}$$

$$\frac{\downarrow p_0 \quad \downarrow F}{\uparrow p_{EW}}$$

$$p_{EW} = p_0 + F \cdot A = 1 \text{ bar} + m \cdot g \cdot \pi \left(\frac{D}{2}\right)^2 = 1,297 \text{ bar}$$

$$p_{g,1} = p_g = p_{EW} = \underline{\underline{1,297 \text{ bar}}}$$

$$\frac{\downarrow p_{EW}}{\uparrow p_g}$$

perfektes Gas:

$$V_1 = 3,19 \cdot 10^{-3} \text{ m}^3$$

$$pV = mRT$$

$$T_1 = 773,15 \text{ K}$$

$$m = \frac{p_1 V_1}{RT_1} = \underline{\underline{0,0312 \text{ kg}}}$$

b)

Da sich an den Umgebungsbedingungen nichts ändert, drücken immer noch 32 kg + 1 bar atmosphärischen Druck,

$$\text{dadurch ist } p_{g,2} = p_{g,1} = \underline{\underline{1,297 \text{ bar}}},$$

$$\Delta E = 0 = m_{EW} (u_{2g} - u_{1g}) + m_g (u_2 - u_1)$$

$$= m_{EW} (u_{2g} - u_{1g}) + m_g (c_v (T_2 - T_1))$$

c) $m_1 = m_2$

$$\Delta E = m_2 u_2 - m_1 u_1 = Q \quad \text{---} \quad \nearrow 0$$

$$Q = m_{EW} (u_2 - u_1)$$

incompressible

$$u_1 = u_1(1,247 \text{ bar}, 0^\circ\text{C}) = u(T=0^\circ\text{C}) =$$

$$u_2 = u(1,247 \text{ bar}, 0,003^\circ) = u(T=0,003^\circ)$$

$$Q = m_g (u_2 - u_1) = m_g u (T_2 - T_1) = \underline{\underline{-9,873 \text{ kJ}}}$$

d)

$$x_1 = 0,6 \quad 0^\circ\text{C}$$

$$u_1 = u(T=0^\circ\text{C}, x=0,6) = u_{\text{fcs}} + x(u_{\text{fcs}} - u_{\text{fcs}}) = -133,4702$$

$u_{2\text{fcs}}$

$$Q = m_{EW} (u_2 - u_1) = \underline{\underline{-9,873 \text{ kJ}}}$$

$$u_2 = \frac{Q}{m_{EW}} + u_1 = 98,73 \text{ kJ/kg} + (-133,4702) \text{ kJ/kg} = -34,7402 \text{ kJ/kg}$$

$$T_2 = 0,003^\circ\text{C}$$

$$x = \frac{u_2 - u_{\text{fcs}}}{u_{\text{fcs}} - u_{\text{fcs}}} = \underline{\underline{0,896081}}$$

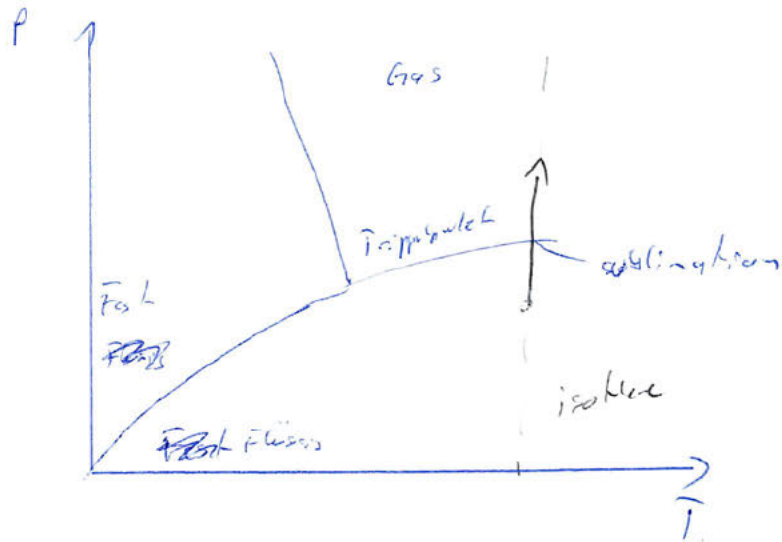
A9)

a) ~~$T_{\text{tripel}} = 0^\circ\text{C}$~~

$$T_{\text{tripel}} = 0^\circ\text{C}$$

$$T_{\text{subl}}(1\text{bar}) = -20^\circ\text{C}$$

Somit liegt T_i mit 10K über T_{subl} bei $T_i = -10^\circ\text{C}$



b) stationärer Flussprozess bei veränderlichen

$$0 = \dot{m}(h_2 - h_3) + \dot{Q} - \dot{W}_t$$

$$\dot{W}_k = \dot{m}(h_2 - h_3) \rightarrow \dot{m}_{139} = \frac{\dot{W}_k}{|h_2 - h_3|} = \frac{28 \cdot 10^{-3}}{|237,79 - 277,314|}$$

1 → 2 isotherme Verdampfung

$$T_v = T_i - 6 = -16^\circ\text{C} = T_2$$

Tab A-11

$$h_2 = h(x_2 = 1, T_2 = -16^\circ\text{C}) = 237,79 \text{ kJ/kg}$$

$$= 8,39 \cdot 10^{-4} \text{ kg/s}$$

$$= \underline{\underline{0,839 \text{ g/s}}}$$

adiabatisch reversibel $3 \rightarrow 9 \Rightarrow$ isobar $s_3 = s_9 = s_g(-16^\circ) = 0,9298$ Tab A-11

$$p_3 = 8 \text{ bar}$$

Tab A-11 8 bar

$$h_3 = h(p_3, s=0,9298) = h_{sat} + \frac{0,9298 - 0,5379}{0,9066 - 0,5379} (h_{g0} - h_{sat}) = 277,314 \text{ kJ/kg}$$

c) $x_9 = 0$ $p_3 = p_9 = 8 \text{ bar}$

Tab A-11

$$s_9 = s_1 = s_f(8 \text{ bar}) = 0,3459$$

$p_2 = 1,5748 \text{ bar}$ ← ablesen aus Tab A-10 bei $T_2 = -16^\circ\text{C}$

$p_1 = p_2 = 1,5748 \text{ bar}$, da isobar

Tab A-11 Interpolation

p	s_f	s_g	h_f	h_g
1,4	0,1035	0,5372	25,77	236,09
1,5748	0,1194	0,529894	29,1945	237,6882
1,6	0,1211	0,5295	29,78	237,97

$$x_1 = \frac{s - s_f}{s_g - s_f} = \underline{\underline{0,2795}}$$

A4)

$$d) \quad \varepsilon_{ic} = \frac{\dot{Q}_{zu}}{\dot{Q} \cdot W_t} = \frac{\dot{Q}_{ic}}{W_t} = \frac{125,3}{28} = \underline{\underline{4,476}}$$

beim Kondensieren

$$\dot{Q}_{zu} \quad 0 = \dot{m}(h_1 - h_2) + \dot{Q} - \dot{W}_t \rightarrow 0$$

$$\dot{Q}_{ic} = \dot{m}(h_2 - h_1) = 0,1253 \frac{\text{kg}}{\text{s}} = 125,3 \text{ W}$$

← Aufgabe C

$$h_1 = h(1,5284 \text{ bar}, x=0,2755) = h_f + x(h_g - h_f) = 87,4689$$

$$h_2 = 237,79 \text{ kJ/kg} \leftarrow \text{Aufgabe b}$$

e) Die Temperatur in innen würde isotherm weiter fallen.