

Recently we have determined that the exiting definitions for theoretical models, data definitions, general definitions and instance models are not entirely consistent. Moreover, the terminology, especially the terminology that combines “theory” and “model” is confusing.

Our new terminology will be based on the concept of different “levels” of theories. We currently have what we are calling context theories, background theories, theories and final theories. Context theories are the starting point for a given scientific model. These theories are not defined, but their implicit presence is necessary for completeness. Typical examples include theories for arithmetic operations, differentiation, integration, vector calculus, etc. Following the context theories we have background theories. Background theories are defined, but not derived. Typically they will be the general forms of the conservation equations, like conservation of thermal energy or momentum, and constitutive equations. Assumptions are often invoked by The level of detail used to define background theories can vary, since we do not always need the full theory; we sometimes just need to know that the theory exists. The background theories are refined into other theories by making assumptions, like plane stress, or linear elasticity, or isothermal material properties, or laminar flow, etc. These theories are combined and refined until the point where we have final theories. All theories are part of the documentation, but the final theories are the ones that will be transformed into code.

We also have data definitions. A data definition is a label for part of a theory. Assumptions will be maintained from the previous terminology.

Below is an attempt to clarify the new concept of different levels of theories by re-writing the relevant parts of the NoPCM model using the new conceptual model of theories.

1 Assumptions

Thermal-Energy-Only: The only form of energy that is relevant for this problem is thermal energy. All other forms of energy, such as mechanical energy, are assumed to be negligible. (RefBy: [BT:consThermE](#).)

Heat-Coeffs-Constant: All heat transfer coefficients are constant over time. (RefBy: [BT:nwtnCooling](#).)

Temp-Across-Tank: The water in the tank is fully mixed, so the temperature of the water is the same throughout the entire tank. (RefBy: [RT:rocTempSimp](#).)

Density-Constant-over-Volume: The density of water has no spatial variation; that is, it is constant over their entire volume. (RefBy: [RT:rocTempSimp](#).)

Specific-Heat-Constant-over-Volume: The specific heat capacity of water has no spatial variation; that is, it is constant over its entire volume. (RefBy: [RT:rocTempSimp](#).)

Cooling-Coil-Water: Newton’s law of convective cooling applies between the heating coil and the water. (RefBy: [RT:htFluxWaterFromCoil](#).)

Coil-Temperature-Constant-over-Time: The temperature of the heating coil is constant over time. (RefBy: [LC:Temperature-Coil-Variable-Over-Day](#) and [RT:htFluxWaterFromCoil](#).)

constant-over-Length: The temperature of the heating coil does not vary along its length. (RefBy: [LC:Temperature-Coil-Variable-Over-Length](#).)

no-Temp-Discharge: The model only accounts for charging the tank, not discharging. The temperature of the water can only increase, or remain constant; it cannot decrease. This implies that the initial temperature is less than (or equal to) the temperature of the heating coil. (RefBy: [LC:Discharging-Tank](#).)

water-Always-Liquid: The operating temperature range of the system is such that the material (water in this case) is always in liquid state. That is, the temperature will not drop below the melting point temperature of water, or rise above its boiling point temperature. (RefBy: [UC:Water-Fixed-States](#), [BT:sensHtE](#), [FT:heatEInWtr](#), and [FT:eBalanceOnWtr](#).)

perfect-Insulation-Tank: The tank is perfectly insulated so that there is no heat loss from the tank. (RefBy: [LC:Tank-Lose-Heat](#) and [FT:eBalanceOnWtr](#).)

no-Generation-By-Water: No internal heat is generated by the water; therefore, the volumetric heat generation per unit volume is zero. (RefBy: [UC:No-Internal-Heat-Generation](#) and [FT:eBalanceOnWtr](#).)

atmospheric-Pressure-Tank: The pressure in the tank is atmospheric, so the melting point temperature and boiling point temperature of water are 0°C and 100°C, respectively. (RefBy: [FT:heatEInWtr](#).)

negligible-Coil-Volume: When considering the volume of water in the tank, the volume of the heating coil is assumed to be negligible. (RefBy: [DD:waterVolume_nopcm](#).)

2 Context Theories

Some theories do not have to be explicitly invoked. They are part of the context for the other theories, without having to be explicitly stated or defined. The context theories for this problem are as follows:

- arithmetic
- operations
- differentiation
- partial differentiation
- integration
- vector calculus (gradient operator, dot product)
- Gauss's divergence theorem [\[Should this be a separate theory? Possibly a theory that is in the background, but not actually printed in the SRS document? —SS\]](#)

3 Background Theories (BT)

Refname	BT:consThermE
Label	Conservation of thermal energy
Equation	$-\nabla \cdot \mathbf{q} + g = \rho C \frac{\partial T}{\partial t}$
Description	<p>∇ is the gradient (Unitless) \mathbf{q} is the thermal flux vector ($\frac{\text{W}}{\text{m}^2}$) g is the volumetric heat generation per unit volume ($\frac{\text{W}}{\text{m}^3}$) ρ is the density ($\frac{\text{kg}}{\text{m}^3}$) C is the specific heat capacity ($\frac{\text{J}}{\text{kg}^\circ\text{C}}$) t is the time (s) T is the temperature ($^\circ\text{C}$)</p>
Notes	<p>The above equation gives the law of conservation of energy for transient heat transfer in a given material. For this equation to apply, other forms of energy, such as mechanical energy, are assumed to be negligible in the system (A:Thermal-Energy-Only). [Should we explicitly say that the above equation relies on vector calculus “theories”, or should we leave that implicit? —SS]</p> <p>Density (ρ) is defined in BT:density</p>
Source	Fourier Law of Heat Conduction and Heat Equation
RefBy	RT:rocTempSimp

Refname	BT:sensHtE
Label	Sensible heat energy (no state change)
Equation	$E = C^L m \Delta T$
Description	<p>E is the sensible heat (J)</p> <p>C^L is the specific heat capacity of a liquid ($\frac{\text{J}}{\text{kg}^\circ\text{C}}$)</p> <p>$m$ is the mass (kg)</p> <p>ΔT is the change in temperature ($^\circ\text{C}$)</p>
Notes	<p>E occurs as long as the material does not reach a temperature where a phase change occurs, as assumed in A:Water-Always-Liquid. [This should actually be an assumption of no state change. This is a generic assumption that is necessary to use this theory. —SS]</p>
Source	Definition of Sensible Heat
RefBy	FT:heatElInWtr

Refname	BT:nwtnCooling
Label	Newton's law of cooling
Equation	$q(t) = h(T(t) - T_{\text{env}}(t))$
Description	<p> q is the heat flux ($\frac{\text{W}}{\text{m}^2}$) t is the time (s) h is the convective heat transfer coefficient ($\frac{\text{W}}{\text{m}^2\text{°C}}$) T is the temperature of the body (°C) T_{env} is the temperature of the environment surrounding the body (°C) </p>
Notes	<p> Newton's law of cooling describes convective cooling from a surface. The law is stated as: the rate of heat loss from a body is proportional to the difference in temperatures between the body and its surroundings. h is assumed to be independent of T (from A:Heat-Transfer-Coeffs-Constant. </p>
Source	[incroperaEtAl2007]
RefBy	RT:htFluxWaterFromCoil

Refname	BT:density
Label	Density
Equation	$\rho = \frac{m}{V}$
Description	<p>ρ is the density of a material ($\frac{\text{kg}}{\text{m}^3}$)</p> <p>$m$ is the mass of the body (kg)</p> <p>V is the volume of the body (m^3)</p>
Notes	Density is the mass per unit volume.
Source	—
RefBy	BT:consThermE, RT:rocTempSimp, DD:waterMass

4 Refined Theories

This section collects the laws and equations that will be used to build the instance models.

Refname	RT:rocTempSimp
Label	Simplified rate of change of temperature
Equation	$mC \frac{dT}{dt} = q_{\text{in}} A_{\text{in}} - q_{\text{out}} A_{\text{out}} + gV$
Description	<p> m is the mass (kg) C is the specific heat capacity ($\frac{\text{J}}{\text{kg}^\circ\text{C}}$) t is the time (s) T is the temperature ($^\circ\text{C}$) q_{in} is the heat flux input ($\frac{\text{W}}{\text{m}^2}$) A_{in} is the surface area over which heat is transferred in (m^2) q_{out} is the heat flux output ($\frac{\text{W}}{\text{m}^2}$) A_{out} is the surface area over which heat is transferred out (m^2) g is the volumetric heat generation per unit volume ($\frac{\text{W}}{\text{m}^3}$) V is the volume (m^3) </p>
Source	–
RefBy	RT:rocTempSimp and FT:eBalanceOnWtr

Detailed derivation of simplified rate of change of temperature: Integrating **BT:consThermE** over a volume (V), we have:

$$- \int_V \nabla \cdot \mathbf{q} dV + \int_V g dV = \int_V \rho C \frac{\partial T}{\partial t} dV$$

Applying Gauss's Divergence Theorem to the first term over the surface S of the volume, with \mathbf{q} as the thermal flux vector for the surface and $\hat{\mathbf{n}}$ as a unit outward normal vector for a surface:

$$- \int_S \mathbf{q} \cdot \hat{\mathbf{n}} dS + \int_V g dV = \int_V \rho C \frac{\partial T}{\partial t} dV$$

We consider an arbitrary volume. The volumetric heat generation per unit volume is assumed constant. Then Equation (1) can be written as:

$$q_{\text{in}}A_{\text{in}} - q_{\text{out}}A_{\text{out}} + gV = \int_V \rho C \frac{\partial T}{\partial t} dV$$

Where q_{in} , q_{out} , A_{in} , and A_{out} are explained in [RT:rocTempSimp](#). [\[Why is this RT referencing itself? This seems like something that could be removed. —SS\]](#) Assuming ρ , C , and T are constant over the volume, which is true in our case by [A:Constant-Water-Temp-Across-Tank](#), [A:Density-Water-Constant-over-Volume](#), and [A:Specific-Heat-Energy-Constant-over-Volume](#), we have:

$$\rho CV \frac{dT}{dt} = q_{\text{in}}A_{\text{in}} - q_{\text{out}}A_{\text{out}} + gV$$

Using the fact that $\rho=m/V$ ([BT:density](#)), Equation (2) can be written as:

$$mC \frac{dT}{dt} = q_{\text{in}}A_{\text{in}} - q_{\text{out}}A_{\text{out}} + gV$$

Refname	RT:htFluxWaterFromCoil
Label	Heat flux into the water from the coil
Units	$\frac{\text{W}}{\text{m}^2}$
Equation	$q_C = h_C (T_C - T_W(t))$
Description	<p>q_C is the heat flux into the water from the coil ($\frac{\text{W}}{\text{m}^2}$)</p> <p>$h_C$ is the convective heat transfer coefficient between coil and water ($\frac{\text{W}}{\text{m}^2\text{°C}}$)</p> <p>$T_C$ is the temperature of the heating coil (°C)</p> <p>T_W is the temperature of the water (°C)</p> <p>t is the time (s)</p>
Notes	<p>q_C is found by assuming that Newton's law of cooling applies (A:Newton-Law-Convective-Cooling-Coil-Water). This law (defined in BT:nwtnCooling) is used on the surface of the heating coil.</p> <p>A:Temp-Heating-Coil-Constant-over-Time</p>
Source	[koothoor2013]
RefBy	FT:eBalanceOnWtr

5 Definitions

This section collects and defines all the data needed to build the instance models.

Refname	DD:waterMass
Label	Mass of water
Symbol	m_W
Units	kg
Equation	$m_W = V_W \rho_W$
Description	m_W is the mass of water (kg) V_W is the volume of water (m^3) ρ_W is the density of water ($\frac{\text{kg}}{\text{m}^3}$)
Notes	Density (ρ) is defined in BT:density
Source	—
RefBy	FR:Find-Mass

Refname	DD:waterVolume.nopcm
Label	Volume of water
Symbol	V_W
Units	m^3
Equation	$V_W = V_{\text{tank}}$
Description	V_W is the volume of water (m^3) V_{tank} is the volume of the cylindrical tank (m^3)
Notes	Based on A:Volume-Coil-Negligible . V_{tank} is defined in DD:tankVolume .
Source	—
RefBy	FR:Find-Mass

Refname	DD:tankVolume
Label	Volume of the cylindrical tank
Symbol	V_{tank}
Units	m^3
Equation	$V_{\text{tank}} = \pi \left(\frac{D}{2} \right)^2 L$
Description	<p>V_{tank} is the volume of the cylindrical tank (m^3)</p> <p>π is the ratio of circumference to diameter for any circle (Unitless)</p> <p>D is the diameter of tank (m)</p> <p>L is the length of tank (m)</p>
Source	—
RefBy	DD:waterVolume_nopcm and FR:Find-Mass

Refname	DD:balanceDecayRate
Label	ODE parameter for water related to decay time
Symbol	τ_W
Units	s
Equation	$\tau_W = \frac{m_W C_W}{h_C A_C}$
Description	τ_W is the ODE parameter for water related to decay time (s) m_W is the mass of water (kg) C_W is the specific heat capacity of water ($\frac{J}{kg^\circ C}$) h_C is the convective heat transfer coefficient between coil and water ($\frac{W}{m^2^\circ C}$) A_C is the heating coil surface area (m ²)
Source	[koothoor2013]
RefBy	FR:Output-Input-Derived-Values and FT:eBalanceOnWtr

6 Final Theories

This section transforms the problem defined in the problem description into one which is expressed in mathematical terms. It uses concrete symbols defined in the [data definitions](#) to replace the abstract symbols in the models identified in [theoretical models](#) and [general definitions](#).

The goal GS:Predict-Water-Temperature is met by [FT:eBalanceOnWtr](#) and the goal GS:Predict-Water-Energy is met by [FT:heatElInWtr](#).

Refname	FT:eBalanceOnWtr
Label	Energy balance on water to find the temperature of the water
Input	$T_C, T_{\text{init}}, t_{\text{final}}, A_C, h_C, C_W, m_W$
Output	T_W
Input Constraints	$T_C \geq T_{\text{init}}$
Output Constraints	
Equation	$\frac{dT_W}{dt} = \frac{1}{\tau_W} (T_C - T_W(t))$
Description	<p>t is the time (s)</p> <p>T_W is the temperature of the water ($^{\circ}\text{C}$)</p> <p>τ_W is the ODE parameter for water related to decay time (s)</p> <p>T_C is the temperature of the heating coil ($^{\circ}\text{C}$)</p>
Notes	<p>τ_W is calculated from DD:balanceDecayRate.</p> <p>The above equation applies as long as the water is in liquid form, $0 < T_W < 100$ ($^{\circ}\text{C}$) where 0 ($^{\circ}\text{C}$) and 100 ($^{\circ}\text{C}$) are the melting and boiling point temperatures of water, respectively (A:Water-Always-Liquid).</p>
Source	[koothoor2013]
RefBy	UC:No-Internal-Heat-Generation, FR:Find-Mass, and FR:Calculate-Temperature-Water-Over-Time

Detailed derivation of the energy balance on water: To find the rate of change of T_W , we look at the energy balance on water. The volume being considered is the volume of water in the tank V_W , which has mass m_W and specific heat capacity, C_W . Heat transfer occurs in the water from the heating coil as q_C (**RT:htFluxWaterFromCoil**), over area A_C . No heat transfer occurs to the outside of the tank, since it has been assumed to be perfectly insulated (**A:Perfect-Insulation-Tank**). Since the assumption is made that no internal heat is generated (**A:No-Internal-Heat-Generation-By-Water**), $g = 0$. Therefore, the equation for **RT:rocTempSimp** can be written as:

$$m_W C_W \frac{dT_W}{dt} = q_C A_C$$

Using **RT:htFluxWaterFromCoil** for q_C , this can be written as:

$$m_W C_W \frac{dT_W}{dt} = h_C A_C (T_C - T_W)$$

Dividing Equation (2) by $m_W C_W$, we obtain:

$$\frac{dT_W}{dt} = \frac{h_C A_C}{m_W C_W} (T_C - T_W)$$

By substituting τ_W (from **DD:balanceDecayRate**), this can be written as:

$$\frac{dT_W}{dt} = \frac{1}{\tau_W} (T_C - T_W)$$

Refname	FT:heatEInWtr
Label	Heat energy in the water
Input	$T_{\text{init}}, m_{\text{W}}, C_{\text{W}}, m_{\text{W}}$
Output	E_{W}
Input Constraints	
Output Constraints	
Equation	$E_{\text{W}}(t) = C_{\text{W}}m_{\text{W}}(T_{\text{W}}(t) - T_{\text{init}})$
Description	<p>E_{W} is the change in heat energy in the water (J)</p> <p>t is the time (s)</p> <p>C_{W} is the specific heat capacity of water ($\frac{\text{J}}{\text{kg}^{\circ}\text{C}}$)</p> <p>$m_{\text{W}}$ is the mass of water (kg)</p> <p>T_{W} is the temperature of the water ($^{\circ}\text{C}$)</p> <p>T_{init} is the initial temperature ($^{\circ}\text{C}$)</p>
Notes	<p>The above equation is derived using BT:sensHtE.</p> <p>The change in temperature is the difference between the temperature at time t (s), T_{W} and the initial temperature, T_{init} ($^{\circ}\text{C}$).</p> <p>This equation applies as long as $0 < T_{\text{W}} < 100^{\circ}\text{C}$ (A:Water-Always-Liquid, A:Atmospheric-Pressure-Tank).</p>
Source	[koothoor2013]
RefBy	FR:Calculate-Change-Heat_Energy-Water-Over-Time

6.0.1 Data Constraints

The **Data Constraints Table** shows the data constraints on the input variables. The column for physical constraints gives the physical limitations on the range of values that can be taken by the variable. The uncertainty column provides an estimate of the confidence with which the physical quantities can be measured. This information would be part of the input if one were performing an uncertainty quantification exercise. The constraints are conservative, to give the user of the model the flexibility to experiment with unusual situations. The column of typical values is intended to provide a feel for a common scenario. The column for software constraints restricts the range of inputs to reasonable values.

Var	Physical Constraints	Software Constraints	Typical Value	Uncert.
A_C	$A_C > 0$	$A_C \leq A_C^{\max}$	0.12 m ²	10%
C_W	$C_W > 0$	$C_W^{\min} < C_W < C_W^{\max}$	4186 $\frac{\text{J}}{\text{kg}^\circ\text{C}}$	10%
D	$D > 0$	$AR_{\min} \leq D \leq AR_{\max}$	0.412 m	10%
h_C	$h_C > 0$	$h_C^{\min} \leq h_C \leq h_C^{\max}$	1000 $\frac{\text{W}}{\text{m}^2^\circ\text{C}}$	10%
L	$L > 0$	$L_{\min} \leq L \leq L_{\max}$	1.5 m	10%
T_C	$0 < T_C < 100$	–	50 °C	10%
T_{init}	$0 < T_{\text{init}} < 100$	–	40 °C	10%
t_{final}	$t_{\text{final}} > 0$	$t_{\text{final}} < t_{\text{final}}^{\max}$	50000 s	10%
t_{step}	$0 < t_{\text{step}} < t_{\text{final}}$	–	0.01 s	10%
ρ_W	$\rho_W > 0$	$\rho_W^{\min} < \rho_W \leq \rho_W^{\max}$	1000 $\frac{\text{kg}}{\text{m}^3}$	10%

Table 1: Input Data Constraints

6.0.2 Properties of a Correct Solution

The **Data Constraints Table** shows the data constraints on the output variables. The column for physical constraints gives the physical limitations on the range of values that can be taken by the variable.

Var	Physical Constraints
T_W	$T_{\text{init}} \leq T_W \leq T_C$
E_W	$E_W \geq 0$

Table 2: Output Data Constraints

7 Requirements

This section provides the functional requirements, the tasks and behaviours that the software is expected to complete, and the non-functional requirements, the qualities that the software

is expected to exhibit.

7.1 Functional Requirements

This section provides the functional requirements, the tasks and behaviours that the software is expected to complete.

Input-Initial-Values: Input the following values described in the table for **Required Inputs**, which define the tank parameters, material properties, and initial conditions.

Find-Mass: Use the inputs in **FR:Input-Initial-Values** to find the mass needed for IM:eBalanceOnWtr, using **DD:waterMass**, **DD:waterVolume_nopcm**, and **DD:tankVolume**.

Physical_Constraints: Verify that the inputs satisfy the required **physical constraints**.

Output-Derived-Values: Output the input values and derived values in the following list: the values (from **FR:Input-Initial-Values**), the mass (from **FR:Find-Mass**), and τ_W (from **DD:balanceDecayRate**).

Water-Over-Time: Calculate and output the temperature of the water ($T_W(t)$) over the simulation time (from IM:eBalanceOnWtr).

Water-Over-Time: Calculate and output the change in heat energy in the water ($E_W(t)$) over the simulation time (from IM:heatEInWtr).

Symbol	Description	Units
A_C	Heating coil surface area	m^2
A_{tol}	Absolute tolerance	—
C_W	Specific heat capacity of water	$\frac{J}{kg^\circ C}$
D	Diameter of tank	m
h_C	Convective heat transfer coefficient between coil and water	$\frac{W}{m^2^\circ C}$
L	Length of tank	m
R_{tol}	Relative tolerance	—
T_C	Temperature of the heating coil	$^\circ C$
T_{init}	Initial temperature	$^\circ C$
t_{final}	Final time	s
t_{step}	Time step for simulation	s
ρ_W	Density of water	$\frac{kg}{m^3}$

Table 3: Required Inputs following **FR:Input-Initial-Values**

8 Likely Changes

This section lists the likely changes to be made to the software.

- Variable-Over-Day: **A:Temp-Heating-Coil-Constant-over-Time** - The temperature of the heating coil will change over the course of the day, depending on the energy received from the sun.
- Variable-Over-Length: **A:Temp-Heating-Coil-Constant-over-Length** - The temperature of the heating coil will actually change along its length as the water within it cools.
- Discharging-Tank: **A:Charging-Tank-No-Temp-Discharge** - The model currently only accounts for charging of the tank. That is, increasing the temperature of the water to match the temperature of the coil. A more complete model would also account for discharging of the tank.
- Tank-Lose-Heat: **A:Perfect-Insulation-Tank** - Any real tank cannot be perfectly insulated and will lose heat.

9 Unlikely Changes

This section lists the unlikely changes to be made to the software.

- Water-Fixed-States: **A:Water-Always-Liquid** - It is unlikely for the change of water from liquid to a solid, or from liquid to gas to be considered.
- l-Heat-Generation: **A:No-Internal-Heat-Generation-By-Water** - Is used for the derivations of IM:eBalanceOnWtr.