

## Design Optimization Project 2

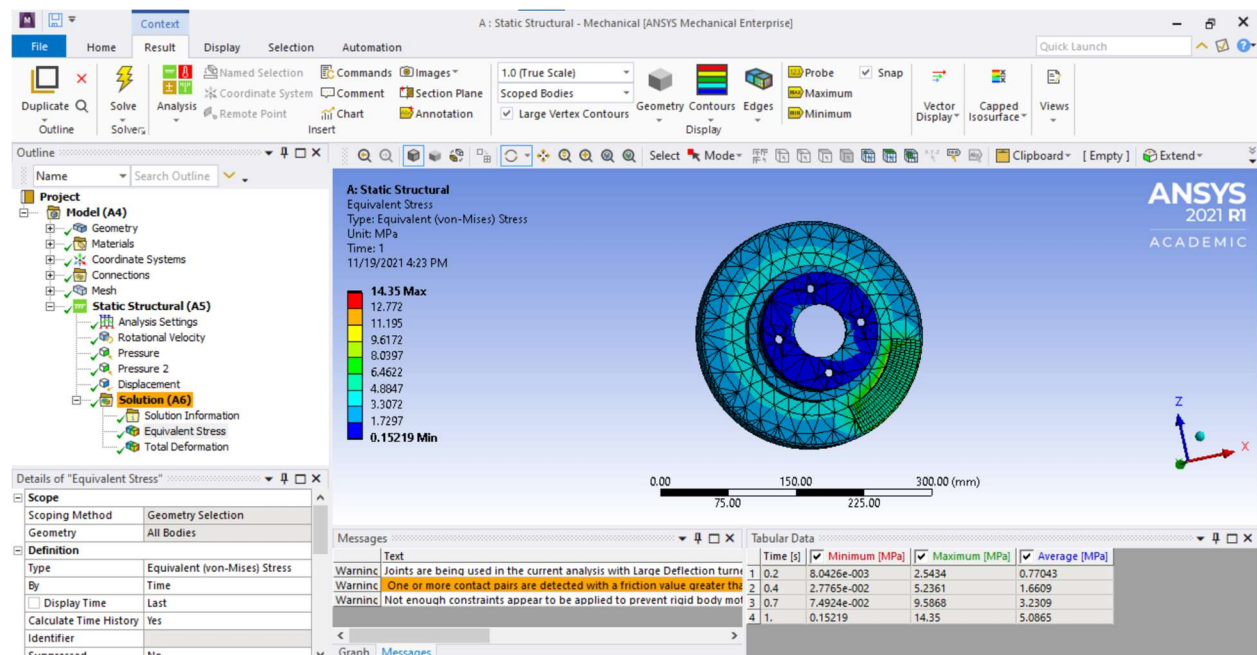
### Mofe Fagade

The aim of this optimization project was to design a brake for emergency conditions with minimal volume that would not only minimize the maximum stress in the brake disc, but also maximize the first natural frequency and minimize the maximum temperature of the brake disc.

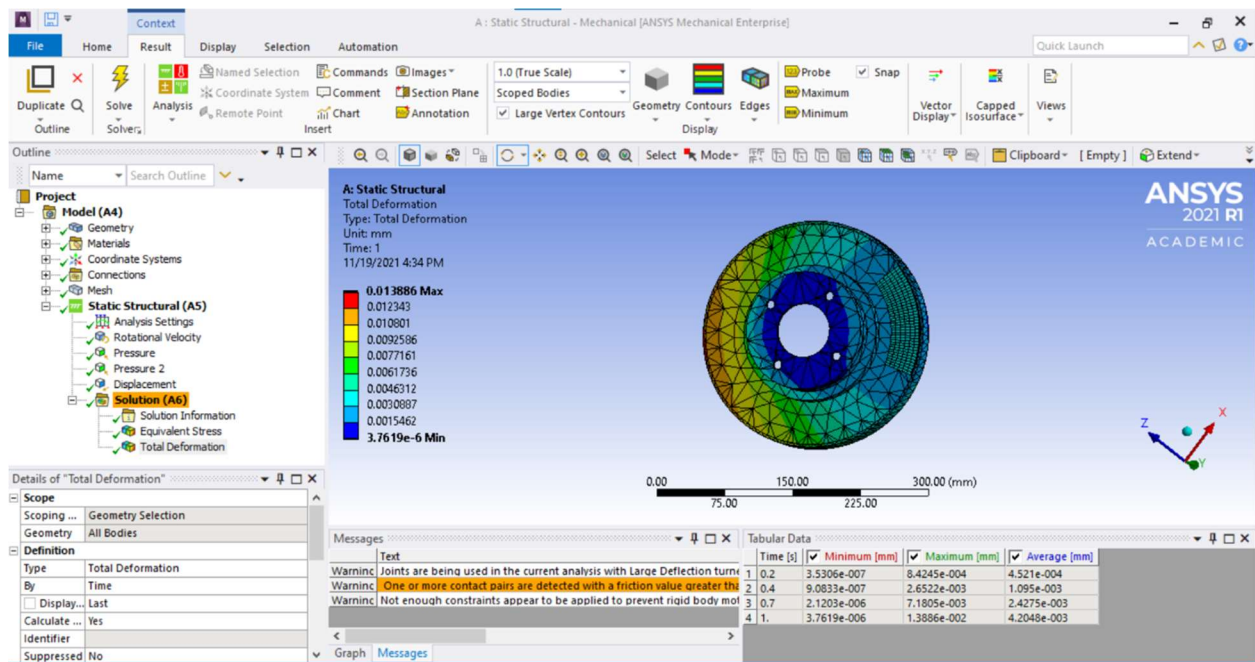
These conditions are desired because if not designed for the material failure could occur due to resultant stresses from centrifugal body forces, failure from resonance, and also wear and tear due to increase in its temperature and thermal stresses.

Thus, three studies were performed in ANSYS which are: Structural Analysis, Modal Analysis and Thermal Analysis. The results from each of these analyses are illustrated below. Furthermore, grey cast iron was the material used for the brake disc, and structural steel was used for the brake pads on the disc.

### STATIC STURCTURAL

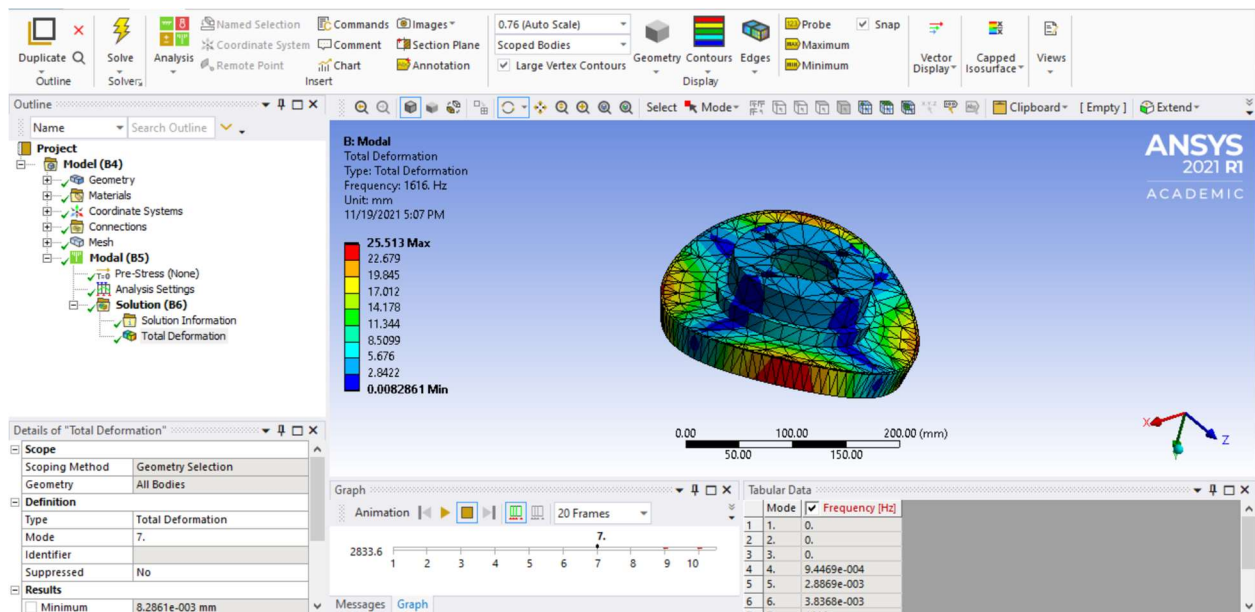


From the figure above, it can be seen from the initial analysis that the maximum stress is found in the region where the pads are in contact with the disc and the value is found to be about 14.35MPa.



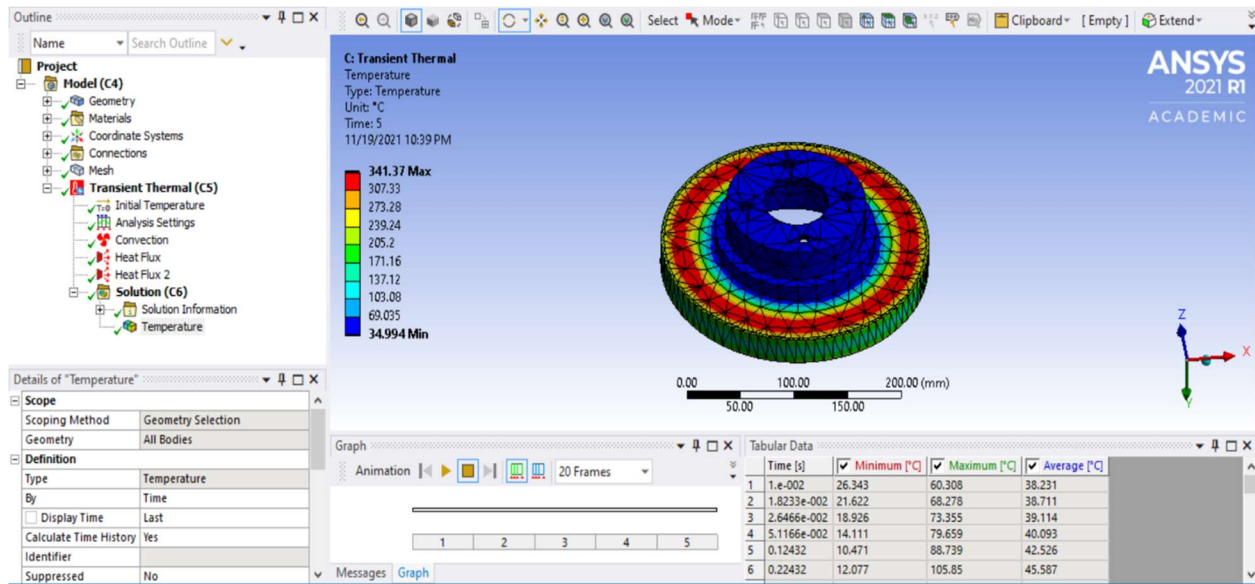
It can also be seen from the structural analysis that the maximum deformation in the disc under the load is 0.013886mm.

## MODAL



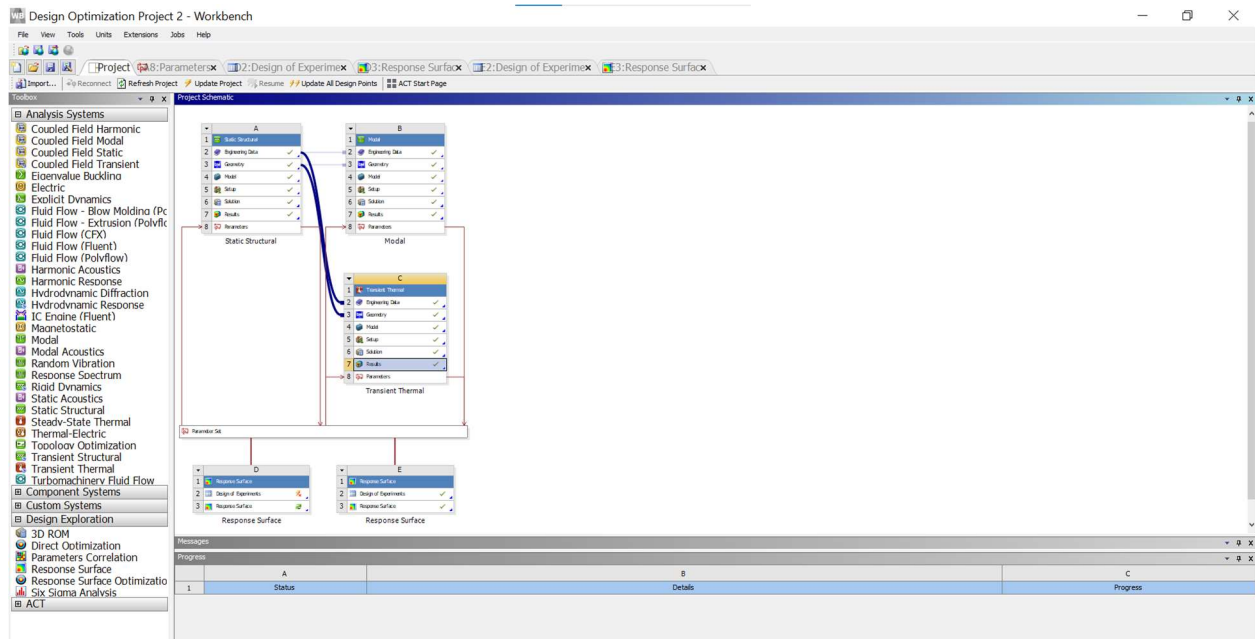
From the modal analysis, it can be seen that the maximum total deformation experienced by the body is 25.513mm and the frequency is 1616Hz.

## THERMAL

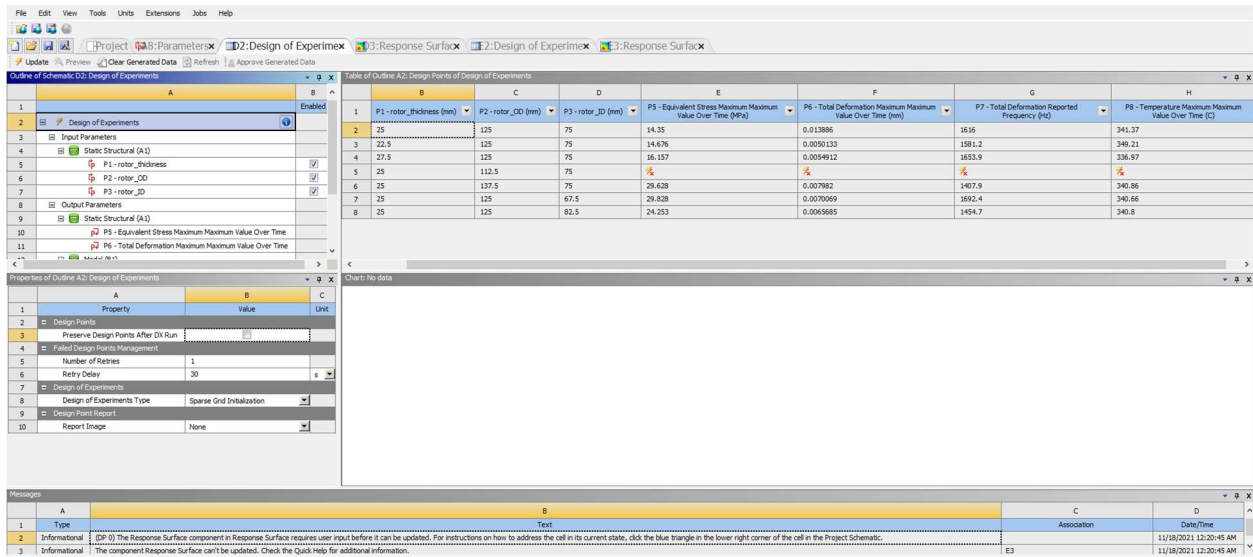


From the thermal analysis, it can be seen that the maximum temperature in the disc under the loading is 341.37°C.

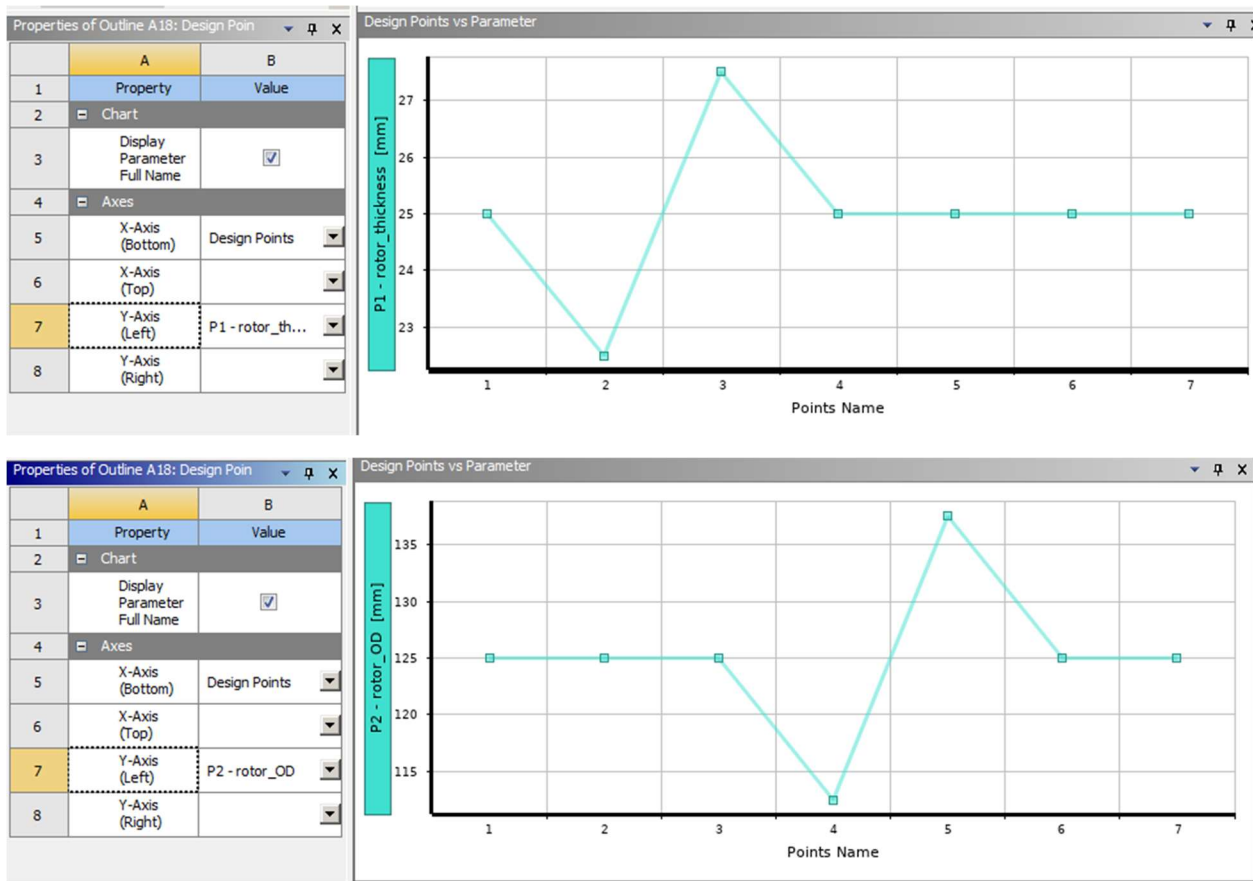
These are the main parameters that we will be optimizing and be utilizing later in the DOE.

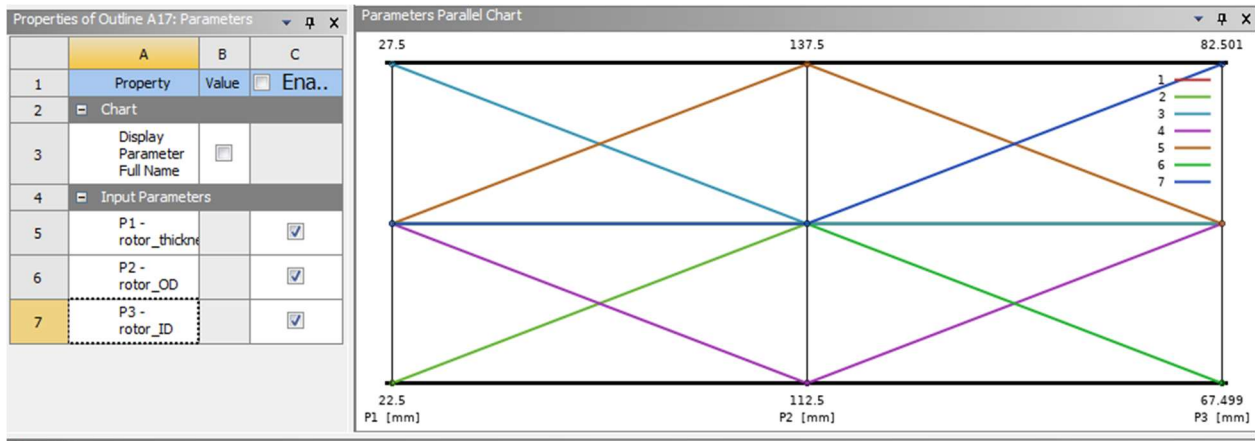
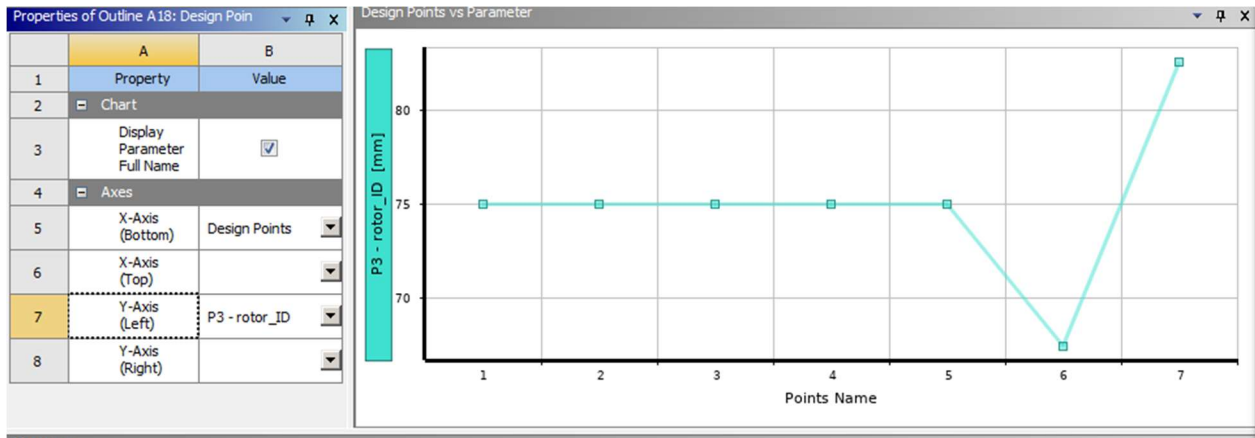


The ANSYS project schematic is shown in the figure above and as can be seen the input parameters are used for the structural, modal and thermal analysis and the output parameters from these analyses is then used for the response surface and design of experiments.



From the screenshot, it can be seen that the input parameters are rotor thickness, outer diameter and inner diameter. The output parameters from the static structural surface are the equivalent stress and the total deformation. From the modal, it is the total deformation reported frequency, and from the thermal it is the maximum temperature value over time.





The Design of Experiments type that was used was Sparse Grid Initialization. Sparse grid is a more advanced sampling technique which only samples a few points initially and then depending on the response surface adaptively needs sample points. As can be seen from the DOE, one of the design points failed to compute due to the geometry and leaving the feasible space. To fix this issue, the .csv file was downloaded and manually adjusted before being uploaded back to ANSYS. This was then used for the response surface. The readjusted data table is illustrated in the figure below.

Update

Preview

Clear Generated Data

Refresh

Approve Generated Data

Outline of Schematic E2: Design of Experiments

	A	B
1	Design of Experiments	Enabled
2	Input Parameters	
3	Static Structural (A1)	
4	P1 - rotor_thickne	<input checked="" type="checkbox"/>
5	P2 - rotor_OD	<input checked="" type="checkbox"/>
6	P3 - rotor_ID	<input checked="" type="checkbox"/>
7	Output Parameters	
8	Static Structural (A1)	
9	P5 - Equivalent Stress Maximum Maximum Value Over Time	
10	P6 - Total Deformation Maximum Maximum Value Over Time	
11	Model (B1)	

Table of Outline A2: Design Points of Design of Experiments

	B	C	D	E	F	G	H
1	P1 - rotor_thickne (mm)	P2 - rotor_OD (mm)	P3 - rotor_ID (mm)	P5 - Equivalent Stress Maximum Maximum Value Over Time (MPa)	P6 - Total Deformation Maximum Maximum Value Over Time (mm)	P7 - Total Deformation Reported Frequency (Hz)	P8 - Temperature Maximum Maximum Value Over Time (C)
2	25	125	75	14.35	0.013886	1516	341.37
3	22.5	125	75	14.676	0.0050133	1581.2	349.21
4	27.5	125	75	16.157	0.0054912	1653.9	336.97
5	25	122.5	75	21.748	0.0054858	1654.3	344.96
6	25	137.5	75	29.438	0.007902	1407.9	340.86
7	25	125	67.5	29.828	0.0070069	1692.4	340.66
8	25	125	82.5	24.253	0.0065685	1454.7	340.8
9	Point						

Properties of Outline A2: Design of Experiments

	A	B
1	Property	Value
2	Design Points	
3	Preserve Design Points After DX Run	<input type="checkbox"/>
4	Response Surface Management	
5	Number of Factors	3
6	Design of Experiments	
7	Design of Experiments Type	Custom
8	Design Results Report	
9	Report Image	None



Due to the adjustment of the data table from the DOE, the Sparse grid type couldn't be used thus I used Kriging instead. I also used refinement to adjust the optimization process and also verification points as well. After running the response surface for a few minutes, it could be seen that the optimal rotor thickness is 25mm, the optimal outer diameter is 125mm, and the optimal inner diameter is 76.35mm. The optimal maximum equivalent stress was found to be 14.259MPa, optimal total deformation was 0.013617mm, and optimal total deformation reported frequency was 1593.2Hz, and finally optimal maximum temperature was found to be 341.36°C. These figures can all be confirmed and verified in the figure below.

	A	B	C	D	E	F	G	H
1	Name	P1 - rotor_thickness (mm)	P2 - rotor_OD (mm)	P3 - rotor_ID (mm)	P5 - Equivalent Stress Maximum Maximum Value Over Time (MPa)	P6 - Total Deformation Maximum Maximum Value Over Time (mm)	P7 - Total Deformation Reported Frequency (Hz)	P8 - Temperature Maximum Maximum Value Over Time (C)
2	Response Point	25	125	76.35	14.259	0.013617	1593.2	341.36
*	New Response Point							

Some of the results from my response surface analysis can also be seen in the figures below:

	A	B	C	D	E	F	G	H
1	Name	P1 - rotor_thickness (mm)	P2 - rotor_OD (mm)	P3 - rotor_ID (mm)	P5 - Equivalent Stress Maximum Maximum Value Over Time (MPa)	P6 - Total Deformation Maximum Maximum Value Over Time (mm)	P7 - Total Deformation Reported Frequency (Hz)	P8 - Temperature Maximum Maximum Value Over Time (C)
2	Output Parameter Minimums							
3	P5 - Equivalent Stress Maximum Maximum Value Over Time	24.132	129.06	75.824	9.4776	0.019499	1525.5	340.35
4	P6 - Total Deformation Maximum Maximum Value Over Time	22.5	112.5	82.5	96.421	-0.076242	1600.3	378.23
5	P7 - Total Deformation Reported Frequency	22.5	137.5	82.5	39.858	-0.0082081	1211.8	348.15
6	P8 - Temperature Maximum Maximum Value Over Time	27.5	131.47	67.5	28.662	0.0055264	1626	332.37
7	Output Parameter Maximums							
8	P5 - Equivalent Stress Maximum Maximum Value Over Time	27.5	112.5	67.5	103.48	-0.075325	1510.6	365.84
9	P6 - Total Deformation Maximum Maximum Value Over Time	25.035	130.33	74.894	10.264	0.021136	1533.1	337.51
10	P7 - Total Deformation Reported Frequency	27.5	112.5	67.5	103.48	-0.075325	1910.6	365.84
11	P8 - Temperature Maximum Maximum Value Over Time	22.5	112.5	75.425	86.4	-0.068959	1794.6	378.79

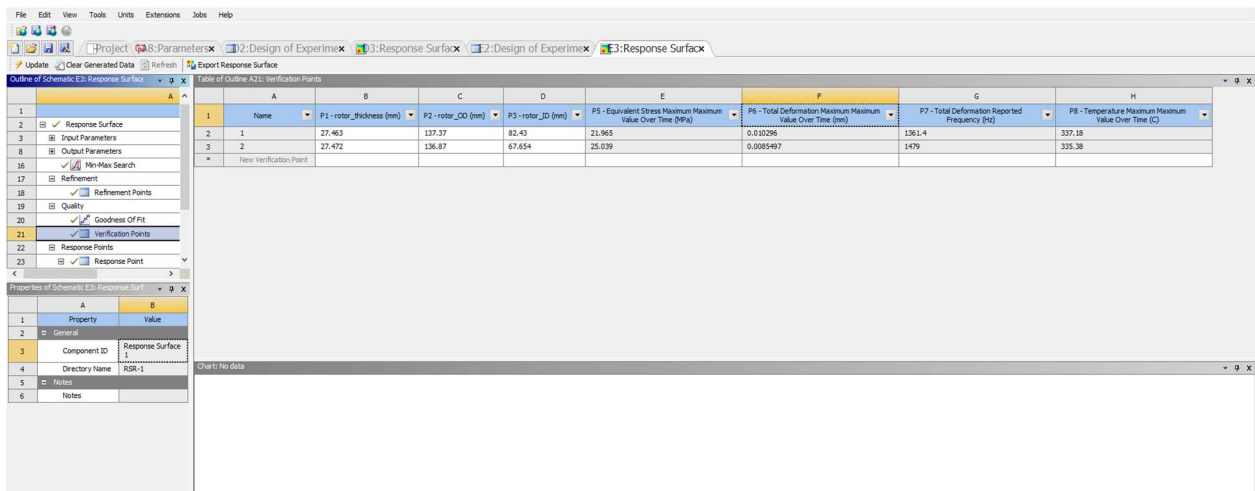
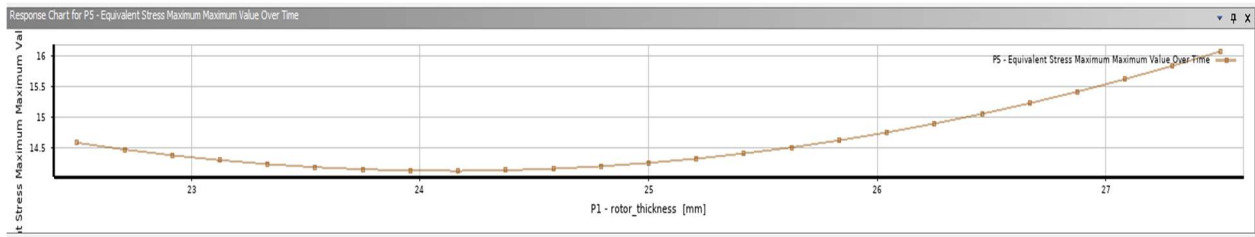
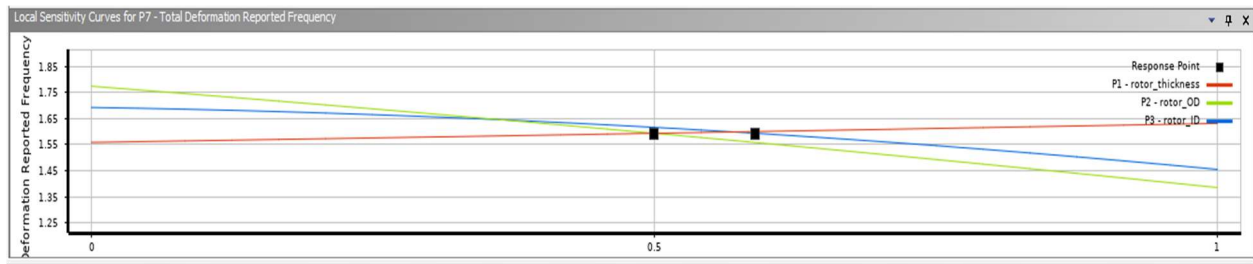
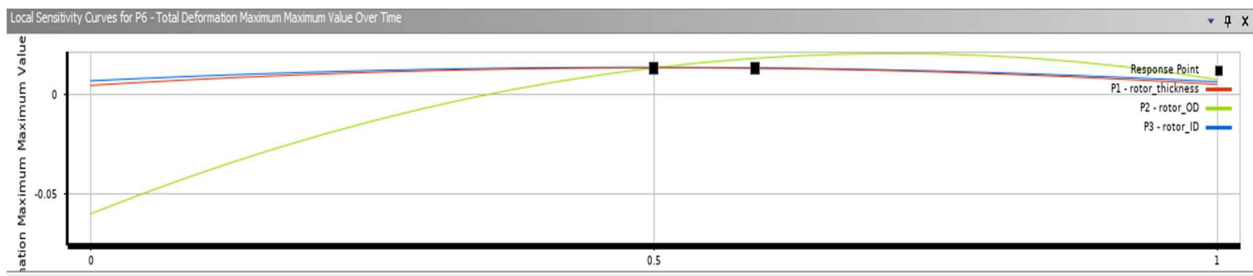
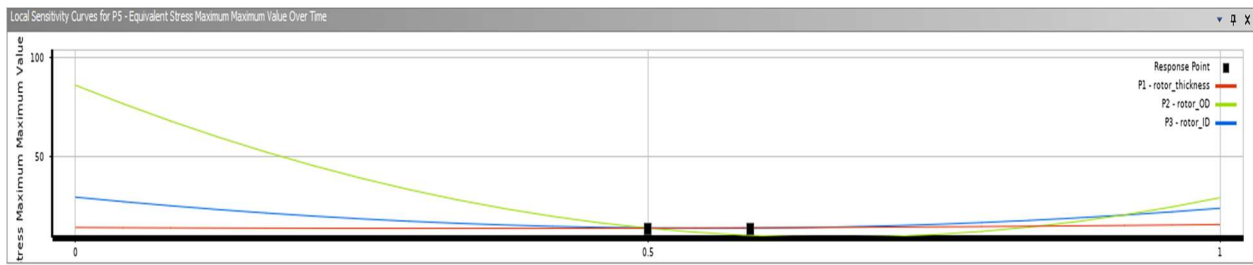
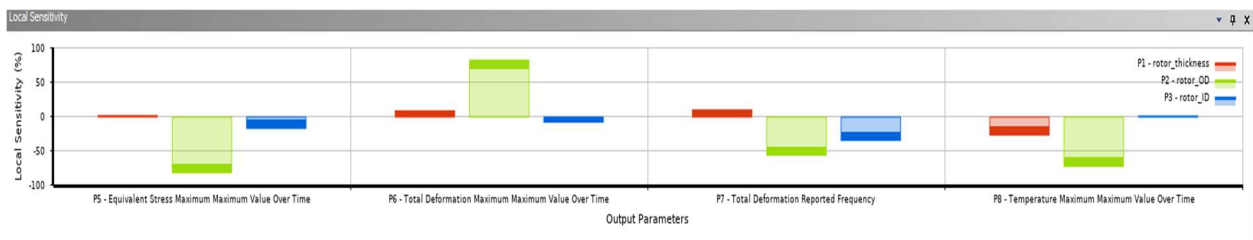
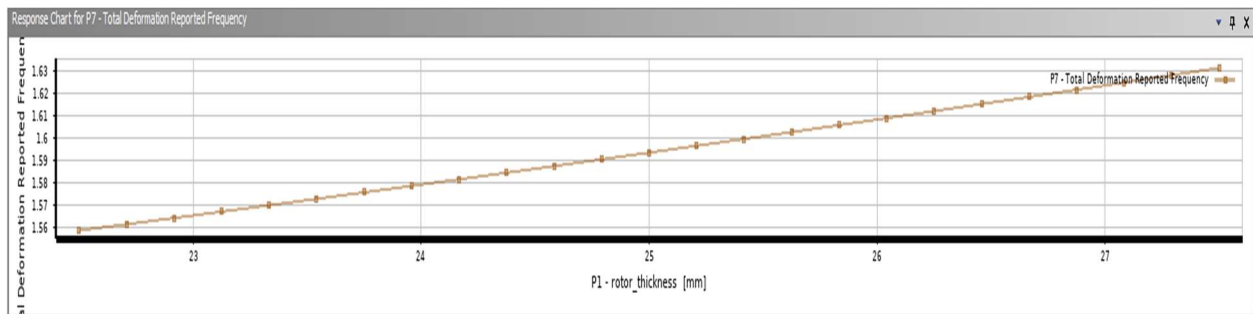
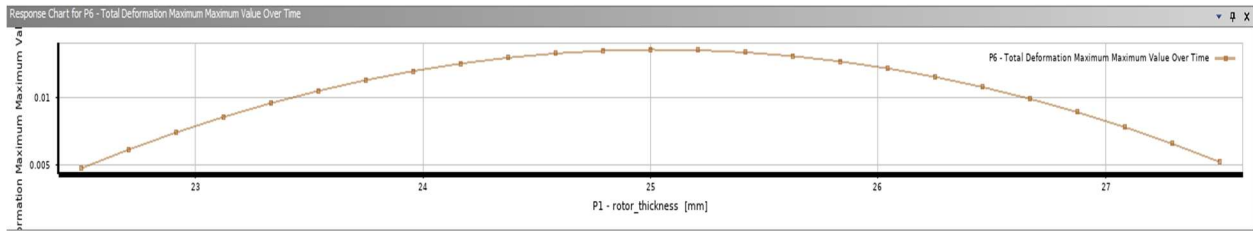


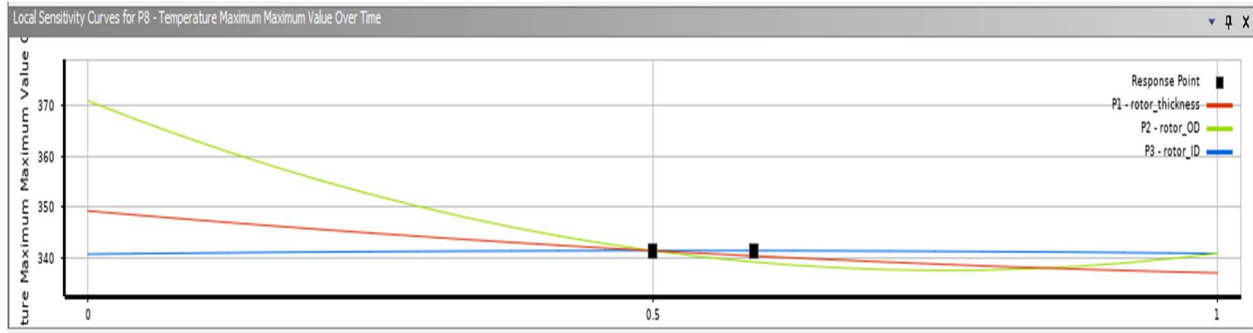
Table of Outline A22: Response Points

	A	B	C	D	E	F	G	H
1	Name	P1 - rotor_thickness (mm)	P2 - rotor_OD (mm)	P3 - rotor_ID (mm)	P5 - Equivalent Stress Maximum Maximum Value Over Time (MPa)	P6 - Total Deformation Maximum Maximum Value Over Time (mm)	P7 - Total Deformation Reported Frequency (Hz)	P8 - Temperature Maximum Maximum Value Over Time (C)
2	Response Point	25	125	76.35	14.259	0.013617	1593.2	341.36
*	New Response Point							









Some of the observations that were made during the optimization analysis are as follows:

- 1.) There was some trade-off between the objectives when adjusting the parameters. After some trial and error, I noticed that I couldn't continue to minimize the upper bound of the temperature from the thermal analysis without compromising the input parameters from the static structural analysis. Design parameters that ran into this issue failed to compute.
- 2.) Furthermore, all the variables that were used for the DoE and response surface were continuous.
- 3.) My objective/constraint functions were analytical but due to some very sharp corners as can be seen in the design points vs parameter charts, they were not always differentiable.
- 4.) The optimization method I chose for the DoE was Sparse Grid Initialization, and I used Kriging for the response surface.
- 5.) Sensitivity Analysis was also performed, and this can be seen with the local sensitivity graphs and sensitivity curves. Monotonicity was only seen with the total deformation and rotor thickness and total deformation. As the rotor thickness increases, the total deformation increases linearly as well.
- 6.) After the optimization analysis, the parameters from the initial are compared with those after the analysis is completed. These values are reported in the table below.

	Rotor Thickness(mm)	Rotor O.D (mm)	Rotor I.D (mm)
Initial	25	125	75
After Analysis	25	125	76.35

Input Parameters

	Equivalent Stress	Total deformation	First natural frequency	Maximum Temperature
Initial	14.350MPa	0.013886mm	1616Hz	341.37°C
After Analysis	14.259MPa	0.013617mm	1593.2Hz	341.36°C

Output parameters