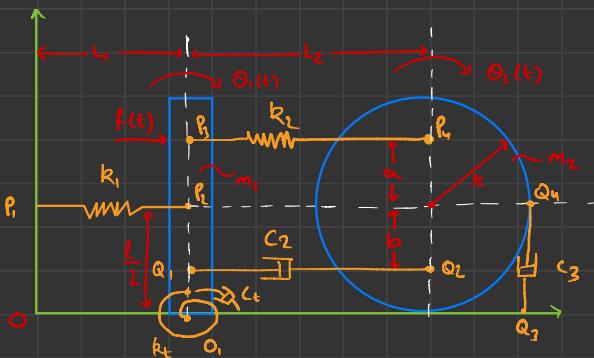
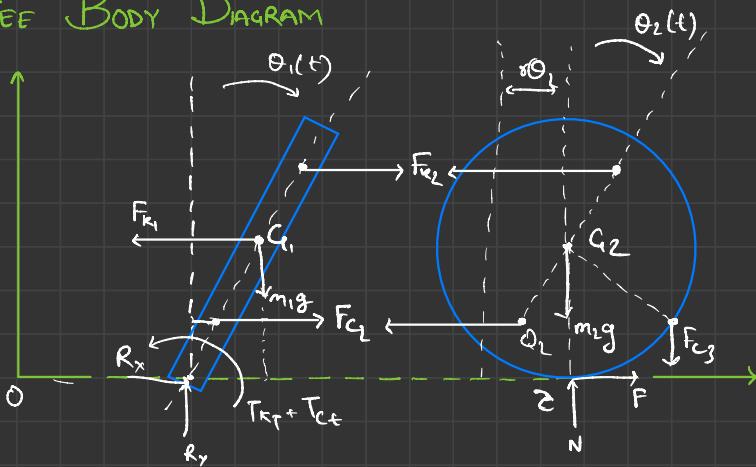


Induced Vibrations



FREE BODY DIAGRAM



Accelerations :-

$$\vec{\alpha}_{0q_1} = \left(L_1 + \frac{L}{2} \sin \theta_1 \right) \hat{i} + \frac{L}{2} \cos \theta_1 \hat{j}$$

$$\vec{\alpha}_{0q_1} = \frac{L}{2} \dot{\theta} \cos \theta_1 \hat{i} - \frac{L}{2} \dot{\theta} \sin \theta_1 \hat{j}$$

$$\vec{\alpha}_{0q_1} = \left(\frac{L}{2} \ddot{\theta} \cos \theta_1 - \frac{L}{2} \dot{\theta}^2 \sin \theta_1 \right) \hat{i}$$

$$- \left(\frac{L}{2} \dot{\theta}^2 \cos \theta_1 + \frac{L}{2} \dot{\theta} \sin \theta_1 \right) \hat{j}$$

$$\overline{\delta_{0\alpha_2}} = (\underline{L_1 + L_2 + \theta\theta_2})\hat{i} + \overline{\delta_j}$$

$$F_{R_1} = k_1 (\overline{\delta_{P_1 P_2}} - L_1)$$

$$F_{R_2} = R_1 (\overline{\delta})$$

$$F_{R_1} = k_1 (\overline{\delta_{P_1 P_2}} - L_1)$$

$$F_{R_2} = R_2 (\overline{\delta_{P_3 P_4}} - L_2)$$

$$F_{C_2} = C_2 \overline{\delta_{Q_1 Q_1}} - \hat{e}_{Q_1 Q_2}$$

$$F_{C_3} = C_3 \overline{\delta_{Q_3 Q_4}} - \hat{e}_{Q_3 Q_4}$$

$$\hat{e}_{Q_3 Q_4} = \frac{\overline{\delta_{Q_3 Q_4}}}{\overline{\delta_{Q_3 Q_4}}}$$

$$\sum M_\gamma = - I \dot{\sum \vec{\phi}}$$

$$= - \frac{3}{2} m_2 \dot{\gamma}_2^2 \dot{\vec{\phi}}$$

$$- \frac{3}{2} m_2 \dot{\gamma}_2^2 \dot{\vec{\phi}} =$$



$$I_C = \frac{3}{2} m_2 \dot{\gamma}_2^2 \quad (\text{End Vib})$$

$$I_C = \frac{1}{12} ml^2 \quad I_{O_1} = \frac{1}{3} ml^2$$

Hello there I am

Janish Ansari and

IDR

Three Phase Drives

1. Three phase half wave converter drive 2Q, 40kW
2. Three phase semi converter drive 1Q, 115 kW
3. Three phase full converter drive 2Q, 1500 kW
4. Three phase dual converter drive 4Q, 1800 kW

$$V_a = \frac{3\sqrt{2}}{2\pi} V_m \cos \alpha$$

$$V_f = \frac{3\sqrt{2}}{2\pi} V_m (1 + \alpha)$$

$$V_a = \frac{3\sqrt{2}}{2\pi} V_m (1 + 0.5\alpha)$$

$$V_f = \frac{3\sqrt{2}}{2\pi} V_m (1 + 0.5\alpha)$$

$$V_a = \frac{3\sqrt{2}}{2\pi} V_m \cos \alpha$$

$$V_f = \frac{3\sqrt{2}}{2\pi} V_m \cos \alpha$$

Single Phase Drives

1. Single Phase half - wave converter drive 1Q, 1/2 kW
2. Single Phase semi converter drive 1Q, 15 kW
3. Single Phase full converter drive 2Q, 15 kW
4. Single Phase dual converter drive 4Q, 15 kW

$$V_m = V_m (H \otimes K)$$

$$V_f = \frac{V_m}{2\pi} (1 + \alpha)$$

$$V_a = \frac{V_m}{2\pi} (1 + 0.5\alpha)$$

$$V_f = \frac{V_m}{2\pi} (1 + 0.5\alpha)$$

$$V_a = \frac{2\sqrt{2}}{\pi} V_m \cos \alpha$$

$$V_f = \frac{2\sqrt{2}}{\pi} V_m \cos \alpha$$

$$V_a = \frac{2\sqrt{2}}{\pi} V_m \cos \alpha$$

$$V_f = \frac{2\sqrt{2}}{\pi} V_m \cos \alpha$$

$$E_g = K_v \omega I_f$$

$$V_a = R_a I_a + E_g = R_a I_a + K_v \omega I_f$$

$$T_d = K_t I_a I_f = K_v I_f I_a$$

$$V_m = \sqrt{2} V_s$$

$$\omega = \Omega_m \cdot \frac{\pi}{30}$$

$$P_a = V_a I_a \quad P_f = V_a I_a + V_f I_f$$

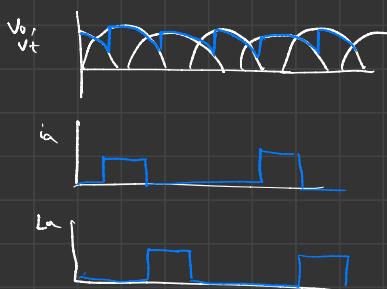
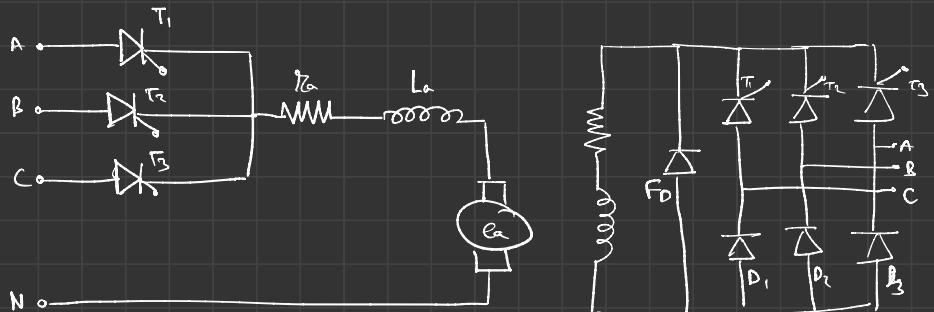
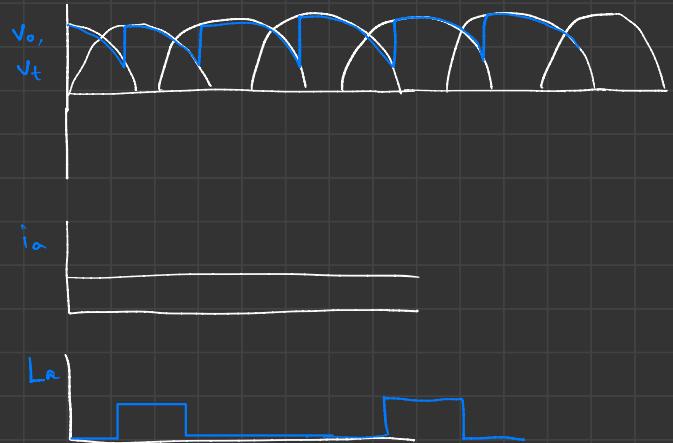
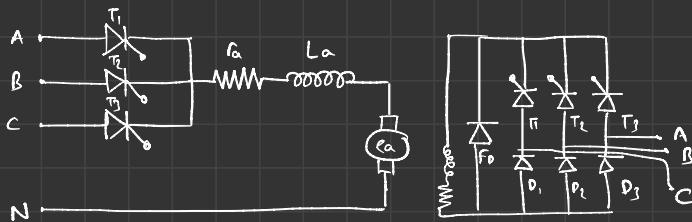
$$P_f = V_s I_{sa} = P_a$$

$$I_{sa} = I_a \left(\frac{\pi - \alpha}{\pi} \right)^{1/2}$$

$$PF = \frac{P_a}{V_s I_{sa}}$$

$$V_f = \frac{V_L}{\sqrt{3}} \quad V_m = \sqrt{2} V_f$$

Three Phase half wave converter drive



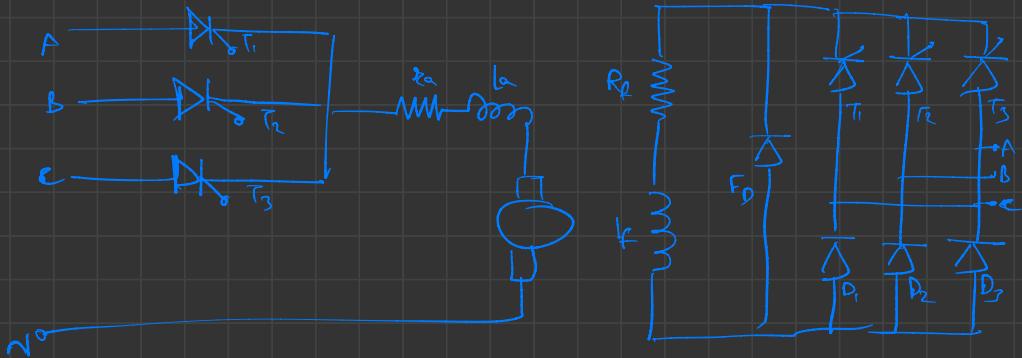
Zigzag output

60Kw/h

Not used in industrial
ac supply owing to comp.

$$V_a = \frac{3\sqrt{3} V_m}{2\pi} \cos \alpha$$

$$V_f = \frac{3\sqrt{3} V_m}{2\pi} (1 + \cos \alpha)$$

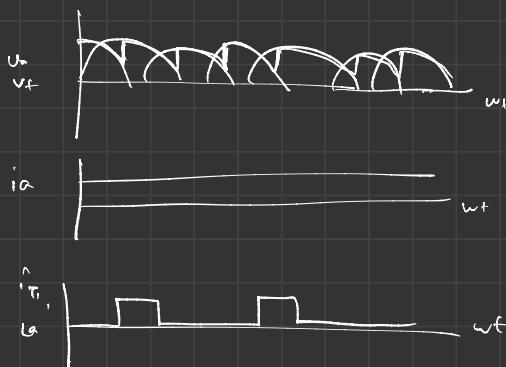


$$V_a = \frac{3\sqrt{3} V_m}{2\pi} \cos \alpha$$

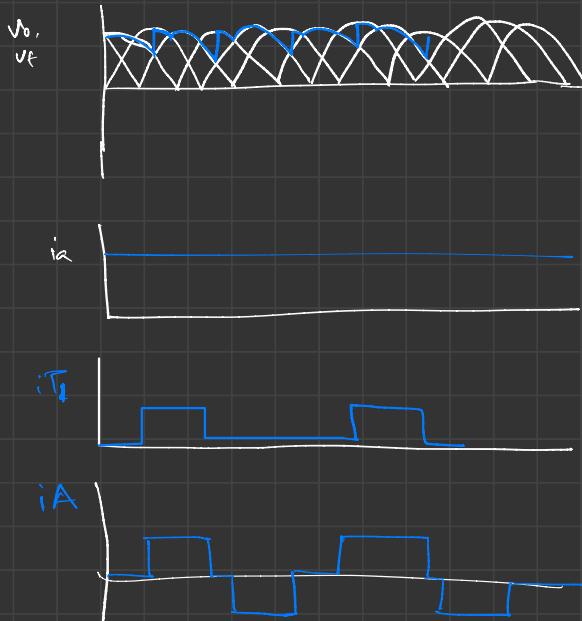
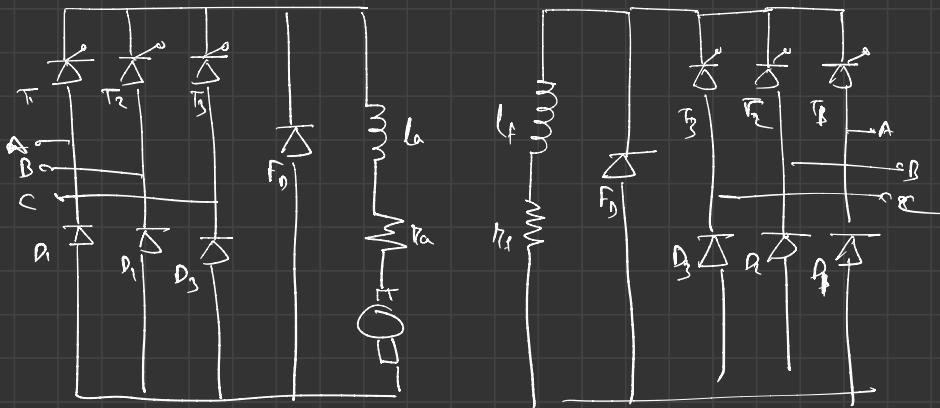
$$V_f = \frac{3\sqrt{3} V_m}{2\pi} (\cos \alpha_f)$$

$$V_a = \frac{3\sqrt{3} V_m}{2\pi} \cos \alpha$$

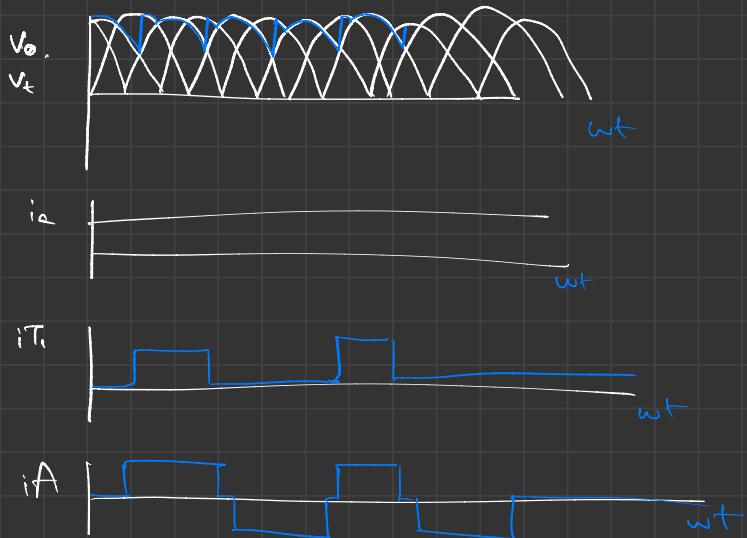
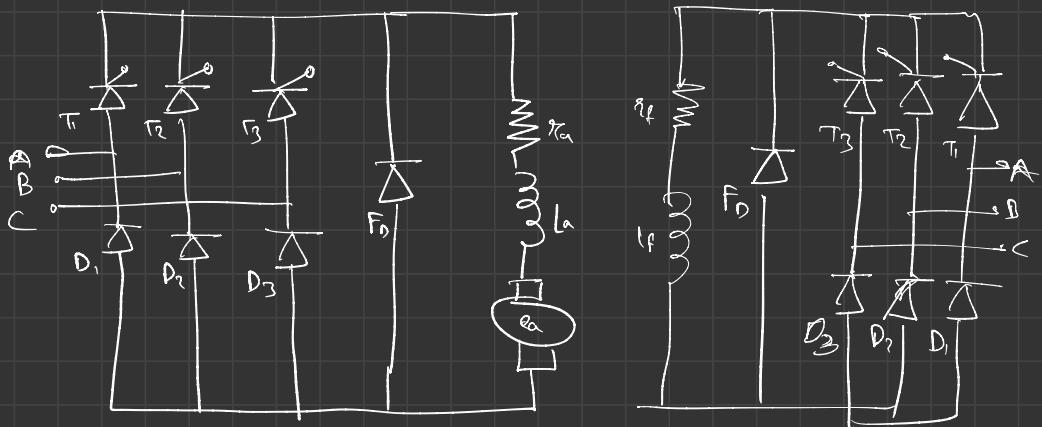
$$V_f = \frac{3\sqrt{3} V_m}{2\pi} (1 + \cos \alpha)$$



Three phase semi converter drive



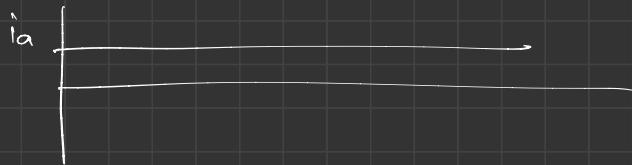
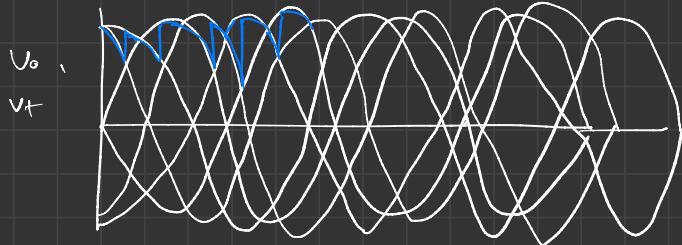
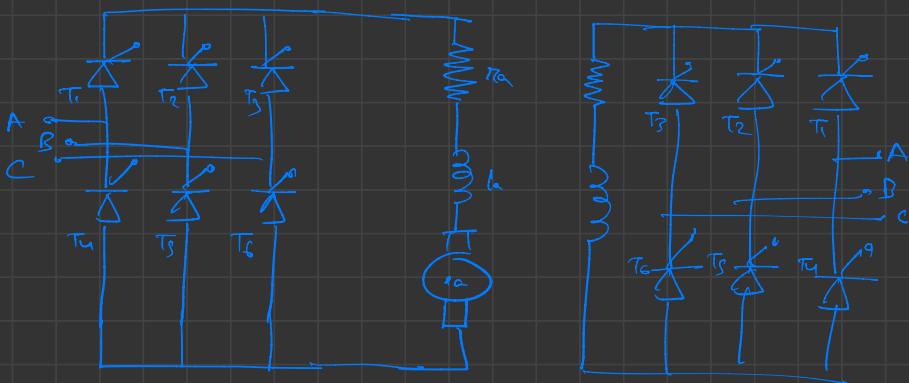
Three phase Semi converter drive



$$V_d = \frac{3\sqrt{3} V_m}{2\pi} (1 + \cos \alpha_a)$$

$$V_f = \frac{3\sqrt{3} V_m}{2\pi} (1 + \cos \alpha_r)$$

Three Phase full converter Drive

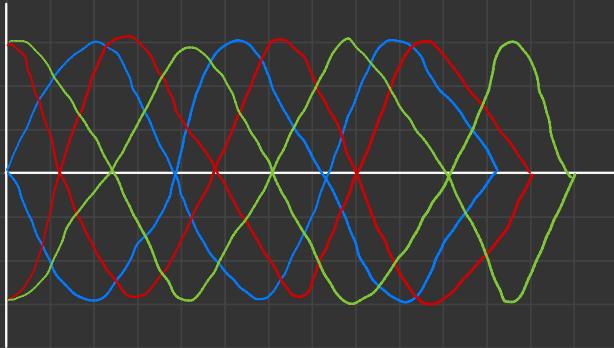


$$V_B = \frac{3\sqrt{3}}{\pi} G_S \alpha_B$$

$$V_T = \frac{3\sqrt{3}}{\pi} G_S \alpha_T$$



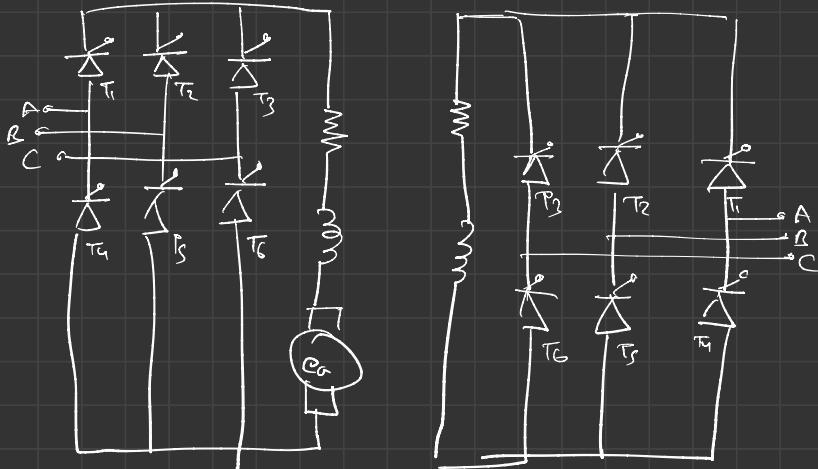
sno.	wt	incmin scr	Conductr pair	Outgoing scr	line Voltry
1	$30 + \alpha$	T_1	(T_6, T_1)	T_6	E_{AB}
2	$90 + \alpha$	T_2	(T_1, T_2)	T_5	E_{AC}
3	$150 + \alpha$	T_3	(T_2, T_3)	T_4	E_{BC}
4	$210 + \alpha$	T_4	(T_3, T_4)	T_2	E_{BA}
5	$270 + \alpha$	T_5	(T_4, T_5)	T_3	E_{CA}
6	$330 + \alpha$	T_6	(T_5, T_6)	T_4	E_{CB}



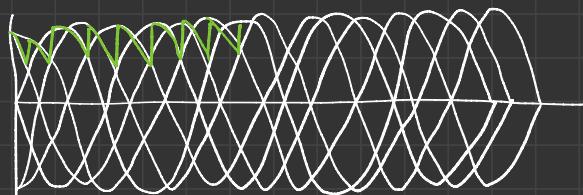
III-Phase Full Wave

Rectifier

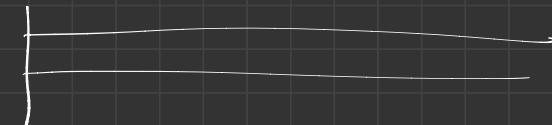
Three phase full converter drive



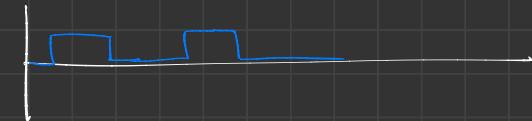
v_g ,
 v_f



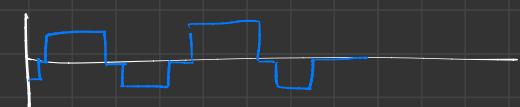
i_a



i_T



i_A



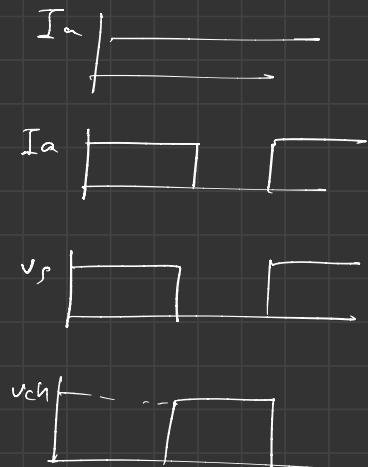
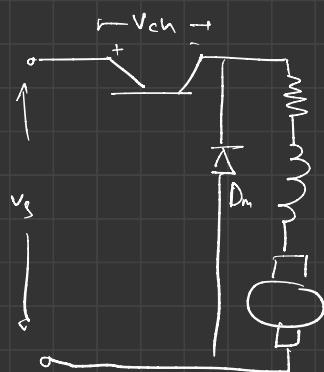
$$v_a = \frac{3\sqrt{3} V_m}{\pi} \cos \alpha_a \quad 0 \leq \alpha_a \leq \pi$$

$$v_f = \frac{3\sqrt{3} V_m}{\pi} \cos \alpha_f \quad 0 \leq \alpha_f \leq \pi$$

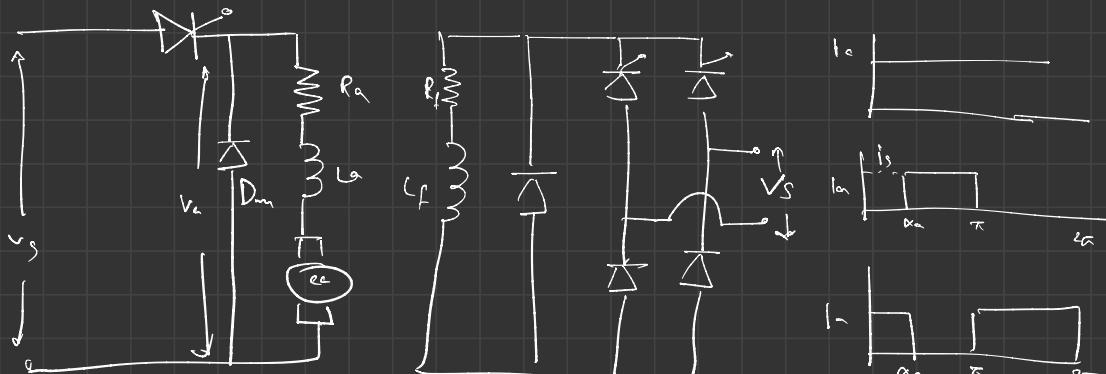
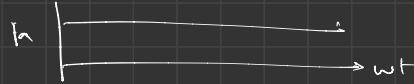
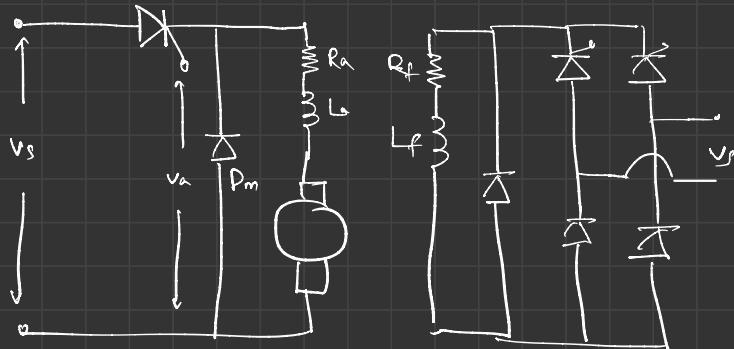
Sno.	wt	latching SCR	Conducting pair	Outgoing SCR	Line Voltage
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SNo.	wt	latching SCR	conducting pair	Outgoing SCR	Line Voltage
1	$30 + \alpha$	T ₁	T ₆ , T ₁	T ₆	E _{AB}
2	$90 + \alpha$	T ₂	T ₁ , T ₂	T ₅	E _{AC}
3	$150 + \alpha$	T ₃	T ₂ , T ₃	T ₁	E _{BC}
4	$210 + \alpha$	T ₄	T ₃ , T ₄	T ₂	E _{BA}
5	$270 + \alpha$	T ₅	T ₄ , T ₅	T ₃	E _{CA}
6	$330 + \alpha$	T ₆	T ₅ , T ₆	T ₄	E _{CB}

Chopper driven



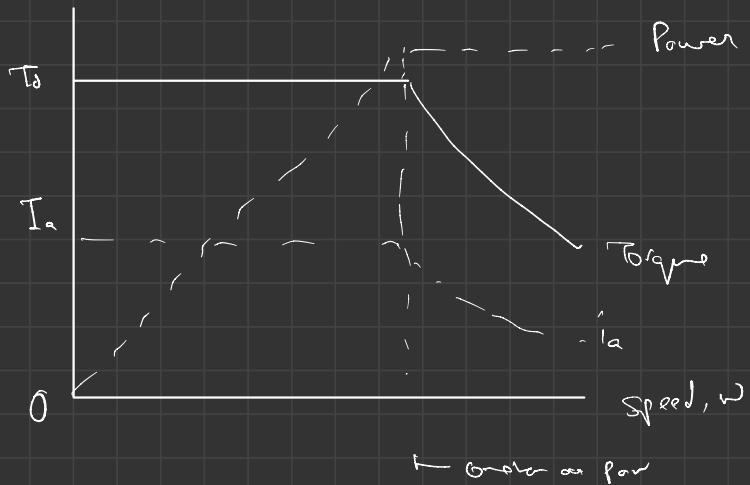
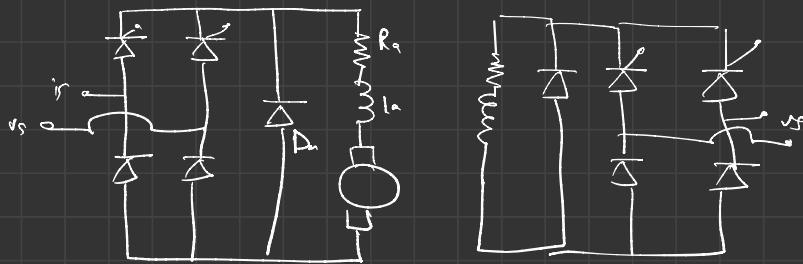
Single Phase half - wave converter drive



$$V_a = \frac{V_m}{\pi} (1 + \cos \alpha_a)$$

$$V_f = \frac{V_m}{\pi} (1 + \cos \alpha_f)$$

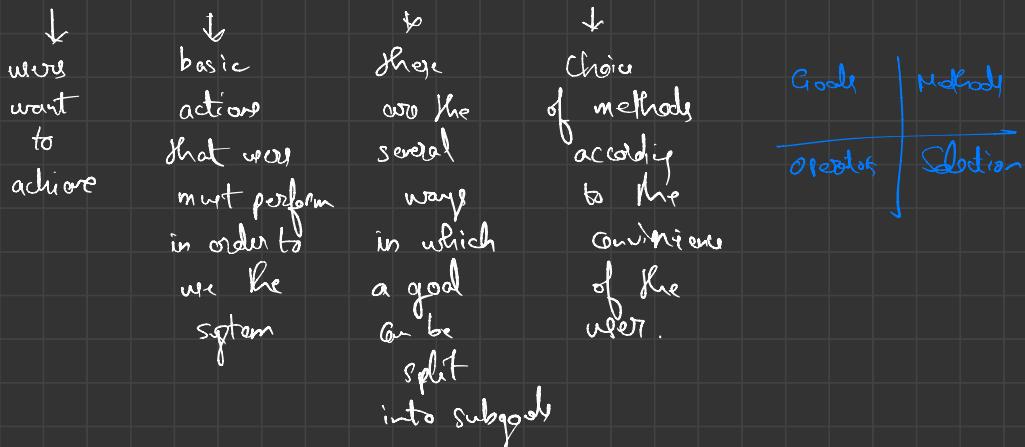
Single Phase Semi Converter drive



HMI

GOMS

→ Goals, Operators, Methods, Selection



Cognitive Model

- An area in computer science which deals with simulating human problem solving and mental processing in a computerized mode.
Cognitive model can be used to simulate and predict human behaviour or performance on tasks in an already modeled system to enhance it.

WATERFALL METHOD

Requirement Specification

↳ Architectural Design

↳ Detailed Design

↳ Coding and Unit testing

↳ Integration and testing

↳ Operation and maintenance .

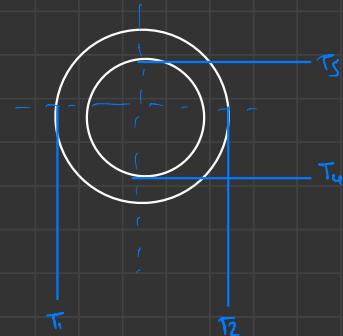
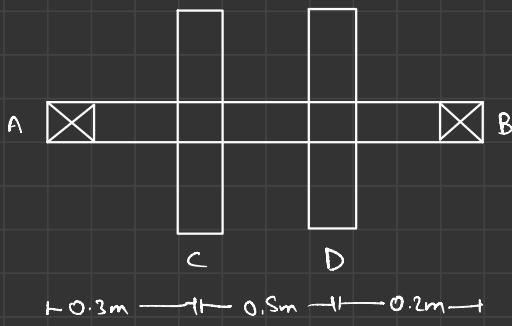
Shneiderman's 8 Golden Rules

1. Strive for consistency
2. Enable frequent users to use shortcuts
3. Offer informative feedback
4. Design dialogs to yield closure
5. Offer error prevention and simple error handling
6. Permit easy reversal of actions
7. Support internal locus of control
8. Reduce short - term memory load .

DME

Q. shaft - pulley $T = \text{shear}$, $\sigma_b = \text{tension}$

A.



$$T_1 = 3.2 \times 10^3 \text{ KN}$$

$$2.3 \log \left(\frac{T_1}{T_2} \right) = \mu \theta$$

$$T_2 = 1.852 \times 10^3 \text{ N}$$

$$T = (T_1 - T_2) R_c$$

$$T = 402 \text{ Nm} = F_D$$

$$W_c = T_1 + T_2 = 5.08 \times 10^3 \text{ N}$$

$$\text{Since, } \sum y = 0$$

$$W_c - R_{AV} - R_{BV} = 0$$

$$W_c = R_{AV} + R_{BV}$$

$$\sum M_B = 0$$

$$R_{AV} \times 1 - W_c \times 0.7 = 0$$

$$R_{AV} = 3556 \text{ N}$$

$$R_{BV} = 1524 \text{ N}$$

$$\text{Since, } T_D = (T_3 - T_4) \times R_D$$

$$T_3 - T_4 = \frac{402}{0.2}$$

$$T_3 - T_4 = 2010$$

$$2.3 \log \left(\frac{T_3}{T_4} \right) = M_0$$

$$\frac{T_3}{T_4} = 1.7$$

$$T_3 = 4881.42 \text{ N}$$

$$T_4 = 2871.42 \text{ N}$$

$$W_D = T_3 + T_4 = 7752.84$$

$$\Sigma M = 0$$

$$R_{AV} - 1 \times W_D \times 0.8 = 0$$

$$R_{AV} = 6202.272$$

$$W_D = R_{AV} + R_{BV}$$

$$R_{BV} = 1550.56$$

Bending Moment Calculations

i) Vertical Loading

$$BM @ A = 0$$

$$BM @ B = 0$$

$$BM @ C = R_{AV} \times 0.3 = 1066.8$$

$$BM @ D = R_{BV} \times 0.2 = 457.2$$

ii) Horizontal Loading

$$BM @ A = 0$$

$$BM @ B = 0$$

$$BM @ C = R_{B1} \times 0.3 = 679.75$$

$$BM @ D = R_{B2} \times 0.2 = 1586.02$$

Resultant Bending Moment

$$@) C = \sqrt{1066.8^2 + 679.75^2} = 1264.95$$

$$@) D = \sqrt{457.2^2 + 1586.02^2} = 1650.60$$

Bending moment $M = 1650.60 \text{ N}$

$$T_e = \sqrt{M^2 + T^2} = 1698.84$$

$$M_e = \frac{1}{2} \left[M + \sqrt{M^2 + T^2} \right] = 1674.72 \text{ N}$$

$$T_e = \frac{\pi}{16} \times T \times d^3$$

$$d = \left(\frac{T_e \times 16}{\pi \times T} \right)^{1/3} = 59.05$$

$$M_e = \frac{\pi}{32} \times \sigma_b \times d^3$$

$$d = \left(\frac{M_e \times 32}{\pi \times \sigma_b} \right)^{1/3}$$

$$d = 0.064 \text{ m} = 64.69 \text{ mm}$$

\therefore Larger Value $\Rightarrow d = 64.69 \text{ mm}$

C. shaft (Pulley)

$$\tau_1, \quad 2.3 \log\left(\frac{\tau_1}{\tau_2}\right) = \mu \Theta \quad \mu=0.4$$

$$\tau_2, \quad T = (\tau_1 - \tau_2) \times R_C = T_D$$

$$W_C = \tau_1 + \tau_2$$

$$\Sigma y = 0, \quad W_C - R_{AV} - R_{BV} = 0$$

$$W_C = R_{AV} + R_{BV}$$

$$\Sigma M = 0 \quad R_{AV} \times 1 - W_C \times 0.7 = 0$$

$$R_{AV}, \quad R_{BV}$$

$$T_D = (\tau_3 - \tau_4) \times R_D$$

$$2.3 \log\left(\frac{\tau_3}{\tau_4}\right) = \mu \Theta$$

$$\tau_3, \quad \tau_4, \quad W_D = \tau_3 + \tau_4$$

$$\Sigma M = 0$$

$$R_{AV} \times 1 - W_D \times 0.8 = 0$$

$$R_{AV}, \quad W_D = R_{AV} + R_{BV}, \quad R_{BV}$$

$$B_M @ C \dots i) \sqrt{L} \Rightarrow B_M @ A = 0$$

$$B_M @ B = 0$$

$$B_M @ C = R_{AV} \times R_C$$

$$B_M @ D = R_{BV} \times R_D$$

$$ii) \sqrt{L} \Rightarrow B_M @ A = 0$$

$$B_M @ B = 0$$

$$B_M @ C = R_{AV} \times R_C$$

$$B_M @ D = R_{BV} \times R_D$$

$$\text{Resultant} \rightarrow @ C = \sqrt{\dots^2 + \dots^2} = \underline{\quad}$$

$$@ D = \sqrt{\dots^2 + \dots^2} = \underline{\quad}$$

Max Value $\rightarrow M$

$$T_e = \sqrt{M^2 + r^2}$$

$$M_e = \frac{1}{2} [M + \sqrt{M^2 + r^2}]$$

$$T_e = \frac{\pi}{16} \times T \times d^3$$

$$M_e = \frac{\pi}{32} \times \sigma_b \times d^3$$

$$T_e = \frac{\pi}{16} \times T \times d^3$$

$$M_e = \frac{\pi}{32} \times \sigma_b \times d^3$$

Q. Shaft (Pulley)

$$T_1, 2.3 \log \left(\frac{T_1}{T_2} \right) = u \theta, T_2$$

$$T = (T_1 - T_2) R_C = T_d$$

$$W_c = T_1 + T_2$$

$$\sum y = 0, W_c - R_{AV} - R_{BV} = 0$$

$$W_c = R_F + R_B$$

$$\sum M_B = 0, R_{AV} \times 1 - W_c \times 0.7 = 0$$

$$R_{AV}, R_{BV}$$

$$T_D = (T_3 - T_4) R_D$$

$$2.3 \log \left(\frac{T_3}{T_4} \right) = u \theta$$

$$T_3, T_4$$

$$W_o = T_3 + T_4$$

$$\sum M = 0$$

$$R_{AH} \times 1 - W_D \times 0.8 = 0$$

R_{AH}

$$W_D = R_{AH} + R_{BU}$$

$$R_{BU}$$

BMC, \Rightarrow VL

$$BM @ A = 0$$

$$BM @ B = 0$$

$$BM @ C = R_{AV} \times R_C$$

$$BM @ D = R_{AV} \times R_D$$

VL) HL $BM @ A = 0$

$$BM @ B = 0$$

$$BM @ C = R_{AV} \times R_C$$

$$BM @ D = R_{AV} \times R_D$$

$$\text{constant } \alpha @ C \Rightarrow \sqrt{...^2 + ...^2} = \text{---}$$

$$@ D = . / = \text{---}$$

large value $\Rightarrow M$

$$T_e = \sqrt{M^2 + T^2}$$

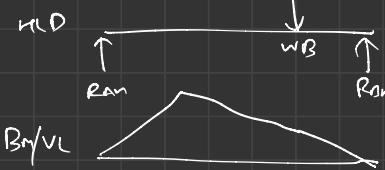
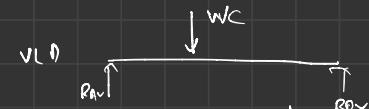
$$M_e = \frac{1}{2} \left[M + \sqrt{M^2 + T^2} \right]$$



$$T_e = \frac{\pi}{16} \times T \times d^3$$

$$M_e = \frac{\pi}{32} \times \sigma_B \times d^3$$

$$\text{Larger } \frac{d}{=}$$



BM/VL

BM/HL

Q) Shaft (gear)

A. $\alpha = 20^\circ$

$T = 35 \text{ mPa}$

$N = 210 \text{ rpm}$

$P = 6.2 \times 10^3 \text{ W}$

$$P = \frac{2\pi NT}{60}$$

$T = 282 \text{ Nm}$

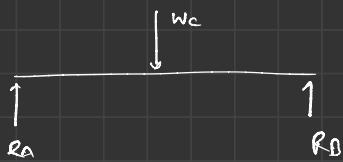
w_c = Normal load acting on gear

F_t = Tangential force acts on gear

$$F_t = \frac{2T}{D}$$

$$w_c = \frac{F_t}{\cos \alpha}$$

$$w_c = 5456.33 \text{ N}$$



$$R_A = R_B = \frac{w_c L}{2} = 2728.16 \text{ N}$$

$$M = \frac{w_c L}{4}$$

$$T_e = \sqrt{M^2 + T^2}$$

$$M_e = \frac{\pi}{32} \times \sigma_b \times d^3$$

$$T_e = \frac{\pi}{16} \times \tau \times d^3$$

$$Q. \quad \alpha = 20^\circ$$

$$T = 35 \text{ N/mm}^2$$

$$N = 210 \text{ rpm}$$

$$P = 6.2 \times 10^3 \text{ W}$$

$$P = \frac{2\pi NT}{60}$$

$$\Gamma = \text{_____}$$



$$F_t = \frac{2T}{D} \quad F_t = \frac{2T}{D} \quad P_e = \frac{2T}{D}$$

$$w_c = \frac{F_c}{G_s \alpha}$$

$$R_A = R_B = \frac{w_c}{2}$$

\therefore Max bending moment

$$M = \frac{wL}{4}$$

$$T_c = \sqrt{M^2 + r^2}$$

$$T_c = \frac{\pi}{16} \times T \times d^3$$

$$Q \quad \alpha = 20'$$

$$C = 85 \text{ N/mm}^2$$

$$N = 210 \text{ rpm}$$

$$P = 6.2 \times 10^3 \text{ W}$$

$$P = \frac{2\pi NT}{60}$$

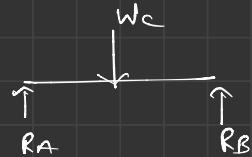
$$T = \underline{\hspace{2cm}}$$

w_c = Normal load

F_t = Tangential force

$$F_t = \frac{2T}{D}$$

$$w_c = \frac{F_t}{G\alpha}$$



∴ Simply Supported beam

$$R_A = R_B = \frac{w_c}{2}$$

∴ Max bending moment ,

$$M = \frac{w_c L}{4}$$

$$M = \underline{\hspace{2cm}}$$

$$Te = \sqrt{M^2 + T^2}$$

$$Te = \frac{\pi}{16} \times C \times d^3$$

$$Q \propto = \frac{1}{T} = 3S$$

$$N = 210 \text{ rpm}$$

$$P = 6.2 \times 10^3 W$$

$$P = \frac{2\pi NT}{60}$$

$$T = \underline{\quad}$$

w_c = normal load

F_t = tangential force

$$F_t = \frac{2F}{D}$$

$$w_c = \frac{F_t}{\cos \alpha}$$

As simply supported beam ,

$$R_A = R_B = \frac{w_c L}{2}$$

\therefore Max bending moment

$$M = \frac{w_c L^2}{4}$$

$$r_e = \sqrt{M^2 + r^2}$$

$$\hookrightarrow r_e = \frac{\pi}{16} \times I \times d^3$$

MCS - (Venkatesh Sir) Mathematical Approach
FCS / DSM & DSA
Plan motion of robot
State Space Model

DSIP - (Dattaroy Sir)
Advance of SAs
Z-transform & Discrete Fourier Transform
Processing images

Embedded System - (Deepak Sir)
MSI & I₂S / I₂C (CAN Protocol)
Program a task on any processor (16-bit
8-bit)

Biomedical Instrumentation - (Nirmal Sir)
ECG's made

ERAT - (Sunny Nanade)
Hybrid Vehicles
Alternatives of IC engine
Using battery to control automotive component
Advisor Software
Matlab & Simulink

Additive Manufacturing - (Ketna)
3-D printing methods of pieces
Cura, Blender & Multi-maker

CAD CAM CIM - (Chetna)
Solidworks, MasterCAM
Simulating 4-axis robot
GCode, MCode (CNC machine)
Designing parts & using
CNC machine to get output
Last practical - make something
in CNC machine.

TQM - (MAE)

- PPT
- Circuit Diagram
- Apparatus
- Softwares Used
- Functioning of our model
- Result

Any arbitrary signal can be expressed as linear combination of harmonically related sine & cosine components.

$$x(t) = a_0 + \sum_{n=1}^{\infty} (a_n \cos(n\omega_0)t + b_n \sin(n\omega_0)t)$$

$$x(t) = a_0 + \sum_{n=1}^{\infty} (a_n \cos(n\omega_0)t + b_n \sin(n\omega_0)t) \quad \text{①}$$

$$\text{Since } A \cos x + B \sin x = C \cos(x - \Theta)$$

$$A \cos x - B \sin x = C \cos(x + \Theta)$$

Then ① can be written as

$$x(t) = C_0 + \sum_{n=1}^{\infty} (C_n \cos(n\omega_0 t - \Theta_n)) \quad \text{or} \quad x(t) = C_0 + \sum_{n=1}^{\infty} C_n \cos(n\omega_0 t + \Theta_n)$$

$$C_0 = a_0$$

$$C_0 = a_0$$

$$|C_n| = \sqrt{a_n^2 + b_n^2}$$

$$|C_n| = \sqrt{a_n^2 + b_n^2}$$

$$\Theta_n = \tan^{-1} \left(\frac{b_n}{a_n} \right)$$

$$\Theta_n = \tan^{-1} \left(\frac{-b_n}{a_n} \right)$$

$$x(t) = a_0 + \sum_{n=1}^{\infty} (a_n \cos(n\omega_0 t) + b_n \sin(n\omega_0 t))$$

$$\text{Since, } A \cos x + B \sin x = C \cos(x - \Theta)$$

$$\text{①} \Rightarrow x(t) = C_0 + \sum_{n=1}^{\infty} C_n \cos(n\omega_0 t - \Theta_n)$$

$$A \cos x - B \sin x = C \cos(x + \Theta)$$

$$x(t) = C_0 + \sum_{n=1}^{\infty} C_n \cos(n\omega_0 t + \Theta_n)$$

$$C_0 = a_0$$

$$|C_n| = \sqrt{a_n^2 + b_n^2}$$

$$\Theta_n = \tan^{-1} \left(\frac{b_n}{a_n} \right)$$

$$C_0 = a_0$$

$$|C_n| = \sqrt{a_n^2 + b_n^2}$$

$$\Theta_n = \tan^{-1} \left(\frac{-b_n}{a_n} \right)$$

$$x(t) = a_0 + \sum_{n=1}^{\infty} (a_n \cos(n\omega_0)t + b_n \sin(n\omega_0)t)$$

Since, $A \cos x - B \sin x = (\cos(x-\theta))$ & $A \cos x + B \sin x = (\cos(x+\theta))$

Then (1) $x(t) = C_0 + \sum_{n=1}^{\infty} C_n \cos(n\omega_0 t - \phi_n)$ & $x(t) = C_0 + \sum_{n=1}^{\infty} C_n \cos(n\omega_0 t + \phi_n)$

$$a_0 = C_0$$

$$C_0 = a_0$$

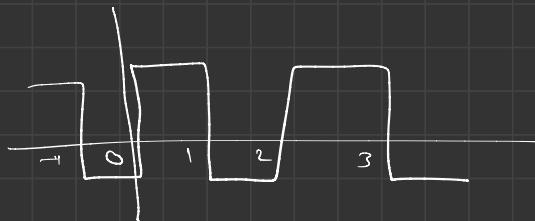
$$|C_n| = \sqrt{a_n^2 + b_n^2}$$

$$|C_n| = \sqrt{a_n^2 + b_n^2}$$

$$\theta = \tan^{-1} \left(\frac{b_n}{a_n} \right)$$

$$\phi = \tan^{-1} \left(\frac{-b_n}{a_n} \right)$$

A.



i) $a_0 = 0$

ii) Odd signal $\therefore x(t) = -x(-t)$

$$x(t) = A, \quad 0 < t < 1$$

$$x(t) = -A, \quad -1 < t < 0$$

$$a_n = 0$$

iii) $x(t) = \sum_{n=1}^{\infty} b_n \sin(n\omega_0 t)$

$$b_n = \frac{2}{T_0} \int_{T_0}^{} x(t) \sin(n\omega_0 t) dt$$

$$T_0 = 2$$

$$\begin{aligned}x(t) &= A, \quad 0 < t < 1 \\x(t) &= -A, \quad 1 < t < 2\end{aligned}$$

$$b_n = \frac{2}{T_0} \int_{T_0}^1 x(t) \sin n\omega_0 t$$

$$b_n = \frac{1}{2} \left[\int_0^1 A \sin n\omega_0 t dt + \int_1^2 -A \sin n\omega_0 t dt \right]$$

$$= \frac{1}{2} \left[A \left(-\frac{\cos n\omega_0 t}{n\omega_0} \right)_0^1 - A \left(\frac{\sin n\omega_0 t}{n\omega_0} \right)_1^2 \right]$$

$$= \frac{A}{n\omega_0} \left[-(\cos n\omega_0 - \cos 0) - (\sin 2n\omega_0 - \sin n\omega_0) \right]$$

$$= \frac{A}{n\omega_0} \left[-(\cos n\omega_0 - \cos 0) - (\sin 2n\omega_0 - \sin n\omega_0) \right]$$

$$= \frac{A}{n\omega_0} \left[-\cos n\omega_0 + \cos 0 + \cos 2n\omega_0 - \cos n\omega_0 \right]$$

$$= \frac{A}{n\omega_0} \left(-(-1)^n + 1 + 1 - (-1)^n \right)$$

$$= \frac{A}{n\omega_0} \left(-2(-1)^n + 2 \right)$$

$$b_n = \frac{2A}{n\pi} [1 - (-1)^n]$$

$$x(t) = a_0 + \sum_{n=1}^{\infty} (a_n \cos n\omega_0 t + b_n \sin n\omega_0 t)$$

$$= 0 + \sum_{n=1}^{\infty} b_n \sin n\omega_0 t$$

$$x(t) = \sum_{n=1}^{\infty} \frac{2A}{n\pi} \left[\frac{1 - (-1)^n}{n} \right] \sin n\omega t$$

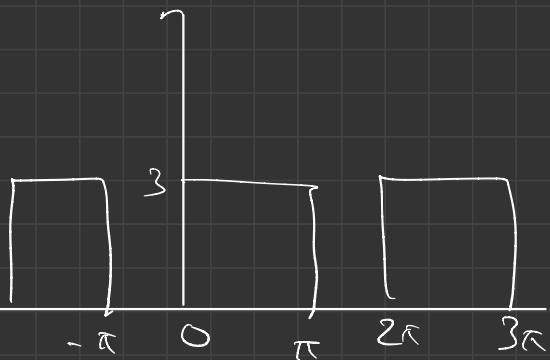
$$\begin{aligned} n &= 2, 4, 6 \quad \text{--- even} \\ &= 1, 3, 5 \quad \text{--- odd} \end{aligned}$$

when $n = \text{even}$, $(1) - (-1)^n = 0$
 $n = \text{odd}$, $(1) - (-1)^n = 2$

$$x(t) = \frac{2A}{\pi n} [2 \sin n\pi t], \quad n = \text{odd}$$

$$x(t) = \frac{4A}{\pi} \left[\frac{\sin \pi t}{1} + \frac{\sin 3\pi t}{3} + \frac{\sin 5\pi t}{5} + \dots \right]$$

Q.



$$x(t) = Q_0 + \sum_{n=1}^{\infty} (a_n \cos n\omega t + b_n \sin n\omega t)$$

i) Not symmetric.

$$a_n = \frac{1}{T_0} \int_{T_0} x(t) dt$$

$$T_0 = 2\pi$$

$$\omega_0 = \frac{2\pi}{T_0} = 1$$

$$\therefore a_0 = \frac{1}{2\pi} \left[\int_0^\pi 3\sin t + \int_\pi^{2\pi} 0 \, dt \right]$$

$$a_0 = \frac{1}{2\pi} (3(t))_0^{\pi}$$

$$a_0 = \frac{3}{2\pi} (\pi)$$

$$a_0 = \frac{3}{2}$$

$$a_n = \frac{2}{T_0} \int_{T_0}^\pi x(t) \cos n\omega_0 t \, dt$$

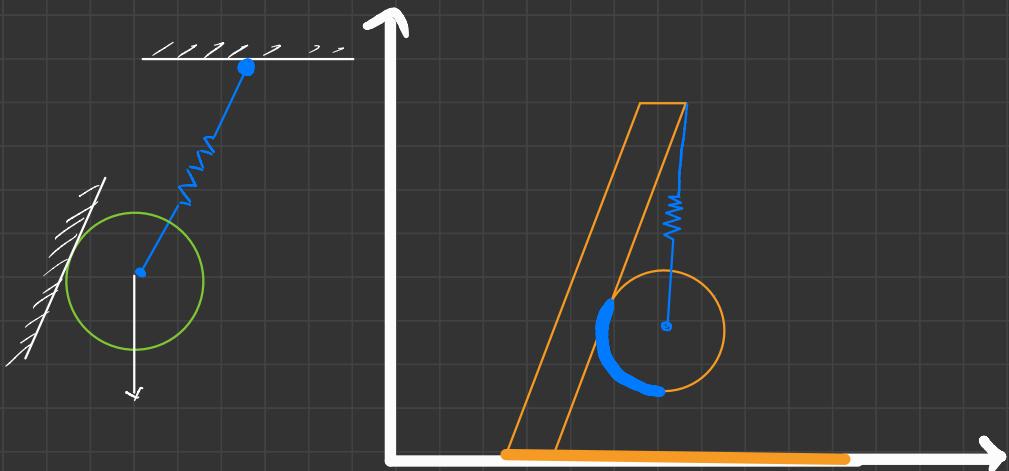
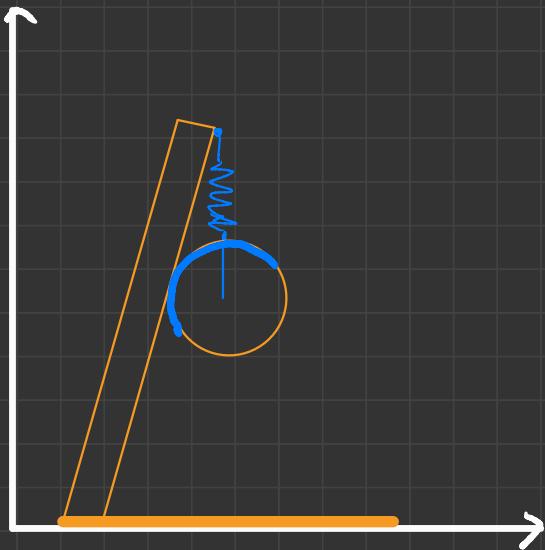
$$a_n = \frac{2}{2\pi} \left[\int_0^\pi 3\sin t \, dt + \int_\pi^{2\pi} 0 \, dt \right]$$

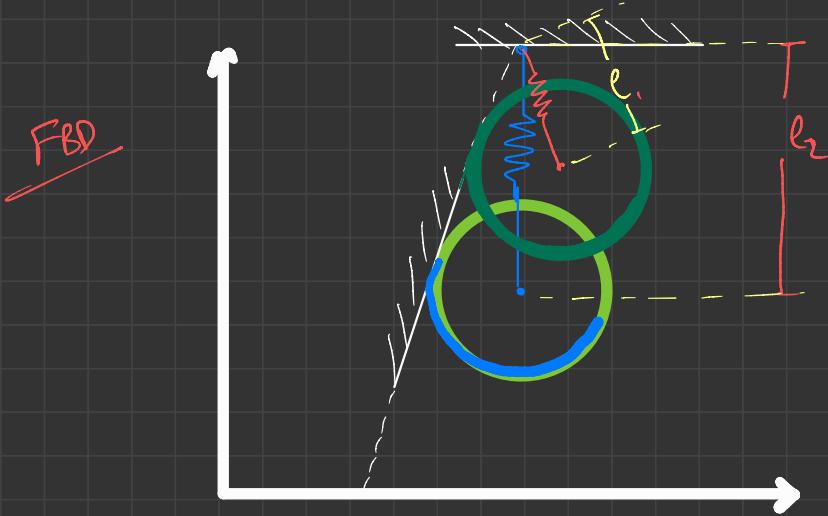
$$a_n = \frac{3}{\pi} \left[\frac{\sin nt}{n} \right]_0^\pi$$

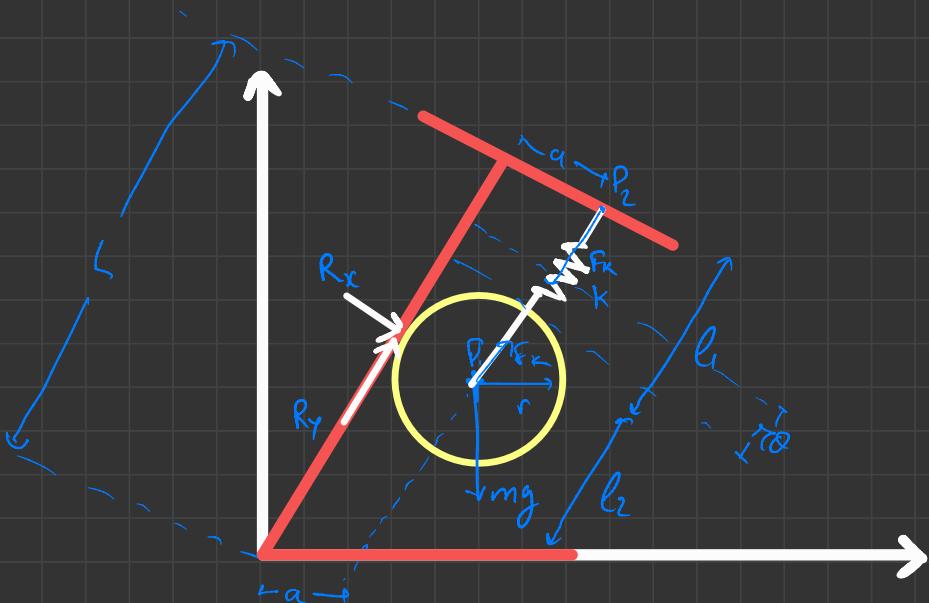
$$a_n = \frac{3}{n\pi} [\sin \pi - \sin 0]$$

$$a_n = 0$$

DSM PROJECT





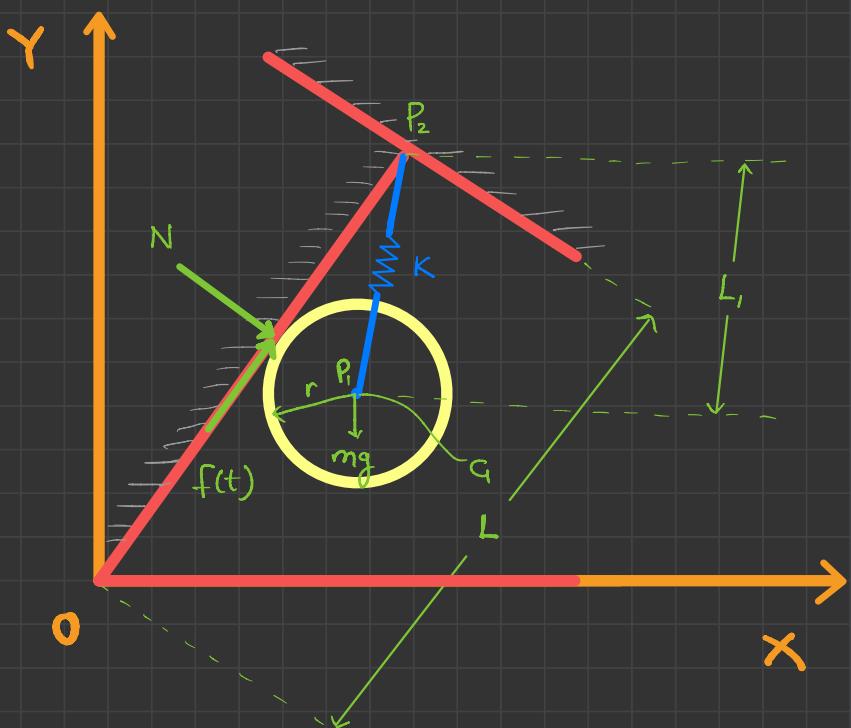


Forces

$$F_{k1} = k_1 (\delta_{R, p_2} - L)$$

$$\overline{\delta_{op_1}} =$$

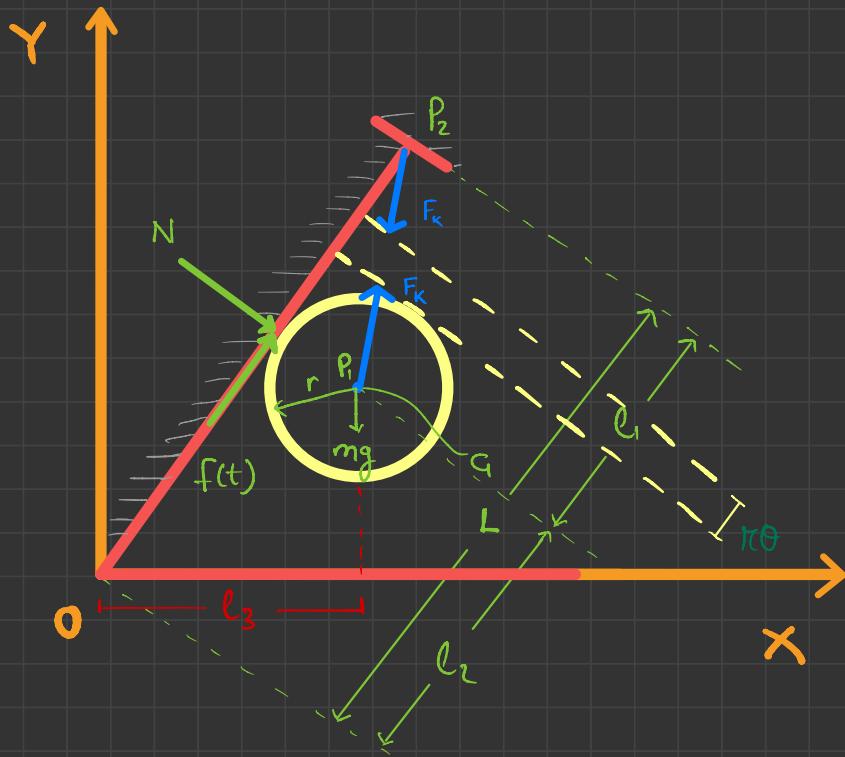




initial velocity :-

$$\text{Time of flight}, \quad t = \frac{2v_0 \sin \theta}{g}$$

$$\text{Initial velocity}, \quad v_0 = \frac{t g \theta}{2 \sin \theta} = 1.56 \text{ m/s}$$



$$\sum F_x = m a_{ax}$$

$$N \cos \theta - F_k \cos \theta_K = m \left[-l_3(\sin \theta) \ddot{\theta} - l_3(\cos \theta) \dot{\theta}^2 \right]$$

$$\sum F_y = m a_{ay}$$

$$f(t) \sin \theta - mg + F_k \sin \theta_K = m \left[-l_3(\cos \theta) \ddot{\theta} + l_3(\sin \theta) \dot{\theta}^2 \right]$$

$$\begin{aligned}
 \text{Translational KE} &= \frac{1}{2} m \dot{\overline{x}}_{\text{OG}} \cdot \dot{\overline{x}}_{\text{OG}} \\
 &= \frac{1}{2} m \left[(\ell_3 \sin \theta) \dot{\theta})^2 + (\ell_3 (\cos \theta) \dot{\theta})^2 \right. \\
 &\quad \left. + 2 \ell_3^2 (\sin \theta \cos \theta) \dot{\theta}^2 \right]
 \end{aligned}$$

$$\begin{aligned}
 \text{Rotational KE} &= \frac{1}{2} I_g \dot{\theta}^2 \\
 &= \frac{1}{2} m r^2 \dot{\theta}^2
 \end{aligned}$$

$$\begin{aligned}
 \text{Restoring PE} &= \frac{1}{2} k_1 (\overline{x}_{\text{PP}_L} - L_1) \\
 &= \frac{1}{2} k_1 \left(\left[[L - \ell_3 \cos \theta]_r + [\ell_3 \sin \theta - \ell_2]_r \right]^2 - L_1 \right)
 \end{aligned}$$

$$\begin{aligned}
 \text{Gravitational PE} &= mg \overline{x}_{\text{OG}} \cdot \uparrow \\
 &= mg (L_2 - \ell_3 \sin \theta)
 \end{aligned}$$

$$\frac{P_{\text{diss}}}{P_{\text{in}}} \times 100$$

- ① M1 & M2 Papers } 50 - 60 marks
② Assignments 1 & 2 }

[25 - Nextion Editor]

- ③ GOMS Model, Waterfall Method Model
④ Design Process
⑤ Implementation
⑥ Schneiderman's 8 Golden Rules
⑦ Standards :- ISO 9241

↳ Principles to support Usability (Design Rules)

→ ↳

Unit → Overview (Point wise)

Software Process

Golden Rules

Design Rules

- ✓ Unit 1 - 3 } User model
✓ Unit 1 - 5 } Stakeholder
 } Socio-organisational issues

{ Cognitive model }
{ Predictive model }

Heuristic Evaluation

GUI &/s webpage } ↳ 3

HMI

GOMS MODEL

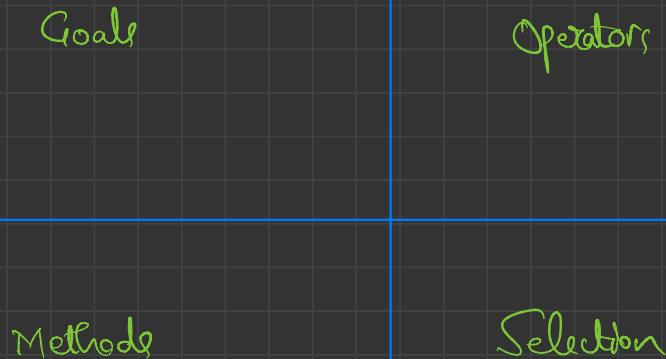
Goals, Operators, Methods, Selection

Goals :- What the user wants to achieve

Operators :- Basic actions that user performs

Methods :- Decomposition of goals into subgoals / operators.

Selection :- Means of choosing b/w competing methods.



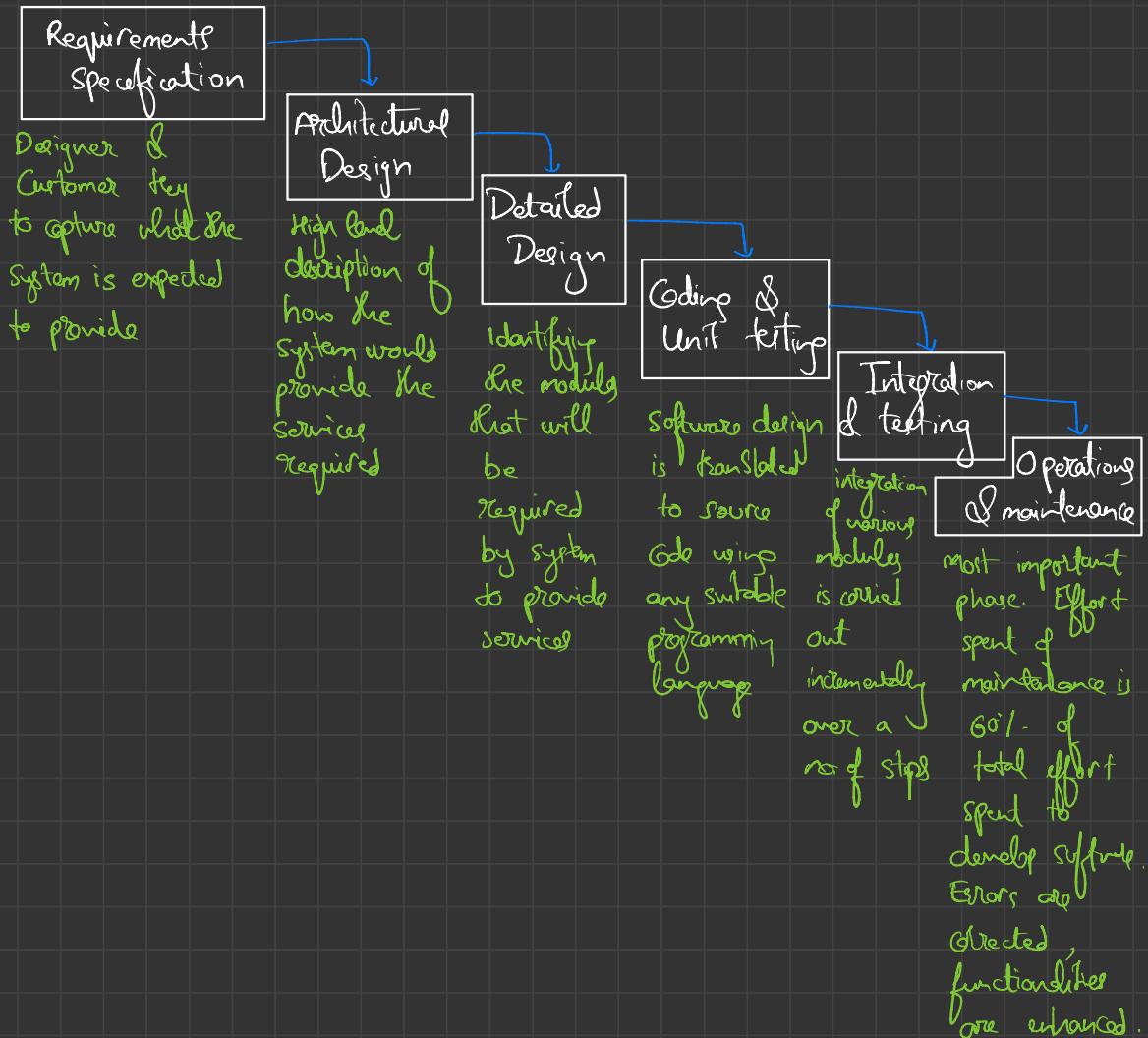
Shneiderman's 8 Golden Rules

SEODOPSR

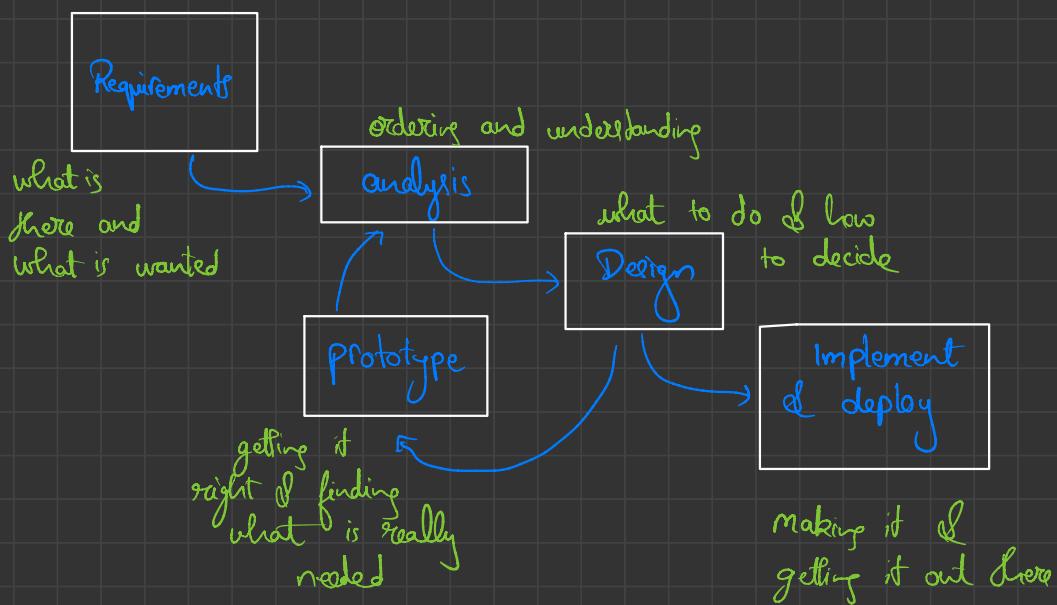
1. Stare for consistency MS Word / Excel / Mac OS
2. Enable frequent users to use shortcuts Ctrl C / Ctrl A
3. Offer informative feedback Software Update / News post notification / Spell checker
4. Design dialogues to yield closure Download complete / Move to trash
5. Offer error prevention & simple error handling Auto correction
6. Permit easy reversal of actions Undo / Redo / Back
7. Support internal locus of control Cancel option / Reset option
8. Reduce short term memory load Captcha / otp paste

The Waterfall Method

RAD CIO



The Process of Design



Scenarios

Scenarios are stories for design : rich stories of interaction. They are perhaps the simplest design representation, but one of the most flexible & powerful.

Scenarios can be augmented by sketches, simulated screenshots, etc. These sketches and pictures are called storyboards.

Scenarios can be used to

- communicate with other designers, clients or users.
- validate other models
- Express dynamics

Heuristic

- is a guideline or general principle or rule of thumb that guides design decision.
- flexible, relatively cheap. Called as discount usability technique.
- Several evaluate independently critique a system.
- 3-5 evaluators
- 10 point evaluation done on project
- each point given a no. (0-4)

Human Ear

consists of outer, middle & inner ear

Outer ear :- protects the middle & inner ear and maintains the middle ear at constant temperature

Middle ear :- Amplifies the sound and has small delicate bones

Inner ear :- Has fluid through which waves are transmitted. Delicate hair bend because of vibrations and release chemicals.

Stakeholders

In an organisation, the different people affected by the introduction of a system are known as stakeholders and their needs can be both complex and conflicting.

Primary Stakeholders : actually using the system
eg. students

Secondary Stakeholders : receive output / provide input
eg. faculty

Tertiary Stakeholders : no direct involvement but affected by success or failure.
eg. parents

Facilitating Stakeholders : involved in development or deployment of system.
eg. IT department, design team

draw line	::= select line + choose points + last point
select line	::= pos mouse + CLICK MOUSE
choose points	::= choose one choose one + choose points
choose one	::= pos mouse + CLICK MOUSE
last point	::= pos mouse + DBL CLICK MOUSE
pos mouse	::= NULL MOVE MOUSE+ pos mouse

Backus - Naur Form (BNF)

Basic syntax :- Nonterminal ::= expression

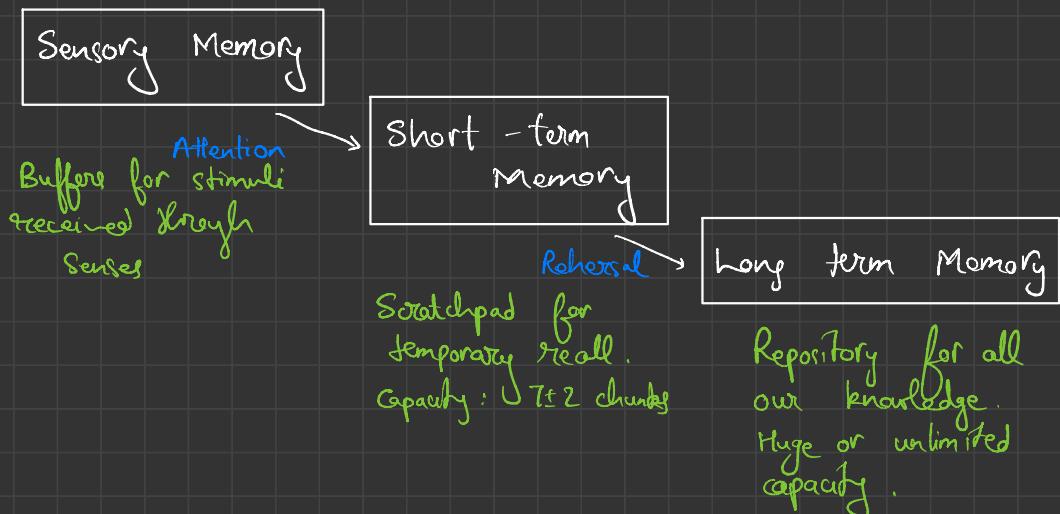
The ::= symbol is read as 'is defined as'. Only non terminals may appear on the left.

The terminals are appeared on the right and are separated with sequence '+' or choice '|'

Terminals - lowest level of human behaviour

Non-terminals - higher level of abstraction.

Memory System



Layout

Grouping of items Order of items } logically together \Rightarrow physically together

Decoration - fonts, boxes etc boxes to group logical items, fonts for emphasis of headings

Alignment of items Text - left to right, Numbers - right to left

White space b/w items use whitespace to separate blocks, texts etc.

Nextion , principles to support usability , GUI vs WEB

DATA ANALYTICS

Data Analytics

Analytics is defined as scientific process for transforming the data into formating better decisions.

4 scale of measurement

Nominal scale - used for naming variable in no particular order

Ordinal scale - used for variables in ranked order , but the difference b/w is not determined

Interval scale - used for numerical variables with known equal intervals of the same distance

Ratio scale - used for variables on a scale that have measurable intervals .

4 data types

List \Rightarrow [" " , " "] , ordered , changeable , duplicate values

Tuple \Rightarrow (" " , " ") , ordered , unchangeable

Set \Rightarrow { " " , " " } , ordered , unchangeable , unique

Dictionary \Rightarrow thisDict = {
 " brand " : " ford " ,
 " model " : " mustang " ,
 " year " : 1964
}

ordered , changeable ,
no duplicate value

Data lakes huge data in native format

Data marts summarized specific data

NoSQL vs RDBMS

1. NoSQL are cheaper to scale rapidly as compared to RDBMS
2. NoSQL systems help to handle large volume data while data volume managed by RDBMS is less.
3. NoSQL has fewer restrictions allowing change management while RDBMS does not allow easy changes
4. NoSQL is cost effective while RDBMS is not
5. NoSQL databases require less management as compared to RDBMS which requires highly trained professionals.