

# **ECE 4470:**

# **Final Project**

Time-Domain and Frequency Domain Controller Design

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Wednesday, April 17, 2024



## Table of Contents:

Introduction:

Analytic and MATLAB Results:

Controller 1: PD

Controller 2: DI

Controller 3: PID

Controller 4: Phase-Lead

Controller 5: Phase-Delay

Comparing/Contrasting the controllers, discussing Pros/Cons, Conclusion:

## Introduction:

Controllers allow us to manage the behavior of complex systems, like industrial manufacturing processes and automotive electronics. The choice and ‘tuning’ of each controller significantly influences a system's performance and parameters such as its response time, overshoot, and stability.

5 controllers were designed with specified parameters: (1)Proportional-Derivative (PD), (2)Proportional-Integral (PI), (3)Proportional-Integral-Derivative (PID), (4)Phase-lead, and (5)Phase-lag. We considered the same system for each type of controller, represented by the open-loop transfer function  $G(s)H(s)$ :

Expanding the given  $G(s)H(s)$ :

Expanded form

$$\frac{10}{s^3 + 27s^2 + 162s}$$

This project employs MATLAB and Simulink simulations to perform calculations and visualize the system's and controller's behavior and impact on system performance. Analytical/handwritten results are provided at the end of the report.

### 1. MATLAB/Simulink Results:

Controller 1: PD

Code:

```

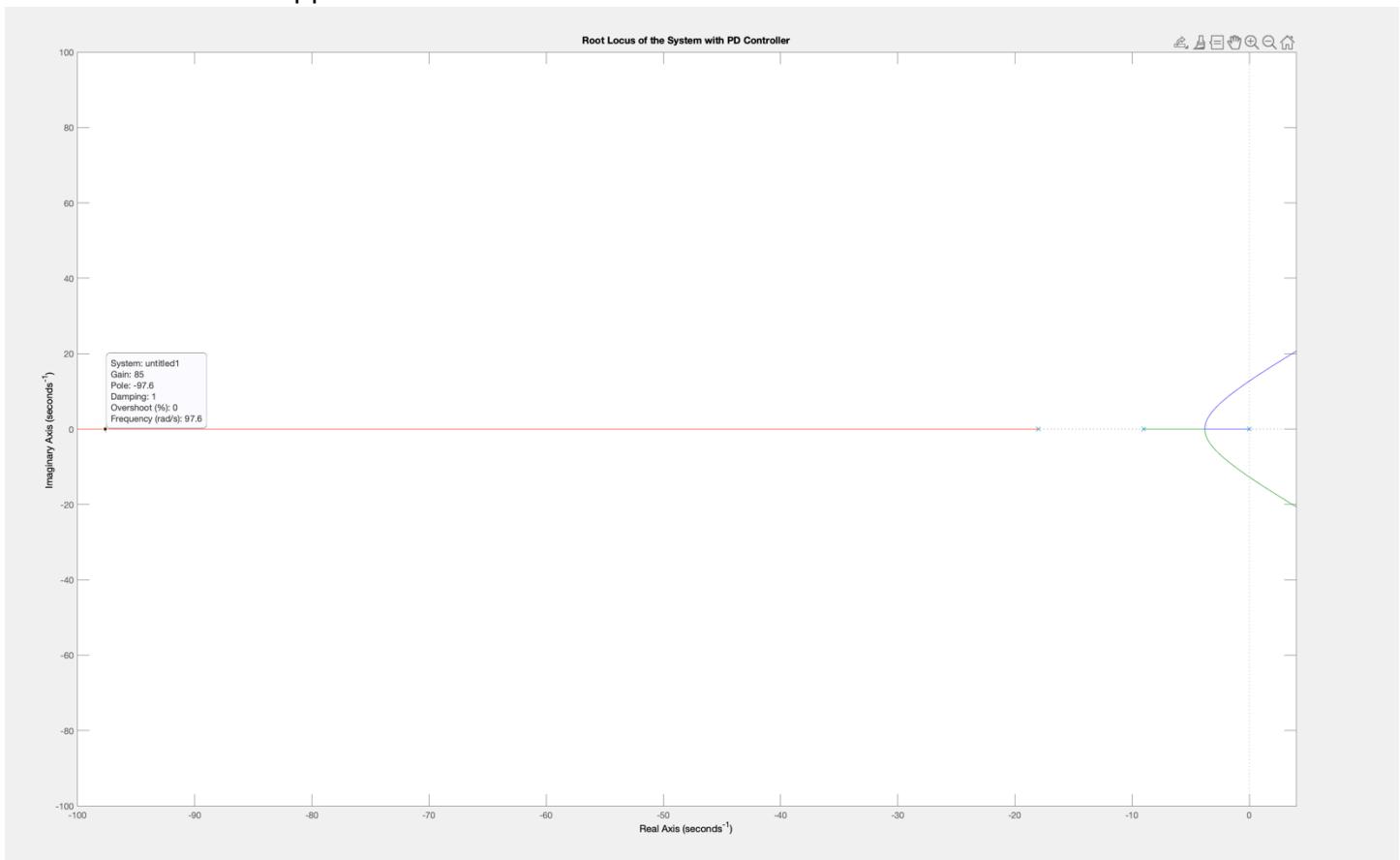
43 %% ECE 4470 - Final Project %%
44 %% David Baron-Vega - GF7068
45 %% Started: Wednesday, April 3. Due: Monday, April 22
46
47 %{
48 "The goal of this project is to design controllers using the design techniques discussed; to simulate
49 closed loop systems; and to compare different controllers.
50 The system to be controlled is modeled by the following transfer function:
51
52 G(s)H(s) = 10/(s)(s+9)(s+18)"
53 %}
54


---

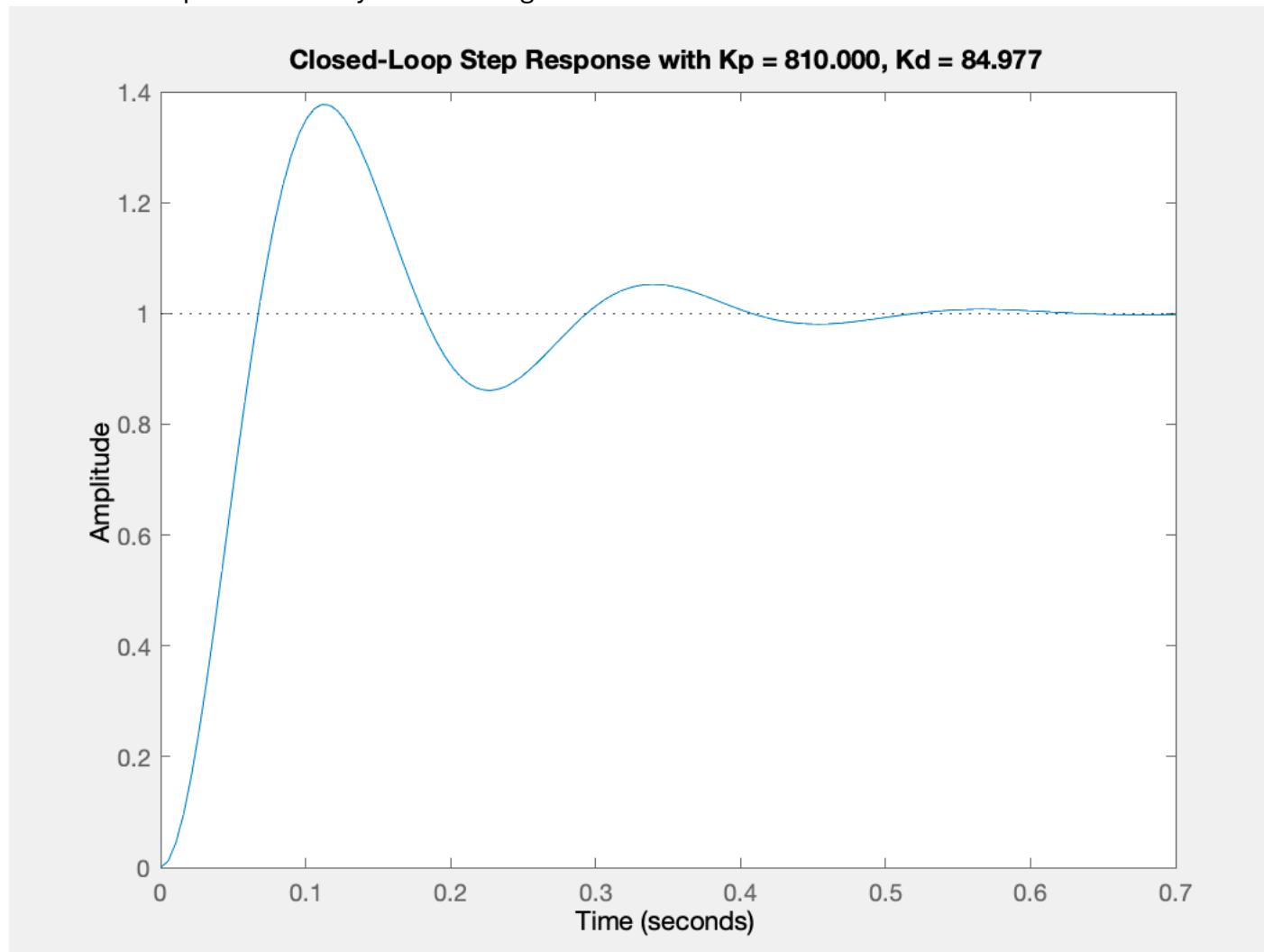

55 %% Part 1 %%
56 %1. Design a PD controller using time-domain design method such that (1) Kv = 50 and (2) overshoot is smallest.
57
58 %Defining the system we are working with. We have been provided with the
59 %system's Open-Loop transfer function, and can use built in MATLAB
60 %functionality 'tf' to work with transfer functions.
61
62 %We need to define the Laplace variable as a continuous function to work with it.
63 s = tf('s');
64
65 %Open-loop transfer function coefficients
66 num = 10;
67 den = [1 27 162 0]; % s(s+9)(s+18) expands to s^3 + 27s^2 + 162s
68
69 %Displaying the open-loop transfer function
70 fprintf('The open-loop function of the system is:\n');
71 G = tf(num, den);
72
73 %Calculating the velocity constant Kv as s approaches 0
74 Kv = dcgain(s*G);
75
76 % Displaying the required open-loop gain to achieve a ramp-error constant Kv of 50
77 fprintf('The open-loop gain necessary to achieve a (Kv) ramp-error of 50 in our system is:\n');
78 Kp = 50 / Kv;
79 disp(Kp);
80
81 %Keeping Kp constant at the value calculated to maintain Kv = 50
82 %Kd will be determined using the rlocfind function on the root locus plot
83 %Defining Td as a placeholder for the derivative time constant
84 Td = s;
85
86 %Plotting the root locus for the system with PD control (using Kp)
87 %The PD controller's derivative term will be represented by Kd*s in the root locus
88 figure;
89 rlocus(Kp*G);
90 title('Root Locus of the System with PD Controller');
91 xlim([-100 4]);
92 ylim([-100 100]);
93
94 %Using rlocfind to pick a point on the root locus to minimize overshoot
95 %This will pause execution and wait for the user to select a point on the plot
96 [Kd, poles] = rlocfind(Kp*G);
97
98 %Displaying the selected Kd and poles of the closed-loop system
99 disp('The selected derivative gain Kd is:');
100 disp(Kd);
101 disp('The closed-loop poles at the selected Kd are:');
102 disp(poles);
103
104 %Forming the PD controller with the fixed Kp and the selected Kd
105 %The derivative term is now properly represented by Kd*s
106 G_PD = Kp + Kd*s;
107
108 %Creating the closed-loop transfer function T with the PD controller and the plant G
109 T = feedback(G_PD*G, 1)
110
111 %Simulating and plot the step response of the closed-loop system with the PD controller
112 figure;
113 step(T);
114 title(sprintf('Closed-Loop Step Response with Kp = %.3f, Kd = %.3f', Kp, Kd));
115

```

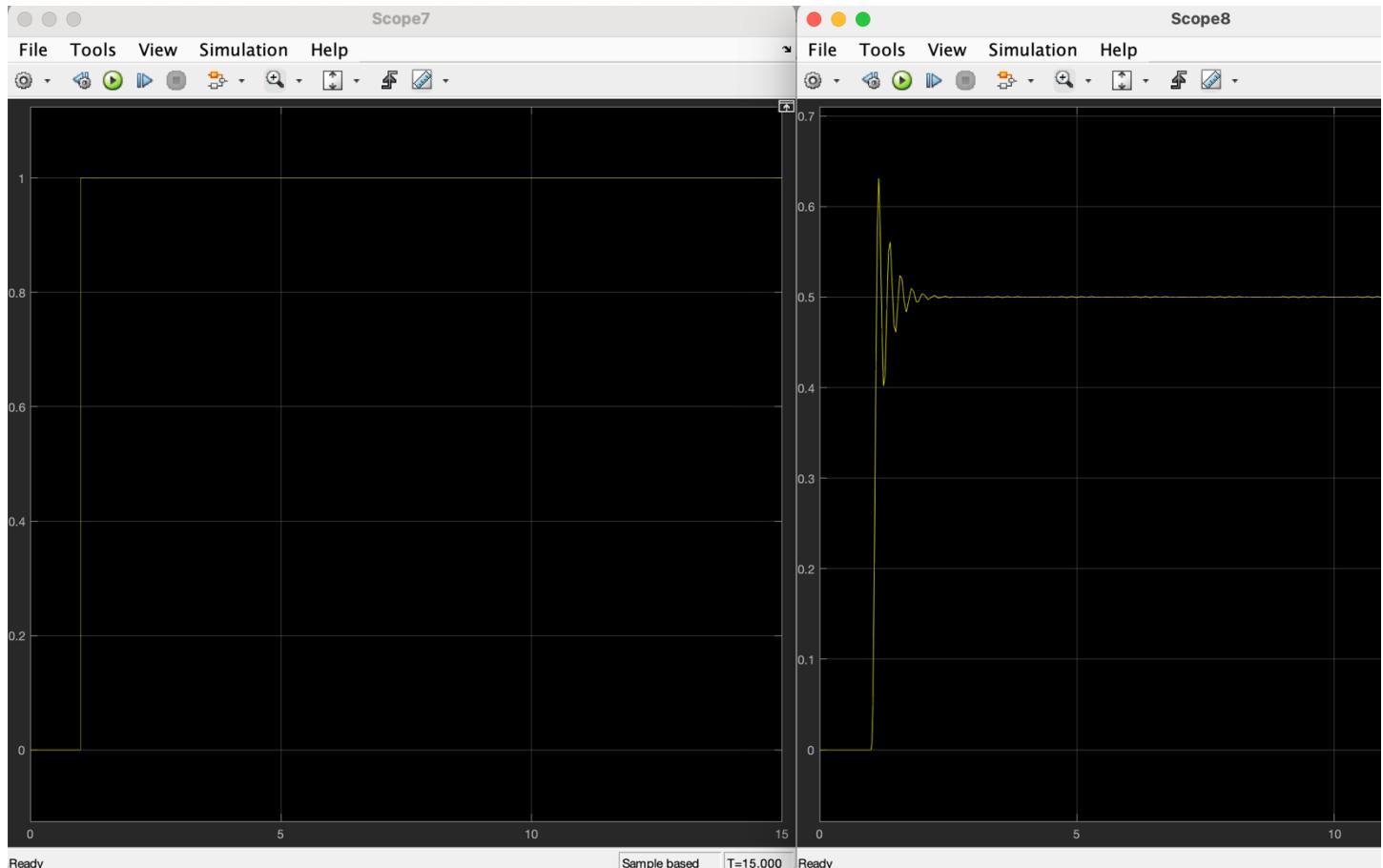
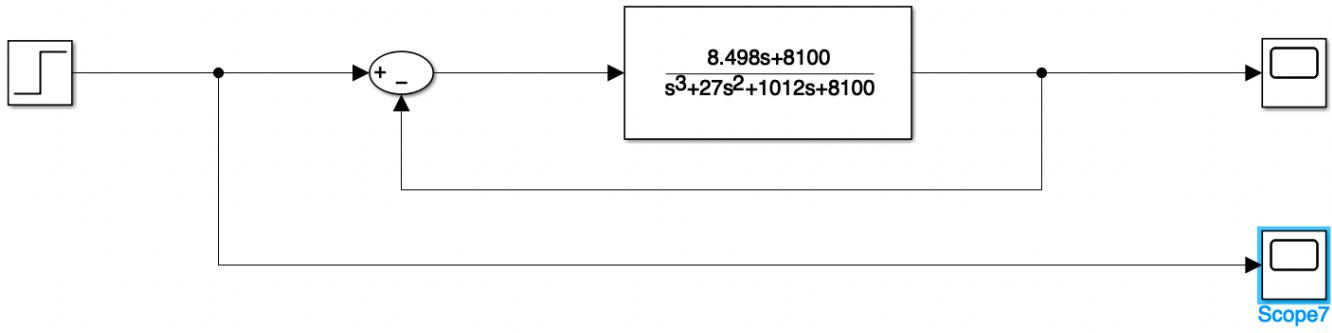
Root Locus Plot and approximate root chosen:



Overshoot response of the system at this given root:



Modeling the closed-loop system in Simulink:

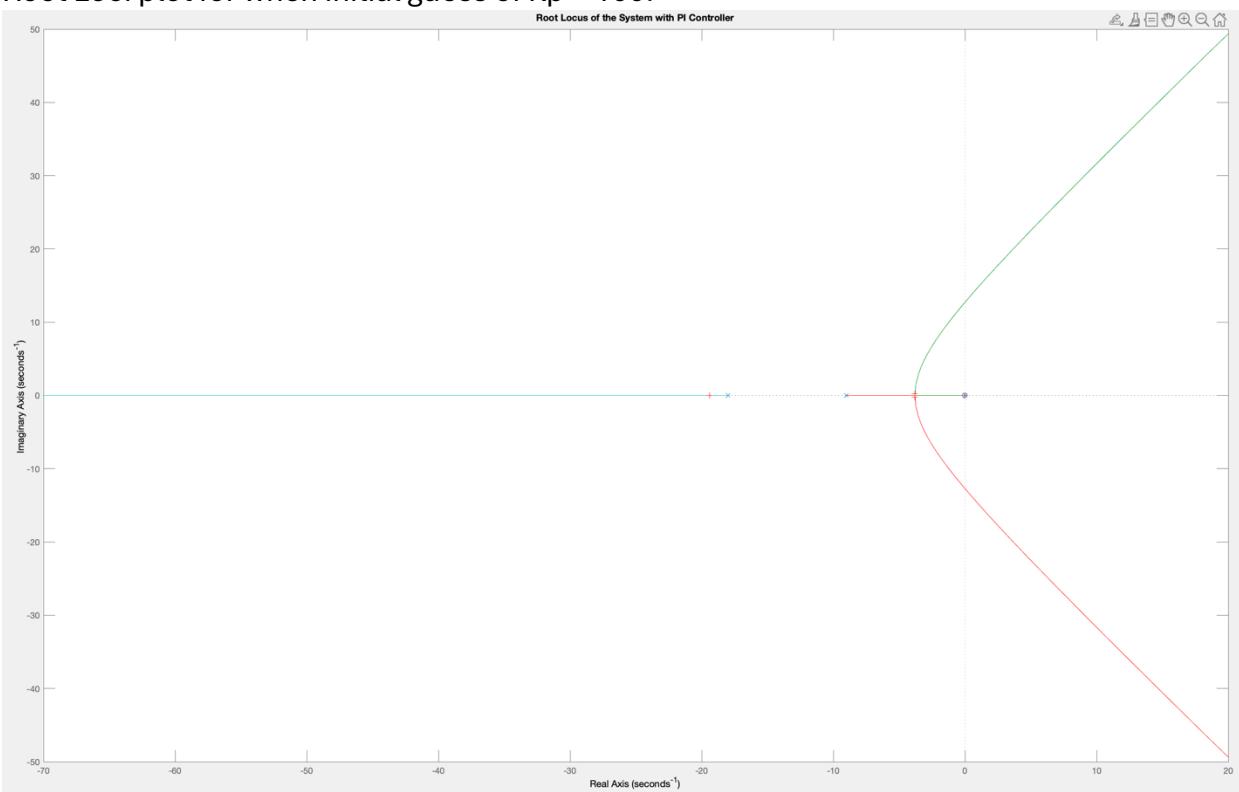


Controller 2: DI

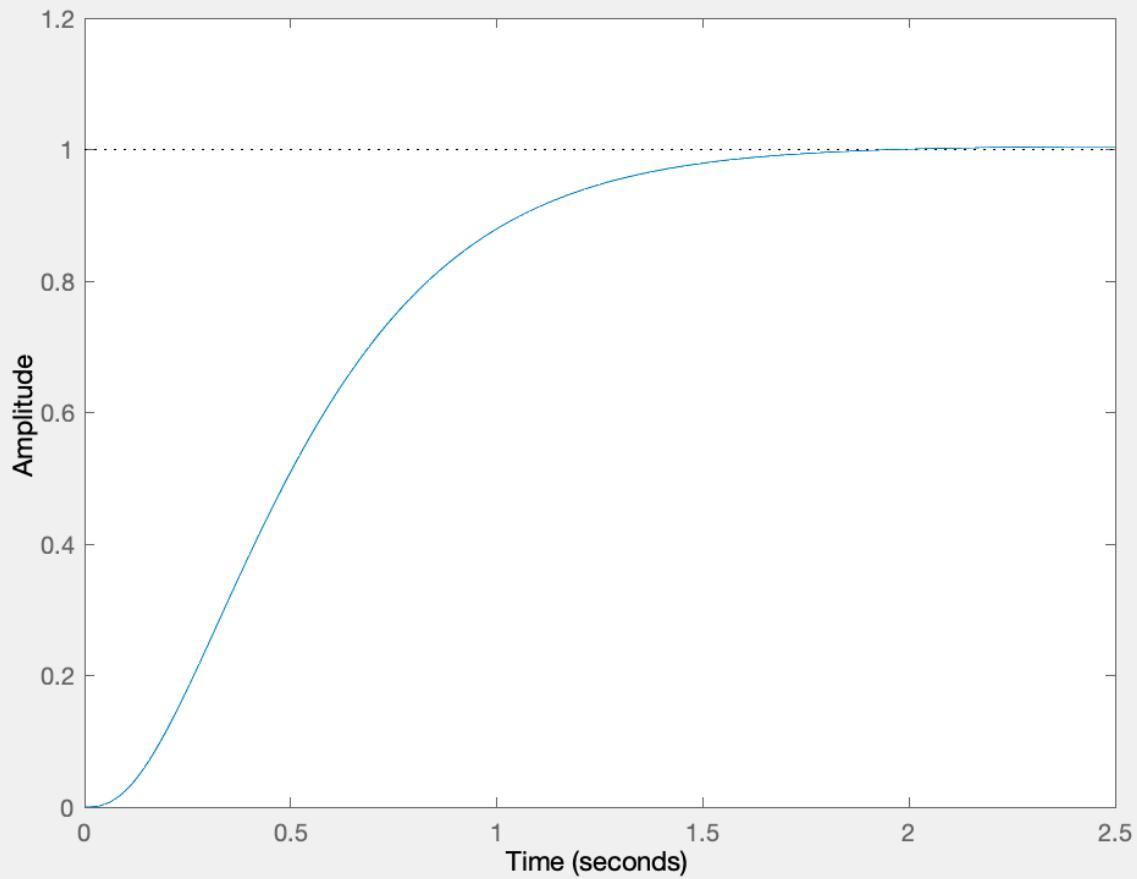
Code:

```
1 %Defining the given system
2 s = tf('s');
3 G = 10 / (s*(s+9)*(s+18));
4
5 %PI Controller parameters (initial guesses, tweaked below)
6 Kp = 100; %Starting with a small value for Proportional gain
7 Ki = 1;   %Starting with a small value for Integral gain
8
9 %Defining the PI Controller Open Loop transfer function
10 PI = Kp + Ki/s;
11 OL_TF = PI * G;
12
13 %Generating the root locus plot of the open-loop transfer function
14 figure;
15 rlocus(OL_TF)
16 title('Root Locus of the System with PI Controller');
17
18 %Using rlocfind to test and select points on the root locus plot
19
20 [K, poles] = rlocfind(OL_TF);
21
22 %Updating Kp and Ki based on the selected gain K
23 %The gain K returned from rlocfind is used to adjust the PI controller gains
24 Kp = Kp * K
25 Ki = Ki * K
26
27 %Redefining the PI controller with the updated gains
28 PI = Kp + Ki/s
29
30 %Creating the closed-loop transfer function with the updated PI controller
31 T = feedback(PI * G, 1)
32
33 %Simulating and plot the step response of the closed-loop system
34 figure;
35 step(T);
36 title(sprintf('Closed-Loop Step Response with Kp = %.3f, Ki = %.3f', Kp, Ki));
37
38 %Fetching step response characteristics
39 info = stepinfo(T);
40
41 %Displaying the overshoot and settling time
42 fprintf('Overshoot: %.2f%\n', info.Overshoot);
43 fprintf('SettlingTime: %.2f seconds\n', info.SettlingTime);
```

# Root Loci plot for when initial guess of $K_p = 100$ :



**Closed-Loop Step Response with  $K_p = 28.219$ ,  $K_i = 0.282$**



Results:

```
>> ECE4470_Controller2Testing  
Select a point in the graphics window
```

```
selected_point =
```

```
-3.8072 + 0.2349i
```

```
Kp =
```

```
28.2190
```

```
Ki =
```

```
0.2822
```

```
T =
```

$$\frac{282.2 s + 2.822}{s^4 + 27 s^3 + 162 s^2 + 282.2 s + 2.822}$$

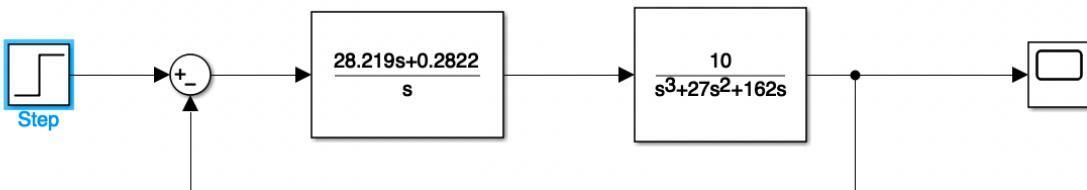
Continuous-time transfer function.

Overshoot: 0.45%

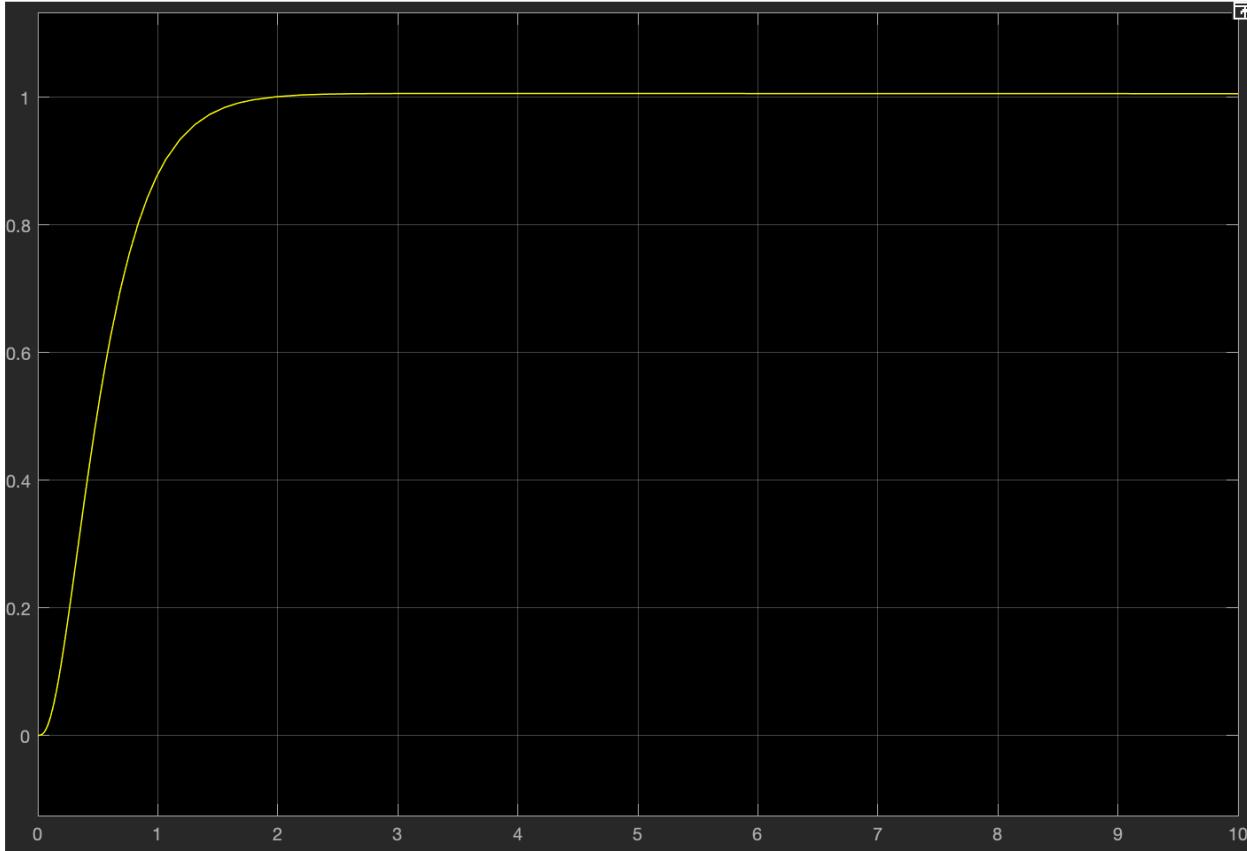
SettlingTime: 1.51 seconds

Implementing in Simulink:

Block Diagram showing  $(K_p + K_i/s) * G(s)H(s)$ :



Output Oscilloscope results, matches the MATLAB step response simulation.



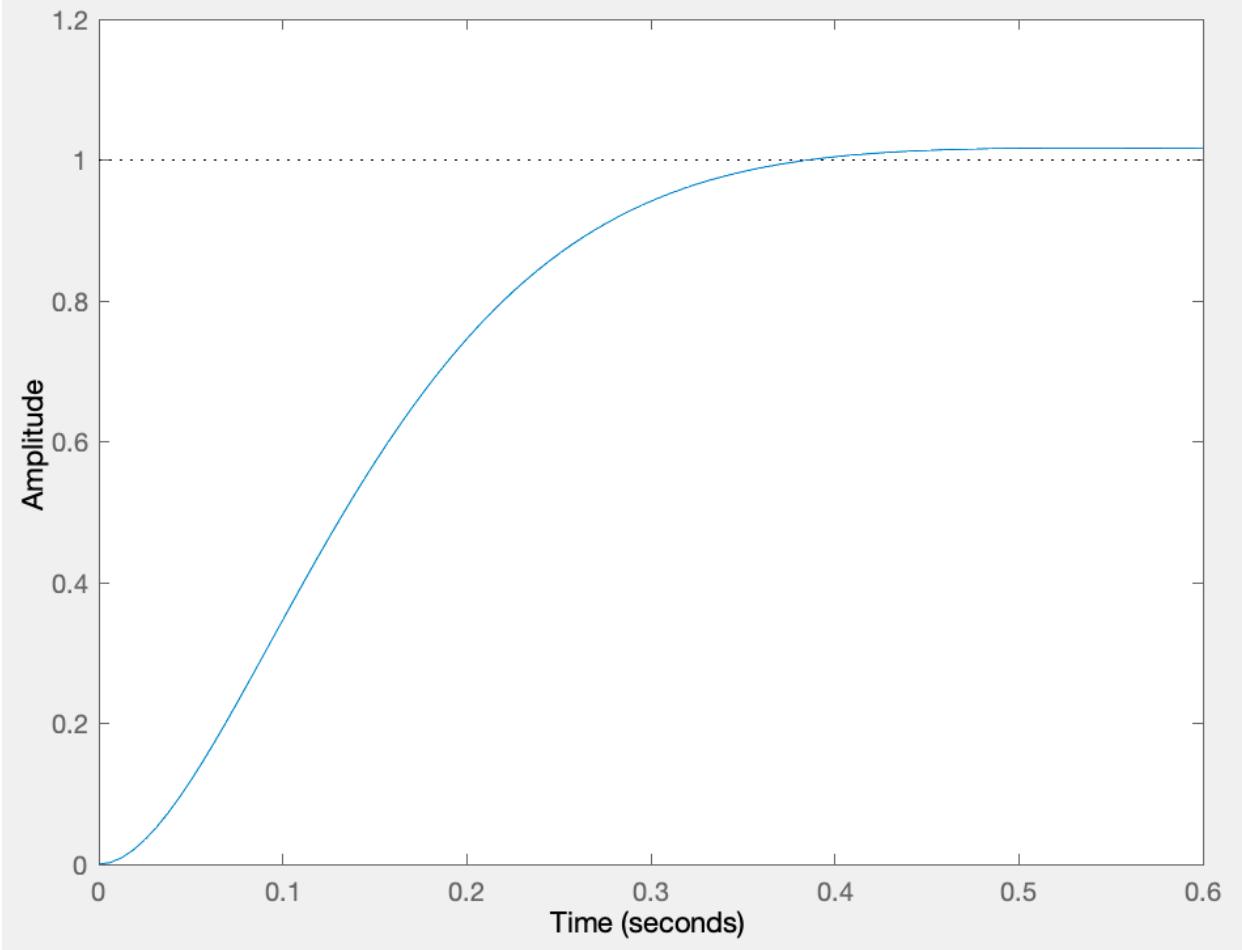
### Controller 3: PID

```

1 %Defining the system:
2 s = tf('s');
3 G = 10 / (s*(s+9)*(s+18));
4
5 %Tuning the PID Controller Gains;
6 Kp = 110;
7 Ki = 12;
8 Kd = 13;
9
10 %PID Controller Transfer Function
11 PID = Kp + Ki/s + Kd*s
12
13 %Closed-loop Transfer Function with PID controller
14 T_PID = feedback(PID * G, 1)
15
16 %Simulating, plotting the step response of the closed-loop system
17 figure;
18 step(T_PID);
19 title(sprintf('Closed-Loop Step Response with Kp = %.3f, Ki = %.3f, Kd = %.3f', Kp, Ki, Kd));
20
21 %Step Response Simulation without plotting:
22 [response, t] = step(T_PID);
23
24 %step response characteristics
25 info = stepinfo(T_PID);
26
27 %Displaying the performance metrics:
28 fprintf('RiseTime: %.2f seconds\n', info.RiseTime);
29 fprintf('SettlingTime: %.2f seconds\n', info.SettlingTime);
30 fprintf('Overshoot: %.2f%\n', info.Overshoot);
31
32

```

### Closed-Loop Step Response with $K_p = 110.000$ , $K_i = 12.000$ , $K_d = 13.000$



T\_PID =

$$\frac{130 s^2 + 1100 s + 120}{s^4 + 27 s^3 + 292 s^2 + 1100 s + 120}$$

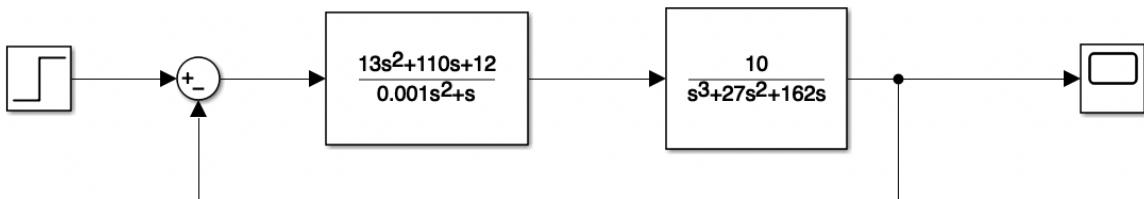
Continuous-time transfer function.

RiseTime: 0.22 seconds

SettlingTime: 0.34 seconds

Overshoot: 1.70%

Simulink Modeling and Simulation Results:



## Output Oscilloscope Results:



Controller 4: Phase-Lead

Code:

```

1 %Defining the System:
2 s = tf('s'); % We need to define the Lapace variable as a continuous function to work with it.
3 num = 10;
4 den = [1 27 162 0]; % s(s+9)(s+18) = s^3 + 27s^2 + 162sa
5 G = tf(num, den)
6
7 %Bode plot to find initial Kv and phase margin of the system without
8 %phase-lead.
9
10 figure;
11 margin(G);
12 [Gm, Pm, Wcg, Wcp] = margin(G);
13
14 %Design parameters for the phase-lead compensator
15 alpha = 5.828; %Solving for a where phi_m is 45, we have a value of 5.828
16 T = .3573; %solved for analytically by deriving desried crossover frequency.
17
18 % Phase-lead transfer function:
19 Gc = (s*T + 1) / (s*alpha*T + 1);
20 K = 161.99; %K needed fpor Kv = 10: (Found analytically) %161.999
21
22
23
24 %Open-loop transfer function with compensator
25 OL_TF = K * Gc * G
26
27 % Bode plot of system with compensator
28 figure;
29 margin(OL_TF);
30 [Gm, Pm, Wcg, Wcp] = margin(OL_TF);
31
32
33 %Tweaking the parameters, these are the best I could make!
34 alpha = 7;
35 T = .858;
36
37 % Phase-lead transfer function:
38 Gc = (s*T + 1) / (s*alpha*T + 1);
39 K = 161.99; %K needed fpor Kv = 10: (Found analytically) %161.999
40
41 %Open-loop transfer function with compensator
42 OL_TF = K * Gc * G
43
44
45 % Bode plot of system with compensator
46 figure;
47 margin(OL_TF);
48 [Gm, Pm, Wcg, Wcp] = margin(OL_TF);
49

```

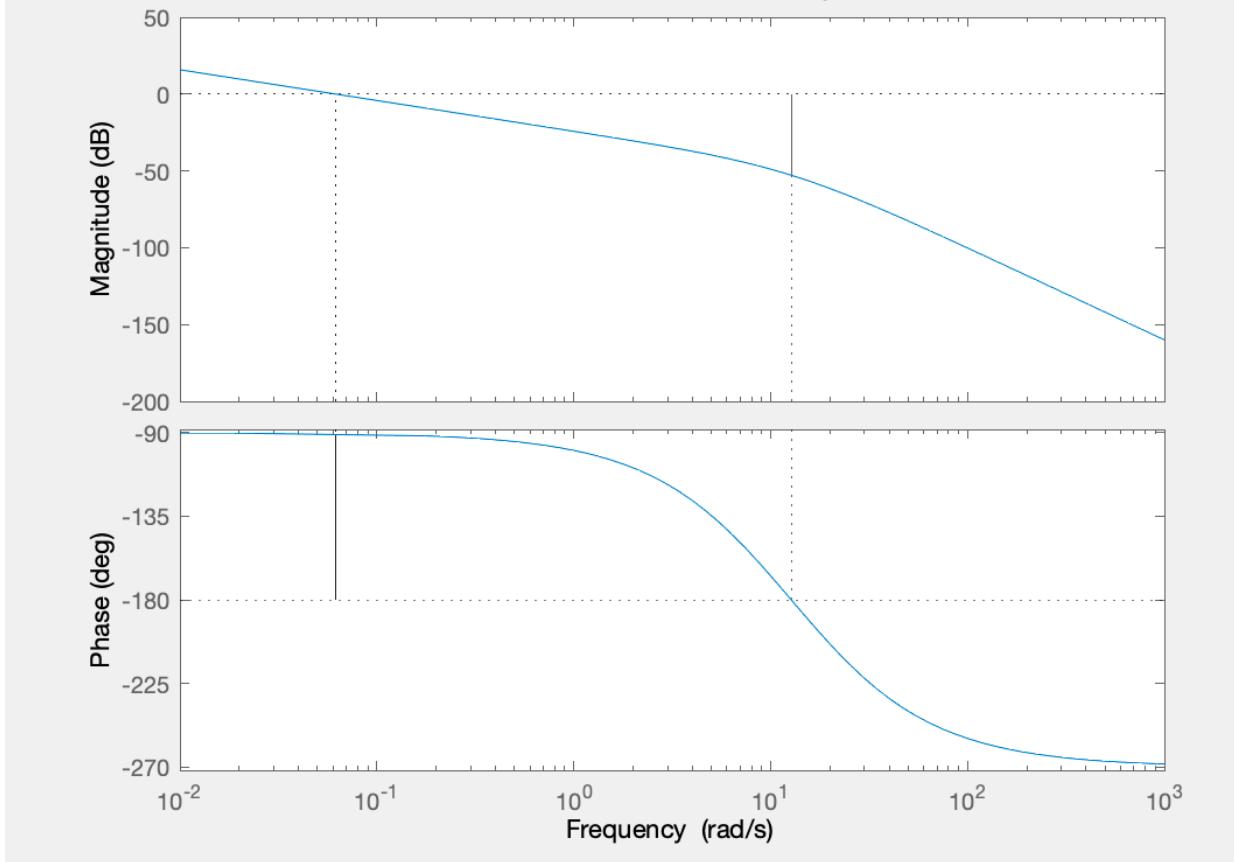
Kv = 10

Phase-Margin = 45 degrees.

First step is to calculate the correct K gain of compensator needed to achieve Kv = 10.

We then plot the initial Bode plot of the system with K, excluding the controller:

**Bode Diagram**  
**G<sub>m</sub> = 52.8 dB (at 12.7 rad/s), P<sub>m</sub> = 89.4 deg (at 0.0617 rad/s)**



We see that our initial phase margin is  $180 - 89.4 = 90.6$  degrees.

We can analytically solve for  $a$  using the following formula:

$$\phi_m = \sin^{-1} \left( \frac{a-1}{a+1} \right)$$

$a$  was analytically found to be 5.828.

Using this value of A, we can specify a value of T that will further calibrate the controller by deriving a new crossover frequency using

$$20\log|G(j\omega_m)| = -10\log a = -3.91$$

From the Bode plot, we find  $\omega_m = 60$ .

Hence,

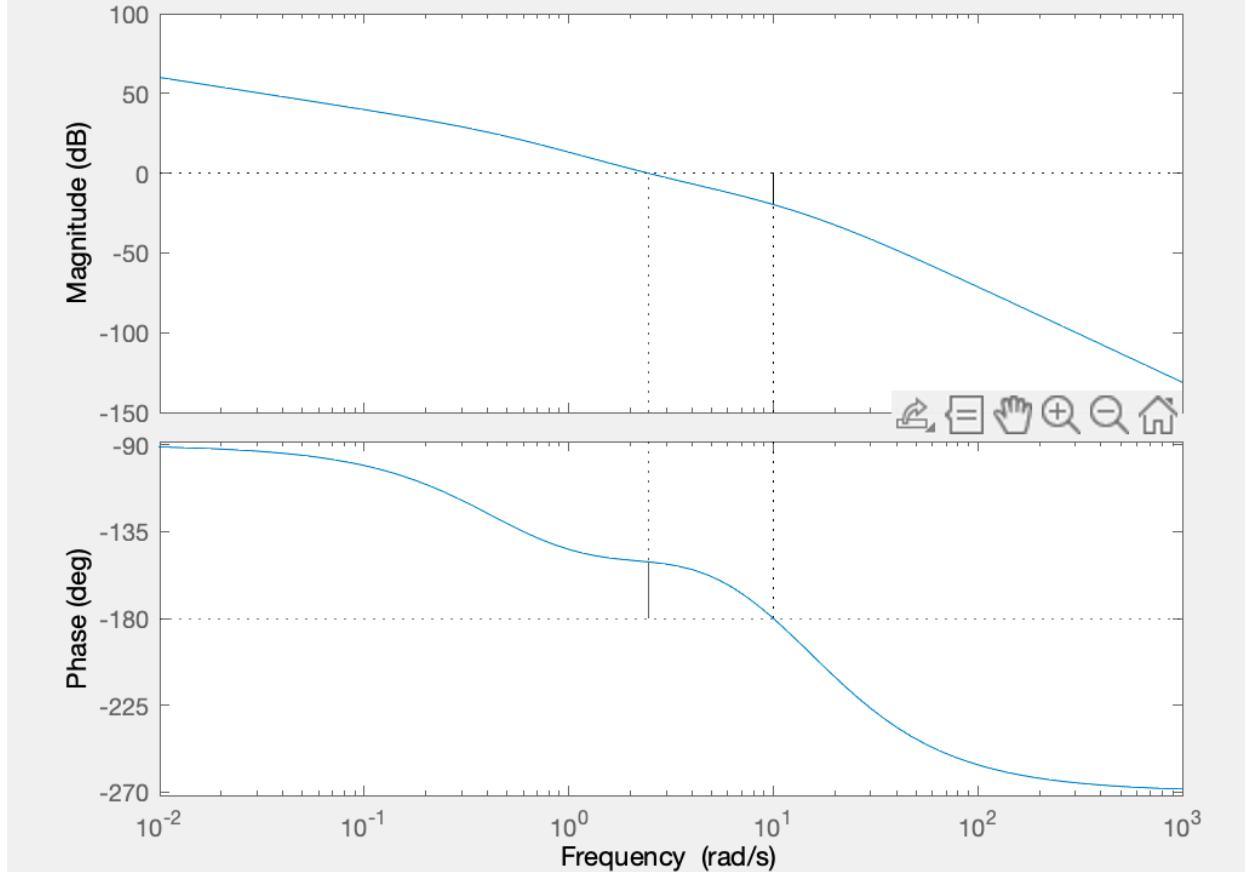
$$\omega_m = \frac{1}{\sqrt{aT}}$$

$$\Rightarrow T = \frac{1}{\sqrt{a}\omega_m} = 0.0106$$

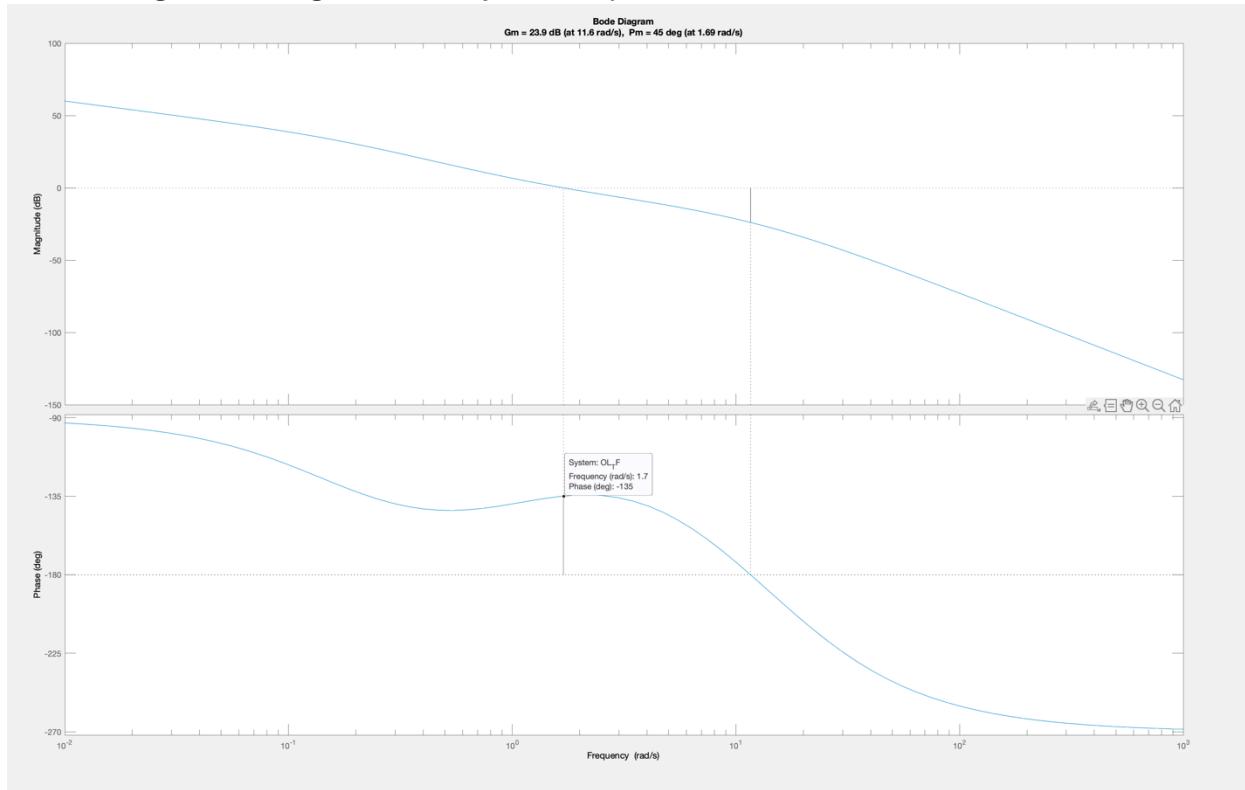
The new T value I used was: .3573.

New Bode Plot Results:

**Bode Diagram**  
**Gm = 19.7 dB (at 10 rad/s), Pm = 29.3 deg (at 2.45 rad/s)**



The Phase margin is too low now, so I will tweak the values of  $\alpha$  and  $T$  until I approximate 45 degrees. Phase margin of 45 degrees, exactly. Final  $\alpha$ ,  $T$  values:



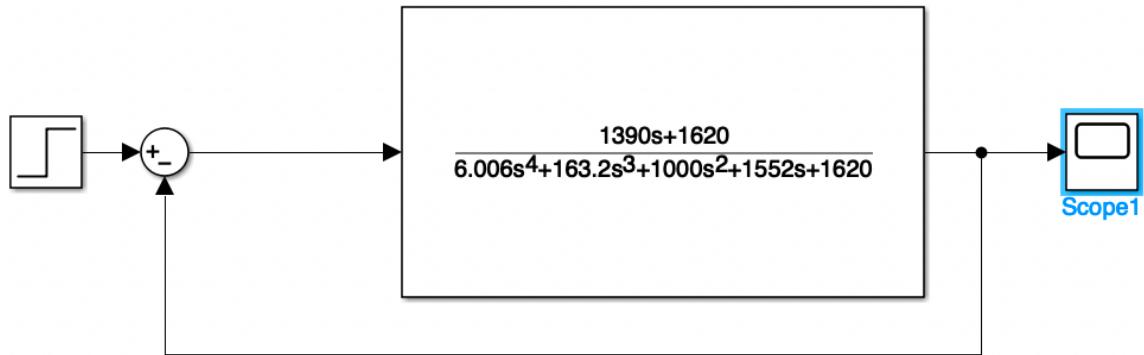
alpha = 7;  
 $T = .858;$

OL\_TF =

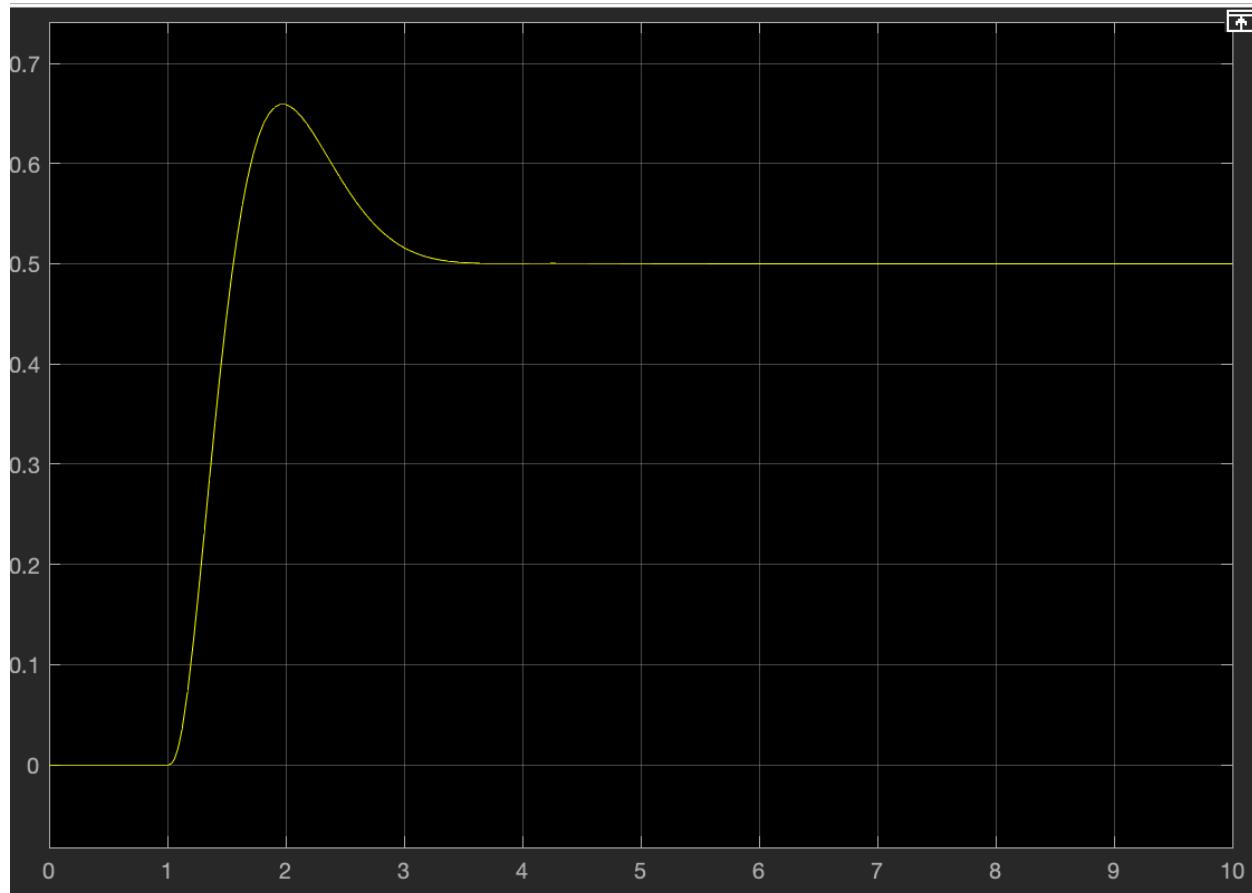
$$\frac{1390 \text{ s} + 1620}{6.006 \text{ s}^4 + 163.2 \text{ s}^3 + 1000 \text{ s}^2 + 162 \text{ s}}$$

Simulink Model and Simulation:

Block diagram of the overall system:



Step response of the system, we can see it is stable.



## Controller 5: Phase-Delay

Code:

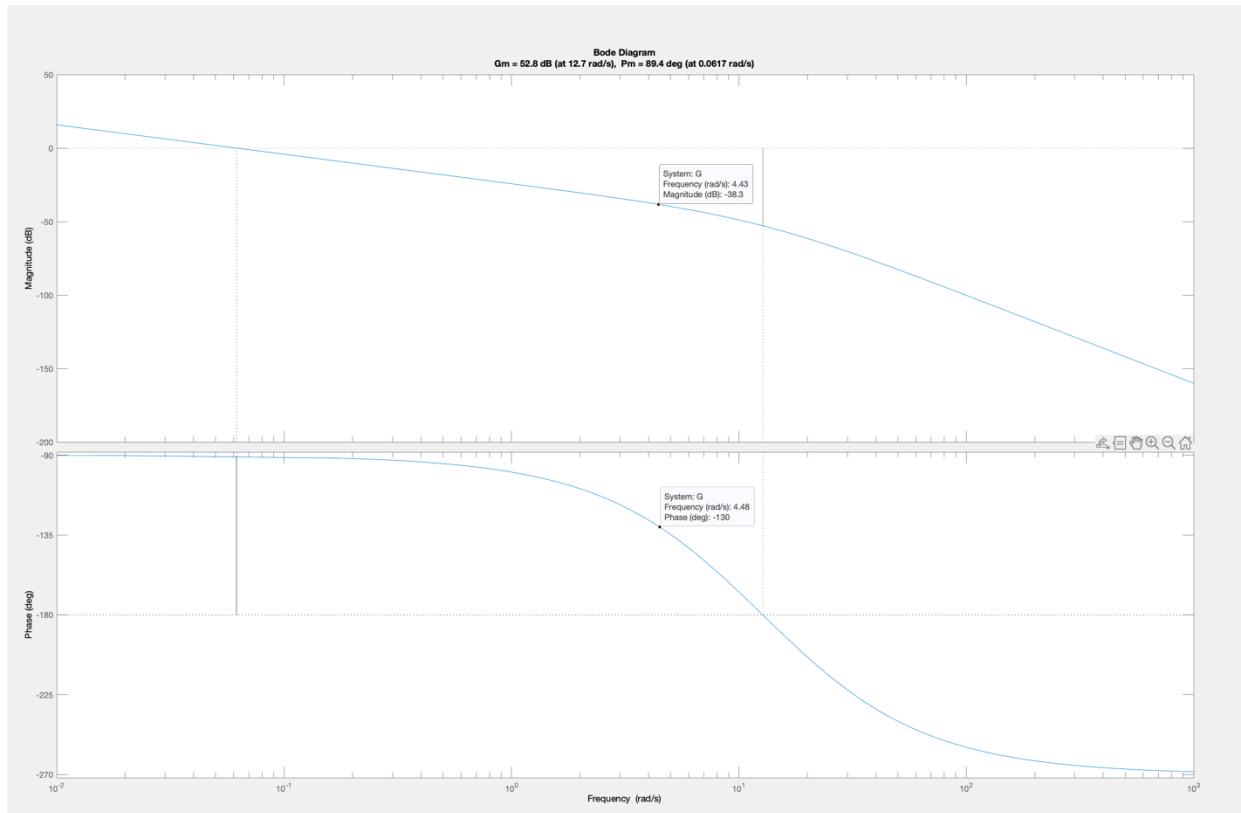
```
ECE4470_Controller5Testing.m +  
1 %%Step 5: Design a Phase-lag controller using frequency-domain design  
2 %%method such that Kv = 10 and Phase margin is 45 degrees.  
3  
4 %Defining the System:  
5 s = tf('s');  
6 num = 10;  
7 den = [1 27 162 0]; % s(s+9)(s+18) = s^3 + 27s^2 + 162sa  
8 G = tf(num, den)  
9  
10 %Bode plot to find initial Kv and phase margin of the system without  
11 %phase-lead, same as before.  
12 figure;  
13 margin(G);  
14 [Gm, Pm, Wcg, Wcp] = margin(G);  
15  
16  
17 %Design parameters for the phase-lead compensator  
18 alpha = 79.433;  
19 T = .0281;  
20  
21 %Phase-lead transfer function:  
22 Gc = (s*T + 1) / (s*alpha*T + 1);  
23 K = 161.99; %K needed for Kv = 10: (Found analytically) %161.999. Same as controller 4.  
24  
25 %Open-loop transfer function with compensator, untuned.  
26 OL_TF = K * Gc * G  
27  
28 %Bode plot of system with phase-lag controlled:  
29 figure;  
30 margin(OL_TF);  
31 [Gm, Pm, Wcg, Wcp] = margin(OL_TF);  
32  
33  
34 %%Definitely needs to be tweaked!  
35  
36 %Design parameters for the phase-lead compensator  
37 alpha = .01259;  
38 T = .042;  
39  
40 %Phase-lead transfer function:  
41 Gc = (s*T + 1) / (s*alpha*T + 1);  
42 K = 161.99; %K needed for Kv = 10: (Found analytically) %161.999. Same as controller 4.  
43  
44 %Open-loop transfer function with compensator, untuned.  
45 OL_TF = K * Gc * G  
46  
47 %Bode plot of system with phase-lag controlled:  
48 figure;  
49 margin(OL_TF);  
50 [Gm, Pm, Wcg, Wcp] = margin(OL_TF);  
51
```

Controller 5 is very similar to controller 4, except we go about changing the phase margin in a different way. Instead of estimating values for  $a$  and  $T$  that will add phase to the output signal, we estimate values for  $a$  and  $T$  that reduce the magnitude, and therefore change the location relative to phase where the crossover frequency occurs.

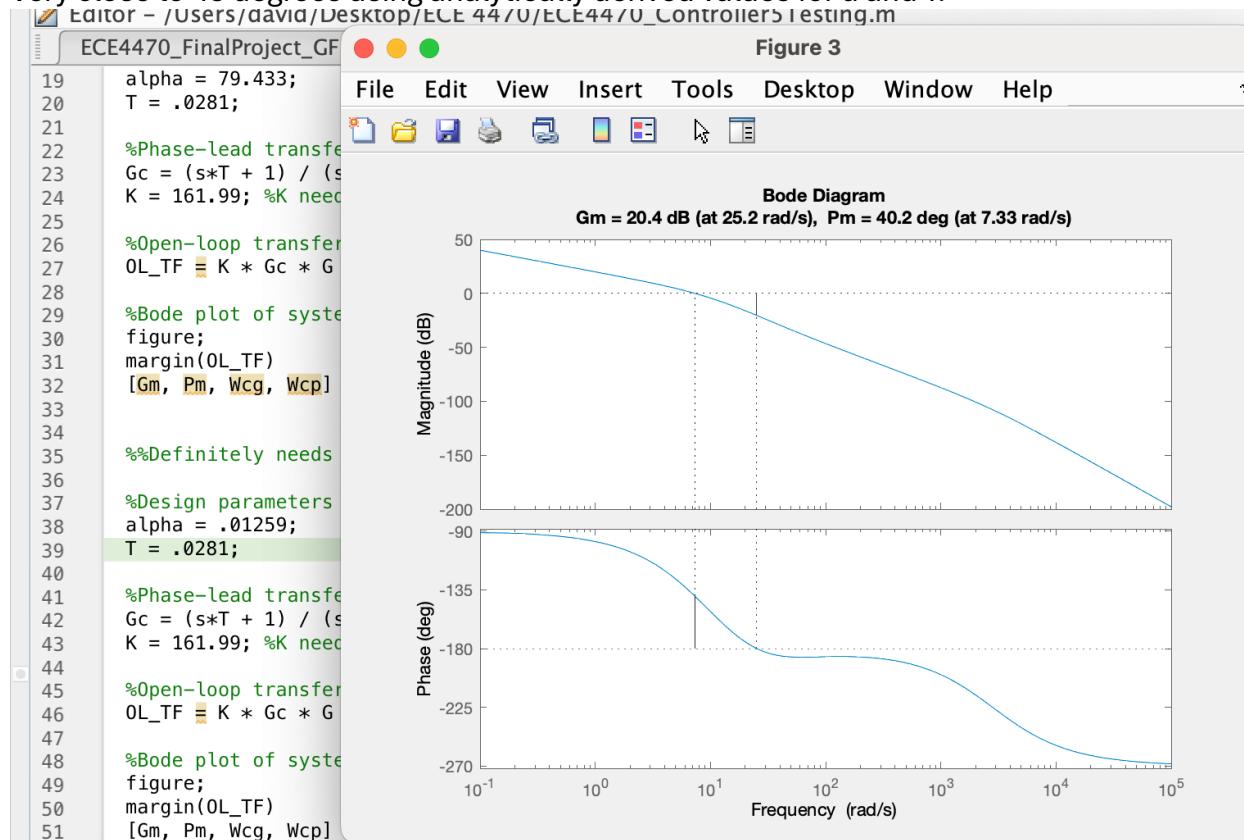
We want to change where  $W_c$  is to where our desired phase margin is. Since we want 45 degrees of margin, that is currently found at  $w = 4.48$  in our system.

The gain at this frequency is approximately 38dB in our system.

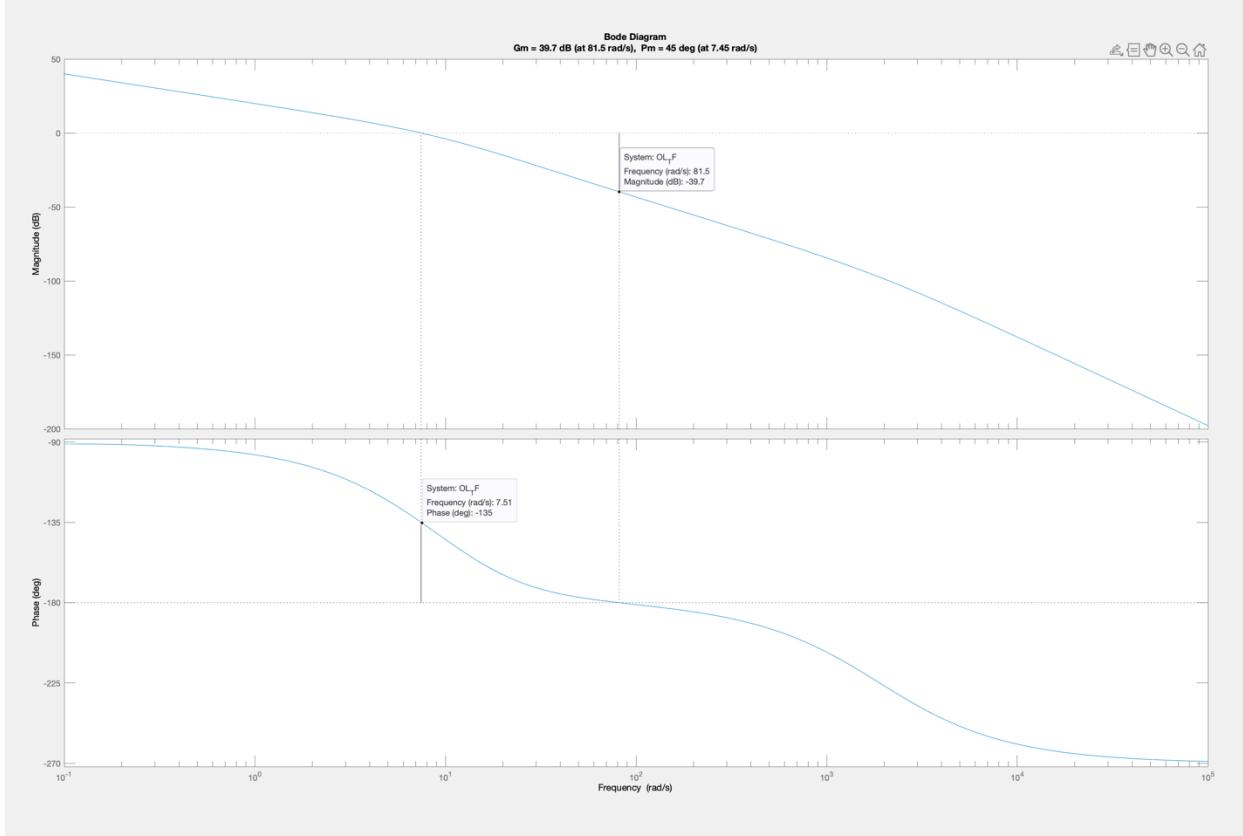
Numerical work:



Very close to 45 degrees using analytically derived values for a and T.



After tweaking the values:



45 degree phase margin achieved.

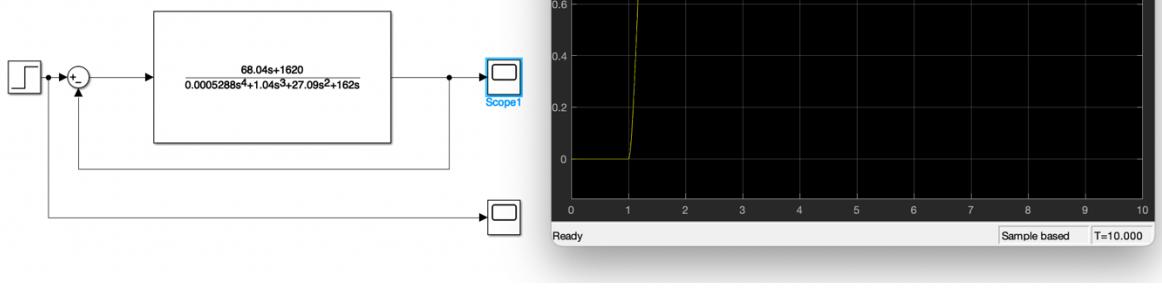
Alpha and T values, resulting closed-loop transfer function of the system:

```
%Design parameters for the phase-lead compensator
alpha = .01259;
T = .042;
```

OL\_TF =

$$\frac{68.04 s + 1620}{0.0005288 s^4 + 1.014 s^3 + 27.09 s^2 + 162 s}$$

Simulink Model and Simulation:



### Comparing/Contrasting the controllers, discussing Pros/Cons, Conclusion:

The comparative analysis of PD, PI, PID, Phase-lead, and Phase-lag controllers on a given transfer function  $G(s)H(s)$  gives insights into the operational strengths and weaknesses of each controller. The PD controller, designed for minimal overshoot, shows good transient response but lacking in steady-state accuracy, making it well-suited for applications where rapid response is critical, and more than long-term precision. In contrast, the PI controller which we focused to eliminate steady-state error, showed superior long-term accuracy at the expense of transient performance. This is good to use in systems where steady-state stability is most important.

The PID controller is a more versatile solution. It balances the fast response with a minimal overshoot, making it an all-round useful design in control applications. This is achieved at the expense of increased complexity, both numerically and analytically. The values of  $K_p$ ,  $K_i$ ,  $K_d$  had to be tweaked, a much more trial-and-error process.

The Phase-lead controller was used to create a phase margin increase, which in this case improved system stability. This is a valuable application in systems requiring consistent phase characteristics. In contrast, the Phase-lag controller optimized the system's gain margin which enhanced the low-frequency response. This is useful in processes that require attenuation of high-frequency noise, IE, applying an LPF within the system.

### Conclusion:

This project explains to us that there is no one-size-fits-all controller. The choice of controller is a chose made to balance and fine-tune specific parameters of the system required.

This was challenging, but fun! I feel that I've learned a lot.

Thank you!

Hand-Written Work:

Controller 1 : Type 0 system, 3rd Order.

$$G(s)H(s) = \left( \frac{10}{(s)(s+4)(s+18)} \right) (K_p + K_D(s))$$

$$C_{ss} = \frac{1}{K_N}$$

$$\text{Input: } R(s) = \frac{1}{s^2}$$

$$C_{ss} = \lim_{s \rightarrow 0} \frac{(s)(\frac{1}{s^2})}{1 + \frac{10(K_p + K_D(s))}{(s)(s+4)(s+18)}} = \lim_{s \rightarrow 0}$$

$$K_N = \lim_{s \rightarrow 0} (s) G(s) H(s) \rightarrow (0) \left( \frac{10}{(s)(s+4)(s+18)} \right) (K_p + K_D(s))$$

$$\lim_{s \rightarrow 0} \frac{(0)(10)(K_p + K_D(s))}{(0+4)(0+18)} = \frac{10 K_p}{162} = K_N$$

$$K_N = 50 = \frac{10}{162} K_p, \boxed{K_p = 810}$$

with  $K_p$  and  $K_D = 0$ , this system becomes unstable.

$$CL - \text{Characteristic Eq: } (1 + G(s)H(s)) = 0$$

$$\rightarrow 1 + \frac{10(K_p + K_D(s))}{(s)(s+4)(s+18)} = 0 \rightarrow (s)(s+4)(s+18) + 10K_p + 10K_Ds = 0$$

$$\rightarrow s^3 + 27s^2 + 162s + 10K_Ds + \underbrace{10K_p}_{8100} = 0$$

$$1 + (10)(K_D) \left[ \frac{(s)}{s^3 + 27s^2 + 162s + 8100} \right] = 0$$

$$= 1 + K \frac{(s)}{(s+30.426)(s-1.713-16.22j)(s-1.713+16.22j)}$$

$\boxed{K = 10K_D}$

$138 = 10K_D$

$K_D = 18.8$  MIGUELRIUS

$$0 \quad s^3 + 27s^2 + 162s + ks + 8100 = 0$$

$$\begin{matrix} s_3 & 1 & 162+k \\ s_2 & 27 & 8100 \end{matrix}$$

$$x_{3,1} = \left(\frac{1}{27}\right) \begin{bmatrix} 27 & 8100 \\ 1 & 162+k \end{bmatrix}$$

$s_1$

$s_1$

$$= \frac{1}{27} (27(162+k) - 8100)$$

$$= 162+k - 300 > 0$$

$$[k \geq 138]$$

$$k = 138, n = 10 k_d,$$

$$k = \frac{l_1 l_2 l_4}{l_3}$$

$$k_d \geq 13.8$$

$$\text{if } k = .005.$$

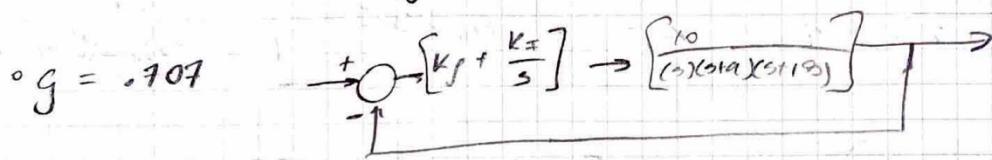
$$\left[ CL \quad T(s) = \frac{0.4708s + 381.3}{s^3 + 27s^2 + 162.5s + 381.3} \right]$$

~~H final~~

$$(10k_p + 10 \frac{v_u}{s}) = 10k_p(1 + \frac{k_u}{k_p} s)$$

Controller 2: PI,  $g = .907$ .

$$= 10k_p(1 + 0.1s)$$



$$\circ g = .907 \rightarrow G(s) H(s) = (10) k_p (s + 0.1)$$

$$\frac{(10)(k_p + \frac{k_i}{s})}{(s)(s+9)(s+18)} = \frac{(10)(k_p(s) + k_i)}{(s)^2 (s+9)(s+18)} \rightarrow$$

$$\rightarrow \left[ \frac{k_i}{k_p} \right] = 0.1 \rightarrow \frac{10k_p(s + 0.1)}{(s^2)(s+9)(s+18)} \approx \frac{10k_p}{(s)(s+9)(s+18)}$$

CL T(f)

Char Eq:

$$1 + \frac{10k_p}{(s)(s+9)(s+18)} \rightarrow 1 + [K] \frac{1}{(s)(s+9)(s+18)} = 0$$

$\downarrow$

$$[K = 10k_p]$$

$\circ K = 0$ , then poles @  $0, -9, -18$

$$\rightarrow s^3 + 27s^2 + 162s + K = 0$$

$$\begin{matrix} s^3 & 1 & 162 \\ s^2 & 27 & K \end{matrix}$$

$$x_{3,1} = \left(\frac{1}{27}\right) \begin{bmatrix} 27 & K \\ 1 & 162 \end{bmatrix} = \frac{1}{27} (27 \cdot 162 - K)$$

$$\begin{matrix} s^1 \\ s^0 \end{matrix}$$

$$\circ \text{stability: } 162 - \frac{K}{27} > 0 \quad \& \quad K > 0$$

$$\checkmark \quad 0 < K < 4374?$$

$$K > \frac{-162}{-27}$$

$$\circ \text{O.S.} \approx e^{-\frac{g\pi}{\sqrt{1-g^2}}} \left[ \begin{array}{l} \text{if } g = .907, \text{ O.S.} \approx .04325 \\ \text{or} \\ \approx 4.3\% \end{array} \right]$$

$$\left\{ K_p = 28.211, K_i = 0.2822 \right]$$

Values used for simulation!

$$(K_p + K_i/s) \rightarrow \frac{28.211(s) + 0.2822}{s}$$

$$(K_p + K_i/s) \left( \frac{10}{(s)(s+9)(s+18)} \right) = \frac{28.211(s) + 0.2822}{s^3}$$

$$(s^3 + 27s + 162)(s) = \dots$$

#3 PID : Fast response, small overshoot, no ess for  $R(s) = 1/s^2$

$$K_p + \frac{K_{i2}}{s} + K_{D2}(s)$$

$$= (1 + K_{D1}(s)) (K_{p2} + \frac{K_{i2}}{s})$$

$$G(s) H(s) = \frac{(10)(1 + K_{D1}(s))(K_{i2} + K_{p2}(s))}{(s)^2 (s+9)(s+18)}$$

[110, 12, 13]

$$P.S.: \text{Let } K_{D2} = \phi, \quad \frac{K_{i2}}{K_{p2}} = 0.1$$

$$\rightarrow G(s) H(s) \approx (10)(1)(0.1 + (s)) = \dots$$

$$\approx \frac{10 K_p}{(s)(s+9)(s+18)}$$

$$O.C.L. \text{ char eq: } 1 + [K] \frac{1}{(s)(s+9)(s+18)}, \quad K = 10 K_{p2}$$

$30, s, s$  is the best fit soln ..

Controller #4.) Phase-Law  $[K > 0, T > 0, \alpha > 1]$

•  $K_V = 10 \quad \therefore \quad \underline{\Phi M = 45^\circ}$ .

$$K \left( \frac{1 + \alpha T_s}{1 + T_s} \right) \rightarrow \left[ \frac{10}{(s)(s+9)(s+18)} \right]$$

$$\begin{aligned} \text{• } K_V &= \lim_{s \rightarrow 0} (s) G(s) H(s) = \lim_{s \rightarrow 0} \phi(K) \left( \frac{1 + \alpha T(s)}{1 + T(s)} \right) \left( \frac{10}{(s)(s+9)(s+18)} \right) \\ &= \frac{(K)(1)(10)}{(1)(9)(18)} = (K) \frac{5}{81}. \end{aligned}$$

• If  $K_V = 10$ ,  $10 = (K) \frac{5}{81}$ ,  $\boxed{K = 161.999}$ .

$45^\circ$  = well damped, good transient response.

$$0dB \quad \underline{\omega \cdot G(\omega) = 10}$$

When drawing bode of  $K = 161.999$ ,

$$\omega_{c_0} = 12.7 \text{ rad/sec}, \quad \underline{\Phi M = 89.4^\circ} \quad \underline{-0.617 \text{ rad/sec}}$$

$$\Phi M = 180^\circ - 89.4^\circ = 90.6^\circ$$

For  $\Phi M = 45^\circ$ , increase:  $45 - 90.6^\circ = -45.6^\circ$  ??

$$\Phi M = \sin^{-1} \left( \frac{a-1}{a+1} \right)$$

phase at  $G_{\omega_c} = -135^\circ$

$$a = \frac{1 + \sin \Phi_m}{1 - \sin \Phi_m}$$

$$180 - 135 = 45^\circ$$

$$a = \frac{1 + \sin(45)}{1 - \sin(45)}$$

• Low  $T$ , increase  $a$ ?

$$\therefore a \approx 5.828!$$

Select Cross Over frequency  $\omega_n$

$$20 \log |G(j\omega_m)| = -10 \log(a)$$
$$= -7.655$$

looking at Bode plot, we see a gain of  $\text{dB} = -7.61$   
pertains to  $\omega = 0.198 \text{ rad/sec}$

Solve now for  $\omega_m^T$ :  $T = \frac{1}{(G_a)(\omega)} = \underline{0.3543}$

Controller #5

PL Controller

$$K_v = 10, \quad \phi_M = 45^\circ$$

→ This yields the same D.L. function as Controller #4.

• K necessary for  $K_v = 10 = 161.999$

→ Achieve  $\Delta\phi_M$  by reducing  $|G(j\omega)|$

$$\begin{aligned} \omega_c' & \quad \angle G(j\omega_c') = -180^\circ + \phi_M + (5^\circ \sim) \\ & = -180^\circ + 45^\circ + (5^\circ \sim 10^\circ) \end{aligned}$$

$$\omega_c' = \frac{4.48}{\text{Eq}} \rightarrow \angle G(j\omega_c') = -130^\circ$$

$$20 \log |G(j\omega_c')| = -20 \log a$$

$$\text{if } \omega_c' = 20, \quad 20 \log |G(j20)| = ? \rightarrow \text{Let's check in Matlab.}$$

$$-38.58 = -20 \log(a)$$

$$a = 10^{(30/20)} = \underline{79.433}. \quad \text{That's big!}$$

$$a = 79.433$$

$$\frac{1}{aT} = \frac{\omega_c'}{10} \rightarrow (T) \frac{\omega_c'}{10} = \frac{1}{a}, \quad T = \frac{1}{a} \frac{10}{\omega_c'}$$

$$T = \frac{10}{(79.433)(9.43)} = \underline{.0281} \quad 1.17?$$