

# Fixed-Axis Rotation

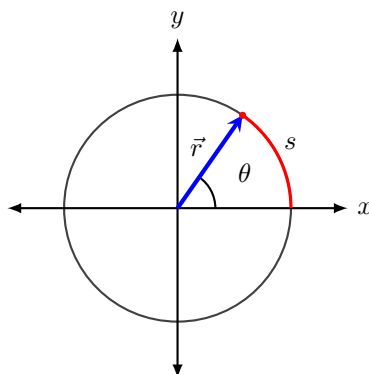
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## 10.1 Rotational Variables

### Angular Velocity

Uniform circular motion is motion in a circle at constant speed, although this is the simplest case of rotational motion, it is used here to introduce rotational variables.

The figure shows a particle moving in a circle. Its position vector from the origin of the circle to the particle sweeps out the angle  $\theta$ , which increases in the counterclockwise direction as the particle moves along its path. The angle  $\theta$  is called the angular position of the particle. As the particle moves, it traces an arc length  $s$ .



The angle is related to the radius of the circle and the arc length by

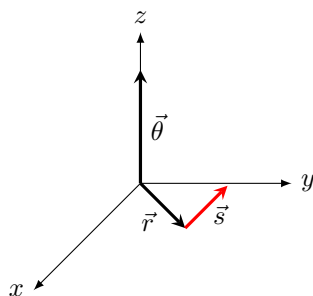
$$\theta = \frac{s}{r} \quad (1)$$

The angle  $\theta$ , the angular position of the particle moving along its path has units of radians (rad). As the particle moves along its circular path, its angular position changes and it undergoes angular displacements  $\Delta\theta$ .

We can assign vectors to the quantities in equation 1, the angle  $\vec{\theta}$  is a vector out of the page. The angular position vector  $\vec{r}$  and the arc length vector  $\vec{s}$  both lie in the plane of the page, they are related by:

$$\vec{s} = \vec{\theta} \times \vec{r} \quad (2)$$

The arc length is the cross product of the angle vector and the position vector



The magnitude of the angular velocity, denoted by  $\omega$ , is the time rate of change of the angle  $\theta$  as the particle moves in a circular path. The instantaneous angular velocity, defined as the limit as  $\Delta t \rightarrow 0$  of the average angular velocity  $\bar{\omega} = \frac{\Delta\theta}{\Delta t}$

$$\omega = \lim_{\Delta t \rightarrow 0} \frac{\Delta\theta}{\Delta t} = \frac{d\theta}{dt} \quad (3)$$

Where  $\theta$  is the angle of rotation. The units of angular velocity are radians per second ( $\text{rad s}^{-1}$ ). Angular velocity can also be referred to as the rotation rate in radians per second. In many cases, rotation rate is given in revolutions/s or cycles/s, to find angular velocity, multiply revolutions/s by  $2\pi$  (since there are  $2\pi$  radians per revolution). Since a positive angle in a circle is counterclockwise, we take counterclockwise rotations as being positive and clockwise rotations as negative.

We can see how angular velocity is related to the tangential speed of the particle by differentiating equation 1 with respect to time. Equation 1 can be rewritten as:

$$s = \theta r$$

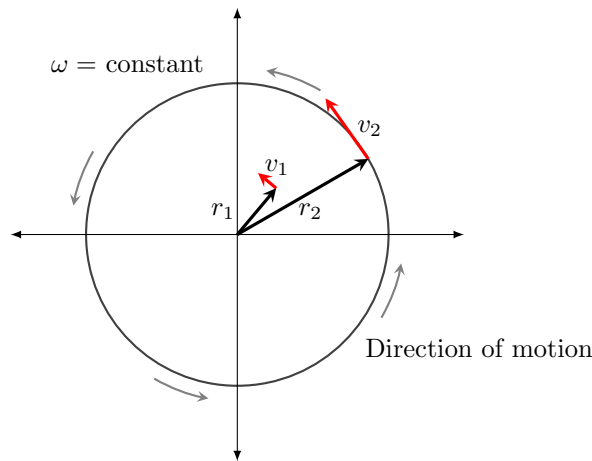
Taking the derivative with respect to time and noting that the radius  $r$  is constant gives:

$$\frac{ds}{dt} = \frac{d}{dt}(r\theta) = r\frac{d\theta}{dt} = r\frac{d\theta}{dt}$$

Where  $\theta\frac{dr}{dt} = 0$ . Here,  $\frac{ds}{dt}$  is just the tangential speed  $v_t$  of the particle moving in a circular path. Using equation 3 we arrive at:

$$v_t = r\omega \quad (4)$$

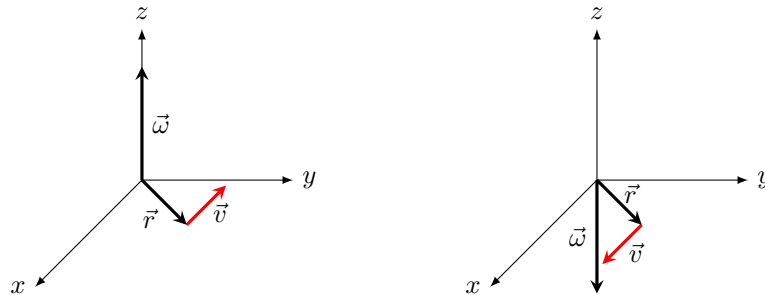
The tangential speed of the particle is its angular velocity times the radius of the circle. The tangential speed of the particle increases with its distance from the axis of rotation for a constant angular velocity. The figure shows two particles placed at different radii on a rotating disk with constant angular velocity. As it rotates, the tangential speed increases linearly with the radius from the axis of rotation. We see that  $v_1 = r_1\omega_1$  and  $v_2 = r_2\omega_2$ . The disk has a constant angular velocity so  $\omega_1 = \omega_2$ . This means that  $\frac{v_1}{r_1} = \frac{v_2}{r_2}$  or  $v_2 = \left(\frac{r_2}{r_1}\right)v_1$ . Thus, since  $r_2 > r_1$ ,  $v_2 > v_1$



Similar to equation 2, one can state a cross product relation to the vector of the tangential velocity as stated in equation 4, therefore:

$$\vec{v} = \vec{\omega} \times \vec{r} \quad (5)$$

The tangential velocity is the cross product of the angular velocity and the position vector as shown below. On the left we see that with the angular velocity in the  $+z$  direction, the rotation in the  $xy$  plane is counterclockwise. On the right, the angular velocity is in the  $-z$  direction, which gives a clockwise rotation in the  $xy$  plane.



### Example 10.1: Rotation of a Flywheel

A flywheel rotates such that it sweeps out an angle at the rate of  $\theta = \omega t = (45.0 \text{ rad s}^{-1})t$  radians. The wheel rotates counterclockwise when viewed in the plane of the page.

- (a) What is the angular velocity  $\omega$  of the flywheel?

$$\omega = \frac{d\theta}{dt} = 45 \text{ rad s}^{-1}, \text{ angular velocity is constant}$$

- (b) What direction is the angular velocity?

The direction of rotation is counterclockwise, so the direction of angular velocity is  $+z$

- (c) How many radians does the flywheel rotate through in 30 s?

$$\theta(t) = \omega t \rightarrow \Delta\theta = \theta(30 \text{ s}) - \theta(0 \text{ s}) = \theta(30 \text{ s}) \rightarrow (45.0 \text{ rad s}^{-1})(30 \text{ s}) = 1350.0 \text{ rad}$$

- (d) What is the tangential speed of a point on the flywheel 10 cm from the axis of rotation

$$v_t = r\omega = (0.1 \text{ m})(45.0 \text{ rad s}^{-1}) = 4.5 \text{ ms}^{-1}$$

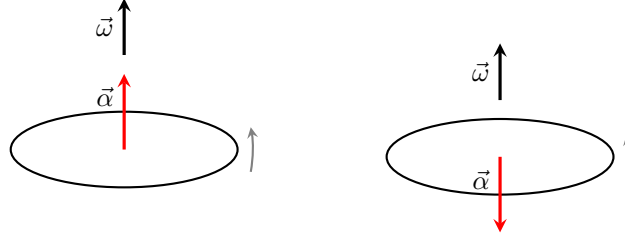
## Angular Acceleration

For describing situations where  $\omega$  changes, we need to define angular acceleration. The faster the change in  $\omega$ , the greater the angular acceleration. Instantaneous angular acceleration  $\alpha$  is defined as the derivative of angular velocity with respect to time:

$$\alpha = \lim_{\Delta t \rightarrow 0} \frac{\Delta\omega}{\Delta t} = \frac{d\omega}{dt} = \frac{d^2\theta}{dt^2} \quad (6)$$

Where we have taken the limit of the average angular acceleration  $\bar{\alpha} = \frac{\Delta\omega}{\Delta t}$  as  $\Delta t \rightarrow 0$ . The units of angular acceleration are radians/s per second, or  $\text{rad s}^{-2}$ .

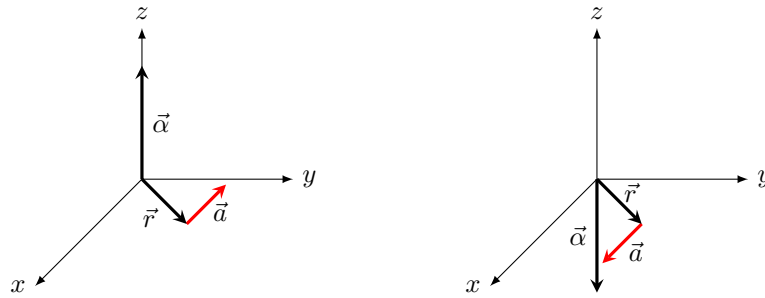
In the same way that the vector associated with angular velocity  $\vec{\omega}$  was defined, we can define  $\vec{\alpha}$ , the vector associated with angular acceleration. If the angular velocity is along the  $+z$  axis and  $\frac{d\omega}{dt}$  is positive, the angular acceleration  $\vec{\alpha}$  is positive and points along the  $+z$  axis, if  $\frac{d\omega}{dt}$  is negative, the angular acceleration is negative and points along the  $-z$  axis.



The tangential acceleration vector can be expressed as a cross product of the angular acceleration and position vectors. This equation can be found by taking the derivative of equation 5

$$\vec{a} = \vec{\alpha} \times \vec{r} \quad (7)$$

The vector relationships for angular acceleration and tangential acceleration are shown below



Tangential acceleration of a point on a rotating body at a distance from the axis of rotation can be related in the same way as tangential velocity and angular velocity. Differentiating equation 4 with respect to time (the radius  $r$  is constant) gives:

$$a_t = r\alpha \quad (8)$$

The tangential acceleration  $a_t$  is the radius times the angular acceleration

### Example 10.2: A Spinning Bike Wheel

A bicycle mechanic mounts a bicycle on the repair stand and starts the rear wheel spinning from rest to a final angular velocity of 250 rpm in 5.00 s.

- (a) Calculate the average angular acceleration in  $\text{rad s}^{-2}$

$$\bar{\alpha} = \frac{\Delta\omega}{\Delta t} = \frac{250 \text{ rpm}}{5.00 \text{ s}}$$

Converting from rpm to  $\text{rad s}^{-1}$ :

$$\Delta\omega = 250 \frac{\text{rev}}{\text{min}} \cdot \frac{2\pi \text{ rad}}{\text{rev}} \cdot \frac{1 \text{ min}}{60 \text{ s}} = 26.2 \text{ rad s}^{-1}$$

Entering this back into the expression for  $\alpha$  gives:

$$\alpha = \frac{\Delta\omega}{\Delta t} = \frac{26.2 \text{ rad s}^{-1}}{5.00 \text{ s}} = 5.24 \text{ rad s}^{-2}$$

- (b) If the brakes are hit, causing an angular acceleration of  $-87 \text{ rad s}^{-2}$ , how long does it take the wheel to stop?  
Angular velocity decreases from  $26.2 \text{ rad s}^{-1}$  to zero so  $\Delta\omega = -26.2 \text{ rad s}^{-1}$ , and  $\alpha$  is given to be  $-87.3 \text{ rad s}^{-2}$

$$\Delta t = \frac{\Delta\omega}{\alpha} = \frac{-26.2 \text{ rad s}^{-1}}{-87.3 \text{ rad s}^{-2}} = 0.300 \text{ s}$$

**Example 10.3:** Wind Turbine

A wind turbine in a wind farm is being shut down for maintenance. It takes 30 s for the turbine to go from its operating angular velocity to a complete stop in which the angular velocity function is  $\omega(t) = \left[ \frac{(ts^{-1} - 30.0)^2}{100.0} \right] \text{rad s}^{-1}$ , where  $t$  is the time in seconds. If the turbine is rotating counterclockwise looking into the page:

- (a) What are the directions of the angular velocity and acceleration vectors?

Since the turbine is rotating counterclockwise, angular velocity  $\vec{\omega}$  points towards  $+z$ . Since the angular velocity is decreasing, the angular acceleration  $\vec{\alpha}$  points towards  $-z$ .

- (b) What is the average angular acceleration?

At  $t = 0$ , the initial angular velocity of the turbine is  $\omega = 9.0 \text{ rad s}^{-1}$ , the final angular velocity is zero, so the average angular velocity  $\bar{\alpha}$  is:

$$\bar{\alpha} = \frac{\Delta\omega}{\Delta t} = \frac{\omega - \omega_0}{t - t_0} = \frac{0 - 9.0 \text{ rad s}^{-1}}{30.0 - 0 \text{ s}} = -0.3 \text{ rad s}^{-2}$$

- (c) What is the instantaneous angular acceleration at  $t = 0.0, 15.0, 30.0$  s?

Taking the derivative of angular velocity with respect to time gives

$$\alpha = \frac{d\omega}{dt} = \left[ \frac{(t - 30.0)}{50.0} \right] \text{rad s}^{-2}$$

Thus:  $\alpha(0.0 \text{ s}) = -0.6 \text{ rad s}^{-2}$ ,  $\alpha(15.0 \text{ s}) = -0.3 \text{ rad s}^{-2}$ , and  $\alpha(30.0 \text{ s}) = 0 \text{ rad s}^{-2}$

## 10.2 Rotation With Constant Angular Acceleration

In this section, the definitions from the previous section are used to derive relationships among these variables, and use these relationships to analyze rotational motion for a rigid body about a fixed axis under a constant angular acceleration, forming the basis for rotational kinematics. If angular acceleration is constant, the equations of rotational kinematics simplify, similar to the equations of linear kinematics.

### Kinematics of Rotational Motion

In the previous section we saw that if a flywheel has an angular acceleration in the same direction as its angular velocity, its angular velocity increases with time and its angular displacement also increases. If the angular acceleration is opposite to the angular velocity vector, its angular velocity decreases with time. Under a constant angular acceleration, we can describe these physical situations with a consistent set of rotational kinematic equations.

If the system is rotating under a constant acceleration, then the average angular velocity follows a simple relation because the angular velocity is increasing linearly with time. The average angular velocity is just half of the sum of the initial and final values:

$$\bar{\omega} = \frac{\omega_0 + \omega_f}{2} \quad (9)$$

Using the definition of average angular velocity, an equation that relates the angular position, average angular velocity, and time can be found:

$$\bar{\omega} = \frac{\Delta\theta}{\Delta t}$$

Solving for  $\theta$  gives:

$$\theta_f = \theta_0 + \bar{\omega}t \quad (10)$$

Where  $t_0 = 0$ . This equation can be useful when the average angular velocity of the system is known. Then the angular displacement over a given period of time could be found. To determine an equation relating  $\omega, \alpha$ , and  $t$ , we start with the definition of angular acceleration:

$$\alpha = \frac{d\omega}{dt}$$

This is rearranged to  $\alpha dt = d\omega$ , then we integrate both sides of the equation from initial to final values, from  $t_0$  to  $t$  and from  $\omega_0$  to  $\omega_f$ . (angular acceleration is constant and can be pulled outside)

$$\alpha \int_{t_0}^t dt = \int_{\omega_0}^{\omega_f} d\omega$$

Setting  $t_0 = 0$  gives:

$$\alpha t = \omega_f - \omega_0$$

This is rearranged to obtain

$$\omega_f = \omega_0 + \alpha t \quad (11)$$

Where  $\omega_0$  is the initial angular velocity. This equation is the rotational counterpart to the linear kinematic equation  $v_f = v_0 + at$ . With equation 11, the angular velocity of an object at any specified time  $t$  can be found given the initial angular velocity and angular acceleration.

Doing a similar thing to the equation  $\omega = \frac{d\theta}{dt}$ , rearranging it to  $\omega dt = d\theta$  and integrating both sides from initial to final values, noting that angular acceleration is constant and does not have a time dependence. This time angular velocity is not constant, so equation 11 is substituted in:

$$\begin{aligned} \int_{t_0}^{t_f} (\omega_0 + \alpha t) dt &= \int_{\theta_0}^{\theta_f} d\theta \\ \int_{t_0}^t \omega_0 dt + \int_{t_0}^t \alpha t dt &= \int_{\theta_0}^{\theta_f} d\theta \\ \left[ \omega_0 t + \frac{1}{2} \alpha t^2 \right]_{t_0}^t &= \omega_0 t + \frac{1}{2} \alpha t^2 = \theta_f - \theta_0 \end{aligned}$$

Where  $t_0 = 0$ . Now we rearrange to obtain:

$$\theta_f = \theta_0 + \omega_0 t + \frac{1}{2} \alpha t^2 \quad (12)$$

This is the rotational counterpart to the linear kinematic equation  $s_f = s_0 + v_0 t + \frac{1}{2} at^2$ . This equation gives the angular position of a rotating rigid body at any time  $t$  given the initial conditions ( $\theta_0$  and  $\omega_0$ ) and the angular acceleration

We can find an equation that is independent of time by solving for  $t$  in equation 11 and substituting into equation 12:

$$\begin{aligned}
 \theta_f &= \theta_0 + \omega_0 \left( \frac{\omega_f - \omega_0}{\alpha} \right) + \frac{1}{2} \alpha \left( \frac{\omega_f - \omega_0}{\alpha} \right)^2 \\
 &= \theta_0 + \frac{\omega_0 \omega_f}{\alpha} - \frac{\omega_0^2}{\alpha} + \frac{1}{2} \frac{\omega_f^2}{\alpha} - \frac{\omega_0 \omega_f}{\alpha} + \frac{1}{2} \frac{\omega_0^2}{\alpha} \\
 &= \theta_0 + \frac{1}{2} \frac{\omega_f^2}{\alpha} - \frac{1}{2} \frac{\omega_0^2}{\alpha} \\
 \theta_f - \theta_0 &= \frac{\omega_f^2 - \omega_0^2}{2\alpha}
 \end{aligned}$$

This rearranges to:

$$\omega_f^2 = \omega_0^2 + 2\alpha(\Delta\theta) \quad (13)$$

Equations 10 - 13 describe fixed-axis rotation for constant acceleration and are summarized below

Rotational	Linear
$\theta_f = \theta_0 + \bar{\omega}t$	$s_f = s_0 + \bar{v}t$
$\omega_f = \omega_0 + \alpha t$	$v_f = v_0 + at$
$\theta_f = \theta_0 + \omega_0 t + \frac{1}{2}\alpha t^2$	$s_f = s_0 + v_0 t + \frac{1}{2}at^2$
$\omega_f^2 = \omega_0^2 + 2\alpha(\Delta\theta)$	$v_f^2 = v_0^2 + 2a(\Delta s)$

### **Example: 10.4/5** Calculating the Acceleration of a Fishing Pole

A deep-sea fisherman hooks a big fish that swims away from the boat, pulling the fishing line from his fishing reel. The whole system is initially at rest, and the fishing line unwinds from the reel at a radius of 4.50 cm from its axis of rotation. The reel is given an angular acceleration of  $110 \text{ rad s}^{-2}$  for 2.00 s

- (a) What is the final velocity of the reel after 2 s?

Since  $\alpha$  and  $t$  are given, the most straightforward equation to use to find  $\omega$  is  $\omega_f = \omega_0 + \alpha t$ . Because the system starts from rest,  $\omega_0 = 0$ , so:

$$\omega_f = 0 + (110 \text{ rad s}^{-2})(2.00 \text{ s}) = 220 \text{ rad s}^{-1}$$

- (b) How many revolutions does the reel make?

To find the number of revolutions, find  $\theta$  in radians (1 rev =  $2\pi$  rad).  $\alpha$  and  $t$  are given, and  $\omega_0 = 0$ , so  $\theta$  can be obtained by using:

$$\begin{aligned}
 \theta_f &= \theta_0 + \omega_0 t + \frac{1}{2}\alpha t^2 \\
 &= 0 + 0 + \frac{1}{2}(110 \text{ rad s}^{-2})(2.00 \text{ s})^2 \\
 &= 200 \text{ rad}
 \end{aligned}$$

Converting from radians to revolutions:

$$(200 \text{ rad}) \left( \frac{1 \text{ rev}}{2\pi \text{ rad}} \right) = 35.0 \text{ rev}$$

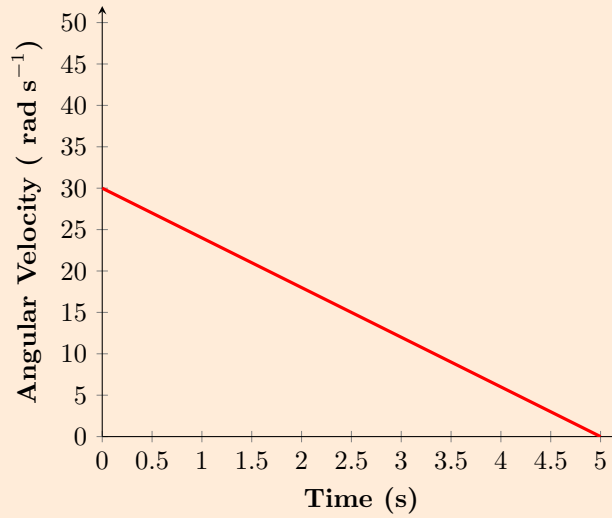
- (c) Now the fisherman applies a brake to the spinning wheel, achieving an angular acceleration of  $-300 \text{ rad s}^{-2}$ . How long does it take for the reel to come to a stop?

Solving the equation  $\omega_f = \omega_0 + \alpha t$  for  $t$ , and substituting in known values gives:

$$t = \frac{\omega_f - \omega_0}{\alpha} = \frac{0 - 220.0 \text{ rad s}^{-1}}{-300 \text{ rad s}^{-2}} = 0.733 \text{ s}$$

**Example 10.6:** Angular Acceleration of a Propeller

The figure shows a graph of the angular velocity of a propeller on an aircraft as a function of time. Its angular velocity starts at  $30 \text{ rad s}^{-1}$  and drops linearly to  $0 \text{ rad s}^{-1}$  over the course of 5 s.



- (a) Find the angular acceleration of the object and verify the result using kinematic equations

Because angular velocity varies linearly with time, angular acceleration is constant and not dependent on time. The angular acceleration is the derivative of angular velocity,  $\alpha = \frac{d\omega}{dt}$ . At  $t = 0 \text{ s}$ ,  $\omega_0 = 30 \text{ rad s}^{-1}$  and at  $t = 5 \text{ s}$ ,  $\omega_f = 0 \text{ rad s}^{-1}$

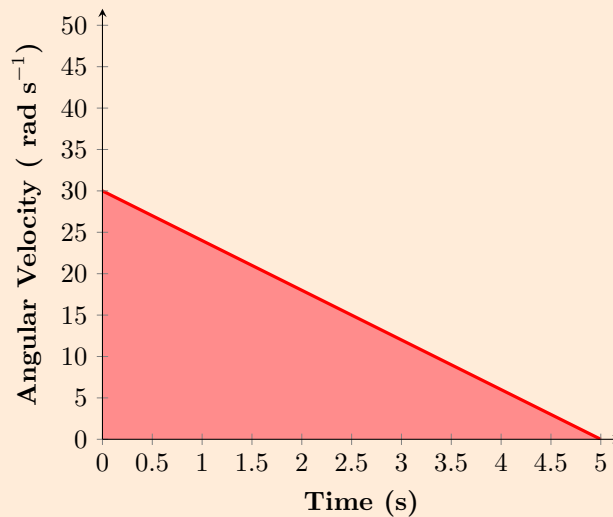
$$\alpha = \frac{\omega - \omega_0}{t - t_0} = \frac{(0 - 30.0) \text{ rad s}^{-1}}{(5.0 - 0) \text{ s}} = -6.0 \text{ rad s}^{-2}$$

- (b) Find the angle through which the propeller rotates during these 5 seconds and verify your result using kinematic equations

Since  $\omega = \frac{d\theta}{dt}$ , angular displacement  $\Delta\theta$  can be calculated by integrating the angular velocity

$$\int_{\theta_0}^{\theta_f} d\theta = \theta_f - \theta_0 = \int_{t_0}^{t_f} \omega(t) dt$$

The area under the curve can be found by calculating the area of the right triangle shown:



$$\Delta\theta = A(\text{triangle}) = \frac{1}{2}l \times h$$

$$\Delta\theta = \frac{1}{2}(30 \text{ rad s}^{-1})(5 \text{ s})$$

This is verified using equation 12 ( $\theta_f = \theta_0 + \omega_0 t + \frac{1}{2}\alpha t^2$ ), and setting  $\theta_0 = 0$  gives:

$$\theta_f = (30.0 \text{ rad s}^{-1})(5.0 \text{ s}) + \frac{1}{2}(-6.0 \text{ rad s}^{-2})(5.0 \text{ rad s}^{-1})^2 = 150.0 - 75.0 = 75.0 \text{ rad}$$

## 10.3 Relating Angular and Translational Quantities

### Angular vs. Linear Variables

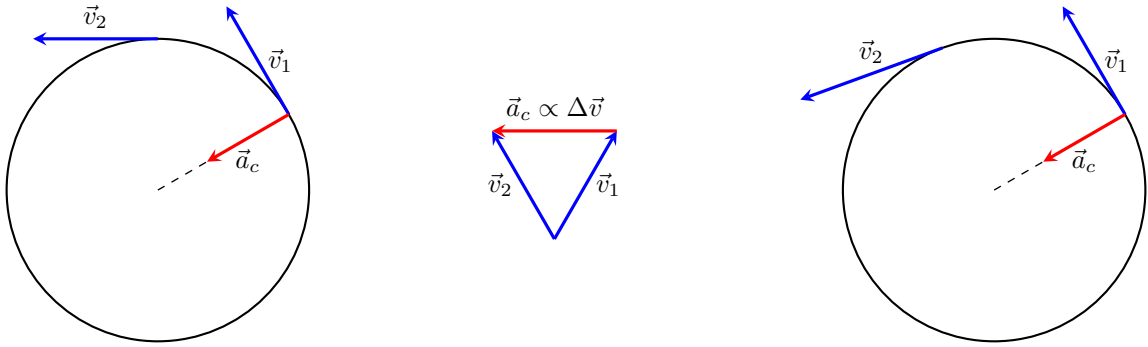
Comparing the definitions of the rotational variables with the definitions of linear kinematic variables shows that there is a mapping of the linear variables to the rotational ones. Linear position, velocity, and acceleration have their rotational counterparts, as seen below.

	Linear	Rotational
Position	$x$	$\theta$
Velocity	$v = \frac{dx}{dt}$	$\omega = \frac{d\theta}{dt}$
Acceleration	$a = \frac{dv}{dt}$	$\alpha = \frac{d\omega}{dt}$

In uniform and nonuniform circular motion, there exists a centripetal acceleration  $a_c$ . The centripetal acceleration vector points inward from the particle toward the axis of rotation. The magnitude of centripetal acceleration is:

$$a_c = \frac{v_t^2}{r} \quad (14)$$

In uniform circular motion, when  $\omega = \text{constant}$  and  $\alpha = 0$ , there is a linear acceleration - centripetal acceleration - since  $v_t = \text{constant}$ . If nonuniform circular motion is present, the rotating system has an angular acceleration, and there is both a linear centripetal acceleration that is changing ( $\Delta v_t \neq 0$ ) as well as a linear tangential acceleration.



The centripetal acceleration is due to the change in the direction of tangential velocity, whereas the tangential acceleration is due to any change in the magnitude of the tangential velocity. The tangential and centripetal acceleration vectors  $\vec{a}_t$  and  $\vec{a}_c$  are always perpendicular to each other (the direction of  $\vec{v}_t$  = the direction of  $\vec{a}_t$ ).

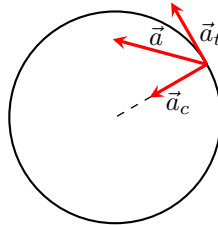
We can assign a total linear acceleration vector to a point on a rotating rigid body or a particle executing circular motion at a radius  $r$  from a fixed axis. The total linear acceleration vector  $\vec{a}$  is the vector sum of the centripetal and tangential accelerations:

$$\vec{a} = \vec{a}_c + \vec{a}_t \quad (15)$$

The total linear acceleration vector in the case of nonuniform circular motion points at an angle between the centripetal and tangential acceleration vectors. Since  $\vec{a}_c \perp \vec{a}_t$ , the magnitude of the total linear acceleration is

$$|\vec{a}| = \sqrt{a_c^2 + a_t^2}$$

If the angular acceleration is zero, the total linear acceleration is equal to the centripetal acceleration



### Relationship Between Rotational & Translational Motion

The relationship between the rotational and translational kinematic equations was mentioned above. The second relationship between rotational and translational motion relates linear and rotational variables in the special case of circular motion

Rotational	Translational	Relationship
$\theta$	$s$	$\theta = \frac{s}{r}$
$\omega$	$v_t$	$\omega = \frac{v_t}{r}$
$\alpha$	$a_t$	$\alpha = \frac{a_t}{r}$
	$a_c$	$a_c = \frac{v_t^2}{r}$



**Example 10.7** Linear Acceleration of a Centrifuge

A centrifuge has a radius of 20 cm and accelerates from a maximum rotation rate of 10,000 rpm to rest in 30 seconds under a constant angular acceleration, rotating counterclockwise. What is the magnitude of the total acceleration for a point at the tip of the centrifuge at  $t = 29.0$  s? What is the direction of the total acceleration vector?

Angular acceleration is:

$$\alpha = \frac{\omega - \omega_0}{t} = \frac{0 - (1 \cdot 10^4)2\pi/60.0 \text{ s}(\text{ rad s}^{-1})}{30.0 \text{ s}} = -34.9 \text{ rad s}^{-2}$$

Tangential acceleration is  $a_t = r\alpha$ , therefore:

$$a_t = 0.2 \text{ m}(-34.9 \text{ rad s}^{-2}) = -7.0 \text{ ms}^{-2}$$

The angular velocity at  $t = 29.0$  s can be found using the equation  $\omega = \omega_0 + \alpha t$ :

$$\begin{aligned}\omega &= \left[ (1.0 \cdot 10^4 \text{ rpm}) \left( \frac{2\pi}{60 \text{ s}} \right) \right] \text{ rad s}^{-1} + (-34.9 \text{ rad s}^{-2})(29.0 \text{ s}) \\ &= (1047.2 - 1012.71) \text{ rad s}^{-1} = 35.1 \text{ rad s}^{-1}\end{aligned}$$

The tangential velocity  $v_t$  is given by  $v_t = r\omega$ , therefore  $v_t$  at  $t = 29.0$  s is:

$$v_t = 0.2 \text{ m}(35.1 \text{ rad s}^{-1}) = 7.0 \text{ ms}^{-1}$$

The centripetal acceleration  $a_c$  at  $t = 29.0$  s can now be calculated using  $a_c = \frac{v^2}{r}$ , therefore:

$$a_c = \frac{(7.0 \text{ ms}^{-1})^2}{0.2 \text{ m}} = 245.0 \text{ ms}^{-2}$$

Since the two acceleration vectors are perpendicular to each other, the magnitude of the total linear acceleration is:

$$\begin{aligned}|\vec{a}| &= \sqrt{a_c^2 + a_t^2} = \sqrt{(245.0 \text{ ms}^{-2})^2 + (-7.0 \text{ ms}^{-2})^2} \\ &= 245.1 \text{ ms}^{-2}\end{aligned}$$

Since the centrifuge has a negative angular acceleration, it is slowing down. The angle of the total acceleration vector with respect to the centripetal acceleration vector is:

$$\theta = \tan^{-1} \frac{-7.0 \text{ ms}^{-2}}{245.0 \text{ ms}^{-2}} = -1.6^\circ$$

## 10.4 Moment of Inertia and Rotational Kinetic Energy

This section introduces two new quantities that are helpful for analyzing properties of rotating objects: moment of inertia and rotational kinetic energy

### Rotational Kinetic Energy

The energy associated with rotational motion is the same as kinetic energy in translational motion however, because kinetic energy is given by  $K = \frac{1}{2}mv^2$ , and velocity is different for every point on a rotating body, it makes sense to write kinetic energy in terms of  $\omega$ . For a single particle rotating around a fixed axis, we relate the angular velocity to the magnitude of the translational velocity using the relation  $v_t = \omega r$ , where  $r$  is the distance of the particle from the axis of rotation and  $v_t$  is its tangential velocity. Substituting into the equation for kinetic energy gives:

$$K = \frac{1}{2}mv_t^2 = \frac{1}{2}m(\omega r)^2 = \frac{1}{2}(mr^2)\omega^2$$

In the case of a rigid rotating body, the total body can be divided up into a large number of smaller masses, each with mass  $m_j$  and distance from the axis of rotation  $r_j$ , such that the total mass of the body  $M$  is equal to the sum of the original masses  $M = \sum_j m_j$ . Each smaller mass has a tangential velocity  $v_j$ . The total kinetic energy of the rigid rotating body is:

$$K = \sum_j \frac{1}{2}m_j v_j^2 = \sum_j \frac{1}{2}m_j (r_j \omega_j)^2$$

Since  $\omega_j = \omega$  for all masses:

$$K = \frac{1}{2} \left( \sum_j m_j r_j^2 \right) \omega^2 \quad (16)$$

Like translational kinetic energy, the units of this equation are joules (J)

### Moment of Inertia

The quantity  $\sum_j m_j r_j^2$  is the counterpart for mass in the equation for rotational kinetic energy. This quantity is called the moment of inertia  $I$  and has units of  $\text{kg m}^2$

$$I = \sum_j m_j r_j^2 \quad (17)$$

For now the expression is left in summation form, representing the moment of inertia of a system of point particles rotating about a fixed axis. The moment of inertia of a single point particle about a fixed axis is simply  $mr^2$ .

The moment of inertia is the quantitative measure of rotational inertia, just as in translational motion, mass is the quantitative measure of linear inertia - that is, the more massive an object is, the more inertia it has, and the greater is its resistance to change in linear velocity. Similarly, the greater the moment of inertia of a rigid body or system of particles, the greater its resistance to change in angular velocity about a fixed axis of rotation. Rigid bodies and systems of particles with more mass concentrated at a greater distance from the axis of rotation have greater moments of inertia than bodies/systems of the same mass but concentrated near the axis of rotation. Substituting equation 17 into equation 16, the expression for the kinetic energy of a rotating rigid body becomes:

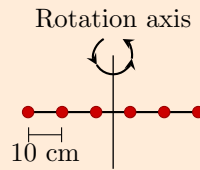
$$K = \frac{1}{2}I\omega^2 \quad (18)$$

The kinetic energy of a rotating rigid body is directly proportional to the moment of inertia and the square of the angular velocity. The rotational and translational quantities for kinetic energy are summarized below:

Rotational	Translational
$I = \sum_j m_j r_j^2$	$m$
$K = \frac{1}{2}I\omega^2$	$K = \frac{1}{2}mv^2$

**Example 10.8:** Moment of Inertia of a System of Particles

Six small washers are spaced 10 cm apart on a rod of negligible mass and 0.5 m in length. The mass of each washer is 20 g. The rod rotates about an axis located at 25 cm as shown.



- (a) What is the moment of inertia of the system?

Use the definition of moment of inertia for a system of particles:  $I = \sum_j m_j r_j^2$

$$\begin{aligned} I &= (0.02 \text{ kg})[2(0.25 \text{ m})^2 + 2(0.15 \text{ m})^2 + 2(0.05 \text{ m})^2] \\ &= 3.5 \cdot 10^{-3} \text{ kg m}^2 \end{aligned}$$

- (b) If the two washers closest to the axis are removed, what is the moment of inertia of the remaining four washers?  
Repeat the steps from part (a):

$$\begin{aligned} I &= (0.02 \text{ kg})[2(0.25 \text{ m})^2 + 2(0.15 \text{ m})^2] \\ &= 3.4 \cdot 10^{-3} \text{ kg m}^2 \end{aligned}$$

- (c) If the system with six washers rotates at 5 rev/s, what is its rotational kinetic energy?  
Insert the result from part (a) into the expression for rotational kinetic energy:

$$\begin{aligned} K &= \frac{1}{2} I \omega^2 \\ &= \frac{1}{2} (3.5 \times 10^{-3} \text{ kg m}^2) [5.0(2\pi \text{ rad s}^{-1})]^2 \\ &= 1.73 \text{ J} \end{aligned}$$

## Applying Rotational Kinetic Energy

### Problem Solving Strategy: Rotational Energy

1. Determine that energy or work is involved in the rotation
2. Determine the system of interest
3. Analyze the situation to determine the types of work and energy involved
4. If there are no losses of energy due to friction and other nonconservative forces, mechanical energy is conserved:  
 $K_0 + U_0 = K_f + U_f$
5. If nonconservative forces are present, mechanical energy is not conserved, determine what they are and calculate them as necessary
6. Eliminate terms wherever possible to simplify the algebra
7. Evaluate the numerical solution to see if it makes sense in the physical situation presented in the wording of the problem

### Example 10.9: Calculating Helicopter Energies

A typical small rescue helicopter has four blades, each is 4.00 m long and has a mass of 50.0 kg. The blades can be approximated as thin rods that rotate about one end of an axis perpendicular to their length. The helicopter has a total loaded mass of 1000 kg.

- (a) Calculate the rotational kinetic energy in the blades when they rotate at 300 rpm

The rotational kinetic energy is given by  $\frac{1}{2}I\omega^2$ . First, the angular velocity must be converted to  $\text{rad s}^{-1}$ , the angular velocity  $\omega$  is

$$\omega = \frac{300 \text{ rev}}{1.00 \text{ min}} \cdot \frac{2\pi \text{ rad}}{1 \text{ rev}} \cdot \frac{1.00 \text{ min}}{60.0 \text{ s}} = 31.4 \text{ rad s}^{-1}$$

The moment of inertia of one blade is that of a thin rod rotated about its end, the total  $I$  is 4 times this because there are 4 blades.

$$I = 4 \frac{ML^2}{3} = 4 \left( \frac{(50.0 \text{ kg})(4.00 \text{ m})^2}{3} \right) = 1067.0 \text{ kg m}^2$$

Entering  $\omega$  and  $I$  into the expression for rotational kinetic energy gives:

$$\begin{aligned} K &= \frac{1}{2}I\omega^2 \\ &= \frac{1}{2}(1067 \text{ kg m}^2)(31.4 \text{ rad s}^{-1})^2 \\ &= 5.26 \cdot 10^5 \text{ J} \end{aligned}$$

- (b) Calculate the translational kinetic energy of the helicopter when it flies at  $20.0 \text{ ms}^{-1}$ , and compare it with the rotational energy in the blades

Entering the given values into the equation for translational kinetic energy gives:

$$\begin{aligned} K &= \frac{1}{2}mv^2 \\ &= \frac{1}{2}(10^3 \text{ kg})(20 \text{ ms}^{-1})^2 \\ &= 2.00 \cdot 10^5 \text{ J} \end{aligned}$$

To compare kinetic energies take the ratio of translational kinetic energy to rotational kinetic energy:

$$\frac{2.00 \cdot 10^5 \text{ J}}{5.26 \cdot 10^5 \text{ J}} = 0.380$$

This tells us that most of the energy of the helicopter is in its spinning blades

**Example 10.10:** Energy in a Boomerang

A person hurls a boomerang into the air with a velocity of  $30.0\text{ m/s}$  at an angle of  $40.0^\circ$  with respect to the horizontal. It has a mass of  $1.0\text{ kg}$  and it rotating at  $10.0\text{ rev/s}$ . The moment of inertia of the boomerang is given as  $I = \frac{1}{12}mL^2$  where  $L = 0.7\text{ m}$ .

- (a) What is the total energy of the boomerang when it leaves the hand?

The total kinetic energy of the boomerang is:  $K_{tot} = K_R + K_T$

The moment of inertia is:  $I = \frac{1}{12}mL^2 = \frac{1}{12}(1.0\text{ kg})(0.7\text{ m})^2 = 0.041\text{ kg m}^2$

The angular velocity is:  $\omega = 2\pi(10.0\text{ rev/s}) = 62.83\text{ rad s}^{-1}$

Therefore the rotational and translational kinetic energies are:

$$K_R = \frac{1}{2}I\omega^2$$

$$= \frac{1}{2}(0.041\text{ kg m}^2)(62.83\text{ rad s}^{-1})^2$$

$$= 80.93\text{ J}$$

$$K_T = \frac{1}{2}mv^2$$

$$= \frac{1}{2}(1.0\text{ kg})(30.0\text{ ms}^{-1})^2$$

$$= 450.0\text{ J}$$

Therefore, the total kinetic energy of the boomerang is  $K_{tot} = (80.93 + 450.0)\text{ J} = 530.93\text{ J}$

- (b) How high does the boomerang go from the elevation of the hand, neglecting air resistance?

The maximum height can be found using the conservation of mechanical energy ( $E_0 = E_f$ ). Since the boomerang is launched at an angle, the total energy of the system must be written in terms of its linear kinetic energies using velocity in the  $x$  and  $y$  directions.

The energy when the boomerang leaves the hand is:  $E_0 = \frac{1}{2}mv_x^2 + \frac{1}{2}mv_y^2 + \frac{1}{2}I\omega^2$

The total energy at the maximum height  $h$  is:  $E_f = \frac{1}{2}mv_x^2 + \frac{1}{2}I\omega^2 + mgh$

Equating and canceling like terms gives:

$$\begin{aligned} \cancel{\frac{1}{2}mv_x^2} + \frac{1}{2}mv_y^2 + \cancel{\frac{1}{2}I\omega^2} &= \cancel{\frac{1}{2}mv_x^2} + \cancel{\frac{1}{2}I\omega^2} + mgh \\ \frac{1}{2}mv_y^2 &= mgh \end{aligned}$$

Rearranging and entering known values gives:

$$\begin{aligned} h &= \frac{\cancel{mv_y^2}}{2\cancel{mg}} \rightarrow h = \frac{v_y^2}{2g} \\ h &= \frac{(19.28\text{ ms}^{-1})^2}{2(9.8\text{ ms}^{-2})} = 18.97\text{ m} \end{aligned}$$

## 10.5 Calculating Moments of Inertia

This section covers how to calculate the moment of inertia for several standard types of objects, as well as how to use known moments of inertia to find the moment of inertia for a shifted axis or for a compound object.

### Moment of Inertia

The moment of inertia  $I$  of an object is defined to be  $I = \sum_i m_i r_i^2$  for all the point masses that make up the object.

Because  $r$  is the distance to the axis of rotation for each piece of mass that makes up the object, the moment of inertia depends on the chosen axis. Take a simple example of two masses at the end of a massless (negligibly small mass) rod and calculate the moment of inertia about two different axes. The summation over the masses is simple because the two masses can be approximated as point masses, and the sum has only two terms.

In the case with the axis in the center of the barbell, each of the two masses  $m$  is a distance  $R$  away from the axis giving a moment of inertia of

$$I_1 = mR^2 + mR^2 = 2mR^2$$

In the case with the axis at the end of the barbell (passing through one of the masses) the moment of inertia is:

$$I_2 = m(0)^2 + m(2R)^2 = 4mR^2$$

From this result we can conclude that it is twice as hard to rotate the barbell about the end than about its center.

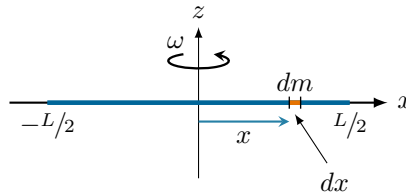
This example uses two point masses, making the sum easy to calculate. In the derivation of equation 17, each piece had the same magnitude of velocity, meaning the whole piece had to have a single distance  $r$  to the axis of rotation. This is only possible taking an infinitesimally small piece of mass  $dm$ . This suggests that the moment of inertia can be written by evaluating an integral over infinitesimal masses rather than doing a discrete sum over finite masses:

$$I = \sum_i m_i r_i^2 \rightarrow I = \int r^2 dm \quad (19)$$

This is all that is needed to generalize the equation for complex shapes

### A Uniform Thin Rod With an Axis Through The Center

Consider a uniform ( $\rho$  & shape) thin rod of mass  $M$  and length  $L$  as shown. We want a thin rod so that we can assume the cross-sectional area of the rod is small and the rod can be thought of as a string of masses along a 1D straight line. In this example, the axis of rotation is perpendicular to the rod and passes through the midpoint. The axes are oriented so that the  $z$  axis is the axis of rotation and the  $x$  axis passes through the length of the rod.



We define  $dm$  to be a small element of the mass making up the rod. The moment of inertia integral is an integral over the mass distribution. However, we know how to integrate over space, not mass and therefore need to find a way to relate mass to spatial variables. This is done using the linear mass density  $\lambda$  of the object, which is the mass per unit length. Since the mass and density of the object are both uniform:

$$\lambda = \frac{m}{L} \rightarrow m = \lambda L$$

Since  $\lambda$  is constant, taking the differential of each side of the equation gives:

$$dm = d(\lambda L) = \lambda(dL)$$

This is where the choice to orient the rod along the  $x$  axis becomes helpful. A piece of the rod  $dl$  lies completely along the  $x$  axis and has a length  $dx$ , in this situation  $dL = dx$ . Therefore  $dm = \lambda(dx)$ , giving an integration variable that can be used. The distance of each piece of mass  $dm$  from the axis is given by the variable  $x$ , putting this all together gives:

$$I = \int r^2 dm = \int x^2 dm = \int x^2 \lambda dx$$

The last step is to be careful about the limits of integration. The rod extends from  $x = -L/2$  to  $x = L/2$ , since the axis is in the middle of the rod at  $x = 0$ . This gives:

$$\begin{aligned} I &= \int_{-L/2}^{L/2} x^2 \lambda dx = \lambda \frac{x^3}{3} \Big|_{-L/2}^{L/2} = \lambda \left( \frac{1}{3} \right) \left[ \left( \frac{L}{2} \right)^3 - \left( -\frac{L}{2} \right)^3 \right] \\ &= \lambda \left( \frac{1}{3} \right) \frac{L^3}{8} (2) = \frac{M}{L} \left( \frac{1}{3} \right) \frac{L^3}{8} (2) = \frac{1}{12} ML^2 \end{aligned}$$

### A Uniform Thin Rod With Axis at The End

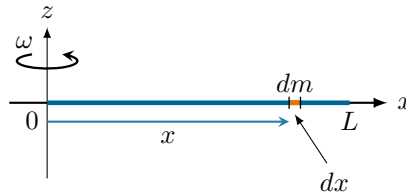
Now consider the same uniform thin rod of mass  $M$  and length  $L$  this time with the axis of rotation at the end of the rod. The quantity  $dm$  is again defined to be a small element of mass making up the rod, just as before:

$$I = \int r^2 dm = \int x^2 dm = \int x^2 \lambda dx$$

However, this time there are different limits of integration. The rod extends from  $x = 0$  to  $x = L$ , since the axis is at the end of the rod at  $x = 0$ . Therefore:

$$\begin{aligned} I &= \int_0^L x^2 \lambda dx = \lambda \frac{x^3}{3} \Big|_0^L = \lambda \left( \frac{1}{3} \right) [(L)^3 - (0)^3] \\ &= \lambda \left( \frac{1}{3} \right) L^3 = \frac{M}{L} \left( \frac{1}{3} \right) L^3 = \frac{1}{3} ML^2 \end{aligned}$$

Note that the rotational inertia of the rod about its endpoint is larger than the rotational inertia about its center by a factor of four.



### Parallel-Axis Theorem

The parallel-axis theorem provides a simpler method for determining the moment of inertia for a rod about any axis parallel to the axis through the center of mass.

Let  $m$  be the mass of an object and let  $d$  be the distance from an axis through the object's center of mass to a new axis:

$$I_{pa} = I_{cm} + md^2 \quad (20)$$

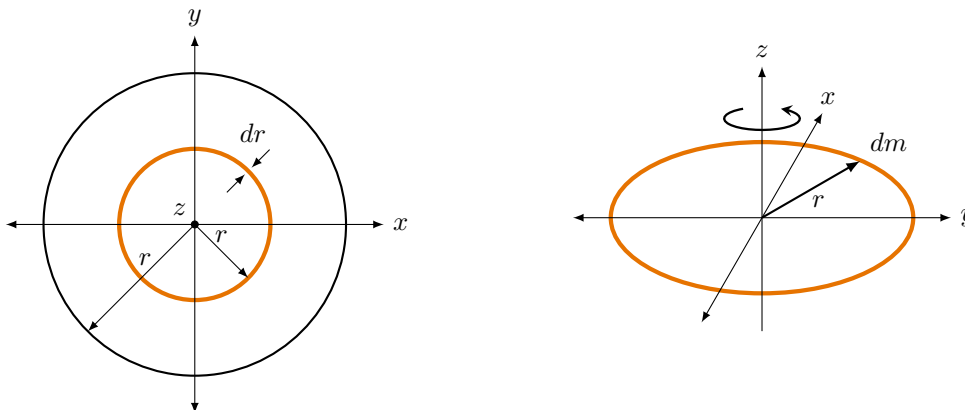
Applying this to the examples solved above:

$$I_{end} = I_{cm} + md^2 = \frac{1}{12} mL^2 + m \left( \frac{L}{2} \right)^2 = \left( \frac{1}{12} + \frac{1}{4} \right) mL^2 = \frac{1}{3} mL^2$$

This result agrees with the more lengthy calculation above.

### A Uniform Thin Disk About an Axis Through The Center

Integrating to find the moment of inertia of a 2D object is a little bit trickier however, one shape is commonly done at this level of study, a uniform thin disk about an axis through its center



Since the disk is thin, we can take the mass as distributed entirely in the  $xy$  plane. Starting again with the relationship for the surface mass density, which is the mass per unit surface area. Since it is uniform, the surface mass density  $\sigma$  is constant:

$$\sigma = \frac{m}{A} \rightarrow \sigma A = m, \text{ so: } dm = \sigma(dA)$$

Now we use a simplification for the area. The area can be thought of as made up of a series of thin rings, where each ring is a mass increment  $dm$  of radius  $r$  equidistant from the axis. The infinitesimal area of each ring  $dA$  is therefore given by the length of each ring ( $2\pi r$ ) times the infinitesimal width of each ring  $dr$ :

$$A = \pi r^2, \quad dA = d(\pi r^2) = \pi dr^2 = 2\pi r dr$$

The full area of the disk is then made up from adding all the thin rings with a radius range from 0 to  $R$ . This radius range then becomes our limits of integration for  $dr$ , we integrate from  $r = 0$  to  $r = R$ . Putting this together gives:

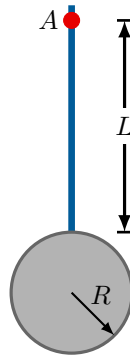
$$\begin{aligned} I &= \int_0^R r^2 \sigma (2\pi r) dr = 2\pi\sigma \int_0^R r^3 dr = 2\pi\sigma \left. \frac{r^4}{4} \right|_0^R = 2\pi\sigma \left( \frac{R^4}{4} - 0 \right) \\ &= 2\pi \frac{m}{A} \left( \frac{R^4}{4} \right) = 2\pi \frac{m}{\pi R^2} \left( \frac{R^4}{4} \right) = \frac{1}{2} m R^2 \end{aligned}$$

### Calculating The Moment of Inertia For Compound Objects

Now consider a compound object like a disk at the end of a thin rod. This cannot be easily integrated to find the moment of inertia because it is not a uniformly shaped object. However, going back to the initial definition of moment of inertia as a summation, we can reason that a compound object's moment of inertia can be found from the sum of each part of the object:

$$I_{tot} = \sum_i I_i \quad (21)$$

It is important to note that the moments of inertia of the objects in equation 21 are about a common axis. In the case of this object, that would be a rod of length  $L$  rotating about its end, and a thin disk of radius  $R$  rotating about an axis shifted off the center by a distance  $L + R$ , where  $R$  is the radius of the disk. We define the mass of the rod to be  $m_r$  and the mass of the disk to be  $m_d$



The moment of inertia of the rod is simply  $\frac{1}{3} m_r L^2$ , but to find the moment of inertia of the disk about the axis shown, parallel-axis theorem must be used. The moment of inertia of the disk about its center is  $\frac{1}{2} m_d R^2$ , applying the parallel-axis theorem  $I_{pa} = I_{cm} + m d^2$  to find

$$I_{pa} = \frac{1}{2} m_d R^2 + m_d (L + R)^2$$

### Applying Moment of Inertia Calculations to Solve Problems

#### Example 10.11: Person on a Merry-Go-Round

A 25 kg child stands a distance  $r = 1.0$  m from the axis of a rotating merry-go-round. The merry-go-round can be approximated as a uniform solid disk with a mass of 500 kg and radius of 2.0 m. Find the moment of inertia of the system

The mass and distance to the axis of rotation of the child as well as the mass and radius of the merry-go-round are given. Since the mass and size of the child are much smaller than the merry-go-round, we can approximate the child as a point mass

$m_c = 25$  kg,  $r_c = 1.0$  m,  $m_m = 500$  kg,  $r_m = 2.0$  m, The goal is to find  $\sum_i I_i$

For the child,  $I_c = m_c r_c^2$ , and for the merry-go-round,  $I_m = \frac{1}{2} m_m r_m^2$ , therefore

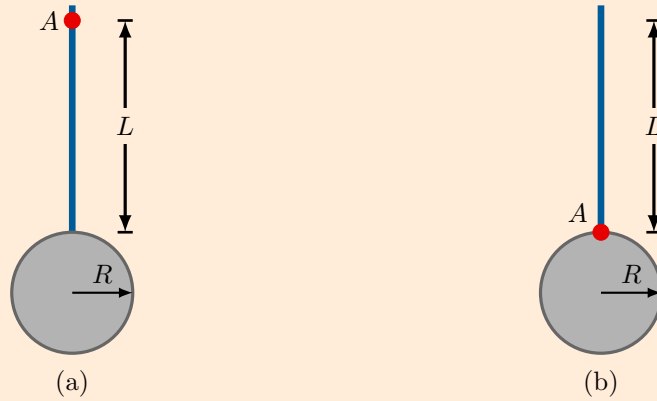
$$\begin{aligned} I_{tot} &= I_c + I_m \\ &= 25(1)^2 + \frac{1}{2}(500)(2)^2 = 25 + 1000 \\ &= 1025 \text{ kg} \end{aligned}$$

The value should be close to the moment of inertia of the merry-go-round by itself because it has much more mass distributed away from the axis than the child does.



**Example 10.12:** Rod and Solid Sphere

Find the moment of inertia of the rod and solid sphere combination about the two axes as shown below. The rod has a length of 0.5 m and a mass of 2.0 kg. The radius of the sphere is 20.0 cm and has a mass of 1.0 kg.



In (a), the center of mass of the sphere is located a distance  $L + R$  from the axis of rotation. In (b), the center of mass is located a distance  $R$  from the axis of rotation. In both cases, the moment of inertia of the rod is about an axis at one end.

1. For axis (a), the moment of inertia is:

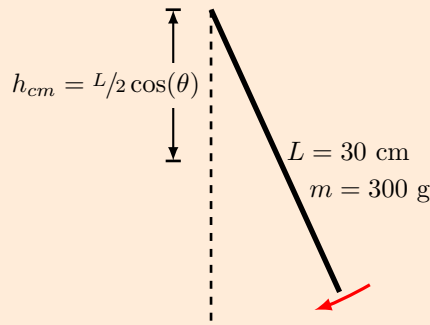
$$\begin{aligned}
 I_{tot} &= \sum_i I_i = I_r + I_s \\
 I_s &= I_{cm} + m_s(L + R)^2 = \frac{2}{5}m_s R^2 + m_s(L + R)^2 \\
 I_{tot} &= I_r + I_s = \frac{1}{3}m_r L^2 + \frac{2}{5}m_s R^2 + m_s(L + R)^2 \\
 &= \frac{1}{3}(2.0 \text{ kg})(0.5 \text{ m})^2 + \frac{2}{5}(1.0 \text{ kg})(0.2 \text{ m})^2 + (1.0 \text{ kg})(0.5 \text{ m} + 0.2 \text{ m})^2 \\
 &= (0.167 + 0.016 + 0.490) \text{ kg m}^2 = 0.673 \text{ kg m}^2
 \end{aligned}$$

- (b) For axis (b), the moment of inertia is:

$$\begin{aligned}
 I_s &= \frac{2}{5}m_s R^2 + m_s R^2 \\
 I_{tot} &= I_r + I_s = \frac{1}{3}m_r L^2 + \frac{2}{5}m_s R^2 + m_s R^2 \\
 &= \frac{1}{3}(2.0 \text{ kg})(0.5 \text{ m})^2 + \frac{7}{5}(1.0 \text{ kg})(0.2 \text{ m})^2 \\
 &= (0.167 + 0.056) \text{ kg m}^2 = 0.223 \text{ kg m}^2
 \end{aligned}$$

**Example 10.13:** Angular Velocity of a Pendulum

A pendulum in the shape of a rod is released from rest at an angle of  $30^\circ$ . It has a length of 30 cm and a mass of 300 g. What is its angular velocity  $\omega$  at its lowest point?



Use conservation of energy to solve the problem. At the point of release, the pendulum has gravitational potential energy which is determined from the height of the center of mass above the lowest point in the swing. At the bottom of the swing, all of the gravitational potential energy is converted into rotational kinetic energy.

The change in Potential energy is equal to the change in rotational kinetic energy,  $\Delta U + \Delta K = 0$

At the top of the swing,  $U = mgh_{cm} = mg\frac{L}{2}$ . At the bottom of the swing,  $U = mg\frac{L}{2}$  with respect to the lowest point

At the top of the swing, the rotational kinetic energy is  $K = 0$ . At the bottom of the swing  $K = \frac{1}{2}I\omega^2$ , therefore:

$$\Delta U + \Delta K = 0 \rightarrow \left( mg\frac{L}{2}(1 - \cos(\theta)) - 0 \right) + \left( 0 - \frac{1}{2}I\omega^2 \right) = 0$$

Or,

$$\frac{1}{2}I\omega^2 = mg\frac{L}{2}(1 - \cos(\theta))$$

Solving for  $\omega$  gives:

$$\omega = \sqrt{mg\frac{L}{I}(1 - \cos(\theta))} = \sqrt{mg\frac{L}{\frac{1}{3}mL^2}(1 - \cos(\theta))} = \sqrt{g\frac{3}{L}(1 - \cos(\theta))}$$

Plugging in the known values gives:

$$\omega = \sqrt{9.8 \text{ ms}^{-2} \cdot \frac{3}{0.3 \text{ m}} (1 - \cos(30^\circ))} = 3.6 \text{ rad s}^{-1}$$

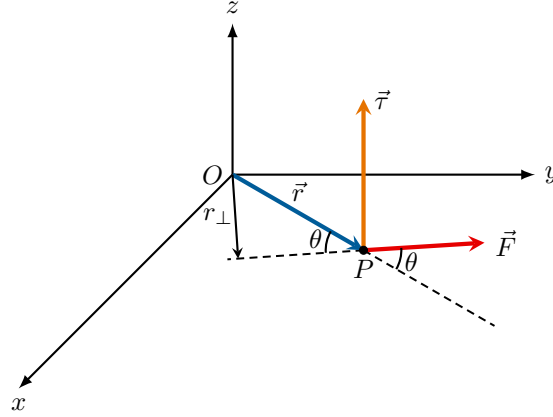
Note that the angular velocity of the pendulum does not depend on its mass

## 10.6 Torque

So far many variables have been defined which are rotational equivalents to their translational counterparts. Since force changes the translational motion of objects, the rotational counterpart must be related to changing the rotational motion of an object about an axis, this is called torque. Note that for rotation in a plane, torque has two possible directions, it is either clockwise or counterclockwise relative to the pivot point.

Now consider how to define torque in the general 3D case. When a force  $\vec{F}$  is applied to a point  $P$  whose position  $\vec{r}$  is relative to  $O$ , the torque  $\vec{\tau}$  around  $O$  is:

$$\vec{\tau} = \vec{r} \times \vec{F} \quad (22)$$



From the definition of the cross product, the torque  $\vec{\tau}$  is perpendicular to the plane containing  $\vec{r}$  and  $\vec{F}$  and has magnitude  $|\vec{\tau}|$  where  $\theta$  is the angle between the vectors  $\vec{r}$  and  $\vec{F}$

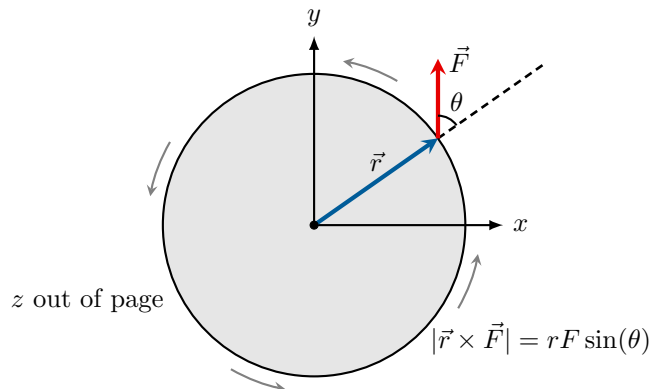
$$|\vec{\tau}| = |\vec{r} \times \vec{F}| = rF \sin(\theta)$$

The SI unit of torque is newtons times meters (usually written as Nm or N·m). The quantity  $r_{\perp} = r \sin(\theta)$  is the perpendicular distance from  $O$  to the line determined by the vector  $\vec{F}$  and is called the **lever arm**. The greater the lever arm, the greater the magnitude of the torque. In terms of the lever arm, the magnitude of torque is

$$|\vec{\tau}| = r_{\perp} F \quad (23)$$

The cross product  $\vec{r} \times \vec{F}$  also tells us the sign of the torque. In the figure, the cross product  $\vec{r} \times \vec{F}$  is along the  $+z$  axis, which by convention is a positive torque.

Considering a disk that is free to rotate about an axis through the center, we can see how the angle between the radius  $\vec{r}$  and the force  $\vec{F}$  affects the magnitude of the torque. If the angle is zero, torque is zero; torque is at a maximum when the angle is  $90^\circ$ . The torque in the figure is positive because the direction of the torque by the right hand rule is out of the page along  $+z$ . The disk rotates counterclockwise due to the torque, in the same direction as a positive angular acceleration.



Any number of torques can be calculated about a given axis. The individual torques add to produce a net torque about the axis. When the appropriate sign is assigned to the magnitudes of individual torques about a specified axis, the net torque about the axis is the sum of the individual torques:

$$\vec{\tau}_{net} = \sum_i \vec{\tau}_i \quad (24)$$

## Calculating Net Torque for Rigid Bodies on a Fixed Axis

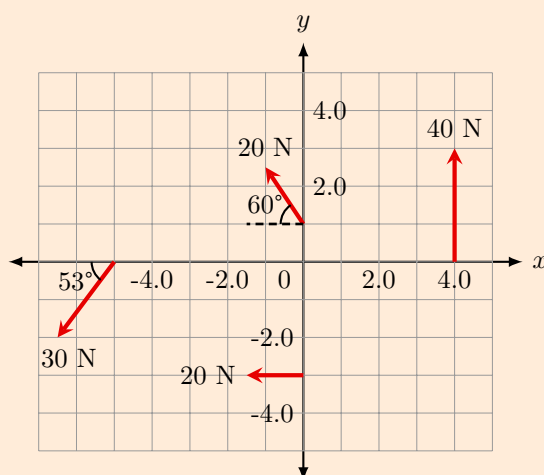
In the following examples, torque is calculated both abstractly and as applied to a rigid body.

### Problem Solving Strategy: Finding Net Torque

1. Choose a coordinate system with the pivot axis of rotation as the origin
2. Determine the angle between the lever arm  $\vec{r}$  and the force vector  $\vec{F}$
3. Take the cross product  $\vec{r} \times \vec{F}$  to determine if torque is positive or negative about the pivot point or axis
4. Evaluate the magnitude of the torque using  $r_{\perp}F$
5. Assign the appropriate sign, +, or -, to the magnitude
6. Sum all the torques to find the net torque

### Example 10.14: Calculating Torque

Forces are shown in the figure at particular locations and orientations with respect to a given  $xy$  coordinate system. Find the torque due to each force about the origin, then use the results to find the net torque about the origin.



This problem requires calculating torque. All known quantities - forces with directions and lever arms - are given. The goal is to find each individual torque and the net torque by summing the individual torques. Be careful to assign the correct sign to each torque by using the cross product  $\vec{r} \times \vec{F}$

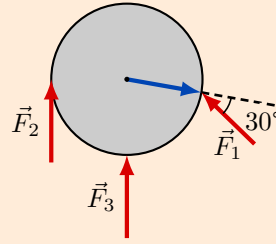
Use  $|\vec{\tau}| = r_{\perp}F = rF \sin(\theta)$  to find the magnitude and  $\vec{\tau} = \vec{r} \times \vec{F}$  to determine the sign of the torque

- The torque from the 40 N force in the first quadrant is given by  $(4)(40) \sin(90^\circ) = 160 \text{ Nm}$   
The cross product  $\vec{r} \times \vec{F}$  is out of the page, positive
- The torque from the 20 N force in the third quadrant is given by  $(-3)(20) \sin(90^\circ) = -60 \text{ Nm}$   
The cross product  $\vec{r} \times \vec{F}$  is into the page, negative
- The torque from the 30 N force in the third quadrant is given by  $(5)(30) \sin(53^\circ) = 120 \text{ Nm}$   
The cross product  $\vec{r} \times \vec{F}$  is out of the page, positive
- The torque from the 20 N force in the first quadrant is given by  $(1)(20) \sin(30^\circ) = 10 \text{ Nm}$   
The cross product  $\vec{r} \times \vec{F}$  is out of the page, positive
- The net torque is therefore  $\sum_i |\vec{\tau}_i| = 160 - 60 + 120 + 10 = 230 \text{ Nm}$

Note that each force that acts in the counterclockwise direction has a positive torque, whereas each force that acts in the clockwise direction has a negative torque. The torque is also greater when the distance, force, or perpendicular components are greater.

**Example 10.15:** Calculating Torque on a Rigid Body

The figure shows several forces acting at different locations and angles on a flywheel. We have  $|\vec{F}_1| = 20 \text{ N}$ ,  $|\vec{F}_2| = 30 \text{ N}$ ,  $|\vec{F}_3| = 30 \text{ N}$ , and  $r = 0.5 \text{ m}$ . Find the net torque on the flywheel about an axis through the center.



Calculate each torque individually, using the cross product and determine the sign of the torque, then sum them to find the net torque.

Starting with  $\vec{F}_1$ , which makes an angle of  $90^\circ + 60^\circ$  with the radius vector  $\vec{r}$ . Taking the cross product, shows that it is out of the page and therefore positive. This is also seen calculating its magnitude:

$$|\vec{\tau}_1| = rF_1 \sin(150^\circ) = (0.5 \text{ m})(20 \text{ N})(\sin(150^\circ)) = 5.0 \text{ Nm}$$

For  $\vec{F}_2$ , the angle between  $\vec{F}_2$  and  $\vec{r}$  is  $90^\circ$  and the cross product is into the page, so the torque value is negative. Its value is:

$$|\vec{\tau}_2| = -rF_2 \sin(90^\circ) = (-0.5 \text{ m})(30 \text{ N}) = -15.0 \text{ Nm}$$

For  $\vec{F}_3$ , the angle it makes with  $\vec{r}$  is zero so  $\vec{r} \times \vec{F}_3 = 0$ . Therefore,  $\vec{F}_3$  does not produce any torque on the flywheel.

The net torque is the sum of the individual torques:

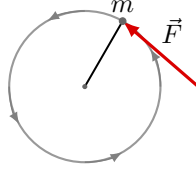
$$\begin{aligned} \tau_{net} &= \sum_i |\tau_i| \\ &= (5 - 15) \text{ Nm} = -10 \text{ Nm} \end{aligned}$$

The axis of rotation is at the center of mass of the flywheel. Since the flywheel is on a fixed axis, it is not free to translate. If it were on a frictionless surface and not fixed in place,  $\vec{F}_3$  would cause the flywheel to translate, as well as  $\vec{F}_1$ . Its motion would be a combination of translation and rotation.

## 10.7 Newton's Second Law For Rotation

Is there an analogous equation to Newton's second law,  $\sum \vec{F} = m\vec{a}$ , which involves torque and rotational motion? We start with Newton's second law for a single particle rotating around an axis and executing circular motion. If we exert a force  $\vec{F}$  on a point mass  $m$  that is at a distance  $r$  from a pivot point, the particle is constrained to move in a circular path with fixed radius and the force is tangent to the circle. Apply Newton's second law to determine the magnitude of acceleration  $a = F/m$  in the direction of  $\vec{F}$ . Recall that the magnitude of the tangential acceleration is proportional to the magnitude of the angular acceleration by  $a = r\alpha$ . Substituting this into Newton's second law gives:

$$F = mr\alpha$$



Multiply both sides of the equation by  $r$ :

$$rF = mr^2\alpha$$

Note that the left side of this equation is the torque about the axis of rotation, where  $r$  is the lever arm and  $F$  is the force, perpendicular to  $r$ . Recall that the moment of inertia for a point particle is  $I = mr^2$ . The torque applied perpendicularly to the point mass in the figure is therefore

$$\tau = I\alpha$$

The torque on the particle is equal to the moment of inertia about the rotation axis times the angular acceleration. This can be generalized to a rigid body rotating about a fixed axis.

$$\tau_{net} = \sum_i \tau_i = I\alpha \quad (25)$$

If more than one torque acts on a rigid body about a fixed axis, then the sum of the torques equals the moment of inertia times the angular acceleration. The term  $I\alpha$  is a scalar quantity and can be positive or negative depending on the sign of the net torque. The convention is that counterclockwise is positive, thus if a rigid body is rotating clockwise and experiences a positive torque, the angular acceleration is positive.

Equation 25 is Newton's second law for rotation and tells us how to relate torque, moment of inertia, and rotational kinematics. This is called the equation for rotational dynamics.

### Deriving Newton's Second Law For Rotation in Vector Form

Like before when we found the angular acceleration vector, we may also find the torque vector. The second law tells us the relationship between net force and how to change the translational motion of an object. A vector rotational equivalent of this equation can be found using equations 7 and 8 ( $a_t = r\alpha$ ). Equation 7 relates the angular acceleration to the position and tangential acceleration vectors:

$$\vec{a} = \vec{\alpha} \times \vec{r}$$

We form the cross product of this equation with  $\vec{r}$  and use a cross product identity (note  $\vec{r} \cdot \vec{\alpha} = 0$ ):

$$\vec{r} \times \vec{a} = \vec{r} \times (\vec{\alpha} \times \vec{r}) = \vec{\alpha}(\vec{r} \cdot \vec{r}) - \vec{r}(\vec{r} \cdot \vec{\alpha}) = \alpha(\vec{r} \cdot \vec{r}) = \alpha r^2$$

Now we form the cross product of Newton's second law with the position vector  $\vec{r}$ :

$$\sum (\vec{r} \times \vec{F}) = \vec{r} \times (m\vec{a}) = m\vec{r} \times \vec{a} = mr^2\vec{\alpha}$$

Identifying the first term on the left as the sum of the torques and  $mr^2$  as the moment of inertia, we arrive at Newton's second law of rotation in vector form:

$$\tau_{net} = \sum \vec{\tau} = I\vec{\alpha} \quad (26)$$

This is exactly the same as equation 25 but with the torque and angular acceleration as vectors. An important point is that the torque vector is in the same direction as the angular acceleration.

## Applying The Rotational Dynamics Equation

### Problem Solving Strategy: Rotational Dynamics

1. Examine the situation to determine that torque and mass are involved in the rotation; draw a sketch of the situation
2. Determine the system of interest
3. Draw a free-body diagram, draw and label all external forces acting on the system
4. Identify the pivot point If the object is in equilibrium, it must be in equilibrium for all possible pivot points, chose the one that simplifies your work the most
5. Apply  $\sum_i \tau_i = I\alpha$ , the rotational equivalent of Newton's second law, to solve the problem. Care must be taken to use the correct moment of inertia and to consider the torque about the point of rotation
6. As always, check the solution to see if it is reasonable

### Example 10.16: Calculating The Effect of Mass Distribution on a Merry-Go-Round

Consider a father pushing a playground merry-go-round. He exerts a force of 250 N at the edge of the 50.0 kg merry-go-round, which has a 1.50 m radius. Calculate the angular acceleration produced when:

- (a) When no one is on the merry-go-round

The moment of inertia of a solid disk about this axis is  $I = \frac{1}{2}MR^2$ . We have  $M = 50.0$  kg and  $R = 1.50$  m, so:

$$I = \frac{1}{2}(50.0 \text{ kg})(1.50 \text{ m})^2 = 56.25 \text{ kg m}^2$$

To find the net torque, note that applied force is perpendicular to the radius and friction is negligible, so that:

$$\tau = rF \sin(\theta) = (1.50 \text{ m})(250.0 \text{ N}) \text{ Nm} = 375.0 \text{ Nm}$$

Now to find the angular acceleration, substitute in the known values

$$\alpha = \frac{\tau}{I} = \frac{375.0 \text{ Nm}}{56.25 \text{ kg m}^2} = 6.67 \text{ rad s}^{-2}$$

- (b) When an 18.0 kg child sits 1.25 m away from the center

We expect the angular acceleration of the system to be less in this part because the moment of inertia is greater when the child is on the merry-go-round. To find the total moment of inertia  $I$ , we must first find the child's moment of inertia  $I_c$  by approximating the child as a point mass at a distance of 1.25 m from the axis, then:

$$I_c = mR^2 = (18.0 \text{ kg})(1.25 \text{ m})^2 = 28.13 \text{ kg m}^2$$

The total moment of inertia is the sum of the moments of inertia of the merry-go-round and the child (about the same axis):

$$\begin{aligned} I &= I_m + I_c \\ &= 56.25 \text{ kg m}^2 + 28.13 \text{ kg m}^2 \\ &= 84.38 \text{ kg m}^2 \end{aligned}$$

Substituting known values into the equation for  $\alpha$  gives:

$$\begin{aligned} \alpha &= \frac{\tau}{I} = \frac{375.0 \text{ Nm}}{84.38 \text{ kg m}^2} \\ &= 4.44 \text{ rad s}^{-2} \end{aligned}$$

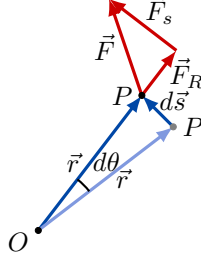
The angular acceleration is less when the child is on the merry-go-round than when it is empty, as expected. The angular accelerations found are quite large, partly due to the fact that friction was considered negligible.

## 10.8 Work and Power For Rotational Motion

In this section, work and power are defined within the context of rotation about a fixed axis. The discussion of work and power makes our treatment of rotational motion almost complete, with the exception of rolling motion and angular momentum.

### Work For Rotational Motion

Now that we can calculate the kinetic energy for rotating rigid bodies, we can proceed with a discussion of the work done on a rigid body rotating about a fixed axis. A rigid body that has rotated through an angle  $d\theta$  from  $A$  to  $B$  while under the influence of a force  $\vec{F}$ . The external force  $\vec{F}$  is applied to point  $P$ , whose position is  $\vec{r}$ , and the rigid body is constrained to rotate about a fixed axis that is perpendicular to the page and passes through  $O$ . The rotational axis is fixed, so that the vector  $\vec{r}$  moves in a circle of radius  $r$ , and the vector  $d\vec{s}$  is perpendicular to  $\vec{r}$ .



From equation 2, we have  $\vec{s} = \vec{\theta} \times \vec{r}$ , thus:

$$d\vec{s} = d(\vec{\theta} \times \vec{r}) = d\vec{\theta} \times \vec{r} + d\vec{r} \times \vec{\theta} = d\vec{\theta} \times \vec{r}$$

Note that  $d\vec{r}$  is zero because  $\vec{r}$  is fixed on the rigid body from the origin  $O$  to point  $P$ . Using the definition of work we can obtain

$$W = \int \sum \vec{F} \cdot d\vec{s} = \int \sum \vec{F} \cdot (d\vec{\theta} \times \vec{r}) = \int d\vec{\theta} \cdot \left( \vec{r} \times \sum \vec{F} \right)$$

We use the identity  $\vec{a} \cdot (\vec{b} \times \vec{c}) = \vec{b} \cdot (\vec{c} \times \vec{a})$ . Noting that  $(\vec{r} \times \sum \vec{F}) = \sum \vec{\tau}$ , we arrive at the expression for the rotational work done on a rigid body:

$$W = \int \sum \vec{\tau} \cdot d\vec{\theta} \quad (27)$$

The total work done on a rigid body is the sum of the torques integrated over the angle through which the body rotate. The incremental work is:

$$dW = \left( \sum_i \tau_i \right) d\theta \quad (28)$$

Where we have taken the dot product from equation 27, leaving only torques along the axis of rotation. In a rigid body, all particles rotate through the same angle, thus the work of every external force is equal to the torque times the common incremental angle  $d\theta$ . The quantity  $\left( \sum_i \tau_i \right)$  is the net torque on the body due to external forces.

Similarly, we found the kinetic energy of a rigid body rotating around a fixed axis by summing the kinetic energy of each particle that makes up the rigid body. Since the work-energy theorem,  $W_i = \Delta K_i$ , is valid for each particle, it is valid for the sum of the particles and the entire body.

### Work-Energy Theorem For Rotation

The work-energy theorem for a rigid body rotating around a fixed axis is:

$$W_{AB} = K_B - K_A \quad (29)$$

Where  $K = \frac{1}{2} I \omega^2$  and the rotational work done by a net force rotating a body from point  $A$  to point  $B$  is:

$$W_{AB} = \int_{\theta_A}^{\theta_B} \left( \sum_i \tau_i \right) d\theta \quad (30)$$

#### Problem Solving Strategy: Work-Energy Theorem For Rotational Motion

1. Identify the forces on the body and draw a free-body diagram. Calculate the torque for each force.
2. Calculate the work done during the body's rotation by every torque
3. Apply the work-energy theorem by equating net work done on the body to the change in rotational kinetic energy



### Example 10.17: Rotational Work and Energy

A 12.0 Nm torque is applied to a flywheel that rotates about a fixed axis and has a moment of inertia of 30.0 kg m<sup>2</sup>. If the flywheel is initially at rest, what is its angular velocity after it has turned through eight revolutions?

Apply the work-energy theorem, torque and angular displacement are given, making it possible to solve for final angular velocity

The flywheel turns through eight revolutions which is  $2\pi(8 \text{ rev}) = 16\pi \text{ rad}$ . The work done by the torque, which is constant and can be pulled outside the integral in equation 30 is:

$$W_{AB} = \tau(\theta_B - \theta_A)$$

Applying the work-energy theorem:

$$\begin{aligned} W_{AB} &= \Delta K = K_B - K_A \\ \tau(\theta_B - \theta_A) &= \frac{1}{2}I\omega_B^2 - \frac{1}{2}I\omega_A^2 \end{aligned}$$

With  $\tau = 12.0 \text{ Nm}$ ,  $(\theta_B - \theta_A) = 16.0\pi \text{ rad}$ ,  $I = 30.0 \text{ kg m}^2$ , and  $\omega_A = 0$ , we have:

$$(12.0 \text{ Nm})(16.0\pi \text{ rad}) = \frac{1}{2}(30.0 \text{ kg m}^2)(\omega_B^2) - 0$$

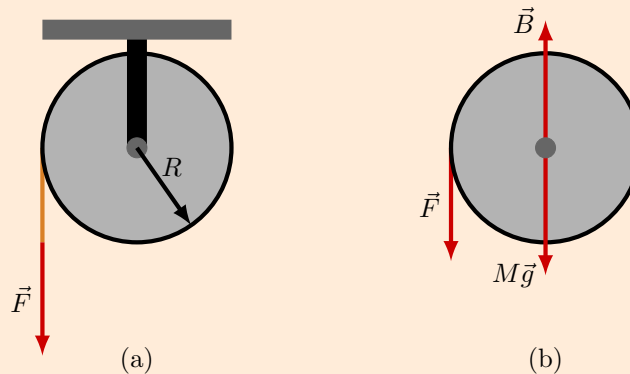
Therefore,

$$\omega_B = 6.3 \text{ rad s}^{-1}$$

This is the angular velocity of the flywheel after eight revolutions. The work-energy theorem provides an efficient way to analyze rotational motion, connecting torque and kinetic energy

### Example 10.18: Rotational Work of a Pulley

A string wrapped around a pulley is pulled with a constant downward force  $\vec{F}$  of magnitude 50 N. The radius is  $R = 0.10 \text{ m}$  and the moment of inertia is  $I = 2.5 \cdot 10^{-3} \text{ kg m}^2$ . If the string does not slip what is the angular velocity of the pulley after 1.0 m of string has unwound? (Assume the pulley starts from rest)



Looking at the free-body diagram, we see that neither  $\vec{B}$ , the force on the bearings of the pulley, nor  $M\vec{g}$ , the weight of the pulley, exerts a torque around the rotational axis, and therefore does no work on the pulley. As the pulley rotates through an angle  $\theta$ ,  $\vec{F}$  acts through a distance  $d$  such that  $d = R\theta$

Since torque due to  $\vec{F}$  has magnitude  $\tau = RF$ , we have:

$$W = \tau\theta = (FR)\theta = Fd$$

If the force on the string acts through a distance of 1.0 m, we have, from the work-energy theorem:

$$\begin{aligned} W_{AB} &= K_B - K_A \\ Fd &= \frac{1}{2}I\omega^2 - 0 \\ (50.0 \text{ N})(1.0 \text{ m}) &= \frac{1}{2}(2.5 \times 10^{-3} \text{ kg m}^2)\omega^2 \end{aligned}$$

Solving for  $\omega$ , we obtain:

$$\omega = 200.0 \text{ rad s}^{-1}$$

## Power For Rotational Motion

Power for rotational motion is equally as important as power in linear motion and can be derived in a similar way as in linear motion when the force is constant. The linear power when the force is constant is  $P = \vec{F} \cdot \vec{v}$ . If the net torque is constant over the angular displacement, equation 25 ( $\tau_{net} = \sum_i \tau_i = I\alpha$ ) simplifies and the net torque can be taken out of the integral. From chapter 7, the instantaneous power is defined as the rate of doing work:

$$P = \frac{dW}{dt}$$

If net torque is constant, equation 25 becomes  $W = \tau\theta$  and the power is:

$$P = \frac{dW}{dt} = \frac{d}{dt}(\tau\theta) = \tau \frac{d\theta}{dt}$$

Rewriting where  $\omega = \frac{d\theta}{dt}$ , gives:

$$P = \tau\omega \quad (31)$$

### Example 10.19: Torque on a Boat Propeller

A boat engine starts operating at  $9.0 \cdot 10^4$  W is running at 300 rev/min. What is the torque on the propeller shaft?

Rotation rate in rev/min and power consumption are given, torque can be calculated simply

Converting from rev/min to  $\text{rad s}^{-1}$ :

$$\frac{300 \text{ rev}}{1 \text{ min}} \cdot \frac{2\pi \text{ rad}}{1 \text{ rev}} = \frac{1885 \text{ rad}}{1 \text{ min}} \cdot \frac{1 \text{ min}}{60 \text{ s}} = 31.4 \text{ rad s}^{-1}$$

Rearranging the equation  $P = \tau\omega$  and substituting in the values:

$$\begin{aligned} \tau &= \frac{P}{\omega} = \frac{9.0 \cdot 10^4 \text{ W}}{31.4 \text{ rad s}^{-1}} \\ &= 2864.8 \text{ Nm} \end{aligned}$$

It is important to note that the radian is a dimensionless unit because its definition is the ratio of two lengths, it therefore does not appear in the solution

## Rotational and Translational Relationships Summarized

The rotational quantities and their linear analogs are summarized in three tables. The first summarizes the rotational variables for circular motion about a fixed axis with their linear analogs and the connecting equation (where applicable). The second summarizes the rotational and translational kinematic equations. The third summarizes the rotational dynamics equations with their linear analogs

**Table 1**

Rotational	Translational	Relationship
$\theta$	$x$	$\theta = \frac{s}{r}$
$\omega$	$v_t$	$\omega = \frac{v_t}{r}$
$\alpha$	$a_t$	$\alpha = \frac{a_t}{r}$
	$a_c$	$a_c = \frac{v_t^2}{r}$

**Table 2**

Rotational	Translational
$\theta_f = \theta_0 + \bar{\omega}t$	$x_f = x_0 + \bar{v}t$
$\omega_f = \omega_0 + \alpha t$	$v_f = v_0 + at$
$\theta_f = \theta_0 + \omega_0 t + \frac{1}{2}\alpha t^2$	$x_f = x_0 + v_0 t + \frac{1}{2}at^2$
$\omega_f^2 = \omega_0^2 + 2\alpha(\Delta\theta)$	$v_f^2 = v_0^2 + 2a(\Delta x)$

**Table 3**

Rotational	Translational
$I = \sum_i m_i R_i^2$	$m$
$K = \frac{1}{2}I\omega^2$	$K = \frac{1}{2}mv^2$
$\sum_i \tau_i = I\alpha$	$\sum_i \vec{F}_i = m\vec{a}$
$W_{AB} = \int_{\theta_A}^{\theta_B} \left( \sum_i \tau_i \right) d\theta$	$W = \int \vec{F} \cdot d\vec{s}$
$P = \tau\omega$	$P = \vec{F} \cdot \vec{v}$