

ECE 345 / ME 380

Introduction to Control Systems

Lecture Notes 9

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Outline

- What is the frequency response?
- Review: Log magnitude
- Sketching the Bode plot
- The Nyquist criterion
- Gain and Phase Margin
- Transient characteristics in frequency response
- Steady-state characteristics in frequency response



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Learning Objectives

- Define and plot the frequency response of a system
- Use the Nyquist criterion to determine the stability of a system
- Find stability and gain and phase margins via
 - Bode plots
 - Nyquist diagrams
- Find the closed-loop time response parameters of peak time, settling time, and percent overshoot given the open-loop frequency response

References:

- Nise, Chapter 10.1–10.3, 10.5–10.8, 10.11. Recommended: 10.12–10.13.

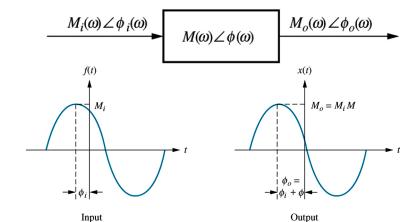
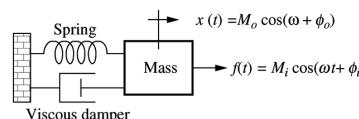


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What is the frequency response?

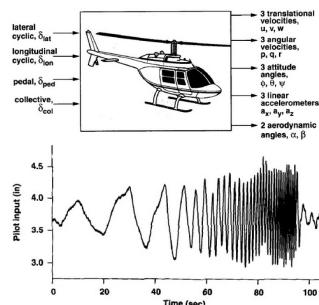
For a stable system,

- A sinusoidal input generates a sinusoidal output
- Magnitude and phase depend on the frequency of the input signal
- A Bode plot is the frequency response described in magnitude and phase



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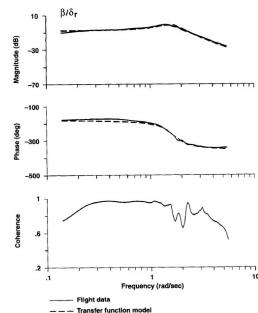
What is the frequency response?



Images from "Aircraft and Rotorcraft System Identification", M. Tischler and R. Remple, AIAA Education Series, 2006.



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Review: Magnitude and phase

Rewrite the transfer function as

$$G(s) = \frac{K(s+z_1)(s+z_2)\cdots(z+z_m)}{s^k(s+p_1)(s+p_2)\cdots(s+p_{n-k})}$$

Then take the magnitude of $G(s)$ and convert it into dB

$$20 \log |G(j\omega)| = 20 \log K + 20 \log |(s+z_1)| + \dots + 20 \log |(s+z_m)| - 20 \log |s^k| - 20 \log |(s+p_1)| - \dots - 20 \log |(s+p_{n-k})| \Big|_{s=j\omega}$$

Note that phase is similarly additive

$$\angle G(j\omega) = \angle K + \angle(s+z_1) + \cdots + \angle(s+z_m) - \angle s^k - \angle(s+p_1) - \cdots - \angle(s+p_{n-k}) \Big|_{s=j\omega}$$



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Sketching the Bode plot

What this additivity means: A Bode plot can be sketched by adding individual Bode plots of each of its elements.

For example:

- The Bode plot of $G(s) = \frac{1}{(s+1)(s+20)}$ looks like the sum of the Bode plots of $G_1(s) = \frac{1}{s+1}$ and the Bode plot of $G_2(s) = \frac{1}{s+20}$.
 - This is equivalent to

$$\begin{aligned} 20 \log |G(s)| &= 20 \log |1| - 20 \log |s+1| - 20 \log |s+20| \\ \angle G(s) &= \angle(1) - \angle(s+1) - \angle(s+20) \end{aligned}$$

- Nise Example 10.2.

Straight-line approximations of these elements are often very good for back-of-the-envelope sketches.

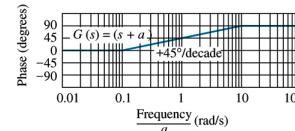
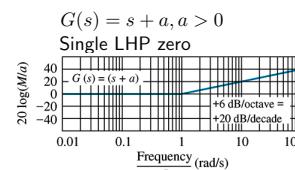
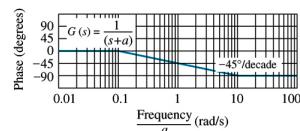
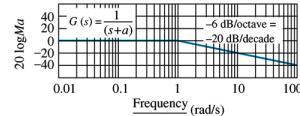


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Sketching the Bode plot

Common elements

- $G(s) = \frac{1}{s+a}$, $a > 0$
Single LHP pole

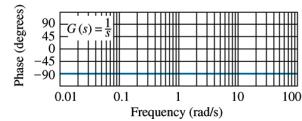


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Sketching the Bode plot

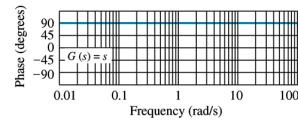
Common elements

- $G(s) = \frac{1}{s}$
Single LHP pole at the origin
-



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- $G(s) = s$
Single LHP zero at the origin
-

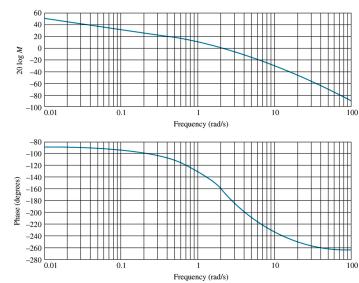


Clicker question

Consider a negative unity feedback system with *open-loop* transfer function $G(s)$, shown in the Bode plot to the right.

What is the **type number** of the *closed-loop system* $Y(s)/R(s)$?

- 0.
- 1.
- 2.
- Type number cannot be deduced from the plot.



Exercise: What feature of this diagram indicates the magnitude of steady-state error?

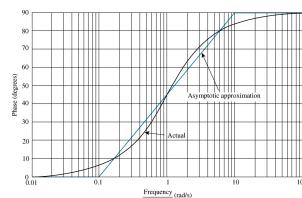
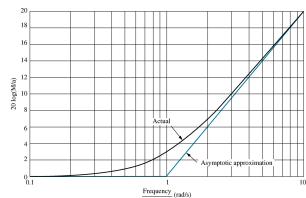


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Sketching the Bode plot

How reasonable are straight line approximations?

- $G(s) = s + a, a > 0$ (Single LHP zero)

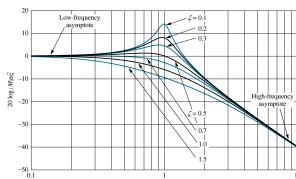


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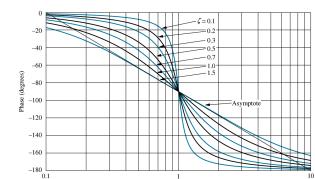
Sketching the Bode plot

Common elements

- $G(s) = \frac{1}{s^2 + 2\zeta\omega_n s + \omega_n^2}$, (Complex conjugate LHP pole pair)



- Peak magnitude $M_p = \frac{1}{2\sqrt{1-\zeta^2}}$ occurs at frequency $\omega_p = \omega_n \sqrt{1-2\zeta^2}$



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Sketching the Bode plot

1. Straight-line approximations

- Drop-off in slope of -20dB/dec for each pole
- Increase in slope of +20dB/dec for each zero
- Flat slope for frequencies lower than the pole / zero when pole / zero is not located at the origin
- 3dB rule

2. Complex conjugate pair of poles

- Break-point occurs near ω_n
- Steepness of phase change increases as ζ decreases
- Peak of magnitude increases as ζ decreases

Sketching the Bode plot

For a given Bode diagram, it is possible to identify certain features of the unknown system $G(s)$:

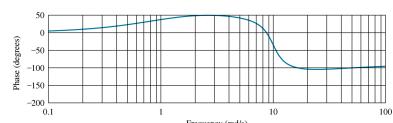
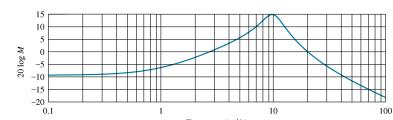
- Order of $G(s)$ ($= n - m$)
- Type number of the closed-loop system under negative unity feedback (with open-loop transfer function $G(s)$)
- Steady state error of the closed-loop system under negative unity feedback (with open-loop transfer function $G(s)$)

This can be very useful in designing control laws for systems whose model may be based on experimental data (and not on first principles, e.g., $F = ma$).

Sketching the Bode plot

Clicker question

Consider the Bode plot of a system $G(s)$. What is the order of $G(s)$, that is, what is the difference between the number of poles and the number of zeros ($n - m$)?

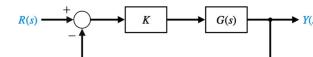


- A. $G(s)$ is of order 1.
- B. $G(s)$ is of order 2.
- C. $G(s)$ is of order 3.
- D. The number of poles can be deduced, but not the number of zeros.
- E. No information about the number of poles or zeros of $G(s)$ can be deduced.

Exercise: What features of this diagram indicate the existence of a zero?

Nyquist Criterion

For a negative unity feedback system



The Nyquist criterion relates the stability of the closed-loop system to the open-loop frequency response and open-loop pole location.

Similar to root locus, in that stability of $\frac{Y(s)}{R(s)}$ can be determined by analyzing $G(s)$ only.

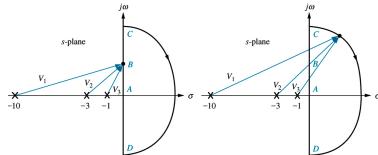
Efficient computational tools in Matlab

- `bode(sys)`
- `nyquist(sys)`
- `margin(sys)`

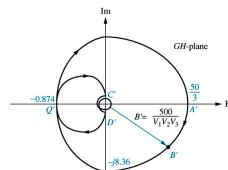
Nyquist Criterion

Main idea:

- Evaluate $G(s)$ for values of s that lie on a specially chosen closed contour that encloses the entire RHP.
- This generates a new closed contour in the complex plane.
- For the new closed contour, count encirclements of the point -1 .



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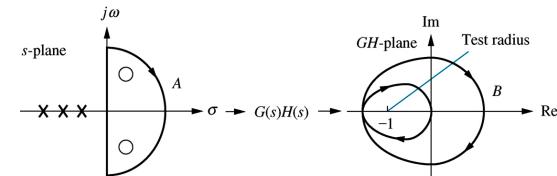


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Nyquist Criterion

$$Z = P - N$$

- Z : number of poles of the closed-loop system $Y(s)/R(s)$ in the RHP
- N : number of counter-clockwise encirclements of -1
- P : number of poles of open-loop system $G(s)$ in the RHP



For stable closed-loop systems, we will have $Z = 0$.

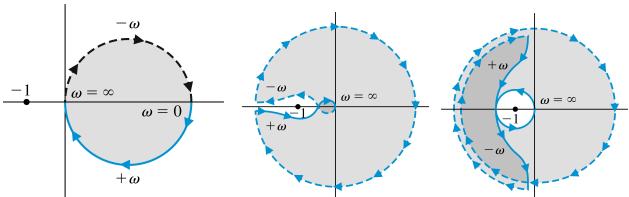
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Nyquist Criterion

Clicker question

Which of the following Nyquist diagrams shows 2 clockwise encirclements of -1 (e.g., $N = -2$)?



- A.
B.
C.
D. Both B. and C.
E. None of the above.

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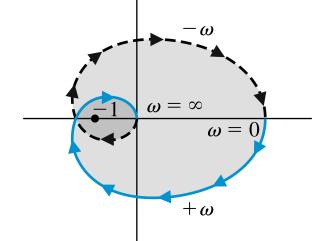
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Nyquist Criterion

Clicker question

Consider a system whose open-loop transfer function $G(s) = \frac{1}{(s+3)(s+1)^2}$ has the following Nyquist diagram. Is the closed-loop system stable under negative unity feedback with $K = 1$?

- A. Yes, because $Z = P - N = 0 - 0 = 0$.
B. Yes, because $Z = P - N = 2 - 2 = 0$.
C. No, because $Z = P - N = 3 - 2 = 1$.
D. No, because $Z = P - N = 0 - (-2) = +2$.
E. Very little confidence in any of the above answers.



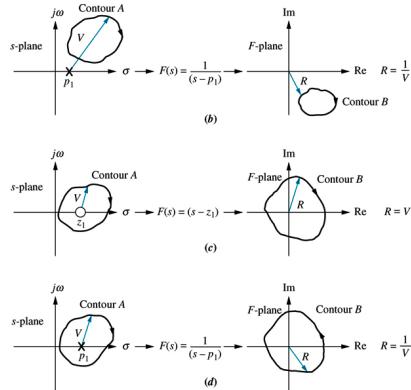
Exercise: How does the Nyquist diagram change if $K \neq 1$?

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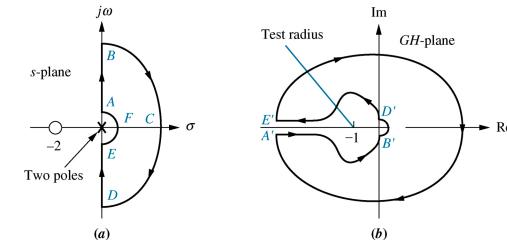
Nyquist Criterion

Where does the Nyquist Criterion come from?



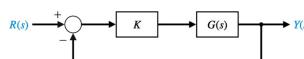
Nyquist Criterion

What if there are poles / zeros on the imaginary axis?



Gain and Phase Margin

Relative measures of stability



- Useful for systems which have branches of the root locus extending into the RHP
- Most often for systems that are stable in open-loop configuration, but can become unstable when gain is increased.

Gain margin: Factor of gain by which you need to would need to increase K in order to destabilize the system

Phase margin: Amount of phase which would needed to be removed from the system in order to destabilize it.

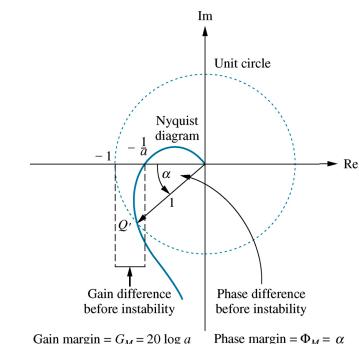
Measures of *how close* the current configuration is to instability.

Gain and Phase Margin

Measured by distance to -1 point in complex plane

Gain margin: Change in open-loop gain, at 180° phase shift, to make the closed-loop system unstable.
(= Reciprocal of real-axis crossing)

Phase margin: Change in open-loop phase shift, required at unity gain, to make the closed-loop system unstable.

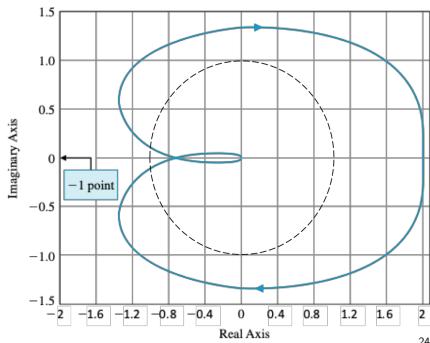


Gain and Phase Margin

Clicker question

What is the approximate gain margin and phase margin?

- A. $G_M \approx 0.7$, $\Phi_M \approx \tan^{-1}(0.2)$
- B. $G_M \approx \frac{1.0}{0.7}$, $\Phi_M \approx \tan^{-1}\left(\frac{0.2}{1.0}\right)$
- C. $G_M \approx -\frac{1.0}{0.7}$, $\Phi_M \approx \tan^{-1}\left(-\frac{0.2}{1.0}\right)$
- D. $G_M \approx \frac{1.0}{0.7}$, $\Phi_M \approx \tan^{-1}\left(\frac{-0.2}{1.0}\right)$
- E. $G_M \approx \frac{1.0}{0.3}$, $\Phi_M \approx \tan^{-1}(-0.2)$

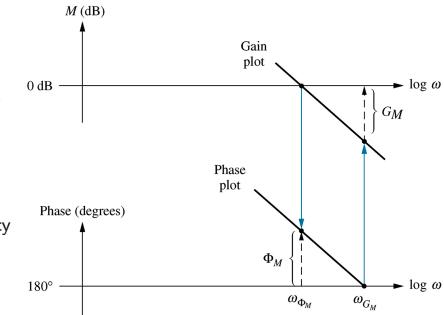


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Gain and Phase Margin

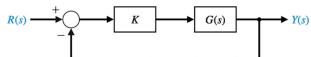
Gain margin and phase margin can also be determined from the Bode diagram



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Gain and Phase Margin



- Assess *open-loop* transfer function to determine *closed-loop* system stability
- Some systems may have infinite gain margin (e.g., $G(s) = \frac{1}{s+2}$ which *cannot* be destabilized by turning up the gain K)
- **Both $G_M > 0$ dB and $\Phi_M > 0^\circ$ are required for the closed-loop system to be stable**
- The system is unstable if *either* gain margin or phase margin are negative!



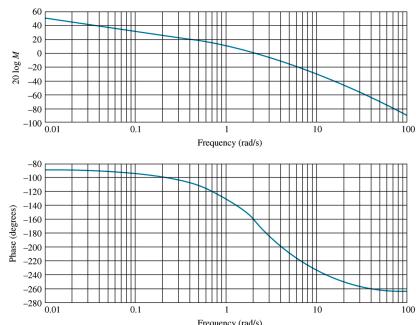
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Gain and Phase Margin

Clicker question

Is the closed-loop system with Bode diagram below stable?

- A. Yes, $G_M \approx 5$ dB, $\Phi_M \approx 20^\circ$
- B. Yes, $G_M \approx -5$ dB, $\Phi_M \approx 20^\circ$
- C. No, $G_M \approx -5$ dB, $\Phi_M \approx -160^\circ$
- D. No, $G_M \approx 5$ dB, $\Phi_M \approx -20^\circ$



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