



Simulation of the first stages of a turbofan using OpenFOAM

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Contents

1	Intr	roduction	2	
2	Cas	se	3	
	2.1	Description of the case	3	
	2.2	Hypotheses	4	
3	Pre	-processing	6	
	3.1	Mesh generation	6	
		3.1.1 Selection of the tutorial	6	
		3.1.2 blockMesh	6	
		3.1.3 Mesh refinement	12	
		3.1.4 Comparison of different types of mesh	17	
		3.1.4.1 Coarse mesh	18	
		3.1.4.2 Standard mesh	19	
		3.1.4.3 Dense mesh	20	
	3.2	Rotation	22	
	3.3	Boundary conditions	24	
	3.4	Properties of the flow	28	
	3.5		30	
	3.6	fvSchemes and fvSolution	33	
4	Sim	nulation of the turbofan	37	
	4.1	Selection of the solver	37	
	4.2	Commands to run the simulation	38	
5	Post-process 39			
	5.1	Analysis of the results	40	
	5.2	Conclusions	41	

1 Introduction

During the development of the course 'Application of Open-Source CFD to engineering problems' we have learned the basics of how to use and solve real-world cases and situations related with fluid mechanics using OpenFOAM, an open source CFD tool. The first days of the course, several possible projects were presented and we had to choose one of them and make a report. After agreeing with the professor, we decided to simulate an option that was not on that list. Since we are really interested in propulsion, we thought that it was a good idea to try to simulate the flow inside the first stages of compression of a turbofan engine.

These type of engines are the most used propulsion system in the aerospace industry. It presents several advantages to other systems such as the turbohelix or the turbojet; for example, this kind of engine takes advantage of the flow that goes through the fan (that cannot be more compressed or heated given that the combustion chamber has a limited volume) and comes out through the rear nozzle. This results in a higher thrust for the same amount of fuel that is burned. Thus, it is no surprise that state-of-the-art planes such as the Airbus 380 or the Boeing 747 use this kind of propulsion system.

In order to do a realistic simulation, we have been gathering lots of information of this type of engines and their typical working conditions. Also, we have been searching for information about how could we solve a geometry that is rotating. Several simulations and comparisons will be presented in this report to analyze the validity of the results obtained.

It will be presented in this report how to use the Multiple Reference Frame utility (MRF) as well as the boundary conditions, the mesh generation and the solver that has been used for the case. Additionally, several hypotheses will be considered in order to alleviate the computation time (given that this project has been simulated in a laptop).

2 Case

2.1 Description of the case

The aim of the current project is to analyze the airflow inside the first stages of a turbofan. The turbofan consists of an initial stage where a fan is placed. This first stage increases the pressure of the air that goes through it and, as it can be seen in 2.1, a part of the incoming airflow goes to the low pressure compression stage (main or primary flow) and the other goes to the conducts (secondary flow). As the secondary flow advances through the conduct, the main flow enters the engine core where it goes through the low pressure and high pressure compression stages. By increasing the pressure of the air, the density also increases; thus, a higher mass flow can be mixed with the fuel and burned in the combustion chamber. After this process, the gases go through the turbines, interchanging energy with them (pressure and velocity, mainly), and they arrive at the nozzle, where the air is accelerated, and a thrust force is obtained.



Figure 2.1: Parts of a turbofan

To simulate the flow in the first stages of the turbofan, we have downloaded the following turbine model from GrabCad, shown in 2.2. This model is pretty similar with the one shown above (2.1) and the simulation will take place between the back side of the fan and the back side of the second compressor. There are several realistic and potentially functional models on GrabCad; however, this model has been selected because it did not present incompatibilities with SolidWorks and Salome. Given that all the parts of the model have to be clearly differentiated in order to export them as STL files, we could not use an assembly.



Figure 2.2: Model used

2.2 Hypotheses

Given that we are not able to use a supercomputer and the computational power that is available to us is very limited since we are using our own personal computers to run this simulation, several hypotheses have to be made to alleviate the calculation time, as mentioned before. Some of them can be assumed and some other will make the case a non-realistic project. These hypotheses are presented below.

- Viscous flow
- Incompressible flow

- Newtonian flow
- Stationary flow
- Laminar flow

Clearly, the flow is not incompressible in reality; the aim of the turbofan is to compress the air to seek for a more efficient combustion. However, all of the compressible solvers are really difficult to use and the computational time is also higher given that we have a high number of control volumes (as discussed in the next section) due to a really complex geometry. Thus, although it will not be a completely real flow, we will be also able to see how the low pressure section of the engine increases the pressure of the flow and how the velocity field changes as it goes through the turbine.

The simulation will take into account that it is a tridimensional, that it behaves as a newtonian fluid and it will be run under stationary flow conditions (that is: the velocity in the inlet is always uniform and has a fixed value). With these hypothesis, we can begin to discuss the geometry of the blockMesh as well as the refined mesh and the boundary conditions and final results within the next sections.

3 Pre-processing

3.1 Mesh generation

3.1.1 Selection of the tutorial

The selection of the tutorial case is the first thing that needs to be done to run the simulation of the turbofan since all the files that will be used are linked and, starting a new case would mean spending a lot of time creating files and establishing all these links in order to make OpenFOAM work properly. Among all the tutorials that can be found on the OpenFOAM folder, we have selected the mixerVesselAMI2D for several reasons.

3.1.2 blockMesh

The definition of the *blockMeshDict* is the first part that needs to be modified. The *blockMesh* must contain the geometry that has to be simulated and we must have an idea of the vertices of the parallelogram that will contain the first stages of the compressor. In order to do that, the geometry has to be opened using either *Salome* or *Paraview*. Then the axes must be showed and the points have to be written down on the *blockMesh* file.

Now that the points enclosing the desired geometry are clear, they can be modified in the *blockMeshDict* file. This change is presented below:

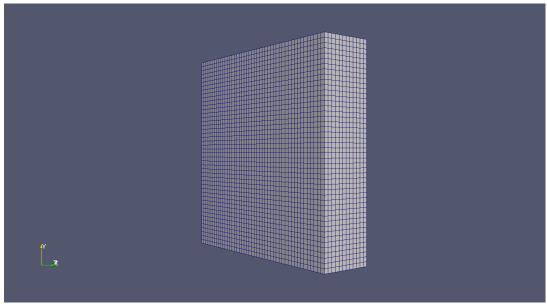


Figure 3.1: blockMeshcaption

It can be clearly seen that the domain of the mesh is a 0.58x2.3x2.3m rectangular prism.

Once the boundaries of the *blockMesh* are defined, the number of cells that it will have has to be set. It has to be taken into account that a very dense mesh will not be efficient when simulating the case given that we will use the *snappyHexMesh* later and it will be over densified. On the other hand, a very coarse mesh will not be efficient either because additional divisions will have to be set when generating the refined mesh and the computational time might grow. Thus, a solution between a mesh with a very high number of cells and a very low number of cells has to be attained.

This basic mesh has been divided every 0.05m. It means that we have done 12 divisions in the x direction, 46 on the y direction and 46 more on the z direction.

```
blocks
(
hex (0 1 2 3 4 5 6 7) (12 46 46) simpleGrading (1 1 1)
);
```

Finally, the differents faces of the mesh must be defined depending on if they are the inlet and outlet faces or the lateral faces. To do this, the vertices are numbered according to their appearance in the *blockMeshDict* and the faces are defined by those numbers. Since, the vertex numeration has been kept the same as the one that comes as default in every tutorial case; it is easier to define the inlet face of the *blockMesh* as well as the outlet face that will be used to define the boundary conditions.

```
boundary
    frontAndBack
    {
        type patch;
        faces
        (
             (3762)
             (1540)
             (0 \ 3 \ 2 \ 1)
             (4567)
        );
    }
    inlet
    {
        type patch;
        faces
        (
             (0 4 7 3)
        );
    }
    outlet
    {
        type patch;
        faces
             (2651)
        );
    }
);
```

The *blockMesh* file is presented below. This is the final file that has been used to generate the *blockMesh* and it is included in the *.zip* file attached to this report. Only the parameters mentioned above have been modified; the rest is equal to the tutorial case that has been selected.

```
-----*\
1 \\
       / F ield
                      | OpenFOAM: The Open Source CFD Toolbox
                      | Version: 4.0
          O peration
   \\ /
                      | Web:
                               www.OpenFOAM.org
          A nd
    \\/
          M anipulation |
\*-----*/
FoamFile
{
   version
            2.0;
   format
            ascii;
   class
            dictionary;
   object
            blockMeshDict;
}
convertToMeters 1;
vertices
(
   (0.77 0
           0)
   (1.35 0
           0)
   (1.35 2.3 0)
   (0.77 2.3 0)
   (0.770
           2.3)
   (1.35 0
           2.3)
   (1.35 2.3 2.3)
   (0.77 2.3 2.3)
);
blocks
   hex (0 1 2 3 4 5 6 7) (12 46 46) simpleGrading (1 1 1)
);
```

```
edges
(
);
boundary
    {\tt frontAndBack}
        type patch;
        faces
        (
            (3 7 6 2)
            (1 5 4 0)
            (0 3 2 1)
            (4 5 6 7)
        );
    }
    inlet
    {
        type patch;
        faces
        (
            (0 4 7 3)
        );
    }
    outlet
    {
        type patch;
        faces
        (
            (2 6 5 1)
        );
    }
);
```

The log obtained when the *blockMesh* has been generated is presented below. As it can be seen, no errors were found during the computation of this basic mesh. We can see that the number of cells is relatively high (**POSAR N. CELLS**) but perfectly suitable to proceed with the refinement of the mesh. Additionally, a caption of the basic mesh is presented in 3.1.

```
Writing polyMesh
_____
Mesh Information
_____
boundingBox:
             (0.77 0 0) (1.35 2.3 2.3)
nPoints:
          28717
nCells:
         25392
nFaces:
         79396
nInternalFaces:
                 72956
Patches
_____
patch 0 (start:
                 72956 size:
                               2208) name:
                                            frontAndBack
patch 1 (start:
                 75164 size:
                               2116) name:
                                            inlet
patch 2 (start:
                 77280 size:
                              2116) name:
                                            outlet
```

End

3.1.3 Mesh refinement

To refine the mesh, the *snappyHexMesh* utility is used (included in *OpenFOAM*) and several parameters have to be modified in order to obtain a dense mesh that is suitable for the simulation of this complex geometry. It has to be considered that a particular geometry is has a relative velocity as well; this is, the rotor is rotating, and so the first and second stages of the Low Pressure Compression of the turbofan engine, while the nacelle and the combustor are static.

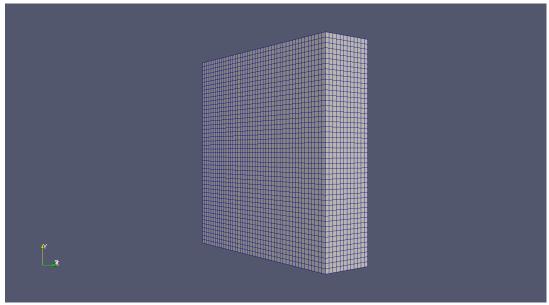


Figure 3.2: blockMeshcaption

In the snappyHexMeshDict several parameters have to be modified. First, the snappyHexMesh must be aware of the geometries that it has to take into account. As it can be seen below, the files Fan.stl, HPSpool.stl, LPSpool.stl and NacelleStator.stl have been included here.

```
geometry
{
    box1x1x1
    {
        type searchableBox;
        min (1.5 1 -0.5);
        max (3.5 2 0.5);
    }
    Fan.stl
    {
        type triSurfaceMesh;
        regions
        {
    }
    HPSpool.stl
    {
```

```
type triSurfaceMesh;
        regions
        {
        }
    }
    LPSpool.stl
        type triSurfaceMesh;
        regions
        {
        }
    }
    NacelleStator.stl
        type triSurfaceMesh;
        regions
        {
        }
    }
};
```

Next, the number of control volumes has to be limited to ensure that the laptop is capable of running a simulation. This number has been limited to two million cells, which is a pretty high number and the following lines have to be modified.

```
// Overall cell limit (approximately). Refinement will stop immediately
// upon reaching this number so a refinement level might not complete.
// Note that this is the number of cells before removing the part which
// is not 'visible' from the keepPoint. The final number of cells might
// actually be a lot less.
maxGlobalCells 2000000;
```

The next step is to define the refinement required of the mesh for the different geometries that will be simulated. It can be clearly seen in this section that we have included the same STL files that we did before. The higher the level of the refinement, the denser the mesh will be and the better it will resemble to the real geometry. But the limitation here is the computational power so, the maximum refinement number cannot be

as high as we would like to. Thus, depending on the complexity of the geometry, several minimum (the first number) and maximum (the second number) refinement leves have been defined.

```
// Surface based refinement
    // Specifies two levels for every surface. The first is the minimum level,
    // every cell intersecting a surface gets refined up to the minimum level.
    // The second level is the maximum level. Cells that 'see' multiple
    // intersections where the intersections make an
    // angle > resolveFeatureAngle get refined up to the maximum level.
    refinementSurfaces
       Fan.stl
           // Surface-wise min and max refinement level
           level (3 5);
           // Optional region-wise level specification
           regions
           {
           patchInfo
           {
               type patch;
               inGroups (meshedPatches);
           }
}
       HPSpool.stl
        {
           // Surface-wise min and max refinement level
           level (3 5);
           // Optional region-wise level specification
           regions
           {
           }
           patchInfo
           {
```

```
type patch;
                inGroups (meshedPatches);
            }
}
        LPSpool.stl
        {
            // Surface-wise min and max refinement level
            level (6 8);
    cellZone rotor; cellZoneInside inside;
            // Optional region-wise level specification
            regions
            {
            }
            patchInfo
                type patch;
                inGroups (meshedPatches);
            }
}
        NacelleStator.stl
            // Surface-wise min and max refinement level
            level (4 6);
            // Optional region-wise level specification
            regions
            {
            }
            patchInfo
            {
                type patch;
                inGroups (meshedPatches);
            }
}
    }
```

It is worth mentioning that a zone has been defined within the *LPSpool.stl* of the surface refinement from above. The MFR options must have a particular set of cellzones defined that will rotate. So, in order to make

these first stages of the compressor to rotate, this has to be defined.

```
LPSpool.stl
{
          // Surface-wise min and max refinement level
          level (6 8);
    cellZone rotor; cellZoneInside inside;
[...] %MODIFIED ^
```

The next step is to select a point within the mesh. So, the *locationInMesh* has to be modified with the x, y, and z-coordinates of a point with that feature.

```
// After refinement patches get added for all refinementSurfaces and
// all cells intersecting the surfaces get put into these patches. The
// section reachable from the locationInMesh is kept.
// NOTE: This point should never be on a face, always inside a cell, even
// after refinement.
locationInMesh (.97443222 1.40534444343 1.24221211);
```

Finally, the *surfaceFeatureExtract* has been used. What this option does it to refine even more the mesh near the points that have complex geometries such as the edges of the blades. This is particularly useful for the turbofan given that it has a high number of blades and a twisting geometry that will resemble more to the reality when using this option. Thus, the following lines within the *snappyHexMeshDict* must be modified.

All of the other parameters of the snappyHexMeshDict have not been modified.

3.1.4 Comparison of different types of mesh

A comparison between the coarse mesh and the refined mesh generated is presented below. The mesh that has been used to simulate the case is the refined mesh (for obvious reasons). Also, it has to be kept in mind that the coarser the mesh, the worse that the geometry will be (this is, it will not resemble to the real geometry), as can be seen below.

3.1.4.1 Coarse mesh

The number of control volumes in the mesh is 40, which is a very small number given the size of the turbofan. It can be seen in 3.2 and 3.3 that this mesh cannot be used to run a simulation.

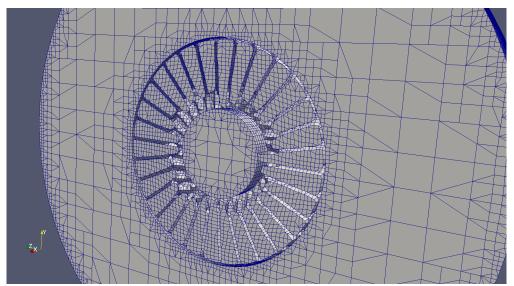


Figure 3.3: Detail of the coarse mesh

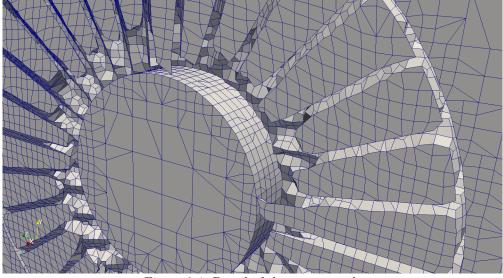


Figure 3.4: Detail of the coarse mesh

3.1.4.2 Standard mesh

Next, it is presented the standard mesh. It has a higher number of cells compared to the coarse mesh, but it cannot be used either given that the size of the turbine is pretty big. The figures 3.4 and 3.5 are a proof of that.

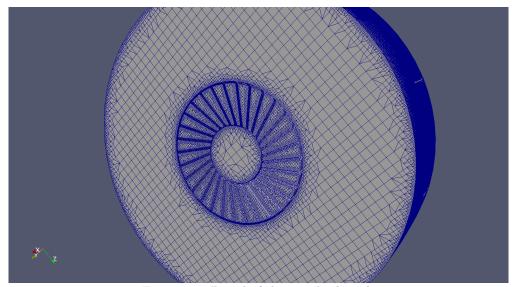


Figure 3.5: Detail of the standard mesh

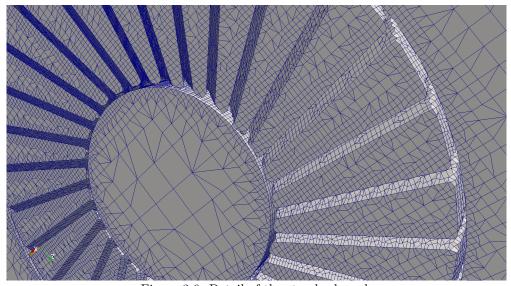


Figure 3.6: Detail of the standard mesh

3.1.4.3 Dense mesh

Finally, the dense mesh is presented. This is by far the best mesh that has been generated for the case. It can be clearly seen on 3.6 and 3.7 that it can be suitable to run the simulation of the first stages of this turbofan.

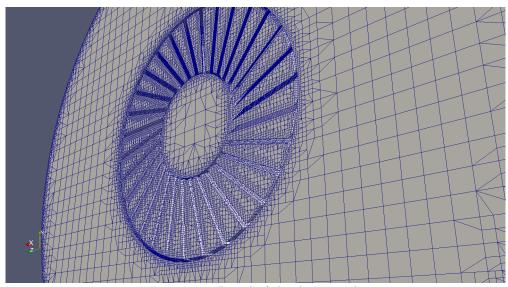


Figure 3.7: Detail of the dense mesh

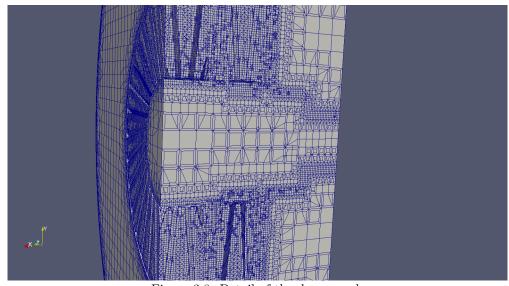


Figure 3.8: Detail of the dense mesh

To end with this section, the *log* file that has been obtained when the mesh has been defined with the *snappyHexMesh* utility is presented below. It has to be taken into account that it is a huge file since the number of iterations is really big and only the final part of the *log* file is shown.

```
Snapped mesh : cells:1626270 faces:5479413 points:2344680
Cells per refinement level:
    0 8592
    1 24456
    2 126790
    3 918976
    4 299109
    5 118950
    6 129397
Writing mesh to time constant
Wrote mesh in = 97.48 \text{ s.}
Mesh snapped in = 1121.77 s.
Checking final mesh ...
Checking faces in error :
    non-orthogonality > 65 degrees
                                                             : 0
    faces with face pyramid volume < 1e-13
                                                             : 0
    faces with face-decomposition tet quality < 1e-15
    faces with concavity > 80 degrees
                                                             : 0
    faces with skewness > 4
                              (internal) or 20 (boundary) : 0
    faces with interpolation weights (0..1) < 0.05
                                                             : 0
    faces with volume ratio of neighbour cells < 0.01
                                                             : 0
    faces with face twist < 0.02
                                                             : 0
    faces on cells with determinant < 0.001
                                                             : 0
Finished meshing without any errors
Finished meshing in = 1705 \text{ s.}
End
```

3.2 Rotation

An important consideration for this project is that the rotor is spinning at a high speed and this condition has to be somehow communicated to *OpenFOAM*. To work with this condition the Multiple Frame Reference (MFR) method has to be used. The method itself is based on adding a 'source' to the momentum equation in a previously defined zone named 'rotor'. How to do that has been shown in the previous section.

The **constant/MRFProperties** file has to be modified with the appropriate parameters for the case. Since the axis of OpenFOAM are based on the right hand rule, the rotation axis must be defined as follows: **CAMBIAR PER ARA!** (1,0,0). From information that can be found only, it has been chosen $5000 \ rpm$ as the angular velocity of the rotor. It should be noted that the units used must be those of the International System, so the $5000 \ rpm$ must be converted to rad/s. Furthermore, an origin point has to be indicated in the axis of rotation; in this case, the origin point is as follows: $(0 \ 1.1675354 \ 1.15542633)$.

Finally, it has to be indicated wheter some geometries are rotating or not. Thus, the next parameter has to be modified:

```
nonRotatingPatches (
NacelleStator.stl
);
```

FoamFile

So, the file **constant/MRFProperties** for the current case is the following:

```
{
    version
                2.0;
    format
                ascii;
    class
                dictionary;
    location
                "constant";
    object
                MRFProperties;
}
MRF1
{
    cellZone
                rotor;
    active
                yes;
    // Fixed patches (by default they 'move' with the MRF zone)
    nonRotatingPatches (
NacelleStator.stl
);
    origin
               (0 1.1675354 1.15542633);
    axis
               (1 0 0);
    omega
              523.5987756;
}
```

It can clearly be seen that the modification of any parameter is pretty straigh-forward in the file since there is no possible confusion with the names.

3.3 Boundary conditions

Some initial conditions and values have to be defined in order to run the simulation. The initial velocity field and pressure are especially important since these inputs are the starting values of the simulation. So, it is clear that the values at the *inlet*, which was defined within the previous sections, have to be defined. Additionally, the values at the *outlet* of the geometry have to be defined as well, but the can be, unlike the *inlet*, not fixed values. Thus, the following modifications have to be made in the $\theta.orig/U$.

```
internalField
                 uniform (0 0 0);
boundaryField
{
[...]
    inlet
                          fixedValue;
        type
value uniform (30 0 0);
    }
    outlet
    {
                         zeroGradient;
        type
    }
[...]
```

These conditions for the pressure have to be modified as well in the $\theta.orig/p$ file.

```
internalField uniform 0;
boundaryField
{
[...]
   inlet
   {
     type      fixedValue;
   value uniform 0;
```

```
outlet
{
    type zeroGradient;
}
```

Given that we have more solids that will be simulated (the Nacelle and the rotor), their conditions for the velocity and the pressure have to be defined as well. Since we have a newtonian flow, the condition for the velocity in these solids will be the noSlip condition. For the pressure, the condition will be the zeroGradient condition.

Finally, both files are presented below. These are the files that have been used to run the simulation for this case.

Velocity

```
/ F ield
                       | OpenFOAM: The Open Source CFD Toolbox
          O peration
                       | Version: 4.0
                                www.OpenFOAM.org
   \\ /
          A nd
                       | Web:
    \\/
          M anipulation |
\*-----
FoamFile
{
   version
             2.0;
   format
             ascii;
   class
             volVectorField;
   object
             U;
             [0 1 -1 0 0 0 0];
dimensions
            uniform (0 0 0);
internalField
```

```
{\tt boundaryField}
{
 LPSpool.stl
   type
      noSlip;
 NacelleStator.stl
   type noSlip;
 }
 inlet
      fixedValue;
   type
value uniform (30 0 0);
 }
 outlet
 {
      zeroGradient;
   type
 }
}
Pressure
/*----*\
| =======
FoamFile
 version 2.0;
 format
      ascii;
```

```
class volScalarField;
object p;
}
dimensions [0 2 -2 0 0 0 0];
internalField uniform 0;
boundaryField
  LPSpool.stl
  {
        zeroGradient;
   type
  }
  NacelleStator.stl
   type zeroGradient;
  }
  inlet
   type fixedValue;
  value uniform 0;
  }
  outlet
    type
         zeroGradient;
  }
}
```

3.4 Properties of the flow

The properties of the flow have to be defined taking into account the hypotheses made for the case. Given that the flow is air, the kinematic viscosity $nu(\nu)$ is $1.5e^{-5}m^2/s$. Also, as mentioned previously, the simulation will work with a Newtonian flow and this has to be defined here as well.

To change all these flow parameters and properties, the *constant/transportProperties* file has to be modified as follows. There is no need to dedicate more time to this file since everything is really clear within it.

```
1
         F ield
                      | OpenFOAM: The Open Source CFD Toolbox
          O peration
                      | Version: 4.0
   \\ /
          A nd
                      | Web:
                               www.OpenFOAM.org
    \\/
          M anipulation |
FoamFile
{
   version
            2.0;
   format
            ascii;
   class
            dictionary;
   object
            transportProperties;
}
transportModel Newtonian;
            [0 2 -1 0 0 0 0] 1.5e-05;
nu
```

Another hypotheses that has been made in order to alleviate the computation time is that the flow is laminar, although it would not be a very realistic flow in this case. To impose this condition, a modification has to be made in *constant/turbulenceProperties* file.

```
/*----*\
\*-----*/
FoamFile
{
 version 2.0;
 format ascii;
class dictionary;
object turbulenceProperties;
}
simulationType laminar;
RAS
{
        kOmegaSST;
 RASModel
 turbulence
       on;
 printCoeffs
        on;
}
```

Since we are only interested in the topology of the flow of this file, the other parameters have not been modified.

3.5 ControlDict

This file is really important for the simulation since it contains the timesteps that it will pursue. So, to begin, the initial and final times of the simulation have to be defined. In this case, the startTime is 0s and the endTime has been set to 20 s. No more simulation time is needed since it has been assumed that the flow is stationary. These changes can be made in the system/controlDict file.

NOTA: DEPEN DEL SOLVER! Si no fem servir SIMPLEFOAM s'he de tornar a COPIAR!

```
application simpleFoam;
startFrom latestTime;
startTime 0;
stopAt endTime;
endTime 20;
```

The next thing that has to be done is to define the time step for the simulation as well as the write interval (this is; the number of time that must pass in order to make a file for the simulation in the current simulation time).

```
deltaT 1;
writeControl timeStep;
writeInterval 5;
purgeWrite 0;
writeFormat binary;
writePrecision 6;
```

```
writeCompression uncompressed;
timeFormat general;
timePrecision 6;
```

Since it is really clear what the other variables do, they will not be explained here. Finally, the file that has been used to run the simulation is presented below.

```
-----*\
/ O peration | Version: 4.0
/ A nd | Web: www.OpenFOAM.org
| \\ / A nd
  \\/ M anipulation |
FoamFile
{
  version 2.0;
  format
         ascii;
  class
         dictionary;
  object
          controlDict;
application
         simpleFoam;
startFrom
         latestTime;
startTime
          0;
stopAt
         endTime;
endTime
          20;
deltaT
          1;
```

```
writeControl
            timeStep;
writeInterval
            5;
purgeWrite
            0;
writeFormat
            binary;
writePrecision 6;
writeCompression uncompressed;
timeFormat
            general;
{\tt timePrecision}
             6;
runTimeModifiable true;
/*functions
{
   #include "streamLines"
   #include "wallBoundedStreamLines"
   #include "cuttingPlane"
   #include "forceCoeffs"
}
```

3.6 fvSchemes and fvSolution

The fvSchemes dictionary contains the numerical schemes that they are used in the simulation. In the hypotheses section we have considered that the flow is stationary. To impose this condition we must modify the **system/fvSchemes** file, the section of *ddtSchemes*, which is the temporal integration scheme. We must write steadyState. Finally, the file that has been used to run the simulation is presented below.

```
-----*\
                          | OpenFOAM: The Open Source CFD Toolbox
           F ield
           O peration
                          | Version: 4.0
            A nd
                          | Web:
                                    www.OpenFOAM.org
    \\/
           M anipulation |
FoamFile
{
              2.0;
   version
   format
              ascii;
   class
              dictionary;
   object
              fvSchemes;
ddtSchemes
{
                  steadyState;
   default
}
gradSchemes
{
   default
                  Gauss linear;
   grad(U)
                  cellLimited Gauss linear 1;
}
divSchemes
{
   default
                  none;
   div(phi,U)
                  bounded Gauss linearUpwindV grad(U);
```

```
div(phi,k)
               bounded Gauss upwind;
   div(phi,omega) bounded Gauss upwind;
   div((nuEff*dev2(T(grad(U))))) Gauss linear;
}
laplacianSchemes
   default
               Gauss linear corrected;
}
interpolationSchemes
{
   default
               linear;
}
snGradSchemes
{
   default
              corrected;
}
wallDist
   method meshWave;
}
```

The fvSolution contains the resolution methods and the tolerances for each equation, but in our case it has not been modified. So, the file fvSolution for the current case is the following:

```
FoamFile
{
    version
                2.0;
    format
                ascii;
    class
                dictionary;
    object
                fvSolution;
}
solvers
{
    р
    {
                          GAMG;
        solver
        tolerance
                          1e-7;
        relTol
                          0.01;
        smoother
                          GaussSeidel;
    }
    Phi
    {
        $p;
    }
    U
    {
        solver
                          smoothSolver;
        smoother
                          GaussSeidel;
        tolerance
                          1e-8;
        relTol
                          0.1;
                          1;
        nSweeps
    }
    k
    {
                          smoothSolver;
        solver
        smoother
                          GaussSeidel;
        tolerance
                          1e-8;
        relTol
                          0.1;
        nSweeps
                          1;
    }
```

```
omega
    {
         solver
                           smoothSolver;
         smoother
                           GaussSeidel;
        tolerance
                           1e-8;
        relTol
                           0.1;
        nSweeps
                           1;
    }
}
SIMPLE
{
    nNonOrthogonalCorrectors 0;
    consistent yes;
}
potentialFlow
{
    nNonOrthogonalCorrectors 10;
}
{\tt relaxation} {\tt Factors}
{
    equations
        U
                          0.9;
        k
                          0.7;
        omega
                          0.7;
    }
}
cache
{
    grad(U);
}
```

4 Simulation of the turbofan

4.1 Selection of the solver

This solver used has been designed to work with non compressible flows, specifically. It solves the Navier Stokes equations for a given incompressible flow in steady-state conditions.

$$\nabla \vec{u} = 0 \tag{4.1}$$

$$\frac{\partial \vec{u}}{\partial t} + \vec{u}(\nabla \vec{u}) = \nu \nabla^2 \vec{u} - \nabla p + \vec{g}$$
(4.2)

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4.2 Commands to run the simulation

The first thing that must be done when the *blockMesh* has been modified is to type the following lines in a terminal in the folder where the case is located. As it can be seen, it will save the log on a new file named blockMesh.log.

blockMesh >blockMesh.log

When the *blockMesh* has been generated, the next thing to do is the refinement of the mesh. So, the *snappyHexMesh* utility will be used, provided that all the parameters in *snappyHexMeshDict* have been modified. In the terminal, the following command has to be typed:

foamJob decomposePar
foamJob surfaceFeatureExtract
foamJob -s -p snappyHexMesh -overwrite

The previous command has several features that will make the mesh refinement to run in parallel (notice the decomposePar and the -p). Additionally, a log will be created when running the last command that will contain all the information about the mesh refinement.

Next, the simulation can be run. To do that, the name of the solver as well as several other parameters have to be typed.

5 Post-process

5.1 Analysis of the resultsNO SE QUE ESCRIBO...TENGO SUEÑO

5.2 Conclusions

NO SE QUE ESCRIBO...TENGO SUEÑO