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## Title

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### Abstract

The Hybrid High Order (HHO) method is a powerful discretization method which has only been recently applied to non linear computational mechanics.

The Hybrid High Order method divides the domain of interest in cells of arbitrary polyhedral shape, whose boundaries form the skeleton of the mesh, and introduces two kinds of degrees of freedom: the displacements in cells and the displacements of the skeleton.

Most introductory materials to the HHO method is focused on mathematical aspects. While they are important, an approach based on physical considerations would help spreading this method to the computational mechanics and engineering communities.

This paper derives Hybrid High Order method from the classical Hu–Washizu functional.

Practical implementation of the method is discussed in depth using notations closed to the ones used in standard finite elements textbooks, highlighting the use of polyhedral cells and the use of approximation spaces based on polynomials of arbitrary orders.

From the point of view of numerical performances, the elimination of the cell degrees of freedom is mandatory to reduce the size of the stiffness matrix. The standard static condensation, is presented, as well as a novel strategy called "cell equilibrium". Advantages and disadvantages of both strategies are discussed.

The resolution of axi-symmetrical problems, which has seldom, if ever, been discussed in the literature, is then presented.

Numerical examples prove the robustness of the method with regards to volumetric locking

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*Keywords:*

Computational mechanics, Hybrid High Order method, Static condensation, Cell equilibrium algorithm, Volumetric locking, Axi-symmetric modelling hypothesis

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## 1. Introduction

The Hybrid High Order method (HHO) is a discontinuous discretization method, that takes root in the Discontinuous Galerkin method (DG). From the physical standpoint, DG methods ensure the continuity of the flux across interfaces, by seeking the solution element-wise, hence allowing jumps of the potential across elements. They can be seen as a generalization of Finite Volume methods, and are able to capture physically relevant discontinuities without producing spurious oscillations. The origin of DG methods dates back to the pioneering work of [1], where an hyperbolic formulation is used to solve the neutron transport equation. The first application of the method to elliptic problems originates in [2] where Nitsche's method [3] is used to weakly impose continuity of the flux across interfaces. In 2002, Hansbo and Larson [4] were the first to consider the Nitsche's classical DG method for nearly incompressible elasticity. They showed, theoretically and numerically, that this method is free from volumetric locking. However, the bilinear form arising from this formulation is not symmetric. A so called interior penalty term has been introduced in [5], leading to the Symmetric Interior Penalty (SIP) DG method. A first study of the method to linear elasticity has been devised by [6], where optimal error estimate has been proved. [7] generalized the Symmetric Interior Penalty method to linear elasticity. In about the same period of time, DG methods were proposed for other linear problems in solid mechanics, such as Timoshenko beams [8], Bernoulli-Euler beam and the Poisson-Kirchhoff plate [9, 10] and Reissner-Mindlin plates [11]. In the mid 2000's, the first applications of DG methods to nonlinear elasticity problems

was undertaken by [12, 13], and in 2007, Ortner and Süli [14] carried out the a priori error analysis of DG methods for nonlinear elasticity. DG methods then solicited a vigorous interest, mostly in fluid dynamics [15, 16] due to their local conservative property and stability in convection dominated problems. However, except some applications for instance in fracture mechanics using XFEM methods [17, 18], or gradient plasticity [19, 20] DG methods did not break through in computational solid mechanics because of their numerical cost, since nodal unknowns need be duplicated to define local basis functions in each element. To address this problem, in the early 2010's, [21, 22] introduced additional faces unknowns on element interfaces for linear elastic problem, hence leading to the hybridization of DG methods, or Hybridizable Discontinuous Galerkin method (HDG). By adding supplementary boundary unknowns, the authors actually allowed to eliminate original cell unknowns by a static condensation process, in order to express the global problem on faces ones only. Extension of HDG methods to non-linear elasticity were first undertaken in [23] and have then fueled intense research works for various applications such as linear and non-linear convection-diffusion problems [24, 25, 26], incompressible Stokes flows [26, 27] and non-linear mechanics [28]. In [29, 30], the authors introduced a higher order potential reconstruction operator in the classical HDG formulation for elliptic problems, providing a  $h^{k+1}H^1$ -norm convergence rate as compared to the usual  $h^k$ -rate. This higher order term coined the name for the so called HHO method. Recent developments of HHO methods in computational mechanics include the incompressible Stokes equations (with possibly large irrotational forces) [31], the incompressible Navier–Stokes equations [32], Biot's consolidation problem [33], and nonlinear elasticity with small deformations [34].

## 2. The model problem

### 2.1. The standard Hu–Washizu Lagrangian

This paragraph introduces the standard Hu–Washizu three field principle. For the sake of simplicity, and without loss of generality, let consider the case of an hyperelastic material. Extensions to mechanical behaviours with internal state variables are treated in classical textbooks of computational mechanics. We will treat this extension in the Section 5 discussing the numerical implementation of the Hybrid High Order method and in Section 6 which provides several examples in plasticity.

#### 2.1.1. Description of the mechanical problem and notations

*Solid body.* Let us consider a solid body whose reference configuration is denoted  $\Omega$ . At a given time  $t > 0$ , the body is in the current configuration  $\Omega_t$ .

*Mechanical loading.* The body is assumed to be submitted to a body force  $f_v$  acting in  $\Omega_t$ , a prescribed displacement  $\mathbf{u}_d$  on the Dirichlet boundary  $\partial_d\Omega_t$ , and a contact load  $\mathbf{t}_n$  on the Neumann boundary  $\partial_n\Omega_t$ .

*Deformation.* The transformation mapping  $\Phi$  takes a point from the reference configuration  $\Omega$  to the current configuration  $\Omega_t$  such that

$$\Phi(\mathbf{X}) = \mathbf{x} = \mathbf{X} + \mathbf{u}(\mathbf{X}) \quad (1)$$

where  $\mathbf{X}$ ,  $\mathbf{x}$  and  $\mathbf{u}$  denote respectively the position in the reference configuration  $\Omega$ , the position in the current configuration  $\Omega_t$  and the displacement.

*Deformation gradient, gradient of the displacement.* The deformation gradient  $\mathbf{F}$  is defined as

$$\mathbf{F} = \nabla \Phi = \mathbf{I} + \mathbf{G} \quad (2)$$

where  $\nabla$  is the gradient operator in the reference configuration and

$$\mathbf{G} = \nabla \mathbf{u} \quad (3)$$

denotes the gradient of the displacement.

*Stress tensor.* The body is assumed made of an hyperelastic material described by a free energy  $\psi_\Omega$  which relates the deformation gradient  $\tilde{\mathbf{F}}$  and the first Piola-Kirchhoff stress tensor  $\tilde{\mathbf{P}}$  such that

$$\tilde{\mathbf{P}} = \frac{\partial \psi_\Omega}{\partial \tilde{\mathbf{F}}} \quad (4)$$

### 2.1.2. Primal problem and Principle of Virtual Work

*Total lagrangian.* The total Lagrangian  $L_\Omega^{VW}$  of the body is defined as the stored energy minus the work of external loadings, as follows:

$$L_\Omega^{VW} = \int_{\Omega} \psi_\Omega(\tilde{\mathbf{F}}(\mathbf{u})) - \int_{\Omega} \mathbf{f}_V \cdot \mathbf{u} - \int_{\partial_N \Omega} \mathbf{t}_N \cdot \mathbf{u}|_{\partial_N \Omega} \quad (5)$$

where the body forces  $\mathbf{f}_V$  and conctat tractions  $\mathbf{t}_N$  in the reference configuration have been obtained from their counterparts  $\mathbf{f}_v$  and  $\mathbf{t}_n$  using the Nanson formulae.

*Principle of Virtual Works.* The displacement  $\mathbf{u}$  satisfying the mechanical equilibrium minimizes the Lagragian  $L_\Omega^{VW}$ . The first order variation of Lagrangian is given by:

$$\frac{\partial L_\Omega^{VW}}{\partial \mathbf{u}} \cdot \delta \mathbf{u} = \int_{\Omega} \tilde{\mathbf{P}} : \nabla \delta \mathbf{u} - \int_{\Omega} \mathbf{f}_V \cdot \delta \mathbf{u} - \int_{\partial_N \Omega} \mathbf{t}_N \cdot \delta \mathbf{u}|_{\partial_N \Omega} \quad (6)$$

which must be null for the the solution displacement. The solution displacement thus satisfies the principle of virtual work:

$$\int_{\Omega} \tilde{\mathbf{P}} : \nabla \delta \mathbf{u} = \int_{\Omega} \mathbf{f}_V \cdot \delta \mathbf{u} + \int_{\partial_N \Omega} \mathbf{t}_N \cdot \delta \mathbf{u}|_{\partial_N \Omega} \quad \forall \delta \mathbf{u}$$

### 2.1.3. The Hu-Washizu Lagrangian

The Hu-Washizu Lagrangian  $L_\Omega^{HW}$  [35, 36] generalizes the previous variational principle by considering that the gradient of the displacement  $\tilde{\mathbf{G}}$  and the first Piola-Kirchoff  $\tilde{\mathbf{P}}$  stress are independent unknowns of the problem, such that:

$$L_\Omega^{HW} = \int_{\Omega} \psi_\Omega(\tilde{\mathbf{I}} + \tilde{\mathbf{G}}) + (\nabla \mathbf{u} - \tilde{\mathbf{G}}) : \tilde{\mathbf{P}} - \int_{\Omega} \mathbf{f}_V \cdot \mathbf{u} - \int_{\partial_N \Omega} \mathbf{t}_N \cdot \mathbf{u}|_{\partial_N \Omega} \quad (7)$$

The solution  $(\mathbf{u}, \tilde{\mathbf{G}}, \tilde{\mathbf{P}})$  satisfying the mechanical equilibrium minimizes the Lagragian  $L_\Omega^{HW}$ . The first order variation of the Hu-Washizu Lagragian with respect to  $\mathbf{u}$ ,  $\tilde{\mathbf{G}}$ , and  $\tilde{\mathbf{P}}$  yields

$$\frac{\partial L_\Omega^{HW}}{\partial \mathbf{u}} \cdot \delta \mathbf{u} = \int_{\Omega} \tilde{\mathbf{P}} : \nabla \delta \mathbf{u} - \int_{\Omega} \mathbf{f}_V \cdot \delta \mathbf{u} - \int_{\partial_N \Omega} \mathbf{t}_N \cdot \delta \mathbf{u}|_{\partial_N \Omega} \quad \forall \delta \mathbf{u} \quad (8a)$$

$$\frac{\partial L_\Omega^{HW}}{\partial \tilde{\mathbf{P}}} : \delta \tilde{\mathbf{P}} = \int_{\Omega} (\nabla \mathbf{u} - \tilde{\mathbf{G}}) : \delta \tilde{\mathbf{P}} \quad \forall \delta \tilde{\mathbf{P}} \quad (8b)$$

$$\frac{\partial L_\Omega^{HW}}{\partial \tilde{\mathbf{G}}} : \delta \tilde{\mathbf{G}} = \int_{\Omega} \left( \frac{\partial \psi}{\partial \tilde{\mathbf{G}}} - \tilde{\mathbf{P}} \right) : \delta \tilde{\mathbf{G}} \quad \forall \delta \tilde{\mathbf{G}} \quad (8c)$$

where equation (8b) and (8c) account for (3) and (4) respectively in a weak sense.

## 2.2. On the use of the Hu-Washizu Lagrangian in mechanics to circumvent volumetric locking

In the continuous framework, the Hu-Washizu functional is not relevant, since equations (8b) and (8c) would hold true in a strong sense.

*Pressure swelling formulations.* Since volumetric locking is pressure dependent phenomenon, considering for instance a decomposition of the stress and strain fields into *e.g.* deviatoric and spherical components, one can express a mixed problem in terms of pressure and swelling, which is at the origin of so-called UPG methods [37, 38, 39]. The scalar pressure and swelling unknowns replace respectively the stress and strain tensorial unknowns in (7) and a modified deformation gradient is introduced in the constitutive equation.

*Enhanced assumed strains formulations.* Another approach of the use of the Hu-Washizu consists in studying the equilibrium of a single element. Such a framework falls into the scope of so-called Enhanced Assumed Strains methods [40, 41], which results for instance in the B-bar method, that consists in defining a modified derivation operator, such that one gets rid of the three-field formulation, to express the problem in terms of primal unknowns only.

*Towards Discontinuous methods.* In the present document, we propose an introduction to so-called *non-conformal* methods, by means of the Hu-Washizu Lagrangian. At the origin of these methods is the Discontinuous Galerkin (DG) method, which postulates the discontinuity of the displacement across elements. This feature allows the method to be robust to volumetric locking. However, its formulation takes root in a possibly dry mathematical background, and the ingredients of the method are not introduced in the literature through physical arguments. The goal of the next section aims at introducing the whole framework of non-conformal methods, including the displacement discontinuity, through the Hu-Washizu Lagragian.

### 3. Introduction to discontinuous methods through a Hu-Washizu formulation

In this section let  $T$  a subpart of the body  $\Omega$ . In the following, one assumes that the cell  $T$  is located inside the body  $\Omega$ , such that its boundary  $\partial T$  bears contact loads only. This subpart is in equilibrium with the rest of the body  $\Omega \setminus T$  if the displacements and the normal traction are continuous at the boundary  $\partial T$ .

*Conformal methods.* Enforcing the displacement continuity at the interface leads to so-called conformal methods, to which the standard Finite Element (FE), or Lagrange method belongs (see Figure 1(a)).

*Discontinuous Galerkin methods.* On the contrary, this condition can be weakened by introducing an elastic interface of negligible size between  $T$  and  $\Omega \setminus T$ . This representation is at the basis of Discontinuous Galerkin methods (see Figure 1(b)).

*Hybird Discontinuous Galerkin methods.* In this paper, we consider hybrid discontinuous Galerkin (HDG) methods, where two elastic interfaces are introduced: one between  $T$  and its boundary  $\partial T$  and a second one between  $\Omega \setminus T$  and  $\partial T$  (see Figure 1(c)). Following this idea, we show in this section how the use of the Hu-Hashizu Lagrangian allows to recover the main ingredients of the HDG/HHO methods, namely the *reconstructed gradient* and the *stabilisation operator*.

#### 3.1. Element description

*Element geometry.* In the following, the cell  $T$  is assumed to be convex. It is split into a core part  $K \subset T$  with boundary  $\partial K$ , and into an interface part  $I \subset T$  with boundary  $\partial I = \partial K \cup \partial T$ , as shown in Figure 1. The interface  $I$  has some thickness  $\ell > 0$  that is supposed to be small compared to  $h_T$  the diameter of  $T$ . From a geometrical standpoint, the core par of the element  $K$  is an homotethy of  $T$  by some ratio inferior to 1.

*Element boundary description.* The boundary  $\partial T$  of  $T$  is the composition of a Neumann boundary  $\partial_N T$  and a Dirichlet  $\partial_D T$ , if the element  $T$  shares a boundary with  $\partial_D \Omega$ . In the following, for the sake of simplicity, we assume that the element is located inside the body  $\Omega$ , such that it is only subjected to imposed traction forces on  $\partial_N T = \partial T$  with  $\partial_D T = \emptyset$ .

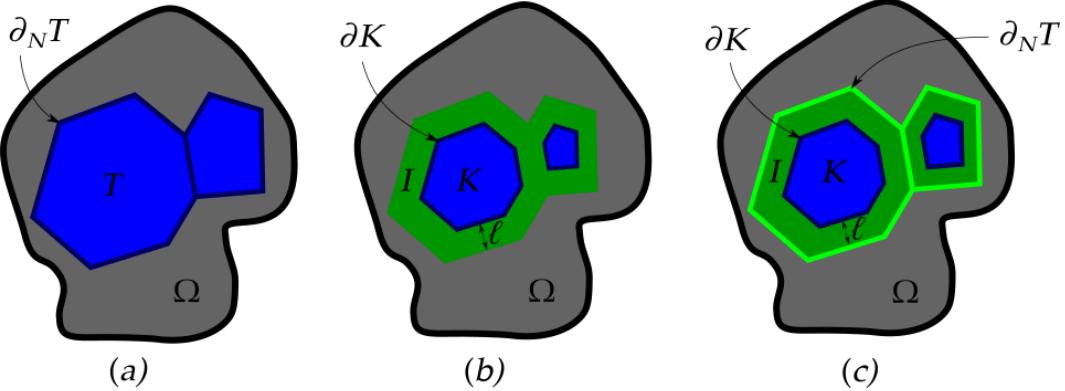


Figure 1. schematic representation of a cell and its surrounding depending on the continuity requirement of the displacement field

*Element behaviour.* The core of the element  $K$  is made out of the same material that composes  $\Omega$  and behaves according to the free energy potential  $\psi_\Omega$ . The interface  $I$  is made out of a pseudo linear elastic material of Young modulus  $\beta(\ell/h_T)$  with a zero Poisson ratio and its behavior is defined by the free energy potential  $\psi_I$  such that

$$\psi_I = \frac{1}{2} \beta \frac{\ell}{h_T} \nabla \mathbf{u}_I : \nabla \mathbf{u}_I \quad (9)$$

where the dimensionless ratio  $\ell/h_T$  balances the accumulated energy with the size of the domain  $T$ .

*Element loading.* The core  $K$  is subjected to the volumetric loading  $f_V$ , and to the traction force applied by the interface  $I$  onto  $\partial K$ . By continuity,  $K$  applies the opposite traction force on  $I$  through  $\partial K$ . The interface  $I$  is also subjected to the exterior traction force  $\mathbf{t}_{\partial_N T}$  acting on  $\partial_N T$ , that accounts for the action of the rest of the solid  $\Omega$  onto the boundary  $\partial T$ .

*Displacement, displacement gradient and stress fields.* Let note  $\mathbf{u}_K$  the displacement field,  $\mathbf{G}_K$  the displacement gradient field and  $\mathbf{P}_K$  the stress field in  $K$ . Similarly, let  $\mathbf{u}_I$  the displacement field,  $\mathbf{G}_I$  the displacement gradient field and  $\mathbf{P}_I$  the stress field in  $I$ . The displacement of the boundary  $\partial T$  is denoted  $\mathbf{u}_{\partial T}$ . By continuity of the displacement field between  $K$  and  $\partial T$ , the displacement  $\mathbf{u}_I$  verifies

$$\mathbf{u}_I|_{\partial K} = \mathbf{u}_K|_{\partial K} \quad (10a)$$

$$\mathbf{u}_I|_{\partial T} = \mathbf{u}_{\partial T} \quad (10b)$$

*Hu-Washizu Lagrangian of the element.* By combining both the Lagrangian of the core  $K$  and that of the interface  $I$ , one obtains the total Lagrangian  $L_T^{HW}$  over the element such that

$$L_T^{HW} = \int_K \psi_\Omega + (\nabla \mathbf{u}_K - \mathbf{G}_K) : \mathbf{P}_K + \int_I \psi_I + (\nabla \mathbf{u}_I - \mathbf{G}_I) : \mathbf{P}_I - \int_K f_V \cdot \mathbf{u}_K - \int_{\partial_N T} \mathbf{t}_{\partial_N T} \cdot \mathbf{u}_{\partial T} \quad (11)$$

### 3.2. Hypotheses

Since the interface is of negligible volume compared to that of the core, let make the following assumptions on the displacement and the stress fields in the interface.

*Displacement in the interface.* The displacement in the interface  $I$  is assumed to be linear with respect to  $\mathbf{n}$ , such that its gradient is homogeneous in  $I$  along  $\mathbf{n}$

$$\nabla \mathbf{u}_I = \frac{\mathbf{u}_{\partial T} - \mathbf{u}_K|_{\partial K}}{\ell} \otimes \mathbf{n} \quad (12)$$

That is, the displacement of the interface  $I$  linearly bridges that of the boundary  $\partial T$  to that of the bulk  $K$ .

*Stress in the interface.* Furthermore, let assume that  $\tilde{\mathbf{P}}_I$  is constant along the direction  $\mathbf{n}$  in  $I$ . By continuity of the traction force across  $\partial K$ , the following equality holds true

$$(\tilde{\mathbf{P}}_I|_{\partial K} - \tilde{\mathbf{P}}_K|_{\partial K}) \cdot \mathbf{n} = 0 \quad (13)$$

### 3.3. Towards Hybrid discontinuous methods from the Hu-Washizu Lagrangian

Using the hypotheses stated in Section 3.2 on the displacement field and the stress field in  $I$ , one can write (11) as a term depending on the thickness of the interface  $\ell$  and on the core and boundary unknowns only. The reader can refer to Appendix A.1 for more details.

*Simplified Hu–Washizu Lagrangian for a vanishing interface.* In particular, making the thickness of the interface  $\ell \rightarrow 0$ , such that  $I$  vanishes and the core part  $K$  identifies to  $T$ , one obtains the simplified Hu–Washizu Lagrangian

$$\begin{aligned} L_T^{HW} = & \int_T \psi_\Omega + (\nabla \mathbf{u}_T - \mathbf{G}_T) : \tilde{\mathbf{P}}_T + \int_{\partial T} (\mathbf{u}_{\partial T} - \mathbf{u}_T|_{\partial T}) \cdot \tilde{\mathbf{P}}_T|_{\partial T} \cdot \mathbf{n} + \int_{\partial T} \frac{\beta}{2h_T} \|\mathbf{u}_{\partial T} - \mathbf{u}_T|_{\partial T}\|^2 \\ & - \int_T \mathbf{f}_V \cdot \mathbf{u}_T - \int_{\partial_N T} \mathbf{t}_{\partial_N T} \cdot \mathbf{u}_{\partial T} \end{aligned} \quad (14)$$

which fully defines the equilibrium of an element for discontinuous methods.

*Hybridization of the primal unknown; the HDG and HHO methods.* Since the interface  $I$  has vanished by making  $\ell \rightarrow 0$ , both  $\mathbf{u}_T|_{\partial T}$  the trace of the displacement of the core part  $T$  onto  $\partial T$  and the displacement of the boundary  $\mathbf{u}_{\partial T}$  coexist on  $\partial T$ . The displacement of the element  $T$  is thus said to be *hybrid*, and is denoted by the pair  $(\mathbf{u}_T, \mathbf{u}_{\partial T})$ .

*The special case of DG methods.* Replacing  $\mathbf{u}_{\partial T}$  by  $\mathbf{u}_{T'}|_{\partial T}$  for any neighboring cell  $T'$  amounts to describe the framework of Discontinuous Galerkin methods, where only the core unknown  $\mathbf{u}_T$  is considered, and the displacement jump on  $\partial T$  depends on  $\mathbf{u}_{T'}|_{\partial T}$  the trace of the displacement of neighboring cells instead.

*Conformal Galerkin formulation.* By strongly enforcing continuity of the displacement across  $\partial T$  such that  $\mathbf{u}_T|_{\partial T} = \mathbf{u}_{\partial T}$ , one recovers the Principle of Virtual Work (7), which defines the framework of conformal methods.

*Mixed problem for HDG methods.* By derivation of the total Lagrangian (14) with respect to each variable of the problem, one obtains the weak equations

$$\frac{\partial L_T^{HW}}{\partial \mathbf{u}_T} \delta \mathbf{u}_T = \int_T \tilde{\mathbf{P}}_T : \nabla \delta \mathbf{u}_T - \int_T \mathbf{f}_V \cdot \delta \mathbf{u}_T - \int_{\partial T} \boldsymbol{\theta}_{\partial T} \cdot \delta \mathbf{u}_T|_{\partial T} \quad \forall \delta \mathbf{u}_T \quad (15a)$$

$$\frac{\partial L_T^{HW}}{\partial \mathbf{u}_{\partial T}} \delta \mathbf{u}_{\partial T} = \int_{\partial_N T} (\boldsymbol{\theta}_{\partial T} - \mathbf{t}_{\partial_N T}) \cdot \delta \mathbf{u}_{\partial T} \quad \forall \delta \mathbf{u}_{\partial T} \quad (15b)$$

$$\frac{\partial L_T^{HW}}{\partial \mathbf{G}_T} \delta \mathbf{G}_T = \int_T \left( \frac{\partial \psi_\Omega}{\partial \mathbf{G}_T} - \tilde{\mathbf{P}}_T \right) : \delta \mathbf{G}_T \quad \forall \delta \mathbf{G}_T \quad (15c)$$

$$\frac{\partial L_T^{HW}}{\partial \tilde{\mathbf{P}}_T} \delta \tilde{\mathbf{P}}_T = \int_T (\nabla \mathbf{u}_T - \mathbf{G}_T) : \delta \tilde{\mathbf{P}}_T + \int_{\partial T} (\mathbf{u}_{\partial T} - \mathbf{u}_T|_{\partial T}) \cdot \delta \tilde{\mathbf{P}}_T|_{\partial T} \cdot \mathbf{n} \quad \forall \delta \tilde{\mathbf{P}}_T \quad (15d)$$

where we introduced the *reconstructed traction force*  $\boldsymbol{\theta}_{\partial T} = \tilde{\mathbf{P}}_T|_{\partial T} \cdot \mathbf{n} + (\beta/h_T)(\mathbf{u}_{\partial T} - \mathbf{u}_T|_{\partial T})$ . In particular, (15a) is the expression of the Principle of Virtual Work in  $T$ , where the *reconstructed traction force*  $\boldsymbol{\theta}_{\partial T}$  replaces the usual expression  $\tilde{\mathbf{P}}_T \cdot \mathbf{n}$  in the external contribution. (15b) denotes a supplementary equation to the usual continuous problem as described in (8), to account for the continuity of the flux  $\boldsymbol{\theta}_{\partial T}$  across the cell boundary. (15c) accounts for the constitutive equation in a weak sense, and (15d) defines the equation of an enhanced gradient field, that does not reduce to the projection of  $\nabla \mathbf{u}_T$  as in (8c), since it is enriched by a boundary component that depends on the displacement jump. This feature is at the origin of the robustness of non-conformal methods to volumetric locking (see Appendix A.2 for more details on this note).

### 3.4. Problem in primal form

*Reconstructed gradient.* Since minimization of (15d) defines a linear problem with any displacement pair  $(\mathbf{v}_T, \mathbf{v}_{\partial T})$ , one can eliminate (15d) from the system (15). The resulting equation defines the so-called *reconstructed gradient*  $\mathbf{G}_T(\mathbf{v}_T, \mathbf{v}_{\partial T})$  associated with any displacement pair  $(\mathbf{v}_T, \mathbf{v}_{\partial T})$ , that solves

$$\int_T \mathbf{G}_T : \boldsymbol{\tau}_T = \int_T \nabla \mathbf{v}_T : \boldsymbol{\tau}_T + \int_{\partial T} (\mathbf{v}_{\partial T} - \mathbf{v}_T|_{\partial T}) \cdot \boldsymbol{\tau}_T|_{\partial T} \cdot \mathbf{n} \quad \forall \boldsymbol{\tau}_T \quad (16)$$

where  $\boldsymbol{\tau}_T$  denotes an arbitrary kinematically admissible stress field.

*Stress tensor.* Likewise, (15c) is eliminated from (15) since it is linear with  $\mathbf{G}_T$ . Assuming in addition that the space of kinematically admissible stress fields is included in that of kinematically admissible displacement gradient fields, (15c) holds in a strong sense such that

$$\mathbf{P}_T = \frac{\partial \psi_\Omega}{\partial \mathbf{G}_T} \quad (17)$$

*Total Lagrangian for the problem in primal form.* By elimination of both (15c) and (15d) from (15), the only remaining unknowns are the cell and boundary displacement fields. Hence, a new Lagrangian that accounts for the problem in primal form arises from (14).

$$L_T^{VW} = \int_T \psi_\Omega + \int_{\partial T} \frac{\beta}{2h_T} \|\mathbf{u}_{\partial T} - \mathbf{u}_T|_{\partial T}\|^2 - \int_T \mathbf{f}_V \cdot \mathbf{u}_T - \int_{\partial_N T} \mathbf{t}_{\partial_N T} \cdot \mathbf{u}_{\partial T} \quad (18)$$

*Principle of virtual works for HDG methods.* Minimization of the Lagrangian (18) amounts to the definition of a Principle of Virtual Work, in the context of HDG methods. It consists in finding the displacement pair  $(\mathbf{u}_T, \mathbf{u}_{\partial T})$  that solves

$$\delta L_{T,\text{int}}^{VW} - \delta L_{T,\text{ext}}^{VW} = 0 \quad (19)$$

where  $\forall (\delta \mathbf{u}_T, \delta \mathbf{u}_{\partial T})$

$$\delta L_{T,\text{int}}^{VW} = \int_T \mathbf{P}_T(\mathbf{G}_T(\mathbf{u}_T, \mathbf{u}_{\partial T})) : \mathbf{G}_T(\delta \mathbf{u}_T, \delta \mathbf{u}_{\partial T}) + \int_{\partial T} (\beta/h_T) \mathbf{Z}_{\partial T}(\mathbf{u}_T, \mathbf{u}_{\partial T}) \cdot \mathbf{Z}_{\partial T}(\delta \mathbf{u}_T, \delta \mathbf{u}_{\partial T}) \quad (20a)$$

$$\delta L_{T,\text{ext}}^{VW} = \int_{\partial_N T} \mathbf{t}_{\partial_N T} \cdot \delta \mathbf{u}_{\partial T} + \int_T \mathbf{f}_V \cdot \delta \mathbf{u}_T \quad (20b)$$

and where we introduced the jump function  $\mathbf{Z}_{\partial T}$  such that

$$\mathbf{Z}_{\partial T}(\mathbf{v}_T, \mathbf{v}_{\partial T}) = \mathbf{v}_{\partial T} - \mathbf{v}_T|_{\partial T} \quad \forall (\mathbf{v}_T, \mathbf{v}_{\partial T}) \quad (21)$$

In particular, one can readily see the resemblance of (20) with (6), where the so called *reconstructed gradient*  $\mathbf{G}_T(\mathbf{u}_T, \mathbf{u}_{\partial T})$  plays the role of the usual displacement Lagrangian gradient  $\nabla \mathbf{u}_T$ , and where an additional *stabilization term* corresponding to a traction energy on the boundary has been added to account for the penalization of the displacement jump on  $\partial T$  through  $\mathbf{Z}_{\partial T}$  (or, equivalently, to account for the infinitesimal interface that lays between the bulk domain and its boundary). Equations (18), (16) and (17) define the mechanical problem to solve at the cell level for HDG methods, and (19) describes the weak form of these equations.

### 3.5. Small strain hypothesis

The proposed large deformation formulation also allows a natural transition to the small deformation framework. In this context, since the gradient of the transformation  $\mathbf{F}_T$  is assumed to be small compared to  $\mathbf{1}$ , we seek the

infinitesimal deformation field  $\underline{\varepsilon}_T$  as the weak formulation of  $\nabla^s \mathbf{u}_T$  rather than the gradient of the displacement field  $\mathbf{G}_T$ . The stress tensor  $\underline{\sigma}_T$  is then identified with the Cauchy stress  $\underline{\sigma}_T$ , so that problem (14) becomes

$$\begin{aligned} L_T^{HW} = & \int_T \psi_\Omega + (\nabla^s \mathbf{u}_T - \underline{\varepsilon}_T) : \underline{\sigma}_T + \int_{\partial T} (\mathbf{u}_{\partial T} - \mathbf{u}_T|_{\partial T}) \cdot \underline{\sigma}_T|_{\partial T} \cdot \mathbf{n} + \int_{\partial T} \frac{\beta}{2h_T} \|\mathbf{u}_{\partial T} - \mathbf{u}_T|_{\partial T}\|^2 \\ & - \int_T \mathbf{f}_V \cdot \mathbf{u}_T - \int_{\partial_N T} \mathbf{t}_{\partial_N T} \cdot \mathbf{u}_{\partial T} \end{aligned} \quad (22)$$

Continuing the same development as above, the energy to be minimized is given by equation 18 as in the large deformation framework. On the other hand, the reconstructed gradient equation 16 becomes

$$\int_T \underline{\varepsilon}_T(\mathbf{v}_T, \mathbf{v}_{\partial T}) : \underline{\tau}_T = \int_T \nabla^s \mathbf{v}_T : \underline{\tau}_T + \int_{\partial T} (\mathbf{v}_{\partial T} - \mathbf{v}_T|_{\partial T}) \cdot \underline{\tau}_T|_{\partial T} \cdot \mathbf{n} \quad \forall \underline{\tau}_T \quad (23)$$

where  $\underline{\tau}_T$  denotes an arbitrary kinematically admissible symmetric stress field, since the deformations  $\underline{\varepsilon}_T$  and the Cauchy stress  $\underline{\sigma}_T$  are symmetric. The expression of the stress as a function of the cell deformation is then given by

$$\underline{\sigma}_T = \frac{\partial \psi_\Omega}{\partial \underline{\varepsilon}_T} \quad (24)$$

### 3.6. Extension to the axi-symmetric framework

In the following section, we devise a Hybrid High order method for an axi-symmetric framework. Owing to geometrical assumptions on the displacement and its gradient, the definition of the reconstructed gradient (16) and of that of the higher order displacement (27) needs be modified accordingly. Details about the definitions of these ingredients can be found in Appendix A.3.

*Axi-symmetric framework.* The cartesian space is expressed in cylindrical coordinates and a point  $X \in \Omega$  has coordinates  $X = (r, z, \theta)$  where  $r$  denotes the radial component,  $z$  the ordinate one, and  $\theta$  is the angular component describing a revolution around the axis  $r = 0$ . By cylindrical symmetry, the angular displacement  $u_\theta$  is supposed to be zero, and both components  $u_r$  and  $u_z$  do not depend on the angular coordinate  $\theta$ .

*Axis faces treatment.* Since in cylindrical coordinates, all integrals depend on the radial component  $r$ , boundary integrals vanish at  $r = 0$  on the symmetry axis. Therefore, the reconstructed gradient (and the stabilization) do not depend on a closed surface wrapping a cell  $T$  located on the symmetry axis. However, this feature is necessary to prove the robustness of the HHO method to volumetric locking (see Appendix A.2). Therefore, in order to restore full mobility of a face located on the symmetry axis, we consider infinitely thin cylindrical faces wrapping it, that are subjected to Dirichlet boundary conditions along the radial direction.

## 4. Discretization

In this section, approximation spaces for unknowns of the global problem are described, which leads to several choices in terms of definition of the stabilization. Depending on such a choice, one recovers either the HDG method, or the HHO one. In a second part, linearization strategies for non-linear problems are discussed, and a new resolution scheme based on the equilibrium of a cell with its boundary is introduced. Finally, the problem to solve at the structural level is presented through the introduction of the so called *skeleton* of the mesh, that bears boundary unknowns.

### 4.1. Functional discretization and stabilization

*Discrete functional space.* For a cell  $T$ , we denote  $U^h(T)$  the approximation displacement space in the cell, and  $V^h(\partial T)$  that on the boundary. Similarly, let  $G^h(T)$  the space used to build the discrete reconstructed gradient and  $S^h(T)$  that chosen for the discrete stress such that

$$\begin{aligned} U^h(T) &= P^l(T, \mathbb{R}^d) \\ V^h(\partial T) &= P^k(\partial T, \mathbb{R}^d) \\ G^h(T) &= P^k(T, \mathbb{R}^{d \times d}) \\ S^h(T) &= P^k(T, \mathbb{R}^{d \times d}) \end{aligned}$$

where the cell displacement polynomial order  $l$  might be chosen different from the face displacement order  $k$  such that  $k - 1 \leq l \leq k + 1$ .

*HDG stabilization.* Accounting for the possible different polynomial order between the cell and faces, one can specify a discrete jump function in a natural way such that it delivers the displacement difference point-wise for any displacement pair  $(\mathbf{v}_T^l, \mathbf{v}_{\partial T}^k) \in U^h(\bar{T})$

$$\mathbf{Z}_{\partial T}^{HDG}(\mathbf{v}_T^l, \mathbf{v}_{\partial T}^k) = \Pi_{\partial T}^k(\mathbf{v}_{\partial T}^k - \mathbf{v}_T^l|_{\partial T}) \quad (25)$$

where  $U^h(\bar{T}) = U^h(T) \times V^h(\partial T)$  and  $\Pi_{\partial T}^k$  denotes the orthogonal projector onto  $V^h(\partial T)$ . This straightforward discrete jump function is at the origin of Hybrid Discontinuous Galerkin methods, and grants a convergence of order  $k$  in the energy norm.

*HHO stabilization.* A richer discrete jump function  $\mathbf{Z}_{\partial T}^{HHO}$  providing a convergence of order  $k + 1$  in the energy norm was introduced in [42], hence giving the Hybrid High Order method its name, such that

$$\mathbf{Z}_{\partial T}^{HHO}(\mathbf{v}_T^l, \mathbf{v}_{\partial T}^k) = \Pi_{\partial T}^k(\mathbf{v}_{\partial T}^k - \mathbf{v}_T^l|_{\partial T} - ((I_T^{k+1} - \Pi_T^k)(\mathbf{w}_T^{k+1}))|_{\partial T}) \quad (26)$$

where  $\Pi_T^k$  is the projector onto  $P^k(T, \mathbb{R}^d)$ ,  $I_T^{k+1}$  is the identity function in  $D^h(T) = P^{k+1}(T, \mathbb{R}^d)$ .

*Reconstructed higher order displacement.* The term  $\mathbf{w}_T^{k+1}$  in (26) denotes a higher order discrete displacement in  $D^h(T)$  that solves for any displacement pair  $(\mathbf{v}_T^l, \mathbf{v}_{\partial T}^k) \in U^h(\bar{T})$

$$\int_T \nabla \mathbf{w}_T^{k+1} : \nabla \mathbf{d}_T^{k+1} = \int_T \nabla \mathbf{v}_T^l : \nabla \mathbf{d}_T^{k+1} + \int_{\partial T} (\mathbf{v}_{\partial T}^k - \mathbf{v}_T^l) \cdot \nabla \mathbf{d}_T^{k+1} \cdot \mathbf{n} \quad \forall \mathbf{d}_T^{k+1} \in D^h(T) \quad (27a)$$

$$\int_T \mathbf{w}_T^{k+1} = \int_T \mathbf{v}_T^l \quad (27b)$$

*Local discrete problem.* Following discretization of cell and faces unknowns, the local discrete problem to solve reads : find  $(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k)$ , such that  $\forall (\delta \mathbf{u}_T^l, \delta \mathbf{u}_{\partial T}^k)$

$$\delta L_{T,\text{int}}^{HHO} - \delta L_{T,\text{ext}}^{HHO} = 0 \quad (28)$$

with the respective Lagrangian variations

$$\begin{aligned} \delta L_{T,\text{int}}^{HHO} &= \int_T \mathbf{P}_T^k(\mathbf{G}_T^k(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k)) : \mathbf{G}_T^k(\delta \mathbf{u}_T^l, \delta \mathbf{u}_{\partial T}^k) + \int_{\partial T} (\beta/h_T) \mathbf{Z}_{\partial T}^{HHO}(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k) \cdot \mathbf{Z}_{\partial T}^{HHO}(\delta \mathbf{u}_T^l, \delta \mathbf{u}_{\partial T}^k) \\ \delta L_{T,\text{ext}}^{HHO} &= \int_{\partial T} \mathbf{t}_{\partial_N T} \cdot \delta \mathbf{u}_F^k + \int_T \mathbf{f}_V \cdot \delta \mathbf{u}_T^l \end{aligned} \quad (29)$$

and where the discrete reconstructed gradient  $\mathbf{G}_T^k(\mathbf{v}_T^l, \mathbf{v}_{\partial T}^k) \in G^h(T)$  solves  $\forall (\mathbf{v}_T^l, \mathbf{v}_{\partial T}^k) \in U^h(\bar{T})$

$$\int_T \mathbf{G}_T^k(\mathbf{v}_T^l, \mathbf{v}_{\partial T}^k) : \boldsymbol{\tau}_T^k = \int_T \nabla \mathbf{v}_T^l : \boldsymbol{\tau}_T^k + \int_{\partial T} (\mathbf{v}_{\partial T}^k - \mathbf{v}_T^l|_{\partial T}) \cdot \boldsymbol{\tau}_T^k|_{\partial T} \cdot \mathbf{n} \quad \forall \boldsymbol{\tau}_T^k \in S^h(T) \quad (30)$$

*Shape functions.* Since the displacement is discontinuous, the usual Lagrange basis functions are not necessarily needed for the description of the discrete displacement, gradient and stress fields. A natural choice amounts to choose monomial basis functions, in the form

$$a(\mathbf{x}) = \sum_j a_j \prod_{1 \leq i \leq d} \frac{(x_i - x_{Ti})^{\alpha_i^j}}{h_T} \quad \text{with} \quad \alpha^j = \left\{ (m_l)_{1 \leq l \leq d}, \sum_{1 \leq l \leq d} m_l = j \right\} \quad (31)$$

where  $a(\mathbf{x})$  denotes a polynomial scalar field of order  $k$  with coefficients  $a_{1 \leq j \leq k}$ . Using monomial basis functions, the coefficients  $a_{1 \leq j \leq k}$  do not necessarily represent nodal displacements as is the case with Lagrange shape functions, and are just scalar coordinates in polynomial basis. Moreover, the notion of reference element is not used in such a context, since monomial basis functions are directly expressed in the deformed element configuration, by scaling field values by the element diameter  $h_T$  and centroid  $\mathbf{x}_T$ . Monomial basis functions are not necessarily one-valued at the element nodes.

#### 4.2. Linearization

In the following section, we express the non-linear problem arising from (28) to solve in incremental form, and devise an iterative cell resolution algorithm to express the equilibrium of the cell with its boundary. In the latter, cell unknowns are locally eliminated, and expressed in terms of boundary displacements solely. The cell displacement then solves a non-linear system at a fixed boundary displacement, in order to verify the equilibrium of the cell with the boundary. The usual static condensation procedure that is used in the literature [43, 44] to eliminate cell unknowns is then retrieved by considering that cell unknowns evolve independently from boundary ones.

*Residual.* For non-linear behaviors and large deformations, problem (28) does not depend linearly on the displacement pair  $(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k)$ , since the stress  $\mathbf{P}_T^k$  is not linear with respect to  $\mathbf{G}_T^k$ . Hence, one defines the residual quantity  $R_T$  such that

$$R_T = \delta L_{T,\text{int}}^{HHO} - \delta L_{T,\text{ext}}^{HHO} \quad (32)$$

*Linearization.* From a numerical standpoint, problem (28) is solved iteratively by seeking a displacement correction pair  $(\alpha \mathbf{u}_T^l, \alpha \mathbf{u}_{\partial T}^k)$  such that

$$\frac{\partial R_T}{\partial \mathbf{u}_T^l} \alpha \mathbf{u}_T^l + \frac{\partial R_T}{\partial \mathbf{u}_{\partial T}^k} \alpha \mathbf{u}_{\partial T}^k = -R_T \quad (33)$$

for a given displacement pair  $(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k)$  at some iteration  $n$ . At iteration  $n+1$ , the displacement  $(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k)$  is updated by the correction  $(\alpha \mathbf{u}_T^l, \alpha \mathbf{u}_{\partial T}^k)$ , and a new value of the residual is computed. The correction at iteration  $n+1$  is sought by solving (33) for the updated residual, and the procedure is repeated until  $R_T < \epsilon$  for some tolerance  $\epsilon$ .

*Cell and boundary residual.*  $R_T$  can be decomposed into a cell contribution  $R_T^T$  and a boundary contribution  $R_T^{\partial T}$ , where  $R_T = R_T^T + R_T^{\partial T}$ . The problem amounts to seeking the displacement correction  $(\alpha \mathbf{u}_T^l, \alpha \mathbf{u}_{\partial T}^k)$  such that

$$\frac{\partial R_T^T}{\partial \mathbf{u}_T^l} \alpha \mathbf{u}_T^l + \frac{\partial R_T^T}{\partial \mathbf{u}_{\partial T}^k} \alpha \mathbf{u}_{\partial T}^k = -R_T^T \quad \text{and} \quad \frac{\partial R_T^{\partial T}}{\partial \mathbf{u}_T^l} \alpha \mathbf{u}_T^l + \frac{\partial R_T^{\partial T}}{\partial \mathbf{u}_{\partial T}^k} \alpha \mathbf{u}_{\partial T}^k = -R_T^{\partial T} \quad (34)$$

##### 4.2.1. Cell equilibrium scheme

*Implicit cell displacement.* Since the boundary  $\partial T$  is linked to the cell  $T$  through the interface (or equivalently, the stabilization term), a variation of boundary displacement yields a variation of cell displacement for the cell to be in equilibrium with its boundary. Hence, the cell displacement can be expressed implicitly as a function of the boundary displacement such that

$$\mathbf{u}_T^l = \mathbf{u}_T^l(\mathbf{u}_{\partial T}^k) \quad (35)$$

*Cell equilibrium.* Assuming that the boundary is fixed, the cell displacement correction that expresses the equilibrium of the cell with the boundary solves

$$\frac{dR_T^T}{d\mathbf{u}_T^l} \alpha \mathbf{u}_T^l = -R_T^T \quad (36)$$

and the equilibrium is reached for  $R_T^T$  close to 0

*Element equilibrium.* Assuming now that the cell is in equilibrium with the boundary, the variation of cell residual with respect to the boundary displacement yields

$$\frac{dR_T^T}{d\mathbf{u}_{\partial T}^k} = \frac{\partial R_T^T}{\partial \mathbf{u}_T^l} \frac{\partial \mathbf{u}_T^l}{\partial \mathbf{u}_{\partial T}^k} + \frac{\partial R_T^T}{\partial \mathbf{u}_{\partial T}^k} = 0 \quad \text{and thus} \quad \frac{\partial R_T^T}{\partial \mathbf{u}_T^l} \frac{\partial \mathbf{u}_T^l}{\partial \mathbf{u}_{\partial T}^k} = -\frac{\partial R_T^T}{\partial \mathbf{u}_{\partial T}^k} \quad (37)$$

Hence, the variation of boundary residual with respect to a boundary displacement variation writes

$$\frac{dR_T^{\partial T}}{d\mathbf{u}_{\partial T}^k} = \frac{\partial R_T^{\partial T}}{\partial \mathbf{u}_T^l} \frac{\partial \mathbf{u}_T^l}{\partial \mathbf{u}_{\partial T}^k} + \frac{\partial R_T^{\partial T}}{\partial \mathbf{u}_{\partial T}^k} = \frac{\partial R_T^{\partial T}}{\partial \mathbf{u}_{\partial T}^k} - \frac{\partial R_T^{\partial T}}{\partial \mathbf{u}_T^l} \frac{\partial \mathbf{u}_T^l}{\partial R_T^T} \frac{\partial R_T^T}{\partial \mathbf{u}_{\partial T}^k} \quad (38)$$

and the boundary displacement correction solves

$$\frac{dR_T^{\partial T}}{d\mathbf{u}_{\partial T}^k} \alpha \mathbf{u}_{\partial T}^k = -R_T^{\partial T} \quad (39)$$

#### 4.2.2. Static condensation scheme

*Cell displacement correction.* Assuming that the cell and boundary displacements are independent from one another, one can express the displacement correction as a term depending on the cell residual and the boundary displacement correction using (34)

$$\alpha \mathbf{u}_T^l = -\frac{\partial \mathbf{u}_T^l}{\partial R_T^T} \frac{\partial R_T^T}{\partial \mathbf{u}_{\partial T}^k} \alpha \mathbf{u}_{\partial T}^k - \frac{\partial \mathbf{u}_T^l}{\partial R_T^T} R_T^T \quad (40)$$

*Condensation of cell unknowns.* Using (40), cell unknowns can be eliminated from the problem such that it writes in terms of boundary displacement correction only

$$\left( \frac{\partial R_T^{\partial T}}{\partial \mathbf{u}_{\partial T}^k} - \frac{\partial R_T^{\partial T}}{\partial \mathbf{u}_T^l} \frac{\partial \mathbf{u}_T^l}{\partial R_T^T} \frac{\partial R_T^T}{\partial \mathbf{u}_{\partial T}^k} \right) \alpha \mathbf{u}_{\partial T}^k = -R_T^{\partial T} + \frac{\partial R_T^{\partial T}}{\partial \mathbf{u}_T^l} \frac{\partial \mathbf{u}_T^l}{\partial R_T^T} R_T^T \quad (41)$$

which yields the usual static condensation formulae.

#### 4.2.3. Comparison between both schemes

The static condensation algorithm is the one used in the literature [42, 45, 46, 43] to eliminate cell unknowns. Contrary to the introduced cell resolution algorithm, this scheme needs not iterate at the cell level to accomodate the cell correction, and is hence, faster. However, the actualization of the cell unknown displacement by its correction demands that the quantities  $\partial \mathbf{u}_T^l / \partial R_T^T$  and  $\partial R_T^T / \partial \mathbf{u}_{\partial T}^k$  computed at the previous iteration are known. From a numerical standpoint, this results in keeping the associated matrices (see Section 5) in memory from an iteration to another.

The novel cell resolution scheme needs iterate at the cell level. It is hence more costly than the static condensation one, and requires to integrate the constitutive equation more times than the static condensation algorithm does. However, it allows to exactly evaluate the equilibrium of the cell with its boundary, what does not the former. Moreover, it allows to consider extending the present cell correction iterative resolution to e.g. constrained resolution algorithm, in order to solve inequality constrained problems, as encountered in multi-field plasticity [47] for instance.

Finally, a schematic representation of the principle of both schemes is given in Figure 2

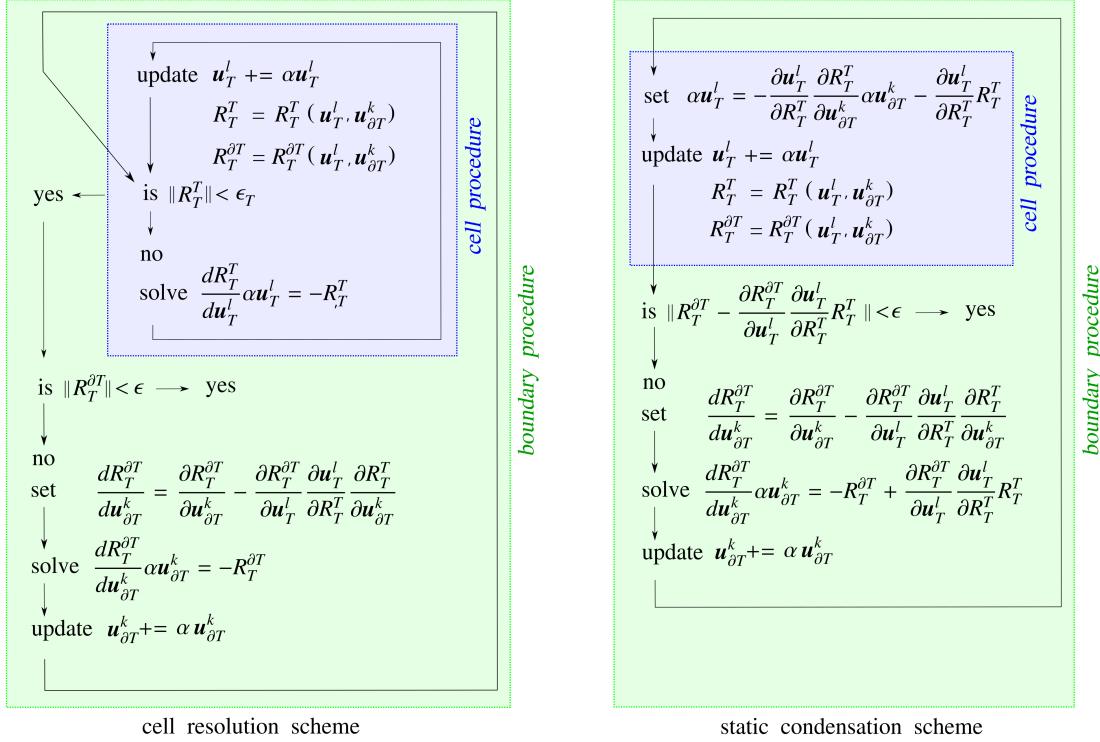


Figure 2. Schematic representation of both resolution schemes

#### 4.3. Spatial discretization

*Faces and skeleton of the mesh.* The boundary  $\partial T$  of each element is decomposed in faces, such that a face  $F$  is a subset of  $\Omega$ , and either there are two cells  $T$  and  $T'$  such that  $F = \partial T \cap \partial T'$  ( $F$  is then an interior face), or there is a single cell  $T$  such that  $F = \partial T \cap \partial\Omega$  ( $F$  is then an exterior face). For any cell  $T$ , let  $\mathcal{F}(T) = \{F \in \mathcal{F} \mid F \subset \partial T\}$  the set of faces composing the boundary of  $T$ . Let finally  $\mathcal{F}(\Omega) = \{F_i \subset \Omega \mid 1 \leq i \leq N_F\}$  the skeleton of the mesh, collecting all element faces  $F_i$  in the mesh, where  $N_F$  denotes the number of faces. The set of faces subjected to Neumann boundary conditions is denoted  $\mathcal{F}_N(\Omega)$ , and  $\mathcal{F}_D(\Omega)$  denotes that subjected to Dirichlet boundary conditions.

*Mesh description.* Likewise, one defines the collection of all cells in the mesh as  $\mathcal{T}(\Omega) = \{T_i \subset \Omega \mid 1 \leq i \leq N_T\}$ , where  $N_T$  denotes the total number of cells. The composition of both  $\mathcal{T}(\Omega)$  and  $\mathcal{F}(\Omega)$  forms the hybrid mesh  $\bar{\mathcal{T}}(\Omega) = \{\mathcal{T}(\Omega), \mathcal{F}(\Omega)\}$ .

#### 4.4. Global discrete problem

*Discrete global approximation spaces.* Let  $U^h(\mathcal{T}) = \prod_{T \in \mathcal{T}(\Omega)} U^h(T)$  the global discrete cell displacement space. Let  $V^h(\mathcal{F}) = \prod_{F \in \mathcal{F}(\Omega)} V^h(F)$  the global discrete face displacement space, and  $U^h(\bar{\mathcal{T}}) = U^h(\mathcal{T}) \times V^h(\mathcal{F})$  the global unknown approximation space.

*Discrete global problem.* The global problem in discrete form writes : find the pair  $(\mathbf{u}_T^l, \mathbf{u}_F^k) \in U^h(\bar{\mathcal{T}})$  verifying  $\mathbf{u}_F^k|_{\partial_D \Omega} = \mathbf{u}_D$  on  $\partial_D \Omega$  such that  $\forall (\delta \mathbf{u}_T^l, \delta \mathbf{u}_F^k) \in U^h(\bar{\mathcal{T}})$

$$\delta L_{T,\text{int}}^{HHO} - \delta L_{T,\text{ext}}^{HHO} = 0 \quad (42)$$

with respective Lagrangian variations

$$\delta L_{\mathcal{T},\text{int}}^{HHO} = \sum_{T \in \mathcal{T}(\Omega)} \int_T \mathbf{P}_T^k(\mathbf{G}_T^k(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k)) : \mathbf{G}_T^k(\delta \mathbf{u}_T^l, \delta \mathbf{u}_{\partial T}^k) + \int_{\partial T} (\beta/h_T) \mathbf{Z}_{\partial T}^{HHO}(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k) \cdot \mathbf{Z}_{\partial T}^{HHO}(\delta \mathbf{u}_T^l, \delta \mathbf{u}_{\partial T}^k) \quad (43a)$$

$$\delta L_{\mathcal{T},\text{ext}}^{HHO} = \sum_{F \in \mathcal{F}_N^e(\Omega)} \int_F \mathbf{t}_N \cdot \delta \mathbf{u}_F^k + \sum_{T \in \mathcal{T}(\Omega)} \int_T \mathbf{f}_V \cdot \delta \mathbf{u}_T^l \quad (43b)$$

*Linearization.* Let  $R_{\mathcal{T}}$  the global residual associated to the non-linear resolution of problem (42) such that

$$R_{\mathcal{T}} = \delta L_{\mathcal{T},\text{int}}^{HHO} - \delta L_{\mathcal{T},\text{ext}}^{HHO} = \sum_{T \in \mathcal{T}} R_T \quad (44)$$

where  $R_{\mathcal{T}}$  can also be written as the sum of elementary residuals  $R_T \forall T \in \mathcal{T}$ , since the sum of Neumann external contributions cancels out by equality of the traction force  $\mathbf{t}_{\partial_N T} = -\mathbf{t}_{\partial_N T'}$  for any  $T'$  adjacent to  $T$ . Likewise, the global residual  $R_{\mathcal{T}}$  can be decomposed into a mesh contribution  $R_{\mathcal{T}}^{\mathcal{T}}$  and into a skeletal contribution  $R_{\mathcal{T}}^{\mathcal{F}}$  such that  $R_{\mathcal{T}} = R_{\mathcal{T}}^{\mathcal{T}} + R_{\mathcal{T}}^{\mathcal{F}}$ .

*Cell unknowns elimination and assembly.* Since cell unknowns are locally eliminated, the problem expresses in terms of faces displacements only, and consists in finding the global faces unknowns correction  $\alpha \mathbf{u}_{\mathcal{F}}^k$  that solves

$$\frac{dR_{\mathcal{T}}^{\mathcal{F}}}{d\mathbf{u}_{\mathcal{F}}^k} \alpha \mathbf{u}_{\mathcal{F}}^k = -R_{\mathcal{F}} \quad (45)$$

where  $R_{\mathcal{F}}$  is the global obtained residual depending on the cell unknown elimination strategy. A schematic representation of the global iterative procedure is given in Figure 3

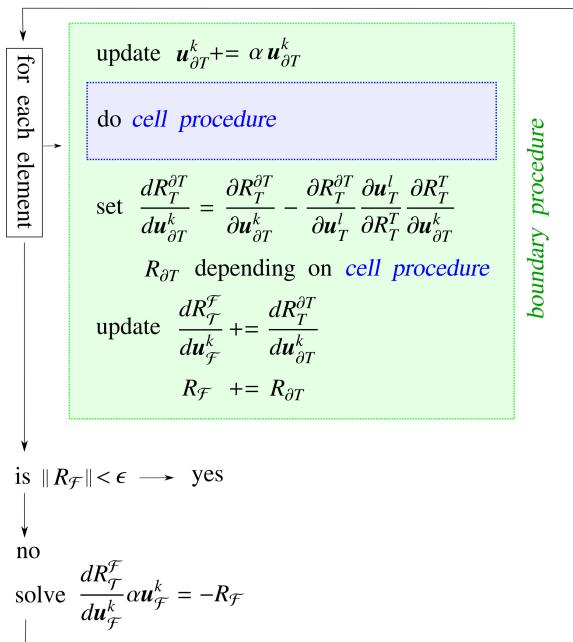


Figure 3. Schematic representation the global resolution scheme

## 5. Implementation

In this section, we specify the underlying matricial implementation of problem (43). In the following, the expression  $\{\cdot\}$  denotes a real-valued vector, and the notation  $[\cdot]$  a real-valued matrix.

*Quadrature.* As is customary with finite element methods, integrals are evaluated numerically by means of a quadrature rule on an element shape. Hence, let  $Q_T$  a quadrature rule for the cell  $T$  of order at least  $2k$ . A quadrature point is denoted  $X_q$  and a quadrature weight  $w_q$ .

### 5.1. Reconstructed gradient and stabilization operators

*Reconstructed gradient operator.* From an algebraic standpoint, (30) defines a linear problem consisting in inverting a mass matrix in  $G^h(T)$ . One can thus defines  $[B_T]$  the discrete gradient operator acting on the pair  $(\mathbf{v}_T^l, \mathbf{v}_{\partial T}^k)$  at a quadrature point  $X_q \in Q_T$  to evaluate the discrete gradient  $\mathbf{G}_T^k(\mathbf{v}_T^l, \mathbf{v}_{\partial T}^k)$  such that

$$\{\mathbf{G}_T^k(\mathbf{v}_T^l, \mathbf{v}_{\partial T}^k)\}(X_q) = [B_T](X_q) \cdot \begin{Bmatrix} \mathbf{v}_T^l \\ \mathbf{v}_{\partial T}^k \end{Bmatrix} \quad \forall (\mathbf{v}_T^l, \mathbf{v}_{\partial T}^k) \in U^h(\bar{T}) \quad (46)$$

where  $[B_T]$  is composed by a cell block and a boundary block.

*Stabilization operator.* Similarly, the algebraic realization of (26) amounts to compute the stabilization operator  $[Z_T]$  such that

$$\{\mathbf{Z}_{\partial T}^{HHO}(\mathbf{v}_T^l, \mathbf{v}_{\partial T}^k)\} = [Z_T] \cdot \begin{Bmatrix} \mathbf{v}_T^l \\ \mathbf{v}_{\partial T}^k \end{Bmatrix} \quad \forall (\mathbf{v}_T^l, \mathbf{v}_{\partial T}^k) \in U^h(\bar{T}) \quad (47)$$

as for  $[B_T]$ , the operator  $[Z_T]$  is composed by a cell and a boundary block.

*Offline computation.* Since (30) and (26) depend on the geometry of the element  $T$  only, one can compute the operators  $[B_T]$  and  $[Z_T]$  for each element once and for all in an offline pre-computation step by working in the reference configuration. Once this offline step is performed, the algebraic form of the problem resembles closely to the standard finite element one, where the operator  $[B_T]$  replaces the usual shape function gradient operator, and the stabilization operator  $[Z_T]$  is incorporated in the expression of the tangent matrix and in that of internal forces.

### 5.2. Elementary residual and its derivatives

*Internal forces.* The internal forces vector depends on the product of the stress values with the gradient operator computed at quadrature points, plus a supplementary force corresponding to that of the traction between the cell and its boundary

$$\{F_T^{int}(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k)\} = \sum_{X_q \in Q_T} w_q [B_T]^{\text{trans}}(X_q) \cdot \{\mathbf{P}_T^k(\mathbf{G}_T^k(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k))\}(X_q) + \frac{\beta}{h_T} [Z_T]^{\text{trans}} \cdot [Z_T] \cdot \begin{Bmatrix} \mathbf{u}_T^l \\ \mathbf{u}_{\partial T}^k \end{Bmatrix} \quad (48)$$

*External forces.* The external forces vector is the evaluation of the given bulk and boundary loads at respective cell and face quadrature points tested against the respective cell and face shape functions, and is denoted

$$\{F_T^{ext}\} = \begin{Bmatrix} \mathbf{f}_V \\ \mathbf{t}_N \end{Bmatrix} \quad (49)$$

*Residual.* Following (32), the elementary residual vector writes

$$\{R_T(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k)\} = \{F_T^{int}(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k)\} - \{F_T^{ext}\} \quad (50)$$

*Tangent matrix.* The tangent matrix  $[K_T(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k)]$  expresses the derivative of  $\{R_T(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k)\}$  with respect to  $\{\mathbf{u}_T^l\}$  and  $\{\mathbf{u}_{\partial T}^k\}$ , and writes as the product of the gradient operators by the tangent operators  $\tilde{A}(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k)$  at quadrature points, plus the additional stabilization term such that

$$[K_T(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k)] = \sum_{X_q \in Q_T} w_q [B_T]^{trans} (X_q) \cdot [\tilde{A}(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k)](X_q) \cdot [B_T](X_q) + \frac{\beta}{h_T} [Z_T]^{trans} \cdot [Z_T] \quad (51)$$

where the tangent operator  $\tilde{A}(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k)$  is the derivative of the stress with respect to the displacement gradient

$$\tilde{A}(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k) = \frac{\partial \mathbf{P}_T^k}{\partial \mathbf{G}_T^k} \quad (52)$$

and the tangent matrix can be decomposed by blocks such that

$$[K_T(\mathbf{u}_T^l, \mathbf{u}_{\partial T}^k)] = \begin{bmatrix} \partial R_T^T / \partial \mathbf{u}_T^l & \partial R_{\partial T}^T / \partial \mathbf{u}_T^l \\ \partial R_T^T / \partial \mathbf{u}_{\partial T}^k & \partial R_{\partial T}^T / \partial \mathbf{u}_{\partial T}^k \end{bmatrix} \quad (53)$$

where one retrieves the expressions of the residual derivatives as introduced in Section 4.2

## 6. Numerical examples in the Axisymmetric framework

In this section, we evaluate the proposed axisymmetric HHO method on classical test cases taken from the literature to emphasize robustness to volumetric locking. We consider both the small and large strains framework, for elasto-plastic behaviors. In this section, we denote by HHO( $k, l$ ) the HHO element of order  $k$  on faces, and order  $l$  in the cell.

*Stabilization parameter.* To ensure coercivity of the HHO method, the stabilization parameter  $\beta$  needs to be chosen according to the material under study. In the literature [42], a value of order  $2\mu$  is advocated, where  $\mu$  denotes the shear modulus of the material. We use this value for all test cases in the present section.

### 6.1. The free dilatation test

The first test case of the following benchmark aims at displaying the robustness of the HHO method for coupled mechanical-thermal problems.

*Specimen and loading.* For this test case, the unit box is fixed on both the right and bottom boundaries in their respective normal directions, and a quadratic thermal load depending on the  $r$ -coordinate is imposed in the solid (see Figure 4). The mesh is composed of 400 quadrangles. The thermal loading is given by

$$T(r, z) = 4(T_{max} - T_{min})r(1 - r) + T_{min} \quad (54)$$

with temperature values  $T_{max} = 2000$  K and  $T_{min} = 293.15$  K.

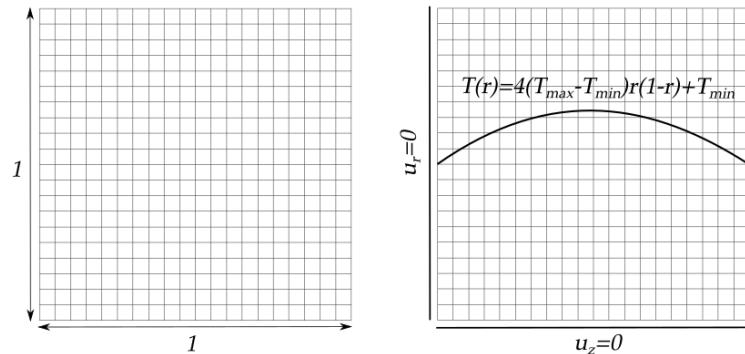


Figure 4. Geometry, displacement boundary conditions and temperature loading for the free dilatation test case

*Material behaviour.* A linear thermo-elastic energy potential is considered with a Young modulus  $E = 150$  GPa. The material is quasi-incompressible with a Poisson ratio  $\nu = 0.499$ . The dilatation parameter is taken as  $\alpha = 1e^{-6}$  K $^{-1}$ .

*Volumetric locking and polynomial approximation for the strain and temperature fields.* Lagrange finite elements of order  $1 \leq k \leq 2$  evaluate a mechanical strain of order  $0 \leq k - 1 \leq 1$ , whereas the thermal strain is of order 2. Using the total strain for the computation of the stress results in strong volumetric locking. Hence, for Lagrange element to accurately model the problem, one needs to choose a temperature fields that is one order lower than that describing the displacement field. This feature is of major importance for mixed elements, where quadratic elements needs be employed to match the the linear pressure unknown polynomial order.

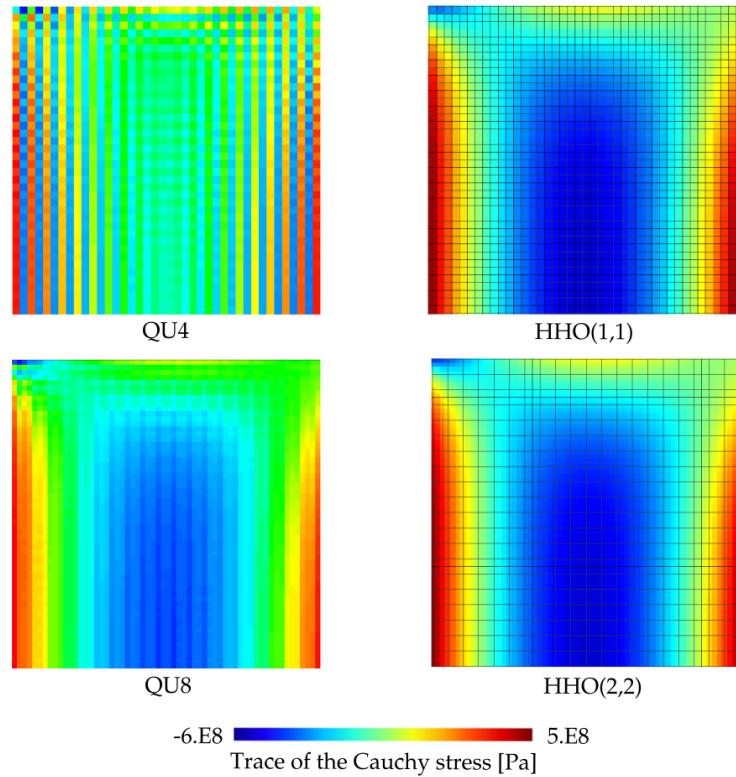


Figure 5. Map fo the Trace of the Cauchy stress at quadrature points for the Free dilatation test case at the last time step

*Comparison of FE and HHO methods.* In Figure 5, one can observe that the pressure map is completely smooth for the HHO computations, even for a quadratic temperature field acting on a linear gradient using HHO(1,1). As expected, the results display mild signs of volumetric locking for the quadratic finite element approximation, and strong oscillations are noted for the linear finite element solution.

## 6.2. Perfect plastic swelling sphere

*Specimen and loading.* This benchmark consists in a quasi-incompressible sphere under uniform internal loading. This test case has an analytical solution and the state of the specimen is known when the plastic region has reached the external border of the sphere. The sphere has an inner radius  $r_{int} = 0.8$  mm and an outer radius  $r_{ext} = 1$  mm. An internal radial displacement  $u$  is imposed. The mesh is composed of XXX quadrangles (see Figure 6). The simulation is performed until the limit load corresponding to an internal displacement of 0.2 mm is reached.

*Material behaviour.* An isotropic hardening energy potential  $\psi_\Omega^p$  is chosen for the description of the plastic evolution of the material such that

$$\psi_\Omega^p(p) = \sigma_0 p + \frac{1}{2} H p^2 + (\sigma_\infty - \sigma_0) \left( p - \frac{1 - e^{-\delta p}}{\delta} \right) \quad (55)$$

where the parameter  $p$  denotes the equivalent plastic strain and a Von Mises yields function  $f$  describes the flow rule

$$f = \sqrt{\frac{3}{2}} \|\text{dev}(\boldsymbol{\sigma})\| - p \quad (56)$$

Moreover, the small strain hypothesis is assumed for this test case.

Perfect plasticity is considered for this test case, where the saturation parameter  $\delta = 0$ , the yield stresses  $\sigma_0 = \sigma_\infty = 6$  MPa, the hardening parameter  $H = 0$  and the elastic potential parameters are the Young modulus  $E = 28.85$  MPa and the Poisson ratio  $\nu = 0.499$ , such that the material is quasi-incompressible.

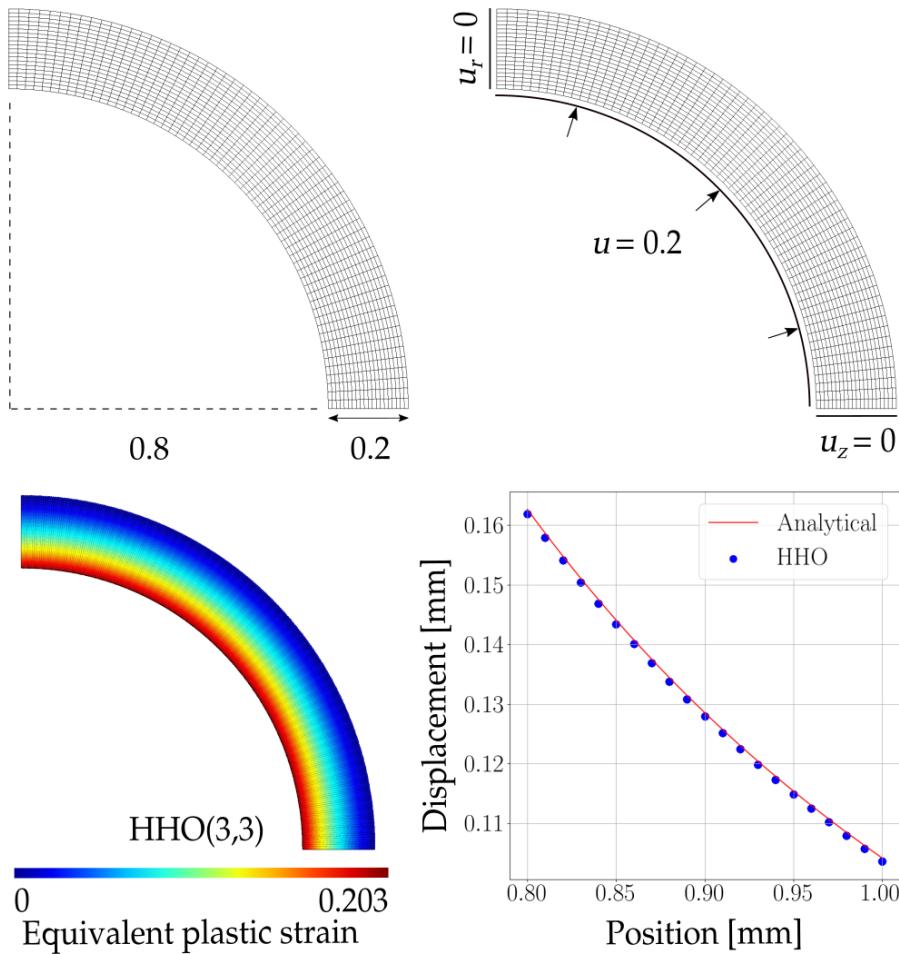


Figure 6. the swelling sphere test case. Geometry, loadings, final displacement along the radius of the sphere, and final equivalent plastic strain map at quadrature points

*Displacement along the radius.* Since an analytical solution is known for this test case, we compare it to the proposed HHO method. The displacement of the section of the sphere at cell nodes is plotted in Figure 6, along with the analytical one, and we observe that the obtained results are in agreement with the analytical response. Figure 6 mentions the label HHO without specifying approximation orders for all computations deliver the same result.

*Trace of the Cauchy stress.* As for the displacement, the analytical solution for the trace of the Cauchy stress tensor is compared to the one computed using the proposed HHO method for three approximation orders. A sign of volumetric locking is the presence of strong oscillations in the trace of the Cauchy stress (or, equivalently, the hydrostatic pressure) within elements. We observe that numerical results at quadrature points fit the analytical curve, and display no sign of volumetric locking. The computed solution is however less smooth at the borders of the specimen for higher orders, a phenomenon that was pointed out in [46] for the three dimensional case, and attributed to the fact that planar faces are considered.

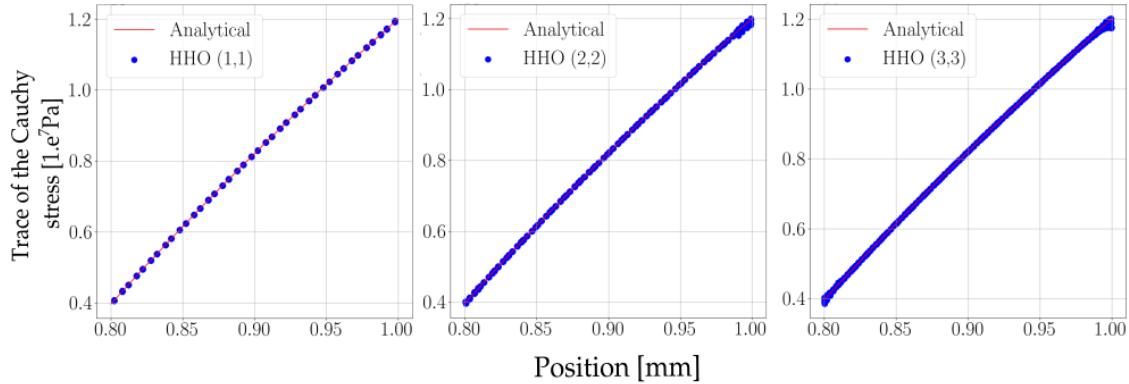


Figure 7. trace of the Cauchy stress tensor along the radius of the sphere at quadrature points

### 6.3. Necking of a notched bar

*Specimen and loading.* We consider a notched bar that is subjected to uniaxial extension. The bar has a length of 30 mm, a top section of radius 5 mm and a bottom section of radius 3 mm. A vertical displacement  $u_z = 0.8$  mm is imposed at the top, as shown in Figure 9. For symmetry reasons, only one-quarter of the bar is discretized, and the mesh is composed of XXX quadrangles.

*Behaviour law.* The same behavior law as that in 6.2 is considered for the present test case. However, the finite strain hypothesis is chosen, based on a logarithmic decomposition of the stress [48].

*Material parameters.* Materials parameters are taken as  $\sigma_0 = 450$  MPa,  $\sigma_\infty = 715$  MPa with a saturation parameter  $\delta = 16.93$ . The Young modulus is  $E = 206.9$  GPa, and the Poisson ratio is  $\nu = 0.29$ .

*Load deflection curve.* The load-displacement curve is plotted in Figure 9, and gives similar results to that obtained with quadratic reduced integration elements.

*Equivalent plastic strain.* Moreover, the equivalent plastic strain  $p$  at quadrature points and at the final load is plotted Figure 8. It has been observed that the equivalent plastic strain might suffer some oscillations at a certain limit load with UPG methods. One notices through the present example, that the proposed HHO method displays no oscillations of the equivalent plastic strain.

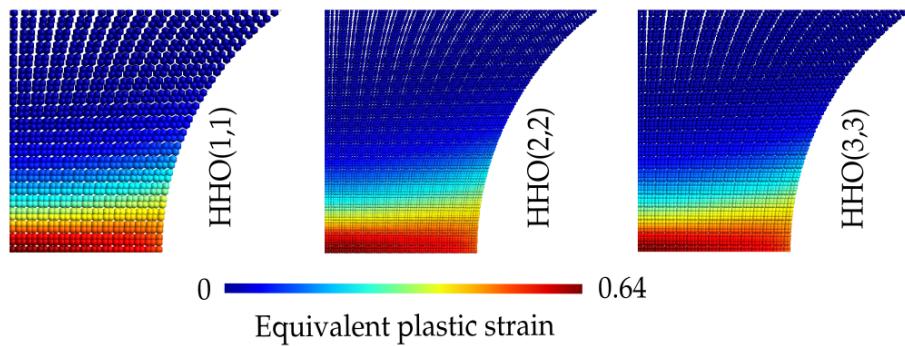


Figure 8. final equivalent plastic strain map at quadrature points in the notch region

*Hydrostatic pressure.* The hydrostatic pressure map at quadrature points and at the final load is shown Figure 9 for three HHO element orders (respectively 1, 2 and 3). As for the swelling sphere test case, one notices that the hydrostatic pressure map is fairly smooth over the whole structure at all approximation orders, even at the bottom left corner where plasticity is confined.

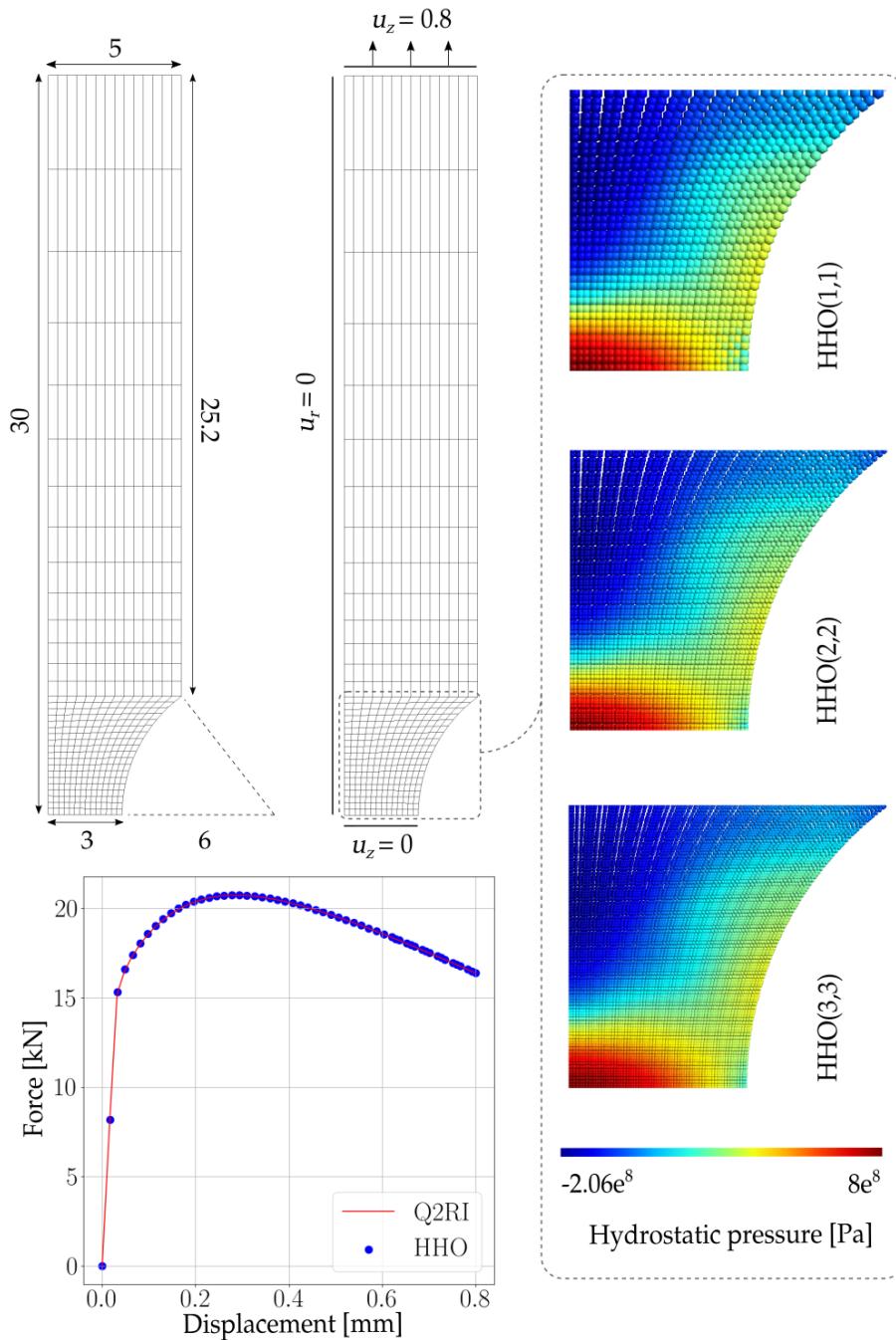


Figure 9. the notched specimen test case. Geometry, loadings, load deflection curve, and final hydrostatic pressure map at quadrature points in the notch region

## 7. Numerical examples using the cell resolution algorithm

The section showcases numerical examples demonstrating the robustness of the cell resolution algorithm. The test cases under study, namely the classical Cook membrane, and the indentation test cases, show that no volumetric locking is encountered using the cell resolution algorithm.

### 7.1. Cook's membrane test case

**Specimen and loading.** Let consider the Cook membrane specimen that is subjected to uniaxial traction. The membrane has a width of 48 mm and a height of 60 mm, and a vertical traction  $t_y = 1000 \text{ N/m}$  is imposed at the top. The HHO computation is run on a polygonal mesh (see Figure 10) and is compared with standard QU4 and QU8 formulations (*i.e.* linear and quadratic approximations)

**Constitutive equation.** The same behavior law as that in 6.2 is considered for the present test case. However, the finite strain hypothesis is chosen, based on a logarithmic decomposition of the stress [48].

**Material parameters.** Materials parameters are taken as  $\sigma_0 = 450 \text{ MPa}$ ,  $\sigma_\infty = 715 \text{ MPa}$  with a saturation parameter  $\delta = 16.93$ . The Young modulus is  $E = 206.9 \text{ GPa}$ , and the Poisson ratio is  $\nu = 0.29$ .

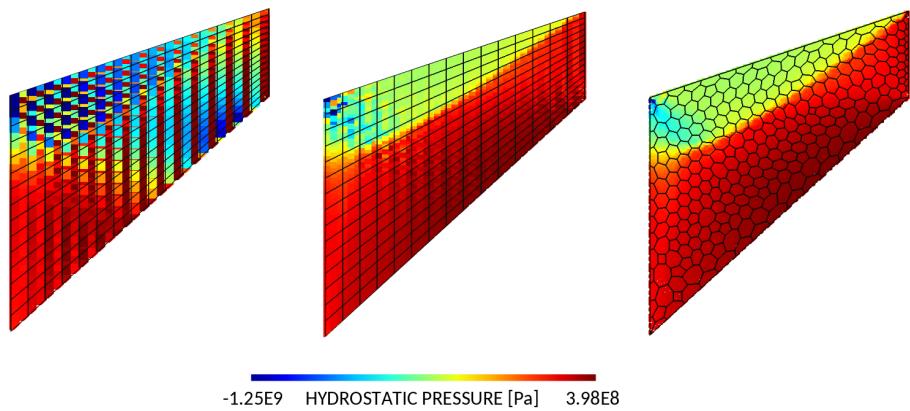


Figure 10. Hydrostatic pressure map one the reference configuration at the limit load

**Numerical results.** As expected, the linear and quadratic finite element methods display respectively strong and mild oscillations of the pressure, whereas the HHO one shows no sign of locking.

### 7.2. Indentation test case

**Specimen and loading.** The last test case consists in the indentation of a cube of size 10 mm. A pressure of 300 MPa is imposed on the top surface see Figure 11).

**Material.** The same perfect plastic material as that in 6.2 is considered for the present test case.

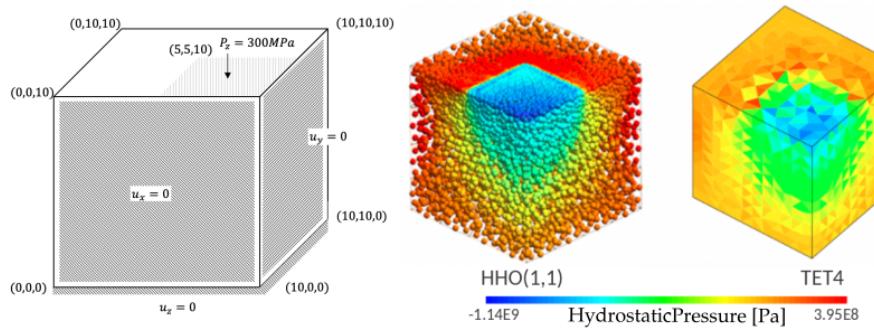


Figure 11. Hydrostatic pressure map one the reference configuration at the limit load

**Numerical results.** The pressure map at the end of the computation is displayed in Figure 11, and no sign of volumetric locking are present on the HHO computation, as opposed to the linear finite element one.

*Performance of the algorithm.* As stated in Section 4.2.3, the cell resolution algorithm needs more iterations (see Figure 12) than the classical linear resolution using a static condensation procedure. Indeed, since after a increment over the skeleton of the mesh, the cell displacement does not yet solve the equilibrium in the cell, the computation of the reconstructed gradient is not accurate, which results in ill-computed stresses at quadrature points. In order to circumvent this phenomenon, a prediction step is used to lift the cell displacement and then solve a non-linear problem within the cell. Using this prediction step displays similar results that those obtained with the static condensation algorithm (see Figure 12).

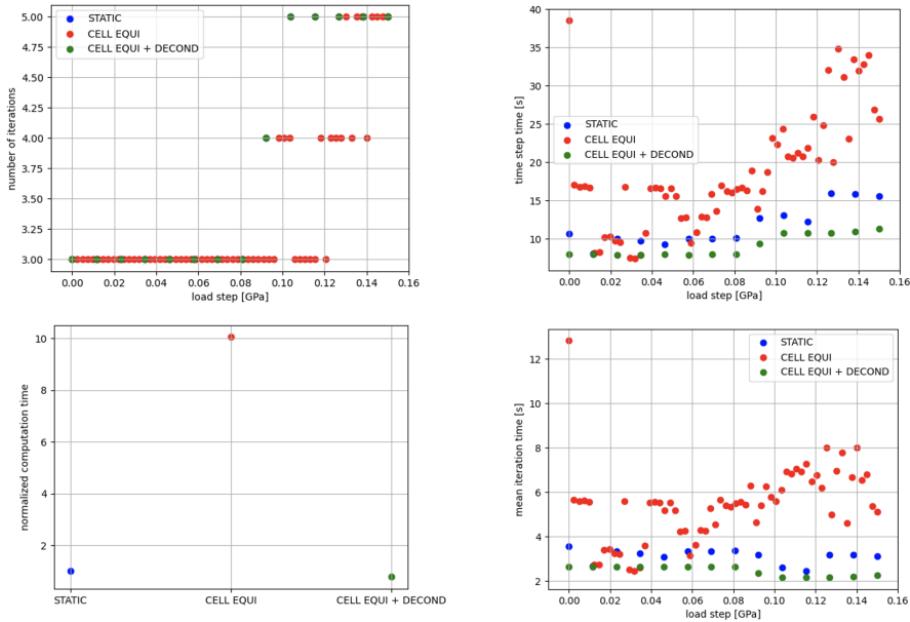


Figure 12. Comparison in terms of performance for different algorithms

## 8. Conclusion

An introduction to HDG and HHO methods has been proposed, based on the minimization of a Hu-Washizu Lagrangian. The expression of the method arising from this approach allows to introduce naturally all the ingredients of the method, as well as the displacement discontinuity, in a unified framework. This new formulation also allows one to draw a connection between HDG methods and other locking-free methods, all based on the minimization of a Hu-Washizu Lagrangian. A natural cell-based resolution scheme emerged from this formulation, leading to the proposition for a novel algorithm, based on the resolution of the equilibrium of the cell. This algorithm has been tested and investigated. Finally, we have devised and evaluated numerically an HHO method to account for mechanical problems in the axisymmetric framework, for both linear thermoelastic behaviours, and plastic behaviours under both the small and finite strain hypotheses. The HHO method exhibits a robust behavior for strain-hardening plasticity as well as for perfect plasticity and produces accurate solutions with a moderate number of degrees of freedom for various benchmarks from the literature.

This work can be pursued in several directions. One could use the cell resolution algorithm to address local resolution problems, such as those encountered with *e.g.* damage irreversibility in phase field fracture mechanics, or multi field plasticity. Moreover, an adaptation of the HHO method to reconstruct pressure-driven gradient terms only could lead to a simpler formulation, closer to that of mixed methods [38].

## 9. Acknowledgements

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## Appendix A. Appendix

### Appendix A.1. From the continuous Hu-Washizu Lagrangian to the HDG Lagrangian

In this part, the development for the expression of the Hu-Washizu Lagrangian (14) is exposed, using the assumptions made in Section 3.2.

*Element geometry.* In the following, the cell  $T$  is assumed to be convex. It is split into a core part  $K \subset T$  with boundary  $\partial K$ , and into an interface part  $I \subset T$  with boundary  $\partial I = \partial K \cup \partial T$ , as shown in Figure 1. The interface  $I$  has some thickness  $\ell > 0$  that is supposed to be small compared to  $h_T$  the diameter of  $T$ .

*Homothetic transformation.* Let  $\Xi_T$  the homothety of ratio  $(1 - \alpha\ell)$  and center  $X_T$  the centroid of  $T$ , with  $0 < \alpha < 1/\ell$  such that  $K$  (respectively  $\partial K$ ) is the image of  $T$  (respectively  $\partial T$ ) by  $\Xi_T$ . Since  $\partial K$  is an homothety of  $\partial T$ , any point  $X_{\partial T} \in \partial T$  and  $X_{\partial K} = \Xi_T(X_{\partial T}) \in \partial K$  share the same unit outward normal  $\mathbf{n}$ .

*Change of reference.* Let the change of frame  $\Psi$  that takes a point from the reference frame to the local frame with origin on  $\partial K$ , and whose first direction is given by the normal vector  $\mathbf{n}$  such that

$$\Psi : X \mapsto \tilde{\mathbf{x}} = \underline{Q}X + \mathbf{c} \quad (\text{A.1})$$

where  $\underline{Q}$  is the rotation matrix whose first row coincides with  $\mathbf{n}$ , and  $\mathbf{c}$  is a constant vector.

*Displacement in the interface.* Assuming that the interface  $I$  is thin enough (*i.e.* that  $\ell$  is small enough) let assume that the displacement  $\mathbf{u}_I$  in  $I$  linearly bridges  $\mathbf{u}_K|_{\partial K}$  to  $\mathbf{u}_{\partial T}$  such that

$$\mathbf{u}_I(\mathbf{x}) = \frac{\mathbf{u}_{\partial T}(\Psi^{-1}(\mathbf{x}_\ell)) - \mathbf{u}_K|_{\partial K}(\Psi^{-1}(\mathbf{x}_o))}{\ell} x_0 + \mathbf{u}_K|_{\partial K}(\Psi^{-1}(\mathbf{x}_o)) \quad (\text{A.2})$$

where  $x_0$  is the first coordinate of a point  $\mathbf{x}$  in the local frame defined by  $\Psi$ . The vector  $\mathbf{x}_o$  denotes a point located in the plane  $x_0 = 0$ , and  $\mathbf{x}_\ell$  a point on the plane  $x_0 = \ell$ , such that they share the same coordinates on their respective planes.

*Displacement gradient in the interface.* The derivative of  $\mathbf{u}_I$  with respect to  $X$  yields

$$\frac{\partial \mathbf{u}_I}{\partial X_j} = \sum_k \frac{\partial u_{Ii}}{\partial x_k} \frac{\partial x_k}{\partial X_j} = \frac{u_{\partial T i}(\Psi^{-1}(\mathbf{x}_\ell)) - u_{K i}|_{\partial K}(\Psi^{-1}(\mathbf{x}_o))}{\ell} Q_{0j} \quad (\text{A.3})$$

which reads

$$\nabla \mathbf{u}_I(X) = \frac{\mathbf{u}_{\partial T}(X_\ell) - \mathbf{u}_K|_{\partial K}(X_o)}{\ell} \otimes \mathbf{n} \quad (\text{A.4})$$

where we have used the fact that the first row of the rotation matrix  $\underline{Q}$  is given by  $\mathbf{n}$ . The points  $X_o$  and  $X_\ell$  are located on the normal plane to  $\mathbf{n}$  on  $\partial K$  and  $\partial T$  respectively, in the reference frame.

*Stress in the interface.* As introduced in Section 3, the stress  $\mathbf{P}_I$  is assumed constant along the direction  $\mathbf{n}$  in  $I$ . By continuity of the traction force across  $\partial K$ , the following equality holds true

$$(\underline{P}_I - \underline{P}_K|_{\partial K}) \cdot \mathbf{n} = 0 \quad \text{in } I \quad (\text{A.5})$$

*Internal Hu-Washizu in the interface.* Let  $L_{I,\text{int}}^{HW}$  the internal contribution of the Hu-Washizu Lagrangian in  $I$

$$L_{I,\text{int}}^{HW} := \int_I \psi_I + (\nabla \mathbf{u}_I - \mathbf{G}_I) : \mathbf{P}_I \quad (\text{A.6})$$

Let  $C_I = \{v \in L^2(I) \mid v \cdot \mathbf{n} = \text{cste}\}$  the set of  $L^2$ -functions which are constant along the normal axis in  $I$ . For any function in  $C_I$ , the following equality holds true:

$$\int_I v \, dV = \int_{\partial K} \int_{\epsilon=0}^{\ell} v(1 - \alpha\epsilon) \, dS \, d\epsilon = \ell(1 - \frac{\alpha}{2}\ell) \int_{\partial K} v \, dS \quad (\text{A.7})$$

Noticing that  $\nabla \mathbf{u}_I \in C_I$ , one has :

$$\begin{aligned} \int_I \psi_I &= \ell(1 - \frac{\alpha}{2}\ell) \int_{\partial K} \frac{1}{2} \beta \frac{\ell}{h_T} \nabla \mathbf{u}_I : \nabla \mathbf{u}_I \\ &= \ell(1 - \frac{\alpha}{2}\ell) \int_{\partial K} \frac{\beta}{2\ell h_T} (\mathbf{u}_{\partial T} - \mathbf{u}_K|_{\partial K}) \otimes \mathbf{n} : (\mathbf{u}_{\partial T} - \mathbf{u}_K|_{\partial K}) \otimes \mathbf{n} \\ &= \ell(1 - \frac{\alpha}{2}\ell) \int_{\partial K} \frac{\beta}{2\ell h_T} \|\mathbf{u}_{\partial T} - \mathbf{u}_K|_{\partial K}\|^2 \\ &= (1 - \frac{\alpha}{2}\ell) \int_{\partial K} \frac{\beta}{2h_T} \|\mathbf{u}_{\partial T} - \mathbf{u}_K|_{\partial K}\|^2 \end{aligned} \quad (\text{A.8})$$

Moreover, for  $\mathbf{P}_I$  in  $C_I$  :

$$\begin{aligned} \int_I \nabla \mathbf{u}_I : \mathbf{P}_I &= \ell(1 - \frac{\alpha}{2}\ell) \int_{\partial K} \nabla \mathbf{u}_I : \mathbf{P}_I \\ &= \ell(1 - \frac{\alpha}{2}\ell) \int_{\partial K} \frac{1}{\ell} (\mathbf{u}_{\partial T} - \mathbf{u}_K|_{\partial K}) \otimes \mathbf{n} : \mathbf{P}_I \\ &= (1 - \frac{\alpha}{2}\ell) \int_{\partial K} (\mathbf{u}_{\partial T} - \mathbf{u}_K|_{\partial K}) \cdot \mathbf{P}_K|_{\partial K} \cdot \mathbf{n} \end{aligned} \quad (\text{A.9})$$

where we have used (A.5). And Finally :

$$L_{I,\text{int}}^{HW} = (1 - \frac{\alpha}{2}\ell) \int_{\partial K} \frac{\beta}{2h_T} \|\mathbf{u}_{\partial T} - \mathbf{u}_K|_{\partial K}\|^2 + (1 - \frac{\alpha}{2}\ell) \int_{\partial K} (\mathbf{u}_{\partial T} - \mathbf{u}_K|_{\partial K}) \cdot \mathbf{P}_K|_{\partial K} \cdot \mathbf{n} - \int_I \mathbf{G}_I : \mathbf{P}_I \quad (\text{A.10})$$

*Total Hu-Washizu Lagrangian in the composite element.* Injecting (A.6) in (11) yields

$$\begin{aligned} L_T^{HW} &= \int_K \psi_\Omega + (\nabla \mathbf{u}_K - \mathbf{G}_K) : \mathbf{P}_K + (1 - \frac{\alpha}{2}\ell) \int_{\partial K} (\mathbf{u}_{\partial T} - \mathbf{u}_K|_{\partial K}) \cdot \mathbf{P}_K|_{\partial K} \cdot \mathbf{n} \\ &\quad + (1 - \frac{\alpha}{2}\ell) \int_{\partial K} \frac{\beta}{2h_T} \|\mathbf{u}_{\partial T} - \mathbf{u}_K|_{\partial K}\|^2 - \int_I \mathbf{G}_I : \mathbf{P}_I - \int_K \mathbf{f}_V \cdot \mathbf{u}_K - \int_I \mathbf{f}_V \cdot \mathbf{u}_I - \int_{\partial_N T} \mathbf{t}_{\partial_N T} \cdot \mathbf{u}_{\partial T} \end{aligned} \quad (\text{A.11})$$

Since  $\ell$  is arbitrary, let  $\ell \rightarrow 0$ , the interface region vanishes such that  $I \rightarrow \emptyset$ ,  $K \rightarrow T$  and  $\partial K \rightarrow \partial T$ , and the expression of the Hu-Washizu functional over the region  $T$  writes

$$\begin{aligned} L_T^{HW} &= \int_T \psi_\Omega + (\nabla \mathbf{u}_T - \mathbf{G}_T) : \mathbf{P}_T + \int_{\partial T} (\mathbf{u}_{\partial T} - \mathbf{u}_T|_{\partial T}) \cdot \mathbf{P}_T|_{\partial T} \cdot \mathbf{n} + \int_{\partial T} \frac{\beta}{2h_T} \|\mathbf{u}_{\partial T} - \mathbf{u}_T|_{\partial T}\|^2 \\ &\quad - \int_T \mathbf{f}_V \cdot \mathbf{u}_T - \int_{\partial_N T} \mathbf{t}_{\partial_N T} \cdot \mathbf{u}_{\partial T} \end{aligned} \quad (\text{A.12})$$

which concludes the development of equation (14).

### Appendix A.2. Reconstructed gradient and Elliptic projection

This section aims at generalizing the elliptic projection property of the reconstructed gradient, as introduced in [29]. In the following, subspaces for the cell and faces approximations are not assumed to be polynomial necessarily.

Let  $U(T)$  the space of cell kinematically admissible displacements, and  $V(\partial T)$  that of face kinematically admissible displacements. The space for statically admissible stress and strain is denoted  $S(T)$ .

Let  $U^h(T) \subset U(T)$  and  $U^\perp(T) \subset U(T)$  such that  $U(T) = U^h(T) \oplus U^\perp(T)$ , and set  $\mathbf{u}_T = \mathbf{u}_T^h + \mathbf{u}_T^\perp$  with  $\mathbf{u}_T^h \in U^h(T)$  and  $\mathbf{u}_T^\perp \in U^\perp(T)$  the orthogonal projections of  $\mathbf{u}_T$  onto  $U^h(T)$  and  $U^\perp(T)$  respectively. Let  $V^h(\partial T) \subset V(\partial T)$  and  $\mathbf{u}_{\partial T}^h \in V^h(\partial T)$  the orthogonal projection of  $\mathbf{u}_T$  onto  $V^h(\partial T)$ . The orthogonal projection of  $\mathbf{u}_T$  onto  $U^h(\bar{T}) = U^h(T) \times V^h(\partial T)$  is then the displacement pair  $(\mathbf{u}_T^h, \mathbf{u}_{\partial T}^h)$ . Let  $S^h(T) = \{\boldsymbol{\tau}_T^h \in S(T) \mid \nabla \cdot \boldsymbol{\tau}_T^h \in U^h(T) \mid \boldsymbol{\tau}_T^h|_{\partial T} \cdot \mathbf{n} \in V^h(\partial T)\}$ , and  $\mathbf{G}_T^h \in S^h(T)$  the solution of (16) for  $(\mathbf{u}_T^h, \mathbf{u}_{\partial T}^h)$  such that

$$\int_T \mathbf{G}_T^h(\mathbf{u}_T^h, \mathbf{u}_{\partial T}^h) : \boldsymbol{\tau}_T^h = \int_T \nabla \mathbf{u}_T^h : \boldsymbol{\tau}_T^h + \int_{\partial T} (\mathbf{u}_{\partial T}^h - \mathbf{u}_T^h|_{\partial T}) \cdot \boldsymbol{\tau}_T^h|_{\partial T} \cdot \mathbf{n} \quad \forall \boldsymbol{\tau}_T^h \in S^h(T) \quad (\text{A.13})$$

using the fact that  $\mathbf{u}_{\partial T}^h$  is the projection of  $\mathbf{u}_T$  onto  $V^h(\partial T)$  and that  $\boldsymbol{\tau}|_{\partial T} \cdot \mathbf{n} \in V^h(\partial T)$ :

$$\begin{aligned} \int_T \mathbf{G}_T^h(\mathbf{u}_T^h, \mathbf{u}_{\partial T}^h) : \boldsymbol{\tau}_T^h &= \int_T \nabla \mathbf{u}_T^h : \boldsymbol{\tau}_T^h + \int_{\partial T} (\mathbf{u}_T|_{\partial T} - \mathbf{u}_T^h|_{\partial T}) \cdot \boldsymbol{\tau}_T^h|_{\partial T} \cdot \mathbf{n} && \forall \boldsymbol{\tau}_T^h \in S^h(T) \\ &= \int_T \nabla \mathbf{u}_T^h : \boldsymbol{\tau}_T^h + \int_{\partial T} \mathbf{u}_T^\perp|_{\partial T} \cdot \boldsymbol{\tau}_T^h|_{\partial T} \cdot \mathbf{n} && \forall \boldsymbol{\tau}_T^h \in S^h(T) \end{aligned} \quad (\text{A.14})$$

using the divergence theorem and the fact that  $\nabla \cdot \boldsymbol{\tau}_T^h \in U^h(T)$ , one has :

$$\int_T \nabla \mathbf{u}_T^\perp : \boldsymbol{\tau}_T^h = \int_{\partial T} \mathbf{u}_T^\perp|_{\partial T} \cdot \boldsymbol{\tau}_T^h|_{\partial T} \cdot \mathbf{n} \quad (\text{A.15})$$

such that :

$$\begin{aligned} \int_T \mathbf{G}_T^h(\mathbf{u}_T^h, \mathbf{u}_{\partial T}^h) : \boldsymbol{\tau}_T^h &= \int_T \nabla \mathbf{u}_T^h : \boldsymbol{\tau}_T^h + \int_T \nabla \mathbf{u}_T^\perp : \boldsymbol{\tau}_T^h && \forall \boldsymbol{\tau}_T^h \in S^h(T) \\ &= \int_T \nabla \mathbf{u}_T : \boldsymbol{\tau}_T^h && \forall \boldsymbol{\tau}_T^h \in S^h(T) \end{aligned} \quad (\text{A.16})$$

which states that  $\mathbf{G}_T^h(\mathbf{u}_T^h, \mathbf{u}_{\partial T}^h)$  is the orthogonal projection of  $\nabla \mathbf{u}_T$  onto  $S^h(T)$ . By linearity of the algebraic trace operator, one has that  $\text{Tr}(\mathbf{G}_T^h(\mathbf{u}_T^h, \mathbf{u}_{\partial T}^h))$  is the orthogonal projection of  $\nabla \cdot \mathbf{u}$ , which proves robustness of the method for linear elastic materials, since the Lamé coefficient  $\lambda$  acts on  $\nabla \cdot \mathbf{u}$ .

### Appendix A.3. Operators in the axi-symmetric framework

This part specifies the formulation of HHO operators in the axi-symmetric framework.

*Reconstructed gradient.* For any displacement pair  $(\mathbf{v}_T^l, \mathbf{v}_{\partial T}^k) \in U^h(T) \times V^h(\partial T)$ , the component  $G_{T\theta\theta}(v_{Tr}, v_{\partial Tr})$  solves

$$\int_T 2\pi r G_{T\theta\theta}(v_{Tr}, v_{\partial Tr}) \tau_{T\theta\theta} = \int_T 2\pi r \frac{u_{Tr}}{r} \tau_{T\theta\theta} = \int_T 2\pi u_{Tr} \tau_{T\theta\theta} \quad \forall \boldsymbol{\tau}_T \in S(T) \quad (\text{A.17})$$

In the radial and ordonal directions, i.e.  $\forall i, j \in \{r, z\}$ , the expression given in (16) is retrieved, and the component  $G_{Ti j}(v_{Ti}, v_{\partial Ti})$  solves

$$\int_T 2\pi r G_{Ti j}(v_{Ti}, v_{\partial Ti}) \tau_{Ti j} = \int_T 2\pi r \frac{\partial u_{Ti}}{\partial j} \tau_{ij} - \int_{\partial T} 2\pi r (u_{\partial Ti} - u_{Ti}|_{\partial T}) \tau_{Ti j}|_{\partial T} n_j \quad \forall \boldsymbol{\tau}_T \in S(T) \quad (\text{A.18})$$

*Reconstructed higher order displacement.* For any  $\mathbf{d}_T^{k+1} \in D^h(T)$ , the radial component  $w_{Tr}^{k+1}$  solves

$$\begin{aligned} \int_T 2\pi r \left( \sum_{i \in \{r,z\}} \frac{\partial w_{Tr}^{k+1}}{\partial i} \frac{\partial d_{Tr}^{k+1}}{\partial i} + \frac{w_{Tr}^{k+1}}{r} \frac{d_{Tr}^{k+1}}{r} \right) &= \int_T 2\pi r \left( \sum_{i \in \{r,z\}} \frac{\partial u_{Tr}}{\partial i} \frac{\partial d_{Tr}^{k+1}}{\partial i} + \frac{u_{Tr}}{r} \frac{d_{Tr}^{k+1}}{r} \right) \\ &\quad + \int_{\partial T} 2\pi r \sum_{i \in \{r,z\}} (u_{\partial Tr} - u_{Tr}|_{\partial T}) \frac{\partial d_{Tr}^{k+1}}{\partial i} |_{\partial T} n_i \end{aligned} \quad (\text{A.19})$$

where the mean value condition is not needed on the radial component of the higher order displacement since the left hand side of the system described by (A.19) depends directly on the displacement unknown and not only on its gradient as in (A.20). The ordinate component  $w_{Tz}^{k+1}$  solves :

$$\int_T 2\pi r \sum_{i \in \{r,z\}} \frac{\partial w_{Tz}^{k+1}}{\partial i} \frac{\partial d_{Tz}^{k+1}}{\partial i} = \int_T 2\pi r \sum_{i \in \{r,z\}} \frac{\partial u_{Tz}}{\partial i} \frac{\partial d_{Tz}^{k+1}}{\partial i} - \int_{\partial T} 2\pi r \sum_{i \in \{r,z\}} (u_{\partial Tz} - u_{Tz}|_{\partial T}) \frac{\partial d_{Tz}^{k+1}}{\partial i} |_{\partial T} n_i \quad (\text{A.20a})$$

$$\int_T 2\pi r w_{Tz}^{k+1} = \int_T 2\pi r u_{Tz} \quad (\text{A.20b})$$

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