

Equilibrium condition requires that

$$M_1 + M_2 = P_z \text{ acting to counteract}$$

$$\text{For balanced Doweling Condition } M_1 = M_2 = M$$

$$2M = Pz$$

$$M = \frac{Pz}{2}$$

~~Embed portions of~~

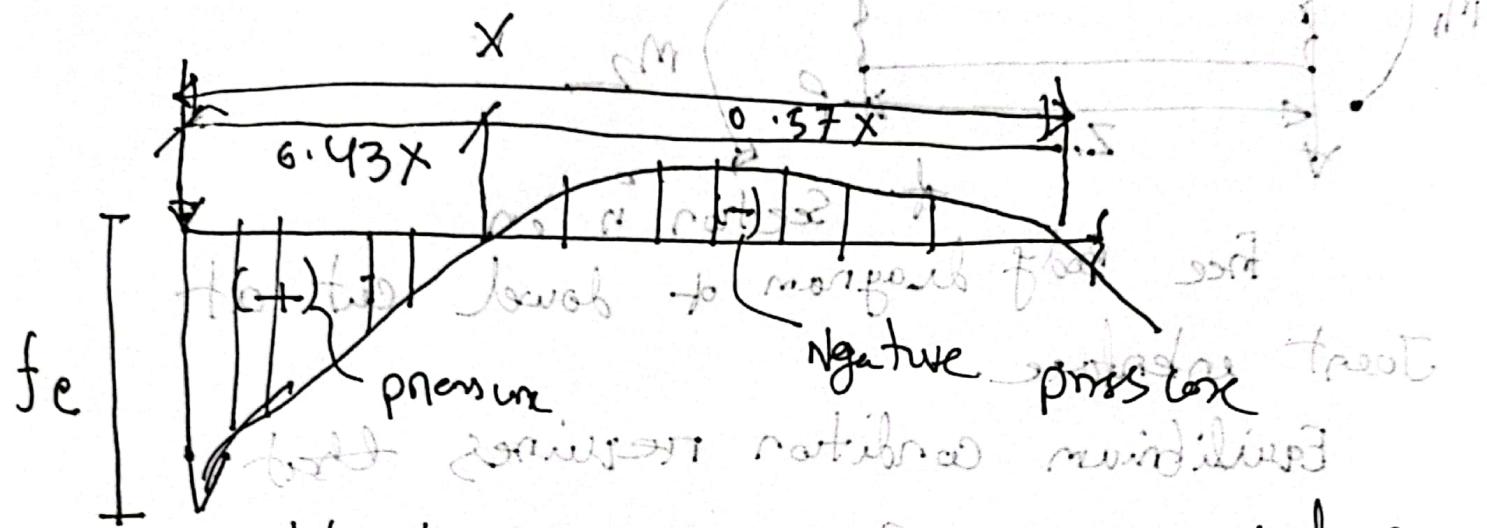
~~Portions of beam dowel being embedded~~

Concrete may be considered as being tightly bonded by ~~material~~ elastic material (of material of yielding character) and

therefore offering resistance either positive or negative deflection of the beam

thus resulting in bottom logo is

Tenovsekko and first solved this problem



Distribution of pressure on Embedded Bar

~~Inflection points caused by load applied at end.~~

$$\Sigma q = M_S$$

$$\frac{\Sigma q}{S} = M$$

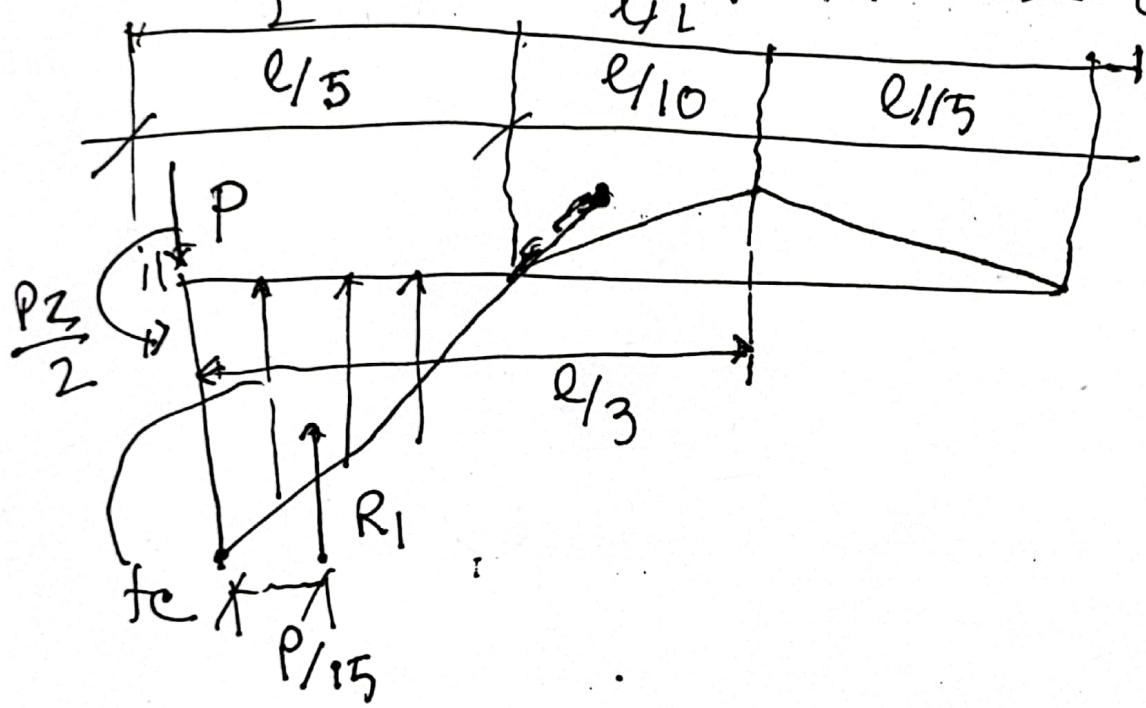
Distance from End	0	0.1x	0.2x	0.3x	0.4x	0.5x	0.6x	0.7x	0.8x	0.9x	x
Pressure Intensity $\times 10^3$	1000	793	448	177	24	-35	-42	-30	-16	-5	0

~~1000 N/mm² load applied over 0.4x length of beam to monogram~~

~~negative pressure is developed from bottom~~  
 pointwise pressure occurs over 0.4x embedment  
 negative pressure occurs over 0.6 embedment  
 negative pressure is only 4.2%  
 positive and negative pressure both can  
 be approximated by straight line very  
 closely.

pressure beyond  $X$  is negligible.

From above observation Bradbury arrived at following approximation of forced bolt active and Reactive (Free body diagram) for  $\frac{1}{2}$  embeded length of the dash bar.



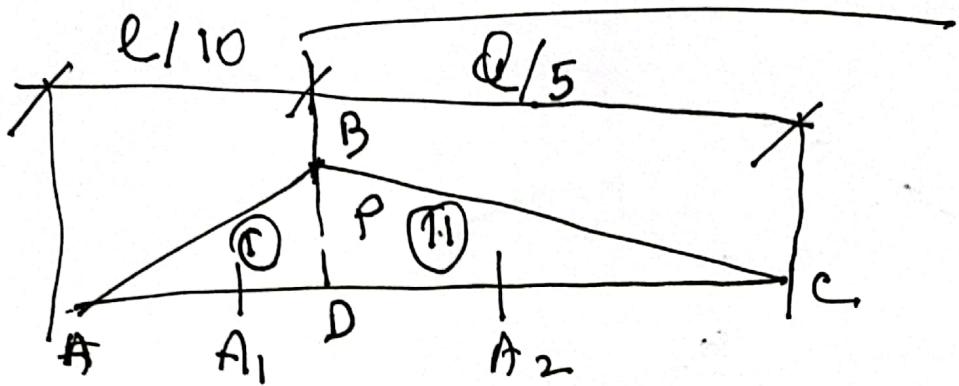
Considering above pressure diagram  
 $R_1$  = resultant positive upward pressure

$=$   
 $R_2$  = resultant of negative upward pressure

$=$

$P - 3$

Calculation Sheet → 2



Area

$$A \oplus ① \frac{1}{2} P \times \frac{l}{10}$$

C.G

$$\frac{l}{10} \times \frac{2}{3} = \frac{2l}{30}$$

Moment

~~$$= \frac{1}{2} \times P \times l \times \frac{l}{10} = \frac{Pl^2}{30}$$~~

~~$$⑪ \quad \frac{1}{2} \times P \times \frac{l}{5} = \frac{Pl}{10}$$~~

~~$$\frac{l}{10} + \frac{l}{15} = \frac{5}{30}l$$~~

~~$$= \frac{Pl}{10} \times \frac{3}{5} = \frac{Pl}{6}$$~~

~~$$① \quad \frac{1}{2} P \times \frac{l}{10} = \frac{Pl}{20}$$~~

~~$$\frac{2l}{30}$$~~

~~$$= \frac{Pl}{30}$$~~

⑪

$$\frac{1}{2} \times P \times \frac{l}{5} = \frac{Pl}{10}$$

$$\frac{l}{10} + \frac{l}{15} = \frac{3+2}{30}l = \frac{l}{6}$$

$$\frac{Pl}{60}$$

$\sum$

$$Pl \times \left( \frac{3}{20} \right)$$

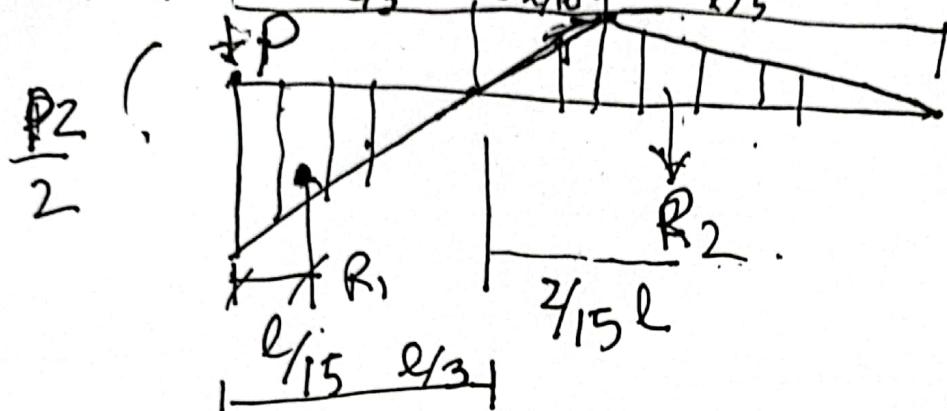
$$Pl \left( \frac{1+5}{30} \right)$$

$$C.G = \frac{\frac{Pl}{6} \times \frac{1}{50}}{Pl \times \frac{3}{20}} = l \times \left( \frac{1}{50} \times \frac{2}{3} \right) = \frac{2}{15}l$$

$$C.G \text{ from left end of embedment} = \frac{l}{5} + \frac{2}{15}l = \frac{7}{15}l$$

## Calculation Sheet - 2

Resultant acts at  $\frac{e}{15}$  negative upward pressure



Taking moment w.r.t location of  $R_1 = C \rightarrow$

$$\rightarrow R_2 \left( \frac{e}{3} - \frac{e}{15} \right) + \frac{P_z}{2} + P \times \frac{e}{15} = 0$$

$$\therefore \Rightarrow -R_2 \times \frac{4}{15} e = -\frac{P_z}{2} - \frac{Pe}{15}$$

$$\Rightarrow R_2 \times \frac{4}{15} e = P \left( -\frac{P}{15} - \frac{e}{2} + \frac{e}{15} \right)$$

$$\Rightarrow Pe \left( \frac{P}{20} \right) \times \frac{4}{15} e = \frac{Pe}{15} + \frac{P_z}{2}$$

$$\Rightarrow \frac{1}{25} Pe^2 d = \frac{P}{15} (e + 7.5z)$$

$$P = \frac{P}{15}$$

$$P = \frac{P \times 25}{15} \times \frac{e + 7.5z}{e^2 d}$$

$$\Rightarrow \frac{5P(e + 7.5z)}{3e^2 d}$$

$$P = 5$$

$$R_2 = \frac{3}{20} Pld$$

$$= \frac{3}{20} \times 4l \times \frac{P(l+7.5z)}{4l} = \frac{3}{20} P(l+7.5z)$$

from force equilibrium  $\sum f_f = 0$

$$R_1 = P + R_2$$

$$\Rightarrow R_1 = P + \frac{3}{20} P(l+7.5z)$$

$$\Rightarrow R_1 = P \left[ 1 + \frac{3}{20}(l+7.5z) \right]$$

$$= P \left( 1 + 1.5z \right)$$

$$= \frac{5P}{4l} (l + 1.5z)$$

$$R_1 = \frac{1}{2} f_c l \times \frac{l}{4l} = \frac{f_c l^2}{8l} = 1.25 f_c$$

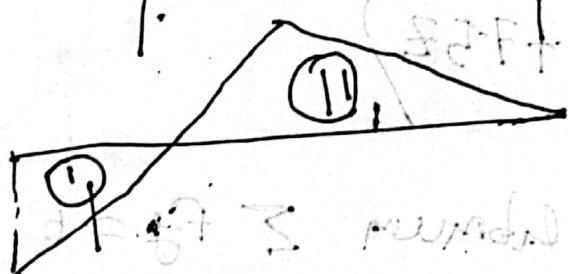
$$\frac{5P}{4l} (l + 1.5z) = \frac{f_c l^2}{8l}$$

$$\Rightarrow \frac{f_c l^2}{8l} = \frac{5P}{4l} (l + 1.5z)$$

$$\Rightarrow f_c = \frac{10}{l^2} \times \frac{5P}{4l} (l + 1.5z) = \frac{25P}{4l^3} (l + 1.5z)$$

$$f_c = \frac{25P(l+1.5z)}{2d} = 25$$

$$f_c = \frac{25P(l+1.5z)}{x_0 + 2d} = 25$$



Determination of point of zero shear

$$\text{Concrete (+) stress slope} = \frac{9f_c}{l/5} = \frac{45f_c}{l}$$

at  $x_0$  concrete shear stress =  $\frac{0.5f_c}{l} \times x_0$

total one concrete stress (+) force

$$= \frac{1}{2} \times 20 \times \frac{5f_c}{l} \times x_0 \times d = R$$

total negative (-) force (-) force at concrete

$$R_2 = \frac{3}{20} Pld = \frac{P}{4e} (l + 7.5z)$$

at  $x_0 \rightarrow$  shear force zero so positive  
and negative forces are equal

$$\frac{5}{2} \frac{x_0}{l} f_c d = \frac{P}{4e} (l + 7.5z)$$

$$\Rightarrow 9$$

$$\Rightarrow \frac{5}{2} \cdot \frac{x_0}{e} f_c \cancel{\times \frac{3}{50} \times \frac{P}{f_e}} = l$$

$$\Rightarrow \frac{x_0}{20} = \frac{3}{20} \times \frac{2}{5} \times \frac{P}{f_e}$$

~~$$\therefore = \frac{10}{50} \times \frac{P}{f_e}$$~~

$$\Rightarrow x_0 = \sqrt{\frac{3}{50} \times \frac{P}{f_e} \times l}$$

~~$$= \frac{3}{50} \times \frac{P}{f_e} \times l$$~~

~~$$\frac{P}{f_e} =$$~~

$$= \frac{\frac{3}{50} \times \frac{5P(l+7.5z)}{3cd}}{25P(l+1.5z)}$$

$$= \frac{2}{15} \times \frac{(l+7.5z)}{(l+1.5z)}$$

~~$$\therefore \sqrt{\frac{P}{f_e}}$$~~

$$= \sqrt{\frac{2}{15} \times \frac{l+7.5z}{l+1.5z}}$$

$$x_0 = \sqrt{\frac{3}{50} \times \frac{2}{15} \times \frac{l+7.5z}{l+1.5z}} l = l \times \sqrt{\frac{1}{125}}$$

P-8

$$= l \sqrt{\frac{1}{125}} \times \sqrt{\frac{l+7.5z}{l+5z}}$$

~~Find Max. unif bending moment~~

Draw SFD upto  $x_0$



$$\frac{1}{2} \times P \times \frac{l}{5} \times d = \frac{1}{16}$$



$$\frac{1}{5} - \frac{1}{5\sqrt{5}}$$

$$\frac{\sqrt{5}-1}{5\sqrt{5}}$$

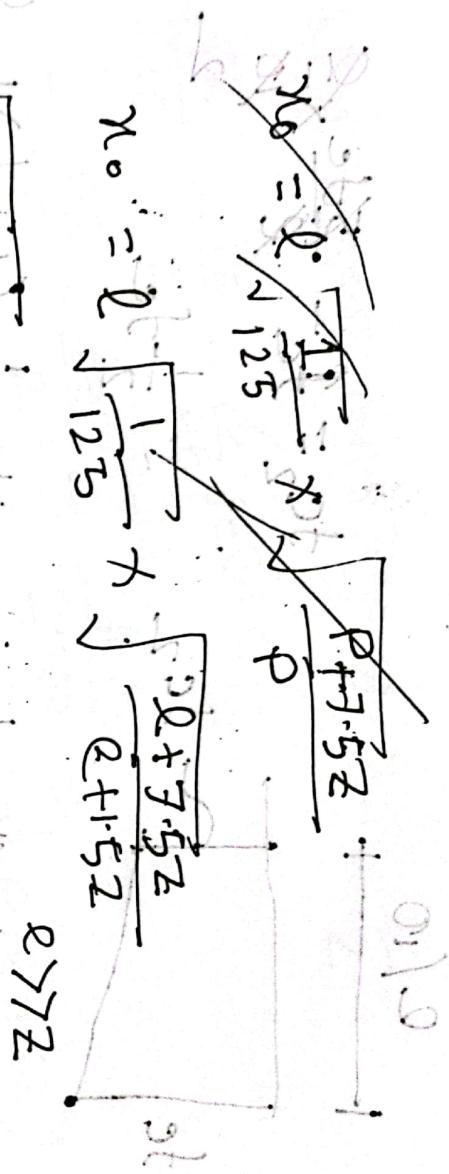
$$\frac{(\sqrt{5}-1)(\sqrt{5}+1)}{5\sqrt{5}(\sqrt{5}+1)}$$

$$\frac{4}{5} \times \frac{1}{5} = \frac{1}{5}$$

$$\frac{1}{5} - \frac{1}{11}$$

$$\frac{1}{5} - \frac{5}{11}$$

$$\frac{6}{55} = \frac{6}{55}$$



$$\frac{P}{e + 7.5z} = \frac{x_0^2}{125}$$

$$x_0^2 = \frac{P^2}{125} \times \frac{1}{e + 1.5z}$$

Assigning value of  $e$



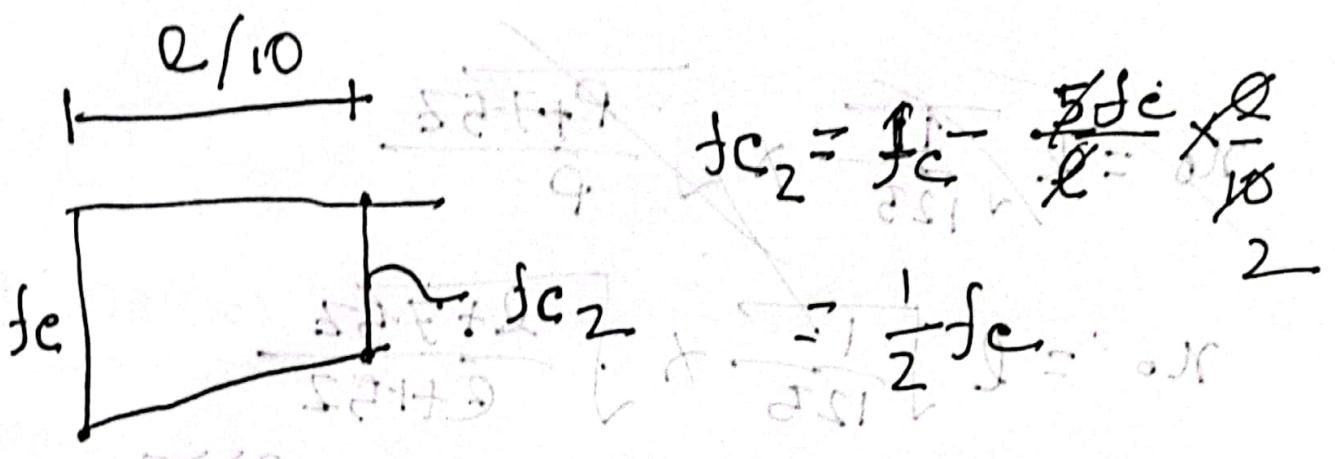
From left

From support distance of zero given

$$x = \frac{P}{5} = \frac{5}{55} \times \frac{2}{10}$$

$$P = 10$$

$N_{\text{sg}}$



Area of the trapezoid =  $\frac{1}{2} \left(1 + \frac{1}{2}\right) f_c \times \frac{l}{10}$

$$= \frac{1}{2} \times \frac{3}{2} \times \frac{l}{10} \text{ m}^2$$

$\frac{l}{10} \approx \frac{1}{3} \cdot \frac{3}{2} \cdot \frac{3}{40} \text{ field}$

Centroid of trapezoid

$$= \frac{1}{3} \left( f_c_2 + \frac{1}{2} f_c + 2 f_c \right) \times \frac{l}{10}$$

$$= \frac{1}{3} \times \frac{\frac{1}{2} + 2}{2} f_c \times \frac{l}{10}$$

$$= \frac{1}{3} \times \frac{\frac{5}{2}}{2} \times \frac{l}{10}$$

$$= \frac{1}{3} \times \frac{5}{4} \times \frac{l}{10}$$

$$= \frac{5}{12} \times \frac{l}{10}$$

Ans

$$= \frac{l}{18}$$

taking moment (cont. zero shear)

$\frac{1}{2}$

$$\frac{P_e l}{10} + \frac{P_z}{2} - \frac{\beta}{46} f_e d \times \frac{l}{18}$$
$$= \frac{P_e l}{10} + \frac{P_z}{2} - \frac{f_e d}{2400}$$
$$= \frac{P_e}{10} + \frac{P_z}{2} - \frac{e^2 d}{2400} \times \frac{25 P (l+5z)}{25 d}$$
$$= P \left( \frac{l}{10} + \frac{z}{2} - \frac{25l}{4800} - \frac{1.5 \times 25}{4800} \right)$$
$$= P f$$

$$\frac{25}{4800} = \frac{480 - 25}{4800} = \frac{455}{4800}$$
$$= P \left( \frac{l}{10} + \frac{z}{10} - \frac{455}{4800} \right)$$

p.12.

Q. 9

taking moment (cont. zero shear)  $\frac{1}{2}$

$$\frac{P_e l}{10} + \frac{P_z}{2} - \frac{3}{46} f_e d \times \frac{l}{18}$$

$$= \frac{P_e l}{10} + \frac{P_z}{2} - \frac{f_e d}{2400}$$

$$= \frac{P_e}{10} + \frac{P_z}{2} - \frac{e^2 d}{2400} \times \frac{25 P (l+5z)}{205x}$$

$$\text{transf}(1) \quad P \left( \frac{l}{10} + \frac{z}{2} \right) - \left( \frac{25l}{4800} - \frac{1.5 \times 25}{4800} \right)$$

$$= P f$$

$$\frac{1}{10} \cdot \frac{25}{4800} - \frac{480 - 25}{4800} = \frac{455}{4800}$$

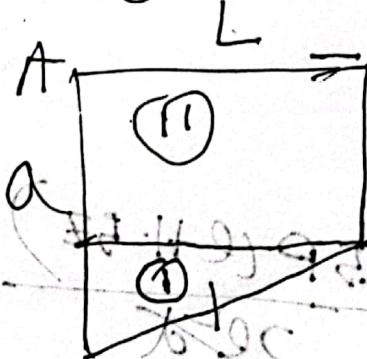
$$= P \left( \frac{l}{10} + \frac{25}{10} \right) - \frac{455}{4800}$$

$$P \cdot 12 = \text{a trans P}$$

$$\frac{1}{10} \cdot \frac{25}{4800} - \frac{480 - 25}{4800} = \frac{455}{4800}$$

# Calculation of Maximum Moment

Bending Moment



$$A = \frac{1}{2}(a+b)L$$

$$= \frac{a}{2} + \frac{b}{2}$$

c. g w.r.t AOT

Moment

	Area		
	$\frac{1}{2}(a+b)L$	$\frac{1}{3}L + \frac{a}{2}$	$= \frac{1}{6}(a+b)L^2$
(1)	$6L$	$\frac{L}{2}$	$\frac{1}{2}6L^2$
	$\left\{ \frac{1}{2}(a-b) + b \right\} L$	$\frac{1}{2}(a-b+2b)L$	$\frac{1}{6}(a-b+3b)L^2$
		$= \frac{1}{2}(a+b)L$	$\frac{1}{6}(a+2b)L^2$

$$\begin{aligned} c. g \text{ w.r.t } A &= \frac{\frac{1}{6}(a+2b)L^2}{\frac{1}{2}(a+b)L} \\ &= \frac{\frac{1}{6}}{\frac{1}{2}} \times \frac{a+2b}{a+b} \times L \\ &= \left( \frac{\frac{1}{6}}{\frac{1}{2}} \times \frac{2}{4} \right) \times \frac{a+2b}{a+b} L = \frac{1}{3} \frac{a+2b}{a+b} L \end{aligned}$$

Diagram shows a beam of length  $L/10$  with a central load  $fc$ . The beam is supported by a roller at the left end and a fixed support at the right end. The deflection curve is parabolic, starting from zero at the left end, reaching a maximum at the center, and returning to zero at the right end.

The deflection equation is given as:
 
$$C \cdot g \cdot x = \frac{1}{3} \times \left( \frac{\frac{1}{2}fc + 2fc}{\frac{1}{2}fc + fc} \right) \frac{L}{10}$$

$$\therefore C \cdot g \cdot x = \frac{1}{3} \times \left( \frac{\frac{1}{2} + 2}{\frac{1}{2} + 1} \right) \frac{L}{10}$$

$$\therefore C \cdot g \cdot x = \frac{1}{3} \times \frac{5/2}{3/2} \times \frac{L}{10}$$

The reaction force  $R$  is calculated as:
 
$$R = \frac{1}{3} \times \frac{5}{9} \times \frac{L}{10}$$

$$= \frac{5}{9} \times \frac{L}{30} = \frac{5}{27} L$$

The reaction moment  $M$  is calculated as:
 
$$M = \frac{1}{6} \left( \frac{1}{2}fc + 2fc \right) \times \frac{L}{10}$$

$$= \frac{1}{6} \times \frac{1}{2} \times \frac{5}{2}fc \times \frac{L}{100}$$

$$= \frac{5}{120} f c e d$$

Page number: P-14

Moment of Trapezoidal force about

Free end

$$M_A = \frac{1}{2} (a+b) L \times \frac{1}{3} \frac{a+2b}{a+b} \bar{x} L$$

$$\frac{1}{6} (a+2b) L^2$$

So Maximum moment:

$$M_0 = \frac{P_l}{10} + \frac{P_z}{2} - \frac{5de^2d}{1200}$$

$$\frac{5e^2d}{1200} \times de = \frac{5e^2d}{1200} \times \frac{25P(l+1.5z)}{2e^2d}$$

$$(st + st \frac{1}{2}) \frac{l}{2} = \frac{5P}{96} (l + 1.5z)$$

$$M_{0s} = \frac{P_l}{10} + \frac{P_z}{2} - \frac{5P}{96} (l + 1.5z)$$

P1-9  
P1-7

$$\begin{aligned}
 &= \frac{P}{100} \cdot \frac{P}{960} \left[ 9.6l + 480z - 50l - 75z \right] \text{et} \\
 &= \frac{P^2}{960} [46l + 405z] \text{ just part of ps} \\
 &= \frac{P \times 46}{960} \cdot [l + 8.8z] \quad \frac{9.6}{100} \cdot \frac{7}{146} = 0.0479 \\
 &\quad \frac{480}{23 P} (l + 8.8z) \quad \left\{ \begin{array}{l} \text{Brabung's Gaus} \\ \text{It is} \end{array} \right. \\
 &\quad \frac{23 P}{480} (l + 8.8z) \quad \frac{7P(l + 8.8z)}{146} \\
 &\text{dowd. is as Circular Section:}
 \end{aligned}$$

Moment of Inertia =  $\frac{\pi}{4} d^4$

$$I = \frac{\pi}{4} d^4 = 64 \quad \text{Q}$$

$$Z = \frac{I}{c} = \frac{64}{d/2} = \frac{\pi d^4}{64} \times \frac{2}{d} = \frac{\pi d^3}{32}$$

Maximum moment that can be carried by a dowd section

$$f_s \geq \frac{M_{max}}{S.F. Z}$$

$$\Rightarrow M_{max} = f_s Z = f_s \times \frac{\pi d^3}{32}$$

equating this to  $M_o$  we get

$$\frac{M_o}{S.F. Z} = \frac{7P}{146} (l + 8.8Z) = f_s \frac{\pi d^3}{32}$$

$$\Rightarrow P = \frac{146 \times f_s \times \pi d^3}{7 \times 32} \times \frac{l}{l + 8.8Z}$$

$$\frac{(58.8+3) 95}{2P} = \left\{ \frac{146 \times \pi}{7 \times 32} \times \frac{d^3 f_s}{l + 8.8Z} \right\} \rightarrow 2.04 \text{ N/mm}^2$$

(Broad flange approx)

$$P_f \approx \frac{2d^3 f_s}{l + 8.8Z} \text{ ut. w/ 2.0}$$

$$\frac{E_{b,II}}{E_c} = \frac{E_b}{E_c} \times \frac{P_{b,II}}{P_0} = \frac{1}{\sqrt{b}} = \frac{I}{J} = 5$$

$E_c$   $\rightarrow$   $E_c$  tensile test  $\rightarrow$   $E_c$   $\rightarrow$   $E_c$   $\rightarrow$   $E_c$   $\rightarrow$   $E_c$

W.H.

Maximum Shear Stress of a Concrete Bar

$$= \frac{\pi}{4} d^2 f_s$$

$$P = 0.785 d^2 f_s' - \textcircled{7}$$

From Bearing Concrete:

$$f_c = \frac{25 P (l + 1.5z)}{2d^2}$$

$$\begin{aligned} P &= f_c \times 2d^2 \times \frac{1}{25} \times \frac{1}{l + 1.5z} \\ &= \left(\frac{2}{25}\right) \times \frac{f_c d^2}{l + 1.5z} \\ &= \frac{1}{12.5} \times \frac{f_c d^2}{l + 1.5z} \end{aligned}$$

Eq ⑨ says increasing dowel length increases capacity in concrete bearing parabolically while Eq 8 tell for flat bending capacity of dowel decreases linearly by per bohally.

$$\frac{321+2}{32.849} \times P = 1.8 \times 62 = 1$$

We need to combine these two similarly

$$\frac{2d^3 fs}{l + 8.8z} = \frac{1}{12.5} \times \frac{\text{field}}{l + 1.5z}$$

$$\Rightarrow \frac{4d^3 fs}{l + 8.8z} = \frac{(1+2)4.25}{10} = 0.9$$

$$\Rightarrow \frac{d^3 fs}{l + 8.8z} = \frac{1}{25} \times \frac{\text{field}}{l + 1.5z}$$

$$\Rightarrow 25 \times \frac{fs}{fc} \times \frac{d^3}{d} \times \frac{l + 1.5z}{l + 8.8z} = l$$

$$\Rightarrow l = 5 \times \sqrt{\frac{fs}{fc}} \times \sqrt{\frac{l + 1.5z}{l + 8.8z}}$$

$$l = 5d \times \sqrt{\frac{fs}{fc}} \times \sqrt{\frac{l + 1.5z}{l + 8.8z}}$$

Letting dowel length for equal capacity in both

$$l = 5d \times \sqrt{\frac{fs}{fc}} \times \sqrt{\frac{l + 1.5z}{l + 8.8z}}$$

$$\text{If } l \gg z \quad \sqrt{\frac{l+1.5z}{l+8.8z}} \approx 1$$

$$l = d\sqrt{2}$$

$$= 5d \sqrt{\frac{f_s}{f_c}}$$

Actual length of dowel bar to be taken

Actual load per unit length of dowel bar

Proper doweling means each slab end will carry  $\frac{1}{2}$  of the total system load at least.

Main load due to concentrated load is main at point of application. According to W. Westergaard theory moment is approximately  $\frac{1}{2}$  at a distance  $9/15$  times from load.

Reasonably close spacers of dowels  
not more than 24 inch any wise

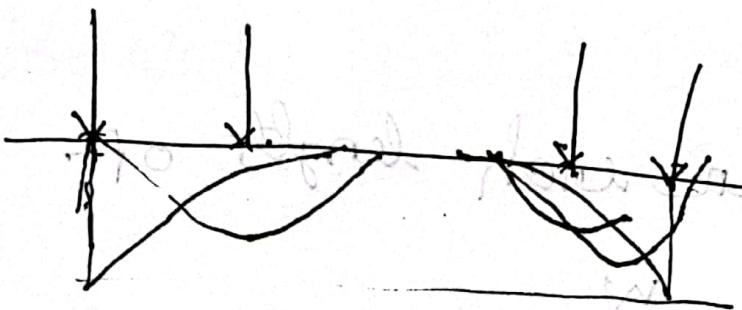
Total no of dowels  $\geq$  ~~All loads applied at joints~~  
~~Maximum Load Transfer capacity of a single~~

~~has less mass of load~~  
~~Maximum wheel load~~  $\leq$  ~~friction force~~

Towbar hitch

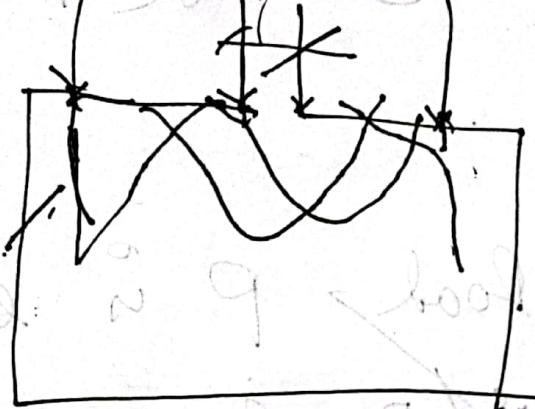
→ load distribution of each load must be same  
→ two long front suspension with  
→ two short front hub mounting with luggage  
→ two passenger vehicle [2 axle load]  
→ load must be same

0.5 ft 21 ft



(a)

~~Case A~~ Case A  
Vertex on  
Edge = 0.6



Fully uniform  
distribution of  
load over  
a width of  
8 feet (two-  
celled box)

Uniform distribution of load over width  
of 16 feet of 4 celled boxes:

So It is reasonable to assume

So if Two celled load is uniformly distributed  
over a large width shall be  
a reasonable estimate.

To get the area of abutments = 22  
feet <sup>p-22</sup>

Applied load per unit length of

Joint would be  $\frac{W}{66}$

$\frac{1}{2}$  of joint load shall be transferred by dowels.

If  $w$  is wheel load,  $P$  is the single load transfer capacity of single dowel bar.

Now  $S = \text{load borne by } P \rightarrow \text{tensile stress}$

Stresses

Allowable Stresses:

$f_e =$  allowable elastic stress of concrete and is

$$= 0.4 f'_c$$

$f_s =$  allowable tensile stress of steel.

$f_s = 0.75 f_y$  [working stress of steel]

Required Maximum concrete cover for Dowel bars.

$$\text{Dowel bar} = D = 3d$$

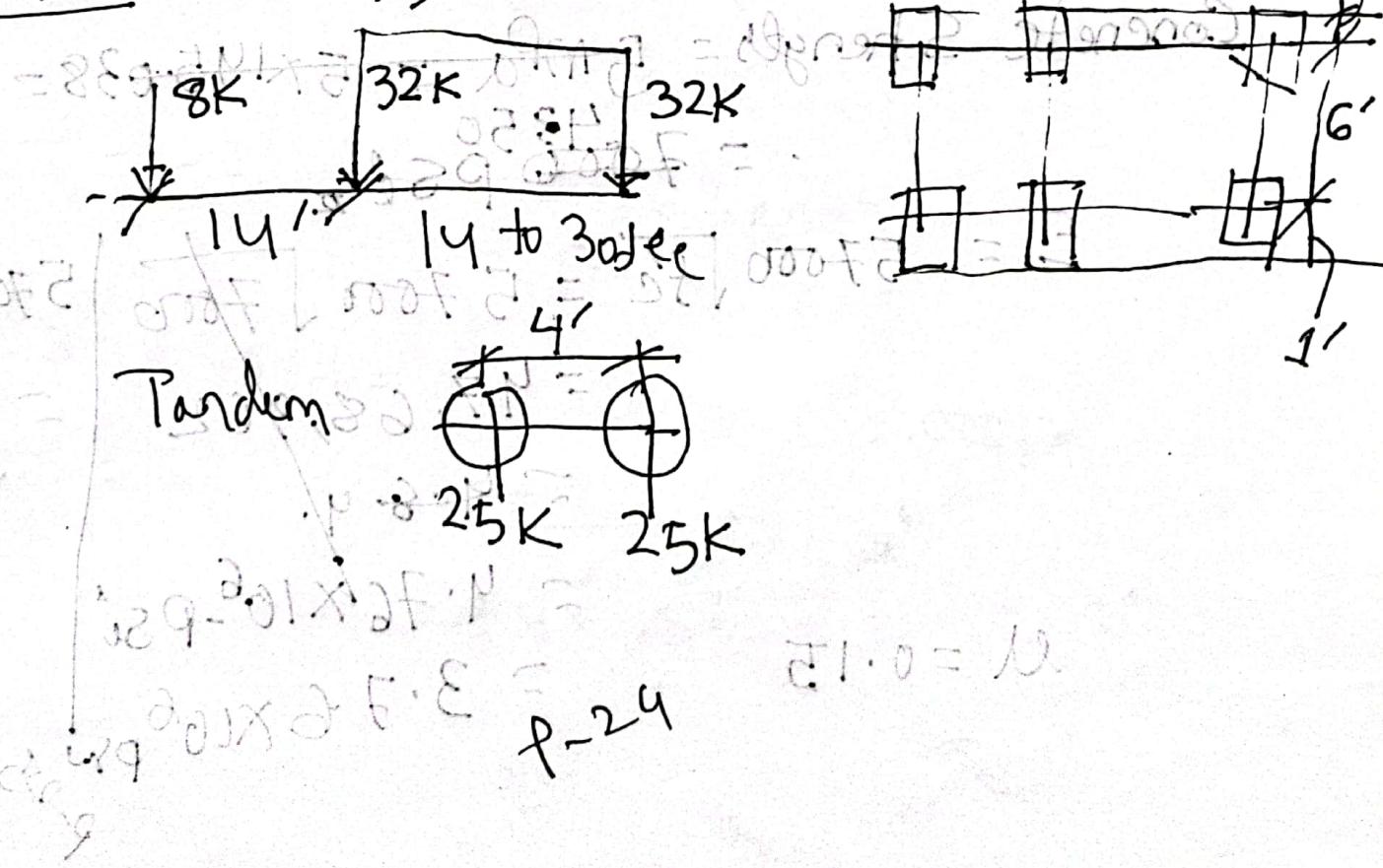
$D$  = distance of nearest concrete face from center of dowel bar.

$d$  = diameter of dowel bar.

MSE = diameter of dowel bar.

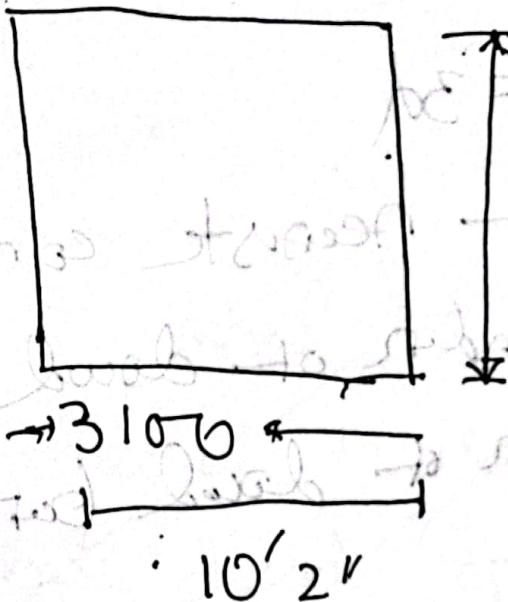
Loads

HL-93



Large load 0.64 kN found from step 3

## Drawing Problems :



Maximum Load

4100

≈ 13' 5"

Thickness h  
= 30cm

≈ 12 inch

Concrete Strength = ~~5 MPa~~ ~~5 × 145.038 =~~  
~~= 7000 psi~~

$$E_i = 57000 \sqrt{f'_c} = 57000 \sqrt{7000}$$

~~≈ 4.7, 68, 962~~  
~~≈ 4.84.~~  
~~≈ 4.76 × 10^6 - psi~~  
~~≈ 3.76 × 10^6 psi~~

~~57000 √4350~~  
~~= 3.76~~  
~~× 10^6~~  
~~psi~~

$$\mu = 0.15$$

CBR value = 25

$$K = 28.6926 \text{ CBR}^{0.7288}$$

$$= 34350 \text{ PSI/cm}^3$$

Radius of relative stiffness = 100 mm

$$= 4 \sqrt{\frac{4.76 \times 10^6 \times 12^3}{1.2 \times 350 \times (1 \mu^L)}}$$

$$= 4 \sqrt{\frac{8225.28 \times 10^6}{1.2 \times 350 \times 0.9775}}$$

$$= 4 \sqrt{\frac{8225.28 \times 10^6}{4105.5}}$$

$$= \sqrt{2.0035 \times 10^6}$$

$$= 37.63 \text{ cm}$$

$f_s = 0.45 f_y = 0.45 \times 60000 = 27,000 \text{ psi}$

$$f_s = 0.75 \times 60000 = 45,000 \text{ psi}$$

$$f_c = 0.4 f'_c = 0.4 \times 4350 = 1740 \text{ psi}$$

For 13' 5" for 4100 mm (13' 5") Lane width = 161 inch Lane width

For One Axle may be considered

$$\text{Axle load UDL} = \frac{32 \times 1000}{161 \text{ in}} = 200 \text{ #/inch}$$

$$\text{Tandem load} = \frac{25 \times 1000}{4 \times 12} = \frac{2500}{48} = 520 \text{ #/inch}$$

$$\text{Lane Load} = 0.64 \text{ klf}$$

$$= \frac{0.64 \times 1000}{12} = 53 \text{ #/inch}$$

$$\text{Design live load} = 520 \text{ #/inch}$$

Considering Impact factor 1.25% 25%

$$\text{Design live load} = 1.25 \times 520 = 650 \text{ #/inch}$$

Dowel Capacity:  $\phi 3.2 \text{ cm} \text{ dowel} = 1.26 \text{ inch}^2$

$$\phi 3.2 \text{ cm dowel} = 1.26 \text{ inch}^2$$

2' 28

$$L_d = 5d \sqrt{\frac{f_s}{f_c}} \times \sqrt{\frac{L_d + 1.5d}{L_d + 8.8d}}$$

for flanged beam

$\delta = 20\text{ cm} \approx 1\text{ inch}$

$$L_d = 1.255 \times 1.25 \sqrt{\frac{42000}{1740}} \times \sqrt{\frac{L_d + 1.5}{L_d + 8.8}}$$

$$= 6.25 \times 4.91 \sqrt{\frac{L_d + 1.5}{L_d + 8.8}}$$

$$= 30.68 \sqrt{\frac{L_d + 1.5}{L_d + 8.8}} = 30.68 \text{ inches}$$

Needs trial

cos we approximate equation

$$L_d = 5d \sqrt{\frac{f_s}{f_c}} = 2$$

$$= 5d \times \sqrt{\frac{42000}{1740}} = 5d \times 4.91 = 24.56d$$

$$24.56d = 26d \Rightarrow 32.76 = 33 \text{ inches}$$

$$L_d + 33 + 1 = 34 \text{ inches} \quad (1 \text{ inch for joint})$$

Design capacity in shear

$$P_s = 0.785 d^2 f_s' \delta$$

$$= 0.785 \times 1.26^2 \times 27000$$

$$= 33650 \text{ ft-lb}$$

f-28

Dowel capacity in bending

$$P_f = \frac{2d^3 fs}{8.8 + b} = \frac{2 \times 1.26 \times 420}{8.8 + b}$$
$$= \frac{34.82 \times 1.26}{8.8 + b} = 3926 \text{ N}$$

Dowel Spans = ~~Maximum live load~~

where dowage = ~~Minimum load capacity~~  
= ~~best shear~~

$$S = \frac{\text{Minimum Dowel Capacity}}{\text{Maximum live load}}$$
$$b \times P_c = 104 \times 6.2 = \frac{650}{3926}$$

$$\therefore S = \frac{650}{3926} = 6.04$$

(Ansatz mit Ansatz)  $\therefore$   $S = 1.26$  inch

So we have 1.26 inch dowel @ 6.04 c/c

3.25 inches dowel @ 15 cm c/c

$$650 \times 1.26 \times 6.04 =$$

$$0.06288 =$$

1-29

Ans

Reducing

2 tandem over 13'

$$\frac{2 \times 25 \times 2}{161} = 621 \text{#/each}$$

50% shall be transfer by dowel  
So design tandem load

$$= 312 \text{#/each}$$

Maximum dowel spacing

$$= \frac{3926}{401} = 9.79 \text{ inch C/C}$$

$$= \frac{3926}{312} = 12.58 \text{ in C/C}$$

f-30