

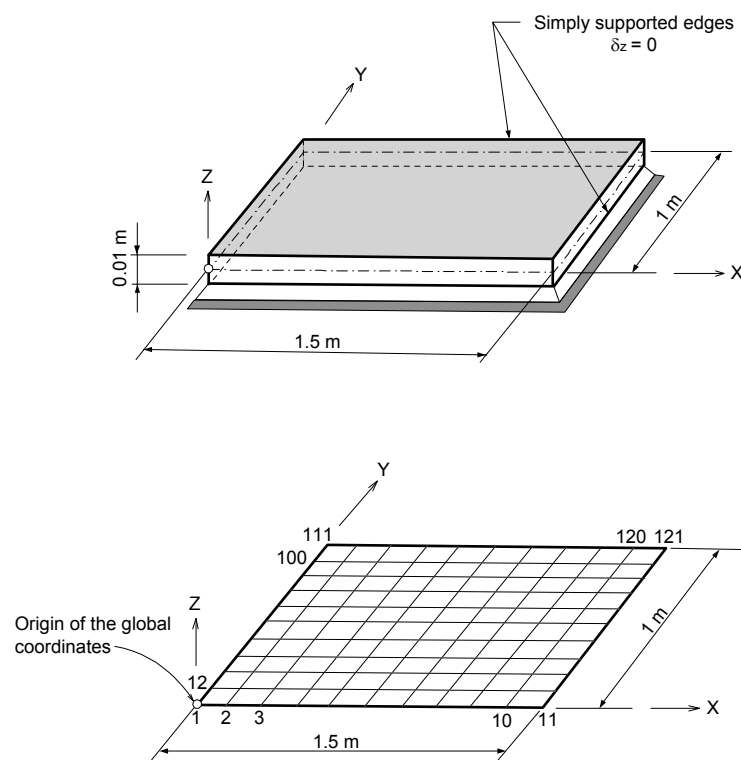
# Eigen-11

## Title

Eigenvalue analysis of a thin simply supported rectangular plate

## Description

Calculate the natural frequencies of a simply supported plate.



*Structural geometry and analysis model*

## MODEL

### *Analysis Type*

3-D eigenvalue analysis

### *Unit System*

m, tf

### *Dimension*

Length 1.5m   Width 1m   Thickness 0.01m

### *Element*

Plate element (Thin type)

### *Material*

Modulus of elasticity    $E = 2.1 \times 10^5$  MPa

Poisson's ratio    $\nu = 0.3$

Weight density    $\gamma = 7.8$  tf/m<sup>3</sup>

### *Sectional Property*

Rectangular cross-section: width 1 m,   thickness 0.01 m

### *Boundary Condition*

All edges are simply supported

### *Analysis Case*

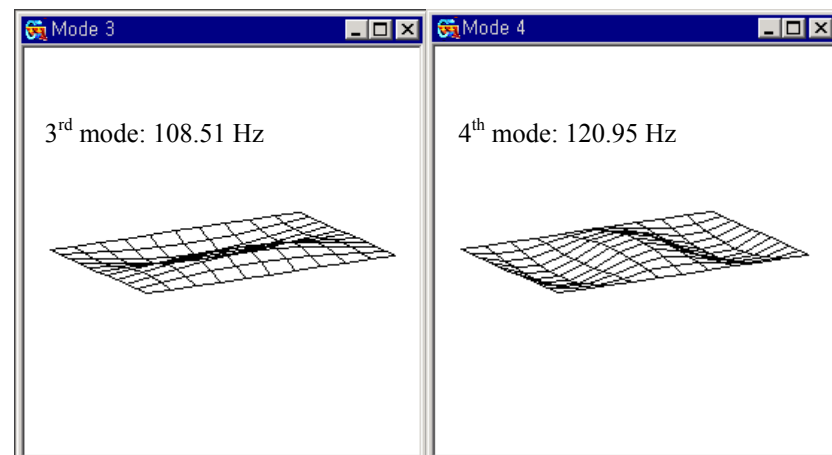
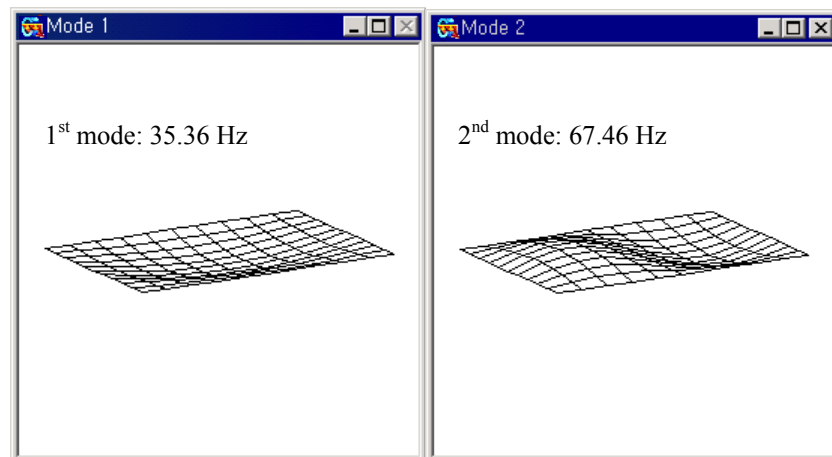
Self weight is converted to nodal masses automatically. (Gravity acceleration  $g=9.806\text{m/sec}^2$ )

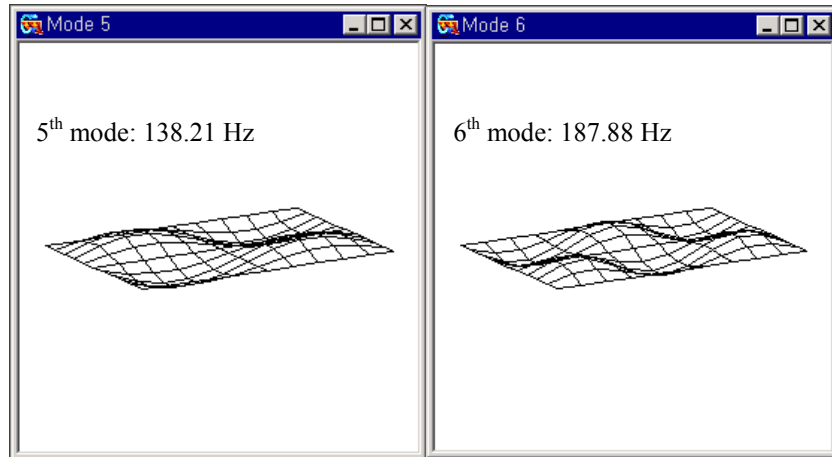
Number of natural frequencies to be computed = 6

## Results

EIGENVALUE ANALYSIS						
Mode No	Frequency		Period (sec)	Tolerance		
	(rad/sec)	(cycle/sec)				
1	222,14	35,36	0,03	0,0000e+000		
2	423,86	67,46	0,01	3,2400e-016		
3	681,80	108,51	0,01	3,0428e-014		
4	759,96	120,95	0,01	1,4735e-013		
5	868,41	138,21	0,01	1,7462e-011		
6	1180,46	187,88	0,01	2,6212e-007		

*The natural frequencies*





*The mode shapes*

## Comparison of Results

Unit: Hz			
Results	Mode	Theoretical	MIDAS/Civil
Frequency	1 <sup>st</sup>	35.63	35.36
	2 <sup>nd</sup>	68.51	67.46
	3 <sup>rd</sup>	109.62	108.51
	4 <sup>th</sup>	123.32	120.95
	5 <sup>th</sup>	142.51	138.21
	6 <sup>th</sup>	197.32	187.88

## References

“*Guide de Validation des Progiciels de Calcul de Structures*”, SFM, Afnor Technique, France, 1990.

Barton, M. V. “*Vibration of Rectangular and Skew Cantilever Plate*”, J. Appl. Mech. 18, 129-134.