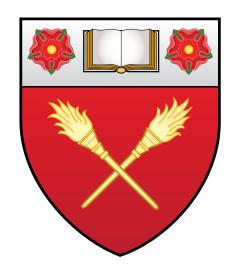
Dislocation Based Modelling of Fusion Relevant Materials.





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Dedication

"Ohana significa familia, y tu familia nunca te abandona, ni te olvida."

"Ohana means family, family means nobody gets left behind, or forgotten."

— Stitch, Lilo & Stitch

Malle santa, cuando te hablé desde el trabajo para darte la noticia me dijiste que siempre lo supiste. Recuerdo que lloraste en el teléfono, se te cerró la garganta y a mí también. A diferencia de otras veces, no pudimos platicar mucho porque nos quedamos sin aliento. Así que le hablaste a abuelita Raquel mientras le hablé a papá. Madre, David y yo te debemos tanto que no podemos repagar en mil vidas, pero ten por seguro que eres nuestro ejemplo a seguir. Nuestra madre chingona y chambeadora, solo hay una y como ella no hay ninguna. Sin tí el mundo sería un lugar más cruel y pobre, no merece un ángel tan grande y puro como tú.

Dad, "Igualito que tu jefe, wey." con tu risa característica fue lo primero que dijiste cuando te avisé "Sí dad, igualito que mi jefe." entre risas respondí. Como siempre, no platicamos mucho pero esta vez porque le querías avisar a abuelita Teté y a mis tíos, y yo le quería avisar a David. Pensé que seguirías aquí para carcajearte al verme en la túnica ridícula, así como lo hicimos nosotros cuando te vimos a tí en la tuya. Escribo esta dedicatoria antes de acabar porque una promesa es una promesa y esta madre la voy a acabar. No te tendré para pedirte consejos y contarte mis avances y tropiezos. Pero a veces pretendo que me escuchas mientras intento comprender o arreglar algo. Te queremos y te extrañamos muchísimo. Buenas noches, dad.

David, "Te mamaste we." me dijiste cuando me abrazaste al llegar a casa después del trabajo el día que me aceptaron. Me has hecho un chingo de falta ojete, te extraño mucho we. Perdón por no hablar tan seguido, pero me duele colgar we. No estoy tan chisqueado como mamá pero siento feo cuando terminamos de platicar. Me gusta mucho ver tus streams porque me recuerda un poco a sentarme a tu lado a verte jugar... o a cuando veíamos Twitch juntos y veíamos a TB missing legal.

Esto es para mi Ohana por sangre y por elección. Ustedes creyeron en mí cuando yo no lo hacía. Me empujaron a ser mejor. Me extendieron la mano cuando nadie más lo hizo. Me hicieron reir cuando solo sabía llorar. Gracias por hacerme quien soy.

This is for my Ohana by blood and by choice. You believed in me when I did not. You pushed me to be better. Offered me a helping hand when nobody would. Made me laugh when all I knew was sorrow. Thank you for making me who I am.

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"You can always judge a man by the quality of his enemies."

— Oscar Wilde

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I guess I'll content myself with being judged unfavourably by the world.

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Outline

- 1. Lit review
- 2. DDD-FEM
- 3. Analytic tractions
 - (a) Traction errors
 - (b) Reaction force errors
 - (c) Simulations
 - (d) Recommendations and Conclusions
- 4. Parallelising tractions
 - (a) Algorithms
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- 5. EasyDD v2.0
 - (a) Adaptive integration
 - (b) New mobility law
 - (c) Surface velocities
 - (d) Collisions
 - (e) Code redesign
- 6. Future work
 - (a) Julia redesign
 - (b) Preliminary comparisons
- 7. Conclusions

Preface

0.1 Notation

For the sake of clarity the following notation conventions have been used:

- Tensors are denoted by sans serif bold italics, T.
- Matrices are denoted by serif bold roman, M.
- Vectors are denoted by serif bold italics, V.
- Scalars are denoted by serif italics, S.
- Host variables are denoted by $^{\rm h}a$.
- Global device variables are denoted by $^{\rm d}a$.
- Thread variables are denoted by ^ta.

0.2 Conventions

All angles are given in radians unless stated otherwise. All pseudo-code is C-style and assuming C conventions (row-order, 0-start indexing, memory allocation).

Elements, be they Surface Elements (SEs), Finite Elements (FEs), dislocation line segments or else, are denoted by e and their total as E. Similarly nodes are denoted by n and their total N. Parallel indices are denoted by idx to distinguish them from regular indices/counters i, j, k. For disambiguation purposes the exponential function will de denoted as $\exp(x)$.

0.3 Typesetting

This document was typeset using a custom-made LaTeX document class created by the author, compiled with XaLaTeX, and the bibliography was produced with BibTeX. The custom document class can be found in https://github.com/dcelisgarza/latex_temp

0.4 Diagrams

All diagrams are vector graphics drawn using the open-source image editing software InkScape for its ease of use, elegant simplicity, high quality outputs, and "Draw Freely" philosophy.

Introduction

- 1.1 Section 1
- 1.1.1 Subsection 1

Coupling Discrete Dislocation Dynamics to Finite Element Methods

2.1 Superposition Scheme

Coupling Discrete Dislocation Dynamics (DDD) to Finite Element Methods (FEMs) [1] is important to properly simulate micromechanical tests because DDD provides us with a more precise set of inputs and greater granularity for solving the FE problem. This can be achieved by using a so-called superposition scheme fig. 2.1 that enables the independent solution of both problems, whilst feeding information from one to the other in a continuous feedback loop.

2.2 Extracting Surface Nodes

The FEM coupler arranges the nodes starting on the xz-plane where $y=0,\ldots,n\mathrm{d}y,\ n\in\mathbb{N}$. However in order to couple DDD to FEM we only require the surface nodes where displacements are not calculated. Because we're working with rectangular prisms, we can easily pick out the surface nodes using a search algorithm with a logical mask. MATLAB and Fortran provide vector intrinsics that allow one to do so. Figure 2.2a illustrates only the surface nodes according to our implementation's node arrangement—which is the xz-plane going from $y=y_{\min}\to y=y_{\max}$. However, due to the nature of the analytical solutions in chapter 3, we need all the surface nodes of the rectangular faces for which there are no displacements. This means that edge nodes are shared between 2 adjacent faces and corner nodes between 3 adjacent faces. In order to properly apply the logical mask to find only the surface nodes need to know that each FEs' nodes are numbered according to

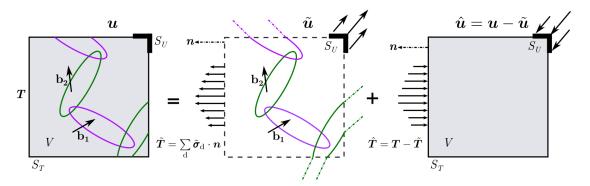
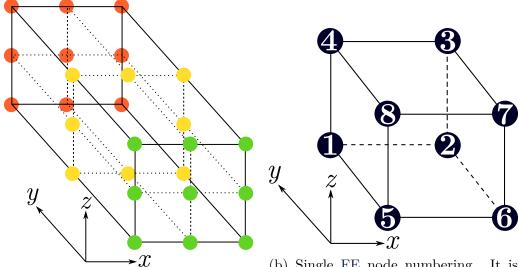


Figure 2.1: The dislocation ensemble in a volume V is bounded by surface S. First, the traction field $\sum_{\mathbf{d}} \tilde{\boldsymbol{\sigma}}_{\mathbf{d}}$ due to the dislocation ensemble is evaluated at the surface. Then, a traditional FEM or Boundary Element Method (BEM) calculates the image traction field $\hat{\boldsymbol{\sigma}} \times \boldsymbol{n}$. Which is then fed back to the DDD problem to evolve the dislocation positions and repeat the cycle. Image edited from [1].



(a) Arrangement of the surface nodes of our FE model. FE nodes are arranged in chunks of $\Delta x \Delta z$ nodes in our implementation, but we only want the surface nodes.

(b) Single FE node numbering. It is necessary to know which FE plane corresponds to which node labels. This lets us design the auxiliary matrix that selects nodes according to the planes we want to extract.

Figure 2.2: FE node arrangement for coupling to DDD.

fig. 2.2b.

Using fig. 2.2 one can work out which nodes are of interest to whichever surface is being extracted. The ordering of the nodes in the final array will depend on the definition of the problems in chapters 3 and 4.

Node selection remains an expensive operation and minimising array indexing is of the utmost importance for the best perfomance. Selecting nodes in the traditional sense, i.e. with code branching such as if statements or case selection is unmantainable, verbose and very prone to mistakes. The issue was solved by introducing an auxiliary matrix which defines various parameters that aid node selection and greatly reduces code size, improves readability, and eliminates the need for code branching. The matrix can be constructed utilising fig. 2.2 in order to know which nodes correspond to which FE planes. The p^{th} column of the matrix¹ corresponds to the p^{th} plane (according to an arbitrary plane numbering) and is defined as,

$$\boldsymbol{V_p}^{\mathsf{T}} = \begin{bmatrix} L_{1p} & L_{2p} & \cdots & L_{Np} & A_p & C_p \end{bmatrix}, \tag{2.1}$$

where L_{np} is the numeric label for node n as given by fig. 2.2, A_p is the area of the plane, and C_p is the numeric label of the orthogonal coordinate to the plane $C_p = 1, 2, 3$ for the x, y, z coordinates respectively. A_p lets us segment our output and transitional arrays so that the only data being modified is that which corresponds to the correct plane and C_p lets us know which coordinate we must use in our selection criteria. Using our particular node labelling scheme (with dimensions Δx , Δy , Δz respectively in the x, y, z directions), the matrix is defined as,

$$\mathbf{V} = \begin{bmatrix} 5 & 2 & 6 & 1 & 5 & 4 \\ 1 & 6 & 5 & 2 & 6 & 3 \\ 8 & 3 & 7 & 4 & 1 & 8 \\ 4 & 7 & 8 & 3 & 2 & 7 \\ \Delta y \Delta z & \Delta y \Delta z & \Delta x \Delta z & \Delta x \Delta z & \Delta x \Delta y & \Delta x \Delta y \\ 1 & 1 & 2 & 2 & 3 & 3 \end{bmatrix}.$$
 (2.2)

The information codified in eq. (2.2) lets us index and process only the necessary columns to extract the surface nodes we're interested in. The advantage of this setup over a naïve implementation is that it can be relatively easily expanded, maintained, and is general enough that it lends itself to a variety of selection criteria. The columns from left to right (1 to 6) represent: face $1 \equiv \min(x)$, yz-

 $^{^1\}mathrm{MATLAB}$ uses column-major ordering, so this gives us the best performance for vectorised code.

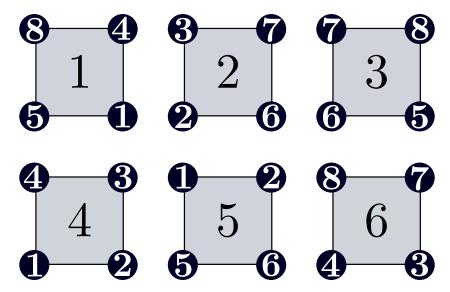


Figure 2.3: Self-consistent, chirality preserving surface planes. Plane and node numbering according to the columns of eq. (2.2) and fig. 2.2.

plane; face $2 \equiv \max(x)$, yz-plane; face $3 \equiv \min(y)$, xz-plane; face $4 \equiv \max(y)$, xz-plane; face $5 \equiv \min(z)$, xy-plane; face $6 \equiv \max(z)$, xy-plane.

It is important to make sure the nodes are extracted in a self consistent manner because the problem definitions in chapters 3 and 4 assume a specific node ordering. In particular, the labelling chirality must remain the same. This is due to the fact that the problems in chapters 3 and 4 utilise internal coordinates derived from a set of basis vectors, and scalar projections onto them. If the relationship between the vectors is kept, but the chirality is different (opposite) the calculated quantities will have the wrong sign. If however the relationship between the vectors is different to the problem formulation, the program will crash due to division by 0. Self-consistency was achieved by placing an imaginary observer inside the FE model facing the $\min(x)$, yz-plane (face 1) and rotating it to view all 6 planes according to fig. 2.3.

2.3 Mapping Forces

 \hat{F} only cares about the global node numbers, of which there are M. The columns of N_L , F_N represent the 4 nodes of a SE, the rows represent the SE they are part of. The column vector \boldsymbol{x} represents the x, y, z-coordinates of the node-element or global node corresponding to their subscript. The column vector γ_t is the list

Algorithm 2.1 If \hat{F} 's columns are arranged the same way as γ_t .

- 1: ▷ Loop through the array containing the node labels of the relevant surface nodes.
- 2: for i = 0; $i < \text{length}(\gamma_t)$; i + + do
- 3: ▷ Save the global node label for the current iteration.
- 4: $n \leftarrow \gamma_t[i]$
- 5: Use the node label to find a vector with the linearised index of all surface nodes whose labels correspond to the node whose forces we want to pass on to the FEM coupler.
- 6: $L \leftarrow \text{find}(\mathbf{N_L} == n)$
- 7: ▷ Loop over coordinates.
- 8: **for** k = 0; k < 3; k + + do
- 9: Arrows Use global node label vector to index the force array from the analytical force calculation due to dislocations on surface nodes. Multiplied by 3 because there are three coordinates per node. We sum the forces from the analytical calculation because the same global node can be part of multiple surface elements. We add k because the x, y, z coordinates are consecutively stored in $\mathbf{F_n}$.

10:
$$\mathbf{\hat{F}}[3 \times \mathbf{L} + k] \leftarrow \mathbf{\hat{F}}[3 \times \mathbf{L} + k] + \sum \mathbf{F_n}[3 \times \mathbf{L} + k]$$

- 11: end for
- 12: end for

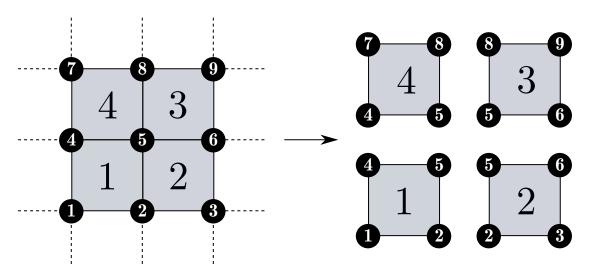


Figure 2.4: FE nodes shared by multiple SEs.

of global nodes for which the forces induced by dislocations must be calculated.

$$\mathbf{x}_{e,n}^{\mathsf{T}} \equiv \begin{bmatrix} x_{en} & y_{en} & z_{en} \end{bmatrix}, \quad \hat{\mathbf{x}}_{n}^{\mathsf{T}} \equiv \begin{bmatrix} x_{n} & y_{n} & z_{n} \end{bmatrix}$$
 (2.3)

$$\mathbf{N_{L}} = \begin{bmatrix} l_{1,1} & l_{1,2} & l_{1,4} & l_{1,5} \\ l_{2,2} & l_{2,3} & l_{2,5} & l_{2,6} \\ l_{3,5} & l_{3,6} & l_{3,8} & l_{3,9} \\ l_{4,4} & l_{4,5} & l_{4,7} & l_{4,8} \end{bmatrix}, \quad \boldsymbol{\gamma} = \begin{bmatrix} l_{1} \\ l_{2} \\ \vdots \\ l_{9} \end{bmatrix}$$

$$\mathbf{F_{e}} = \begin{bmatrix} \boldsymbol{x_{1,1}} & \boldsymbol{x_{1,2}} & \boldsymbol{x_{1,3}} & \boldsymbol{x_{1,4}} \\ \boldsymbol{x_{2,1}} & \boldsymbol{x_{2,2}} & \boldsymbol{x_{2,3}} & \boldsymbol{x_{2,4}} \\ \boldsymbol{x_{3,1}} & \boldsymbol{x_{3,2}} & \boldsymbol{x_{3,3}} & \boldsymbol{x_{3,4}} \\ \boldsymbol{x_{4,1}} & \boldsymbol{x_{4,2}} & \boldsymbol{x_{4,3}} & \boldsymbol{x_{4,4}} \end{bmatrix}, \quad \hat{\boldsymbol{F}} = \begin{bmatrix} \hat{\boldsymbol{x}_{1}} \\ \hat{\boldsymbol{x}_{2}} \\ \hat{\boldsymbol{x}_{3}} \\ \hat{\boldsymbol{x}_{4}} \end{bmatrix}. \quad (2.5)$$

$$\mathbf{F}_{\mathbf{e}} = \begin{bmatrix} \mathbf{x}_{1,1} & \mathbf{x}_{1,2} & \mathbf{x}_{1,3} & \mathbf{x}_{1,4} \\ \mathbf{x}_{2,1} & \mathbf{x}_{2,2} & \mathbf{x}_{2,3} & \mathbf{x}_{2,4} \\ \mathbf{x}_{3,1} & \mathbf{x}_{3,2} & \mathbf{x}_{3,3} & \mathbf{x}_{3,4} \\ \mathbf{x}_{4,1} & \mathbf{x}_{4,2} & \mathbf{x}_{4,3} & \mathbf{x}_{4,4} \end{bmatrix}, \quad \hat{\mathbf{F}} = \begin{bmatrix} \hat{\mathbf{x}}_{1} \\ \hat{\mathbf{x}}_{2} \\ \hat{\mathbf{x}}_{3} \\ \hat{\mathbf{x}}_{4} \end{bmatrix}.$$
(2.5)

Analytical Forces Induced by Dislocations on Linear Rectangular Surface Elements

3.1 Forces Exerted by a Dislocation Line Segment on Linear Rectangular Surface Elements

Coupling DDD to FEM requires the traction field $\sigma^{\infty}(x) \cdot n(x)$ to be distributed among the set of relevant discrete nodes of a FE or Boundary Element (BE) model. This is usually achieved as follows,

$$\mathbf{F}^{(m)} = \int_{S^e} \left[\mathbf{\sigma}^{\infty}(\mathbf{x}) \cdot \mathbf{n}(\mathbf{x}) \right] N_m(\mathbf{x}) \, dS_e \,, \tag{3.1}$$

where dS_e is the infinitesimal surface element with surface area S_e . $N_m(\boldsymbol{x})$ are so-called shape functions (interpolation functions) that distribute the traction field among the surface element's nodes.

The problematic singularity associated with the classical Volterra dislocation is avoided by using the non-singular formulation of Cai et al. [2]. The stress field of a dislocation in a homogenous infinite linear elastic domain can be calculated as a contour integral along the loop [3],

$$\sigma_{ij}^{\infty}(\boldsymbol{x}) = C_{ijkl} \oint \epsilon_{lnh} C_{pqmn} \frac{\partial G_{kp}(\boldsymbol{x} - \boldsymbol{x}')}{\partial x_q} b_m dx_h', \qquad (3.2)$$

where C_{ijkl} is the elastic stiffness matrix, ϵ_{lnh} the permutation operator, \boldsymbol{b} the Burgers vector, \boldsymbol{x}' the coordinate that spans the dislocation, and $G_{kp}(\boldsymbol{x}-\boldsymbol{x}')$ is Green's function of elasticity [3]. $G_{kp}(\boldsymbol{x}-\boldsymbol{x}')$ is defined as the displacement component in the x_k direction at point \boldsymbol{x} due to a force applied in the x_p direction

at point \mathbf{x}' . The traditional singularity comes from taking the Burgers vector distribution as a delta function. Cai et al. [2] proposed an alternative definition of $G_{kp}(\mathbf{x} - \mathbf{x}')$ which has a wider isotropic spread mainly localised in a radius a around the dislocation core,

$$G_{ij}(\boldsymbol{x} - \boldsymbol{x}') = \frac{1}{8\pi\mu} \left[\delta_{ij} \partial_{pp} - \frac{1}{2(1-\nu)} \partial_{ij} \right] R_a, \qquad (3.3)$$

where μ , ν are the isotropic shear modulus and Poisson's ratio respectively, δ_{ij} is the Kronecker Delta, $\partial_{x_1...x_n} \equiv \frac{\partial^n}{\partial x_1...\partial x_n}$. R_a is the defined as,

$$R_a(\mathbf{x}) = R(\mathbf{x}) * w(\mathbf{x}) = \int R(\mathbf{x} - \mathbf{x}') w(\mathbf{x}') d^3 \mathbf{x}'$$
$$= \sqrt{R^2 + a^2}$$
(3.4a)

$$w(\mathbf{x}) = \frac{15a^4}{8\pi (R^2 + a^2)^{7/2}},$$
(3.4b)

where $w(\mathbf{x})$ is the isotropic Burgers vector distribution derived in the appendix of [2], $\mathbf{x} = (x, y, z)$ and $R(\mathbf{x}) = \sqrt{x^2 + y^2 + z^2}$.

For two dislocation nodes (1, 2) connected by straight line segments eq. (3.2) becomes,

$$\frac{\mu}{8\pi} \int_{x_1}^{x_2} \left(\frac{2}{R_a^3} + \frac{3a^2}{R_a^5}\right) [(\mathbf{R} \times \mathbf{b}) \otimes d\mathbf{x}' + d\mathbf{x}' \otimes (\mathbf{R} \times \mathbf{b})]
+ \frac{\mu}{4\pi(1-\nu)} \int_{x_1}^{x_2} \left(\frac{1}{R_a^3} + \frac{3a^2}{R_a^5}\right) [(\mathbf{R} \times \mathbf{b}) \cdot d\mathbf{x}'] \mathbf{I}_2
- \frac{\mu}{4\pi(1-\nu)} \int_{x_1}^{x_2} \frac{1}{R_a^3} [(\mathbf{b} \times d\mathbf{x}') \otimes \mathbf{R} + \mathbf{R} \otimes (\mathbf{b} \times d\mathbf{x}')]
+ \frac{\mu}{4\pi(1-\nu)} \int_{x_1}^{x_2} \frac{3}{R_a^5} [(\mathbf{R} \times \mathbf{b}) \cdot d\mathbf{x}'] \mathbf{R} \otimes \mathbf{R}$$
(3.5)

3.1.1 Resolving Singularities when Dislocation Line Segments are Parallel to Surface Elements

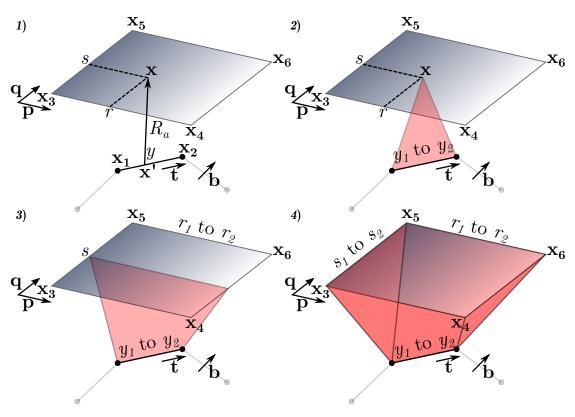


Figure 3.1: Diagram of the line integral method used to find analytical expressions for the forces exerted by dislocations on linear rectangular SEs [1]. 1) For any given point \boldsymbol{x} on the SE and any given point \boldsymbol{x}' on the dislocation line segment, define distance R_a . 2) Integrate from $x_1 \to x_2$ along line direction \boldsymbol{t} . 3) Integrate from $r_1 \to r_2$ along vector \boldsymbol{p} . 4) Integrate from $s_1 \to s_2$ along vector \boldsymbol{q} .

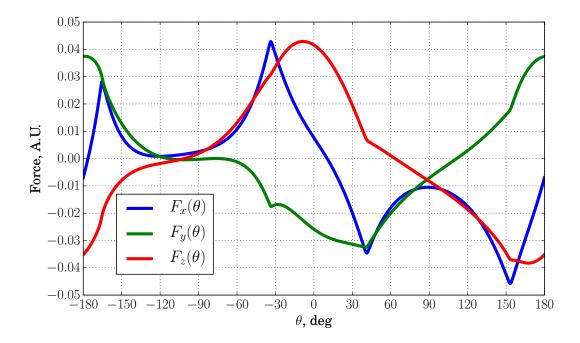


Figure 3.2: Effects of rotating a single dislocation line segment on the forces exerted by it on a linear rectangular SE. The specific values of this function are not known a priori, all that is known is that it must be periodic $(T=2\pi)$ and have finite maximum and minimum values. The singularity is avoided by perturbing the angle $\theta=0 \to \theta=\pm\epsilon$, $\epsilon\gtrsim 0$.

Analytical Forces Induced by Dislocations on Quadratic Triangular Surface Elements

4.1 Forces Exerted by a Dislocation Line Segment on Quadratic Triangular Surface Elements

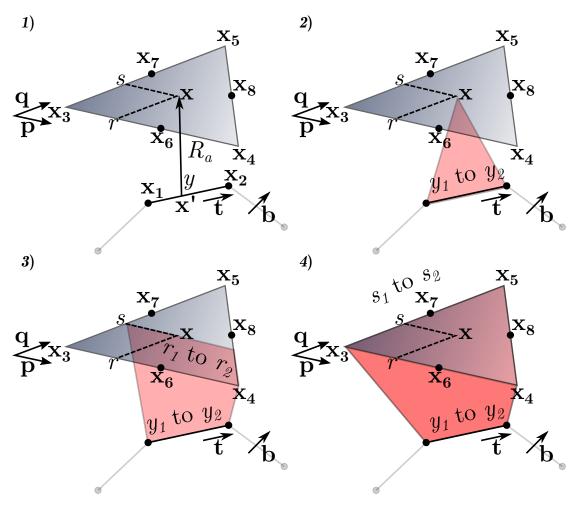


Figure 4.1: Diagram of the line integral method used to find analytical expressions for the forces exerted by dislocations on quadratic triangular SEs.

Screw Dislocation Mobility BCC

The way screw dislocations have been treated in BCC mobility laws has been by allowing them to move freely in the crystal via the drag matrix eq. (5.1)

$$\mathbf{B}(\boldsymbol{\xi}) = B_s(\mathbf{I} - \boldsymbol{\xi} \otimes \boldsymbol{\xi}), \quad \text{when } \boldsymbol{\xi} \parallel \boldsymbol{b}. \tag{5.1}$$

Edge dislocations have been restricted to on the glide plane via eq. (5.2)

$$\mathbf{B}(\boldsymbol{\xi}) = B_{el}(\boldsymbol{\xi} \otimes \boldsymbol{\xi}) + B_{eq}(\boldsymbol{m} \otimes \boldsymbol{m}) + B_{ec}(\boldsymbol{n} \otimes \boldsymbol{n}), \text{ when } \boldsymbol{\xi} \perp \boldsymbol{b}.$$
 (5.2)

Where the subscripts s, el, eg, ec mean screw, edge line, edge glide and edge climb respectively. The unit vectors $\mathbf{n} \equiv \mathbf{b} \times \boldsymbol{\xi} / \|\mathbf{b} \times \boldsymbol{\xi}\|$ and $\mathbf{m} \equiv \mathbf{n} \times \boldsymbol{\xi}$.

By choosing appropriate drag coefficients, B, one can scale the movement of edge dislocations along any particular direction to fit real dislocation mobilities. However, this is not so for screw dislocations. The result of this is unrestricted movement.

However if we recognise that in BCC screw dislocations only have one family of Burgers vectors $\langle 1\,1\,1 \rangle$, we can construct find the potential glide planes, \boldsymbol{n} orthogonal to the Burgers vector as shown in eq. (5.3),

$$\frac{1}{\sqrt{3}} \begin{bmatrix} 1 & 1 & 1 \end{bmatrix} \perp \frac{1}{\sqrt{2}} \begin{bmatrix} 1 & \overline{1} & 0 \end{bmatrix}, \frac{1}{\sqrt{2}} \begin{bmatrix} 1 & 0 & \overline{1} \end{bmatrix}, \frac{1}{\sqrt{2}} \begin{bmatrix} 0 & 1 & \overline{1} \end{bmatrix}$$
 (5.3a)

$$\frac{1}{\sqrt{3}} \begin{bmatrix} \overline{1} & 1 & 1 \end{bmatrix} \perp \frac{1}{\sqrt{2}} \begin{bmatrix} 1 & 1 & 0 \end{bmatrix}, \frac{1}{\sqrt{2}} \begin{bmatrix} 1 & 0 & 1 \end{bmatrix}, \frac{1}{\sqrt{2}} \begin{bmatrix} 0 & 1 & \overline{1} \end{bmatrix}$$
 (5.3b)

$$\frac{1}{\sqrt{3}} \begin{bmatrix} 1 & \overline{1} & 1 \end{bmatrix} \perp \frac{1}{\sqrt{2}} \begin{bmatrix} 1 & 1 & 0 \end{bmatrix}, \frac{1}{\sqrt{2}} \begin{bmatrix} 0 & 1 & 1 \end{bmatrix}, \frac{1}{\sqrt{2}} \begin{bmatrix} 1 & 0 & \overline{1} \end{bmatrix}$$
 (5.3c)

$$\frac{1}{\sqrt{3}} \begin{bmatrix} 1 & 1 & \overline{1} \end{bmatrix} \perp \frac{1}{\sqrt{2}} \begin{bmatrix} 1 & 0 & 1 \end{bmatrix}, \frac{1}{\sqrt{2}} \begin{bmatrix} 0 & 1 & 1 \end{bmatrix}, \frac{1}{\sqrt{2}} \begin{bmatrix} 1 & \overline{1} & 0 \end{bmatrix}. \tag{5.3d}$$

We can then build a mobility function that limits the free movement of screw dislocations to preferentially occur along physically reasonable planes that is more along the lines of eq. (5.2) rather than freely moving as in eq. (5.1). We can then use the same definition for $\mathbf{m} \equiv \mathbf{n} \times \boldsymbol{\xi}$ and arrive at eq. (5.4),

$$\mathbf{B}(\boldsymbol{\xi}) = B_{sl}(\boldsymbol{\xi} \otimes \boldsymbol{\xi}) + B_{sg} \sum_{i} (\boldsymbol{m}_{i} \otimes \boldsymbol{m}_{i}) + B_{sc} \sum_{i} (\boldsymbol{n}_{i} \otimes \boldsymbol{n}_{i}). \tag{5.4}$$

Where the sum over i is over the candidate planes.

For a screw dislocation with $\boldsymbol{b} = [1 \ 1 \ 1]$

Chapter 6

Parallelisation of the Analytical Forces Induced by Dislocations on Surface Elements

6.1 Parallelisation on Graphics Processing Units

Efficient parallelisation requires coalesced memory access, which means we have to be extremely careful when mapping CPU memory to device memory. The fact that threads work "simultaneously" means that in order to obtain good performance, data which is to be "simultaneously" loaded into each thread must be contiguous. This maximises cache memory use and therefore reduces slow memory fetch operations to global or shared memory.

The parallelisation was done only over the SEs in order to avoid the undesirable and inefficient GPU branching that would occur under other schemes.

The most natural form of parallelisation is to have blocks of 4n threads where $n \leq 8 \in \mathbb{N}$. This also fits nicely into the 32 thread per warp paradigm.

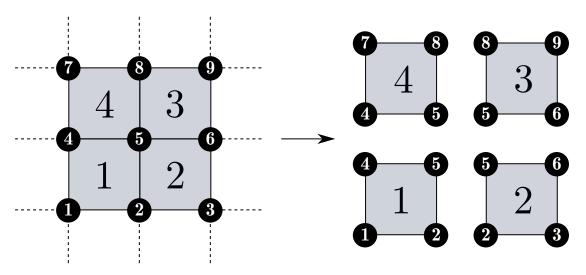


Figure 6.1: Each linear rectangular SE is mapped to one thread.

Algorithm 6.1 Elements in host \mapsto device.

```
1: function ELEMENT_HOST_DEVICE_MAP({}^{\text{h}}\boldsymbol{X}[N][3 \times E])
                                         ▶ Input, temporary, and output indices set to zero.
 2:
         i, j, k \leftarrow 0
 3:
               \triangleright <sup>d</sup>X accommodates all 3 coordinates of all N nodes in all E elements.
 4:
         {}^{\mathrm{d}}\boldsymbol{X} \leftarrow \mathrm{malloc}(3 \times E \times N)
 5:
                                                                     ▷ Loop over nodes in element.
         for (n = 0; n < N; n + +) do
 6:
                \triangleright Set output index to point at the n^{\text{th}} node of the first coordinate of
 7:
     the first element.
 8:
              k \leftarrow j
              for (c = 0; c < 3; c + +) do
                                                                             ▷ Loop over coordinates.
 9:
                   \triangleright Set input index to point at the c^{\text{th}} coordinate of the first element
10:
    of the n^{\text{th}} node.
                   i \leftarrow c
11:
                   for (e = 0; e < E; e + +) do
                                                                                 ▶ Loop over elements.
12:
                        {}^{\mathrm{d}}\boldsymbol{X}[k+e] \leftarrow^{\mathrm{h}} \boldsymbol{X}[n][i]
13:
                              \triangleright Advance output index to point at the n^{\rm th} node of the c^{\rm th}
14:
     coordinate of the first element.
                        i \leftarrow i + 3
15:
16:
                   end for
                              \triangleright Advance output index to point at the n^{\mathrm{th}} node of the c^{\mathrm{th}}
17:
     coordinate of the first element.
                   k \leftarrow k + E \times N
18:
19:
               \triangleright Advance temporary index to point at the (n+1)^{\text{th}} node of the first
20:
     coordinate of the first element.
21:
              j \leftarrow j + E
         end for
22:
         return {}^{\mathrm{d}}X
23:
24: end function
```

Algorithm 6.2 Elements in device \mapsto thread.

```
function ELEMENT_DEVICE_THREAD_MAP({}^{d}X[3 \times E \times N],
 1: GPU
    {}^{\rm t}X[N][3]
                                                                             ▶ Thread, input indices.
 2:
         idx, i \leftarrow \text{threadIdx.} x + \text{blockIdx.} x \times \text{blockDim.} x
 3:
 4:
         for (c = 0; c < 3; c + +) do
                                                                                ▶ Loop over elements.
              for (n = 0; n < N; n + +) do
                                                                     ▶ Loop over nodes in element.
 5:
                   {}^{\mathrm{t}}\boldsymbol{X}[n][c] \leftarrow^{\mathrm{d}} \boldsymbol{X}[i+n \times E]
 6:
 7:
              i \leftarrow i + E \times N \triangleright \text{Advance input index to the } (n+1)^{\text{th}} \text{ node of element}
 8:
    i.
 9:
         end for
         return {}^{\mathrm{t}}X[n][c]
10:
11: end GPU function
```

Algorithm 6.3 Force in thread $\stackrel{+}{\mapsto}$ device.

```
1: GPU function ADD_FORCE_THREAD_DEVICE({}^{\mathrm{t}}F_n[N][3], {}^{\mathrm{t}}F_e[3], {}^{\mathrm{d}}F_n[3 \times
    E \times N,
    {}^{\mathrm{d}}\mathbf{F}_{e}[3\times E], \mathrm{idx})
 2:
                                                                                        ▶ Nodal force.
                 ▷ Set output index to whatever parallelisation index is given to the
3:
    function. In this case the same index we use for the main parallelisation.
         i \leftarrow idx
 4:
         for (n = 0; n < N; n + +) do
                                                                                 ▶ Loop over nodes.
5:
             for (c = 0; c < 3; c + +) do
6:
                                                                          ▶ Loop over coordinates.
                   ▶ Ensure nodal forces are correctly added and mapped from local
 7:
    thread memory to global device memory.
                  atomicAdd({}^{d}\boldsymbol{F_n}[i+c\times E\times N], {}^{t}\boldsymbol{F_n}[n][c])
8:
9:
             end for
                        ▷ Displace output index to point at the first coordinate of the
10:
     (n+1)^{\text{th}} node of the idx<sup>th</sup> SE.
             i \leftarrow i + E
11:
         end for
12:
                                                                                         ▶ Total force.
13:
         i \leftarrow idx
                                                                              ▷ Reset output index.
14:
         for (c = 0; c < 3; c + +) do
                                                                          ▶ Loop over coordinates.
15:
             atomicAdd({}^{\mathrm{d}}\boldsymbol{F_e}[i], {}^{\mathrm{t}}\boldsymbol{F_e}[c])
16:
             \triangleright Advance the output index to point at the (i+1)^{\text{th}} coordinate of the
    idx<sup>th</sup> SE.
             i \leftarrow i + E
18:
19:
         end for
         return {}^{\mathrm{d}}F_n, {}^{\mathrm{d}}F_e
20:
21: end GPU function
```

```
Algorithm 6.4 Force (parallelise over dislocations) in thread \stackrel{+}{\mapsto} device.
```

```
1: GPU function DLN_ADD_FORCE_THREAD_DEVICE({}^{\mathrm{t}}F_{n}[N][3], {}^{\mathrm{t}}F_{e}[3],
    {}^{\mathrm{d}}\boldsymbol{F_n}[3 \times E \times N], {}^{\mathrm{d}}\boldsymbol{F_e}[3 \times E], k)
                                                                                     ▷ Nodal force.
 2:
                           ⊳ Set output index to correspond to whatever SE we're on.
 3:
    By parallelising over dislocation lines, we must loop through SEs in the main
    code, this is the value of k we provide. Multiply by 3 and N because each SE
    has 3 coordinates and N nodes.
 4:
         i \leftarrow 3 \times N \times k
         j \leftarrow 0
 5:
                                                                                for (n = 0; n < N; n + +) do
                                                                               ▶ Loop over nodes.
 6:
             for (c = 0; c < 3; c + +) do
                                                                       ▶ Loop over coordinates.
 7:
                   ▶ Ensure nodal forces are correctly added and mapped from local
    thread memory to global device memory.
                  atomicAdd({}^{\mathrm{d}}\boldsymbol{F_n}[i+j+c], {}^{\mathrm{t}}\boldsymbol{F_n}[n][c])
 9:
10:
             end for
                     ▶ Displace auxiliary index to point at the first coordinate of the
11:
    (n+1)^{\text{th}} node of the k^{\text{th}} SE.
             j \leftarrow j + 3
12:
         end for
13:
                                                                                      ▶ Total force.
14:
                 \triangleright Set auxiliary index to point at the first coordinate of the k^{\text{th}} SE.
15:
         j \leftarrow 3 \times k
16:
         for (c = 0; c < 3; c + +) do
17:
                                                                       ▶ Loop over coordinates.
             atomicAdd({}^{d}\boldsymbol{F_e}[j+c], {}^{t}\boldsymbol{F_e}[c])
18:
         end for
19:
         return {}^{\mathrm{d}}F_n, {}^{\mathrm{d}}F_e
20:
21: end GPU function
```

Algorithm 6.5 Nodal force in device \mapsto host.

```
1: function fx_device_host_map({}^{d}\boldsymbol{F_n}[3 \times E \times N], {}^{h}\boldsymbol{F_n}[N][3 \times E])
                                                    ▶ Set input and output indices to zero.
2:
        i, j \leftarrow 0
        for n = 0; n < N; n + + do
3:
                                                                              ▶ Loop over nodes.
                                      ▷ Reset output index to point at the first element.
 4:
             for e = 0; e < E; e + + do
                                                                          ▶ Loop over elements.
 5:
                 for c = 0; c < 3; c + + do
                                                                       ▶ Loop over coordinates.
6:
                      {}^{\mathrm{h}}\mathbf{F}_{n}[n][j+c] = {}^{\mathrm{d}}\mathbf{F}_{n}[i+e+c\times E\times N]
 7:
8:
                       ▶ Advance output index to point at the first coordinate of the
9:
    (e+1)^{\text{th}} element.
                 j \leftarrow j + 3
10:
             end for
11:
             \triangleright Advance input index to point at the first coordinate of the (n+1)^{\text{th}}
12:
    node of the first element.
             i \leftarrow i + E
13:
        end for
14:
        return {}^{\rm h}F_n
15:
16: end function
```

6.1.1 Data Mapping

- 6.1.1.1 Elements: Host \mapsto Device
- 6.1.1.2 Elements: Device \mapsto Thread
- 6.1.1.3 Force: Thread $\stackrel{+}{\mapsto}$ Device
- 6.1.1.4 Force (Parallelise over Dislocations): Thread $\stackrel{+}{\mapsto}$ Device
- 6.1.1.5 Nodal Force: Device \mapsto Host
- 6.1.1.6 Total Force: Device \mapsto Host

$$\boldsymbol{X}_{en} \coloneqq [x_{en}, y_{en}, z_{en}] \tag{6.1a}$$

 $\boldsymbol{X}_{(1 \to E)n} \mapsto \boldsymbol{X}_n$

$$X_n := [x_{1n}, y_{1n}, z_{1n}, \dots, x_{En}, y_{En}, z_{En}]$$
 (6.1b)

 $oldsymbol{X}_{1 o N}\mapsto oldsymbol{X}^{ ext{SE}}$

$$[x_{11}, \dots, x_{E1}, x_{12}, \dots, x_{E2}, \dots, x_{1N}, \dots, x_{EN},$$

$$\boldsymbol{X}^{\text{SE}} := y_{11}, \dots, y_{E1}, y_{12}, \dots, y_{E2}, \dots, y_{1N}, \dots, y_{EN},$$

$$z_{11}, \dots, z_{E1}, z_{12}, \dots, z_{E2}, \dots, z_{1N}, \dots, z_{EN}]$$

$$(6.1c)$$

where e is the SE label, n the node label, E the number of SEs in the scope. N

¹Not quite but essentially simultaneously.

Algorithm 6.6 Total force in device \mapsto host.

```
1: function ftot_device_host_map({}^{d}F_{e}[3 \times E], {}^{h}F_{e}[3 \times E])
                                                                    ▶ Set output index to zero.
 2:
 3:
         for e = 0; e < E; e + + do
                                                                           ▶ Loop over elements.
             for c = 0; c < 3; c + + do
 4:
                                                                       ▶ Loop over coordinates.
                  {}^{\mathrm{h}}\boldsymbol{F_e}[i+c] = {}^{\mathrm{d}}\boldsymbol{F_e}[e+c\times E]
 6:
                   > Advance the output index to point at the first coordinate of the
     (e+1)^{\text{th}} surface element.
             i \leftarrow i + 3
 8:
 9:
         end for
         return {}^{\rm h}F_e
10:
11: end function
```

the number of nodes in a SE.

Data-mapping according to ?? and relabelling the nodes so they go from $0 \rightarrow N-1$, the data from fig. 6.1 would be arranged in device memory like eq. (6.2),

$$\boldsymbol{X}^{\text{SE}} = \begin{bmatrix} \underbrace{h_x, i_x, f_x, g_x}_{\mathbf{x_0}}, \underbrace{a_x, b_x, i_x, h_x}_{\mathbf{x_1}}, \underbrace{i_x, d_x, e_x, f_x}_{\mathbf{x_2}}, \underbrace{b_x, c_x, d_x, i_x}_{\mathbf{x_3}}, \\ \dots y\text{-coord} \dots, \\ \dots z\text{-coord} \dots \end{bmatrix}$$
(6.2)

In the GPU, each thread will cater to one SE at a time. This means that each thread will have to extract the relevant data from the 1D array with length $3 \times E \times N$ into four 1D arrays of length 3. The purpose of CNE mapping is to provide the warp with coalesced memory access. This is achieved via ??.

Since ?? is performed in a CUDA GPU, threads in a single block execute sequentially from,

threadIdx.
$$x = 0 \rightarrow \text{threadIdx.} x = \text{blockDim.} x - 1$$
, (6.3)

while threads in different blocks execute in parallel. Coalesced memory access is ensured by having each block load a cache line whose entries are contiguously accessed by the threads in the block. Using the same notation as $\ref{eq:local_second}$, full cache line utilisation (optimal cache use) is achieved if cache lines can accommodate l entries

Figure 6.2: Minimum working complete example of the backward coodinate-node-element-map. The explicit calculations of global indices correspond to substituting the values for "idx" and "idxi" as in ??. X_a^b denotes a 1D array of length three containing the xyz-coordinates of node a of element b. Each thread concerns itself with only one element at a time. The dash-dot and dashed boxes represent cache lines for a thread block, the dotted line represents a memory fetch, and the dashed lines in X^E represent steps of E entries (the start of the data for the next node of the element type we're dealing with). The memory operations are not shown twice to minimuse redundancy.

given by eq. (6.4),

$$l = \begin{cases} a \times N \times E &, a > 0 \in \mathbb{N} \\ & \text{or} \\ \frac{1}{2^a} \times N \times E &, a \ge 0 \in \mathbb{N}, N \times E \equiv 0 \pmod{2^a} . \end{cases}$$
 (6.4)

Figure 6.2 shows an example of ?? up to the y-coordinate.

6.1.1.7 Node Coordinate Element Map

The node-coordinate-element (NCE) data mapping in eq. (6.5) is carried out by algorithm 6.7, each thread looks after a given SE.

Algorithm 6.7 NCE data mapping.

```
for all n nodes \in surface element do

for all c coordinates \in [x, y, z] do

for all e surface elements \in surface mesh section do

list.append(data of the n^{\text{th}} node with c^{\text{th}} coordinate of the e^{\text{th}} SE)

end for

end for
```

$$\boldsymbol{X}_{en} \coloneqq [x_{en}, y_{en}, z_{en}] \tag{6.5a}$$

$$X_{(1 \to E)n} \mapsto X_n$$

$$\mathbf{X}_n := [x_{1n}, \dots, x_{En}, y_{1n}, \dots, y_{En}, z_{1n}, \dots, z_{En}]$$
 (6.5b)

$$X_{1 \rightarrow N} \mapsto X^{\operatorname{SE}}$$

$$\mathbf{X}^{\text{SE}} := \begin{bmatrix} x_{11}, \dots, x_{E1}, y_{11}, \dots, y_{E1}, z_{11}, \dots, z_{E1} \\ \dots, \dots, \\ x_{1M}, \dots, x_{EN}, y_{1N}, \dots, y_{EN}, z_{1N}, \dots, z_{EN} \end{bmatrix}$$
(6.5c)

where e is the surface element, n the node, E the total number of SEs in scope, N the total number of nodes in each SE.

Data-mapping according to algorithm 6.7 and relabelling the nodes so they go from $0 \to N - 1$, the data from fig. 6.1 would be arranged in device memory like eq. (6.6),

$$\underbrace{\begin{bmatrix} h_x, i_x, f_x, g_x, h_y, i_y, f_y, g_y, h_z, i_z, f_z, g_z \\ \dots & \mathbf{x_1}, xyz\text{-coords} \dots, \\ \dots & \mathbf{x_2}, xyz\text{-coords} \dots, \\ \dots & \mathbf{x_3}, xyz\text{-coords} \dots \end{bmatrix}}$$
(6.6)

6.1.1.8 Resolving Data Write Conflicts

Data write conflicts can be a problem in parallel applications where global data is changed by multiple threads within the same clock tick. This can be avoided with atomic operations.

6.1.2 Resolving Parallel Dislocation Line Segments to Surface Elements

Dealing with the special case when the dislocation line segment t is parallel to the SE is relatively trivial when in a serial program. By the mean value theorem, we

can slightly perturb t by rotating it by a small angle around the midpoint of t with respect to the axis of rotation defined by $t \times n$. In contrast to serial code, program branches can have a serious effect on parallel performance due to warp divergence. Due to the complexity of the force calculation and relative rarity of the edge case, there is no branching behaviour in the parallel code until *after* the calculation is performed. Where algorithm 6.8 is performed.

Algorithm 6.8 Resolving cases when $t \parallel n$ on GPUs.

```
for all surface elements and line segments do \dots

if t_{\mathrm{thread}} \parallel n_{\mathrm{thread}} then x_{\mathrm{buffer}} \leftarrow x_{\mathrm{thread}}
t_{\mathrm{buffer}} \leftarrow t_{\mathrm{thread}}
else F_{\mathrm{total}} + = F_{\mathrm{thread}}
end if end for return to serial code if x_{\mathrm{buffer}} \neq \mathrm{empty} then for all surface elements and line segments do perform calculation for t \parallel n end for end if
```

6.1.2.1 Implementation

The node mapping would be the same as ??. The parallelisation would be on individual packets of a SE with a slightly rotated dislocation line segment, as in a single iteration of the serial code where the dislocation line segment is rotated. The forces would be averaged with atomic operations and <code>__syncthreads()</code> before they are added to the total nodal forces.

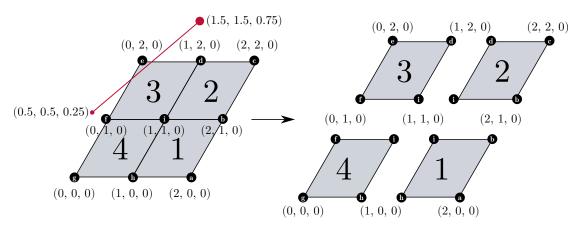


Figure 6.3: Test case for CUDA development. Dislocation line not shown in node separation as the parallelisation happens across surface elements. Not to perspective or scale.

6.1.3 Test Case

According to ?? and the node labelling scheme of fig. 3.1 the data mapping would look like:

$$\mathbf{X}^{\text{SE}} = \underbrace{\begin{bmatrix} \underbrace{x \text{-coord}} \\ 1, 1, 0, 0, \underbrace{2, 2, 1, 1}, \underbrace{1, 1, 0, 0}, \underbrace{2, 2, 1, 1}, \\ \underbrace{y \text{-coord}} \\ \underbrace{y \text{-coord}} \\ \underbrace{x_{3}} \underbrace{x_{4}} \underbrace{x_{5}} \underbrace{x_{5}} \underbrace{x_{6}}$$

$$\underbrace{x_{5}} \underbrace{x_{6}} \underbrace{x_{6}}$$

$$\underbrace{0, 0, 0, 0, \underbrace{0, 0, 0, 0}, \underbrace{0, 0, 0, 0}, \underbrace{0, 0, 0, 0}, \underbrace{0, 0, 0, 0}, \underbrace{0, 0, 0}, \underbrace{0, 0, 0}, \underbrace{0, 0, 0, 0}, \underbrace{0, 0, 0, 0}, \underbrace{0, 0, 0, 0, 0}, \underbrace{0, 0, 0}, \underbrace{$$

In general the number of indices in the array would be $3 \times e \times n$ where e is the number of surface elements, n the number of nodes per element and 3 the number of spatial dimensions.

6.2 Thread Block Size Optimisation

 $http://docs.nvidia.com/cuda/cuda-c-best-practices-guide/index.html\#memory-optimizations \\ http://docs.nvidia.com/cuda/cuda-c-best-practices-guide/index.html\#execution-configuration-optimizations$

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Appendix A

Best Practices

This chapter defines the best practices that will make improve the development and testing pipeline by defining rules and standards that facilitate collaboration.

A.1 Filesystem

All files must be within /project_name.

```
1. All files must be within /project name,
```

```
~/ = project_name/,
~/~/ = project_name/src/ or project_name/dev/.
```

- 2. Release code must be within ~/src/.
- 3. Development code must be within ~/dev/.
- 4. Test data must be within ~/tests/.
- 5. Documentation must be within ~/~/doc/.
- 6. Examples must be within $\sim/\sim/\exp/$.
- 7. External libraries must be within ~/lib/.
- 8. Generated images must be within ~/~/images/.
- 9. Old versions recordkeeping must be within ~/prv/va.b.c/.

A.2 Versioning

1. Use a version control system like GitHub or PasteBin.

- 2. There must be a master branch that is only changed when the code is stable and bug free.
- 3. Development branches should be exploited as seen fit without making things overly convoluted.
- 4. Commits must be as bug free and regular as possible. When to commit is left to the developer's discresion.
- 5. Commit messages should be as descriptive as possible.
- 6. Versions should be specified as va.b.c where a, b, c = integers. The three levels are a = release version (usable, bug free code), b = beta version (code that is undergoing testing), c = alpha version (code that is under active development).

A.3 Documentation

A.3.1 Commenting

Every codefile must be appropriately commented by meeting the following guidelines.

- 1. The start of each codefile must have a heading detailing the creator, date of creation and edit history (date and name of editor).
- 2. Below the heading there must be a general explanation of the code. It must state any procedures, structures, objects and how they are to be utilised. Any backward or forward dependencies must be stated.
- 3. Below the description and edit history any relevant literature must be mentioned (dois are preferred). Must be as detailed as possible, include equation numbers/ranges if necessary.
- 4. The start of every procedure has an explanation of its purpose, inputs, outputs and inputs-outputs.
- 5. Particularly complicated code blocks must have an in-depth explanation of what it does. Comment each line if necessary.
- 6. Corrections or additions must be explicitly bounded by comments at the start and end of the change. Both bounding comments must have the author's name and the date. Below the starting comment, there should be an explanation of the change. Any punctual comments can be made as normal.

A.3.2 README

Every codefile must have an associated README .tex document that documents the codefile's contents. It must meet the following guidelines as appropriate.

- 1. The name must be that of the file it documents (minus the extension of course).
- 2. Description and overall explanation of the codefile's purpose.
- 3. Overall flow chart or pseudo code describing the file's purpose.
- 4. Document the codefile's procedures. This means describing and explaining their corresponding inputs, outputs, inputs-outputs, forwards and backwards dependencies, and flow charts or pseudo codes.
- 5. Unit test designs and results for each procedure. If appropriate also include those of integral tests.

The codefile may also be appended at the end of documentation if desired (the minted package is highly recommended).

A.4 Modularisation

- 1. Code repetition must be kept to a *strict* minimum. Any piece of code that will be reused must be modularised.
- 2. Procedures must be as self-sufficient as possible *without* repeating code. If repeating code is necessary, replace it with a procedure that is to be repeatedly called instead. Minimising repeated code >>> procedure self-sufficiency.

A.5 Coding Style

All names must meet the following guidelines.

- 1. Indent appropriately. Four space tabs are a good compromise between code necking and readability.
- 2. Minimise the use of nested code blocks, use intrisics, libraries or create procedures instead.
- 3. Break up lines that are uncomfortably long, typically anything over 80–100 characters.

- 4. Names should be appropriately descriptive and human readable.
- 5. All code and names must be systematic and logical.
- 6. the use of upper cases should be reserved for parameters (const variables in C).
- 7. Delimit words with "_" not case changes.
- 8. Long and descriptive \gg short and cryptic.

A.5.1 Filenames

1. If old versions are to be kept, the old stable versions of file must have the date of last modification appended suffixed after the file extension in the _yyyymmdd format. For example, if the stable version of the release code hello_world_parallel.c was last modified on April 25, 2017 it should be archived as

hello_world_parallel.c_20170425.

A.5.2 Variables, Structures and Objects

- 1. Only counters and indices can be single letter variables.
- 2. Structure and object definitions are suffixed with _s and _o respectively.
- Inputs, outputs and input-outputs to procedures must be prefixed with i_,
 o_ and io_ respectively.

A.5.3 Procedures

1. Functions and subroutines must be *prefixed* with f_ and s_ respectively.

Appendix B

Coupling Discrete Dislocation Dynamics to Finite Element Methods

```
function node_plane = extract_node_plane(fem_nodes ,
  → fem_node_cnct,...
                               node labels, dim, filter
                                node_plane)
     ~ %%-----%%
     % Written by famed MATLAB hater and fan of compiled languages,
     % Daniel Celis Garza, in 13/11/17.
     \mbox{\it \%} This code has been optimised as much as I can and has been
     → thoroughly
     % tested as of 15/11/17.
11
     \hookrightarrow %------%
     % Extracts a single node plane according to given filtering
     → criteria
     % and puts it into a 2D array arranged in xyz-coordinates like
     %
```

```
Node 1 Node 2 Node 3 ... Node N
      %
16
       %
17
      %
                     x 12
                               x 13 \qquad x 1N
             x\_11
      %
                      y_{-}12
                                y_{\perp}13
                                          y_1N / Element 1
             y_{\perp}11
19
      %
             z\_11
                      z_{12}
                                z_13 \ldots z_1N
                                                  /_____
      %
             x_21
                       x 22
                                x 23
                                          x 2N
21
      %
                                                  | Element 2
             y_{21}
                      y_{22}
                                y_23
                                           y_2N
      %
             z_21
                      z_22
                                z_23
                                           z_2N
                                                  /_____
      %
24
      %
^{25}
      %
26
      %
27
      %
             x\_E1
                     x\_{\it E2}
                               x\_E3 x\_EN
      %
                      y\_E2
                                y\_E3 ... y\_EN | Element E
             y\_E1
29
                               z\_{\it E3} z\_{\it EN}
      %
             z\_E1
                     z\_{\it E2}
       %
31
      % Inputs
33
       %
35
       % fem_nodes := dimension(:, 3). Assumed shape 2D array with 3
       \rightarrow columns.
      % Describes the coordinates of the nodes of the ft_inite
       \rightarrow element model.
       %
38
       % fem_node_cnct := dimension(:, 8). Assumed shape 2D array
       \rightarrow with 8
       % columns. Describes the node connectivity of the finite
       \rightarrow elements in
      % the model.
41
       %
42
      % node_labels := dimension(:). The labels of the nodes that
       \rightarrow are to be
      % extracted from fem_nodes.
      % dim := three times the area of the plane to be extracted (3
      % dimensions).
```

```
%
48
     % filter := the selection criteria used to filter which nodes
49
     \rightarrow we want
     % to obtain.
50
52
        % Dummy variables
54
        %-----%
55
     % n_nodes := number of nodes per planar element.
     % tmp\_nodes := dimension(n\_nodes, 3). Contains the
58
        xyz-coordinates of
     % all nodes with the labels specified by node_label.
59
     %
60
     % mask := same dimension as tmp_nodes. Logical mask whose
        entries are 1
     % only when a given condition is met. It is used to index
62
        tmp_nodes to
     % find only the nodes that correspond to a given plane.
63
     %
65
     % Input/output variables
67
     ~ %-----%
68
     % node_plane := dimension(dim/3, n_nodes). It is the slice of
69
     \rightarrow the array
     % that holds the nodes of the plane. Only pass the array slice
70
     % that corresponds to the current plane of nodes to be
     \rightarrow extracted,
     % a la Fortran intent(in out).
72
73
74
        %%-----%%
```

75

```
%% Error checking
76
        n_nodes
                    = size(node_labels, 2);
77
        if(n nodes ~= size(node plane , 2))
78
            error('extract_node_plane:: n_nodes = %d, size(node_plane,
79
             \rightarrow 2) = %d; they must be the same i.e. the number of nodes
               per planar element', n nodes, size(node plane,2))
        end %if
80
        %% Finding plane nodes
82
        % Find the all the nodes labelled as stated in node_labels
        % (corresponding to a plane family)
84
        tmp_nodes = fem_nodes(fem_node_cnct(:, node_labels), 1:3);
        % Find an array of entries that meet the filter criteria in
           tmp\_nodes
        mask = tmp_nodes(:, filter(1)) == filter(2);
87
        % Reshape the array to a column array containing only the
88
        → nodes which
        % meet the filter criteria.
        tmp nodes = reshape(tmp nodes(mask, :)', n nodes * dim, 1);
90
91
        %% Passing data to the slice of the array that contains the
        → nodes for every plane
        step = 0;
93
        for i = 1: n nodes
94
            node_plane(1: dim, i) = tmp_nodes(1 + step: dim + step);
95
            step = step + dim_3;
        end %for
97
98
        %% Cleanup
99
        clear dim_3; clear idxf; clear tmp_nodes; clear mask; clear
100

    step;

   end %function
101
   function [node_plane_lbl, node_plane] = extract_node_planes(
                                                fem nodes

    fem_node_cnct,...

                                                surf_node_util,
 3

    → fem planes

                                                                  , . . .
```

```
n_elem , n_nodes)
4
5
         %%-----%%
      % Written by famed MATLAB hater and fan of compiled languages,
      % Daniel Celis Garza, in 13/11/17.
      %
      \% This code has been optimised as much as I can and has been
10
      \rightarrow thoroughly
      % tested as of 15/11/17.
11
12
13
      % _lbl := variables suffixed with it are the global label
      \rightarrow equivalent
      % of those without.
15
      % Extracts a single node plane according to given filtering
17
      → criteria
      % and puts it into a 2D array arranged in xyz-coordinates like
      %
           Node 1 Node 2 Node 3 ... Node N
20
      %
21
      %
           x\_11
                    x_{\perp}12
                              x_{-}13
                                        x\_1N
      %
           y_{\perp}11
                    y_{\_}12
                              y_{\_}13
                                        y\_\,1N
                                               | Element 1
23
      %
           z\_11
                    z\_12
                              z_13 \ldots z_1N
                                                /_____
24
      %
           x_21
                     x_22
                              x_{2}3
                                         x_2N
25
           y_{2}
                    y_{22}
                              y_23
                                                / Element 2
      %
                                        y_2N
26
      %
           z_21
                    z_22
                             z_23
                                        z_2N
                                                /_____
27
      %
28
      %
29
      %
30
      %
31
      %
           x\_E1
                    x E2
                              x E3 x EN
32
      %
            y\_E1
                     y\_{\it E2}
                              y\_E3 \ldots y\_EN
                                                / Element E
33
      %
                              z_E3 z_EN
            z\_E1 z\_E2
                                                /
34
      %
35
```

```
36
      % Inputs
39
      % fem_nodes := dimension(:, 3). Assumed shape 2D array with 3
         columns.
          Describes the coordinates of the nodes of the ft_i
          element model.
      % fem_node_cnct := dimension(:, 8). Assumed shape 2D array
       → with 8
          columns. Describes the node connectivity of the finite
          elements in
          the model.
      %
      %
      % surf_node_util := dimension(n_nodes+2, 6). Each column is
         laid out as
      %
          follows:
      %
              [node_label_initial; ...; node_label_final;
              plane_area; coordinate_number].
      %
          Each row corresponds to one plane.
52
      % fem_planes := dimension(:). Contains the planes whose nodes
         are to be
          extracted.
54
      % n_elem := total number of elements to be extracted.
57
      % n_nodes := number of nodes per planar element.
      %
         % Dummy variables
61
```

%

63

```
% plane_idx := the index of the fem plane to be extracted
64
           during the
       %
           current iteration.
66
       % dim := plane_area, there are this many surface elements per
        \rightarrow plane.
       %
68
       % dim_3 := 3*plane_area, there are three coordinates per plane
           node, plane_area is the total number of nodes in a plane
70
           and is found
            in surf_node_utils.
       %
71
       %
72
       % tmp\_nodes := dimension(n\_nodes, 3). Contains the
           xyz-coordinates of
            all nodes with the labels specified by node_label.
       %
74
75
       % mask := same dimension as tmp_nodes. Logical mask whose
76
           entries are 1
           only when a given condition is met. It is used to index
77
           tmp nodes to
           find only the nodes that correspond to a given plane.
79
       % coord := orthogonal coordinate to the plane (for node
           filtering
           purposes).
       %
81
       \% idxi, idxf := initial and final index that slice the
83
        \rightarrow coordinate
            output array into sections for each extracted plane.
84
85
       % idxl, idxm := initial and final index that slice the label
            output array into sections for each extracted plane.
87
       %
       % filter := value for filtering data.
89
90
       % step := step that ensures proper array indexing for each
          section of
            the linear array that contains the nodes of a given plane.
       %
92
       %
93
```

```
% n_nodes_p1 := n_nodes + 1
94
      %
95
      % n \ nodes \ p2 := n \ nodes + 2
97
         % Output variables
100
          %-----
101
      % node plane := dimension(3*n_{elem}, n_{elem}). It is the array
102
          that
          holds the planes' nodes' Cartesian coordinates.
104
105
          %/_______
106
      %% Allocating node plane coordinates and labels
      node plane
                   = zeros(3 * n elem, n nodes);
108
      node plane lbl = zeros(
                             n elem, n nodes);
109
110
      %% Looping through the planes to be extracted
111
      idxl = 1;
112
      idxi = 1;
113
      n_nodes_p_1 = n_nodes + 1;
114
      n_nodes_p_2 = n_nodes + 2;
115
      for i = 1: size(fem_planes, 1)
116
          %% Assigning control variables for the iteration
          plane_idx = fem_planes(i);
118
          dim = surf_node_util(n_nodes_p_1, plane_idx);
119
          \dim 3 = 3 * \dim;
120
          coord = surf_node_util(n_nodes_p_2, plane_idx);
121
          % Set final index of the output array slice containing the
122
          \rightarrow nodes
          % for this iteration's plane
123
          idxm = idxl + dim - 1;
124
          idxf = idxi + dim_3 - 1;
125
          %% Finding global node labels and node coordinates of the
126
          → plane
```

```
% Extracting all nodes whose labels correspond to the
127
             \rightarrow plane family
            % to be extracted this iteration
128
            tmp_nodes_lbl = fem_node_cnct(:, surf_node_util(1:n_nodes,
129
             → plane_idx));
                            = fem nodes(tmp nodes lbl, 1:3);
            tmp nodes
130
            % If the plane index is odd it is a face where its
131
             \rightarrow orthogonal
            % coordinate is at a minimum.
132
            if(mod(plane idx, 2) ~= 0)
133
                 filter = min(tmp nodes(:, coord));
134
            % Otherwise find the face whose orthogonal coordinate is
135
             \rightarrow at a maximum
            else
136
                 filter = max(tmp_nodes(:, coord));
137
            end %if
138
            % Find an array of entries that meet the filter criteria
139
             \rightarrow in tmp_nodes
            mask = tmp nodes(:, coord) == filter;
140
            % Reshape the array to a column array containing only the
141
             → nodes which
            % meet the filter criteria.
142
            tmp nodes = reshape(tmp nodes(mask,:)', n nodes * dim 3,
             \rightarrow 1);
            % Reshape the array with global node labels so it can be
144
             \rightarrow filtered
            % using the same mask.
145
            tmp_nodes_lbl_size = size(tmp_nodes_lbl, 1);
146
            tmp_nodes_lbl
                                 = reshape(tmp_nodes_lbl ,...
147
                                            tmp_nodes_lbl_size * n_nodes,
148
            tmp_nodes_lbl = tmp_nodes_lbl(mask);
149
150
            %% Passing data to the slice of the array that contains
151
             → the nodes labels and coords for every plane
            % Pass the node labels of this iteration to the global
152
             \rightarrow array.
            % Set the slice of the tmp_nodes array to be indexed
153
            step
                      = 0;
154
```

```
step_lbl = 0;
155
             for j = 1: n_nodes
156
                 % Grab the array slice that corresponds to node j in
157
                 % tmp_nodes array and send it off to the corresponding
                 → column
                 % of the slice of the output array containing all
159
                  \rightarrow extracted nodes
                 node plane(idxi: idxf, j) = tmp nodes(...
160
                                                       1 + \text{step: dim } 3 +
161
                                                        \hookrightarrow step...
                                                     );
162
                 node_plane_lbl(idxl: idxm, j) = tmp_nodes_lbl(...
163
                                                       1 + step_lbl: dim +
164

    step_lbl...

                                                     );
165
                 % Advance the step to move onto the array slice that
166
                 → contains
                 % node j+1 in tmp_nodes
167
                         = step
                                      + dim 3;
168
                 step lbl = step lbl + dim ;
169
            end %for
170
             % Set the initial index to shift the output array slice
             \rightarrow onto the
             % next plane to be extracted.
172
             idxi = idxf + 1;
173
             idxl = idxm + 1;
174
             clear tmp_nodes; clear tmp_nodes_lbl; clear mask;
175
        end %for
176
177
        %% Cleanup
178
        clear idxl; clear idxi; clear plane idx; clear dim; clear
179
         \rightarrow dim 3;
        clear coord; clear idxm; clear idxf; clear filter; clear
180

    tmp_nodes_lbl_size;

        clear step; clear step_lbl; clear n_nodes_p1; clear n_nodes_p2;
181
    end %function
182
```

```
function [f_dln, f_dln_se] = analytic_traction(
     . . .
                  se_node_coord, dln_node_coord, burgers ,
2

→ n nodes,...

                  n_nodes_t , n_se
                                     , n_dln
                   → idxf ,...
                  idxi
                            , f_dln_node , f_dln_se
4
                  \rightarrow f_dln ,...
                  mu, nu, a , use_gpu , n_threads ,
                   → para_scheme)
       ///-----///
     \mbox{\it \%} Written by famed MATLAB hater and fan of compiled languages,
     % Daniel Celis Garza, in 12/11/17--16/11/17.
     % Refactored into independent subroutines in
10
     → 11/12/2017--15/12/2017.
     ~ %-----%
     % Calculates analytical forces to the surfaces of a
     \rightarrow rectangular
     % cantilever whose nodes are labelled as follows:
13
     %
14
     % 4. ---- .3
15
     % /\
             /\
16
     % / \
              / \
     % 1. -\---- .2 \
18
     % \ \ \
19
        \ 8. ----\- .7
20
        \ /
     %
21
          \ /
     %
                 \ /
     % 5. ---- .6
23
     % ~z
24
     % / y
25
     % 1 /
26
     % 1/
     % /---->x
28
     %
29
```

```
% Everything has been arranged to conform with the analytical
30
       → solutions
      % for forces exerted on linear rectangular surface elements
         published by:
      % S. Queyreau, J. Marian, B.D. Wirth, A. Arsenlis, MSMSE,
          22(3):035004, (2014).
      %
33
         % Inputs
35
36
          %-----
      %
37
      % se node coord := dimension(n se*n nodes, 3). 2D array
38
          with the cartesian coordinates of the finite element nodes
          subject
          to traction boundary conditions. The analytical solution
      %
40
          requires
          they be arranged element by element in a specific order.
41
42
      % dln_node_coord := dimension(n_dln*2, 3). 2D array with
          the cartesian coordinates of the dislocation line nodes.
44
          The
          analytical solution requires they be arranged line segment
45
          by
      %
          line segment.
47
      % burgers := dimension(n_dln, 3). 2D array with the
          individual dislocation line segments' Burgers vector. The
49
          analytical solution needs the Burgers vector for each line
          segment.
      %
51
      % n_nodes := number of nodes per surface element.
      % n_nodes_t := total number of nodes with traction boundary
         conditions.
          It is needed to map the analytical solutions' forces to
55
          the force
```

```
array used to account for the forces exerted by the
56
           dislocation
       %
           ensemble on the finite element nodes. This does not
           necessarily
           contain a factor of n_nodes less entries than
       %
           se node coord because
           not all nodes in an element may have traction boundary
       %
59
           conditions,
           but still need to be included in the calculation due to
60
           the
           problem's formulation.
       %
61
       %
62
       % n_se := number of surface elements
64
       % n_dln := number of dislocation line segments
65
66
       % idxf := dimension(:). 1D array containing the indices of the
67
           force
           array used to account for the forces exerted by the
68
           dislocation
           ensemble on the finite element nodes. It is needed to map
           analytical solutions' forces to the right index. It is
       %
           usually
           equal to gamma*3 because only the nodes with traction
       %
71
           boundary
       %
           conditions are needed.
72
       %
73
       % idxi := dimension(n_nodes_t*n_nodes). 1D assumed shape array
74
       %
           containing the global label indices of the nodes whose
           tractions
           were calculated by the analytical solution.
       %
76
77
       % mu := shear modulus of material.
78
79
       % nu := poisson's ratio.
80
       %
81
```

```
% a := dislocation core size parameter (non-singular
82
        \rightarrow dislocations).
       %
       %
85
       % Flags:
       % use_gpu := Flag to use the Graphics Processing Unit (GPU) in
           calculation. If use_gpu == 1, the code is run by a
       %
        \hookrightarrow CUDA-enabled
           NVidia GPU. The code will crash if there is none, or if
       %
        → there is no
           compiled MEX CUDA file. If use_gpu == 0, a serial C code
       %
        \rightarrow will be
       %
           executed. The code will crash if there is no compiled MEX
        \rightarrow C file.
           If use_gpu is any other number, a MATLAB version will be
        \rightarrow used. This
           is much slower than either of the others.
       %
       %
        % Optional parameters:
       \% n_threads := when use_gpu ==1, it is the number of threads
        → per block
           to be used in the parallelisation. This may require some
101
           experimentation to optimise. Defaults to 512. Does nothing
       %
102
        \rightarrow if a
           GPU isn't used.
       %
103
104
       % para_scheme := when use_gpu is enabled, it dicates the
105
       % parallelisation scheme used. If para_scheme == 1
        \rightarrow parallelise over
           dislocation line segments. If para_scheme == 0 parallelise
       %
107
```

→ over

```
surface elements. Other parallelisation schemes can be
108
          added in the
       %
          CUDA C code.
109
110
111
          %-----%
       % Dummy variables
112
          114
       % f_{dln_{node}} := dimension(3*n_{se}, 4). 3D force on each
115
           surface element node.
116
117
       % f_{dln_se} := dimension(3*n_se, 1). Total 3D force on each
118
          surface
          element (sum over all nodes of an element). This isn't
119
          used to
          couple the forces to the finite element forces but can be
120
          used to
          provide the evolution of the forces exerted by the
121
          dislocations on
       %
          the surface elements.
122
123
       % tmp := dimension(n\_nodes). 1D array containing the at-most 4
124
           instances where a surface element node appears (a single
125
          node can
           be shared by up to 4 surface elements).
126
127
       % tmp2 := dimension(:). Assumed shape 1D array containing the
128
          at-most 4
           instances where a surface element node appears (a single
129
          node can
          be shared by up to 4 surface elements) but without any
130
          zero-valued
           entries because O indices crash MATLAB.
131
132
       % k := used to traverse idxi to the next block of 4 entries
133
       → which
          contain data relevant to the node in the
134
```

```
%
135
136
          % Inputs/Outputs
137
          %
139
      % f_{dln} := dimension(n_{nodes_t*3}). 1D array containing the
          forces
          exerted by the dislocations on the surface nodes. It is
141
          used to
          correct the finite element forces.
      %
142
      %
144
145
      %% Analytical force calculation.
146
      % Parallel CUDA C calculation.
      if use gpu == 1
148
          % Provide a default number of threads in case none is
149
           → given.
          if ~exist('n_threads', 'var')
150
             n threads = 512;
          end %if
152
          % Provide a default parallelisaion scheme in case none is
153
          → given.
          if ~exist('para_scheme', 'var')
154
              % Parallelise over dislocations.
             para_scheme = 1;
156
          end %if
157
          [f dln node(:, 1), f dln node(:, 2),...
           f_dln_node(:, 3), f_dln_node(:, 4),...
159
           f_dln_se] = nodal_surface_force_linear_rectangle_mex_cuda(
160
              . . .
                               dln_node_coord(:, 1),
161

    dln_node_coord(:, 2)

                                se_node_coord (:, 1), se_node_coord
162
```

```
se_node_coord (:, 3), se_node_coord
163
                                      , . . .
                                      burgers(:), mu, nu, a, n_se, n_dln,
164
                                      para_scheme);
165
        % Serial force calculation in C.
166
        elseif use_gpu == 0
167
            [f dln node(:, 1), f dln node(:, 2),...
168
             f dln node(:, 3), f dln node(:, 4),...
169
             f_dln_se] = nodal_surface_force_linear_rectangle_mex(
170
                                      dln_node_coord(:, 1),
171

→ dln_node_coord(:, 2),...

                                      se_node_coord (:, 1), se_node_coord
172
                                      \rightarrow (:, 2),...
                                      se_node_coord (:, 3), se_node_coord
173
                                      \rightarrow (:, 4),...
                                      burgers(:), mu, nu, a, n_se,
                                      \rightarrow n_dln);
        % Matlab version.
175
        else
176
            [f_dln_node(:, 1), f_dln_node(:, 2),...
177
             f dln node(:, 3), f dln node(:, 4),...
             f_dln_se] = NodalSurfForceLinearRectangle2(
179
              dln_node_coord(:, 1),
180

→ dln_node_coord(:, 2),...

                                      se_node_coord (:, 1), se_node_coord
181
                                      se_node_coord (:, 3), se_node_coord
182
                                      \rightarrow (:, 4),...
                                      burgers(:), mu, nu, a, n_se,
183
                                      \rightarrow n dln);
        end %if
184
185
        %% Map analytical nodal forces into a useful form for the
        → force superposition scheme.
        % Loop through the number of nodes.
187
        k = 0;
188
```

```
tmp = zeros(n_nodes, 1);
189
        for i = 1: n_nodes_t
190
            % Populate tmp array with the indices corresponding to
191
             → nodes of
            % the surface relevant surface element.
192
            tmp = idxi(1 + k: k + n\_nodes);
193
            % Obtain only the indices which are non-zero. Zero indices
194
             → me.a.n.
            % those nodes are not under traction boundary conditions.
195
            tmp2 = tmp(idxi(1+k: k + n_nodes, 1) \sim= 0);
196
            for j = 2:-1:0
197
                % The index is displaced by -2, -1, 0, corresponding
198
                 → to the
                % indices of the x, y, z coordinate respectively.
199
                % We add the force contributions from all surface
200
                 \rightarrow elements a node
                % is part of. This gives us the total x,y,z forces on
201
                 → each node.
                f_dln(idxf(i) - j) = sum(f_dln_node(tmp2 - j));
202
            end %for
203
            % Step to the block in idxi where the next surface node's
204
            → indices
            % are found.
205
            k = k + 4;
206
        end %for
207
        clear tmp; clear tmp2; clear k;
   end % function
209
```

Appendix C

Implementation of Analytical Forces Induced by Dislocations on Linear Rectangular Surface Elements

- C.1 Serial C Code MEX File
- C.2 Parallel CUDA C Code MEX File

Appendix D

Implementation of Analytical Forces Induced by Dislocations on Quadratic Triangular Surface Elements

- D.1 Serial C Code MEX File
- D.2 Parallel CUDA C Code MEX File

Appendix E

Talks

E.1 Durham July 12–14 2017

Bridging the gap between the microscopic and macroscopic world is an on-going challenge for science and technology. If we hope to understand complex emergent phenomena we need to study systems that blur the line between micro and macro. In materials science, one such area is the study of extended defects called dislocations; whose nucleation and movement mediate the permanent deformation of materials. These large, high energy defects present very complex and long ranged interactions with each other, crystal boundaries, impurities, free surfaces, and themselves. As such, their dynamics are difficult to study experimentally. So we make justified assumptions and simplifications and create models that let us study them in detail. However if our models are to prove useful in real applications, they must be continually refined and improved by weakening assumptions and removing simplifications. Unfortunately, with increased refinement comes increased computational cost and new challenges. Therefore, finding faster alternatives that do not sacrifice accuracy is of the utmost importance—better yet if the alternatives are more accurate or exact. With the advent of increasingly accessible graphics processor units (GPUs) typically used in video gaming, the power of parallel processing is no longer exclusive to researchers with access to supercomputers. In this talk we will discuss the GPU implementation of exact solutions for the forces dislocations exert on the surfaces of materials.



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