Section 5.3

Two dimensional oscillators Section 5.4

Damped oscillations

Read Sections 5.3 and 5.4.

Figure 5.7 (a) A restoring force that is proportional to \mathbf{r} defines the isotropic harmonic oscillator. (b) The mass at the center of this arrangement of springs would experience a net force of the form $\mathbf{F} = -k\mathbf{r}$ as it moves in the plane of the four springs.

for small oscillations

5.3. Two dimensional oscillators

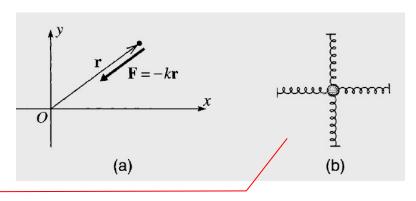
The definition of an "isotropic" oscillator is 2 or 3 dimensions is

$$F = -k r$$

$$U = \frac{1}{2} k r^2 = \frac{1}{2} k (x^2 + y^2 + z^2)$$

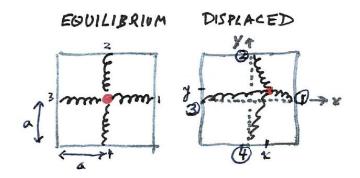
in 3 dimensions

<u>Figure 5.7</u> shows a 2d example; the particle (mass m) attached to the springs moves in the xy plane.



Comments about Figure 5.7.

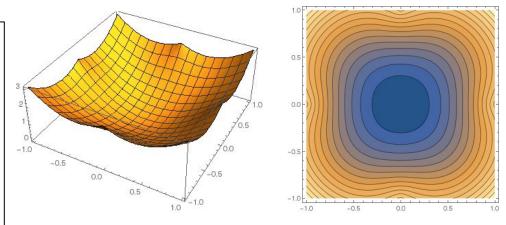
The particle (mass = m) attached to the springs moves in the xy plane.

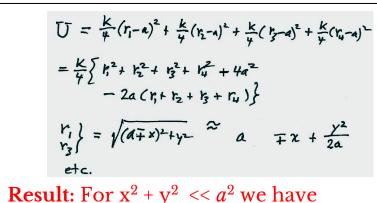


What is the potential energy when the particle is displaced to { x , y }?

 $2a \times 2a$.

Assume that the equilibrium length of each spring is *a*, and the spring constant is k/2. Also, the size of the square is





 $U \approx \frac{1}{2} k (x^2 + y^2) = \frac{1}{2} k r^2$ and $F \approx -k r$.

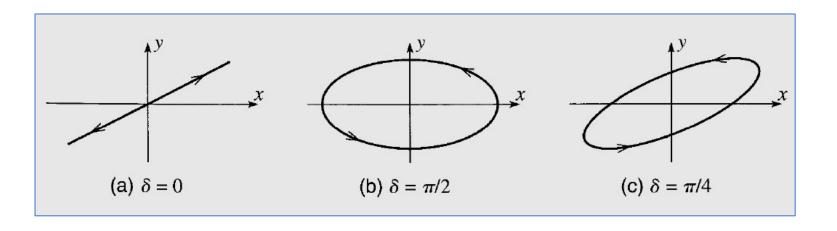
⁵ It is perhaps worth pointing out that one does *not* get a force of the form (5.17) by simply attaching a mass to a spring whose other end is anchored to the origin.

<u>Figure 5.8.</u> Three examples of isotropic oscillations in 2d:

$$U = \frac{1}{2} k x^{2} + \frac{1}{2} k y^{2}$$

$$x(t) = A \cos(\omega t)$$

$$y(t) = B \cos(\omega t - \delta)$$

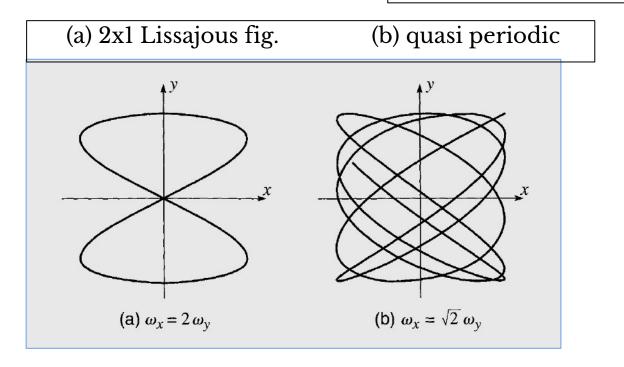


<u>Figure 5.9.</u> Two examples of anisotropic oscillations

$$U = \frac{1}{2} k_x x^2 + \frac{1}{2} k_y y^2$$

$$x(t) = A \cos(\omega_x t)$$

$$y(t) = B \cos(\omega_y t - \delta)$$



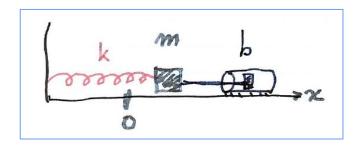
5.4. Damped oscillations

Sometimes in everyday life, oscillations may create problems.

For example, that's why a car has shock absorbers — to damp out the oscillations when the wheels hit a bump in the road or a pothole.

Go back to 1-dimensional oscillations, but now add damping.

Generic picture



The equation of motion is

$$ma = -bv - kx$$

Note the assumption of "linear damping"; i.e., $F_{damping} \propto -v$;

Or we can write it this way,

$$m \ddot{x} + b \dot{x} + k x = 0$$
.

It is useful to rescale the parameters to write the equation in a standard form;

$$x' + 2\beta x' + \omega_0^2 x = 0$$

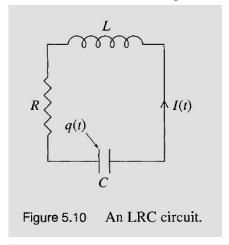
where

$$2\beta = b/m$$
 and $\omega_0^2 = k/m$.

Figure 5.10

THE EQUIVALENT LRC CIRCUIT

Recall from circuit theory



$$L\ddot{q} + R\dot{q} + \frac{1}{C}q = 0.$$

so the math is the same as for the mechanical system.

$$x + 2 \beta x + \omega_0^2 x = 0$$

This is an example of a "homogeneous linear differential equation with constant coefficients".

There is a standard method to solve this kind of diff.eq. (MTH 234)

First, try
$$x(t) = e^{\beta t}$$
.
 $\dot{x} = \rho e^{\beta t}$ and $\ddot{x} = \rho^2 e^{\beta t}$, so
$$\rho^2 + 2\beta \rho + \omega_o^2 = 0$$

$$\rho_{\pm} = -\beta \pm \sqrt{\beta^2 - \omega_o^2}$$

$$\ddot{x} + 2 \beta \dot{x} + \omega_0^2 x = 0$$

We have two solutions, $exp(p_t)$ and $exp(p_t)$ where

$$\beta \pm = -\beta \pm \sqrt{\beta^2 - \omega_0^2}$$

The equation is second order, so the *general solution* depends on two constants. The equation is linear so we can write the general solution as

$$\chi(t) = c_{+}e^{\beta_{+}t} + c_{-}e^{\beta_{-}t}$$

$$= e^{-\beta_{+}t} \left\{ c_{+}e^{\sqrt{\beta_{-}^{2}-\omega_{0}^{2}}t} + c_{-}e^{-\sqrt{\beta_{-}^{2}-\omega_{0}^{2}}t} \right\}$$

The 2 constants, c_+ and c_- , must be determined from the initial conditions or some other information.

Overdamped oscillator; $\beta > \omega_0$ This is the case of *strong* damping. In this case p₁ and p are *real*.

$$\chi(0) = C_{+} + C_{-}$$
 and $\sigma(0) = P + C_{+} + P_{-}C_{-}$

$$C_{\pm} = \left[P_{+} \times (0) - \sigma(0) \right] / \left(P_{+} - P_{\pm} \right)$$

■ <u>Underdamped oscillator</u>; $β < ω_0$ This is the case of *weak* damping. In this case p_1 and p_2 are complex numbers.

Recall $e^{\pm i \theta} = \cos \theta \pm i \sin \theta$ (Euler)

$$\chi(t) = e^{-\beta t} \left[A \cos \omega t + B \sin \omega_1 t \right]$$
where $\omega_1 = \sqrt{\omega_0^2 - \beta^2}$.
$$\chi(0) = A \text{ and } \dot{\chi}(0) = -\beta A + \omega_1 B$$

The critically damped oscillator

 $\beta = \omega_0$

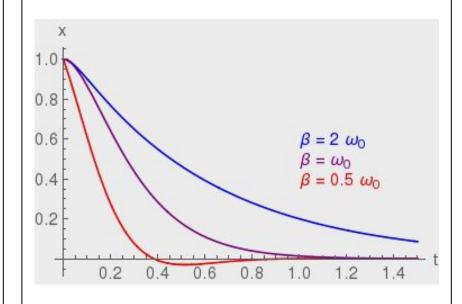
In this case p_+ and p_- are *equal*, $p_+ = p_- = \omega_0$; so exp(pt) is only one solution. To get the general solution we need another solution.

Exercise: Show that $x(t) = t \exp(pt) i$ s also a solution for the critically damped oscillator $(\beta = \omega_0)$.

$$\chi(t) = e^{-\beta t} \left[A + B t \right]$$

$$\chi(0) = A \text{ and } \dot{\chi}(0) = -\beta A + B$$

Example. Consider these initial conditions: x(0) = 1 and v(0) = 0.



<u>Figure 5.11</u> <u>Underdamped oscillator</u>

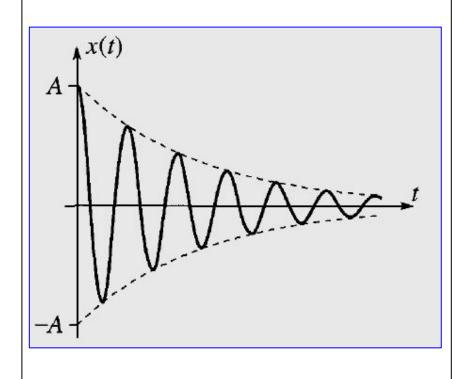


Figure 5.12 Overdamped oscillator

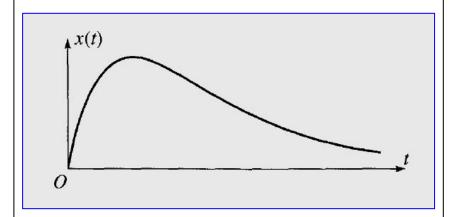
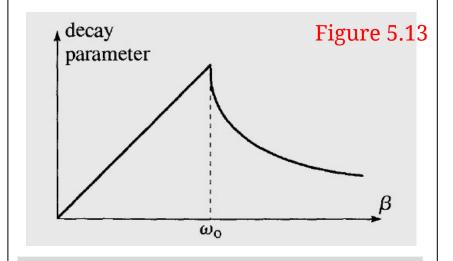


Fig. 5.12 corresponds to these initial conditions: x(0) = 0 and v(0) > 0; i.e., the mass is kicked in the +x direction, reaches a maximum displacement, and returns to equilibrium monotonically.

Critical damping $(\beta = \omega_0)$

This special case has the most rapid return to equilibrium ...

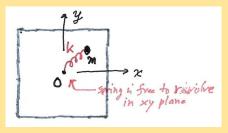


The "decay parameter" p versus β . The decay parameter is largest, so the motion dies out most quickly, for critical damping $\beta = \omega_0$.

damping	β	decay parameter
none	$\beta = 0$	0
under	$\beta < \omega_{\rm o}$	β
critical	$\beta = \omega_{\rm o}$	β
over	$\beta > \omega_{\rm o}$	$eta - \sqrt{eta^2 - \omega_{ m o}^2}$

10

Test yourself:



Mass m moves in the xy-plane, attached to a spring as shown. According to a footnote in Taylor, the force on m is not $-k\mathbf{r}$.

What is the force?

Homework Assignment #9
due in class Friday November 4
[41] Problem 4.41 and Problem 4.43
[42] Problem 5.3 *
[43] Problem 5.5 *
[44] Problem 5.9 *
[45] Problem 5.12 **
[46] Problem 5.18 ***

Use the cover sheet.