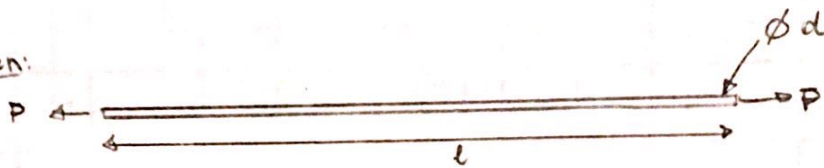


CH2: 2, 9, 17, 21, 33, 35, 43, 51, 58, 61, 79, 98, 100

2.2:

Given:



$$d = 0.25 \text{ in}$$

$$l = 4.8 \text{ ft} = 57.6 \text{ in}$$

$$E = 29 \cdot 10^6 \text{ psi}$$

$$P = 750 \text{ lb}$$

Find: a) elongation of wire -
b) normal stress σ

Solution:

$$E = \frac{\sigma}{\epsilon}, \quad \sigma = \frac{P}{A}, \quad A = \frac{\pi}{4} d^2 = \frac{\pi}{64} \text{ in}^2$$

$$\sigma = \frac{750 \text{ lb}}{\frac{\pi}{64} \text{ in}^2}$$

$$\sigma = 15,279 \text{ psi} \quad \text{b)}$$

$$\epsilon = \frac{\sigma}{E} = \frac{15,279 \text{ psi}}{29 \cdot 10^6 \text{ psi}}$$

$$\epsilon = 5.27 \cdot 10^{-4}$$

let e be elongation:note: will use δ from now on

$$e = \epsilon \cdot l$$

$$e = 0.0303 \text{ in} \quad \text{a)}$$

2.9:

Given: steel rod \rightarrow $l = 4\text{ m}$
 $\delta \leq 2\text{ mm} = 0.002\text{ m}$
 $\sigma_m \leq 150\text{ MPa}$
 $P = 10\text{ kN}$
 $E = 200\text{ GPa}$

Find: required diameter d

Solution:

① stress limit:

$$\sigma_m = \frac{P}{A}$$

$$A = \frac{P}{\sigma_m}, \quad A = \frac{\pi}{4} d^2$$

$$\rightarrow d_\sigma = \sqrt{\frac{P}{\sigma_m} \cdot \frac{4}{\pi}} \quad \text{①}$$

$$d_\sigma = 0.0092\text{ m} = 9.2\text{ mm}$$

② strain limit:

$$\delta = \epsilon \cdot l$$

$$\epsilon = \frac{\delta}{l}$$

$$E = \frac{\sigma}{\epsilon}$$

$$\sigma = E\epsilon$$

from ①: $d = \sqrt{\frac{P}{\sigma} \cdot \frac{4}{\pi}}, \quad \sigma = E\frac{\delta}{l}$

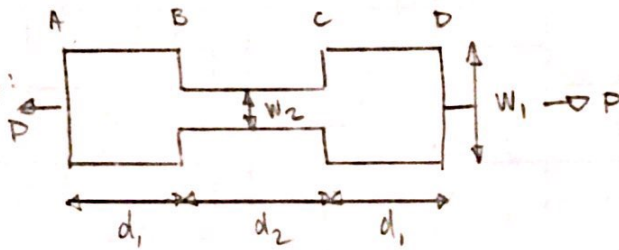
$$d = \sqrt{\frac{Pl}{E\epsilon} \cdot \frac{4}{\pi}}$$

$$d = 0.0092\text{ m} = \boxed{9.2\text{ mm}}$$

oh, they're the same!

2.17:

Given:



$$\begin{aligned} w_1 &= 1 \text{ in}, w_2 = 0.4 \text{ in} \\ d_1 &= 1.6 \text{ in}, d_2 = 2 \text{ in} \\ t &= 0.25 \text{ in} \\ P &= 350 \text{ lb} \\ E &= 0.45 \cdot 10^6 \text{ psi} \end{aligned}$$

Find: a) total deformation of specimen
b) deformation of BC

Solution:

$$\begin{aligned} a) \quad \delta &= \int_0^L \frac{P dx}{AE} \\ &= \int_A^B \frac{P dx}{A_{AB} E} + \int_B^C \frac{P dx}{A_{BC} E} + \int_C^D \frac{P dx}{A_{CD} E} \end{aligned}$$

① ② ③

note, ① and ③ are the same

$$A_{AB} = w_1 t, A_{BC} = w_2 t$$

$$\begin{aligned} \rightarrow \delta &= 2 \int_A^B \frac{P dx}{w_1 t E} + \int_B^C \frac{P dx}{w_2 t E} \\ &= \frac{P}{t E} \left(\frac{2d_1}{w_1} + \frac{d_2}{w_2} \right) \end{aligned}$$

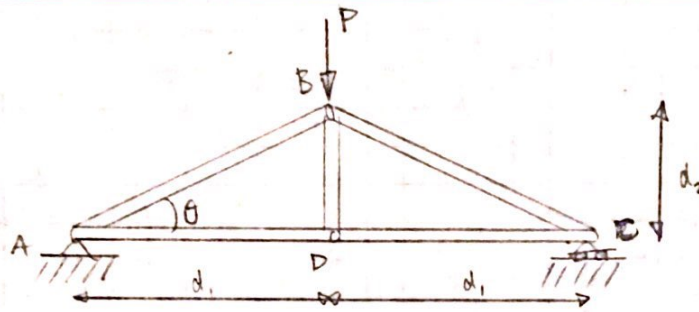
$$\boxed{\delta_t = 0.0255 \text{ in}}$$

$$b) \quad \delta_{BC} = \frac{P}{t E} \frac{d_2}{w_2}$$

$$\rightarrow \boxed{\delta_{BC} = 0.0156 \text{ in}}$$

2.21:

Given:



$$P = 228 \text{ kN}$$

$$d_1 = 4 \text{ m} \quad d_2 = 2.5 \text{ m}$$

$$E = 200 \text{ GPa}$$

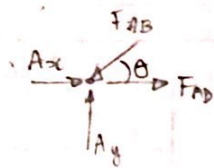
$$A_{AB} = 2400 \text{ mm}^2$$

$$A_{AD} = 1800 \text{ mm}^2$$

Find: δ_{AB} , δ_{AD}

Solution: note BD is zero force member due to no loading at D.

at A:



$$A_y, C_y = \frac{P}{2}$$

$$\theta = \tan^{-1}\left(\frac{d_2}{d_1}\right) = 32^\circ$$

$$\rightarrow A_y = F_{AB} \sin \theta$$

$$F_{AB} = \frac{A_y}{\sin \theta}$$

$$\Sigma F_y$$

$$F_{AB} = 215.1 \text{ kN}$$

$$F_{AB} \cos \theta = F_{AD}$$

$$\Sigma F_x$$

$$F_{AD} = 182.4 \text{ kN}$$

$$\delta = \frac{Pl}{AE}, \quad l_{AB} = \sqrt{d_1^2 + d_2^2}$$

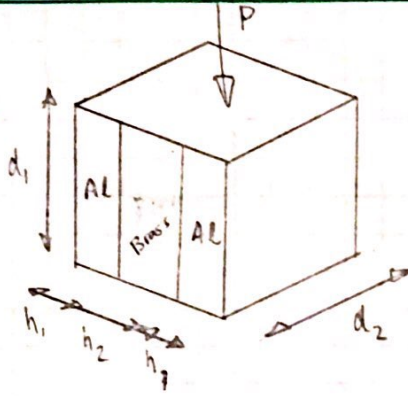
$$l_{AB} = 4.72 \text{ m}, \quad l_{AD} = 4 \text{ m}$$

$$\delta_{AB} = \frac{F_{AD} l_{AB}}{A_{AD} E} = -0.00212 \text{ m} = -2.12 \text{ mm}$$

$$\delta_{AD} = \frac{F_{AD} l_{AD}}{A_{AD} E} = +0.00202 \text{ m} = +2.03 \text{ mm}$$

2.33.

Given:



$$E_{AL} = 70 \text{ GPa}$$

$$E_B = 105 \text{ GPa}$$

$$d_1 = 300 \text{ mm} \quad d_2 = 60 \text{ mm}$$

$$h_1 = 10 \text{ mm} \quad h_2 = 40 \text{ mm}$$

$$P = 450 \text{ kN}$$

Find: σ_B , σ_{AL}

Solution:

$$\delta_{A1} = \frac{P_A l}{A_A E_A}$$

$$l = d_1, \quad A_A = 2 \cdot d_2 h_1$$

$$\delta_B = \frac{P_B l}{A_B E_B}$$

$$l = d_1, \quad A_B = d_2 h_2$$

$$\text{and } \delta_A = \delta_B$$

$$\rightarrow \frac{P_B l}{A_B E_B} = \frac{P_A l}{A_A E_A}, \quad \text{and } P_A + P_B = P$$

system of eq:

P_A	P_B	k
$+\frac{1}{A_A E_A}$	$-\frac{1}{A_B E_B}$	0
1	1	P

$$\Rightarrow P_A = 113 \text{ kN}$$

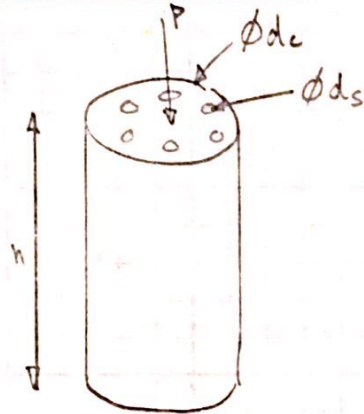
$$P_B = 337 \text{ kN}$$

$$\rightarrow \sigma_A = \frac{P_A}{A_A} = -94 \text{ MPa}$$

$$\sigma_B = \frac{P_B}{A_B} = -140 \text{ MPa}$$

2.35:

Given:



$$\begin{aligned} d_c &= 18 \text{ in} \\ d_s &= 1.125 \text{ in} \\ h &= 54 \text{ in} \\ E_c &= 4.2 \cdot 10^6 \text{ psi} \\ E_s &= 29 \cdot 10^6 \text{ psi} \\ P &= 350 \text{ kip} \end{aligned}$$

Find: σ_s , σ_c

Solution: $A_s = 6 \cdot \frac{\pi}{4} d_s^2 = 5.964 \text{ in}^2$

$$A_c = \frac{\pi}{4} d_c^2 - A_s = 248.5 \text{ in}^2$$

$$\delta_c = \frac{P_c h}{A_c E_c} \quad \delta_s = \frac{P_s h}{A_s E_s}, \quad \delta_s = \delta_c \quad P_s + P_c = P$$

from book, $P_s = \frac{A_s E_s P}{A_s E_s + A_c E_c}$

$$P_s = 49.75 \text{ kip}$$

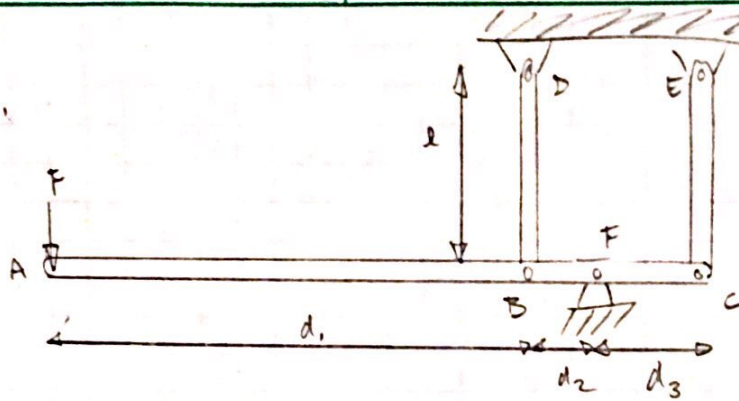
$$\rightarrow P_c = 300.25 \text{ kip}$$

$$\sigma_s = \frac{P_s}{A_s} = -8.342 \text{ ksi}$$

$$\sigma_c = \frac{P_c}{A_c} = -1.208 \text{ ksi}$$

2.43:

Given:



$$\begin{aligned} F &= 2 \text{ kN} \\ l &= 225 \text{ mm} \\ d_1 &= 550 \text{ mm} \\ d_2 &= 75 \text{ mm} \\ d_3 &= 100 \text{ mm} \\ A_{DB} &= A_{EC} = 200 \text{ mm}^2 \\ E &= 105 \text{ GPa} \end{aligned}$$

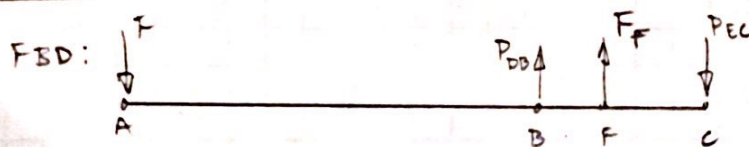
Find: deflection at A, given rigid ABC

Solution: by similar triangles,

$$\delta_{BD} = \frac{d_2}{d_3} \delta_{EC} = 0.75 \delta_{EC}$$

what we're looking for $\rightarrow \delta_A = \frac{(d_1 + d_2)}{d_3} \delta_{EC} = 6.25 \delta_{EC}$

determine forces:



$$\sum F_y: P_{DB} + F_F - P_{EC} - F = 0 \quad (1)$$

$$\sum M_C: +F(d_1 + d_2 + d_3) - P_{DB}(d_2 + d_3) - F_F(d_3) = 0 \quad (2)$$

$$\delta_{BD} = \frac{P_{DB} l}{AE}, \quad \delta_{EC} = \frac{P_{EC} l}{AE}$$

$$\text{since } \delta_{BD} = 0.75 \delta_{EC} \rightarrow P_{DB} = 0.75 P_{EC} \quad (3)$$

system of eq:	P_{DB}	F_F	P_{EC}	lc
	1	1	-1	F
	$+(d_2 + d_3)$	$+d_3$	0	$F(d_1 + d_2 + d_3)$
	1	0	-0.75	0

$$\begin{aligned} P_{DB} &= 6 \text{ kN} \\ F_F &= 4 \text{ kN} \\ P_{EC} &= 8 \text{ kN} \end{aligned}$$

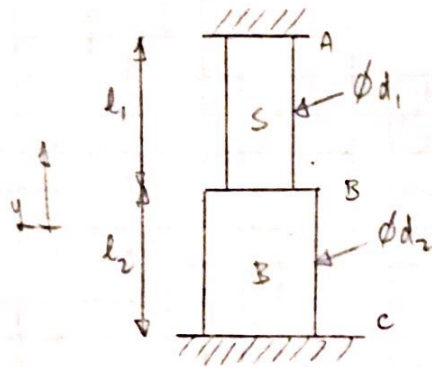
$$\delta_{EC} = \frac{P_{EC} l}{AE} = 8.57 \cdot 10^{-5} \text{ m}$$

$$\delta_A = \delta_{EC} \frac{(d_1 + d_2)}{d_3} = 5.35 \cdot 10^{-4} \text{ m}$$

$$\Rightarrow \delta_A = 0.535 \text{ mm}$$

251:

Given:



$$l_1 = 250 \text{ mm}, l_2 = 300 \text{ mm}$$

$$d_1 = 30 \text{ mm}, d_2 = 50 \text{ mm}$$

$$E_S = 200 \text{ GPa}$$

$$E_B = 105 \text{ GPa}$$

$$\alpha_S = 11.7 \cdot 10^{-6} / ^\circ\text{C}$$

$$\alpha_B = 20.9 \cdot 10^{-6} / ^\circ\text{C}$$

Find: P_c compressive force when T temperature rises 50°C

Solution: $\delta_{TB} + \delta_{TS} = \delta_{PB} + \delta_{PS}$

$$\delta_{TB} = \alpha_B \Delta T l_2$$

$$\delta_{TS} = \alpha_S \Delta T l_1$$

$$\delta_{PB} = \frac{P l_2}{A_B E_B}$$

$$\delta_{PS} = \frac{P l_1}{A_S E_S}$$

$$\rightarrow P = \frac{\delta_{PB} A_B E_B}{l_2}$$

$$P = \frac{\delta_{PS} A_S E_S}{l_1}$$

$$\rightarrow \delta_{PB} = \delta_{PS} \cdot \frac{l_2}{l_1} \cdot \frac{A_S E_S}{A_B E_B}$$

$$\rightarrow \delta_{TB} + \delta_{TS} = \delta_{PS} \left[1 + \frac{l_2}{l_1} \frac{A_S E_S}{A_B E_B} \right]$$

$$\delta_{TB} = \alpha_B \Delta T l_2$$

$$= 0.3135 \text{ mm}$$

$$A_B = \frac{\pi}{4} d_2^2$$

$$= 1.96 \cdot 10^{-3} \text{ m}^2$$

$$\delta_{TS} = \alpha_S \Delta T l_1$$

$$= 0.1463 \text{ mm}$$

$$A_S = \frac{\pi}{4} d_1^2$$

$$= 7.07 \cdot 10^{-4} \text{ m}^2$$

$$\rightarrow 0.4598 \text{ mm} = 1.82 \delta_{PS}$$

$$\delta_{PS} = 0.252 \text{ mm}$$

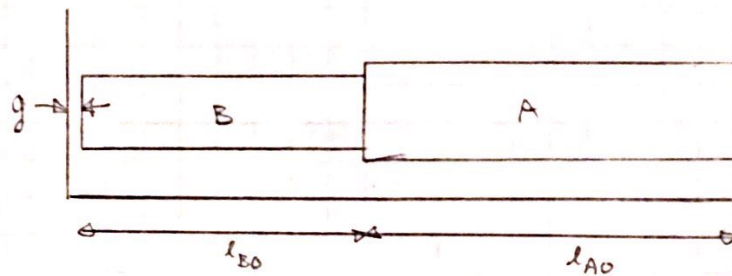
$$\delta_{PS} + \delta_{PB} = 0.4598 \text{ mm}$$

$$\rightarrow \delta_{PB} = 1.97 \text{ mm}$$

$$P = \frac{\delta_{PS} A_S E_S}{l_1} = 142.5 \text{ kN}$$

2.58:

Given:



$$\begin{aligned} g &= 0.02 \text{ in} \\ l_{B0} &= 14 \text{ in} \\ l_{A0} &= 18 \text{ in} \\ T_0 &= 75^\circ \text{F} \end{aligned}$$

Bronze: $A = 2.4 \text{ in}^2$
 $E = 15 \cdot 10^6 \text{ psi}$
 $\alpha_B = 12 \cdot 10^{-6} / ^\circ \text{F}$

Aluminum: $A = 2.8 \text{ in}^2$
 $E = 10.6 \cdot 10^6 \text{ psi}$
 $\alpha_A = 12.9 \cdot 10^{-6} / ^\circ \text{F}$

Find: a) temperature T when $\sigma_A = -11 \text{ ksi}$

b) l_A at T

Solution: first solve for δ 's to close gap, then δ 's under stress

stage ① $\delta_{TB} + \delta_{TA} = g$, $\delta_T = \alpha \Delta T L$

$$\alpha_B \Delta T l_{B0} + \alpha_A \Delta T l_{A0} = g$$

$$\Rightarrow \Delta T_g = \frac{g}{\alpha_B l_{B0} + \alpha_A l_{A0}}$$

$$\Delta T_g = 50^\circ \text{C} \text{ to close gap}$$

stage ②

now, with gap closed:

$$\delta_{TB} + \delta_{TA} = \delta_{PB} + \delta_{PA} \quad ①$$

recall, $\delta_T = \alpha \Delta T L$, $\delta_P = \frac{PL}{EA}$

$$l_{B1} = l_{B0} + \delta_{TB0}$$

$$= l_{B0} + \alpha_B \Delta T_g l_{B0}$$

$$l_{B1} = 14.01 \text{ in}$$

$$l_{A1} = l_{A0} (1 + \alpha_A \Delta T_g)$$

$$l_{A1} = 18.01 \text{ in}$$

from ① $\Delta T_c (\alpha_B l_{B1} + \alpha_A l_{A1}) = P \left(\frac{l_{B1}}{E_B A_B} + \frac{l_{A1}}{E_A A_A} \right)$

we know $\sigma_A = -11 \text{ ksi}$

$$\Rightarrow \sigma_A = \frac{P}{A_A} \Rightarrow P = \sigma_A A_A$$

$$P = 30.8 \text{ ksi}$$

$$\Rightarrow \Delta T_c = P \left(\frac{l_{B1}}{E_B A_B} + \frac{l_{A1}}{E_A A_A} \right) \cdot (\alpha_B l_{B1} + \alpha_A l_{A1})^{-1}$$

$$\Delta T_c = 76.6^\circ \text{F}$$

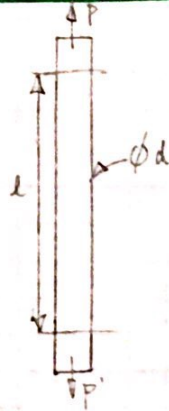
$$\Rightarrow T_f = T_0 + T_g + T_c$$

$$T_f = 201.6^\circ \text{F}$$

$$l_{A1} = 18.01 \text{ in}$$

2.61:

Given:



$$\begin{aligned}l &= 50 \text{ in} \\d &= 0.625 \text{ in} \\P &= 800 \text{ lb} \\\delta_l &= 0.45 \text{ in} \\\delta_d &= -0.025 \text{ in}\end{aligned}$$

Find: E , G , ν

Solution:

$$\delta_l = \frac{Pl}{EA} \rightarrow E = \frac{Pl}{\delta_l A}$$

$$E = 28973 \text{ psi}$$

$$\nu = -\frac{\epsilon_d}{\epsilon_l}$$

$$\epsilon_d = \frac{\delta_d}{d}, \quad \epsilon_l = \frac{\delta_l}{l}$$

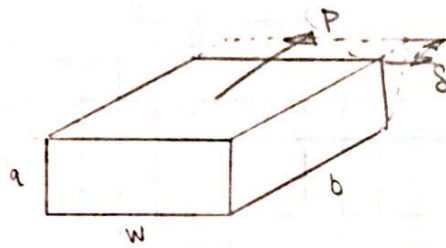
$$\nu = 0.444$$

$$G = \frac{E}{2(1+\nu)}$$

$$G = 10032 \text{ psi}$$

2.79:

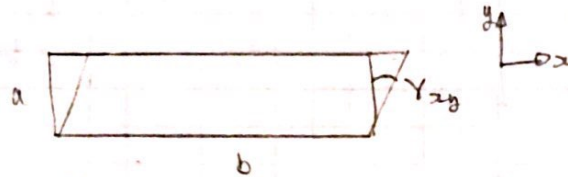
Given:



$$\begin{aligned} w &= 8 \text{ in} \\ P &= 5 \text{ kip} \\ \delta &= 0.375 \text{ in max} \\ \tau &= 60 \text{ psi max} \\ G &= 130 \text{ psi} \end{aligned}$$

Find: a) smallest allowable b
b) smallest required thickness a

Solution:



$$a) \tau = \frac{P}{A}, \quad A = bw$$

$$\rightarrow \tau = \frac{P}{bw}$$

$$b = \frac{P}{\tau w}$$

$$b = 10.4 \text{ in}$$

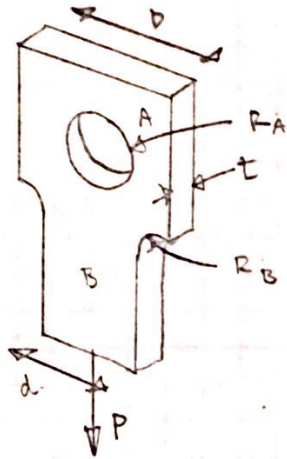
$$b) \tau = G\gamma, \quad \gamma = \frac{\delta}{a} \quad \rightarrow a = \frac{\delta}{\gamma}$$

$$\rightarrow a = \frac{\delta G}{\tau}$$

$$a = 0.8125 \text{ in}$$

2.98:

Given:



$$\begin{aligned} D &= 88 \text{ mm} \\ d &= 64 \text{ mm} \\ R_A &= 20 \text{ mm} \\ R_B &= 15 \text{ mm} \\ P &= 100 \text{ kN} \end{aligned}$$

Find: minimum t for allowable $\sigma = 125 \text{ MPa}$

Hole: $\sigma_{\max} = K \sigma_{\text{ave}}$

$$\frac{2r}{D} = \frac{2R_A}{D} = 0.455 \rightarrow K_h \approx 2.2$$

$$\sigma_{\text{ave}} = \frac{P}{A} \quad A = (D - 2R_A)(t)$$

$$\sigma_{\max} = K \frac{P}{(D - 2R_A)t}$$

$$\rightarrow t = K \frac{P}{(D - 2R_A)\sigma_{\max}}$$

$$t_{\min} = 0.0367 \text{ m} = 36.7 \text{ mm}$$

Fillet: $\frac{r}{d} \rightarrow \frac{R_B}{d} = 0.234$ } $K \approx 1.6$

$$\frac{D}{d} = 1.375$$

$$\sigma_{\text{ave}} = \frac{P}{dt}$$

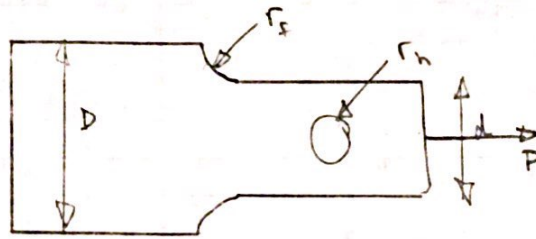
$$\sigma_{\max} = K \frac{P}{dt}$$

$$\rightarrow t = \frac{KP}{d\sigma_{\max}} = 0.02 \text{ m} = 20 \text{ mm}$$

largest of two minimums = 36.7 mm

2.100 :

Given:



$$\begin{aligned} r_f &= 0.5 \text{ in} \\ r_h &= 0.5 \text{ in} \\ D &= 6.5 \text{ in} \\ d &= 5 \text{ in} \\ t &= 0.75 \text{ in} \\ \sigma_{all} &= 20 \text{ ksi} \end{aligned}$$

Find: maximum allowable P

Solution:

fillet: $\frac{D}{d} = 1.3 \quad \frac{r_f}{d} = 0.1$

$$\rightarrow K_f \approx 2.05$$

hole: $\frac{2r_h}{d} = 0.2$

$$\rightarrow K_h \approx 2.5$$

$$\sigma_{avef} = \frac{P_f}{A_f} \quad A_f = dt$$

$$\sigma_{avef} = \frac{\sigma_{max}}{K_f}$$

$$P_f = A_f \frac{\sigma_{max}}{K_f} = 3.66 \cdot 10^4 \text{ lb}$$

$$\sigma_{aveh} = \frac{P_h}{A_h} \quad A_h = (d - 2r_h)t$$

$$P_h = A_h \frac{\sigma_{max}}{K_h}$$

lowest of two is max allowable P

$$\rightarrow P_{max} = 2.4 \cdot 10^4 \text{ lb}$$