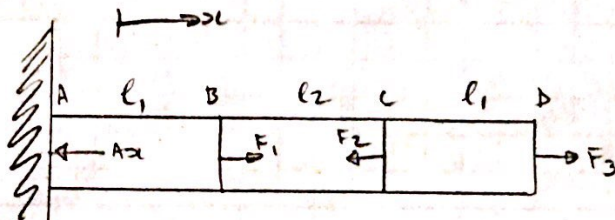


Honor Statement:

I have neither given nor received unauthorized assistance on this Midterm.

signed:  Sean Lai

Q1:



$$A = 900 \text{ mm}^2$$

$$F_1 = 45 \text{ kN}$$

$$F_2 = 90 \text{ kN}$$

$$F_3 = 70 \text{ kN}$$

$$\alpha = 11.7 \cdot 10^{-6} / ^\circ\text{C}$$

$$E = 200 \text{ GPa}$$

$$l_1 = 800 \text{ mm}$$

$$l_2 = 1200 \text{ mm}$$

$$@ 28^\circ\text{C}$$

$$\sum F_x: -A_x + F_1 - F_2 + F_3 = 0$$

$$A_x = -90 \text{ kN} + 45 \text{ kN} + 70 \text{ kN}$$

$$A_x = 25 \text{ kN}$$

a) 1D:

$$AB: P_{AB} = -A_x$$

$$\rightarrow \sigma_{AB} = \frac{P}{A} = \frac{-25 \text{ kN}}{900 \text{ mm}^2} = \boxed{27.8 \text{ MPa tension (AB)}}$$

$$BC: P_{BC} = -A_x + F_1$$

$$\rightarrow \sigma_{BC} = \frac{-A_x + F_1}{A} = \frac{20 \text{ kN}}{900 \text{ mm}^2} = \boxed{22.2 \text{ MPa compression (BC)}}$$

$$CD: P_{CD} = -A_x + F_1 - F_2$$

$$\rightarrow \sigma_{CD} = \frac{-70 \text{ kN}}{900 \text{ mm}^2} = \boxed{77.8 \text{ MPa tension (CD)}}$$

$$b) \delta = \frac{PL}{AE}$$

$$\delta_{\text{tot}} = \delta_{AB} + \delta_{BC} + \delta_{CD}, \text{ tension } +, \text{ comp. } -$$

$$= \frac{1}{E} (\sigma_{AB} l_1 + \sigma_{BC} l_2 + \sigma_{CD} l_1)$$

$$= \frac{1}{E} [(22.8 + 77.8 \text{ MPa})(800 \text{ mm}) - (22.2 \text{ MPa})(1200 \text{ mm})]$$

$$\delta_{\text{tot}} = 2.92 \cdot 10^{-4} \text{ m}$$

$$\rightarrow \boxed{\delta_{\text{totb}} = 0.292 \text{ mm}}$$

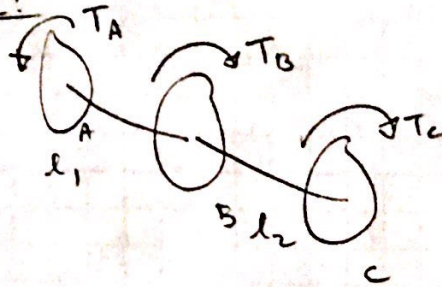
$$c) \delta_{\text{temp}} = \alpha \Delta T \cdot L, \Delta T = 140^\circ\text{C} - 28^\circ\text{C}, L = 2l_1 + l_2$$

$$\delta_{\text{temp}} = 11.7 \cdot 10^{-6} \cdot 112 \cdot 2.8 \text{ m}$$

$$\delta_{\text{temp}} = 3.67 \cdot 10^{-3} \text{ m} = 3.67 \text{ mm}$$

$$\rightarrow \delta_{\text{tot}} = \delta_{\text{totb}} + \delta_{\text{temp}} = \boxed{3.961 \text{ mm}}$$

Q2:



$$\begin{aligned} T_A &= 1600 \text{ Nm} \\ T_B &= 900 \text{ Nm} \\ T_C &= 700 \text{ Nm} \\ G &= 77.9 \text{ GPa} \end{aligned}$$

$$\begin{aligned} l_1 &= 500 \text{ mm} \\ l_2 &= 900 \text{ mm} \\ \phi d &= 30 \text{ mm} \end{aligned}$$

$$\phi = \frac{Tl}{JG}, \quad J = \frac{\pi}{2} c^4, \quad c = 15 \text{ mm}$$

$$\phi_{B/A} = \frac{T_B l_1}{\frac{\pi}{2} c^4 \cdot G}$$

$$\boxed{\phi_{B/A} = 0.00735 \text{ rad} = 4.21^\circ}$$

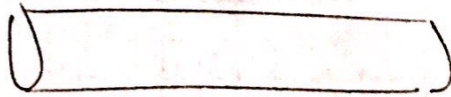
$$\phi_{C/A} = \phi_{B/A} + \phi_{C/B}$$

$$\phi_{C/B} = \frac{T_C l_2 \cdot 2}{\pi c^4 \cdot G}$$

$$\phi_{C/B} = 0.163 \text{ rad} = 5.90^\circ$$

$$\rightarrow \boxed{\phi_{C/A} = 5.90^\circ + 4.21^\circ = 10.11^\circ}$$

Q3:



$$P = 120 \text{ hp}, \quad \omega = 1200 \text{ rpm}$$
$$\tau_{\max} = 7000 \text{ ksi}$$

$$P = T \cdot \omega$$

$$T = \frac{P}{\omega}$$

$$T = \frac{120 \text{ hp} \cdot 6600 \text{ lb}\cdot\text{in}/\text{s}/\text{hp}}{1200 \text{ rpm} \cdot \frac{2\pi}{60} \frac{\text{rad}}{\text{s}} / \frac{\text{rev}}{\text{min}}}$$

$$T = 6.3 \cdot 10^3 \text{ lb}\cdot\text{in} \text{ or } 6.3 \text{ kip}\cdot\text{in}$$

$$\tau_{\max} = \frac{Tc}{J}, \quad J = \frac{\pi}{2} c^4$$

$$\tau_{\max} = \frac{Tc \cdot 2}{\pi c^4}$$

$$c^3 = \frac{2T}{\pi \tau_{\max}}$$

$$c = \left(\frac{2T}{\pi \tau_{\max}} \right)^{1/3}$$

$$c = \left(\frac{2 \cdot 6.3 \text{ kip}\cdot\text{in}}{\pi \cdot 7000 \text{ ksi}} \right)^{1/3}$$

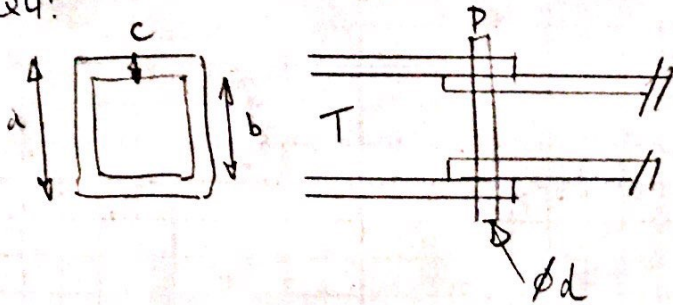
$$c = 8.31 \cdot 10^{-2} \text{ in}$$

$$d = 2c = 0.166 \text{ in}$$

seems small... but τ_{\max} of 7000 ksi
- seems large.

shaft must have a diameter of 0.166 in to transmit
120 hp @ 1200 rpm for $\tau_{\max} = 7000 \text{ ksi}$

Q4:



$$\begin{aligned} a &= 2.5 \text{ in} \\ b &= 2.0 \text{ in} \\ c &= 0.25 \text{ in} \end{aligned}$$

$$\begin{aligned} \sigma_p &= 40 \text{ ksi} & \tau_p &= 21 \text{ ksi} \\ & & \tau_T &= 30 \text{ ksi} \end{aligned}$$

$$d = 0.625 \text{ in}$$

a) $\tau_{max} = \frac{P}{A}$, $A = \pi \left(\frac{d}{2}\right)^2$

2 shearing points:

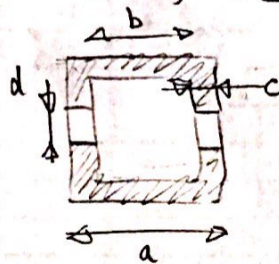
$$\begin{aligned} P &= \tau_T \cdot 2A \\ P &= 18.41 \text{ kips} \end{aligned}$$

F.O.S of 5 $\rightarrow P_{all} = \frac{P}{5} = 3.68 \text{ kips}$

b) $P = 7.5 \text{ kips}$ (note: this is outside of F.O.S. of 5)
Assuming axial load is tension, i.e. not compression when braking down a hill.

$$A = a^2 - b^2 - 2dc$$

$$A = 1.938 \text{ in}^2$$



$$\sigma = \frac{P}{A}$$

$$\sigma = \frac{7.5 \text{ kips}}{1.938 \text{ in}^2} = 3.871 \text{ ksi}$$

c) $P = 7.5 \text{ kips}$, $\sigma_{ball} = 40 \text{ ksi}$

$$\sigma_B = \frac{P}{A} , A = 2 \cdot dc = 0.3125 \text{ in}^2$$

$$\sigma_B = \frac{7.5 \text{ kips}}{0.3125 \text{ in}^2} = 24 \text{ ksi}$$

$$F.O.S. = \frac{\sigma_{ball}}{\sigma_B} = \frac{40}{24} \text{ ksi}$$

$$F.O.S. = 1.67$$

Q5.

$$\nu = \frac{\text{lateral } \epsilon}{\text{axial } \epsilon}$$

so larger ν means more lateral ϵ for each quantity of axial ϵ
→ greater thinning/necking

$$\nu_A = 0.21, \quad \nu_B = 0.33$$



B



A

Q6:

$$\tau = \frac{T_c}{J}, \quad J = \frac{\pi}{2} c^4$$

$$\tau_A = \frac{2T}{\pi c^3}, \quad c = \frac{1}{2} r$$

$$\tau_A = \frac{16T}{\pi r^3} \quad (1)$$

$$\tau_B = \frac{2T}{\pi c^3}, \quad c = r$$

$$\tau_B = \frac{2T}{\pi r^3} \quad (2)$$

$$\tau_c = \frac{2T}{\pi} \cdot \frac{c_0}{c_0^4 - c_1^4}, \quad c_0 = \frac{2}{3} r, \quad c_1 = \frac{1}{2} r$$

$$\tau_c = \frac{2T}{\pi} \cdot \frac{\frac{2}{3} r}{\frac{16}{81} r^4 - \frac{1}{16} r^4}$$

$$\tau_c = \frac{2T}{\pi} \cdot \frac{4.93}{r^3}$$

$$\tau_c \approx \frac{10T}{\pi r^3} \quad (3)$$

$$\tau_A > \tau_c > \tau_B$$