

# Extension of the entropy viscosity method to the low Mach multi-D Euler equations and the seven-equation model.

by  
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# Outline:

- 1 Introduction
- 2 The multi-D Euler equations with variable area
- 3 The multi-D seven-equation model with variable area
- 4 Conclusion

introduction → hyperbolic system of equations and real life

# Mathematical properties

Let us consider a hyperbolic conservation law of the form:

$$\partial_t U + \nabla \cdot F(U) = \mathbf{0} \text{ (strong form)}$$

- wave-dominated problem  $\rightarrow$  eigenvalues.
- known to form shocks even with smooth initial conditions: characteristic equation and variable (REFS)
- require a numerical stabilization method to resolve shocks: approximate Riemann solver, artificial dissipative method, flux-limiting method, ...
- uniqueness of the weak solution is ensured by an entropy condition: the associated entropy equation satisfies an inequality and is peaked in the shock region.

$$\partial_t S(U) + \nabla \Phi(U) \geq 0$$

# The general idea behind the entropy viscosity method

*It is an artificial dissipation method with smart viscosity coefficient capable of tracking the shock so that dissipation is only added into the shock region.*

- it requires a viscous regularization: the dissipative terms are consistent with the entropy inequality.
- each viscosity coefficient is function of a high-order viscosity coefficient and an upper bound called first-order viscosity coefficient.
- the high-order viscosity coefficient is defined proportional to the entropy residual.
- the first-order viscosity coefficient is function of the local maximum eigenvalue.
- also accounts for the inter element jumps → make the definition of the viscosity coefficients also sensitive to all discontinuities.

# The Multiphysics Object-Oriented Simulations Environment (MOOSE)

- Explicit and implicit temporal integrators (first and second-order accuracy)
- Continuous and Discontinuous Galerkin Finite Element Method
- Mesh adaptivity, time step adaptivity
- Support parallel runs
- Built on PETSc and Libmesh
- For implicit solve, requires a preconditioner → either FDP or hard-coded preconditioner
- C++ language, open source

All numerical solutions were run with BDF2 and linear test functions → second-order accuracy in time and space.

## BurRer's Equation (BadGER)



# A simple example of application of the EVM: the 1-D Burger's equation

The 1-D Burger's equation with its viscous regularization

$$\partial_t u(x, t) + \partial_x \left( \frac{u(x, t)^2}{2} \right) = \partial_x (\mu(x, t) \partial_x u(x, t))$$

Definition of the local viscosity coefficient  $\mu(x, t)$

$$\mu(x, t) = \min(\mu_e(x, t), \mu_{max}(x, t))$$

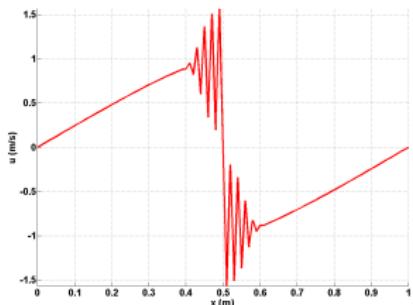
$$\mu_{max}(x, t) = \frac{h}{2} |u(x, t)|$$

$$\mu_e(x, t) = h^2 \frac{\max(R(x, t), J)}{\|\eta(u) - \bar{\eta}(t)\|_\infty}$$

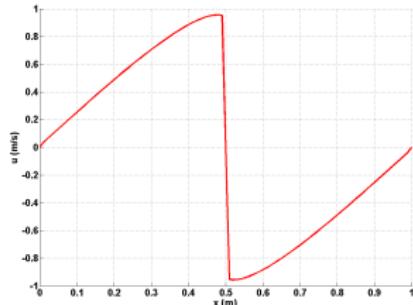
Entropy residual:  $R(x, t) = \partial_t \eta(u) + \partial_x \Phi(u) \leq 0$  with  $\eta(u) = u^2/2$

Jump:  $J = [[\partial_x \Phi(u)]]$

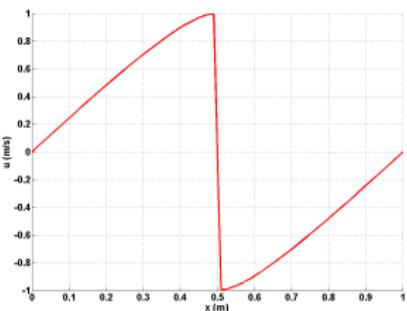
# 1-D numerical results (100 cells and CFL = 1)



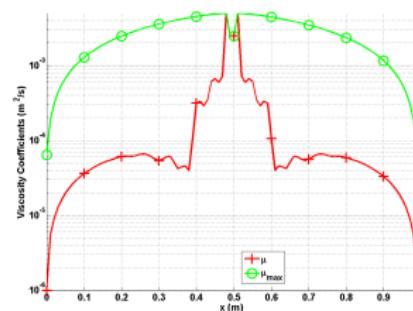
(a) Without stabilization.



(b) With first-order viscosity.



(c) With the EVM.



(d) Viscosity coefficient profiles.

## The MULti-D Euler Equations with variable area (MULe DEEr)



## Numerical methods for continuous and discontinuous schemes

- multi-wave problem, can develop shock waves and other types of discontinuities.
- Numerical methods for both continuous and discontinuous schemes: approximate Riemann solvers (HLL, HLLC, Roe scheme,  $\dots$ ), flux limiters , Lapidus viscosity, Pressure-based viscosity, SUPG, C-method, Entropy Viscosity Method (EVM).
- Numerical method can be ill-scaled in the low-Mach limit, yielding the wrong incompressible system  $\rightarrow$  use of a Mach-based preconditioner for the dissipative terms to obtain the correct behavior in the low Mach limit.
- Low-Mach steady-state solution: time-dependent term preconditioner to accelerate the convergence of the solution to the steady-state (Turkel) when using an explicit scheme  $\rightarrow$  the transient is no longer accurate. *Implicit solvers do not have this issue.*

- The isentropic compressible multi-D Euler equations degenerate into an incompressible system in the low-Mach asymptotic limit.

$$\partial_t \rho + \vec{u} \cdot \nabla \rho = 0$$

$$\partial_t \vec{u} + \vec{u} \cdot \nabla \cdot \vec{u} + \frac{1}{\rho} \nabla P = 0$$

$$\nabla \cdot \vec{u} = 0$$

$$P(\vec{r}, t) = P_0(t) + M^2 P_2(\vec{r}, t)$$

- no energy equation since the flow is assumed isentropic
- pressure fluctuations of the order of the Mach number square

## The multi-D Euler equations with variable area

Mass conservation

$$\partial_t(\rho A) + \nabla \cdot (\rho A \vec{u}) = 0$$

Momentum conservation

$$\partial_t(\rho \vec{u} A) + \nabla \cdot (\rho A \vec{u} \otimes \vec{u} + P \mathbb{I}) = P \nabla A$$

Energy conservation

$$\partial_t(\rho E A) + \nabla \cdot [\vec{u} (\rho E + P) A] = 0$$

Equation of state

$$P = eos(\rho, e)$$

Multi-wave problem:  $\lambda_1 = \vec{u} \cdot \vec{n} - c$ ,  $\lambda_2 = \vec{u} \cdot \vec{n} + c$  and  $\lambda_{2,\dots,2+D} = \vec{u} \cdot \vec{n}$ .

The area  $A$  is only a function of space.

*Objectives: extend the EVM to low-Mach flows while maintaining its capabilities of solving for transonic and supersonic flows, and use an implicit solver.*

## How to do it?

- ➊ recast the entropy equation as a function of the pressure, the density, the velocity and the speed of sound.
- ➋ derive a viscous regularization for the multi-D Euler equations (already done).
- ➌ work with the non-dimensionalized version of the multi-D Euler equations in order to understand how the different terms scale → will define non-dimensionalized numbers (Mach number, numerical Reynolds number, ...)
- ➍ derive a definition for the viscosity coefficients that ensures well-scaled dissipative terms for a wide range of Mach numbers → will consider two cases: isentropic and non-isentropic (with shocks) flows.

# Recast the entropy residual

## New entropy residual

$$D_e(\vec{r}, t) = \partial_t s + \vec{u} \cdot \nabla s = \underbrace{\frac{s_e}{P_e} \left( \frac{dP}{dt} - c^2 \frac{d\rho}{dt} \right)}_{\tilde{D}_e(\vec{r}, t)}$$

## The viscosity coefficients

- The viscosity coefficient will be set proportional to  $\tilde{D}_e(\vec{r}, t)$  (instead of  $D_e(\vec{r}, t)$ ):

$$\mu_e(\vec{r}, t) = h^2 \frac{\max(\tilde{D}_e(\vec{r}, t), J)}{\text{norm}_P^\mu} \text{ and } \kappa_e(\vec{r}, t) = h^2 \frac{\max(\tilde{D}_e(\vec{r}, t), J)}{\text{norm}_P^\kappa}$$

- $\tilde{D}_e(\vec{r}, t)$  is an alternative way of computing the local entropy production
- This new expression offers more diversity in the choice of the **normalization parameter**  $\text{norm}_P$ :  $P$ ,  $\rho c^2$ ,  $\rho c \|\vec{u}\|$  or  $\rho \|\vec{u}\|^2$

# A viscous regularization for the multi-D Euler equations with variable area

Mass conservation

$$\partial_t(\rho A) + \nabla \cdot (\rho A \vec{u}) = \nabla \cdot \vec{f}$$

Momentum conservation

$$\partial_t(\rho \vec{u} A) + \nabla \cdot (\rho A \vec{u} \otimes \vec{u} + P \mathbb{I}) = P \nabla A + \nabla \cdot \left( \mathbb{F}(\vec{u}) + \vec{u} \otimes \vec{f} \right)$$

Energy conservation

$$\partial_t(\rho E A) + \nabla \cdot [\vec{u} (\rho E + P) A] = \nabla \cdot \left( \vec{h} + \vec{u} \cdot \mathbb{F}(\vec{u}) + \frac{\|\vec{u}\|^2}{2} \vec{f} \right)$$

Dissipative terms

$$\vec{f} = A \kappa \nabla \rho, \quad \vec{h} = A \kappa \nabla (\rho e) \text{ and } \mathbb{F}(\vec{u}) = A \mu \nabla^s \vec{u} \text{ or } \mathbb{F}(\vec{u}) = A \mu \nabla \vec{u}$$

→ two positive viscosity coefficients  $\mu$  and  $\kappa$ . Requires a concave physical entropy  $s(\rho, e)$ .

# Non-dimensionalized multi-D Euler equation

We define some reference variables denoted by subscript  $\infty$ :

$$\rho^* = \frac{\rho}{\rho_\infty}, \quad u^* = \frac{u}{u_\infty}, \quad P^* = \frac{P}{\rho_\infty c_\infty^2}, \quad E^* = \frac{E}{c_\infty^2},$$

$$x^* = \frac{x}{L_\infty}, \quad t^* = \frac{t}{L_\infty/u_\infty}, \quad \mu^* = \frac{\mu}{\mu_\infty}, \quad \kappa^* = \frac{\kappa}{\kappa_\infty}$$

$\rightarrow \mu_\infty$  and  $\kappa_\infty$  are function of the normalization parameters  $norm_P^\mu$  and  $norm_P^\kappa$ , respectively. We also define the following reference numbers:

$$\text{Mach number: } M_\infty = \frac{u_\infty}{c_\infty},$$

$$\text{Numerical Reynolds number: } \text{Re}_\infty = \frac{u_\infty L_\infty}{\mu_\infty},$$

$$\text{Numerical P\'echlet number: } \text{P\'e}_\infty = \frac{u_\infty L_\infty}{\kappa_\infty},$$

$$\text{Numerical Prandlt number: } \text{Pr}_\infty = \text{P\'e}_\infty / \text{Re}_\infty$$

# Non-dimensionalized multi-D Euler equation

$$\partial_{t^*} \rho^* + \nabla^* \cdot (\rho^* \vec{u}^*) = \frac{1}{\text{Pé}_\infty} \nabla^* \cdot (\kappa^* \vec{\nabla}^* \rho^*)$$

$$\begin{aligned} \partial_{t^*} (\rho^* \vec{u}^*) + \nabla^* \cdot (\rho^* \vec{u}^* \otimes \vec{u}^*) + \frac{1}{M_\infty^2} \vec{\nabla}^* P^* &= \frac{1}{\text{Re}_\infty} \nabla^* \cdot \left( \rho^* \mu^* \vec{\nabla}^{s,*} \vec{u}^* \right) \\ &+ \frac{1}{\text{Pé}_\infty} \nabla^* \cdot \left( \vec{u}^* \otimes \kappa^* \vec{\nabla}^* \rho^* \right) \end{aligned}$$

$$\begin{aligned} \partial_{t^*} (\rho^* E^*) + \nabla^* \cdot [\vec{u}^* (\rho^* E^* + P^*)] &= \frac{1}{\text{Pé}_\infty} \nabla^* \cdot \left( \kappa^* \vec{\nabla}^* (\rho^* e^*) \right) \\ &+ \frac{M_\infty^2}{\text{Re}_\infty} \nabla^* \cdot \left( \vec{u}^* \rho^* \mu^* \vec{\nabla}^{s,*} \vec{u}^* \right) + \frac{M_\infty^2}{2\text{Pé}_\infty} \nabla^* \cdot \left( \kappa^* (u^*)^2 \vec{\nabla}^* \rho^* \right) \end{aligned}$$

The above equations are valid for both isentropic and non-isentropic flows and for all Mach numbers.

## For an isentropic flow

→ choose  $\text{Re}_\infty$  and  $\text{Pé}_\infty$  so that we recover the incompressible equations.

- assume Ideal gas equation of state for simplicity:  $P = (\gamma - 1)\rho e$
- choose  $\text{Re}_\infty = \text{Pé}_\infty = 1$
- expand each variable in power of the Mach number:  
$$P(\vec{r}, t) = P_0(\vec{r}, t) + P_1(\vec{r}, t)M_\infty + P_2(\vec{r}, t)M_\infty^2 + \dots$$
- derive the leading, first and second-order momentum equations
- derive the leading-order energy and mass equations

## For an isentropic flow: momentum equation

$$\partial_t(\rho \vec{u}) + \nabla \cdot (\rho \vec{u} \otimes \vec{u}) + \frac{1}{M_\infty^2} \nabla P = \frac{1}{\text{Re}_\infty} \nabla \cdot (\rho \mu \vec{\nabla}^s \vec{u}) + \frac{1}{\text{Pé}_\infty} \nabla \cdot (\vec{u} \otimes \kappa \nabla \rho)$$

Leading and first-order momentum equation:

$$\nabla P_0 = \nabla P_1 = 0 \longrightarrow P(\vec{r}, t) = \tilde{P}_0(t) + P_2(\vec{r}, t) M_\infty^2$$

Second-order momentum equation:

$$\partial_t(\rho \vec{u})_0 + \nabla \cdot (\rho \vec{u} \otimes \vec{u})_0 + P_2 = \nabla \cdot (\rho \mu \vec{\nabla}^s \vec{u})_0 + \nabla \cdot (\vec{u} \otimes \kappa \nabla \rho)_0$$

## For an isentropic flow: energy and mass equations

$$\begin{aligned}\partial_{t^*} (\rho^* E^*) + \nabla^* \cdot [\vec{u}^* (\rho^* E^* + P^*)] &= \frac{1}{\text{Pé}_\infty} \nabla^* \cdot \left( \kappa^* \vec{\nabla}^* (\rho^* e^*) \right) \\ &+ \frac{M_\infty^2}{\text{Re}_\infty} \nabla^* \cdot \left( \vec{u}^* \rho^* \mu^* \vec{\nabla}^{s,*} \vec{u}^* \right) + \frac{M_\infty^2}{2\text{Pé}_\infty} \nabla^* \cdot \left( \kappa^* (u^*)^2 \vec{\nabla}^* \rho^* \right)\end{aligned}$$

Leading-order energy equation:

$$\partial_t (\rho E)_0 + \nabla \cdot [\vec{u} (\rho E + P)]_0 = \nabla \cdot (\kappa \nabla (\rho e))_0 \longrightarrow \nabla \cdot \vec{u}_0 = 0$$

$$\partial_t \rho + \nabla \cdot (\rho \vec{u}) = \frac{1}{\text{Pé}_\infty} \nabla \cdot (\kappa \nabla \rho)$$

Leading-order mass equation:

$$\partial_t \rho_0 + \nabla \cdot (\rho \vec{u})_0 = \nabla \cdot (\kappa \nabla \rho)_0 \rightarrow \partial_t \rho_0 + \vec{u}_0 \nabla \rho_0 = \nabla \cdot (\kappa \nabla \rho)_0$$

## For an isentropic flow: derivation of norm $_{P}^{\mu,\kappa}$

We were able to recover the incompressible equations by choosing  $\text{Re}_{\infty} = \text{P\'{e}}_{\infty} = 1$ . What does that imply for  $\text{norm}_{P}^{\mu}$  and  $\text{norm}_{P}^{\kappa}$ ?

Derive an expression for  $\kappa_{\infty}$ :

$$\kappa_e(\vec{r}, t) = h^2 \frac{\max(\tilde{D}_e(\vec{r}, t), J)}{\text{norm}_{P}^{\kappa}} \longrightarrow \kappa_{\infty} = \frac{\rho_{\infty} c_{\infty}^2 u_{\infty} L}{\text{norm}_{P,\infty}^{\kappa}}$$

Then, use the substitute the above expression for  $\text{norm}_{P}^{\kappa}$  into  $\text{P\'{e}}_{\infty}$  to obtain:

$$\text{norm}_{P,\infty}^{\kappa} = \text{P\'{e}}_{\infty} \rho_{\infty} c_{\infty}^2 = \rho_{\infty} c_{\infty}^2$$

Same derivation using  $\mu_e$  and  $\text{Re}_{\infty}$  leads to:

$$\text{norm}_{P,\infty}^{\mu} = \text{Re}_{\infty} \rho_{\infty} c_{\infty}^2 = \rho_{\infty} c_{\infty}^2$$

## For a non-isentropic flow i.e. with shocks

- the flow can experience shocks and other waves → discontinuities
- the low-Mach asymptotic study is no longer valid
- directly work with the non-dimensionalized Euler equations
- determine the scaling of  $\text{Re}_\infty$  and  $\text{Pé}_\infty$  to stabilize the equations
- look at the low-Mach limit

## For a non-isentropic flow: momentum equation

$$\partial_t (\rho \vec{u}) + \nabla \cdot (\rho^* \vec{u} \otimes \vec{u}) + \frac{1}{M_\infty^2} \nabla P = \frac{1}{\text{Re}_\infty} \nabla \cdot (\rho \mu \vec{\nabla}^s \vec{u}) + \frac{1}{\text{Pé}_\infty} \nabla \cdot (\vec{u} \otimes \kappa \nabla \rho)$$

In the shock region, the term  $\frac{1}{M_\infty^2} \nabla P$  will become dominant and will need to be stabilized by a dissipative term of the same scaling:

- (a)  $\text{Re}_\infty = M_\infty^2$  and  $\text{Pé}_\infty = 1$
- (b)  $\text{Re}_\infty = 1$  and  $\text{Pé}_\infty = M_\infty^2$
- (c)  $\text{Re}_\infty = \text{Pé}_\infty = M_\infty^2$

→ each of the above option will affect the other equations (mass and energy equations).

## For a non-isentropic flow: mass equation

$$\partial_t \rho + \nabla \cdot (\rho \vec{u}) = \frac{1}{\text{Pé}_\infty} \nabla \cdot (\kappa \nabla \rho)$$

choice (a)  $\text{Pé}_\infty = 1$

$$\partial_t \rho + \nabla \cdot (\rho \vec{u}) = \nabla \cdot (\kappa \nabla \rho)$$

→ the dissipative term is *well-scaled*

choice (b) and (c)  $\text{Pé}_\infty = M_\infty^2$

$$\partial_t \rho + \nabla \cdot (\rho \vec{u}) = \frac{1}{M_\infty^2} \nabla \cdot (\kappa \nabla \rho)$$

→ the dissipative term is *ill-scaled*

Options (b) and (c) are not inappropriate. Thus, we are left with option (a):  $\text{Re}_\infty = M_\infty^2$  and  $\text{Pé}_\infty = 1$

## For a non-isentropic flow: energy equation

$$\begin{aligned}\partial_{t^*} (\rho^* E^*) + \nabla^* \cdot [\vec{u}^* (\rho^* E^* + P^*)] &= \frac{1}{\text{Pé}_\infty} \nabla^* \cdot \left( \kappa^* \vec{\nabla}^* (\rho^* e^*) \right) \\ &+ \frac{M_\infty^2}{\text{Re}_\infty} \nabla^* \cdot \left( \vec{u}^* \rho^* \mu^* \vec{\nabla}^{s,*} \vec{u}^* \right) + \frac{M_\infty^2}{2\text{Pé}_\infty} \nabla^* \cdot \left( \kappa^* (u^*)^2 \vec{\nabla}^* \rho^* \right)\end{aligned}$$

With option (a) it yields:

$$\begin{aligned}\partial_{t^*} (\rho^* E^*) + \nabla^* \cdot [\vec{u}^* (\rho^* E^* + P^*)] &= \nabla^* \cdot \left( \kappa^* \vec{\nabla}^* (\rho^* e^*) \right) \\ &+ \nabla^* \cdot \left( \vec{u}^* \rho^* \mu^* \vec{\nabla}^{s,*} \vec{u}^* \right) + \frac{M_\infty^2}{2} \nabla^* \cdot \left( \kappa^* (u^*)^2 \vec{\nabla}^* \rho^* \right)\end{aligned}$$

→ all dissipative terms are *well-scaled*.

## For a non-isentropic flow: derivation of norm $_{P}^{\mu,\kappa}$

We derive the scaling of norm $_{P}^{\mu}$  and norm $_{P}^{\kappa}$  when  $\text{Re}_{\infty} = M_{\infty}^2$  and  $\text{Pé}_{\infty} = 1$ :

Derive an expression for  $\kappa_{\infty}$ :

$$\kappa_e(\vec{r}, t) = h^2 \frac{\max(\tilde{D}_e(\vec{r}, t), J)}{\text{norm}_{P}^{\kappa}} \rightarrow \kappa_{\infty} = \frac{\rho_{\infty} c_{\infty}^2 u_{\infty} L}{\text{norm}_{P,\infty}^{\kappa}}$$

Then, use the substitute the above expression for norm $_{P}^{\kappa}$  into  $\text{Pé}_{\infty}$  to obtain:

$$\text{norm}_{P,\infty}^{\kappa} = \text{Pé}_{\infty} \rho_{\infty} c_{\infty}^2 = \rho_{\infty} c_{\infty}^2 \rightarrow \text{same as for isentropic flow}$$

Same derivation using  $\mu_e$  and  $\text{Re}_{\infty}$  leads to:

$$\text{norm}_{P,\infty}^{\mu} = \text{Re}_{\infty} \rho_{\infty} c_{\infty}^2 = \rho_{\infty} u_{\infty}^2 \rightarrow \text{different from isentropic flow}$$

## How to merge the two cases?

$$\mu(\vec{r}, t) = \min \left( \mu_{\max}(\vec{r}, t), \mu_e(\vec{r}, t) \right) \text{ and } \kappa(\vec{r}, t) = \min \left( \mu_{\max}(\vec{r}, t), \kappa_e(\vec{r}, t) \right)$$

where the first-order viscosity is given by

$$\kappa_{\max}(\vec{r}, t) = \mu_{\max}(\vec{r}, t) = \frac{h}{2} \left( \|u\| + c \right)$$

and the entropy viscosity coefficients by

$$\kappa_e(\vec{r}, t) = \frac{h^2 \max(\tilde{R}, J)}{\rho c^2} \text{ and } \mu_e(\vec{r}, t) = \frac{h^2 \max(\tilde{R}, J)}{\text{norm}_P^\mu}$$

where

$$\text{norm}_P^\mu = \text{Re}_\infty \rho_\infty c_\infty^2 = \begin{cases} \rho \|u\|^2 & \text{if } |\tilde{R}^*| \geq M \text{ (i.e., non-isentropic flow)} \\ \rho c^2 = \text{norm}_P^\kappa & \text{otherwise} \end{cases}$$

with the jumps given by

$$J = \| \vec{u} \| \max \left( [[\nabla P \cdot \vec{n}]], c^2 [[\nabla \rho \cdot \vec{n}]] \right)$$

## Numerical results:

Table: Subsonic flow inlet boundary conditions.

boundary type	$U_1^{bc}$	$U_2^{bc}$	$U_3^{bc}$	$U_4^\ell$
static pressure	$P$	$T$	$\theta$	$u$
mass flow rate	$\rho  u  $	$h$	$\theta$	$u$
stagnation pressure	$P_0$	$T_0$	$\theta$	$u$

# 1-D converging-diverging nozzle

## Stiffened gas equation of state

$$P = (\gamma - 1)\rho(e - q) - \gamma P_\infty$$

## Equation of state parameters

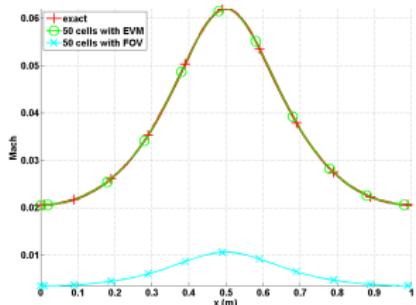
fluid	$\gamma$	$C_v \text{ (J.kg}^{-1}.\text{K}^{-1}\text{)}$	$P_\infty \text{ (Pa)}$	$q \text{ (J.kg}^{-1}\text{)}$
liquid water	2.35	1816	$10^9$	$-1167 \text{ } 10^3$
steam	1.43	1040	0	$2030 \text{ } 10^3$

## Cross-section $A$

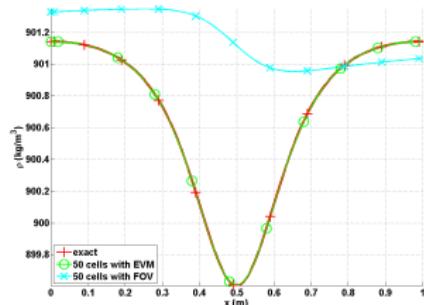
$$A(x) = 1 + 0.5 \cos(2\pi x)$$

- a steady state is reached
- low-Mach flow for liquid water
- supersonic flow for steam

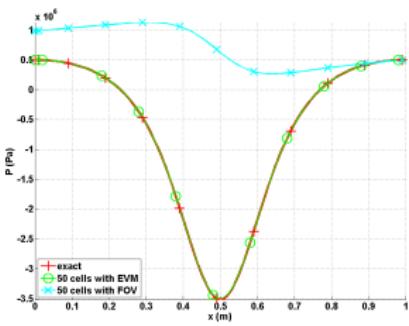
# 1-D converging-diverging nozzle: liquid water



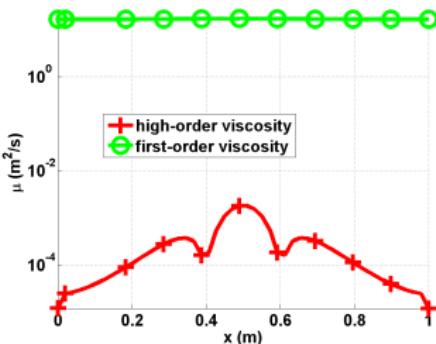
(e) Mach number



(f) Density



(g) Pressure



(h) Viscosity coefficients

# 1-D converging-diverging nozzle: liquid water

Convergence rates for the  $L_1$  norm of the error:

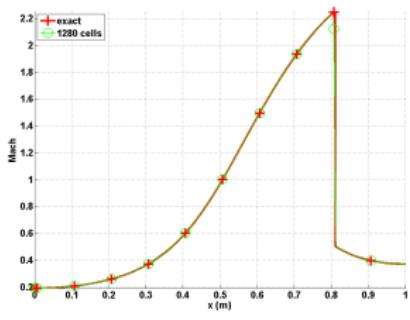
cells	density	rate	pressure	rate	velocity	rate
4	$2.8037 \cdot 10^{-1}$	—	$8.4705 \cdot 10^5$	—	7.2737	—
8	$1.3343 \cdot 10^{-1}$	0.495	$4.7893 \cdot 10^5$	0.24	6.1493	0.0747
16	$2.9373 \cdot 10^{-2}$	2.10	$1.0613 \cdot 10^5$	2.09	1.2275	2.25
32	$5.1120 \cdot 10^{-3}$	2.58	$1.8446 \cdot 10^4$	2.58	$1.8943 \cdot 10^{-1}$	2.78
64	$1.0558 \cdot 10^{-3}$	2.31	$3.7938 \cdot 10^3$	2.31	$3.7919 \cdot 10^{-2}$	2.37
128	$2.3712 \cdot 10^{-4}$	2.18	$8.4471 \cdot 10^2$	2.19	$8.5517 \cdot 10^{-3}$	2.17
256	$5.6058 \cdot 10^{-5}$	2.08	$1.9839 \cdot 10^2$	2.09	$2.0475 \cdot 10^{-3}$	2.07
512	$1.3278 \cdot 10^{-5}$	2.07	$4.6622 \cdot 10^1$	2.08	$4.9516 \cdot 10^{-4}$	2.06
1024	$3.1193 \cdot 10^{-6}$	—	$1.1755 \cdot 10^1$	—	$1.2379 \cdot 10^{-4}$	—

# 1-D converging-diverging nozzle: liquid water

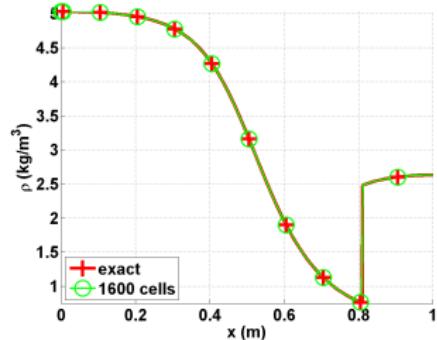
Convergence rates for the  $L_2$  norm of the error:

cells	density	rate	pressure	rate	velocity	rate
4	$3.106397 \cdot 10^{-1}$	—	$5.254445 \cdot 10^5$	—	3.288543	—
8	$7.491623 \cdot 10^{-2}$	2.06	$1.636966 \cdot 10^5$	1.62	1.823880	0.14
16	$2.079858 \cdot 10^{-2}$	1.81	$4.627338 \cdot 10^4$	1.77	$4.990605 \cdot 10^{-1}$	1.83
32	$5.329627 \cdot 10^{-3}$	1.96	$1.180287 \cdot 10^4$	1.96	$1.261018 \cdot 10^{-1}$	1.98
64	$1.341583 \cdot 10^{-3}$	1.99	$2.967104 \cdot 10^3$	1.99	$3.160914 \cdot 10^{-2}$	1.99
128	$3.359766 \cdot 10^{-4}$	1.99	$7.428087 \cdot 10^2$	1.99	$7.907499 \cdot 10^{-3}$	1.99
256	$8.403859 \cdot 10^{-5}$	1.99	$1.857861 \cdot 10^2$	2.01	$1.977292 \cdot 10^{-3}$	2.00
512	$2.10075 \cdot 10^{-5}$	—	$4.7024 \cdot 10^1$	—	$4.9516 \cdot 10^{-4}$	—

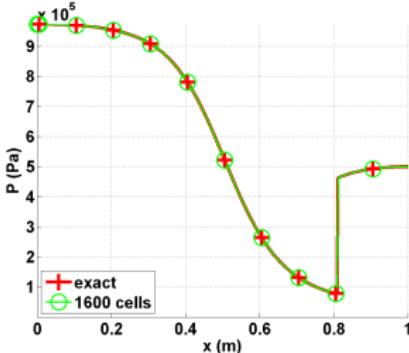
# 1-D converging-diverging nozzle: vapor



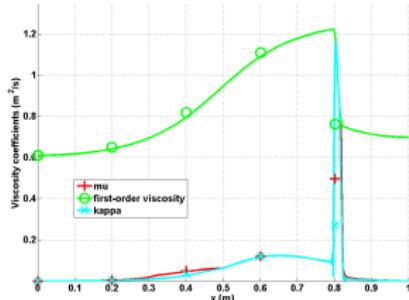
(i) Mach number



(j) Density



(k) Pressure



(l) Viscosity coefficients

# 1-D converging-diverging nozzle: vapor

Convergence rates for the  $L_1$  norm of the error:

cells	density	rate	pressure	rate	velocity	rate
5	$0.72562 \cdot 10^{-1}$	—	$1.5657 \cdot 10^5$	—	173.69	—
10	$0.4165 \cdot 10^{-1}$	0.80088	$9.6741 \cdot 10^4$	0.63425	120.69	0.52519
20	$0.20675 \cdot 10^{-1}$	1.0104	$4.9193 \cdot 10^4$	0.96971	72.149	0.74228
40	$0.093703 \cdot 10^{-1}$	1.1417	$2.0103 \cdot 10^4$	0.72728	34.716	1.0554
80	$0.047328 \cdot 10^{-1}$	0.9854	$1.0208 \cdot 10^4$	0.9777	16.082	1.1101
160	$0.023965 \cdot 10^{-2}$	0.9817	$5.1969 \cdot 10^3$	0.9739	7.9573	1.0150
320	$0.020768 \cdot 10^{-2}$	0.9886	$2.5116 \cdot 10^3$	1.0490	3.7812	1.0734
640	$0.0059715 \cdot 10^{-2}$	1.0160	$1.2754 \cdot 10^3$	0.9776	1.8353	1.0428

# 1-D converging-diverging nozzle: vapor

Convergence rates for the  $L_2$  norm of the error:

cells	density	rate	pressure	rate	velocity	rate
5	$9.7144 \cdot 10^{-1}$	—	$2.0215 \cdot 10^5$	—	236.94	—
10	$5.9718 \cdot 10^{-1}$	0.70195	$1.3024 \cdot 10^5$	0.63425	166.56	0.50854
20	$2.9503 \cdot 10^{-1}$	1.0173	$6.6503 \cdot 10^4$	0.96971	103.36	0.68831
40	$1.8193 \cdot 10^{-1}$	0.69747	$4.0171 \cdot 10^4$	0.72728	66.374	0.6390
80	$1.3366 \cdot 10^{-1}$	0.44485	$2.3163 \cdot 10^4$	0.43576	42.981	0.62692
160	$9.6638 \cdot 10^{-2}$	0.46790	$1.7263 \cdot 10^4$	0.42413	31.717	0.43844
320	$7.0896 \cdot 10^{-2}$	0.44688	$1.2763 \cdot 10^4$	0.43571	23.138	0.45499
640	$5.2191 \cdot 10^{-2}$	0.44190	$9.4217 \cdot 10^3$	0.43790	16.910	0.45238

## 2-D low-Mach flow over a cylinder

### Typical benchmark problem for low-Mach flow:

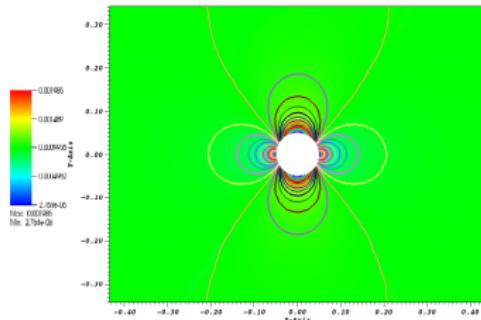
- The steady-state solution is symmetric: the iso-Mach contour lines are used to assess the symmetry of the numerical solution
- The velocity at the top of the cylinder is twice the incoming velocity set at the inlet
- The pressure fluctuations are proportional to the square of inlet Mach number, i.e.,

$$\delta P = \frac{\max(P(\vec{r}, t)) - \min(P(\vec{r}, t))}{\max(P(\vec{r}, t))} \propto M_\infty^2$$

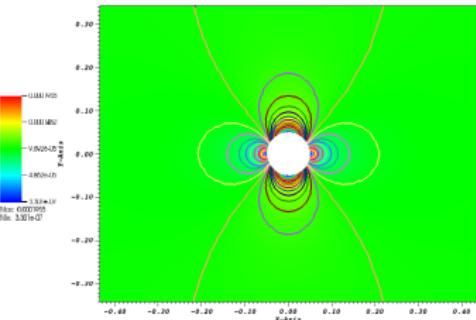
where  $\delta P$  and  $M_\infty$  denote the pressure fluctuations and the inlet Mach number, respectively.

- triangular mesh with 4008 triangular elements
- Ideal Gas equation of state with  $\gamma = 1.4$
- CFL = 20

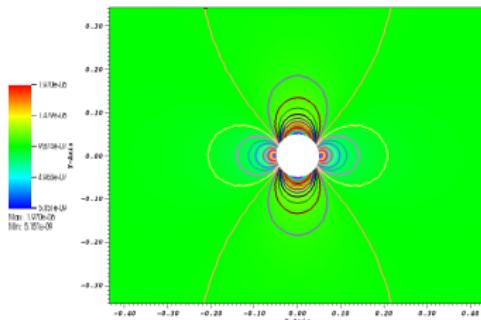
## 2-D low-Mach flow over a cylinder



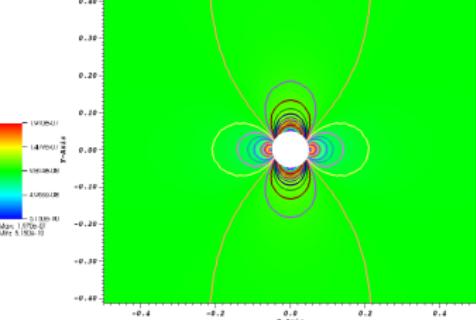
(m)  $M_{\text{inlet}} = 10^{-3}$



(n)  $M_{\text{inlet}} = 10^{-4}$

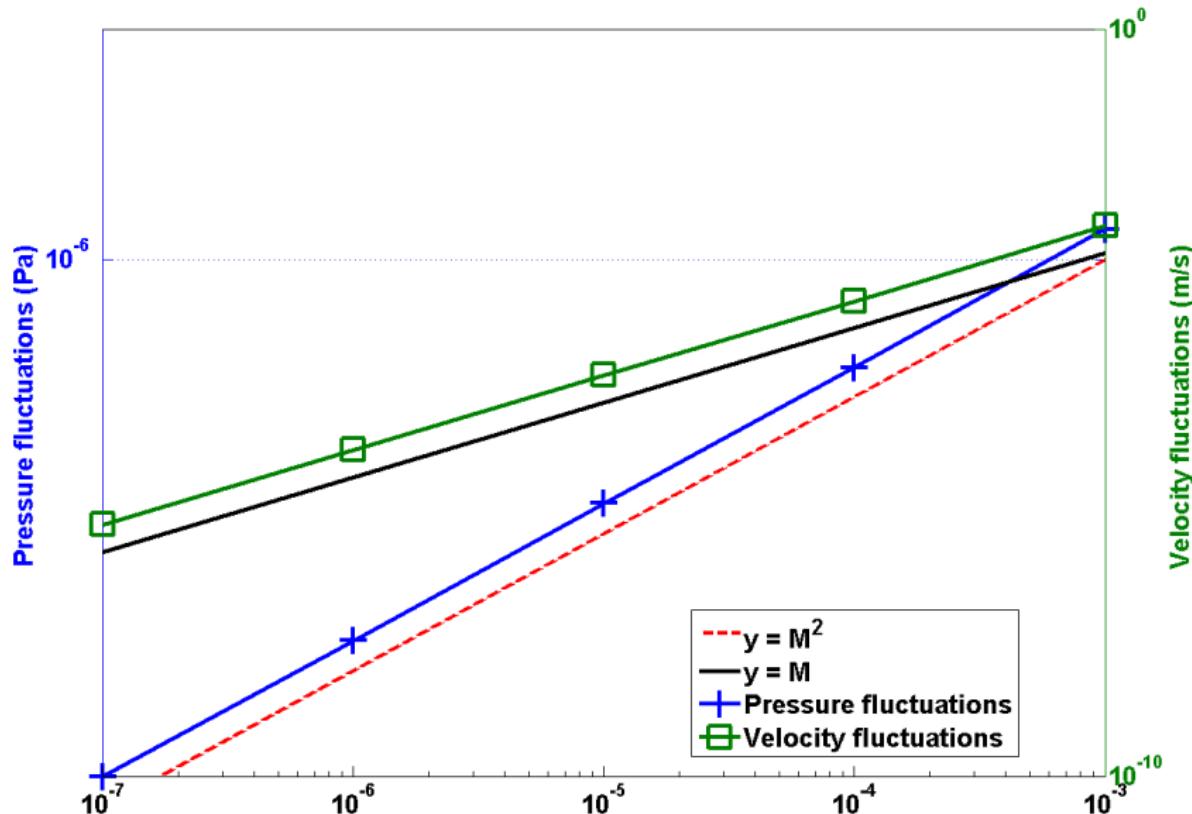


(o)  $M_{\text{inlet}} = 10^{-6}$



(p)  $M_{\text{inlet}} = 10^{-7}$

## 2-D low-Mach flow over a cylinder



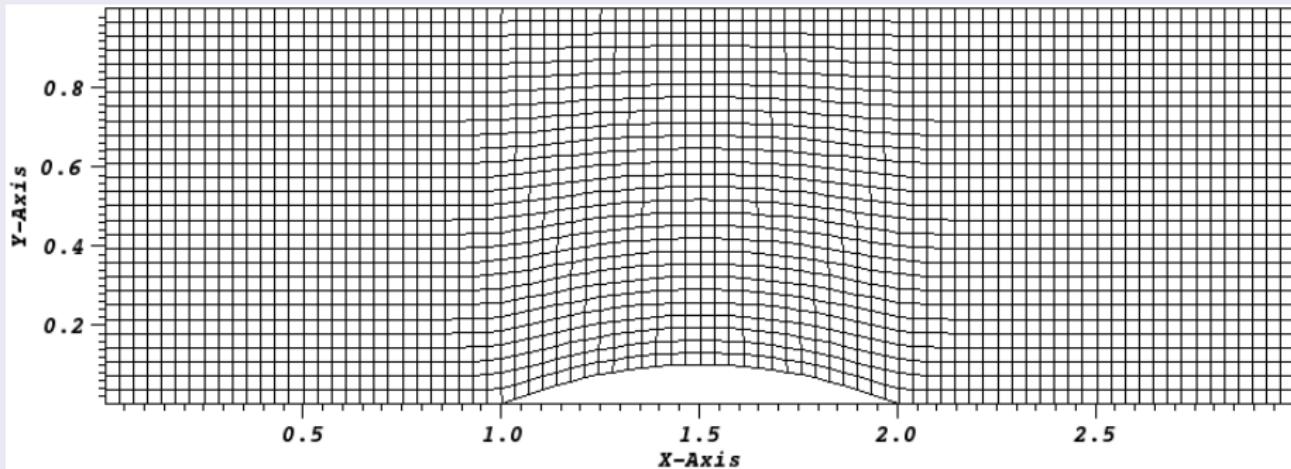
# 2-D low-Mach flow over a cylinder

Velocity ratio for different Mach numbers

Mach number	inlet velocity	velocity at the top of the cylinder	ratio
$10^{-3}$	$2.348 \cdot 10^{-3}$	$1.176 \cdot 10^{-3}$	1.99
$10^{-4}$	$2.285 \cdot 10^{-4}$	$1.145 \cdot 10^{-4}$	1.99
$10^{-5}$	$2.283 \cdot 10^{-5}$	$1.144 \cdot 10^{-5}$	1.99
$10^{-6}$	$2.283 \cdot 10^{-6}$	$1.144 \cdot 10^{-6}$	1.99
$10^{-7}$	$2.283 \cdot 10^{-7}$	$1.144 \cdot 10^{-7}$	1.99

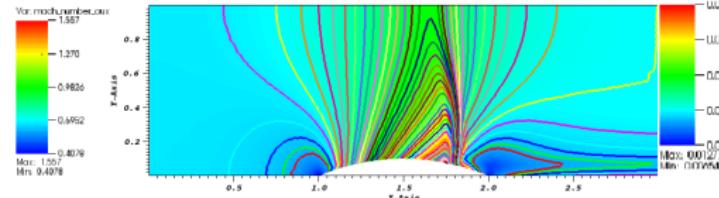
## 2-D low-Mach flow over a bump

Geometry: an uniform grid of 3352  $Q_1$  elements

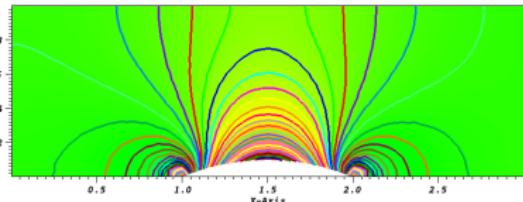


- Ideal Gas equation of state with  $\gamma = 1.4$
- CFL = 20
- Inlet flow for different Mach numbers, static pressure and wall-boundary conditions.

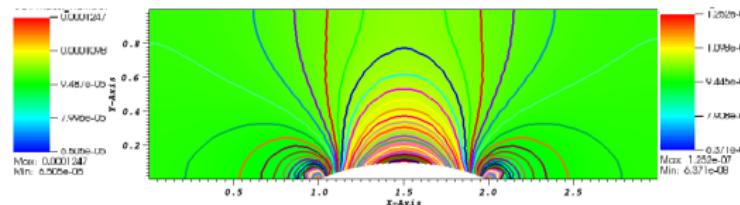
# 2-D low-Mach flow over a bump



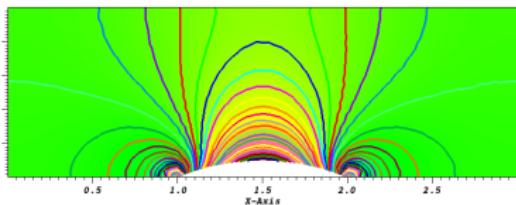
(q)  $M_{\text{inlet}} = 0.7$



(r)  $M_{\text{inlet}} = 10^{-2}$



(s)  $M_{\text{inlet}} = 10^{-4}$



(t)  $M_{\text{inlet}} = 10^{-7}$

# 2-D Mach 3 flow over a forward facing step

# The SEVEN-EquAtion Model with variable area (SEVEN-bandEd ArMadillo)



# The seven-equation model (SEM)

- Each phase obeys the single-phase Euler equations: two continuity equations, two momentum equations and two energy equations.
- Seventh equation: void fraction equation → an internal boundary condition between the two phases at the interface.
- Exchange terms between phases: relaxation terms. These terms were derived using *rational thermodynamic* → consistent with the entropy minimum principle.
- The system of equations is well-posed, and has seven waves.
- The seven-equation model degenerates to single-phase Euler equations when one phase disappears.

# The multi-D seven-equation model (with variable area)

We consider two phases  $j, k$ . Phase  $k$  obeys the following system of equations:

$$\left\{ \begin{array}{lcl} \partial_t (\alpha_k A) & + & \vec{u}_{int} A \nabla \alpha_k = A \mu_{rel} (P_k - P_j) \\ \partial_t (\alpha_k \rho_k A) & + & \nabla \cdot (\alpha_k \rho_k \vec{u}_k A) = 0 \\ \partial_t (\alpha_k \rho_k \vec{u}_k A) & + & \nabla \cdot [\alpha_k A (\rho_k \vec{u}_k \otimes \vec{u}_k)] + \nabla (\alpha_k A P_k) = \\ & & \alpha_k P_k \nabla A + P_{int} A \nabla \alpha_k + A \lambda_{rel} (\vec{u}_j - \vec{u}_k) \\ \partial_t (\alpha_k \rho_k E_k A) & + & \nabla \cdot [\alpha_k A \vec{u}_j (\rho_k E_k + P_k)] = \\ & & P_{int} \vec{u}_{int} A \nabla \alpha_k - \mu_{rel} \bar{P}_{int} (P_k - P_j) + \bar{\vec{u}}_k A \lambda_{rel} (\vec{u}_j - \vec{u}_k) \end{array} \right.$$

$$\left\{ \begin{array}{l} P_{int} = \bar{P}_{int} - \frac{\nabla \alpha_k}{|\nabla \alpha_k|} \frac{Z_k Z_j}{Z_k + Z_j} \cdot (\vec{u}_k - \vec{u}_j) \\ \bar{P}_{int} = \frac{Z_k P_j + Z_j P_k}{Z_k + Z_j} \\ \vec{u}_{int} = \vec{u}_{int} - \frac{\nabla \alpha_k}{|\nabla \alpha_k|} \frac{P_k - P_j}{Z_k + Z_j} \\ \bar{\vec{u}}_{int} = \frac{Z_k \vec{u}_k + Z_j \vec{u}_j}{Z_k + Z_j} \end{array} \right.$$

and 
$$\left\{ \begin{array}{l} \mu_{rel} = \frac{A_{int}}{Z_k + Z_j} \\ \lambda_{rel} = \frac{\mu_{rel}}{2} Z_k Z_j \\ A_{int} = 6.25 \cdot A_{int, max} \cdot \alpha_k (1 - \alpha_k)^2 \end{array} \right.$$

## Viscous regularization for the multi-D SEM:

$$\begin{aligned}\partial_t (\alpha_k A) + \vec{u}_{int} A \nabla \alpha_k &= A \mu_{rel} (P_k - P_j) + \nabla \cdot \vec{I} \\ \partial_t (\alpha_k \rho_k A) + \nabla \cdot (\alpha_k \rho_k \vec{u}_k A) &= \nabla \cdot \vec{f} \\ \partial_t (\alpha_k \rho_k \vec{u}_k A) + \nabla \cdot [\alpha_k A (\rho_k \vec{u}_k \otimes \vec{u}_k + P_k \mathbb{I})] &= \\ &\quad \alpha_k P_k \nabla A + P_{int} A \nabla \alpha_k + A \lambda_{rel} (\vec{u}_j - \vec{u}_k) + \nabla \cdot \vec{g} \\ \partial_t (\alpha_k \rho_k E_k A) + \nabla \cdot [\alpha_k A \vec{u}_j (\rho_k E_k + P_k)] &= \\ P_{int} \vec{u}_{int} A \nabla \alpha_k - \mu_{rel} \bar{P}_{int} (P_k - P_j) + \bar{\vec{u}}_k A \lambda_{rel} (\vec{u}_j - \vec{u}_k) + \nabla \cdot \vec{h} &\end{aligned}$$

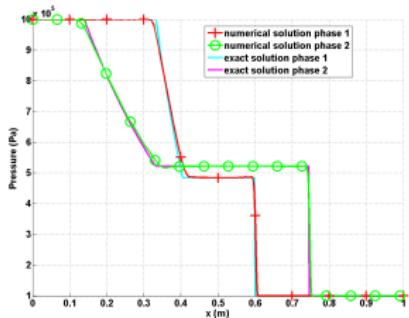
$$\left\{ \begin{array}{l} \vec{I} = A \beta_k \nabla \alpha_k \\ \vec{f} = \alpha_k A \kappa_k \nabla \rho_k + \rho_k \vec{I} \\ \vec{g} = \alpha_k A \rho_k \mu_k \nabla \vec{u} + \vec{u} \otimes \vec{f} \\ \vec{h} = \alpha_k A \kappa_k \nabla (\rho_k e_k) - \frac{\|\vec{u}\|^2}{2} \vec{f} + \vec{u} \cdot \vec{g} + \rho_k e_k \vec{I} \end{array} \right.$$

if  $\alpha_k \rightarrow 1$ , the multi-D Euler equations are retrieved

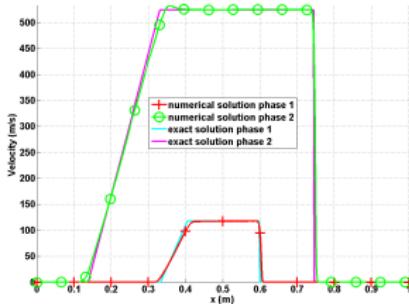
# Numerical results

# 1-D shock tube with two independent fluids

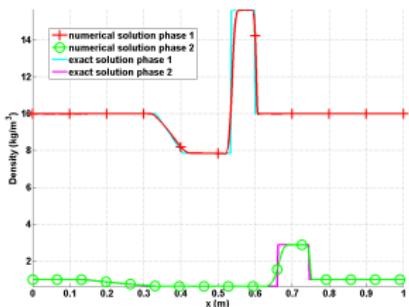
# 1-D shock tube with two independent fluids



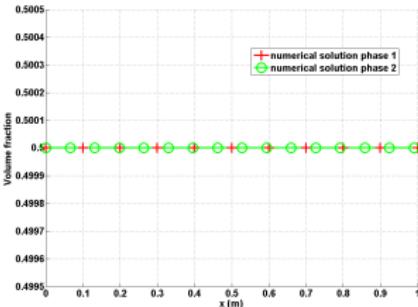
(u) Pressure at  $t = 305 \mu s$



(v) Velocity at  $t = 305 \mu s$

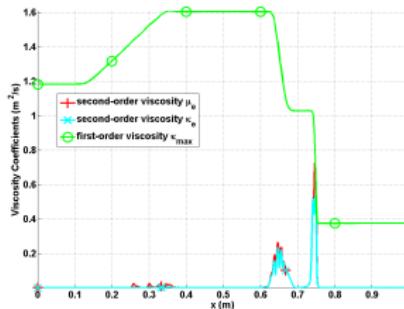


(w) Pressure at  $t = 305 \mu s$



(x) Velocity at  $t = 305 \mu s$

# 1-D shock tube with two independent fluids



(y) Viscosity coefficients phase 1 (z) Viscosity coefficients phase 2

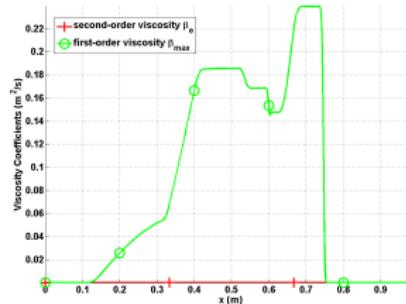
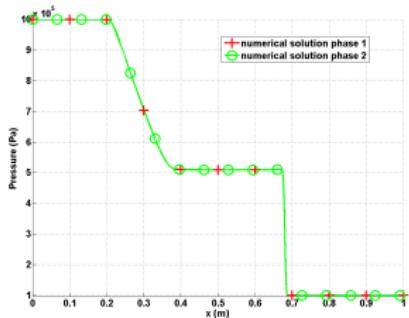


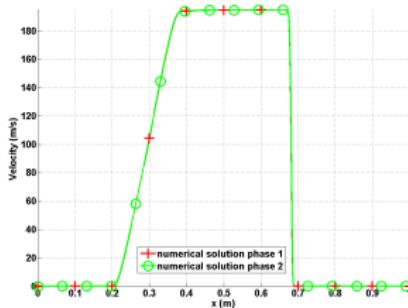
Figure: Viscosity coefficients volume fraction

# 1-D shock tube with infinite relaxation coefficients

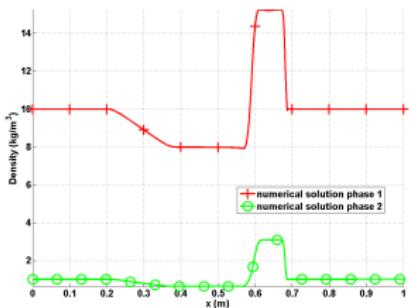
# 1-D shock tube with infinite relaxation coefficients



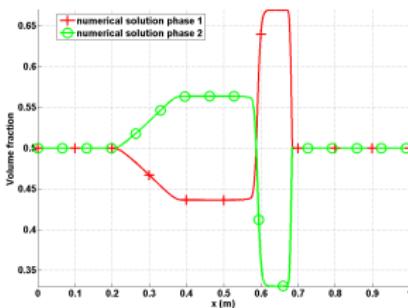
(a) Pressure at  $t = 305 \mu s$



(b) Velocity at  $t = 305 \mu s$



(c) Pressure at  $t = 305 \mu s$



(d) Velocity at  $t = 305 \mu s$

# 1-D shock tube with infinite relaxation coefficients

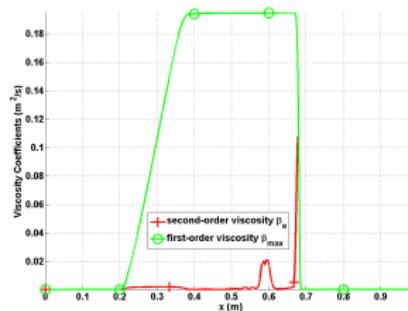
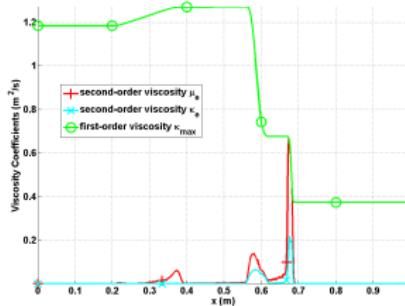
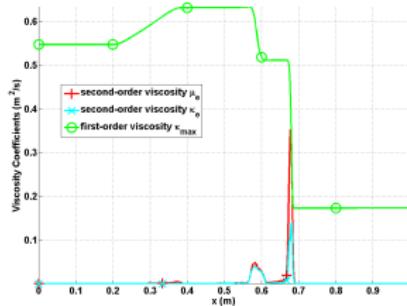


Figure: Viscosity coefficients volume fraction

# Conclusion

# The 1-D Radiation-Hydrodynamic Equation (RHEA)



# QUESTIONS/COMMENTS ?