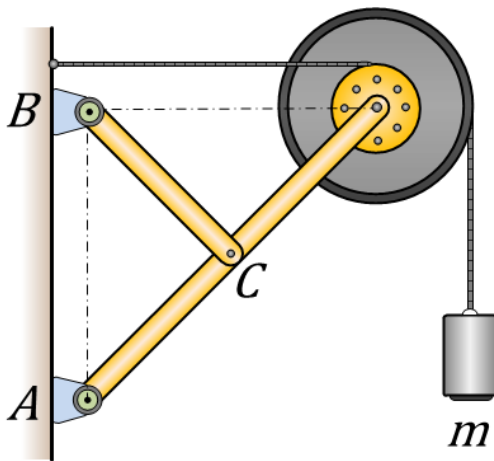


## Week 4, LI-2: Frames, Machines Friction and Springs



### FRAMES

- Rigid vs collapsible structures
- Force analysis and FBDs

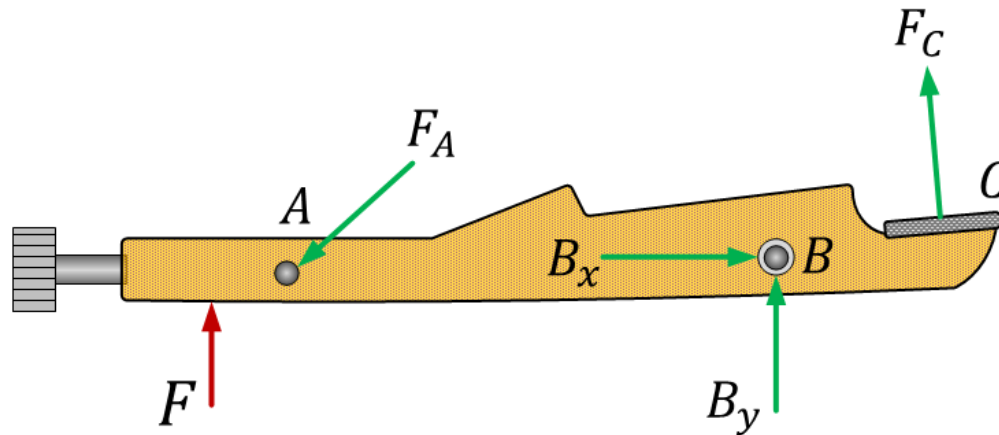
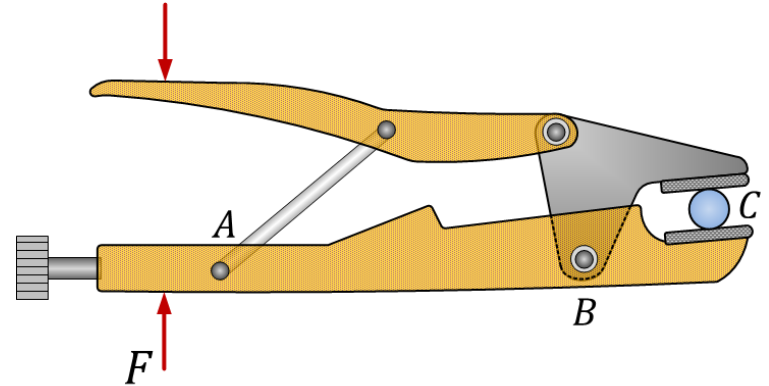
### MACHINES AND PULLEY SYSTEMS

- Definitions: mechanical components
- Pulley systems

### FRICTION AND SPRINGS

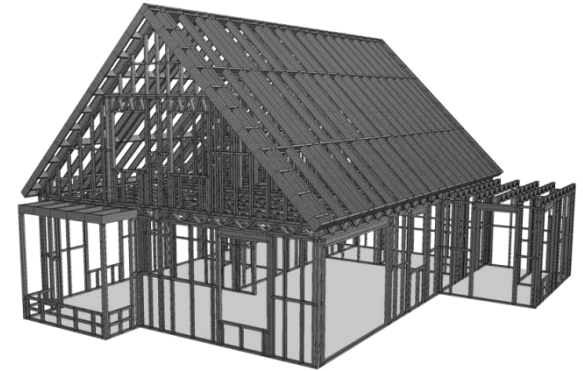
# Frames and Machines are similar to trusses

- The difference is that frames and machines contain at least one multi-force member
- A multi-force member is any member with more than two forces

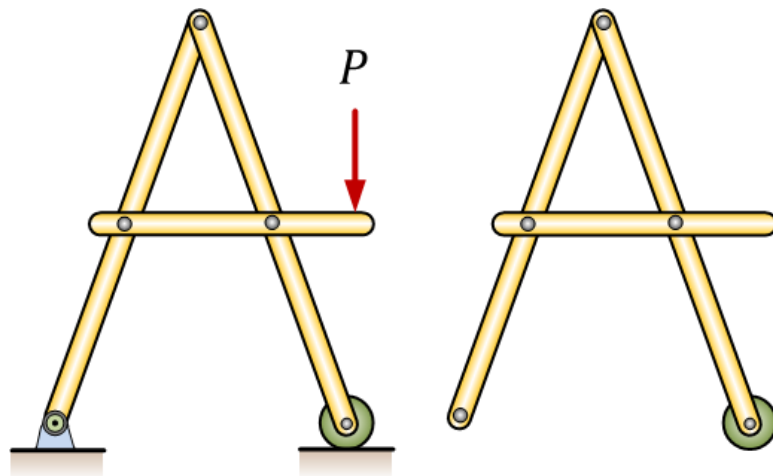


# Frames vs. Machines

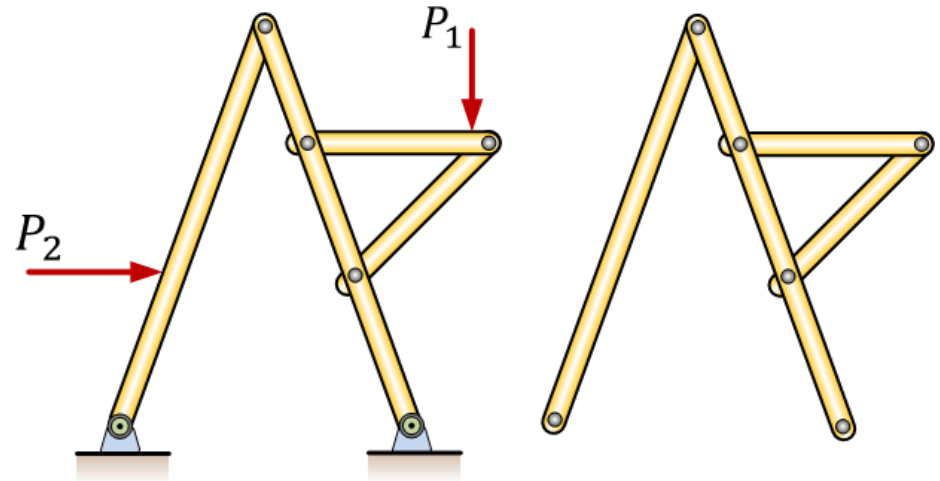
- Frames are structures which are designed to support loads and are usually fixed in position
- Machines are structures which contain moving parts and are designed to transmit forces and/or moments



# Rigid vs. Collapsible structures



(a) Rigid noncollapsible

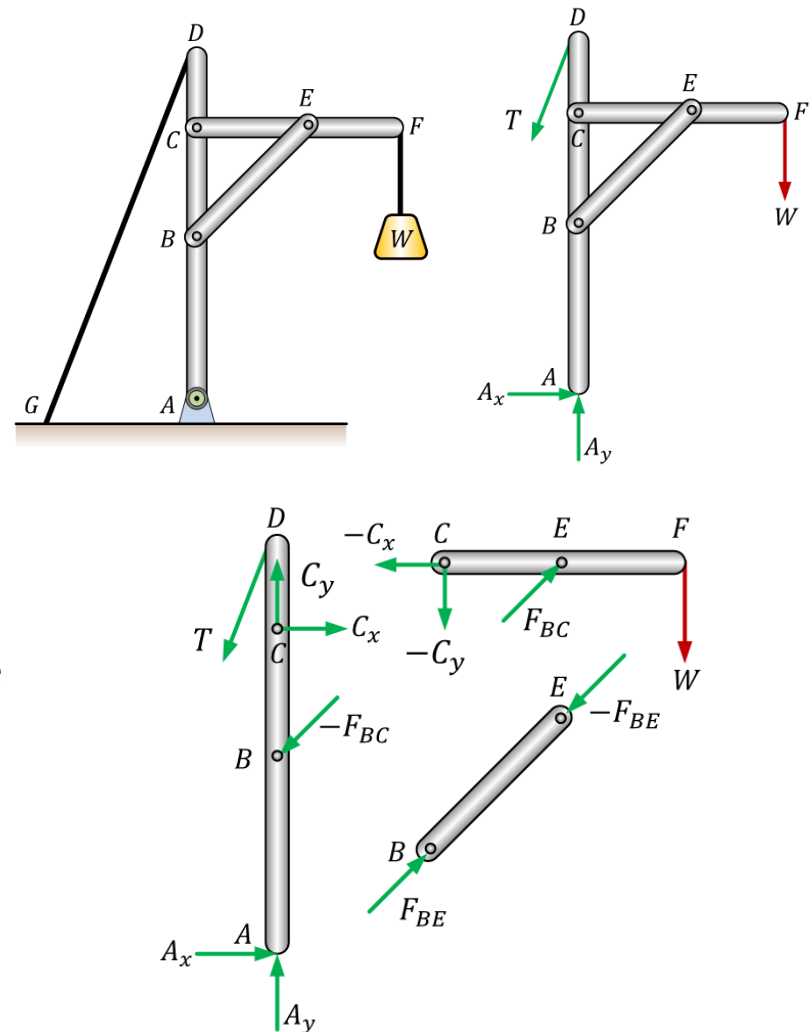


(b) Nonrigid collapsible

- If we disconnect a rigid frame from the rest of the world, its members will remain in the same configuration
- The members of a collapsible frame can move relative to one another if it is disconnected from the rest of the world

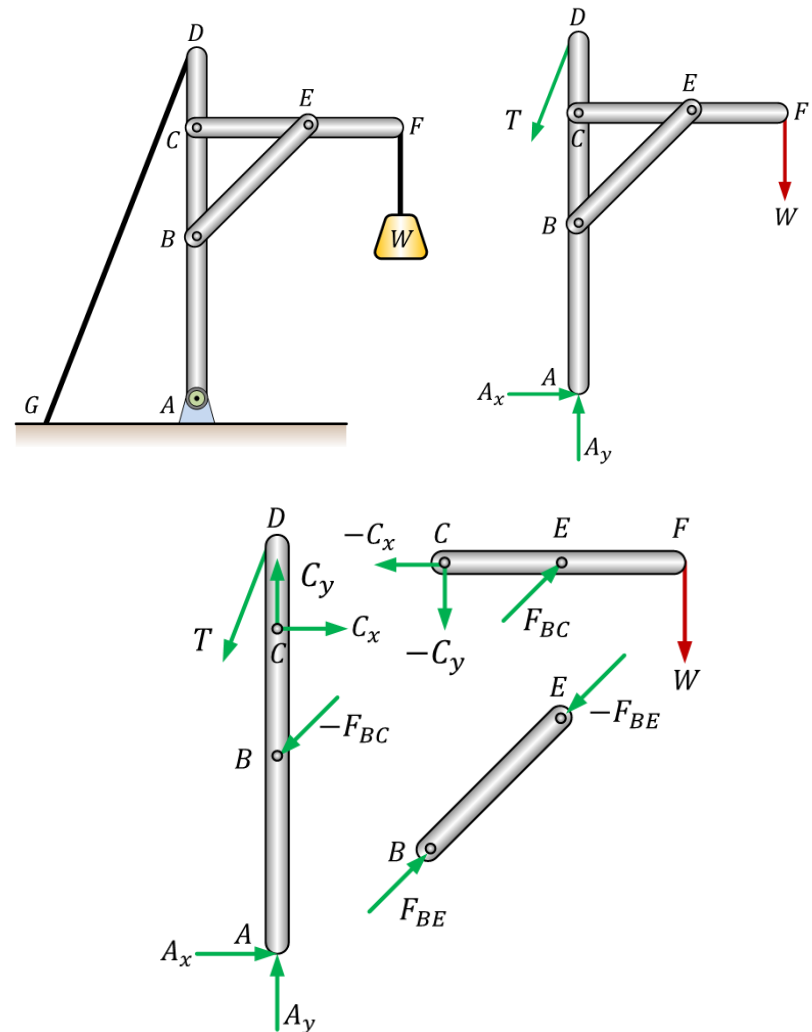
# Force Analysis in Frames and Machines

- Frames and machines are structures with at least one multi-force member. Frames are designed to support loads and are usually stationary. Machines contain moving parts and are designed to transmit and modify forces.
- A free body diagram of the complete frame is used to determine the external forces acting on the frame (similar to the truss analysis)
- Internal forces are determined by dismembering the frame and creating free-body diagram for each component



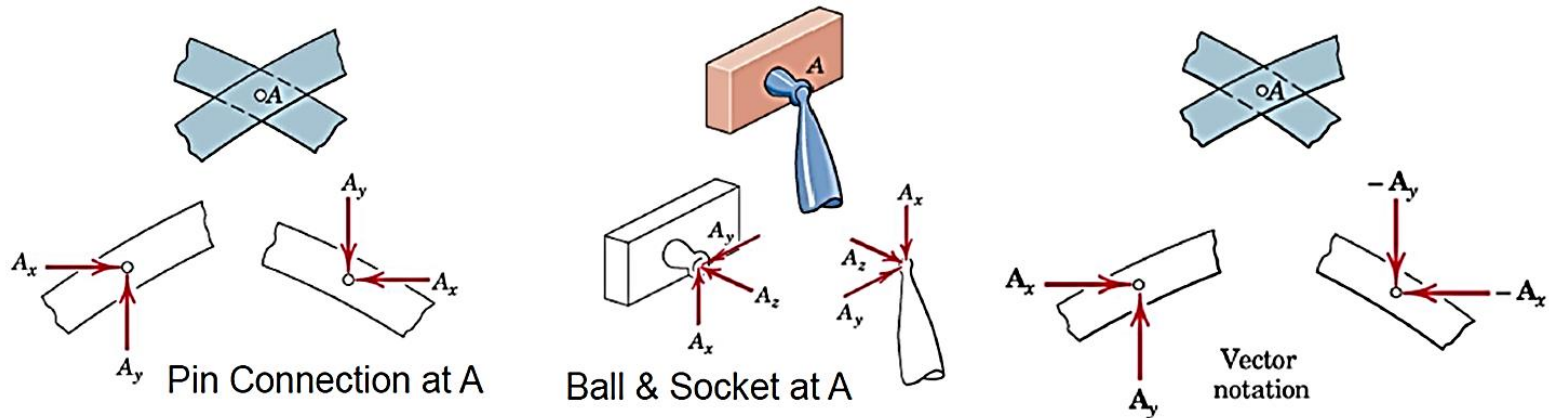
# Force Analysis in Frames and Machines

- Forces on two-force members have known lines of action but unknown magnitude and sense
- Forces on multi-force members have unknown magnitude and line of action. They must be represented with two unknown components
- Forces between connected components are equal, have the same line of action, and opposite sense.



# Free-Body Diagrams: Forces of Interactions

- Force components must be consistently represented in opposite directions on the separate FBDs (e.g. Pin a  $A$ )
- Apply action-and-reaction principle (e.g. Ball and socket at  $A$ )
- Vector notation: use plus sign for an action and a minus sign for the corresponding reaction



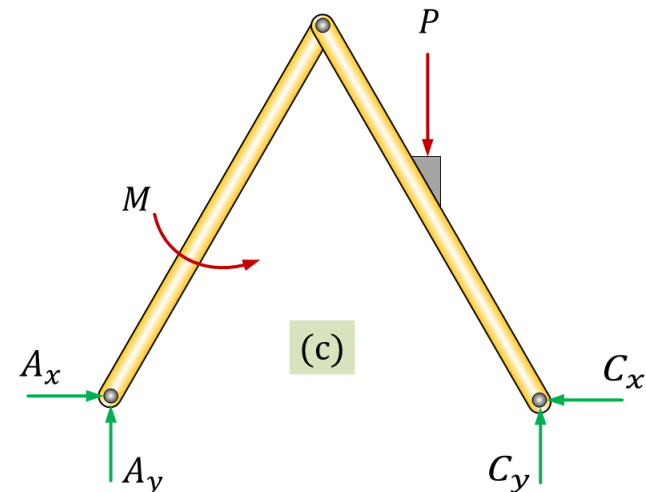
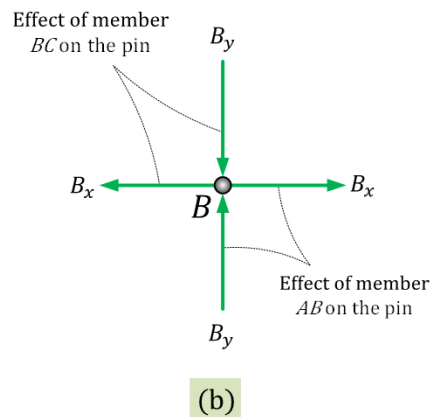
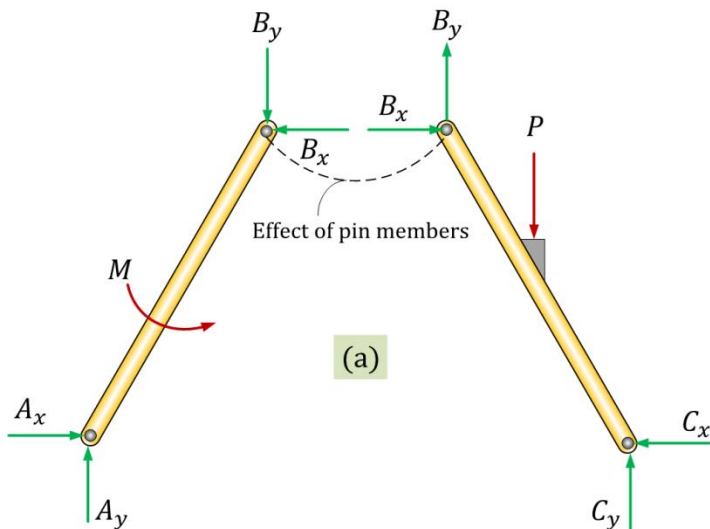
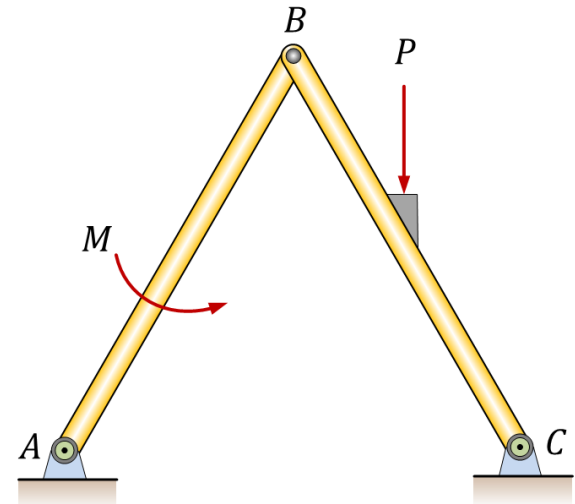
# Example: Free-Body Diagram

Draw FBD of

(a) Each member

(b) pin at B, and

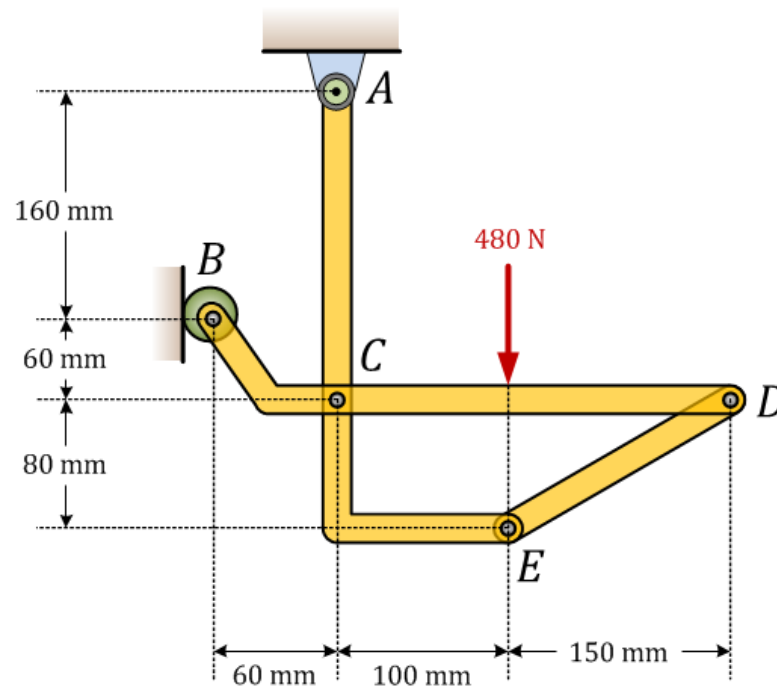
(c) whole system





# Example 1

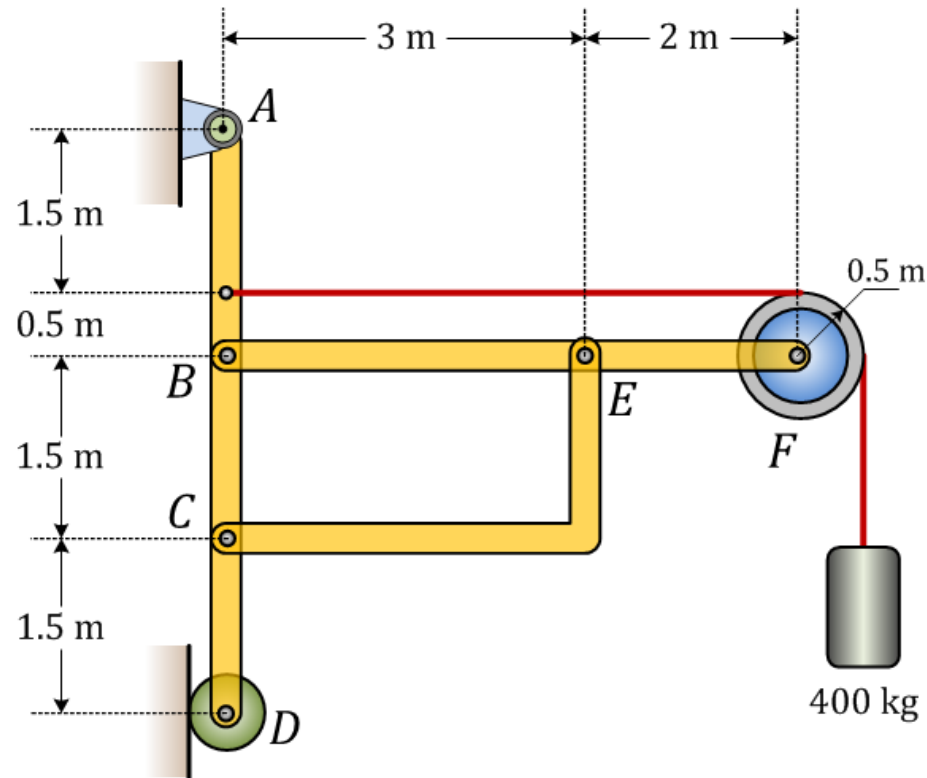
Member  $ACE$  and  $BCD$  are connected by a pin at  $C$  and by the link  $DE$ . For the loading shown, determine the force in link  $DE$  and the components of the force exerted at  $C$  on member  $BCD$ .



W4 Example 1 (Web view)

# Example 2

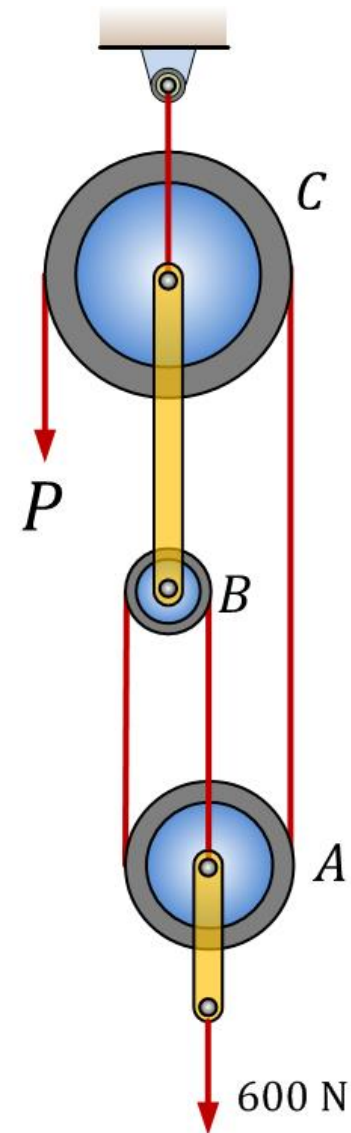
Compute the horizontal and vertical components of all forces acting on each of the members (neglect self weight)



W4 Example 2 (Web view)

## Example 3

Find the tension in the rigid link connecting pulleys  $B$  and  $C$  and the force  $P$  required to support the 600 N force using frictionless pulley system (neglect self weight)



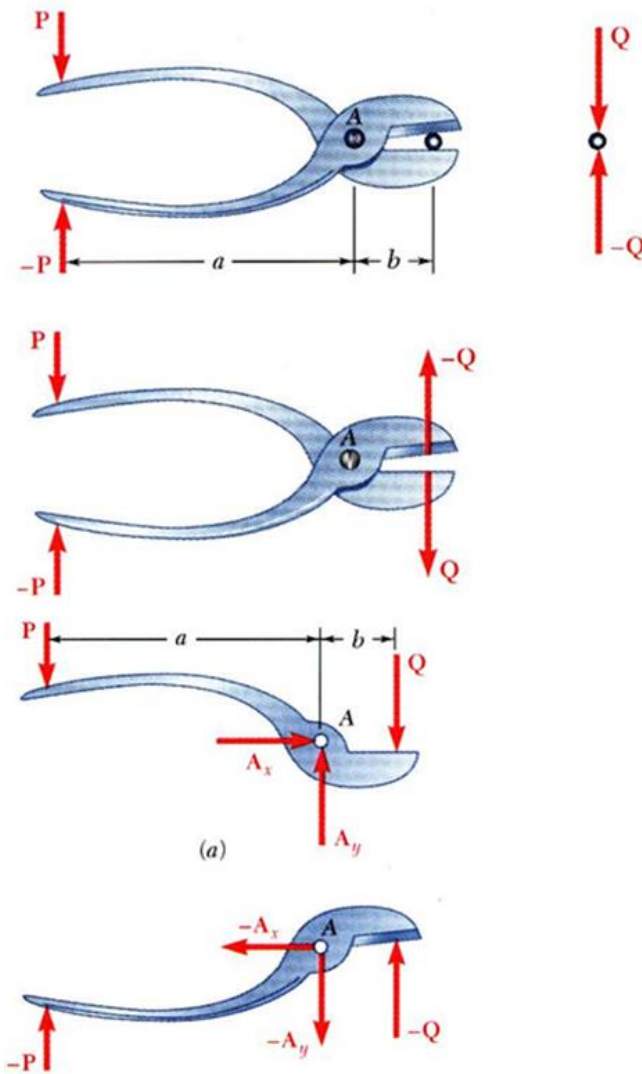
W4 Example 3 (Web view)

# Machines

- Machines are structures designed to transmit and modify forces. Their main purpose is to transform input forces into output forces
- Given the magnitude of  $P$ , determine the magnitude of  $Q$
- Create a free-body diagram of the complete machine, including the reaction that the wire exerts
- The machine is a nonrigid structure. Use one of the components as a free-body
- Taking moments about  $A$

$$\sum M_A = 0 = aP - bQ$$

$$Q = \frac{a}{b} P$$



# Machines

## Definitions

- **Effort:** Force required to overcome the resistance to get the work done by the machine
- **Mechanical Advantage:** Ratio of load lifted ( $W$ ) to the effort applied ( $P$ )

$$\text{Mechanical Advantage} = \frac{W}{P}$$

- **Velocity Ratio:** Ratio of the distance moved by the effort ( $D$ ) to the distance moved by the load ( $d$ ) in the same interval of time

$$\text{Velocity Ratio} = \frac{D}{d}$$

# Machines

## Definitions

- **Input:** Work done by the effort

$$\text{Input} = PD$$

- **Output:** Useful work got out of the machine i.e. the work done by the load

$$\text{Output} = WD$$

- **Efficiency:** Ratio of output to the input

Efficiency of an ideal machine is 1, in that case:

$$Wd = PD \quad \longrightarrow \quad \frac{W}{P} = \frac{D}{d}$$

For an ideal machine, mechanical advantage is equal to velocity ratio

# Machines

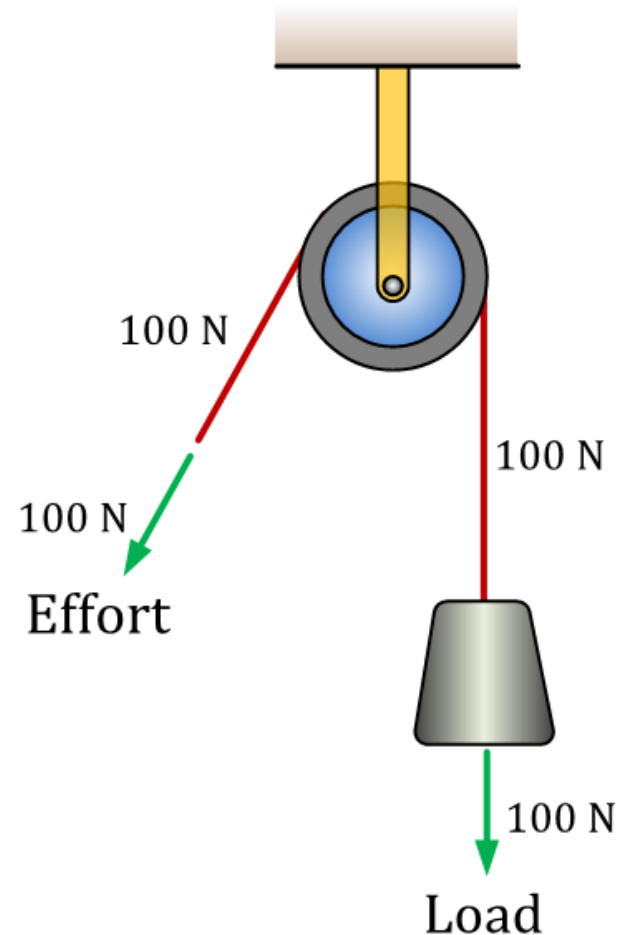
## Fixed Pulley

$$\text{Effort} = \text{Load}$$

→ Mechanical Advantage = 1

Distance moved by effort is equal to the distance moved by the load

→ Velocity Ratio = 1



# Machines

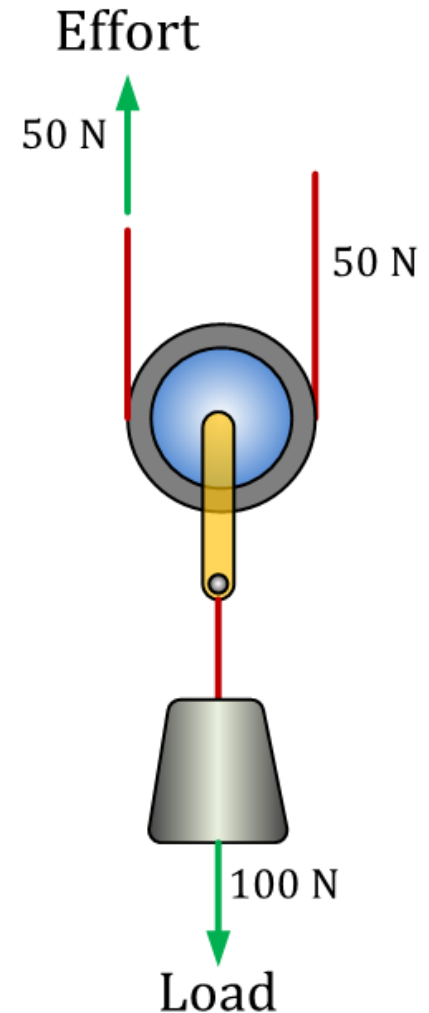
## Movable Pulley

$$\text{Effort} = \text{Load}/2$$

→ Mechanical Advantage = 2

Distance moved by the effort is twice the distance moved by the load (both rope should also accommodate the same displacement by which the load is moved)

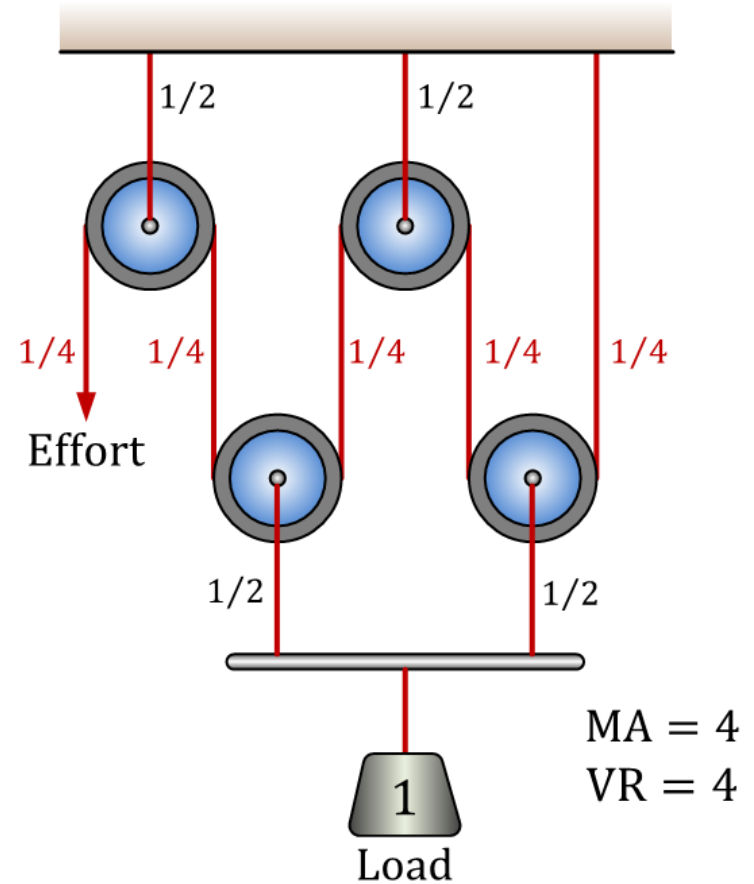
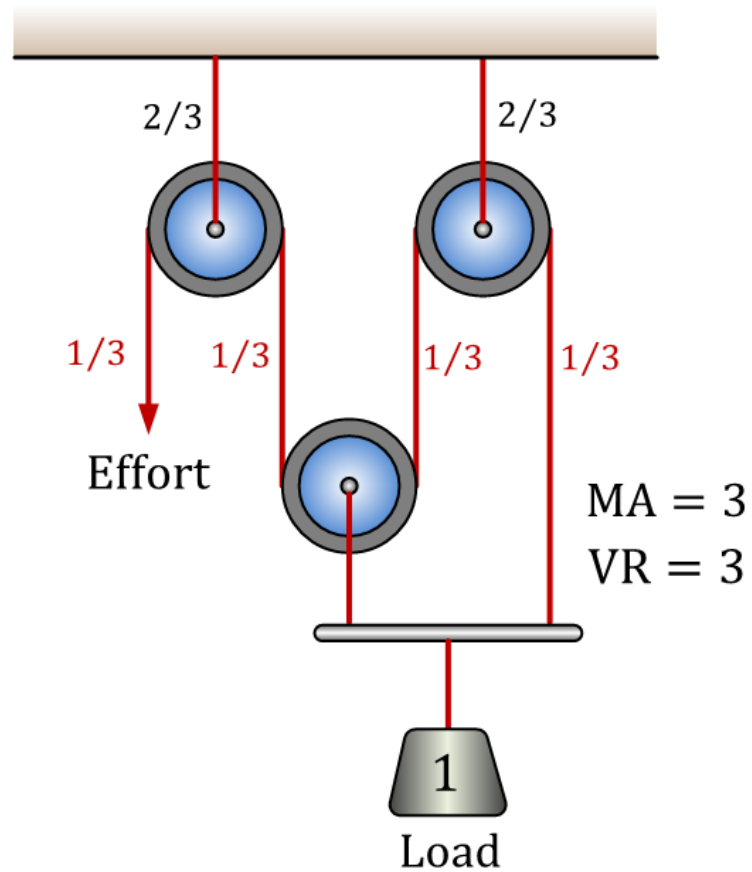
→ Velocity Ratio = 2





# Machines

## Compound Pulley



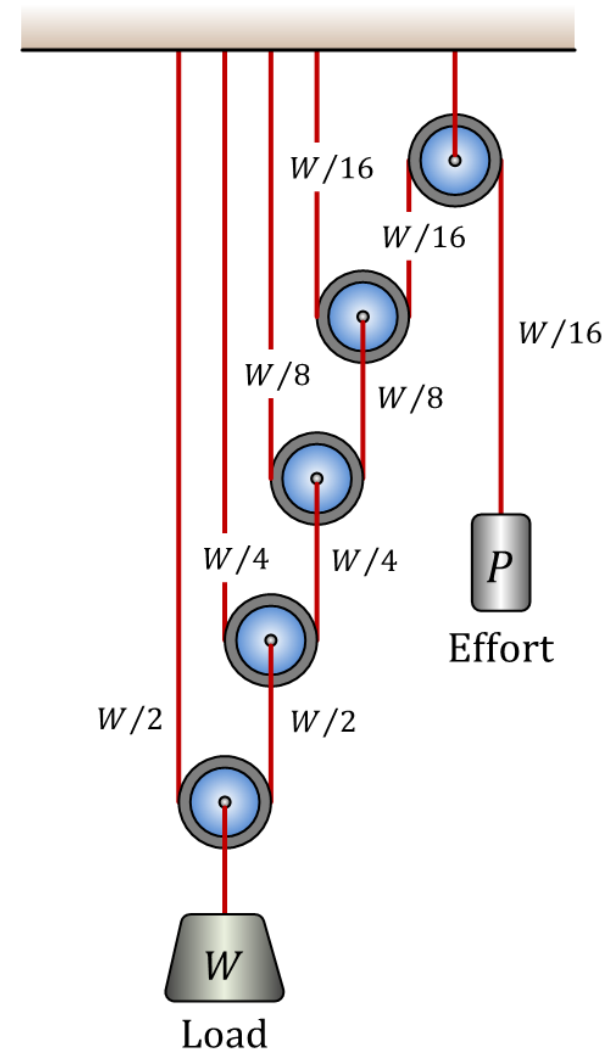
# Machines

## Compound Pulley

Effort required is  $1/16^{\text{th}}$  of the load

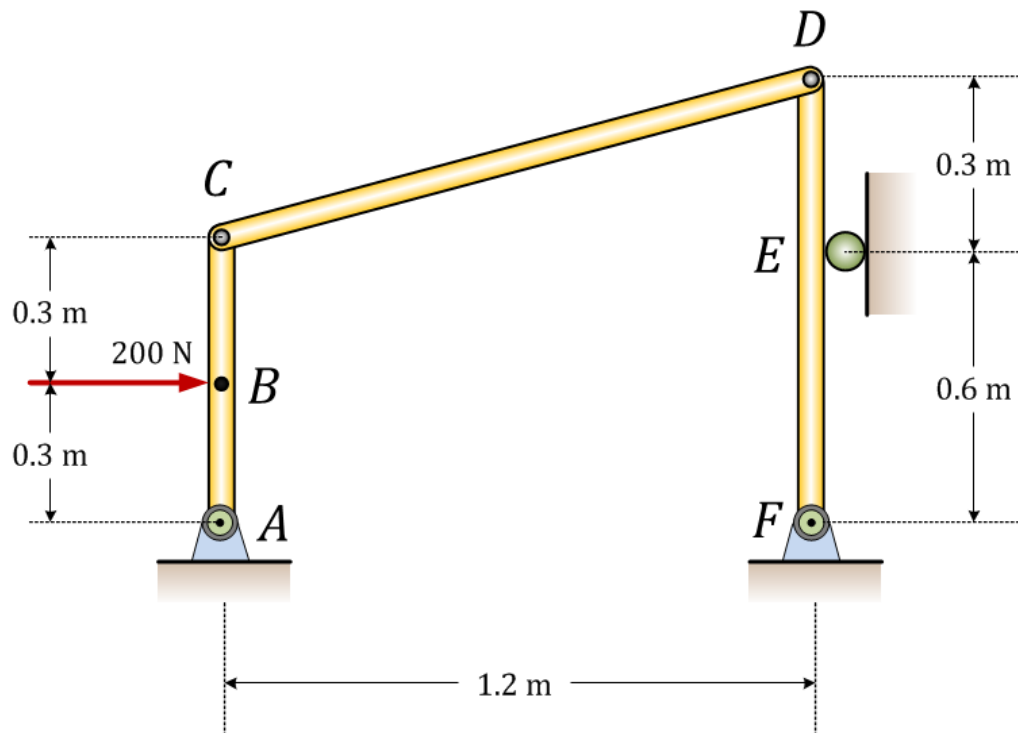
Mechanical Advantage = 16  
(Neglecting frictional forces)

Velocity ratio is 16, which means in order to raise a load to 1 unit height, effort has to be moved by a distance of 16 units



# Example 4

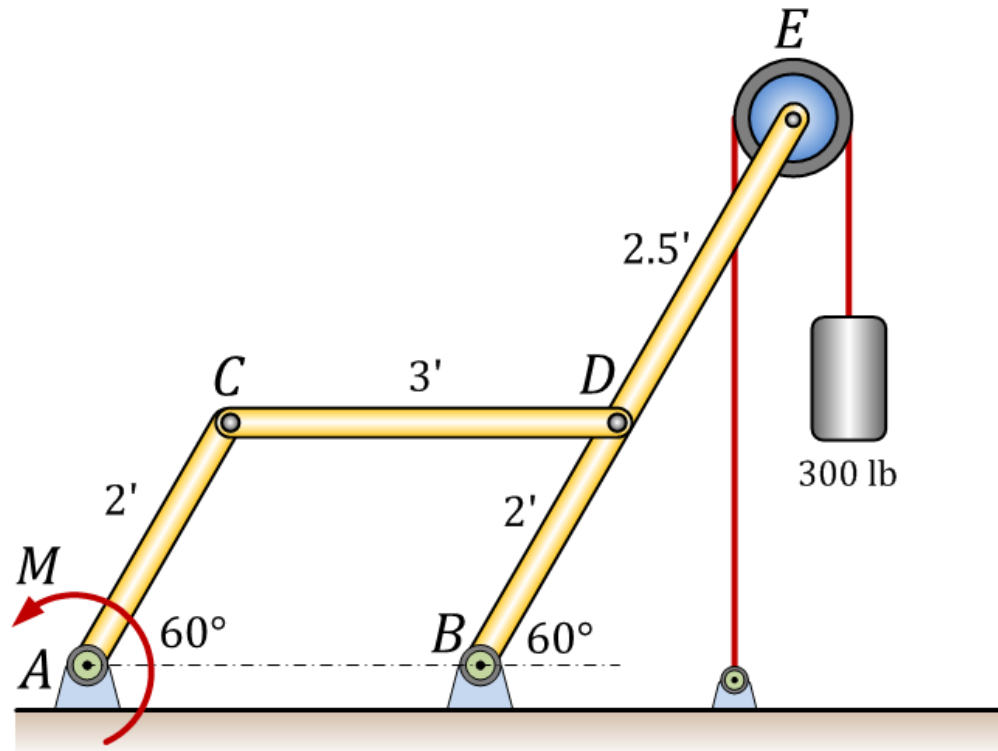
Calculate the force at the point  $E$



W4 Example 4 (Web view)

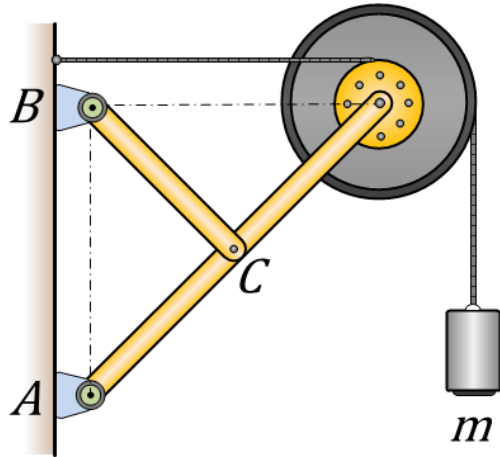
# Example 5

Calculate the force and moment at point A

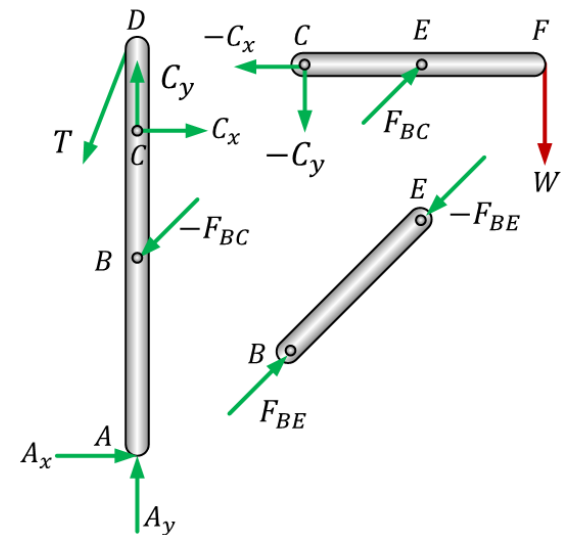
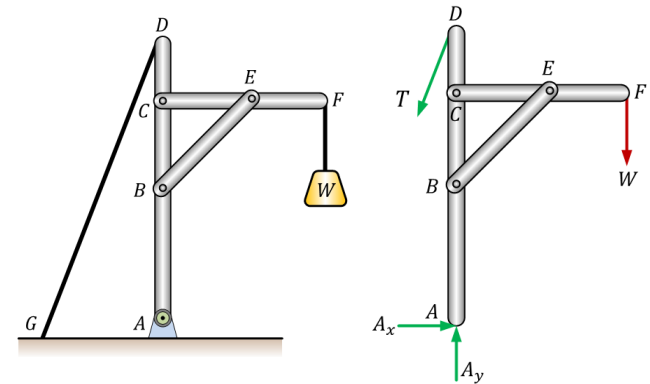


W4 Example 5 (Web view)

# Summary



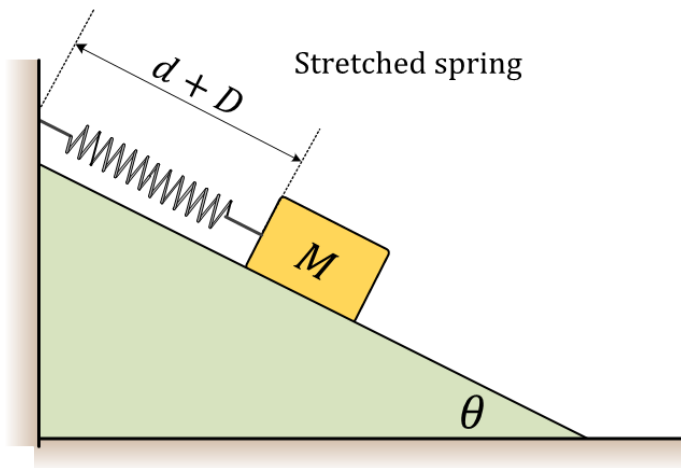
- A free body diagram of the complete frame is used to determine the external forces acting on the frame (similar to the truss analysis)
- Internal forces are determined by dismembering the frame and creating free-body diagram for each component



Next Topic:

*Friction and Springs*

## Week 4, L2 – Friction and Springs



### FRICTION

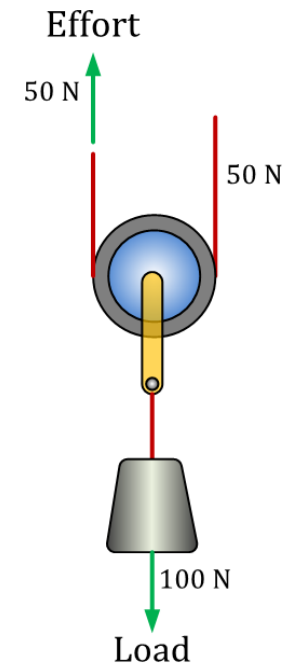
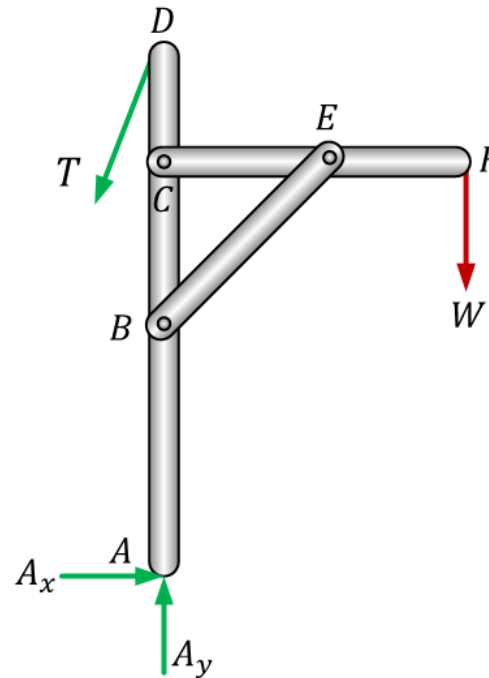
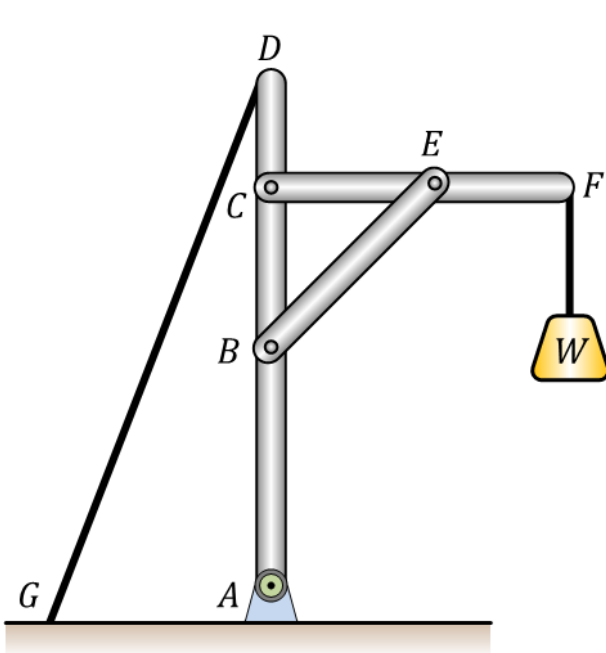
- Static and dynamic coefficient of friction
- Motion threshold
- Actual vs maximum friction force

### SPRINGS

- Spring equation
- Spring force and stretched lengths

# Frames and Machines Review

The mechanical advantage of a machine can be incorporated into a frame





# Example 5 – A Frame/Machine



At Central station, from Platform 22, you can see this tensioning device. It is both a Frame and a Machine.

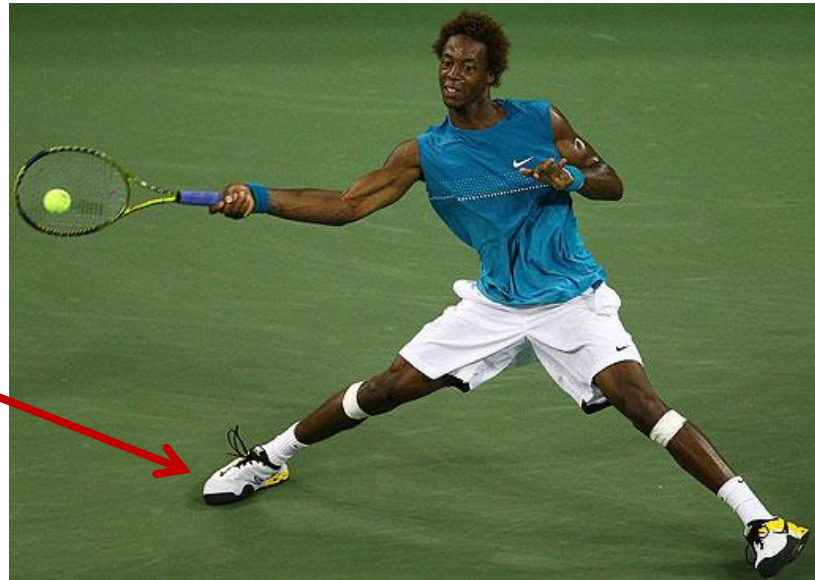
Calculate the declination angle  $\theta$  of the bracket.

Assume the deadweight is 10 kg, the pulley radii are 100mm and the centre spacings are 300mm.

W4 Example 6 (Web view)

# If you try to slide one solid object across another, you will notice a resistance to the sliding

Friction force is acting here



- In mechanics, we call the resistance the **friction force** (it should be obvious that the resistance constitutes a force)

# Friction force has a magnitude and a direction

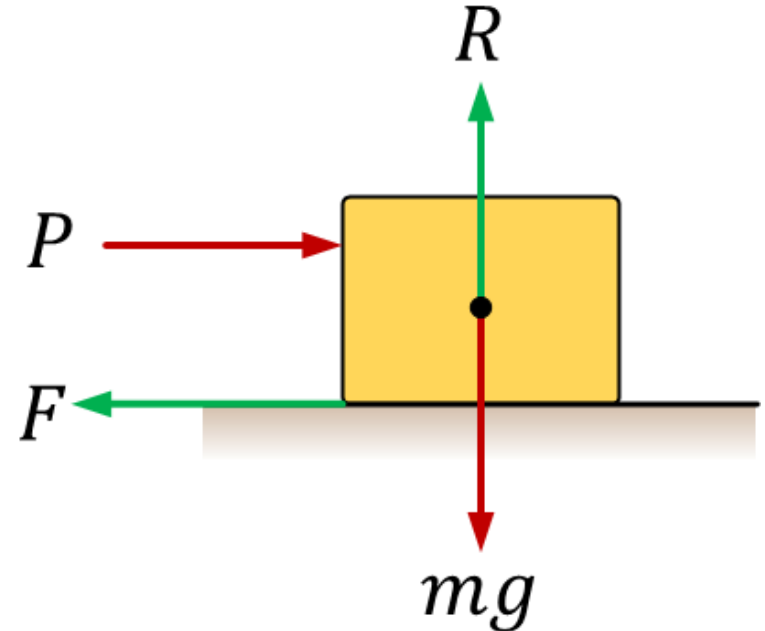
- The direction is parallel to the two contacting surfaces
- The direction is also such that it opposes the sliding
- If the two objects are not sliding with respect to each other, the direction of the force is such that it opposes the impending slip

# But how do we find the magnitude?

- In this course, we will use the Coulomb model of friction forces
- This model has limitations, but is widely applicable in engineering
- As you will see, we have to consider 2 different cases: static and dynamic friction

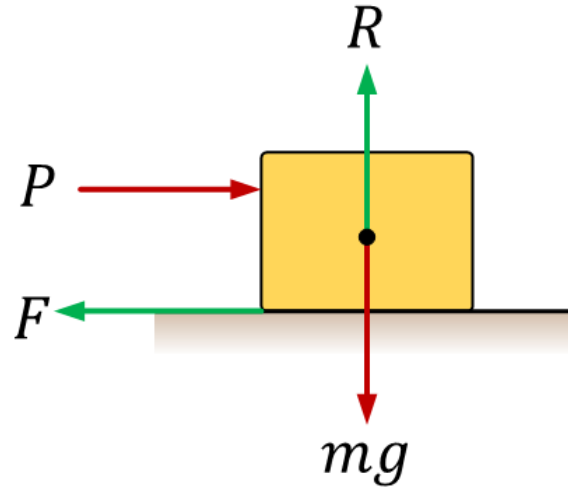
# Let's do a little experiment

- Consider a block of mass  $m$  that is initially at rest on a horizontal plane
- Initially, the applied force  $P = 0$
- Upon increasing  $P$  we can make some observations



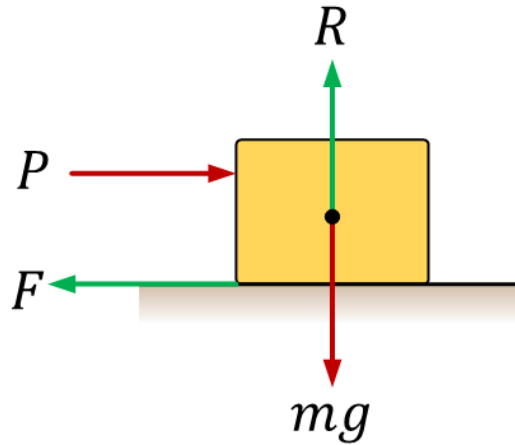
FBD of the block

# What relevant observations do we make?



1. For small  $P$ , the block remains at rest
2. Beyond a critical value  $P = P^*$ , the block starts to move
3. Once in motion, a constant value of  $P = P^{**}$ , is required to move the block at constant speed
4. Both  $P^*$  and  $P^{**}$  are proportional to the magnitude of the reaction force  $R$  and independent of the contact area

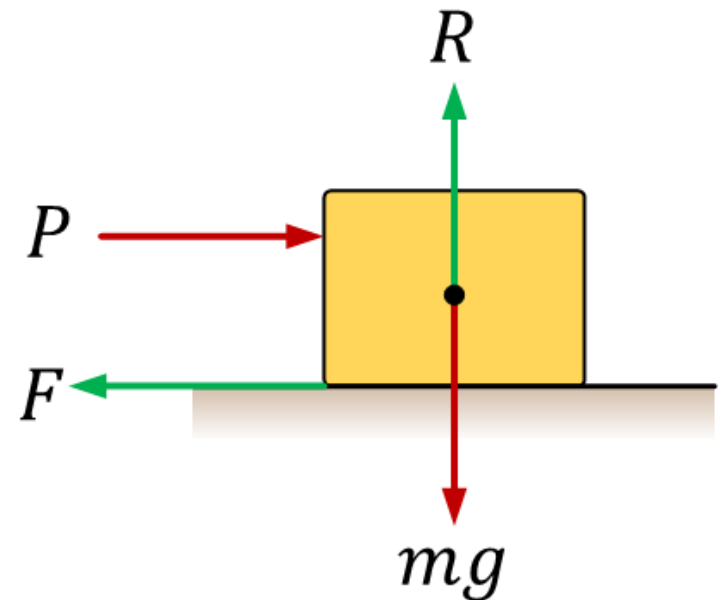
# Let's take a closer look at the static case (up until the block starts to move)



- Take  $\mathbf{F} = m\mathbf{a}$  in the horizontal ( $x$ ) direction
- For the static ( $\mathbf{a} = 0$ ) case,  $F = P$
- If  $P$  is increased beyond a critical value ( $P^*$ ), the block starts to move

We can notice (observation #4 above) that  $P^*$  is proportional to  $R$

- We denote the constant of proportionality by  $\mu_s$
- Hence,  $|F| \leq \mu_s |R|$





# Now let's examine the block once it starts moving

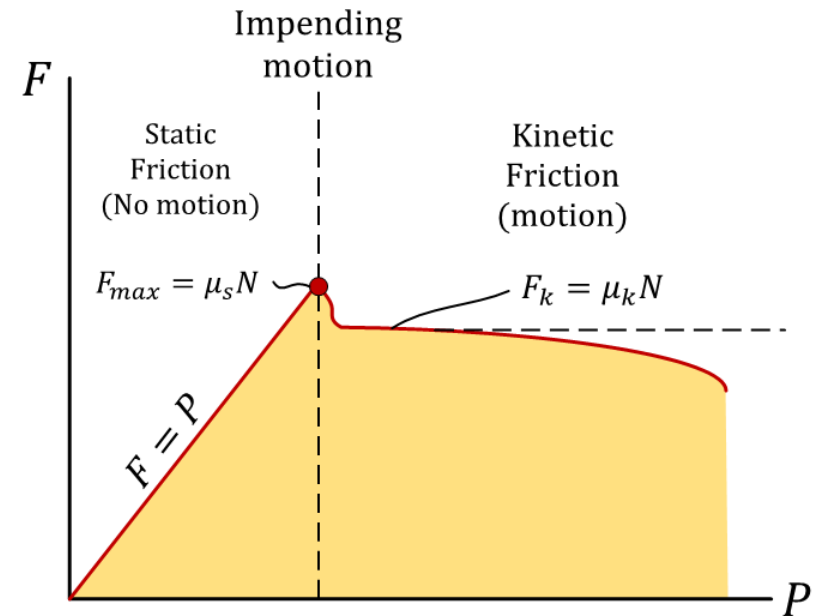
- When the block starts moving, the friction opposes motion
- The direction is opposite  $\mathbf{v}$
- As noted above, its magnitude is proportional to the magnitude of  $R$ , the reaction (normal) force
- Let's call the constant of proportionality  $\mu_k$
- Now Newton's 2<sup>nd</sup> Law gives us  $F = \mu_k R$

# Now we need some terminology

- We will call  $\mu_s$  “The coefficient of **static** friction”
- We will call  $\mu_k$  “The coefficient of **kinetic** friction”
- Generally  $\mu_k < \mu_s$
- Coefficients of friction are particular to pairs of materials (i.e. steel on steel, brass on steel, plastic on wood etc.)

# We **cannot** use $F_f = \mu_s N$ to calculate friction force in the static case

- The motion is known (i.e. there is no relative motion between the contacting objects)
- So we can use  $\mathbf{F} = m\mathbf{a}$  to find  $F_f$  even if  $\mathbf{a} = 0$
- The equation  $F_f = \mu_s N$  is **only valid** at impending slip



# Static Friction is a reaction force!

**DO NOT** use  $F_f = \mu_s N$  to find the static friction force

*(Except for at impending slip)*

# Linear Springs Hooke's Law

- “Ut tensio sic vis”  
–*Robert D. Hooke FRS*
- Translation: The power of any spring is in the same proportion with the tension thereof
- This is known as Hooke's Law



**Robert Hooke**

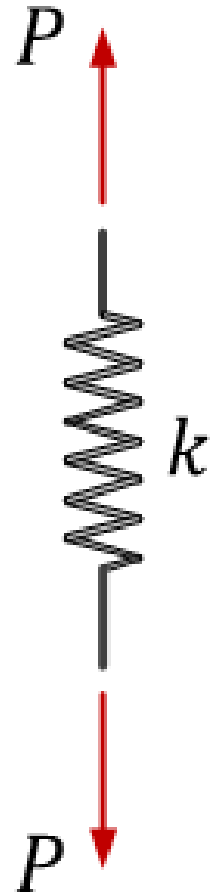
Physicist, Academic,  
Scientist, Scholar

(1635–1703)

# Hooke's Law

- Consider a FBD of a length of spring
- Springs are considered to be massless, but unlike strings, springs may be stretched or compressed
- If the spring extends by an amount  $\delta$  from its **unstretched** length, then the stiffness of the spring is defined as:

$$k = \frac{P}{\delta}$$



# Linear Springs

- With Hooke's Law, the relationship between the force and the displacement is linear
- The magnitude is given as  $F_s = k\delta$ 
  - Where  $\delta = L - L_0$
  - $L$  is the current length of the spring
  - $L_0$  is the unstretched length of the spring
- The direction is opposite the stretch (or compression) of the spring

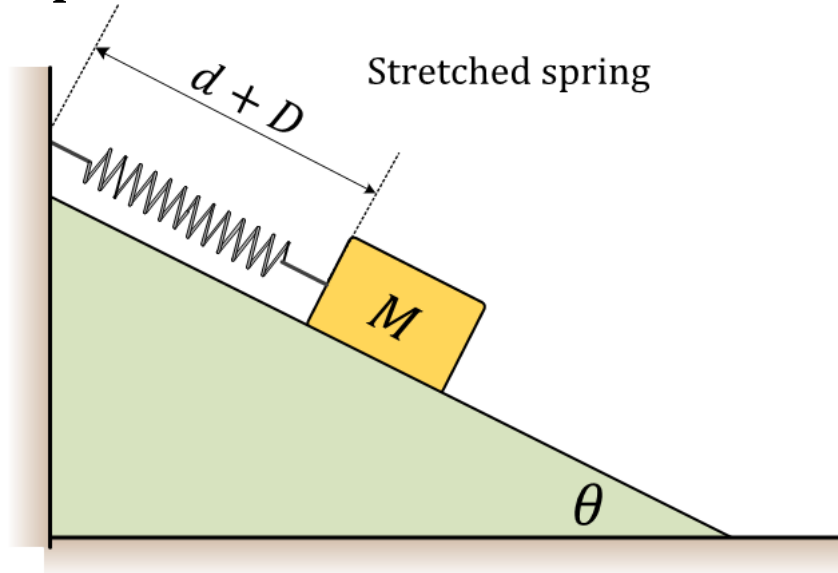
# Some notes on Springs

- It is easy to think of situations where Hooke's Law does not hold (but we will not deal with them in this course)
- In this course, we will solve most problems involving springs using Work/Energy Methods
- These will be covered in the coming weeks

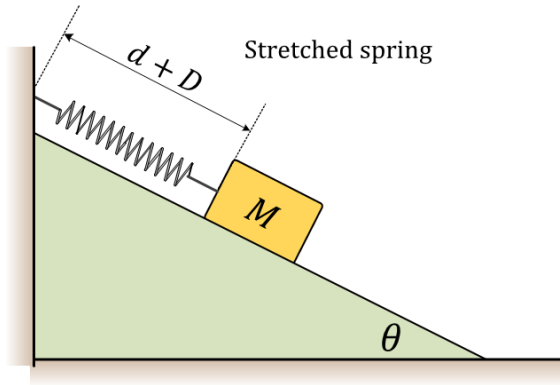


## Example 6

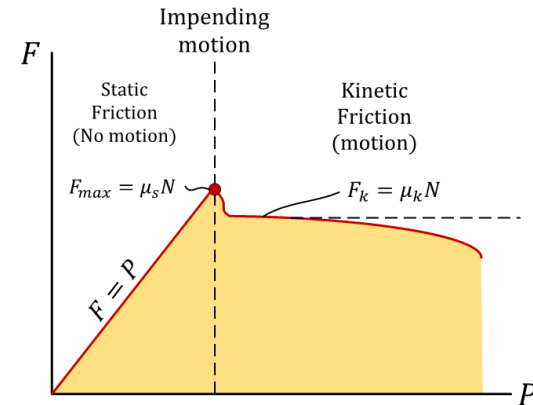
A spring with coefficient  $k$  is stretched by an amount  $d$  from its unstretched length  $D$  by a mass  $M$  on an incline of angle  $\theta$ . Calculate the minimum friction coefficient required to maintain equilibrium on the assumption that the friction is acting up the slope.



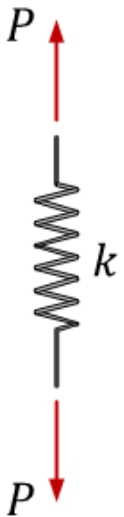
# Summary



- Potential friction force- the equation  $F_f = \mu_s N$  is **only valid** at impending slip



- Spring Force is given as  $F_s = k\delta$ 
  - Where  $\delta = L - L_0$
  - $L$  is the current length of the spring
  - $L_0$  is the unstretched length of the spring



Next Topic:

*Distributed Loads, Shear Force  
and Bending Moments*