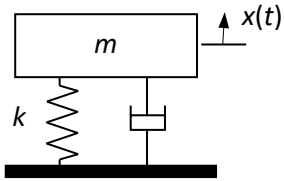


EQUATION SHEET FOR PART A

Spring-Mass-Damper Systems

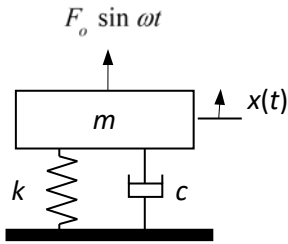
$$\omega = 2\pi f \quad \omega_n = \sqrt{\frac{k}{m}} \quad \omega_d = \omega_n \sqrt{1 - \zeta^2} \quad \zeta = \frac{c}{2\sqrt{km}}$$

$$\delta = \ln\left(\frac{x_n}{x_{n+1}}\right) \quad \delta = \frac{1}{n} \ln\left(\frac{x_1}{x_{n+1}}\right) \quad \zeta = \frac{\delta}{\sqrt{4\pi^2 + \delta^2}} \quad r = \frac{\omega}{\omega_n}$$



$$m\ddot{x} + c\dot{x} + kx = 0$$

$$x(t) = X e^{-\zeta\omega_n t} \sin(\omega_d t + \phi)$$

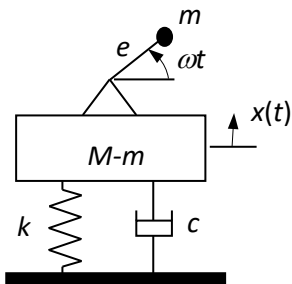


$$m\ddot{x} + c\dot{x} + kx = F_o \sin \omega t$$

$$x(t) = X \sin(\omega t - \phi)$$

$$X = \frac{F_o / k}{\sqrt{(1 - r^2)^2 + (2\zeta r)^2}}; \quad \phi = \tan^{-1}\left(\frac{2\zeta r}{1 - r^2}\right)$$

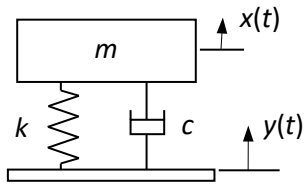
$$TR = \frac{F_T}{F_o} = \frac{\sqrt{1 + (2\zeta r)^2}}{\sqrt{(1 - r^2)^2 + (2\zeta r)^2}}$$



$$M\ddot{x} + c\dot{x} + kx = me\omega^2 \sin \omega t$$

$$x(t) = X \sin(\omega t - \phi)$$

$$\frac{MX}{me} = \frac{r^2}{\sqrt{(1 - r^2)^2 + (2\zeta r)^2}}; \quad \phi = \tan^{-1}\left(\frac{2\zeta r}{1 - r^2}\right)$$



$$m\ddot{x} + c\dot{x} + kx = c\dot{y} + ky$$

$$x(t) = X \sin(\omega t - \phi)$$

$$\frac{X}{Y} = \sqrt{\frac{1 + (2\zeta r)^2}{(1 - r^2)^2 + (2\zeta r)^2}}; \quad \frac{F_r}{kY} = r^2 \frac{\sqrt{1 + (2\zeta r)^2}}{\sqrt{(1 - r^2)^2 + (2\zeta r)^2}}$$

Continuous Systems

Lateral vibration of a taut string

$$\frac{\partial^2 u(x, t)}{\partial x^2} = \frac{1}{c_s^2} \frac{\partial^2 u(x, t)}{\partial t^2}, \quad c_s = \sqrt{\frac{T}{m_L}}$$

$$u(x, t) = \phi(x)q(t), \quad \phi(x) = C \sin kx + D \cos kx, \quad q(t) = A \sin \omega t + B \cos \omega t$$

Boundary Conditions

$$\underline{x = 0}$$

$$\underline{x = L}$$

Fixed

$$u = 0$$

$$u = 0$$

Longitudinal vibrations of rods

$$\frac{\partial^2 u(x, t)}{\partial x^2} = \frac{1}{c_L^2} \frac{\partial^2 u(x, t)}{\partial t^2}, \quad c_L = \sqrt{\frac{E}{\rho}}$$

Boundary Conditions

$$\underline{x = 0}$$

$$\underline{x = L}$$

Fixed

$$u = 0$$

$$u = 0$$

Free

$$\frac{\partial u}{\partial x} = 0$$

$$\frac{\partial u}{\partial x} = 0$$

Spring load

$$k_s u = EA \frac{\partial u}{\partial x}$$

$$k_s u = -EA \frac{\partial u}{\partial x}$$

Inertia load

$$M \frac{\partial^2 u}{\partial t^2} = EA \frac{\partial u}{\partial x}$$

$$M \frac{\partial^2 u}{\partial t^2} = -EA \frac{\partial u}{\partial x}$$

EQUATION SHEET FOR PART B

Constant Linear Acceleration Equations ($t_o = 0$)

$$v = v_o + at$$

$$v^2 = v_o^2 + 2a(s - s_o)$$

$$s = s_o + v_o t + \frac{1}{2} at^2$$

Angular Motion

$$\omega = \frac{d\theta}{dt} \quad \alpha = \frac{d\omega}{dt} \quad \omega d\omega = \alpha d\theta$$

Displacement, Velocity and Acceleration Components

(1) Rectangular Co-ordinates

$$\mathbf{r} = x\mathbf{i} + y\mathbf{j}$$

$$\mathbf{v} = \dot{x}\mathbf{i} + \dot{y}\mathbf{j}$$

$$\mathbf{a} = \ddot{x}\mathbf{i} + \ddot{y}\mathbf{j}$$

(2) Normal and Tangential Co-ordinates

$$\mathbf{v} = v\mathbf{e}_t$$

$$\mathbf{a} = a_t\mathbf{e}_t + a_n\mathbf{e}_n$$

$$v = \omega r$$

$$a_t = \dot{v} = \alpha r$$

$$a_n = \frac{v^2}{\rho} = \omega^2 r$$

Relative Motion

$$\mathbf{r}_A = \mathbf{r}_B + \mathbf{r}_{A/B}$$

$$\mathbf{v}_A = \mathbf{v}_B + \mathbf{v}_{A/B}$$

$$\mathbf{a}_A = \mathbf{a}_B + \mathbf{a}_{A/B}$$

Relative Motion for Rotating Axes

$$\mathbf{v}_A = \mathbf{v}_{A'} + \mathbf{v}_{A/A'}$$

$$\mathbf{a}_A = \mathbf{a}_{A'} + \mathbf{a}_{\text{rel}} + \mathbf{a}_{\text{cor}}$$

Moment of Inertia

Moment of inertia of a uniform disk:

$$I = \frac{1}{2} mr^2$$

Moment of inertia of a uniform slender rod:

$$I = \frac{1}{12} ml^2$$

Parallel axis theorem:

$$I_o = I_G + md^2$$