

Homework #5

(due: Dec. 4)

Show all the work/derivation with neat writing.

1. We learned that if a linear time-invariant system is controllable, then we can design state feedback for which the resulting closed loop can have its eigenvalues at any locations in the complex plane. We validate this using a single-input system.

Consider a controllable single-input linear system given by

$$\dot{x}(t) = Ax(t) + Bu(t) \quad (7)$$

where $x(t) \in \mathbb{R}^n$ and $u(t) \in \mathbb{R}$. Let

$$\mathcal{C}^{-1} = \begin{pmatrix} M_1 \\ M_2 \\ \vdots \\ M_n \end{pmatrix} \quad \text{and} \quad P = \begin{pmatrix} M_n \\ M_n A \\ \vdots \\ M_n A^{n-1} \end{pmatrix}$$

where $\mathcal{C} = (B \ AB \ \cdots \ A^{n-1}B)$ is the controllability matrix. Suppose P is invertible. (Note: M_n in P is same as the last row of \mathcal{C}^{-1})

- (a) Validate that with the similarity transformation $z(t) = Px(t)$, we can derive

$$\dot{z}(t) = \begin{pmatrix} 0 & 1 & \cdots & 0 \\ 0 & 0 & \cdots & 0 \\ \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \cdots & 1 \\ \alpha_0 & \alpha_1 & \cdots & \alpha_{n-1} \end{pmatrix} z(t) + \begin{pmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{pmatrix} u(t) \quad (8)$$

(Hint: Use $\mathcal{C}^{-1}\mathcal{C} = I_n$ and $A^n = \sum_{k=0}^{n-1} \alpha_k A^k$ by Cayley-Hamilton theorem.)

2. Consider a single-input single-output linear time-invariant system given by

$$\begin{aligned} \dot{x}_1(t) &= -x_1(t) - 4x_2(t) + 2u(t) \\ \dot{x}_2(t) &= x_1(t) + 3x_2(t) + u(t) \\ y(t) &= x_1(t) \end{aligned}$$
 - (a) Design state feedback $u(t) = K \begin{pmatrix} x_1(t) \\ x_2(t) \end{pmatrix}$ to stabilize the system. Place the eigenvalues of the closed loop at $(-1, -3)$.

- (b) Design Luenberger observer to estimate the state of the system. Place the eigenvalues of the state equation for the estimation error at $(-1, -3)$.
- (c) Discuss how you can use the answers from (a) and (b) to design output feedback that stabilizes the system..

3. Consider the linear system given by

$$\begin{aligned}\dot{x}_1(t) &= -x_2(t) \\ \dot{x}_2(t) &= x_1(t) + u(t) \\ y(t) &= x_1(t)\end{aligned}$$

with the initial condition $x_1(0) = 1$ and $x_2(0) = 0$.

- (a) Validate that with $u(t) = 0$, the system exhibits the rotational motion with velocity 1 (rad/s) and radius 1.
- (b) Using the Luenberger observer and state feedback, can you design output feedback that allows the system's state to track the rotational motion trajectory with velocity 1 (rad/s) and radius 2: $\bar{x}_1(t) = 2 \cos t$, $\bar{x}_2(t) = 2 \sin t$?
4. Consider the (nonlinear) state equation for a simple racing car (moving at constant velocity 1 m/s) given by

$$\begin{aligned}\dot{p}_x(t) &= \cos \theta(t) \\ \dot{p}_y(t) &= \sin \theta(t) \\ \dot{\theta}(t) &= \tan u(t)\end{aligned}$$

We aim to design state feedback control that allows the car to follow a racing track depicted below.

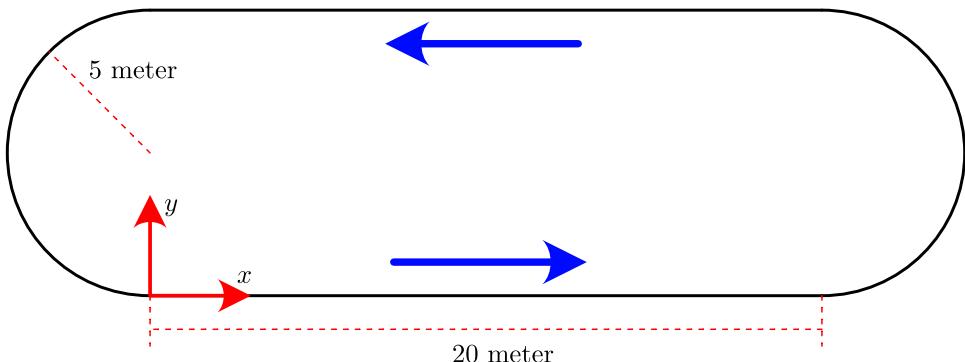
Let

$$u(t) = \mu_{\text{Linear}}(p_y(t), \theta(t)) = K_1 \begin{pmatrix} p_y(t) \\ \theta(t) \end{pmatrix}$$

be a controller that allows the car to track a straight line originating from the origin along the x -axis, and

$$\begin{aligned}u(t) &= \mu_{\text{Circular}}(p_x(t), p_y(t), \theta(t), p_{c,x}, p_{c,y}, r_0) \\ &= \arctan \left(\frac{1}{r_0} \right) + K_2 \left(\theta(t) - \left(\frac{\pi}{2} + \arctan \left(\frac{p_y(t) - p_{c,y}}{p_x(t) - p_{c,x}} \right) \right) \right)\end{aligned}$$

be a controller that allows the car to track a circular track centered at $(p_{c,x}, p_{c,y})$ with radius r_0 . Use both of the controllers μ_{Linear} and μ_{Circular} to design state feedback control to maneuver the car along the below racing track. (No need to compute K_1, K_2 .)



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Homework 5

1. consider a controllable single-input linear system

$$\dot{X}(t) = AX(t) + BU(t). \quad X(t) \in \mathbb{R}^n, U(t) \in \mathbb{R}$$

Let:

$$C^{-1} = \begin{pmatrix} M_1 \\ M_2 \\ M_3 \\ \vdots \\ M_n \end{pmatrix} \quad P = \begin{pmatrix} M_n \\ M_n A \\ M_n A^2 \\ \vdots \\ M_n A^{n-1} \end{pmatrix}$$

where $C = (B \ AB \ \dots \ A^{n-1}B)$ is the controllability matrix

Suppose P is invertible.

a) Validate that with the similarity transformation

$Z(t) = PX(t)$ we can derive

$$\dot{Z}(t) = \begin{vmatrix} 0 & 1 & \cdots & 0 \\ 0 & 0 & \cdots & 0 \\ 0 & 0 & \cdots & 1 \\ d_0 & d_1 & \cdots & d_{n-1} \end{vmatrix} Z(t) + \begin{pmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{pmatrix} U(t)$$

\rightarrow we have state eq $\rightarrow \dot{X}(t) = AX(t) + BU(t)$

and $Z(t) = PX(t)$, $\rightarrow X(t) = P^{-1}Z(t)$

then: $\dot{P}^{-1}\dot{Z}(t) = A\dot{P}^{-1}Z(t) + BU(t)$

$$\boxed{\dot{Z}(t) = PAP^{-1}Z(t) + PBU(t)}$$

Demonstrate that $PAP^{-1} \rightarrow \begin{vmatrix} 0 & 1 & 0 & \cdots & 0 \\ 0 & 0 & 1 & \cdots & 0 \\ \vdots & & & & \\ d_0 & d_1 & d_2 & \cdots & d_{n-1} \end{vmatrix}$

$$PA = \begin{pmatrix} M_n A \\ M_n A^2 \\ \vdots \\ M_n A^n \end{pmatrix} \longrightarrow$$

Using Cayley-Hamilton theorem

$$A^n = \sum_{k=0}^{n-1} \alpha_k A^k \rightarrow M_n A^n = \sum_{k=0}^{n-1} M_n A^{n-k}$$

$$\text{where } \rightarrow PA = \begin{vmatrix} M_n A \\ M_n A^2 \\ \vdots \\ M_n A^n \end{vmatrix} = \begin{vmatrix} 0 & 1 & 0 & \cdots & 0 \\ 0 & 0 & 1 & \cdots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \cdots & 1 & 0 \\ \alpha_0 & \alpha_1 & \cdots & \alpha_{n-1} & \end{vmatrix} \begin{vmatrix} M_n \\ M_n A \\ M_n A^2 \\ \vdots \\ M_n A^{n-1} \end{vmatrix}$$

M P

We can define $PA = MP$

$$\text{then } \rightarrow PAP^{-1} = (MP)P^{-1} = MPP^{-1} = M.$$

$$\rightarrow \text{Now for } PB \rightarrow PB = \begin{vmatrix} M_n B \\ M_n AB \\ \vdots \\ M_n A^{n-1} B \end{vmatrix} \quad \text{define } V = \begin{pmatrix} 0 \\ \vdots \\ 0 \\ 1 \end{pmatrix} \quad V \in \mathbb{R}^n.$$

and $C = (B \ AB \ \dots \ A^{n-1}B)$.

We can define PB as:

$$PB = \begin{vmatrix} M_n B \\ M_n AB \\ \vdots \\ M_n A^{n-1} B \end{vmatrix} = \underbrace{(C^\top V)}_{\downarrow} C = (C^\top C)V = V$$

$$\text{Recall } C^\top = (M_1 \ M_2 \ \dots \ M_n)$$

$$C^\top V = M_n$$

Therefore:

$$\dot{z}(t) = PAP^{-1}z(t) + PBu(t)$$

$$\dot{z}(t) = MPP^{-1}z(t) + (C^\top C)Vu(t) \rightarrow M = \begin{vmatrix} 0 & 1 & 0 & \cdots & 0 \\ 0 & 0 & 1 & \cdots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & 0 & \cdots & 1 \\ \alpha_0 & \alpha_1 & \alpha_2 & \cdots & \alpha_{n-1} \end{vmatrix} \quad V = \begin{pmatrix} 0 \\ \vdots \\ 0 \\ 1 \end{pmatrix}$$

b) Can you find state feedback $u(t) = Kz(t)$ such that the eigenvalues of the resulting closed loop can be assigned to any points in the complex plane?

$$\dot{z}(t) = Mz(t) + Vu(t) \rightarrow u(t) = Kz(t)$$

$$\dot{z}(t) = Mz(t) + VKz(t) = (M + VK)z(t)$$

$$\dot{z}(t) = (M + VK)z(t)$$

For the System $\dot{z}(t) = (M + VK) z(t)$

$$\dot{z}(t) = \begin{pmatrix} 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 1 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \dots & 0 & 1 \\ d_0 & d_1 & d_2 & \dots & d_{n-1} \end{pmatrix} + \begin{pmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{pmatrix} (K_1 \dots K_n) z(t)$$

$$\dot{z}(t) = \begin{pmatrix} 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 1 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \dots & 0 & 1 \\ d_0 + K_1 & d_1 + K_2 & d_2 + K_3 & \dots & d_{n-1} + K_n \end{pmatrix} z(t)$$

We have:

For a single-input, single output linear system the transfer function is $\rightarrow H(s) = \frac{N(s)}{D(s)} = C(SI - A)^{-1}B$

$$\rightarrow H(s) = \frac{1}{\det(SI - A)} C(A \text{adj}(SI - A)) B$$

we can find a state feedback $u(t) = Kz(t)$ if the system has a transfer function $H(s)$ where $N(s)$ and $D(s)$ are coprime, it means that the controller form is a minimal realization which implies that the state equation is controllable and observable \rightarrow resulting in that we can select K that the closed loop system have negative eigenvalues.

c) Apply state feedback $u(t) = KPX(t)$

$$\dot{X}(t) = AX(t) + BU(t)$$

$$\rightarrow \dot{X}(t) = AX(t) + BKPX(t)$$

$$\dot{X}(t) = (A + BKP)X(t). \quad (1)$$

$$\rightarrow \text{we have } X(t) = P^{-1}z(t) \quad (2)$$

$$(1) \ 1 \ (2) \ \therefore P^{-1}\dot{z}(t) = (A + BKP)P^{-1}z(t)$$

$$\dot{z}(t) = P(A + BKP)P^{-1}z(t)$$

$$\rightarrow \dot{\bar{z}}(t) = \bar{P}(A + BK\bar{P})\bar{P}^{-1}\bar{z}(t)$$

$$\dot{\bar{z}}(t) = (PA\bar{P}^{-1} + PBK)\bar{z}(t)$$

\rightarrow we have demonstrated that $PAP^{-1} = M = \begin{vmatrix} 0 & 1 & 0 & \cdots & 0 \\ 0 & 0 & 1 & \cdots & 0 \\ \vdots & & & & \\ d_0 & d_1 & d_2 & \cdots & d_{n-1} \end{vmatrix}$
and that $PB = V = \begin{pmatrix} 0 \\ 0 \\ \vdots \\ 1 \end{pmatrix}$

$$\text{Then } \rightarrow \dot{\bar{z}}(t) = (PA\bar{P}^{-1} + PBK)\bar{z}(t) = (M + VK)\bar{z}(t)$$

$$\dot{\bar{z}}(t) = \begin{vmatrix} 0 & 1 & 0 & \cdots & 0 \\ 0 & 0 & 1 & \cdots & 0 \\ \vdots & & & & \\ d_0 t K_n & d_1 t K_n & d_2 t K_n & \cdots & d_{n-1} t K_n \end{vmatrix} \bar{z}(t)$$

\rightarrow Applying the state feedback control $u(t) = KPx(t)$
we can have the same result that in b \rightarrow this implies
that the $u(t)$ applied can permit the selection of
 K to have determined eigenvalues for the closed loop
System and due to the reasons explained in literal b.

2 → Consider a single input linear time-invariant system given by:

$$\dot{x}_1(t) = -x_1(t) - 4x_2(t) + 2u(t)$$

$$\dot{x}_2(t) = x_1(t) + 3x_2(t) + u(t)$$

$$y(t) = x_1(t)$$

a. Design state feedback $u(t) = K \begin{pmatrix} x_1(t) \\ x_2(t) \end{pmatrix}$ to stabilize the system. place the eigenvalues of the closed loop at $(-1, -3)$

1 → Define the state $x(t) = (x_1(t), x_2(t))^T$

2 → write down the state equation.

$$\dot{x}(t) = \begin{pmatrix} \dot{x}_1(t) \\ \dot{x}_2(t) \end{pmatrix} = \begin{pmatrix} -1 & -4 \\ 1 & 3 \end{pmatrix} \begin{pmatrix} x_1(t) \\ x_2(t) \end{pmatrix} + \begin{pmatrix} 2 \\ 1 \end{pmatrix} u(t)$$

$$y(t) = (1 \ 0)^T x(t)$$

3 → First we want to check if the system is stabilizable.

We want to check if the system is controllable.

using controllability rank condition

$$\text{rank}(B \ AB) \Rightarrow AB = \left| \begin{array}{cc|c} -1 & -4 & 2 \\ 1 & 3 & 1 \end{array} \right| = \left| \begin{array}{c} -6 \\ 5 \end{array} \right|$$

$\text{rank} \left| \begin{array}{cc} 2 & -6 \\ 1 & 5 \end{array} \right| = 2 \rightarrow$ Given that the system is controllable then the system is stabilizable

4 → Design state feedback $u(t) = Kx(t)$ that stabilize the system

$$\dot{x}(t) = Ax(t) + Bu(t)$$

$$\dot{x}(t) = Ax(t) + BKx(t)$$

$$\dot{x}(t) = (A + BK)x(t)$$

$$\text{Compute } BK \rightarrow BK = \begin{pmatrix} 2 \\ 1 \end{pmatrix} (K_1 \ K_2) = \begin{pmatrix} 2K_1 & 2K_2 \\ 1K_1 & 1K_2 \end{pmatrix}$$

Then the Result of state equation + Feedback design.

$$\dot{X}(t) = (A+BK)X(t) = \begin{pmatrix} -1+2K_1 & -9+2K_2 \\ 1+K_1 & 3+K_2 \end{pmatrix} X(t)$$

5 → Compute the K -values to have negative eigenvalues → $\lambda = -1, \lambda = -3$

define → $M = (A+BK)$

$$\text{then } \det(M-\lambda I) = \begin{vmatrix} -1+2K_1-\lambda & -9+2K_2 \\ 1+K_1 & 3+K_2-\lambda \end{vmatrix}$$

$$= (-1+2K_1-\lambda)(3+K_2-\lambda) - (-9+2K_2)(1+K_1)$$

$$= -3 - K_2 + \underline{\lambda} + 6K_1 + 2K_1K_2 - 2K_1\lambda - 3\lambda - K_2\lambda + \underline{\lambda^2} - (-9 - 9K_1 + 2K_2 + 2K_1K_2)$$

$$= \lambda^2 + (-2 - 2K_1 - K_2)\lambda + (1 + 10K_1 - 3K_2)$$

$$\text{we want } \lambda = -1, \lambda = -3 \rightarrow (\lambda+1)(\lambda+3) = \underline{\lambda^2 + 4\lambda + 3 = 0}$$

$-3\lambda - 1\lambda + 3$ we want to have
this equation

$$\rightarrow -2 - K_2 - 2K_1 = 4$$

$$\textcircled{1} \quad K_2 + 2K_1 = -6 \rightarrow K_2 = -6 - 2K_1$$

$$\rightarrow 1 + 10K_1 - 3K_2 = 3$$

$$\textcircled{2} \quad 1 + 10K_1 + 18 + 6K_1 = 3 \rightarrow 16K_1 = -16$$

$$\rightarrow K_1 = -1 \quad \boxed{K_1 = -1} \quad \boxed{K_2 = -4}$$

with $K = (-1, -4) \rightarrow$ System is stable.

b) Design Luenberger observer to estimate the state of the system. Place the eigenvalues of the state equation for the estimation error at $(-1, -3)$

Given the system

$$\dot{\mathbf{x}}(t) = \begin{pmatrix} -1 & -1 \\ 1 & 3 \end{pmatrix} \mathbf{x}(t) + \begin{pmatrix} 2 \\ 1 \end{pmatrix} u(t)$$

$$\mathbf{y}(t) = \begin{pmatrix} 1 & 0 \end{pmatrix} \mathbf{x}(t)$$

1 \rightarrow We want to check if the System is observable to finally design the Luenberger observer.
Using the observability rank condition.

$$\text{rank} \begin{pmatrix} C \\ CA \end{pmatrix} \rightarrow CA = \begin{pmatrix} 1 & 0 \end{pmatrix} \begin{pmatrix} -1 & -1 \\ 1 & 3 \end{pmatrix} = \begin{pmatrix} -1 & -1 \end{pmatrix}$$

$\text{rank} \begin{pmatrix} 1 & 0 \\ -1 & 1 \end{pmatrix} = 2 \rightarrow$ Then the system is observable
we can create a Luenberger observer

2 \rightarrow Design Luenberger observer to estimate
we define $\tilde{\mathbf{x}}(t) = \mathbf{x}(t) - \hat{\mathbf{x}}(t)$

$$\dot{\tilde{\mathbf{x}}} = A\tilde{\mathbf{x}}(t) + L(\mathbf{y}(t) - C\hat{\mathbf{x}}(t))$$

$$\dot{\tilde{\mathbf{x}}} = A\underbrace{\mathbf{x}(t)}_{\tilde{\mathbf{x}}(t)} - A\hat{\mathbf{x}}(t) - L(\mathbf{y}(t) - C\hat{\mathbf{x}}(t))$$

$$\dot{\tilde{\mathbf{x}}} = A\tilde{\mathbf{x}}(t) - L(C\mathbf{x}(t) - C\hat{\mathbf{x}}(t))$$

$$\dot{\tilde{\mathbf{x}}} = A\tilde{\mathbf{x}}(t) - LC\tilde{\mathbf{x}}(t)$$

$$\boxed{\dot{\tilde{\mathbf{x}}}(t) = (A - LC)\tilde{\mathbf{x}}(t)}$$

Define :

$$L = \begin{pmatrix} L_1 \\ L_2 \end{pmatrix} \rightarrow LC = \begin{pmatrix} L_1 \\ L_2 \end{pmatrix} \begin{pmatrix} 1 & 0 \end{pmatrix} = \begin{pmatrix} L_1 & 0 \\ L_2 & 0 \end{pmatrix}$$

Then the Luenberger observer can be expressed:

$$\dot{\tilde{X}}(t) = (A - LC)\tilde{X} = \left(\begin{pmatrix} -1 & -4 \\ 1 & 3 \end{pmatrix} - \begin{pmatrix} L_1 & 0 \\ L_2 & 0 \end{pmatrix} \right) = \begin{pmatrix} -1-L_1 & -4 \\ 1-L_2 & 3 \end{pmatrix}$$

3 → Compute the L_i values to have negative eigenvalues

$$\lambda_1 = -1 \quad \lambda_2 = -3$$

→ we want the characteristic equation of $(A - LC)$

to be: $(\lambda + 1)(\lambda + 3) = 0$

$$\lambda^2 + 3\lambda + \lambda + 3 = 0 \rightarrow \boxed{\lambda^2 + 4\lambda + 3 = 0}$$

Then:

define $\rightarrow M = A - LC = \begin{pmatrix} -1-L_1 & -4 \\ 1-L_2 & 3 \end{pmatrix}$

Then →

$$\begin{aligned} \det(M - \lambda I) &= \begin{vmatrix} -1-L_1-\lambda & -4 \\ 1-L_2 & 3-\lambda \end{vmatrix} = (-1-\lambda)(3-\lambda) - (-4)(1-L_2) \\ &= (-1-L_1-\lambda)(3-\lambda) - (-4)(1-L_2) \\ &= \underline{\lambda - 3} - 3L_1 + \underline{L_1\lambda} - 3\lambda + \lambda^2 - (-4 + 4L_2) \\ &= \lambda^2 + (L_1 - 2)\lambda + (1 - 3L_1 - 4L_2) \end{aligned}$$

$$\rightarrow L_1 - 2 = 4 \rightarrow \boxed{L_1 = 6} \rightarrow 1 - 3L_1 - 4L_2 = 3$$

$$1 - 18 - 4L_2 = 3 \rightarrow \boxed{L_2 = -5}$$

with $L = \begin{pmatrix} 6 \\ -5 \end{pmatrix}$ we have a Luenberger observer
for the estimation error.

C) Discuss how you can use the answers from (a) and (b) to design output feedback that stabilizes the system.

We just proved that the system is controllable and observable, after that we have designed the state feedback giving negative eigenvalues to $(A+BK)$ to make the system stable. Similarly, we designed a Luenberger observer to define an estimation error, and making the system $(A-LC)$ stable giving negative eigenvalues.

i) From state feedback design $\rightarrow U(t) = K\hat{x}(t)$

$$\dot{x}(t) = Ax(t) + Bu(t) \quad \uparrow \text{estimation}$$

$$\dot{x}(t) = Ax(t) + BK\hat{x}(t)$$

ii) From observer design $\dot{\hat{x}}(t) = (A-LC)\hat{x}(t)$

$$\hat{x}(t) = x(t) - \hat{x}(t)$$

\downarrow Estimation

\hookrightarrow Estimation Error

To create an output feedback we can design a custom state to combine $x(t)$ and $\hat{x}(t)$, and create a new state equation:

\rightarrow state: $x'(t) = (x(t), \hat{x}(t))'$

$$\hat{x}(t) = x(t) - \hat{x}(t)$$

$$\begin{pmatrix} \dot{x}(t) \\ \dot{\hat{x}}(t) \end{pmatrix} = \begin{pmatrix} Ax(t) + BK\hat{x}(t) \\ (A-LC)\hat{x}(t) \end{pmatrix} = \begin{pmatrix} (A+BK)x(t) - BK\hat{x}(t) \\ (A-LC)\hat{x}(t) \end{pmatrix}$$

$$\begin{pmatrix} \dot{x}(t) \\ \dot{\hat{x}}(t) \end{pmatrix} = \begin{pmatrix} (A+BK) & -BK \\ 0 & A-LC \end{pmatrix} \begin{pmatrix} x(t) \\ \hat{x}(t) \end{pmatrix}$$

Output feedback design

The observer and state feedback stabilize the system $\rightarrow (A+BK)$ with negative eigenvalues
 $\rightarrow (A-LC)$ with negative eigenvalues.

3 → Consider the linear system given by

$$\dot{x}_1(t) = -x_2(t)$$

$$\dot{x}_2(t) = x_1(t) + u(t)$$

$$y(t) = x_1(t)$$

with initial condition $x_1(0) = 1$ and $x_2(0) = 0$

- a) Validate that with $u(t) = 0$, the system exhibits the rotational motion with velocity 1 (rad/s) and radius 1.

1 → First we define the state. $X(t) = (x_1(t), x_2(t))^T$

2 → State equation

$$\begin{pmatrix} \dot{x}_1(t) \\ \dot{x}_2(t) \end{pmatrix} = \begin{pmatrix} 0 & -1 \\ 1 & 0 \end{pmatrix} \begin{pmatrix} x_1(t) \\ x_2(t) \end{pmatrix} + \begin{pmatrix} 0 \\ 1 \end{pmatrix} u(t) \quad X(0) = \begin{pmatrix} 1 \\ 0 \end{pmatrix}$$

$$y(t) = (1 \ 0) X(t)$$

→ With $u(t) = 0$:

$$\begin{pmatrix} \dot{x}_1(t) \\ \dot{x}_2(t) \end{pmatrix} = \underbrace{\begin{pmatrix} 0 & -1 \\ 1 & 0 \end{pmatrix}}_A \begin{pmatrix} x_1(t) \\ x_2(t) \end{pmatrix}$$

→ Find the solution to this equation.

$$\det(A - \lambda I) = \begin{vmatrix} -\lambda & -1 \\ 1 & -\lambda \end{vmatrix} = \lambda^2 + 1 = 0 \quad \Rightarrow \text{eigenvalues } \lambda = i \quad \lambda = -i$$

Eigenvectors →

$$\lambda = i \rightarrow (A - \lambda I)V = 0 \quad \left| \begin{array}{cc|c} -i & -1 & V_1 \\ 1 & -i & V_2 \end{array} \right| = \begin{pmatrix} 0 \\ 0 \end{pmatrix} \quad \Rightarrow -iV_1 - V_2 = 0 \\ V_2 = -iV_1$$

$$\lambda = -i \rightarrow (A - \lambda I)V = 0 \quad \left| \begin{array}{cc|c} i & -1 & V_1 \\ 1 & i & V_2 \end{array} \right| = \begin{pmatrix} 0 \\ 0 \end{pmatrix} \quad V_{\lambda_1} = (1, -i) \\ V_{\lambda_2} = (1, i)$$

$$iV_1 - V_2 = 0$$

then → Matrix exponential

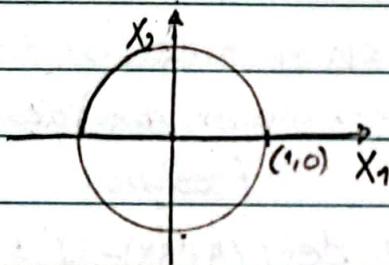
$$e^{At} = \begin{vmatrix} 1 & 1 \\ -i & i \end{vmatrix} \begin{vmatrix} e^{it} & 0 \\ 0 & e^{-it} \end{vmatrix} \begin{vmatrix} 1 & 1 \\ -i & i \end{vmatrix}^{-1} = \begin{vmatrix} e^{it} & e^{-it} \\ -ie^{it} & ie^{-it} \end{vmatrix} \begin{vmatrix} 1/2 & i/2 \\ i/2 & -i/2 \end{vmatrix}$$

$$\rightarrow e^{At} = \begin{vmatrix} \frac{e^{it} + \bar{e}^{-it}}{2} & \frac{-e^{it} + \bar{e}^{-it}}{2i} \\ \frac{e^{it} - \bar{e}^{-it}}{2} & \frac{e^{it} + \bar{e}^{-it}}{2} \end{vmatrix} = \begin{vmatrix} \cos(t) & -\sin(t) \\ \sin(t) & \cos(t) \end{vmatrix}$$

\rightarrow If $u(t)=0 \rightarrow$ the solution to the linear system is

$$X(t) = e^{At} X(0)$$

$$X(t) = \begin{vmatrix} \cos t & -\sin t \\ \sin t & \cos t \end{vmatrix} \begin{pmatrix} 1 \\ 0 \end{pmatrix} = \begin{pmatrix} \cos t \\ \sin t \end{pmatrix} \rightarrow \text{This is a rotational motion.}$$



- b) using the Luenberger observer and state feedback,
can you design output feedback that allows the
system's state to track the rotational motion
trajectory with velocity 1 (rad/s) and radius 2
 $\bar{x}_1(t) = 2 \cos t \quad \bar{x}_2(t) = 2 \sin t.$

1 \rightarrow Define trajectory reference $\bar{x}(0) = (2, 0)^T$

$$\bar{X} = \begin{pmatrix} \bar{x}_1(t) \\ \bar{x}_2(t) \end{pmatrix} = \begin{pmatrix} 2 \cos t \\ 2 \sin t \end{pmatrix} \quad \dot{\bar{X}} = \begin{pmatrix} -2 \sin t \\ 2 \cos t \end{pmatrix} = \begin{pmatrix} 0 & -1 \\ 1 & 0 \end{pmatrix} \begin{pmatrix} 2 \cos t \\ 2 \sin t \end{pmatrix}$$

2 \rightarrow State equation for trajectory reference

$$\bar{x}(t) = \begin{pmatrix} -2 \sin t \\ 2 \cos t \end{pmatrix} = \begin{pmatrix} 0 & -1 \\ 1 & 0 \end{pmatrix} \begin{pmatrix} 2 \cos t \\ 2 \sin t \end{pmatrix} + \begin{pmatrix} 0 \\ 1 \end{pmatrix} \bar{u} \rightarrow \bar{u} = 0$$

3 \rightarrow Define $\tilde{x}(t) = x(t) - x_{ref}(t) \quad \tilde{u}(t) = u(t) - \bar{u} \rightarrow$ But $\bar{u} = 0$.

$$\begin{pmatrix} \tilde{x}_1(t) \\ \tilde{x}_2(t) \end{pmatrix} = \begin{pmatrix} x_1(t) - \bar{x}_1(t) \\ x_2(t) - \bar{x}_2(t) \end{pmatrix}$$

4 → Design a Luenberger observer.

Define $\dot{X}'(t) = \tilde{X}(t) - X(t)$

where $\tilde{X}(t) \rightarrow$ is the estimation of the state and
 $X'(t) \rightarrow$ is the error in the estimation.

→ To design a Luenberger observer we must check
if the system is observable.

$$\text{rank } (C) = \begin{pmatrix} 1 & 0 \\ 0 & 1 \end{pmatrix} = 2 \quad A = \begin{pmatrix} 0 & -1 \\ 1 & 0 \end{pmatrix} \quad C = \begin{pmatrix} 1 & 0 \end{pmatrix}$$

→ The system is observable, then we can compute
a Luenberger observer.

we define $\rightarrow L = \begin{pmatrix} L_1 \\ L_2 \end{pmatrix}$

and $\dot{X}' = (A - LC)X'(t)$. $LC = \begin{pmatrix} L_1 \\ L_2 \end{pmatrix} \begin{pmatrix} 1 & 0 \end{pmatrix} = \begin{pmatrix} L_1 & 0 \\ L_2 & 0 \end{pmatrix}$

$$\rightarrow \dot{X}'(t) = \begin{vmatrix} -L_1 & -1 \\ 1-L_2 & 0 \end{vmatrix} X'(t) \quad M = (A - LC) = \begin{vmatrix} -L_1 & -1 \\ 1-L_2 & 0 \end{vmatrix}$$

→ Compute $\det(M - \lambda I)$ → to set L_1, L_2 that $(A - LC)$ has
eigenvalues with negative real parts

$$\det(M - \lambda I) = \begin{vmatrix} -L_1 - \lambda & -1 \\ 1 - L_2 & -\lambda \end{vmatrix} = \lambda(L_1 + \lambda) + (1 - L_2) = 0$$
$$= \lambda^2 + L_1\lambda + (1 - L_2) = 0$$

→ we would like to have $\lambda = -1 \rightarrow (-1+1)^2 = 1^2 + 2(-1) + 1 = 0$

$$\therefore L_1 = 2 \quad (1 - L_2) = 1 \rightarrow L_2 = 0$$

Finally $L = (2, 0)$

5 → Design state feedback that $\tilde{X}(t) \rightarrow 0$ as $t \rightarrow \infty$

$$\tilde{\tilde{X}}(t) = \begin{pmatrix} 0 & -1 \\ 1 & 0 \end{pmatrix} \tilde{X}(t) + \begin{pmatrix} 0 \\ 1 \end{pmatrix} \tilde{U}(t) \quad \tilde{X}(0) = \begin{pmatrix} 1 \\ 0 \end{pmatrix}$$

→ we would like to design a state feedback

$$U(t) = \bar{U}(t) + K\hat{X}(t) \rightarrow \text{where } \hat{X}(t) \text{ is the estimation of } X(t)$$
$$\bar{U} = 0 \rightarrow U(t) = K\hat{X}(t) \text{ the state.}$$

$$\dot{\hat{X}}(t) = A\hat{X} + BU(t)$$

$$\dot{\hat{X}}(t) = A\hat{X} + BK\hat{X}(t) \rightarrow \text{we have defined}$$

$$\rightarrow \dot{\hat{X}}(t) = A\hat{X} + BK\hat{X} - BKX'(t) \quad X'(t) = \hat{X}(t) - \tilde{X}(t)$$

$$\dot{\hat{X}}(t) = (A+BK)\hat{X} - BKX'(t) \quad \tilde{X}(t) = \hat{X}(t) - X'(t)$$

where $X'(t) \rightarrow \text{Error in estimation}$

First, we have to check if the system is

controllable to be able to implement $U(t) = K\hat{X}(t)$

→ $\text{rank}(B \ AB) = \begin{pmatrix} 0 & -1 \\ 1 & 0 \end{pmatrix} = 2 = n \rightarrow \text{order of the system}$
then we can design
the state feedback.

Define: $BK = \begin{pmatrix} 0 \\ 1 \end{pmatrix} (K_1 \ K_2) = \begin{pmatrix} 0 & 0 \\ K_1 & K_2 \end{pmatrix}$

$$\rightarrow (A+BK) = \begin{pmatrix} 0 & -1 \\ 1+K_1 & K_2 \end{pmatrix} \rightarrow \det((A+BK)-\lambda I) = \begin{vmatrix} -\lambda & -1 \\ 1+K_1 & K_2-\lambda \end{vmatrix}$$
$$= \lambda^2 - K_2\lambda + (1+K_1)$$

∴ we want negative eigenvalues $\lambda = -1, \lambda = -1$

$$(\lambda+1)^2 = 0 \rightarrow \lambda^2 + 2\lambda + 1 = 0,$$

$$\rightarrow -K_2 = 2, K_2 = -2$$

$$K_1+1 = 1, K_1 = 0 \rightarrow K = (0, -2)$$

Finally → $\dot{\hat{X}}(t) = (A+BK)\hat{X} - BKX'(t) = \begin{pmatrix} 0 & -1 \\ 1 & -2 \end{pmatrix} \hat{X}(t) - \begin{pmatrix} 0 & 0 \\ 0 & -2 \end{pmatrix} X'(t)$

For state feedback design.

$$\rightarrow \dot{X}'(t) = (A-LC)\dot{X}'(t) = \begin{pmatrix} -2 & -1 \\ 1 & 0 \end{pmatrix} \dot{X}'(t)$$

6 -> Create output feedback \rightarrow combine the Luenberger observer and the state feedback design.

i) state feedback design.

$$\dot{\tilde{X}}(t) = (A + BK)\tilde{X} - BKX'(t) \Rightarrow X'(t): \text{estimation error.}$$

(i) From observer design

$$\dot{X}'(t) = (A - LC)X'(t)$$

\rightarrow To combine we define: $X_\alpha(t) = (\tilde{X}(t), X'(t))'$

where $\dot{X}(t) = X(t) = \tilde{X}(t) \quad \text{and} \quad X(t) = \tilde{X}(t) - \underbrace{X'(t)}_{\text{reference}}$

state estimation

Then:
$$\begin{pmatrix} \dot{\tilde{X}}(t) \\ \dot{X}'(t) \end{pmatrix} = \begin{pmatrix} (A + BK)\tilde{X}(t) - BKX'(t) \\ (A - LC)X'(t) \end{pmatrix}$$

$$\begin{pmatrix} \dot{\tilde{X}}(t) \\ \dot{X}'(t) \end{pmatrix} = \begin{pmatrix} (A + BK) & -BK \\ 0 & (A - LC) \end{pmatrix} \begin{pmatrix} \tilde{X}(t) \\ X'(t) \end{pmatrix}$$

where $(A + BK) = \begin{pmatrix} 0 & -1 \\ 1 & -2 \end{pmatrix}, BK = \begin{pmatrix} 0 & 0 \\ 0 & -2 \end{pmatrix}$

$$A - LC = \begin{pmatrix} -2 & -1 \\ 1 & 0 \end{pmatrix}$$

Given that $(A + BK)$ and $(A - LC)$ have negative eigenvalues
 \rightarrow the system is exponentially stable

4 → Consider the (nonlinear) state equation for a simple racing car (moving at constant velocity 1 m/s) given by:

$$\dot{P}_x(t) = \cos \theta(t)$$

$$\dot{P}_y(t) = \sin \theta(t)$$

$$\dot{\theta}(t) = \tan u(t)$$

We aim to design state feedback control that allows the car to follow a racing track depicted below.

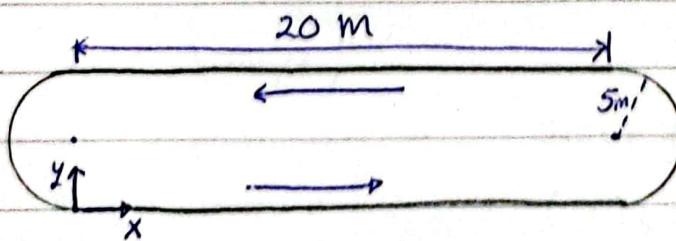
Let:

$$u(t) = M_{\text{linear}}(P_y(t), \theta(t)) = K_1 \begin{pmatrix} P_y(t) \\ \theta(t) \end{pmatrix}$$

be a controller that allows the car to follow a (racing track) straight line originating from the origin along x -axis, and

$$\begin{aligned} \text{and } u(t) &= M_{\text{circular}}(P_x(t), P_y(t), G(t), P_{c,x}, P_{c,y}, r_0) \\ &= \arctan\left(\frac{1}{r_0}\right) + K_2 \left(\sqrt{(P_x(t) - P_{c,x})^2 + (P_y(t) - P_{c,y})^2 - r_0^2} \right. \\ &\quad \left. \theta(t) - (\pi/2 + \arctan\left(\frac{P_y(t) - P_{c,y}}{P_x(t) - P_{c,x}}\right)) \right) \end{aligned}$$

be a controller that allows the car to track a circular track centered at $(P_{c,x}, P_{c,y})$ with radius r_0 . use both controllers M_{linear} , M_{circular} to design state feedback control to maneuver the car along the below racing track → (No need to compute K_1, K_2).



First, we will get concentrated on the straight line state feedback control design.

$$\dot{P}_x(t) = \cos \theta(t)$$

$$P_x(0) = P_y(0) = 0 \quad \theta(0) = 0$$

$$\dot{P}_y(t) = \sin \theta(t)$$

$$\dot{\theta}(t) = \tan u(t)$$

1 → Design a Reference trajectory to follow the line:

$$\begin{cases} P_y^r(t) = 0 \\ \theta^r(t) = 0 \end{cases}$$

Define the

state vector $\rightarrow X(t) = \begin{pmatrix} P_y(t) \\ \theta(t) \end{pmatrix}$

2 → Create new state equation

$$\begin{aligned} \tilde{P}_y(t) &= P_y(t) - P_y^r(t) \\ \tilde{\theta}(t) &= \theta(t) - \theta^r(t) \end{aligned} \quad \left\{ \rightarrow \tilde{X}(t) = X(t) - \underbrace{X_{ref}(t)}_{\text{Reference}} \right.$$

Trajectory

$$\dot{\tilde{P}}_y(t) = \sin \theta(t) - P_y^r(t) = \sin \theta(t)$$

$$\dot{\tilde{\theta}}(t) = \tan u(t) - \theta^r(t) = \tan u(t)$$

3 → Linearization around $\tilde{\theta}=0, \tilde{P}_y(t)=0 \rightarrow \tilde{X}(t)=0$

$$\dot{\tilde{X}}(t) = \begin{pmatrix} \sin \theta(t) \\ \tan u(t) \end{pmatrix} \rightarrow \nabla_x f(x, u) = \begin{vmatrix} 0 & \cos \theta(t) \\ 0 & 0 \end{vmatrix}$$

$$f(x, u) \qquad \qquad \nabla_u f(x, u) = \begin{vmatrix} 0 \\ \frac{1}{1+u^2} \end{vmatrix}$$

$$f(\bar{x}, \bar{u}) = \begin{pmatrix} 0 \\ \tan \bar{u} \end{pmatrix} = 0 \rightarrow \text{Implies that } \bar{u} = 0$$

After Linearization: $\dot{\tilde{X}}(t) = \nabla_x f(\bar{x}, \bar{u})(\tilde{x} - \bar{x}) + \nabla_u f(\bar{x}, \bar{u})(u - \bar{u})$

$$\dot{\tilde{X}}(t) = \begin{pmatrix} 0 & 1 \\ 0 & 0 \end{pmatrix} \tilde{X}(t) + \begin{pmatrix} 0 \\ 1 \end{pmatrix} u(t)$$

4 → Check if the linearized system is controllable.

$$\text{rank } (B \ AB) = \begin{pmatrix} 0 & 1 \\ 1 & 0 \end{pmatrix} = 2 \rightarrow \text{Then the system is controllable.}$$

We can design a state feedback control that stabilizes the system $\rightarrow U(t) = K_1 \tilde{x}(t)$

5 → State feedback design \rightarrow Define $K_1 = (K_a, K_b)$

$$\dot{\tilde{x}}(t) = \begin{pmatrix} 0 & 1 \\ 0 & 0 \end{pmatrix} \tilde{x}(t) + \begin{pmatrix} 0 \\ 1 \end{pmatrix} (K_a, K_b) \tilde{x}(t)$$

$$\dot{\tilde{x}}(t) = \begin{pmatrix} 0 & 1 \\ 0 & 0 \end{pmatrix} \tilde{x}(t) + \begin{pmatrix} 0 & 0 \\ K_a & K_b \end{pmatrix} \tilde{x}(t) = \begin{pmatrix} 0 & 1 \\ K_a & K_b \end{pmatrix} \tilde{x}(t)$$

This controller is used starting from the origin until reach the 20m of the final stage of the straight line.

Second \rightarrow we need to maneuver the car in a circular trajectory.

\rightarrow we have: $\dot{P}_x(t) = \cos \theta(t)$ we will use polar
 $\dot{P}_y(t) = \sin \theta(t)$ coordinates to rewrite
 $\dot{\theta}(t) = \tan u(t)$ the state equation.

①

\rightarrow Define $\tilde{r}(t) = \sqrt{\bar{P}_x^2(t) + \bar{P}_y^2(t)} - r_0$ where $\bar{P}_x = P_x(t) - P_{Cx}$
 $\tilde{\theta}(t) = \theta(t) - \left(\frac{\pi}{2} + \arctan\left(\frac{\bar{P}_y}{\bar{P}_x}\right)\right)$ Reference
 $\bar{P}_y = P_y(t) - P_{Cy}$
 (P_{Cx}, P_{Cy})

2 → Define new state equation

$$\dot{\tilde{r}}(t) = -\sin \tilde{\theta}(t)$$

Coordinates of the center of the circle.

$$\dot{\tilde{\theta}}(t) = \frac{-1}{\tilde{r}(t) + r_0} \cos \tilde{\theta}(t) + \tan u(t)$$

$$\rightarrow \dot{x}(t) = \begin{pmatrix} f(t) \\ \theta(t) \end{pmatrix}$$

$$\tilde{x}(t) = x(t) - x_{ref}(t) \rightarrow \text{where } x_{ref}(t) = \begin{pmatrix} r_0 \\ \theta_r \end{pmatrix}$$

$$\dot{\tilde{x}}(t) = \begin{pmatrix} -\sin \tilde{\theta}(t) \\ -\frac{1}{\tilde{r}(t)+r_0} (\cos \tilde{\theta}(t)) + \tan u(t) \end{pmatrix} \left\{ \begin{array}{l} f(x, u) \\ \tilde{r}(t) \end{array} \right.$$

3 → Linearize → we need that $\tilde{r}(t)=0$ and $\tilde{\theta}(t)=0$

$$\tilde{x}(t) = (0, 0)$$

$$\rightarrow \nabla_x f(\bar{x}, \bar{u}) = \begin{pmatrix} 0 & -\cos \tilde{\theta}(t) \\ (\tilde{r}+r_0)^2 \cos \tilde{\theta}(t) & (\tilde{r}+r_0) \sin \tilde{\theta}(t) \end{pmatrix} \Big|_{\begin{array}{l} \bar{x} = (0, 0) \\ \bar{u} \end{array}}$$

$$\nabla_x f(\bar{x}, \bar{u}) = \begin{pmatrix} 0 & -1 \\ 1/r_0 & 0 \end{pmatrix} \quad f(\bar{x}, \bar{u}) = \begin{pmatrix} 0 \\ -1 + \tan \bar{u} \end{pmatrix}$$

$$\nabla_u f(\bar{x}, \bar{u}) = \begin{pmatrix} 0 \\ \frac{1}{1+u^2} \end{pmatrix} \Big|_{u=\bar{u}} \quad \bar{u} = \arctan \left(\frac{1}{\tilde{r}+r_0} \right)$$

$$\nabla_u f(\bar{x}, \bar{u}) = \begin{pmatrix} 0 \\ \frac{(\tilde{r}+r_0)^2}{1+(\tilde{r}+r_0)^2} \end{pmatrix} \Big|_{\begin{array}{l} x=\bar{x} \\ u=\bar{u} \end{array}} = \begin{pmatrix} 0 \\ \frac{r_0^2}{1+r_0^2} \end{pmatrix} \quad \boxed{\bar{u} = \arctan \left(\frac{1}{r_0} \right)}$$

$$\text{Linearization} \rightarrow \dot{\tilde{x}}(t) = \nabla_x f(\bar{x}, \bar{u})(x - \bar{x}) + \nabla_u f(\bar{x}, \bar{u})(u - \bar{u})$$

$$\dot{\tilde{x}}(t) = \begin{pmatrix} 0 & -1 \\ 1/r_0^2 & 0 \end{pmatrix} \tilde{x}(t) + \begin{pmatrix} 0 \\ \frac{r_0^2}{1+r_0^2} \end{pmatrix} \tilde{u}(t) \rightarrow \tilde{u} = u(t) - \bar{u}$$

9 → we can check that the system is controllable

$$\text{rank}(B A B) = \begin{pmatrix} 0 & -\frac{r_0^2}{1+r_0^2} \\ \frac{r_0^2}{1+r_0^2} & 0 \end{pmatrix} = 2 \rightarrow \text{System is controllable.}$$

\rightarrow Then we can design a state feedback control

$$\tilde{U}(t) = K_2 (X(t) - X_{ref}) + \arctan\left(\frac{1}{r_0}\right)$$

$$\tilde{U}(t) = K_2 \tilde{X}(t) + \bar{U}$$

Define $K_2 = (K_c, K_d)$

$$\dot{\tilde{X}} = A \tilde{X}(t) + B \tilde{U}(t) = A \tilde{X}(t) + B K_2 \tilde{X}(t) + B \bar{U}$$

$$\dot{\tilde{X}} = (A + B K_2) \tilde{X}(t) + B \bar{U}$$

$$\dot{\tilde{X}}(t) = \begin{pmatrix} 0 & -1 \\ \frac{1}{r_0^2} & 0 \end{pmatrix} \tilde{X}(t) + \begin{pmatrix} 0 \\ \frac{r_0^2}{1+r_0^2} \end{pmatrix} (K_c, K_d) \tilde{X}(t) + \begin{pmatrix} 0 \\ \frac{r_0^2}{1+r_0^2} \end{pmatrix} \bar{U}$$

$$\dot{\tilde{X}}(t) = \begin{pmatrix} 0 & -1 \\ \frac{K_c}{1+r_0^2} & \frac{r_0^2 K_d}{1+r_0^2} \end{pmatrix} \tilde{X}(t) + \begin{pmatrix} 0 \\ \frac{r_0^2}{1+r_0^2} \end{pmatrix} \bar{U}$$

where $\rightarrow \bar{U} = \arctan\left(\frac{1}{r_0}\right)$ $\tilde{X}(t) = X(t) - X_{ref}(t)$

$$\tilde{X}(t) = \begin{pmatrix} r(t) - r_0 \\ \theta(t) - \left(\frac{\pi}{2} + \arctan\left(\frac{p_x}{p_y}\right)\right) \end{pmatrix}$$

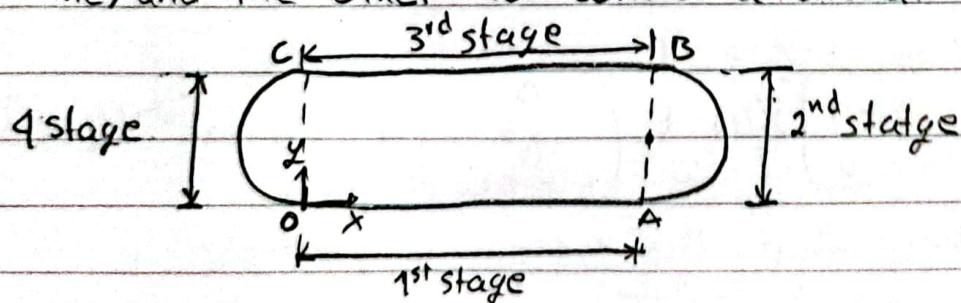
\rightarrow For the circular trajectories

The system developed govern the motion, where $r_0 = 5 \text{ m.}$

$$\bar{p}_x = p_x(t) - p_{c,x}$$

$$\bar{p}_y = p_y(t) - p_{c,y}$$

we have 2 controllers \rightarrow one designed to follow a straight line, and the other for control a circular motion.



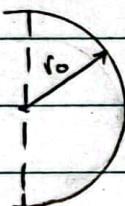
If the car starts in the origin of our coordinate system we devide the task in 4 stages, 2 should be for straightline control motion and the circular trajectories

should be used the circular control design.

→ For stage 1 and 3 we have to make the race car to follow an straight line, the distance is 20m.

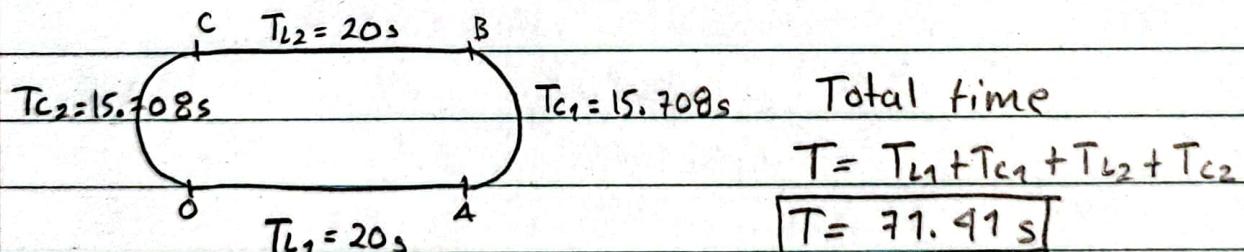
The car has constant velocity of 1m/s, then each stage takes 20s to be completed. → $T_L = 20s$

→ For the 2nd and 4th stage we have 2 half-circle trajectory $r_0 = 5m$, The distance in this curve is → $D = \frac{2\pi r_0}{2} = \pi r_0 = 5\pi m$. The car should travel distance D .



$D = \pi r_0 \rightarrow$ Circunference of the half-circle.

Given that the car is moving at constant velocity the car make the circular trajectory in the time $T_c = 15.708s$



To apply the controllers we can use the time:

$$u(t) = \begin{cases} K_1 \tilde{x}(t) = K_1 \begin{pmatrix} \tilde{p}_y(t) \\ \tilde{\theta}(t) \end{pmatrix} & \rightarrow \text{if } 0 \leq t \leq 20 \text{ (stage 1)} \\ & 35.70 \leq t \leq 55.70 \text{ (stage 3)} \\ K_2 \tilde{x}(t) + \bar{u} = K_2 \left(\begin{pmatrix} r(t) - r_0 \\ \theta(t) - \left(\frac{\pi}{2} + \tan^{-1} \left(\frac{\tilde{p}_y}{\tilde{p}_x} \right) \right) \end{pmatrix} \right) + \bar{u} & \end{cases}$$

If $20 < t \leq 35.70$ (stage 2)

$55.70 < t \leq 71.41s$ (stage 4)