

Design Review
Feb. 23, 2018

What is Hyperloop?

By: Rachel, Raymond



Introduction

Competition Rules

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Maglev

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Stability & Braking

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PCB & Sensors

Project Schedule

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Levitation Competition Rules

By: Rachel

- There-and-back race on outdoor test track
- Pod must **levitate** for entire duration of run
- 5 foot minimum pod length
- **Fastest pod wins!**



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Updated Key Specs

By: Raymond

- Accepted into final competition in July at SpaceX
 - Competing directly against San Jose State University in Levitation Competition



Team Name	University
UCSB Hyperloop	University of California, Santa Barbara
Spartan Hyperloop	San Jose State University

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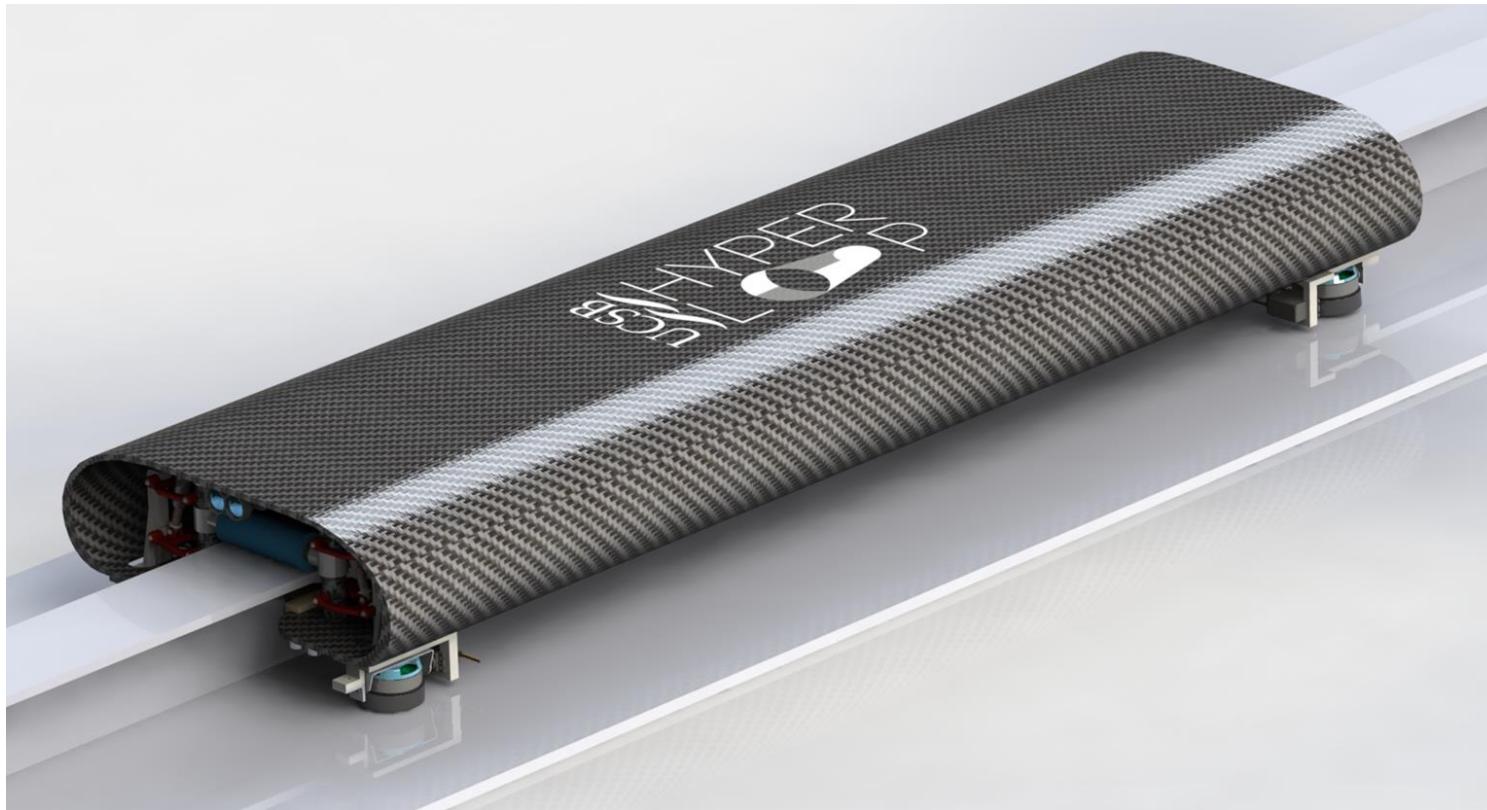
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UCSB Hyperloop III

By: Nate



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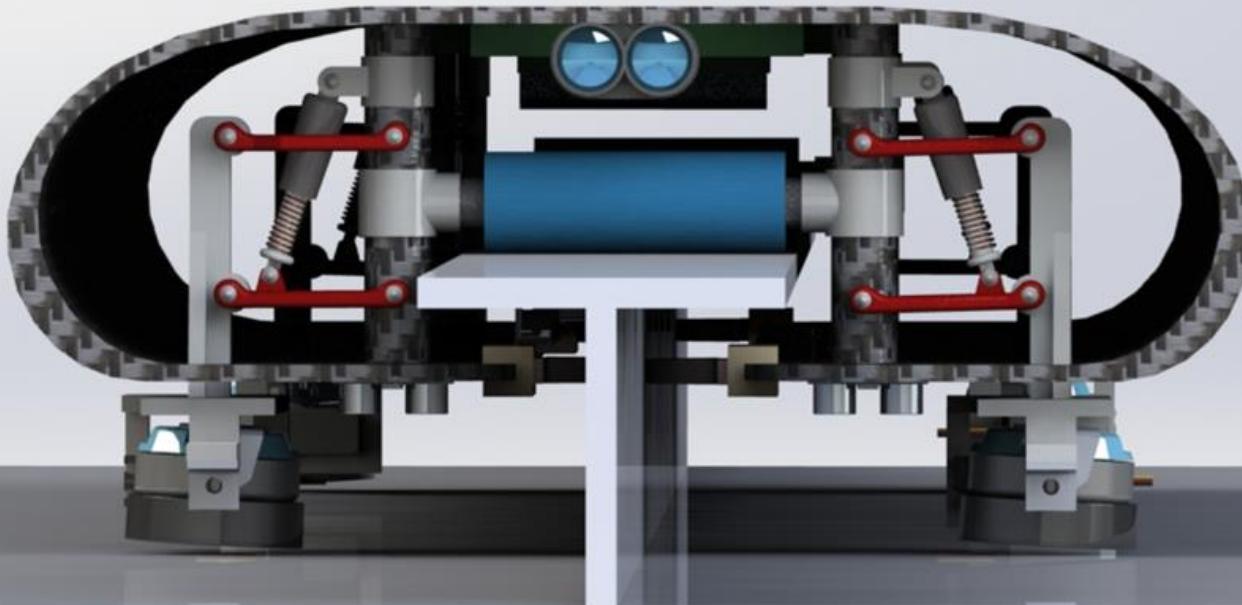
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Pod Overview

By: Nate



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Subsystem Weight Allocation

By: Nate

Subsystem	Mass (kg)	Weight (lb)
Maglev	1.52	3.35
Structure	1.07	2.5
Stability/Braking	0.81	1.78
Electronics	0.62	1.14
Total	3.98 kg	8.77 lb

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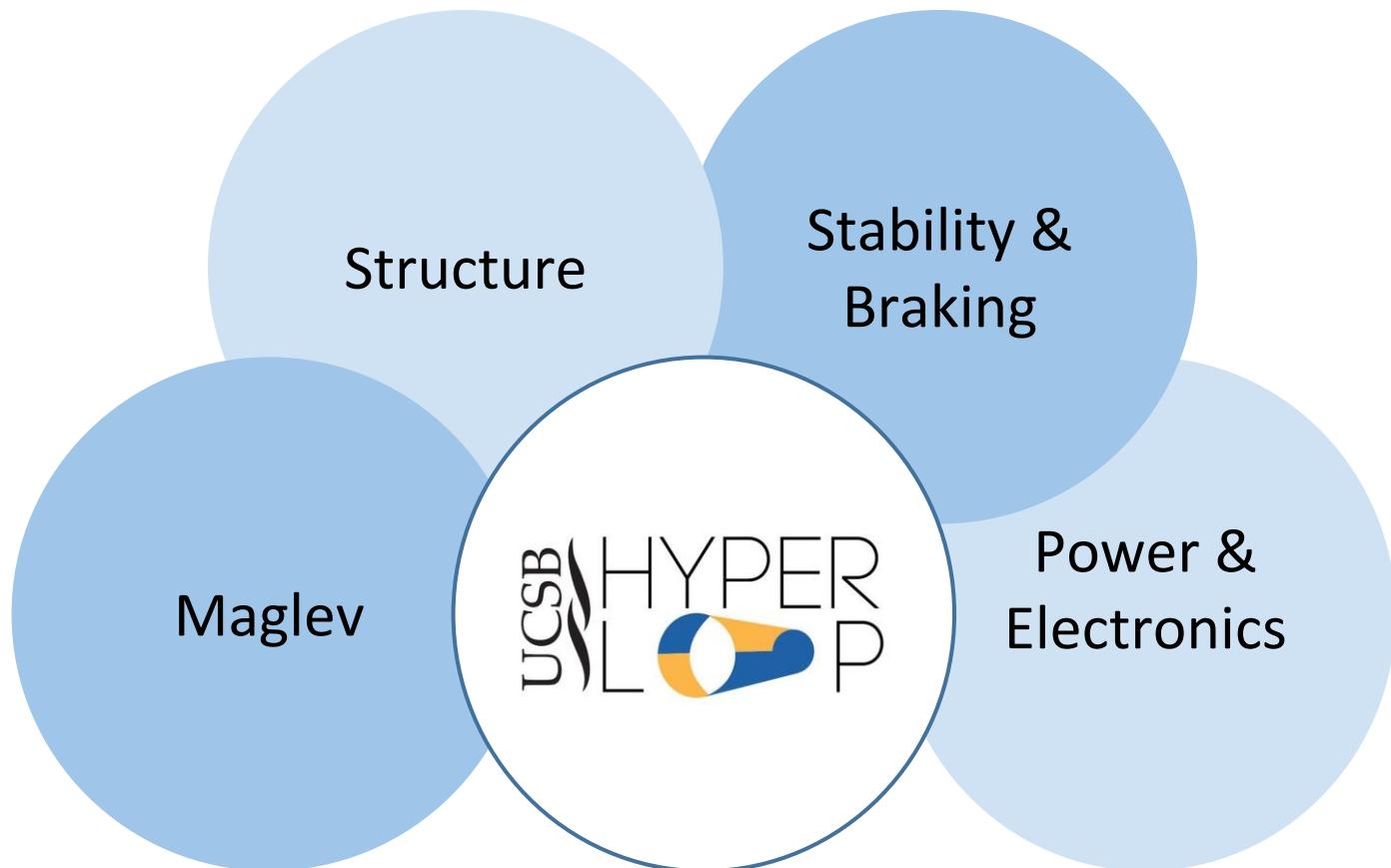
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System Design

By: Nate



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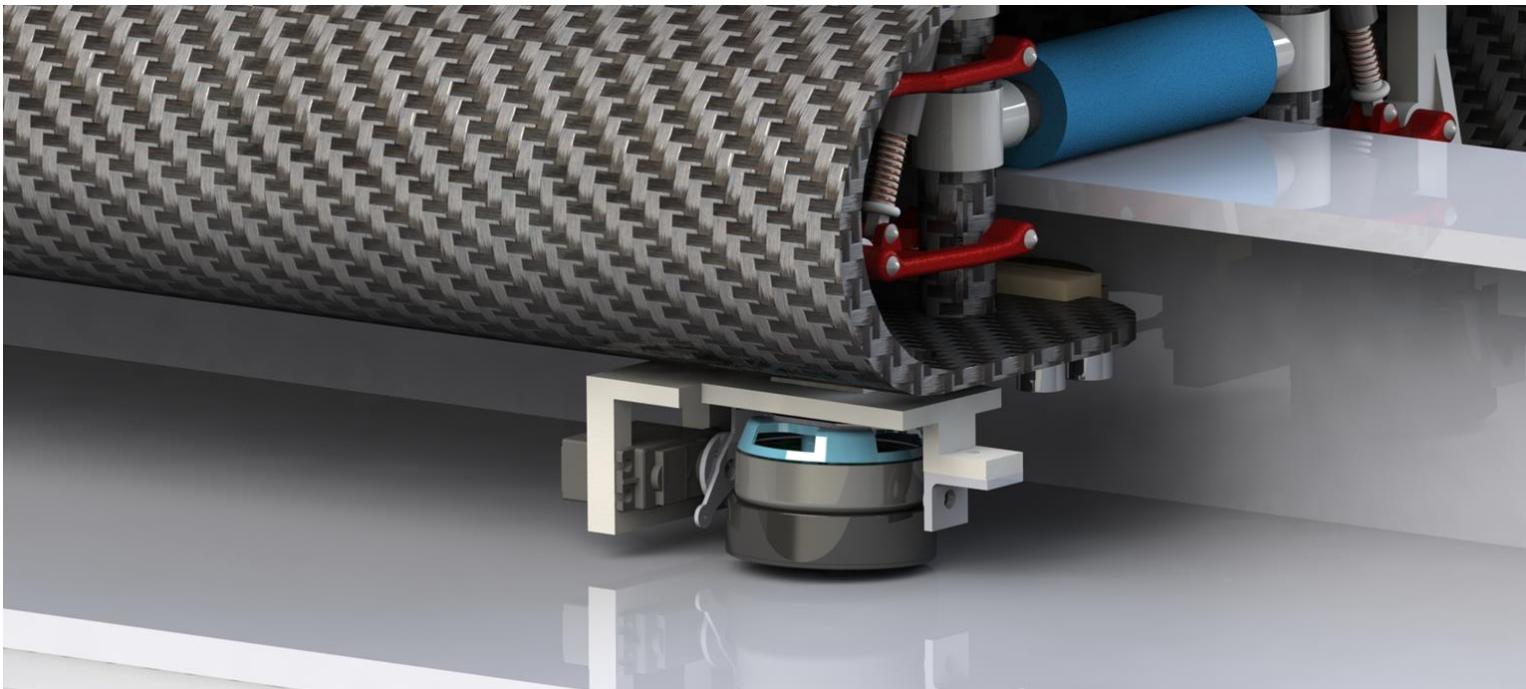
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By: Andrew



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MagLev Propulsion

By: Andrew

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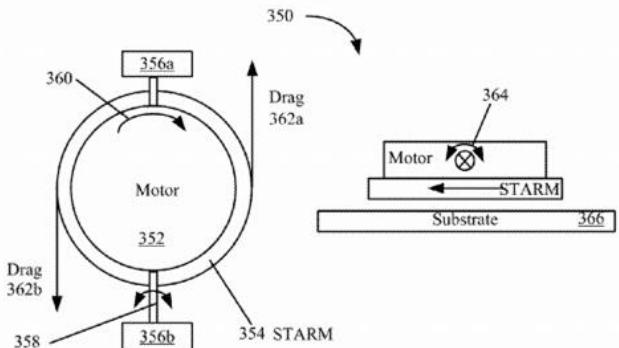


Figure 30A

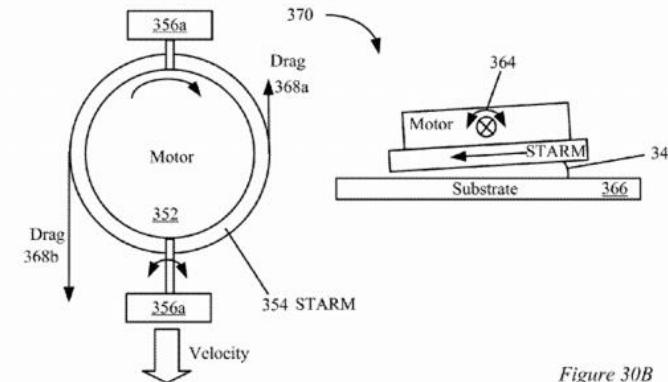
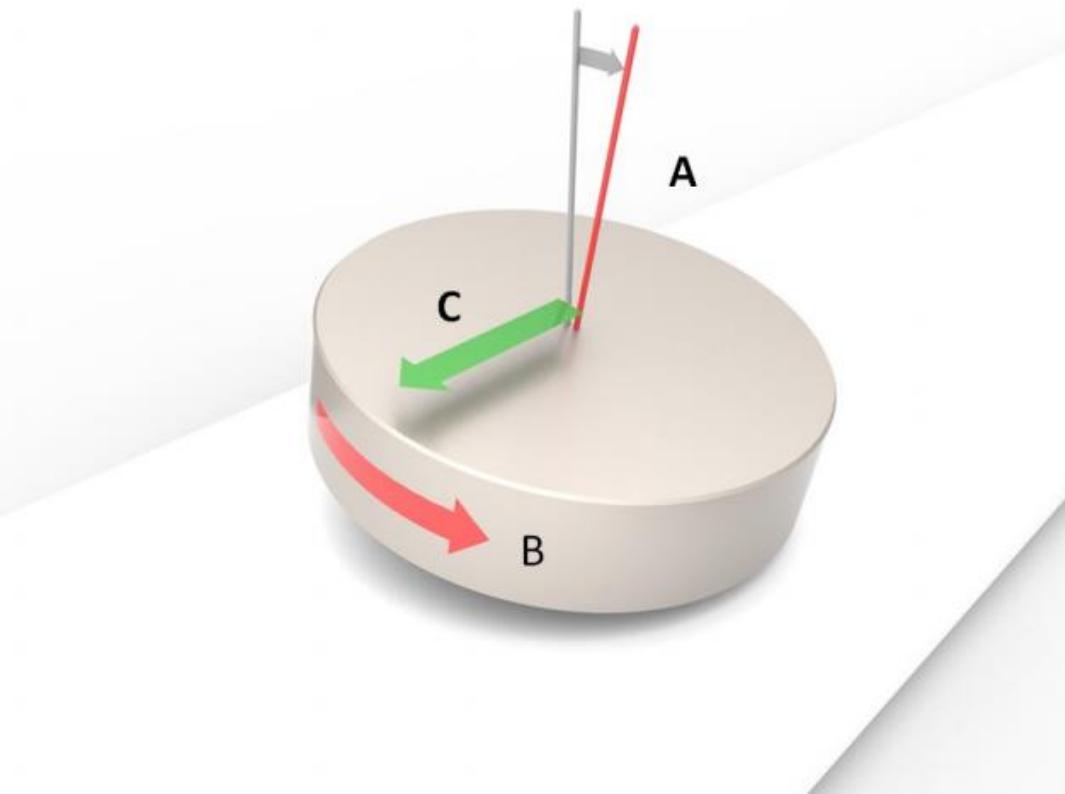


Figure 30B

MagLev Propulsion

By: Andrew



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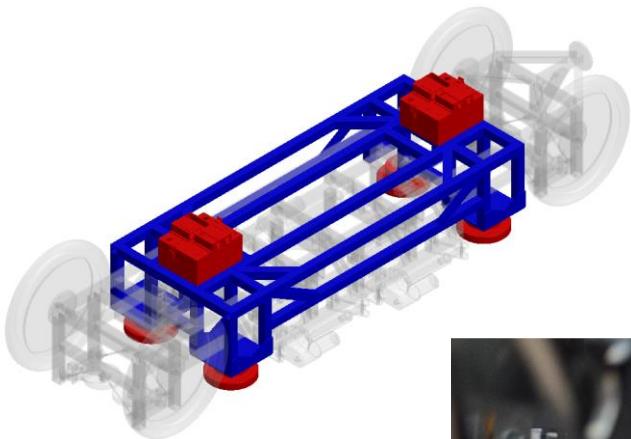
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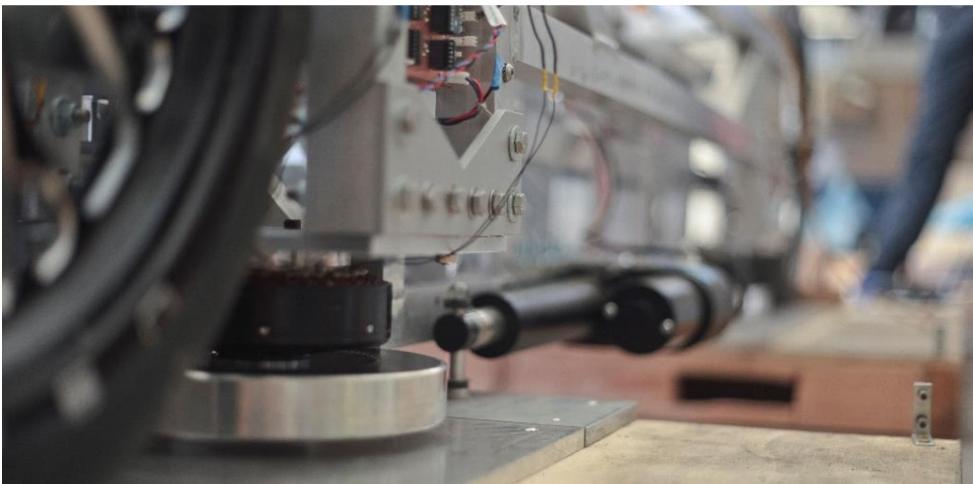
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Last Year: MagLev

By: Nate



- Hybrid Payload System
- Large Arx Pax HE3.0 Maglev engines
- Heavy and power hungry
- Lacked suspension system to dampen oscillations created by MagLev engines



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Levitation Concept Validation

By: Nate

- **Air Bearing** prototype
 - Developed as potential alternative
 - Didn't achieve full levitation
 - Requires separate propulsion system
- **Maglev** prototype
 - Scaled down using **HDK** motors
 - Proved viability of tilt propulsion
 - Showed promising acceleration



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Levitation Concept Development

By: Nate



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By: Nate

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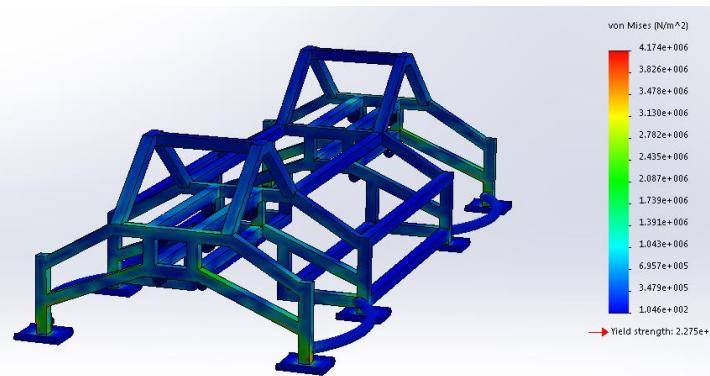
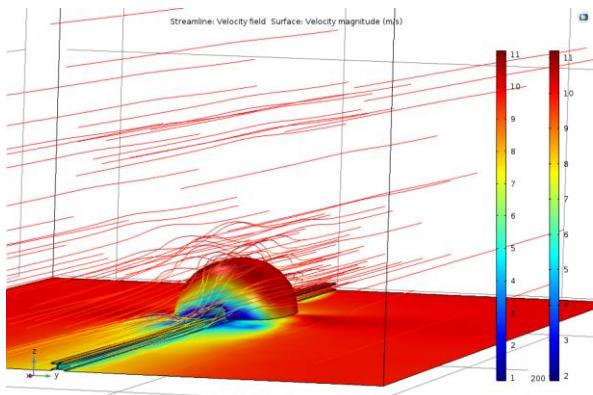
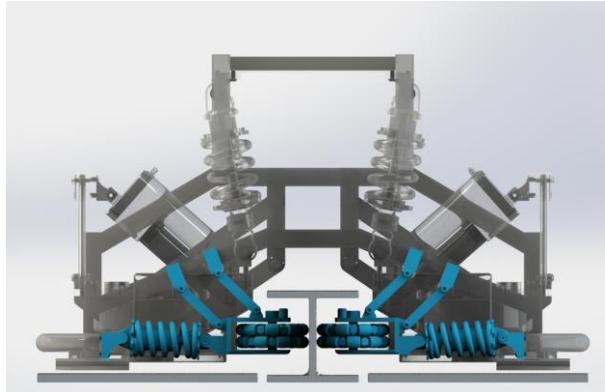
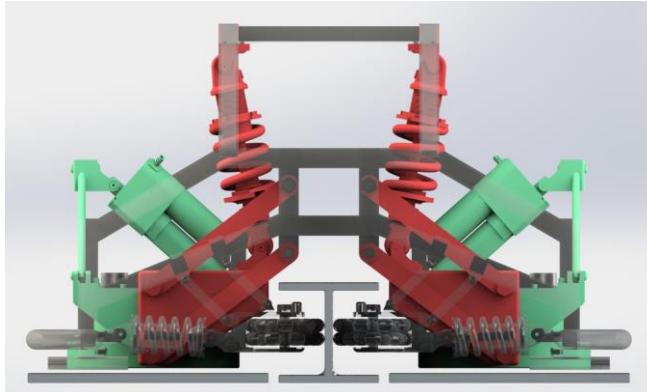
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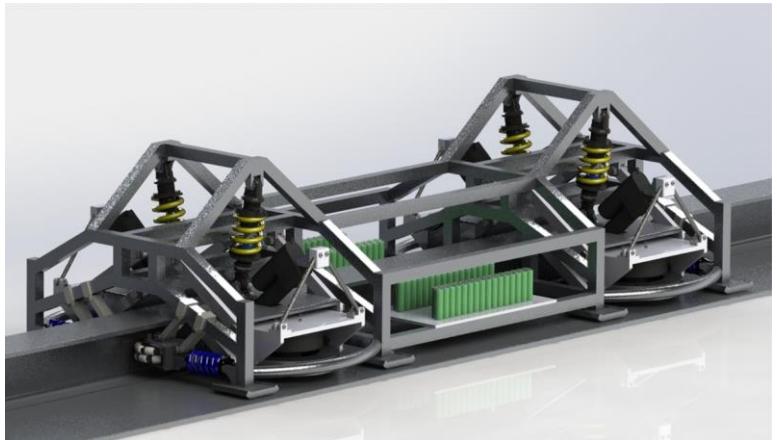
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Levitation Concept Development

By: Nate



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Levitation Concept Assessment

By: Nate

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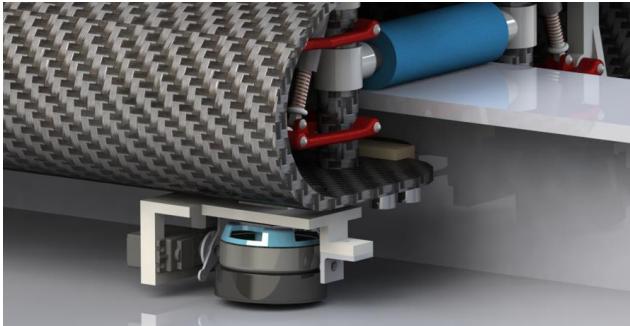
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Design Criteria	HDK (small maglev) Pod	HE3.0 (large maglev) Pod
Proven Concept (0-15)	10.8	5.2
Simplicity of Design (0-10)	7.2	3.43
Cost (0-5)	4.4	2.8
Modifiability (0-10)	9	3.6
Timeline (0-20)	18.4	9.4
Weight Limit (0-25)	8.75	19.17
Acceleration (0-25)	10.8	18
Total	69.35	61.60

Maglev

By: Nate



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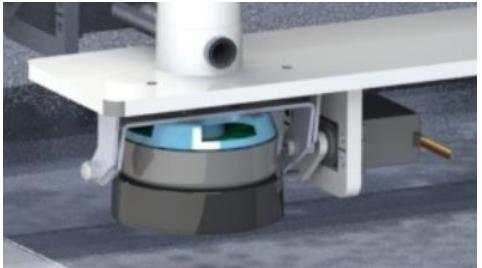
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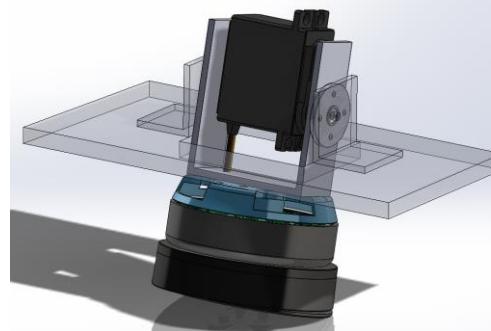
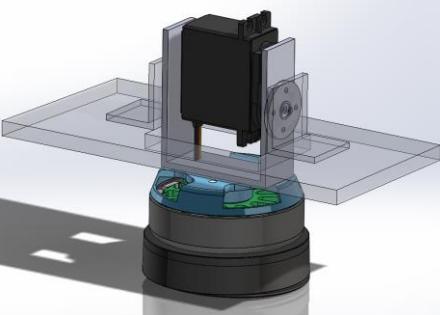
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Tilt Mechanism

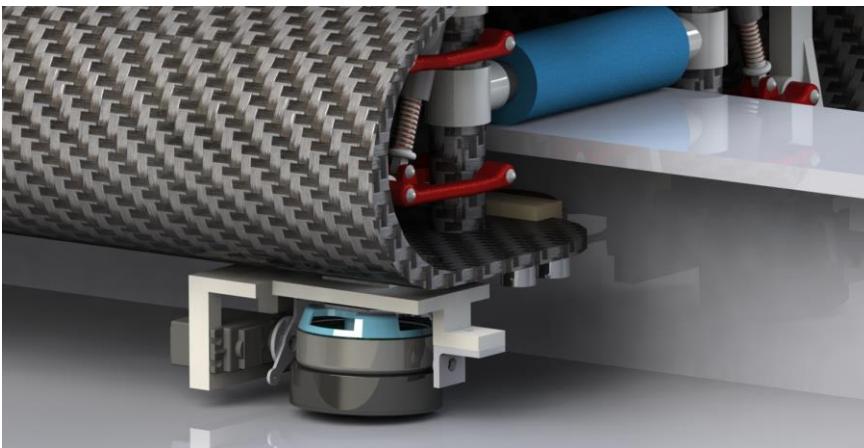
By: Andrew



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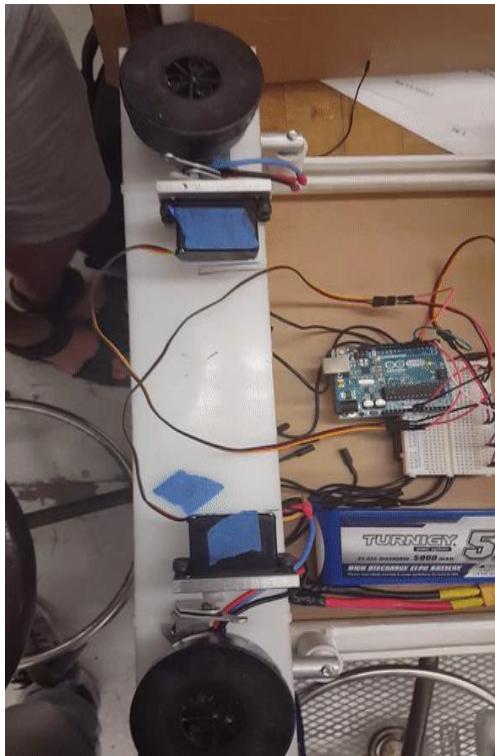
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Maglev Prototype Progress

By: David

- Scaled down prototype 4 maglev engines and 4 servos
- LiPo battery powers servos and engines
- ESC controls servos and engines using PWM signals from the microcontroller



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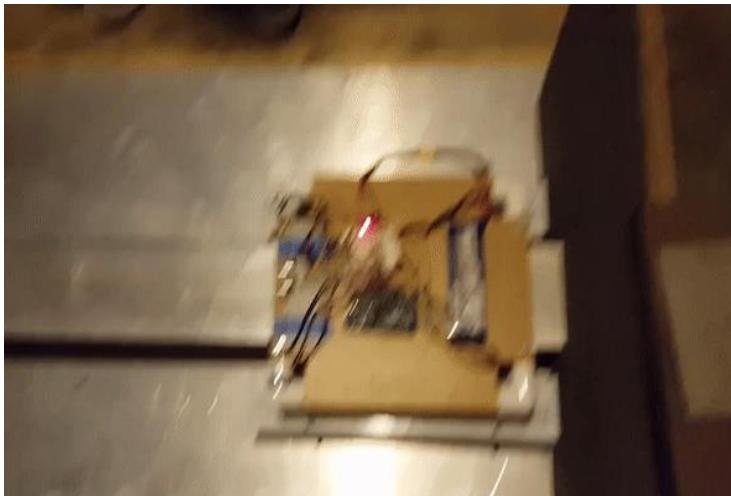
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Maglev Prototype Progress

By: Alex

- Used Arduino to test prototype
- Controls Testing
 - Timing profile
 - Sensor feedback using ultrasonic sensor
- Currently integrating PCB to replace the Arduino



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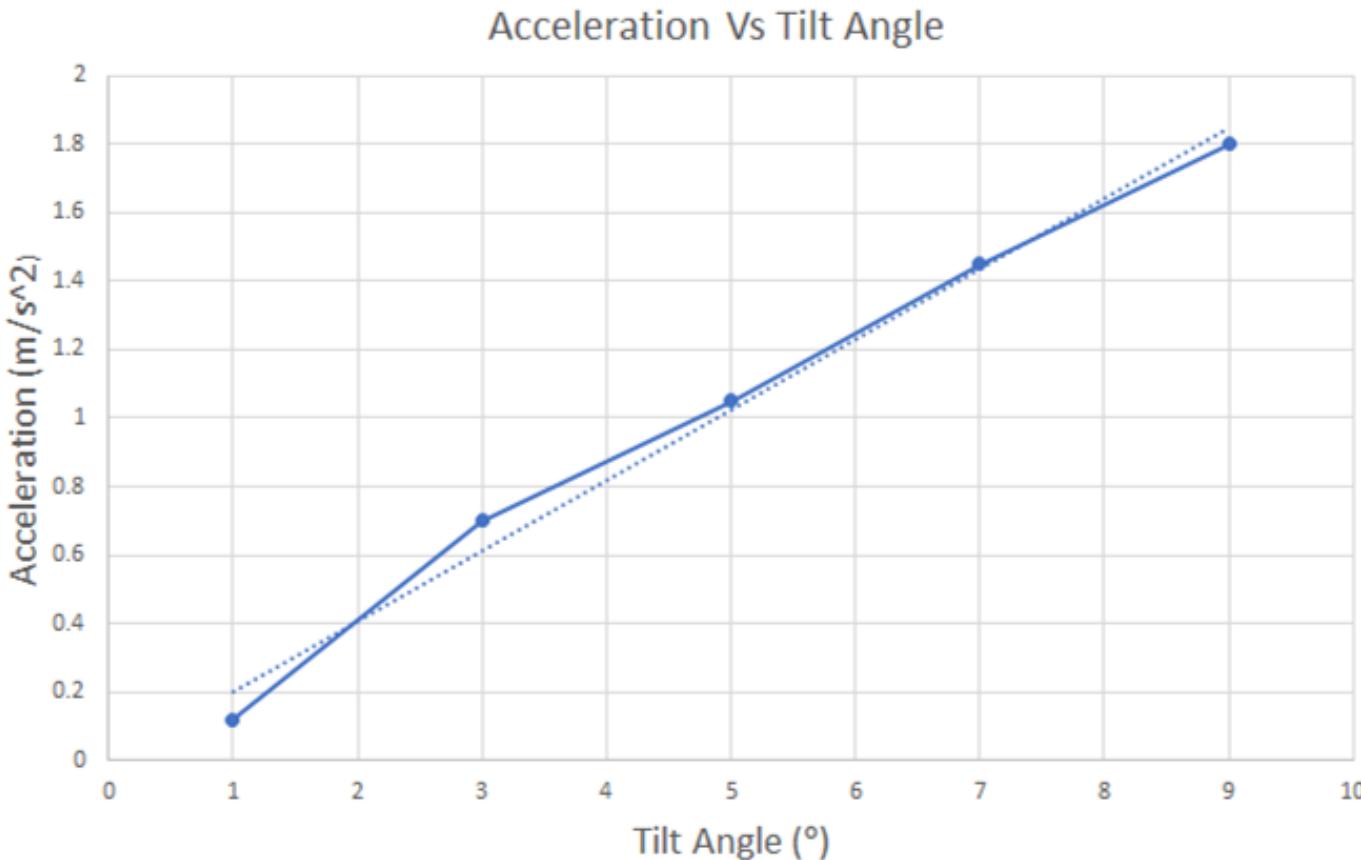
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Maglev Motor Testing

By: Alex



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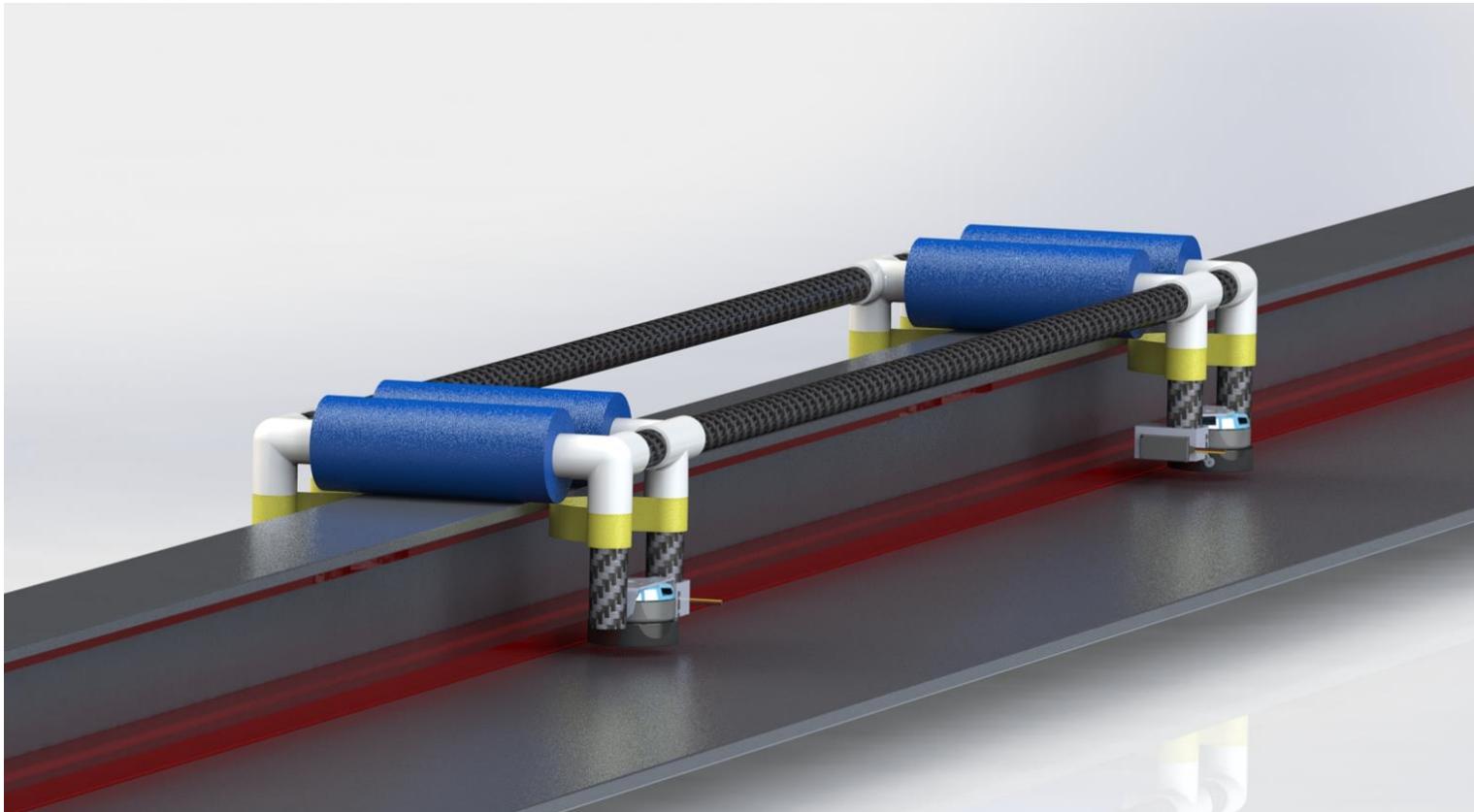
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Maglev Prototype Progress

By: Edsel



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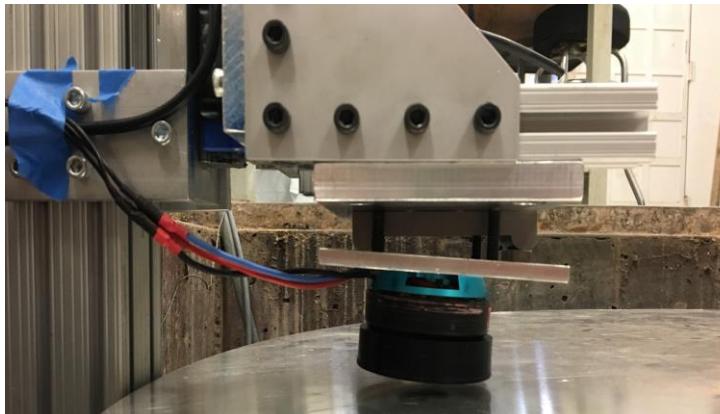
Translational Rig Testing

By: Xingchen, Jack D.

- Aluminum disk simulates translational motion
- Actuator adjusts height
- Sine bar adjusts angle

Updates:

- Translational Test Rig Ready
- Mini-maglev testing in progress
- More extensive testing to come



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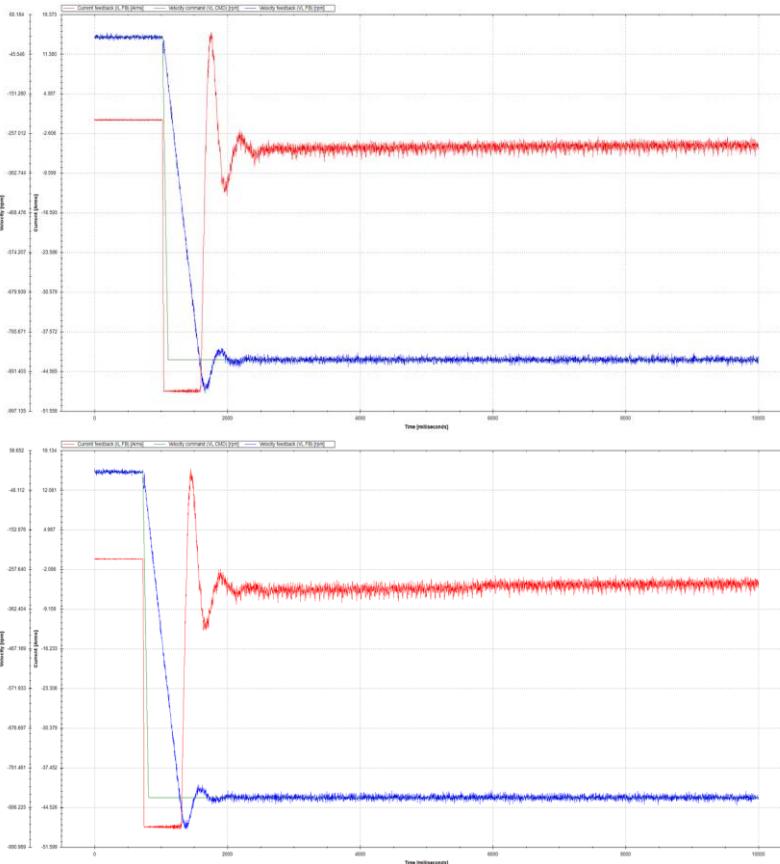
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Collecting Data

By: Jack D.

- Vary height, angle, and speed
- Record load cell data to obtain lift force
- Collect motor current feedback to calculate torque exerted on disk by maglev
- MATLAB program to calculate resultant thrust force



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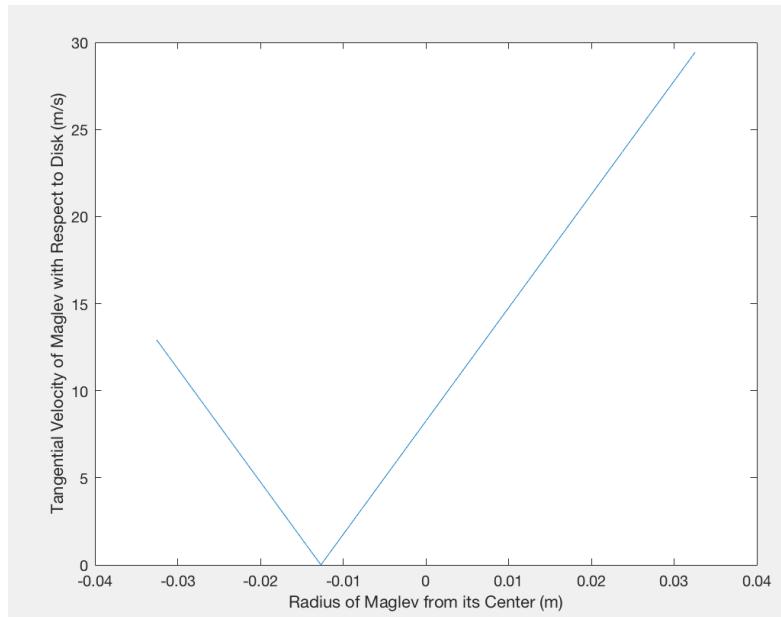
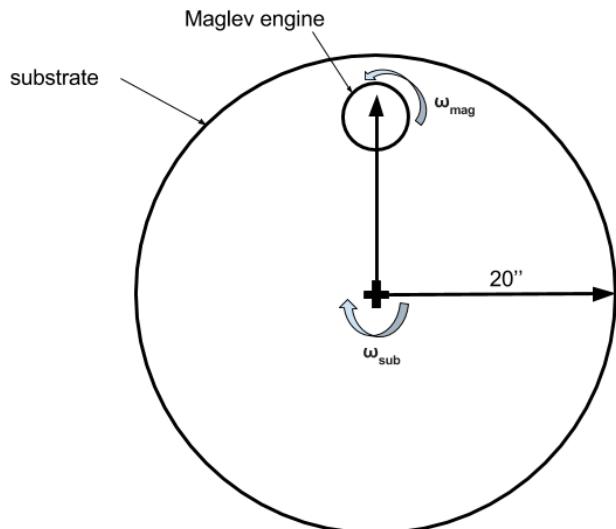
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Analyzing Data

By: Pedro

- The velocity analyzed was the velocity of the Maglev engines with respect to the disk. The velocity vectors were treated similar to a “distributed load” to find a resultant and centroid.
- The linear velocity of the disk were subtracted from the Maglev engine’s linear velocity.



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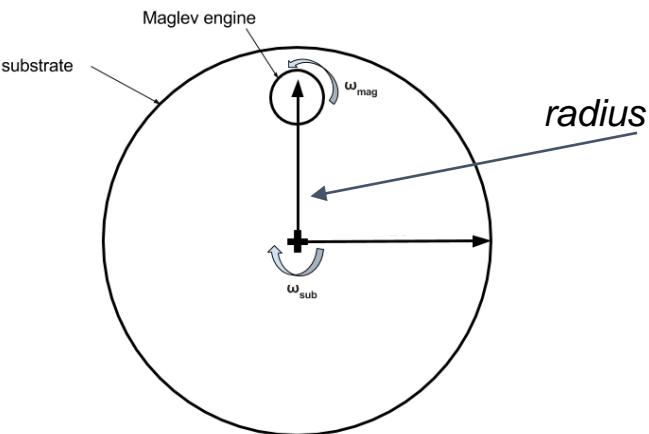
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Analyzing Data

By: Pedro

- We can then use the equation to find the centroid of the load which will be the location where we can apply the basic torque equation.



$$x_S = \frac{\int_0^b x w(x) dx}{\int_0^b w(x) dx}$$

$$\tau_{in} = k_m \times I$$

$$F_{drag} = \frac{\tau_{drag}}{radius}$$

τ =torque
 k_m = Motor Constant
 I = Current

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Magnetic Lift Data

By: Xingchen

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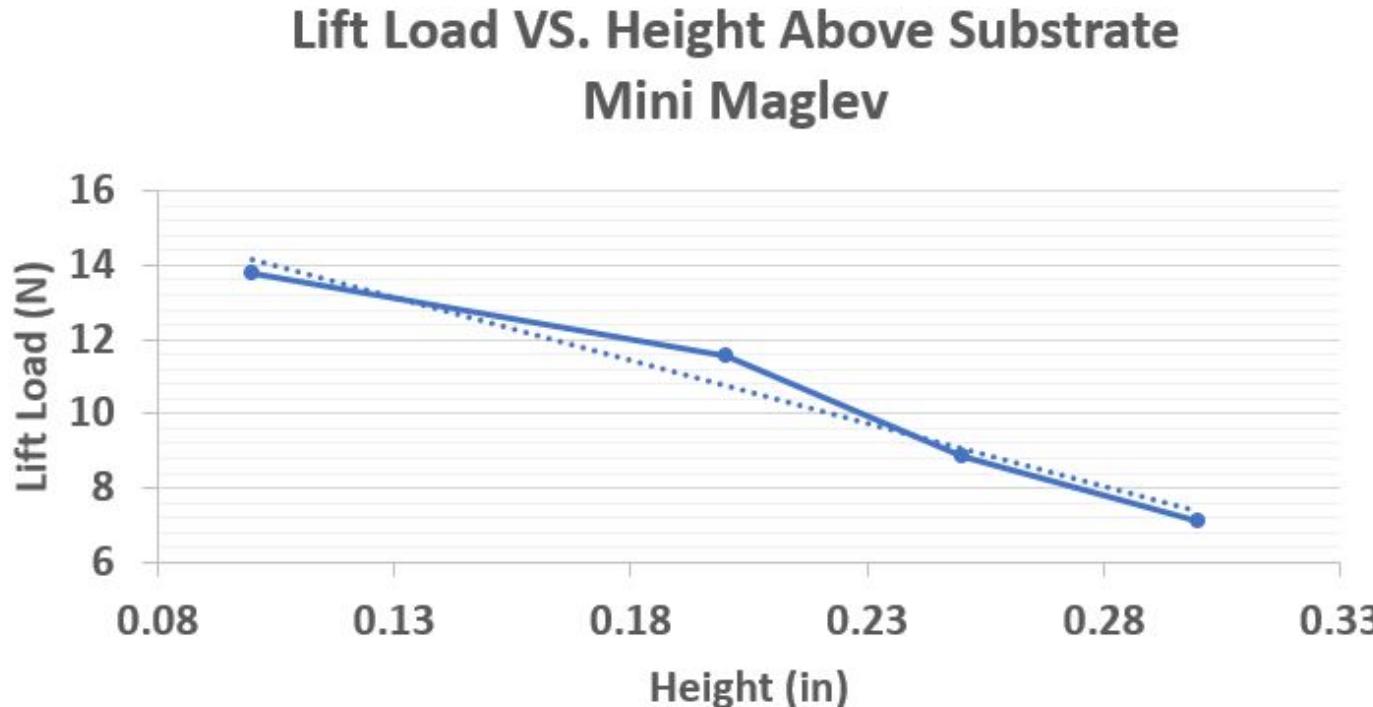
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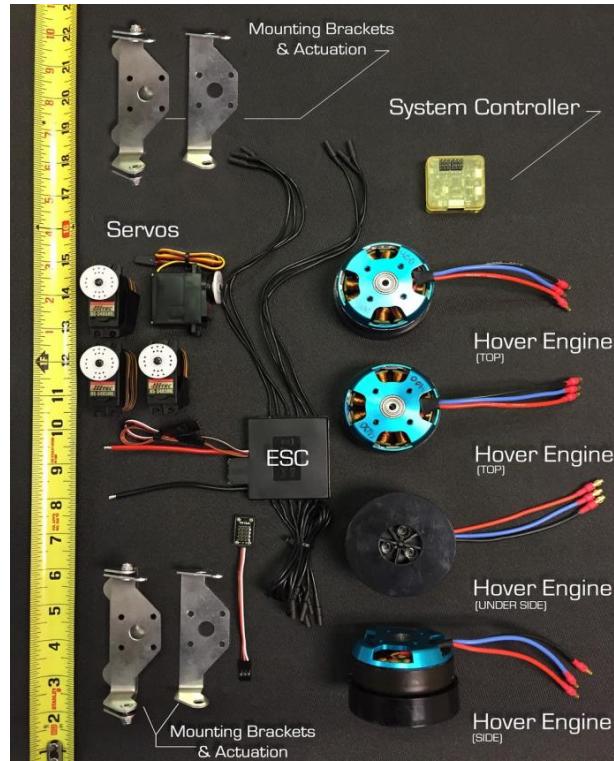
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Custom Maglev Manufacture

By: Andrew

- Currently there are no backups in case of failure.
- Our solution is to design and manufacture custom maglev engines.
- Goal of 10 total custom engines, 4 for the final pod, 4 for potential addition, and 2 for backups.
- The design of these custom motors are meant to replace the HDK maglev engines, allowing current subsystems to continue prototyping with the HDKs and make minimal changes to accommodate for the custom engines.



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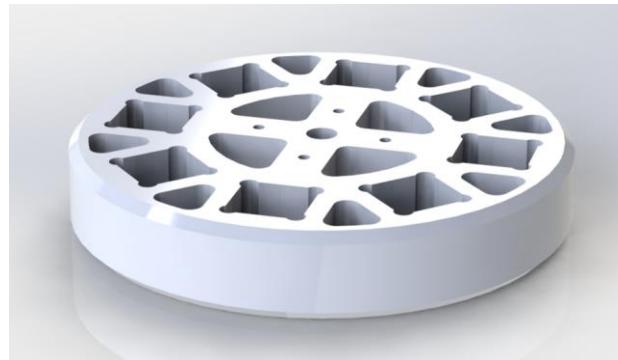
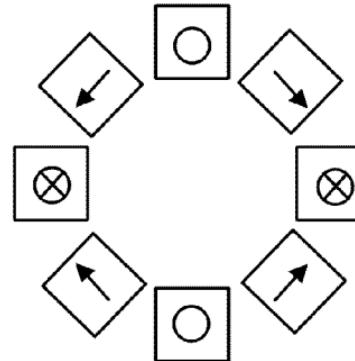
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Custom Maglev Manufacture

By: Andrew

Specifications for Custom Engine:

- Max Voltage: 28.8 V
- Max Current: 24 A
- Max Power: 680 W
- 8* $\frac{1}{2}$ inch cubic N52 Neodymium Magnets
- 3-inch diameter Delrin halbach array housing



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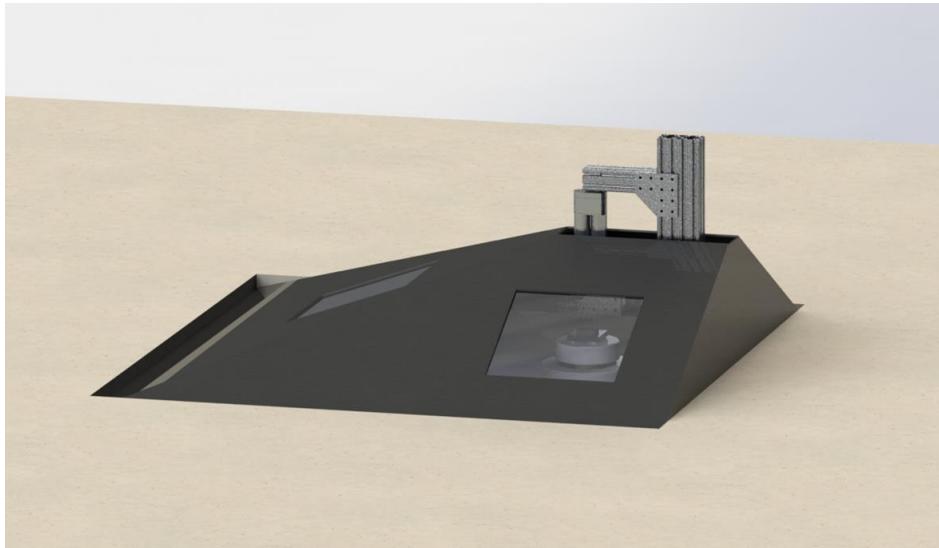
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Custom Maglev Testing

By: Andrew

- Bunker Cover
 - 8 gauge sheet steel
 - Cutouts for acrylic plates
- Spin up custom maglev to 6000 rpm (or failure) in increments of 500 rpm for 1 min.
- Engines are classified safe to use only after 5 successful tests to 6000 rpm.



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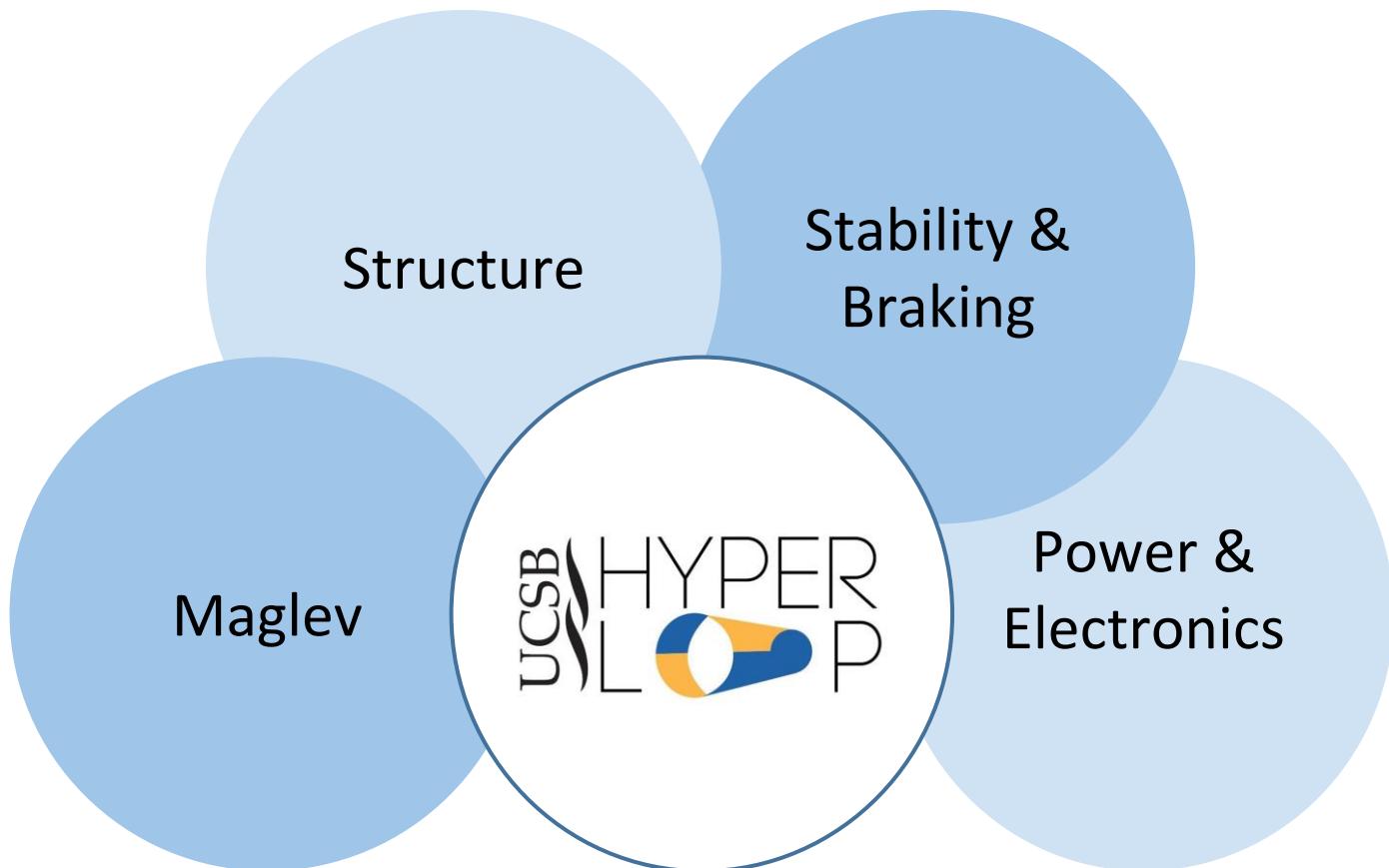
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System Design

By: Nate



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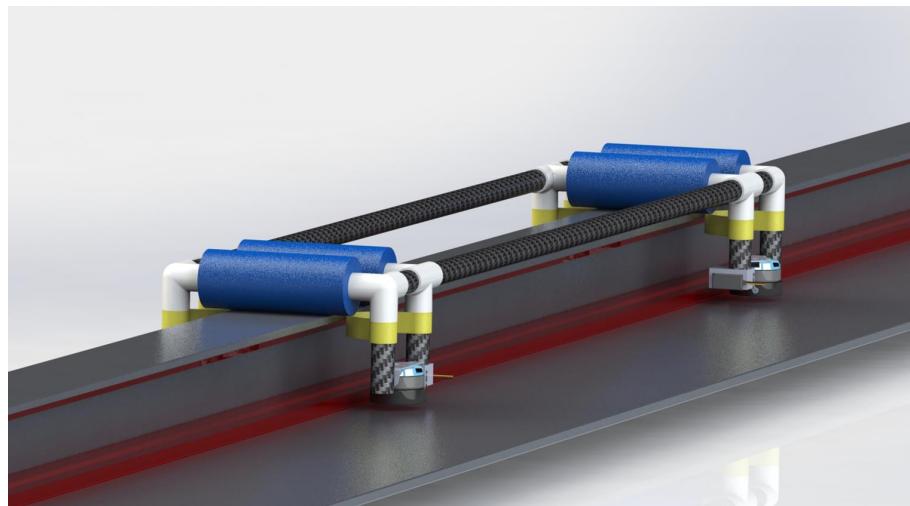
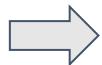
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Structure

By: Ryan P.

Major Structure Design Requirements now focused around Mini Maglev Motors:

- Very Light (2.5 lb or less)
- Strong
- 5 ft. Minimum length



Mini Maglev Prototype

Full Scale Prototype using Carbon Tubing

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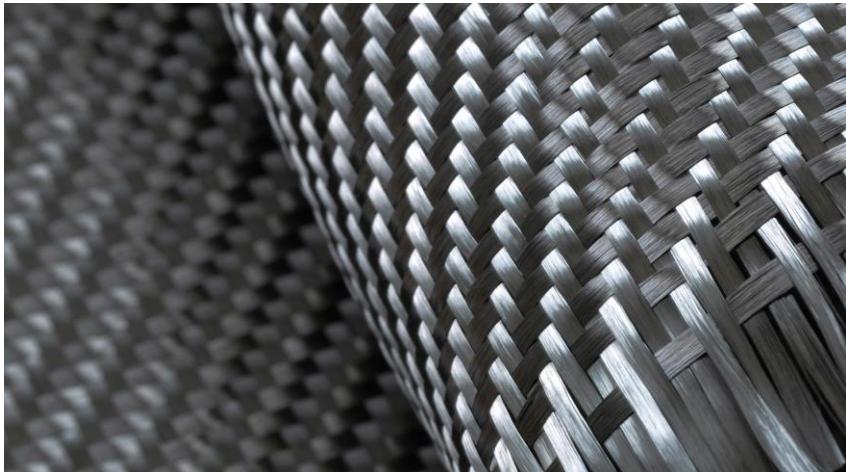
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Structure

By: Ryan P.

Carbon Fiber:

- Not just limited to Tubing/Truss structures.
- Met with Composites One Rep. to learn about different processes



Vacuum Bagging



Autoclave



Honeycomb

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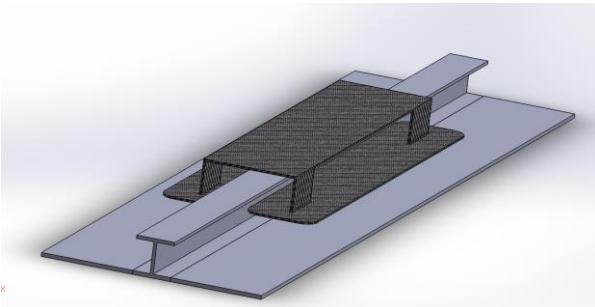
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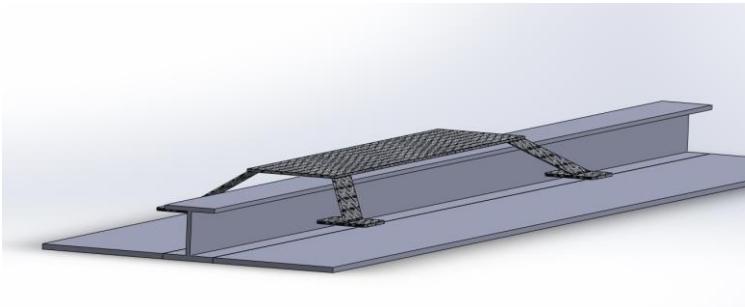
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By: Ryan P.

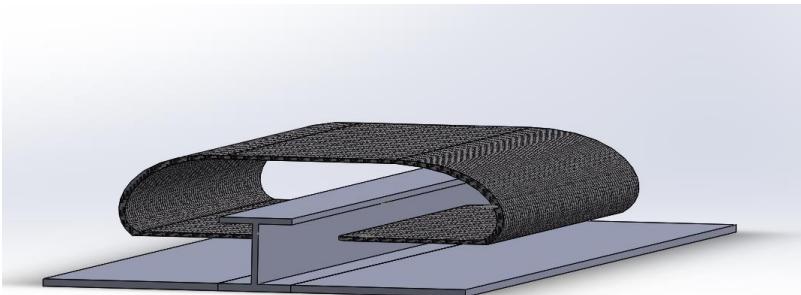
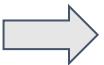
Design Ideas:



Concept 1



Concept 2



Fall Quarter Design

Concept 3

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Structure

By: Ryan P.



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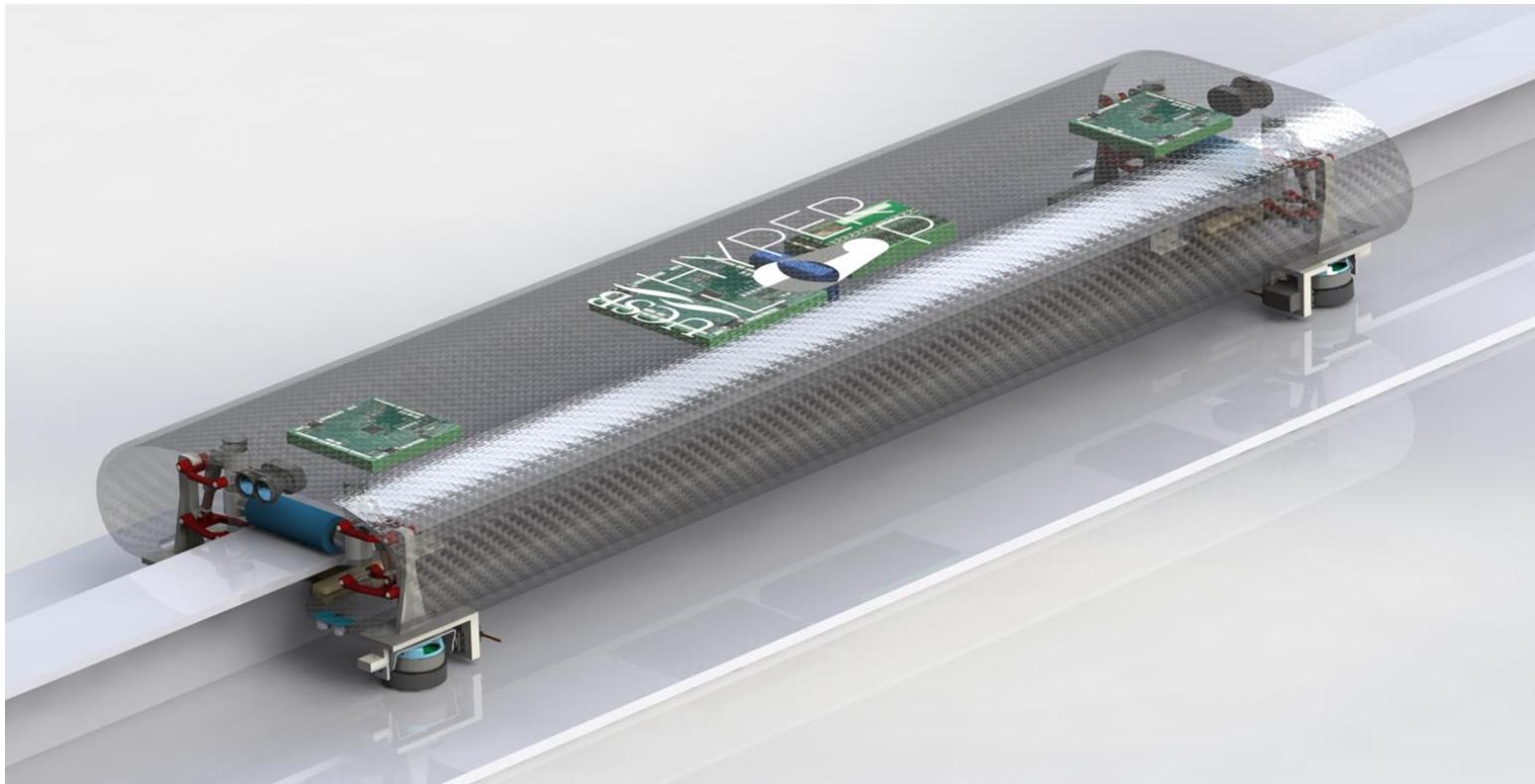
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By: Ryan P.



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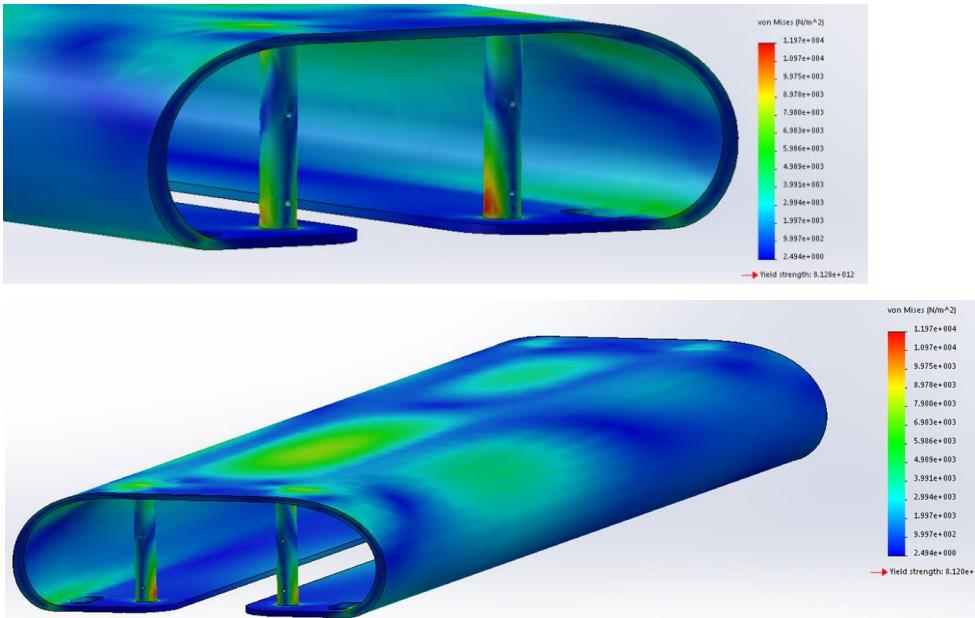
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Structure

By: Edsel

FEA Case 1: Acceleration (max tilt)

- Concern: Base of pillars
- Forces: Lift, Drag, Weight, Thrust
- Results are magnitudes below yield strength



	Hand Calculated Value (KPa)	FEA Value(KPa)
Stress at base of pillar	11.8	12.0

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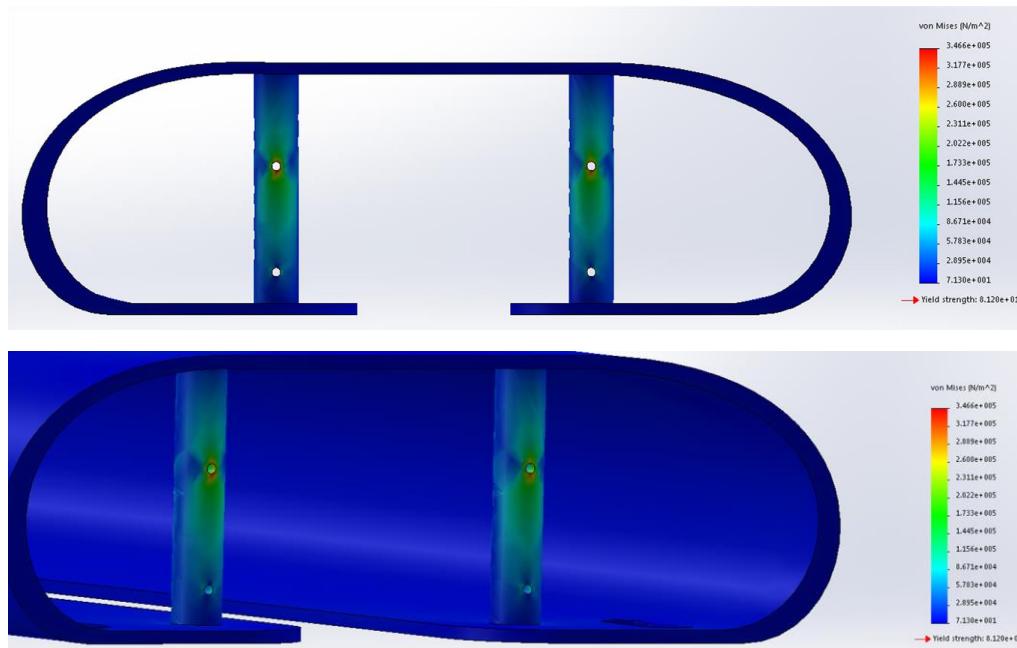
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Structure

By: Edsel

FEA Case 2: Braking (Worst case)

- Concern: Pillar's brake stress
- Forces: Drag, Weight, Inertia, Braking
- Results magnitudes below yield strength



	Hand Calculated Value(KPa)	FEA Value(KPa)
Stress at pillar brakes	31.5	34.7

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By: Ryan P.

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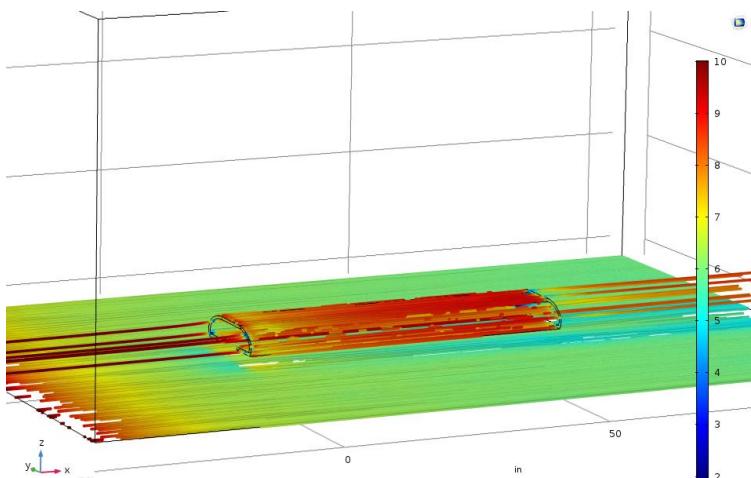
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CFD Analysis of Frame with and without Capped Ends at 10m/s:

Open Ends

Drag Force: 0.58 N

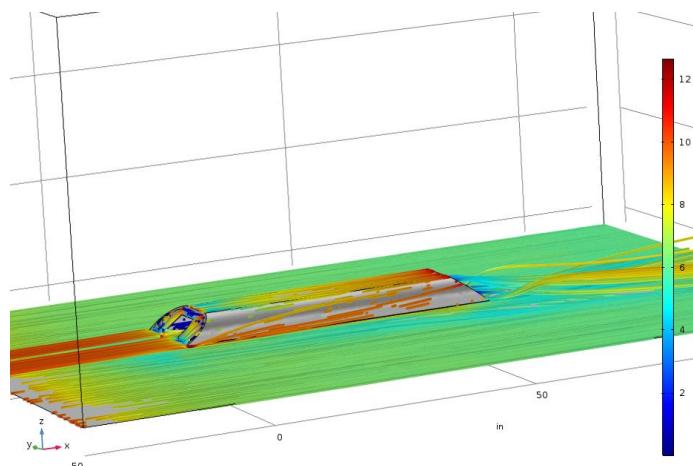
Vertical Force: -0.44 N



Capped Ends

Drag Force: 1.54 N

Vertical Force: -4.64N



Structure

By: Ryan P.

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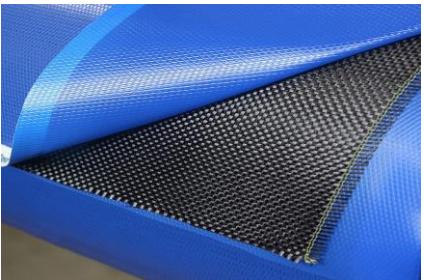
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Prepreg Carbon Fiber



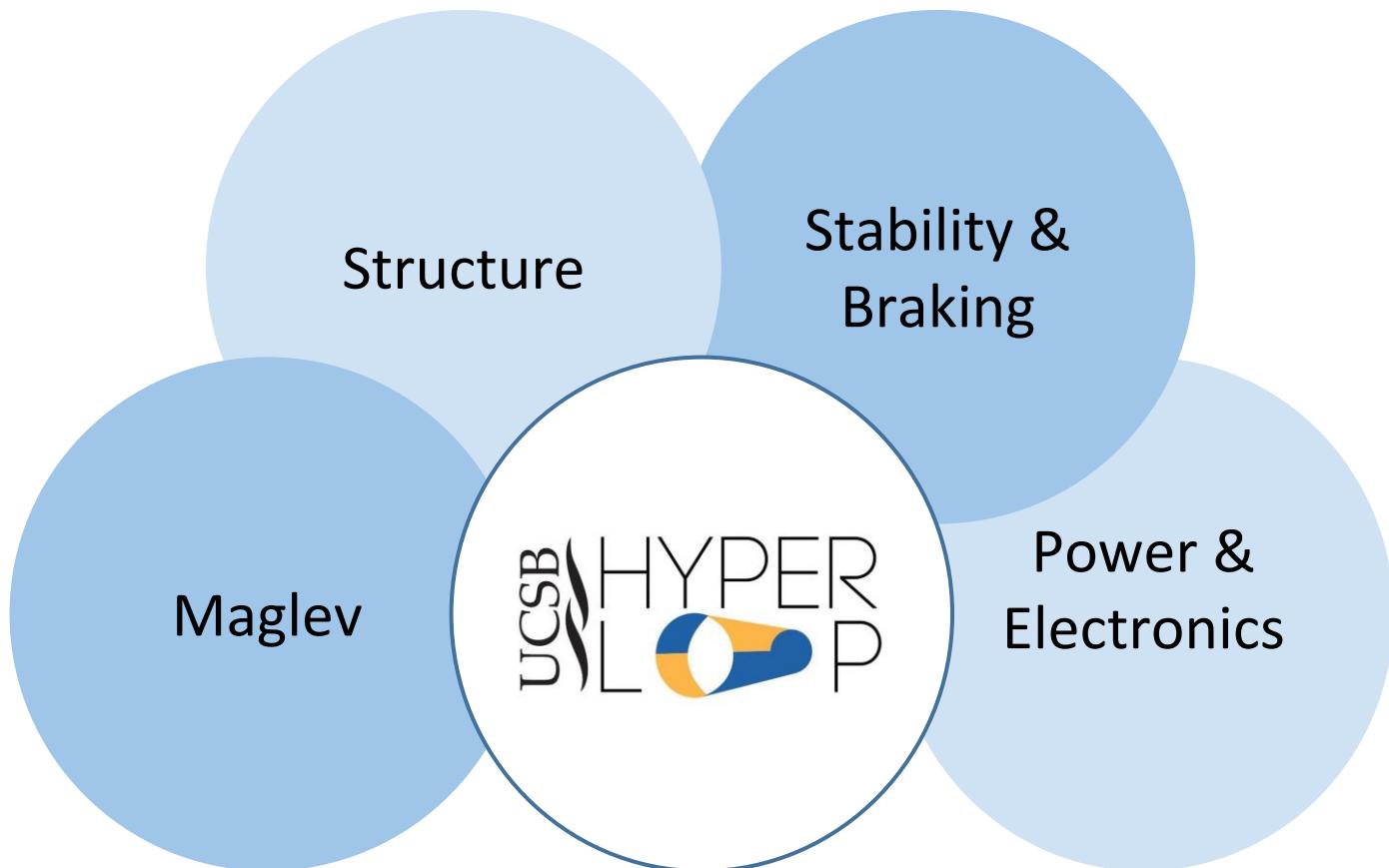
Example Piece



Prototype Mold

System Design

By: Nate



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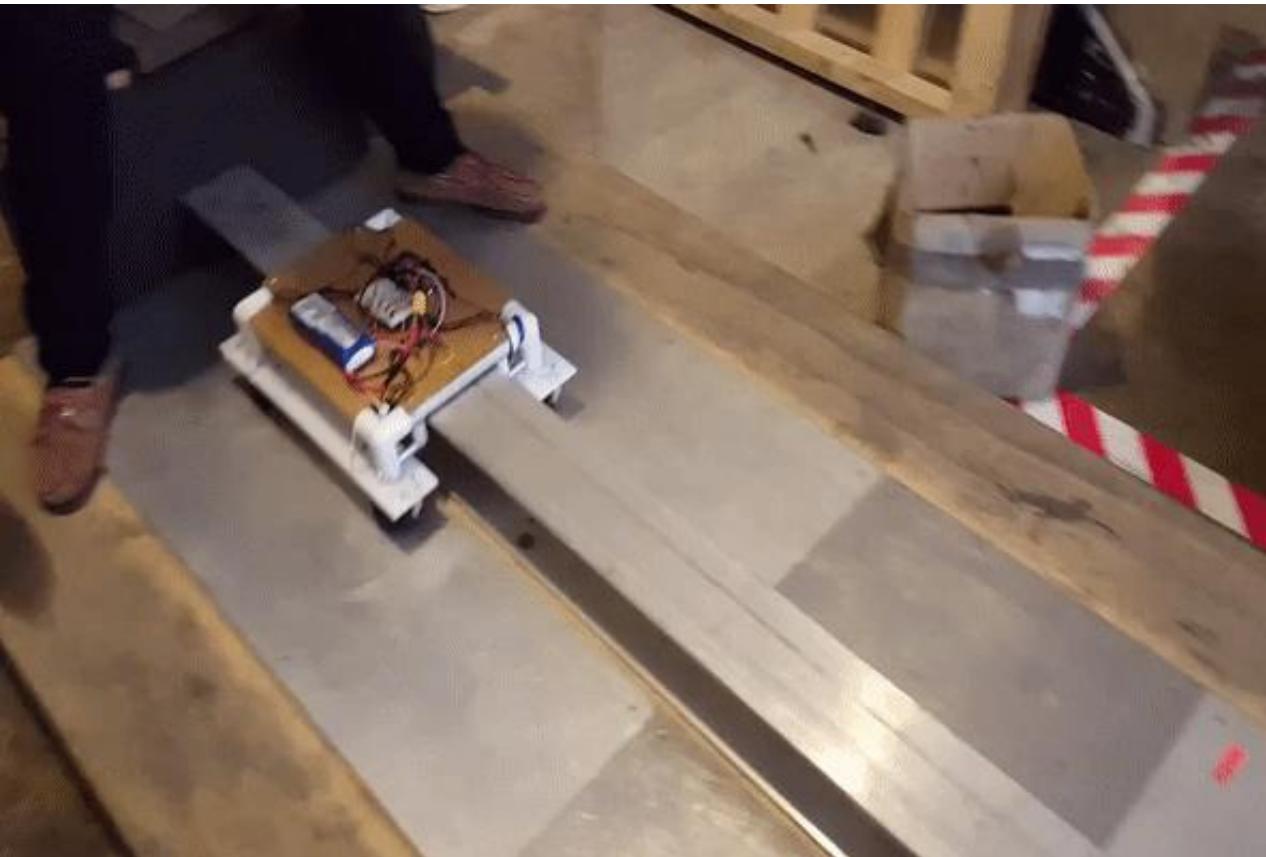
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Primary Braking

By: Gavin M.



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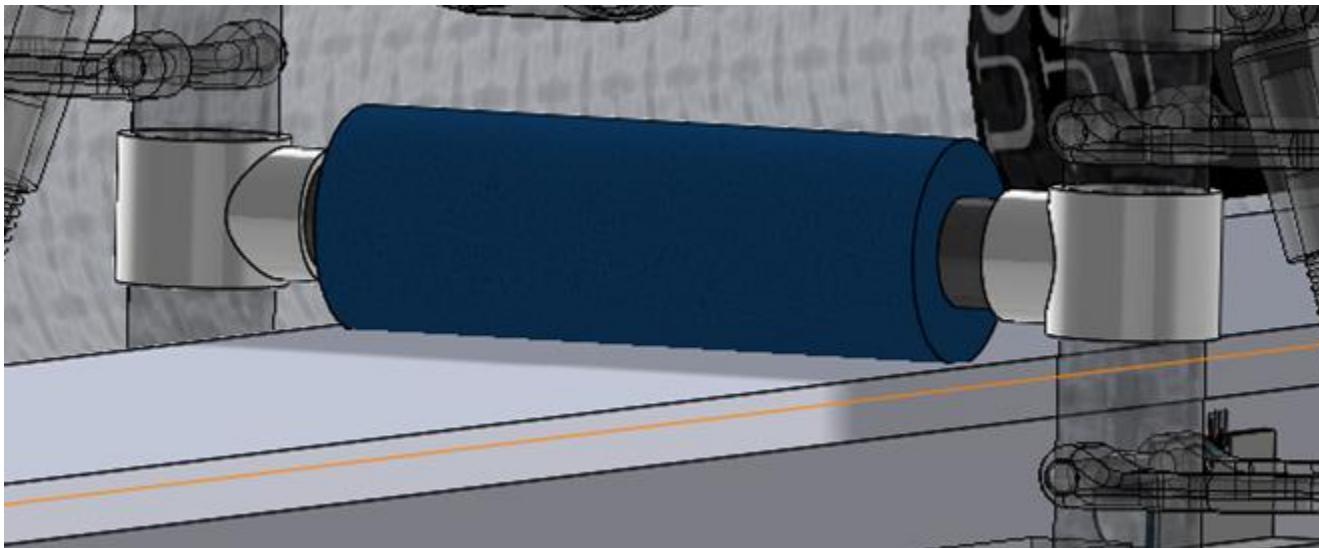
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Secondary Braking

By: Gavin M.

- Only being used if the MagLev engines fail
- Rubber pads on pod
 - They cushion the landing as Maglev engines spin down
 - Generate frictional braking force



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Secondary Braking Analysis

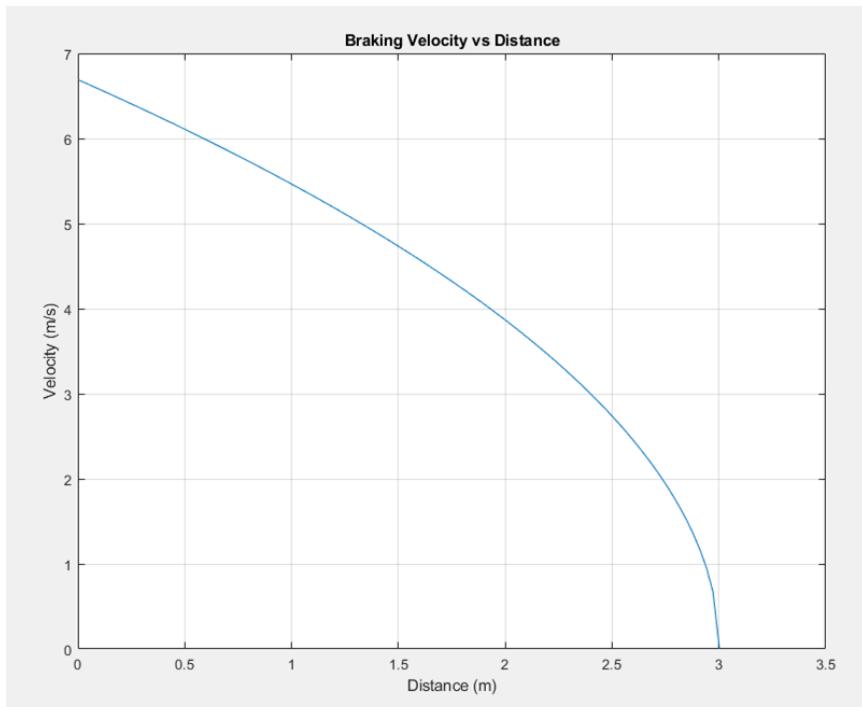
By: Gavin M.

Technical Specifications:

- Coefficient of Friction ~ 0.75
- Max Deflection ~ 2.87 mm
- Thickness of rubber ~ 1 cm
- Weight ~ 560 grams

Deceleration Data:

- Comes to a stop in ~ 0.90 s
 - Travels ~ 3 m



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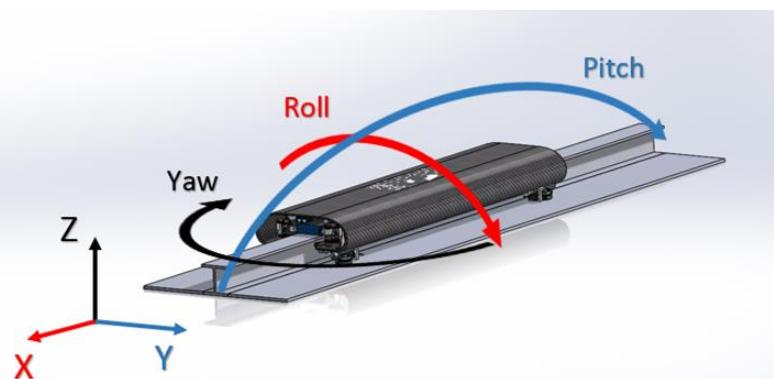
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Stability

By: Gavin M.

Direction	Constraint
X	Maglev gimbal propulsion Failsafe friction rubber brake
Y	Leaf spring lateral suspension
Z	Double wishbone vertical suspension
Roll	Double wishbone vertical suspension
Pitch	Double wishbone vertical suspension
Yaw	Leaf spring lateral suspension



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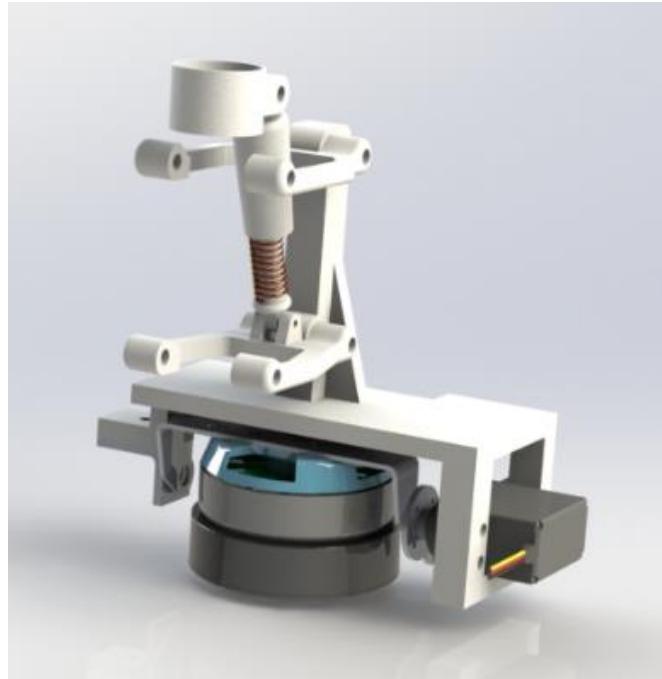
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Vertical Suspension

By: Wesley C.



- Independent double wishbone suspension to maintain pod attitude
- 3D printed Nylon
- Simplified wishbone suspension
- Air piston damper
- Spring constant $\sim 2000 \text{ N/m}$
- Weight per suspension $\sim 350 \text{ grams}$

Total Weight: $\sim 1400 \text{ grams}$

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FEA Vertical Suspension

By: Wesley C.

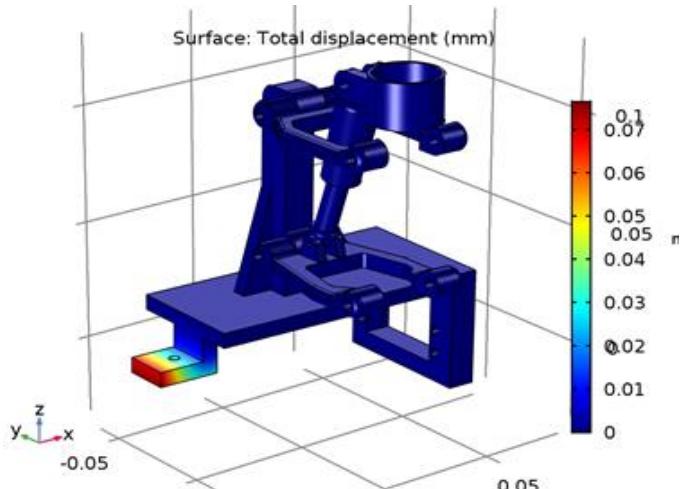


Figure: Maglev Support

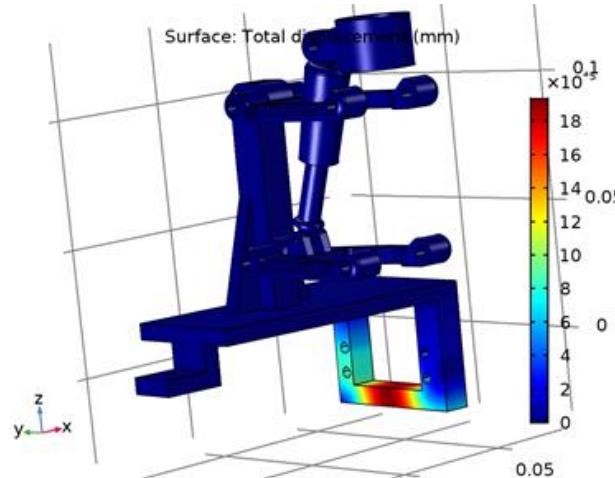


Figure: Servo Support

The displacements of the suspension are very small.

$$S_y = 2 \times 10^9 \text{ Pa}$$

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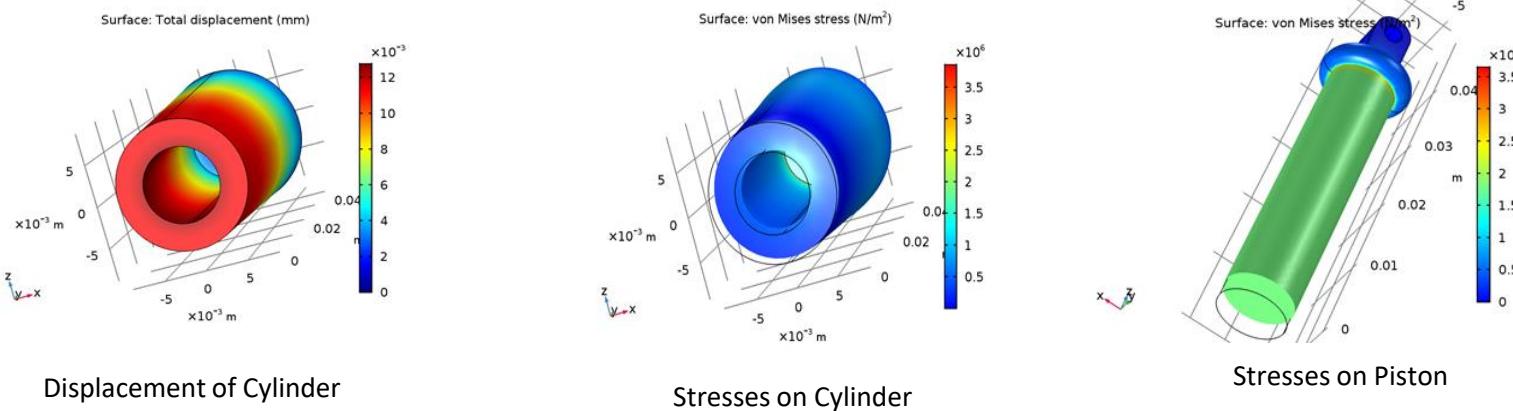
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FEA Vertical Suspension

By: Wesley C.



Hand Calculations:

Boyle's Law:

$$P_1 V_1 = P_2 V_2$$

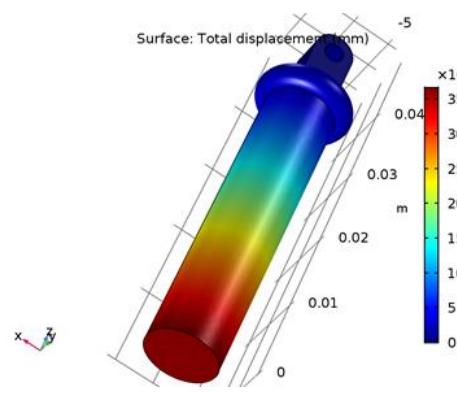
Stress in a Pressure Vessel:
kPa

$$\sigma = (P_2 R) / (2t) = 180$$

$$S_y = 2 \times 10^9 \text{ Pa}$$

$$\sigma < S_y$$

∴ Will not yield



Displacement of Piston

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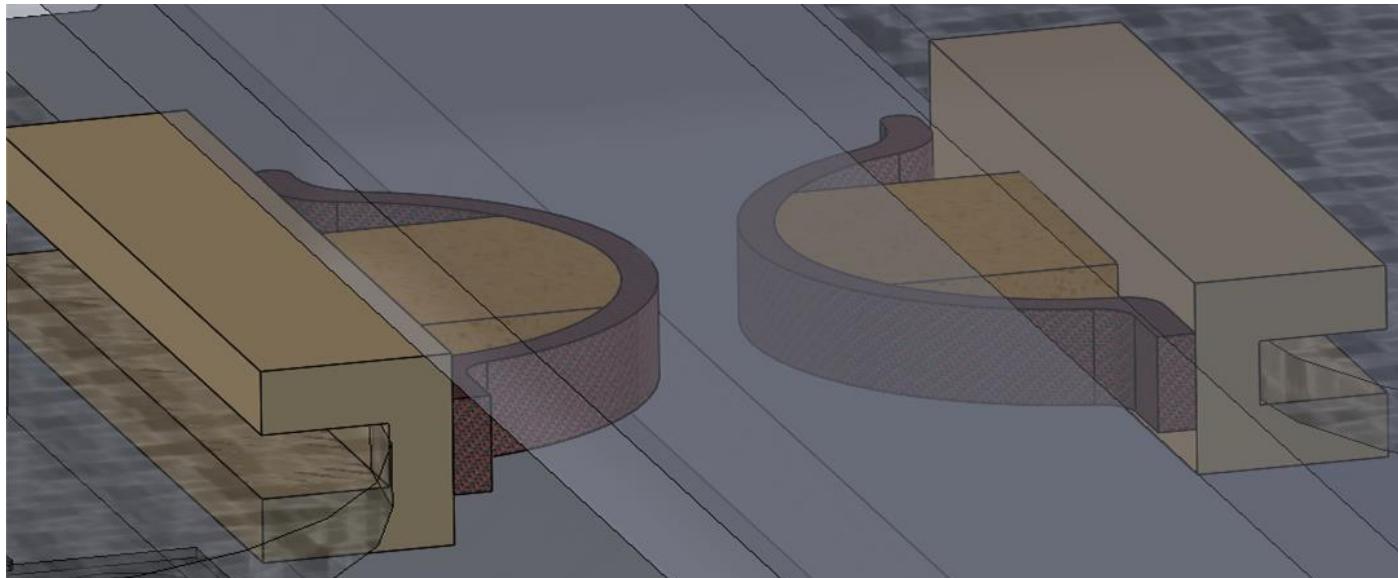
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Lateral Suspension

By: Gavin M.

- Composite E-glass leaf spring
- Polyurethane foam block damper
- Teflon tape on the surface to minimize friction between spring and I-beam
- Weight per leaf spring ~ 30 grams

Total Weight: ~ 120 grams



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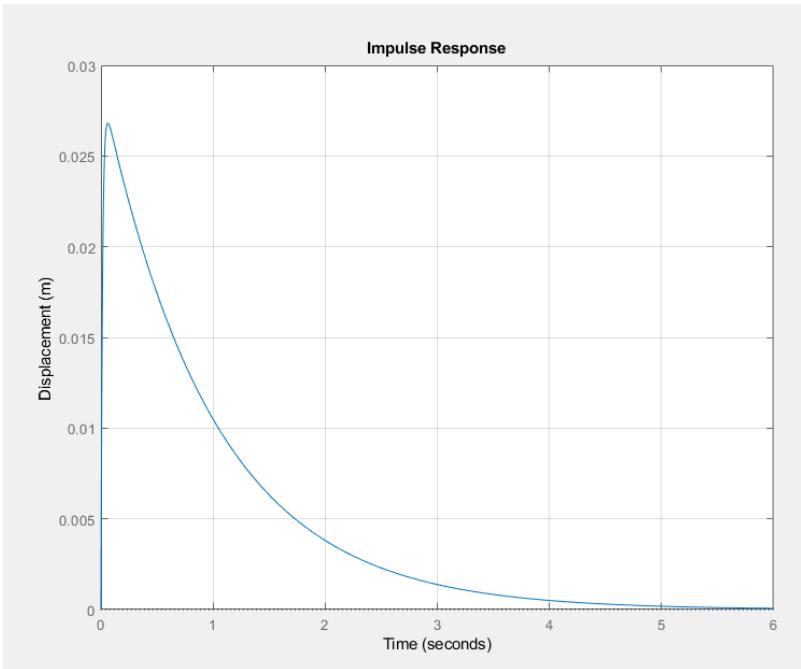
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Lateral Suspension

By: Gavin M.



Damping Time ~ 3 s

- Expected forces on leaf spring only require a spring force with $k \sim 20$ N/m
- Polyurethane foam has a damping coefficient up to $b \sim 40$ Ns/m

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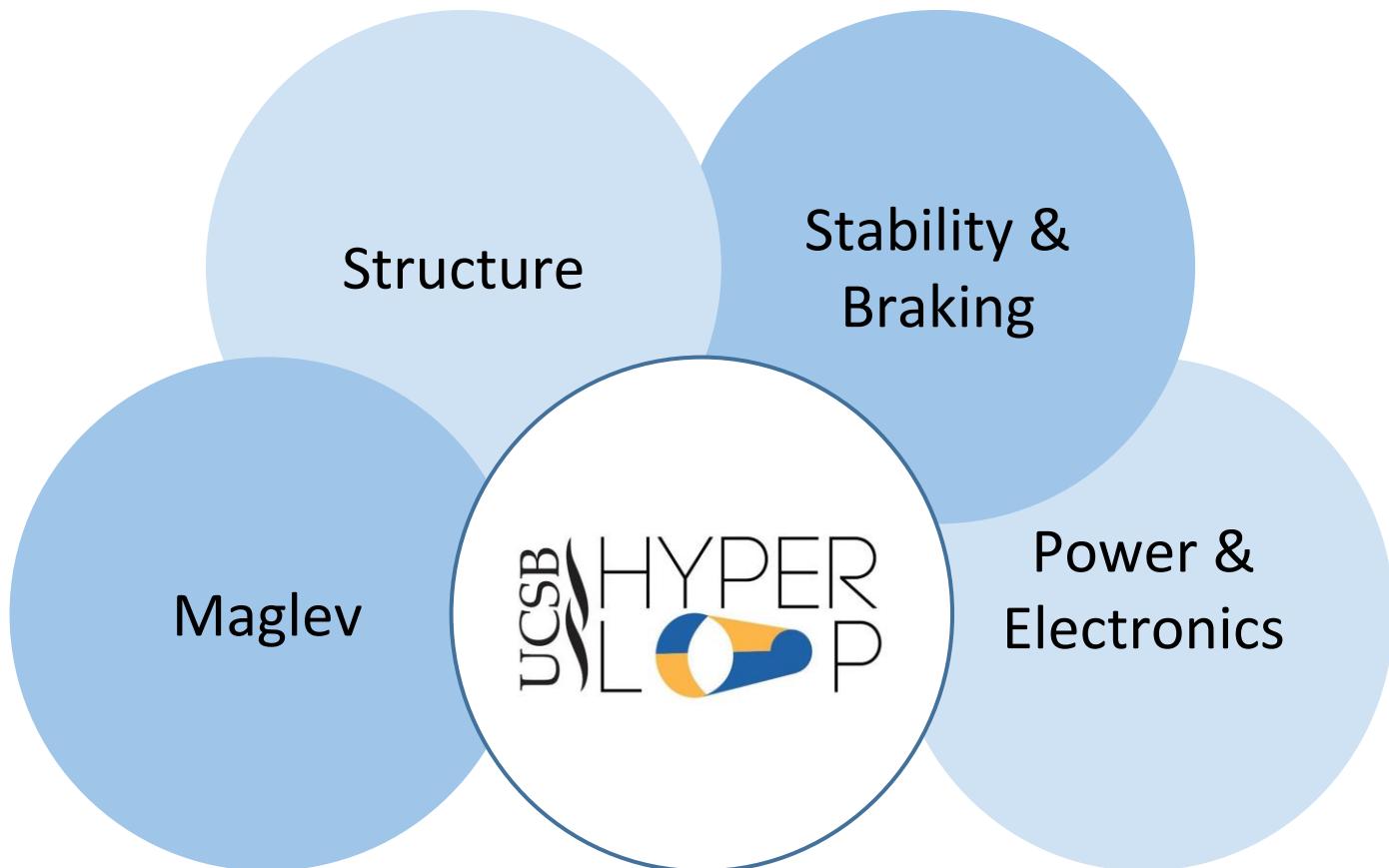
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System Design

By: Himangshu



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Battery Chemistry Decision

By: Himangshu

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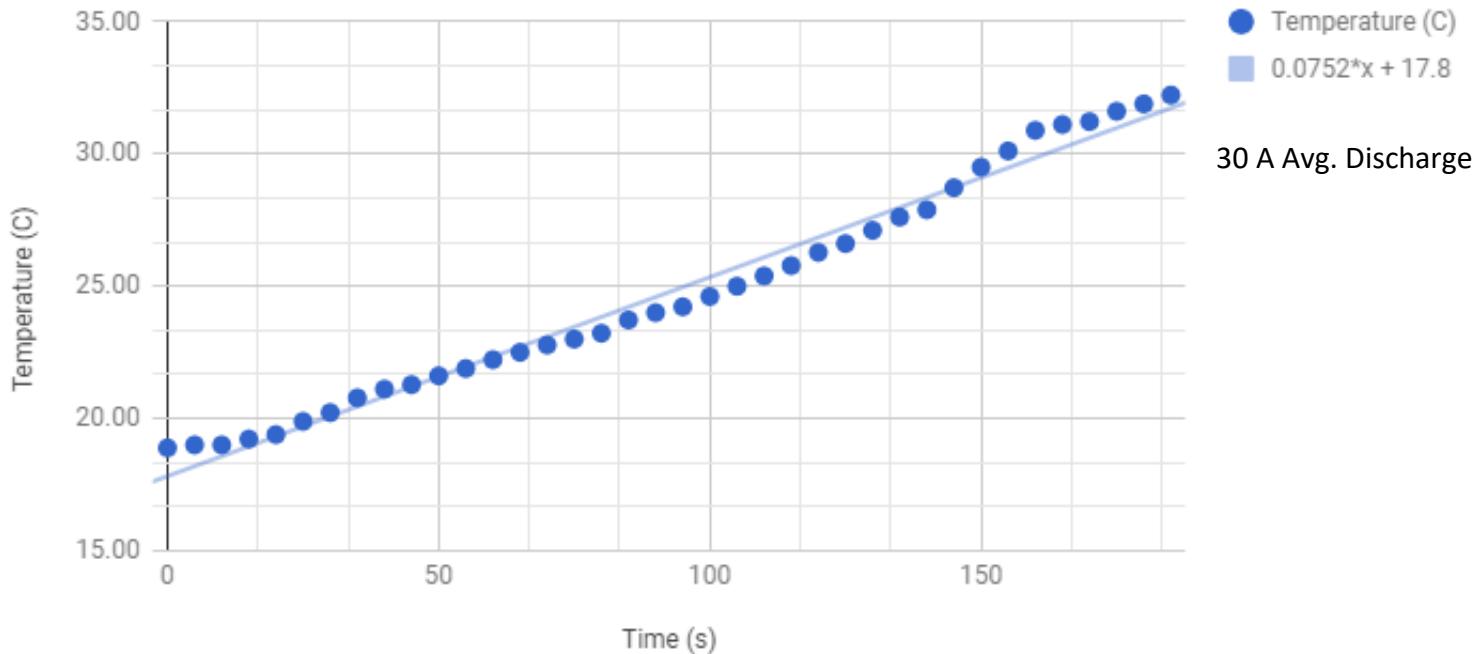
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	Lithium Ion (NMC)	Lithium Polymer (CO)
Cells in Series	4	4
Cells in Parallel	4	1
Total Nominal Voltage (V)	14.8	14.8
Total Cont. Current (A)	120	99
Total Capacity (mAh)	6000	2200
Total Weight (g)	725	245
Power/Weight Ratio (W/g)	2.45	5.98
Total Cost per Pack (\$)	65	30
Safe Temperature (°C)	0-70	0-60(external)

LiPo Temperature Test

By: Mihir

LiPo Pack Temperature (C) vs. Time (s)



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Battery Management System

By: Mihir

- **BMCU (Battery Management Control Unit):**
 - monitors performance parameters (pack voltage, shunt current, resistance, SoC, SoH)
 - RS232 for communication with PCB
 - inputs & outputs for event notification, connects to peripherals
- **LMU (Local Monitoring Unit):**
 - monitor and balance individual cells at up to 0.84A
 - monitor self temp(shunt temp during bleeding) and pack temp using onboard/external sensors



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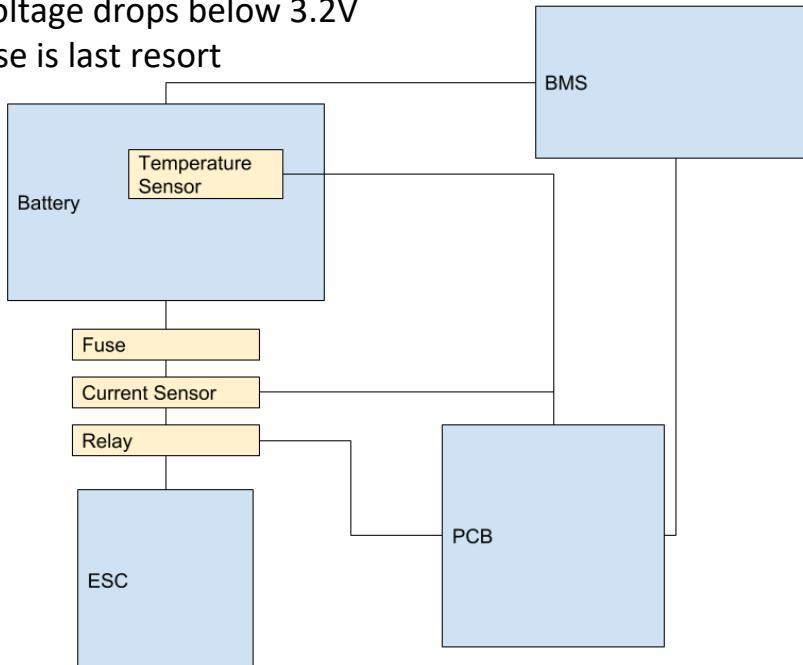
Budget

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Safety Protocols

By: Himangshu

- **Relay:** PCB switches relay to physically connect/disconnect battery under normal operation
 - If current rises too far above 100A
 - If temperature of battery rises above 60C
 - If cell voltage drops below 3.2V
- **Fuse:** 130A fuse is last resort



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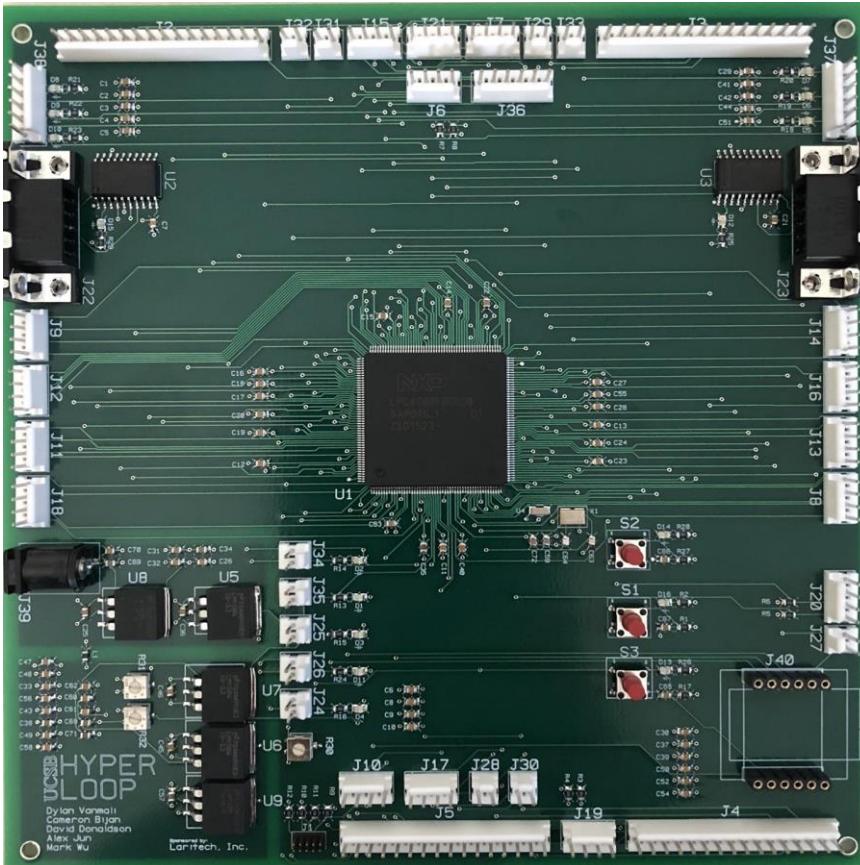
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Main PCB

By: Dylan



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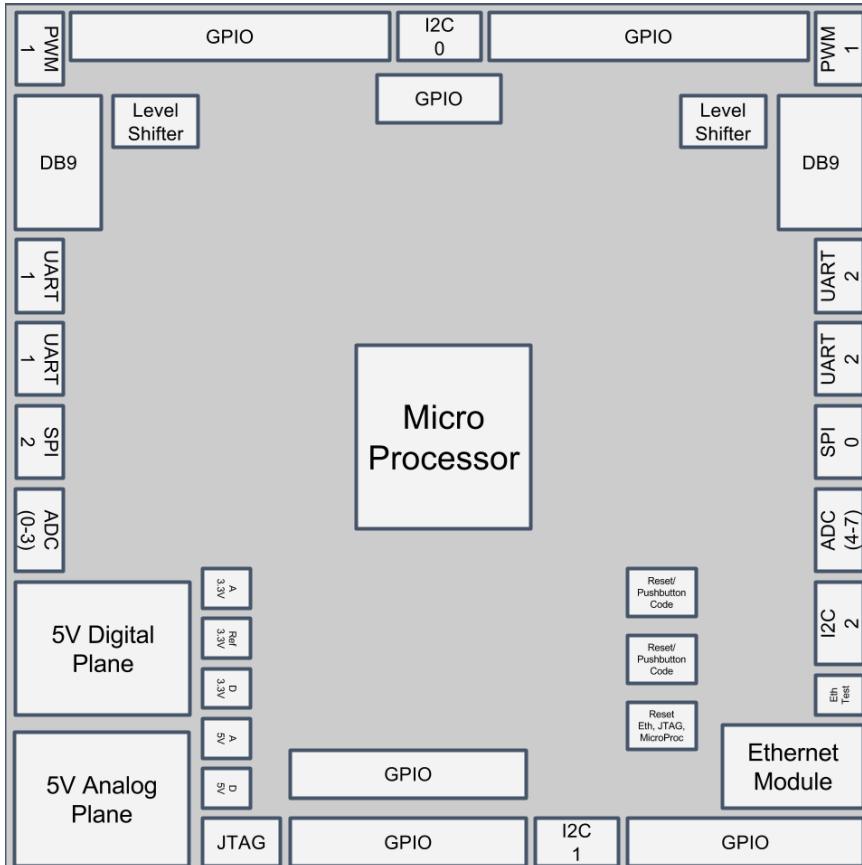
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Main PCB

By: Dylan



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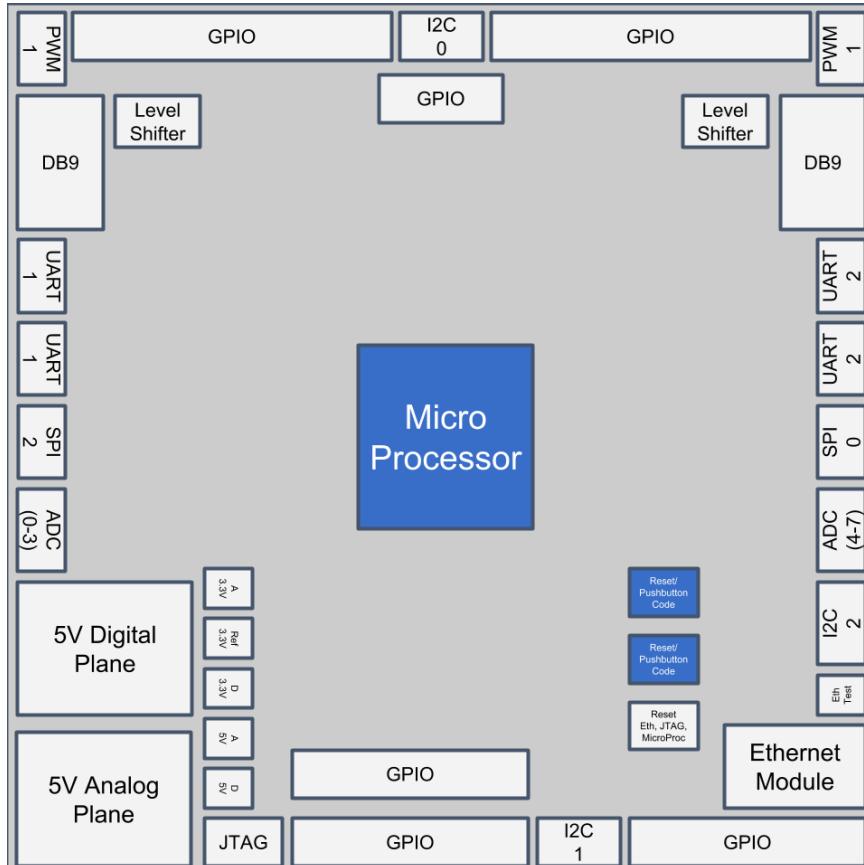
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Computing

By: Dylan



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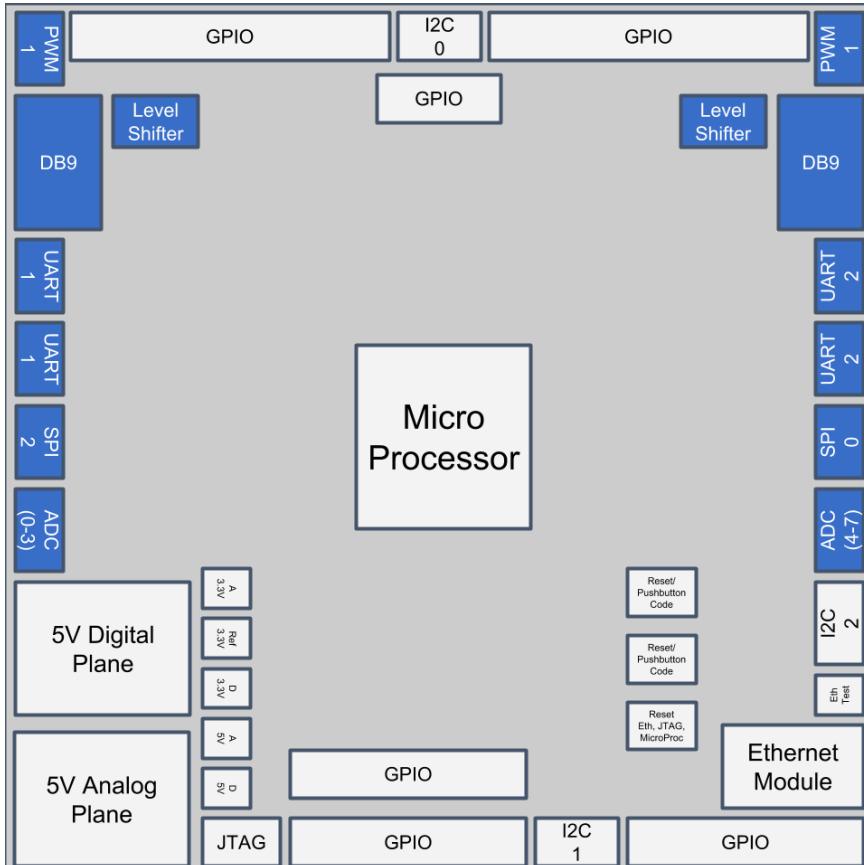
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Communication Ports

By: Dylan



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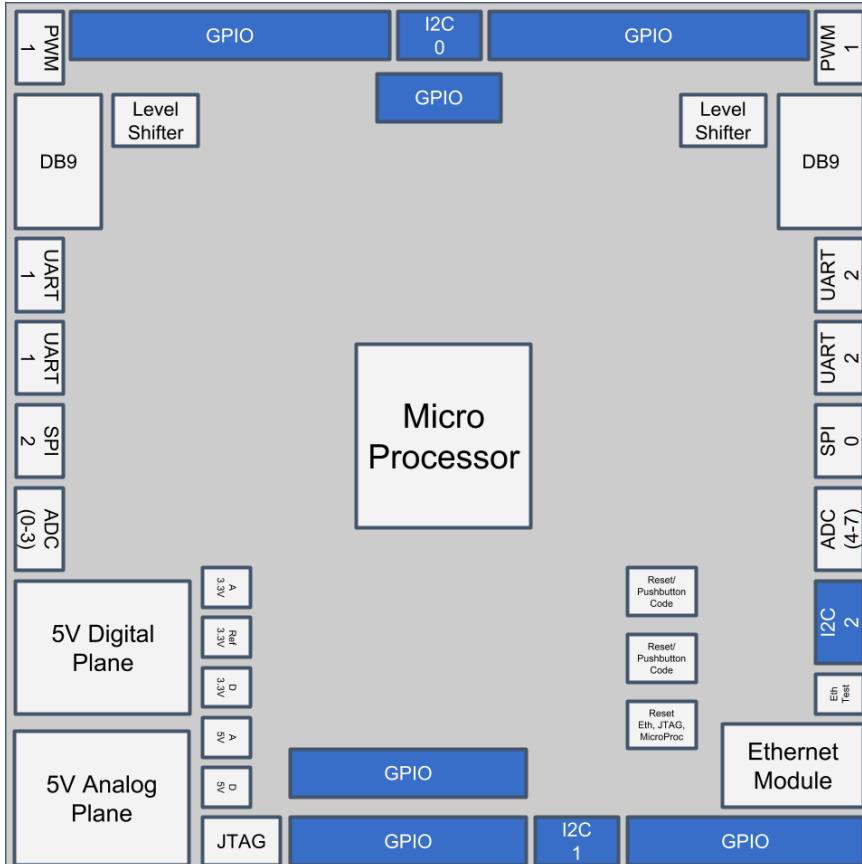
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Local Ports

By: Dylan



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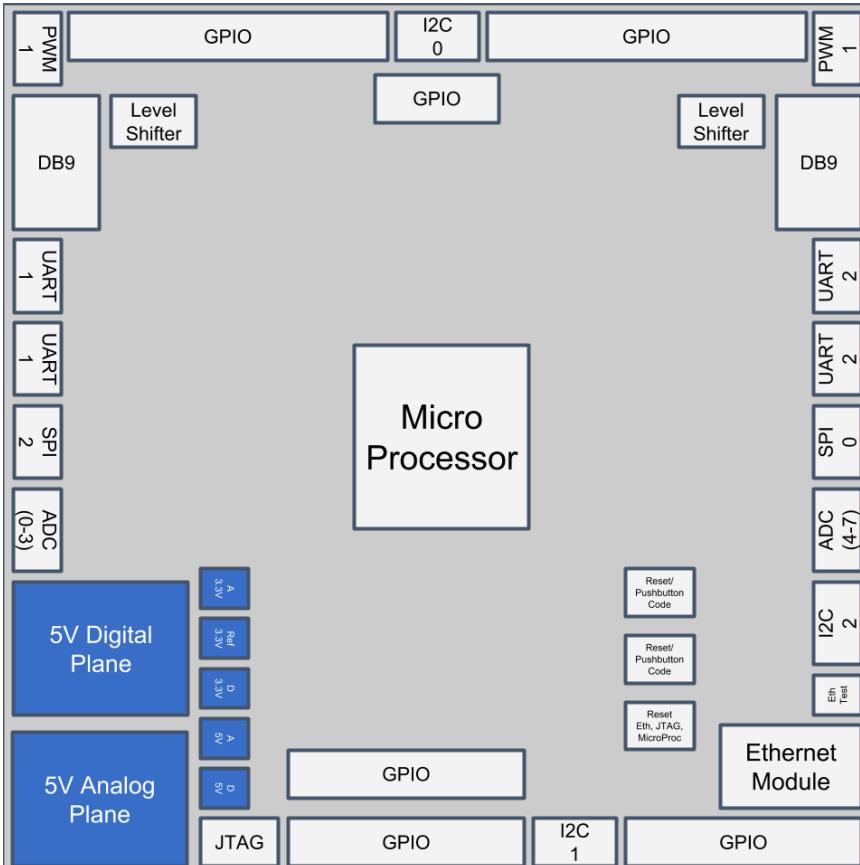
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Power

By: Dylan



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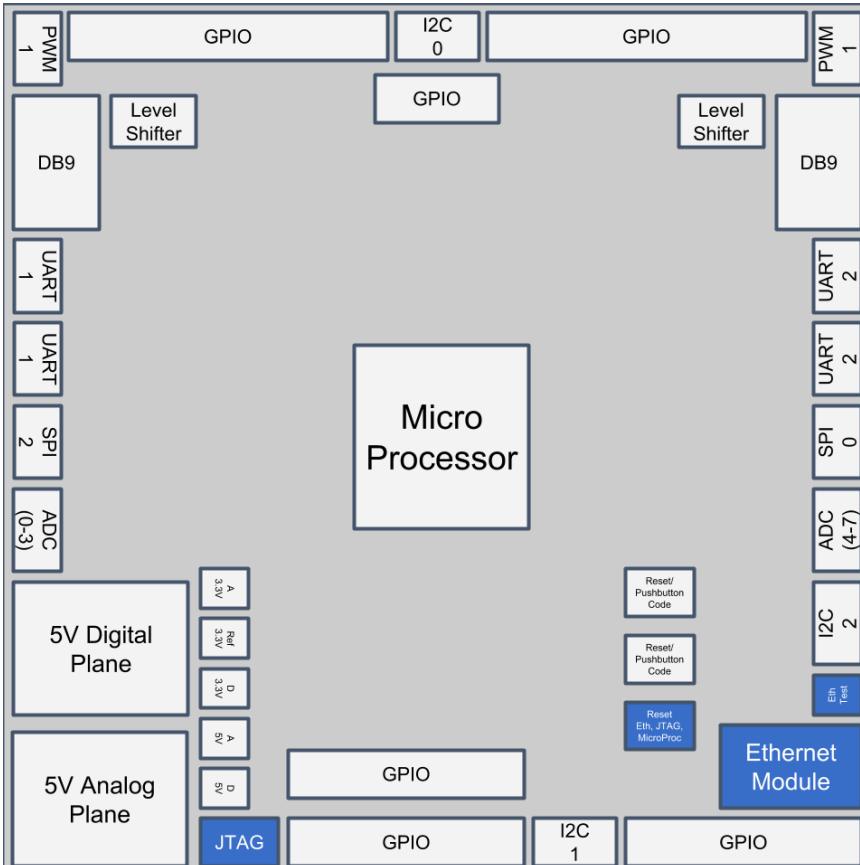
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Programming and Wireless Links

By: Dylan



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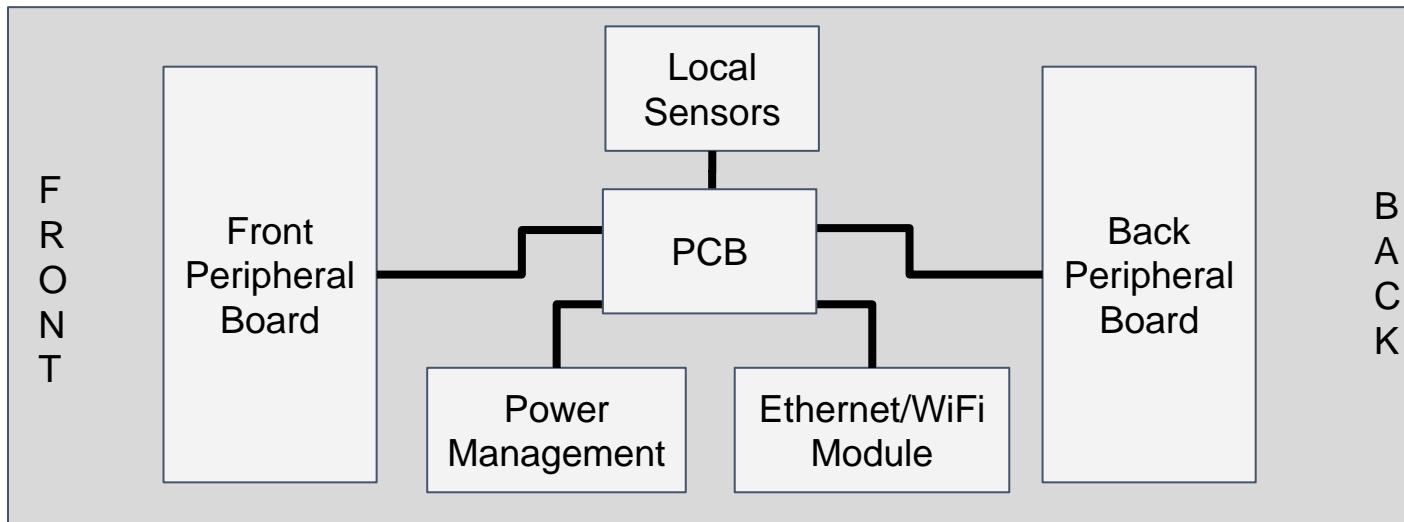
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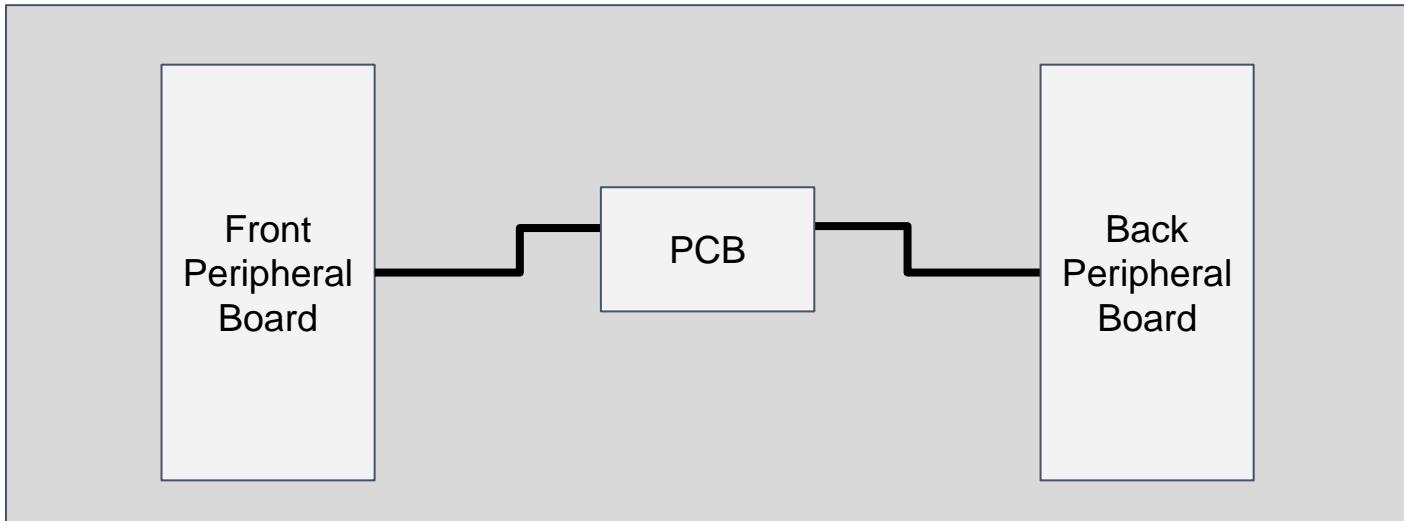
High-Level Electronic Scheme

By: Xiaochang

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Sensor Locations

By: Cameron



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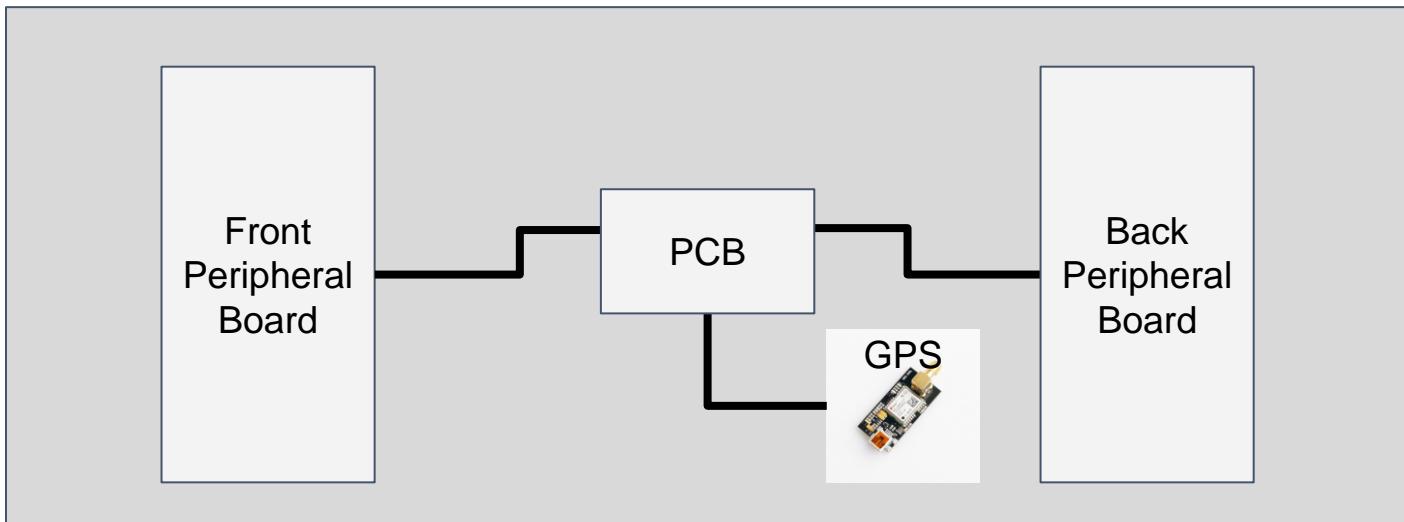
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By: Cameron



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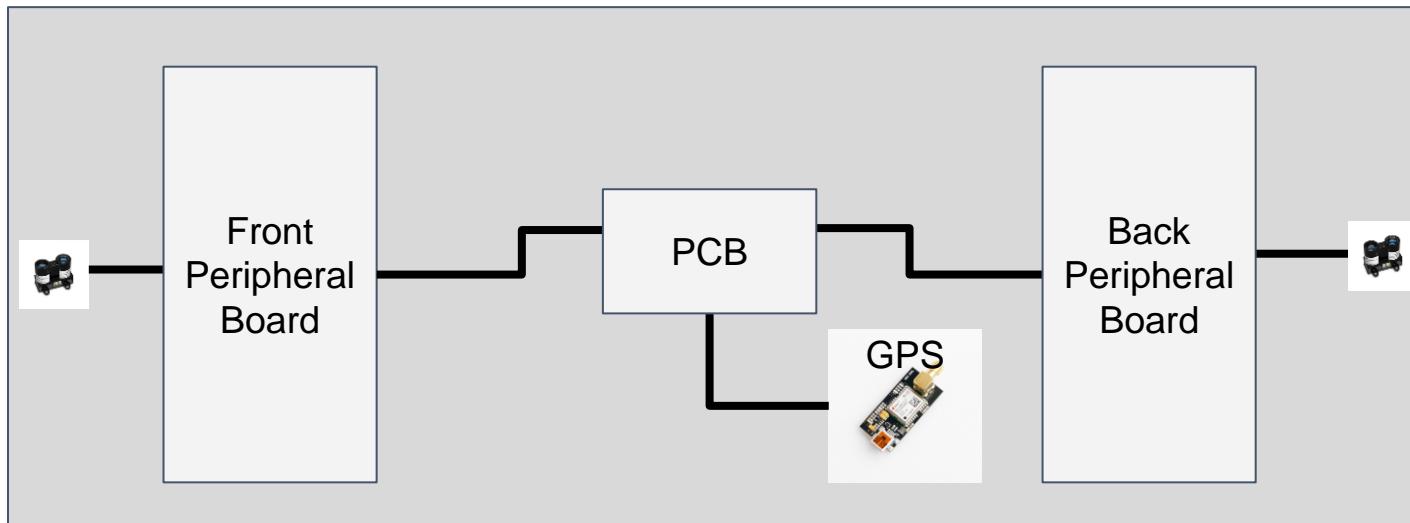
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By: Cameron



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By: Cameron

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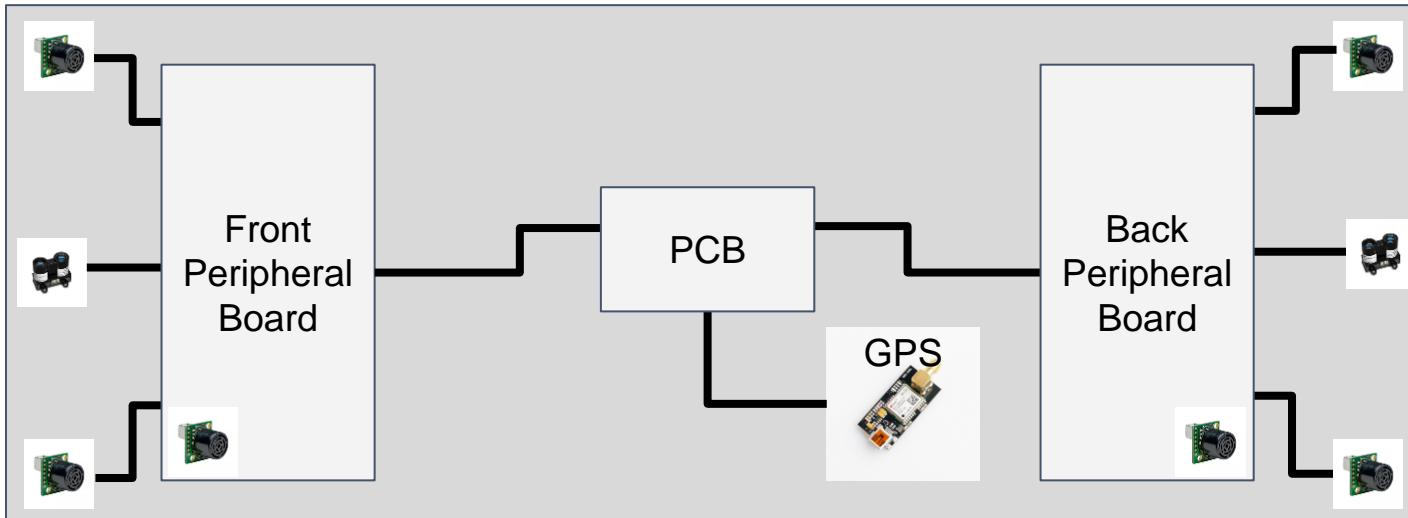
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Sensor Locations

By: Cameron

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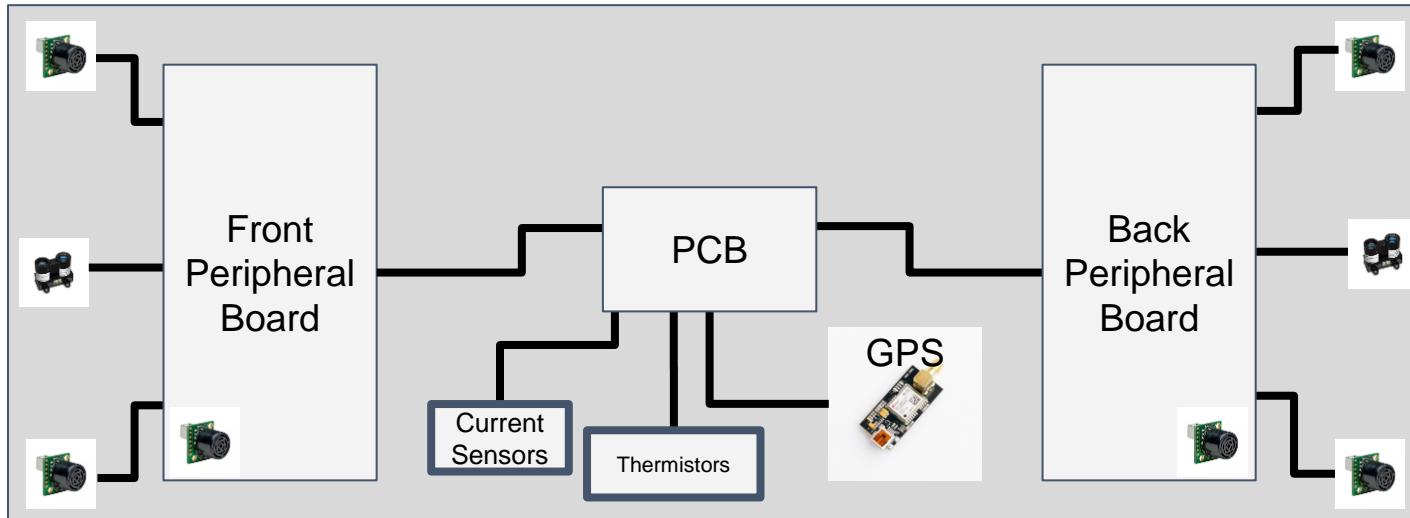
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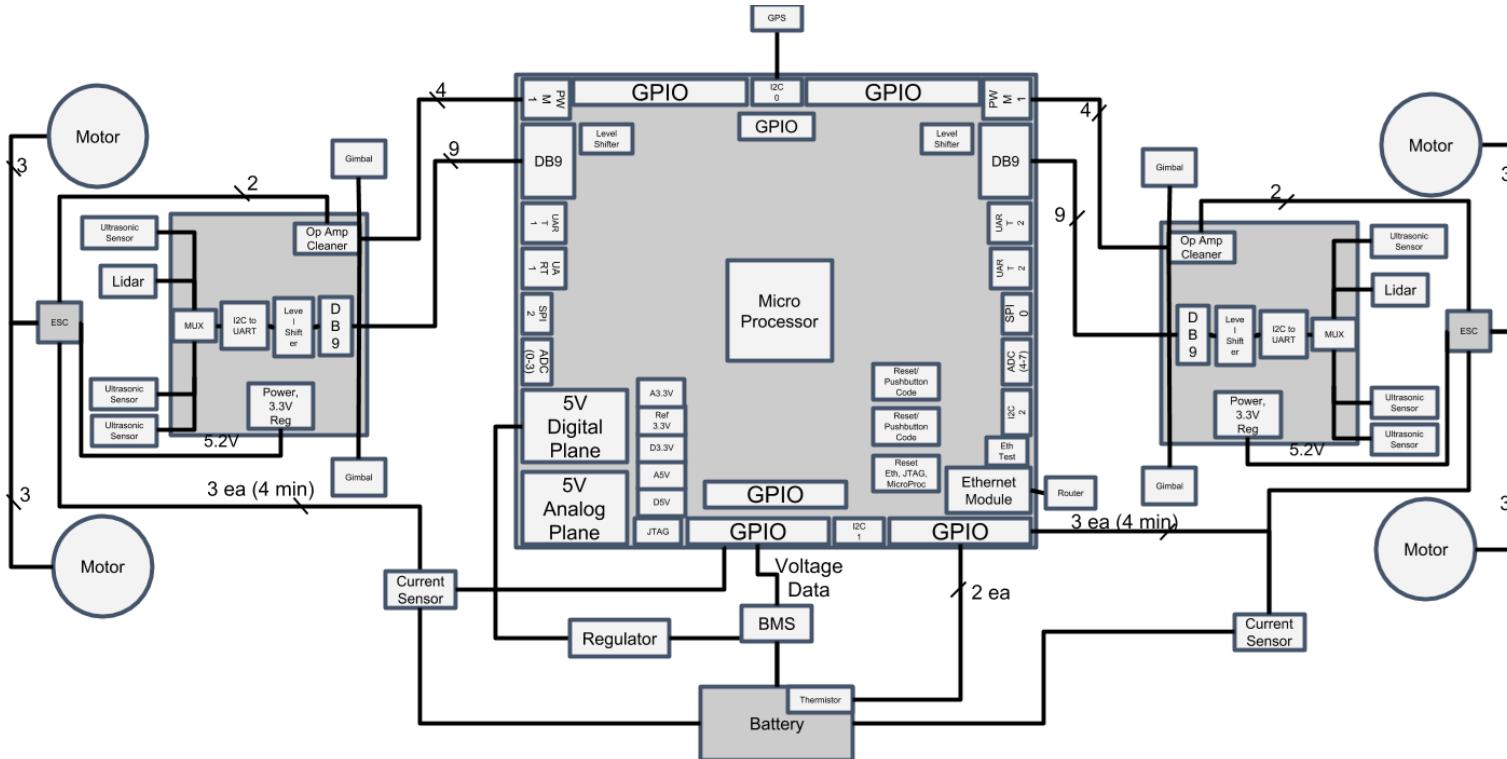
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Detailed Electronic Scheme

By: Cameron



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Peripheral Board

By: Cameron

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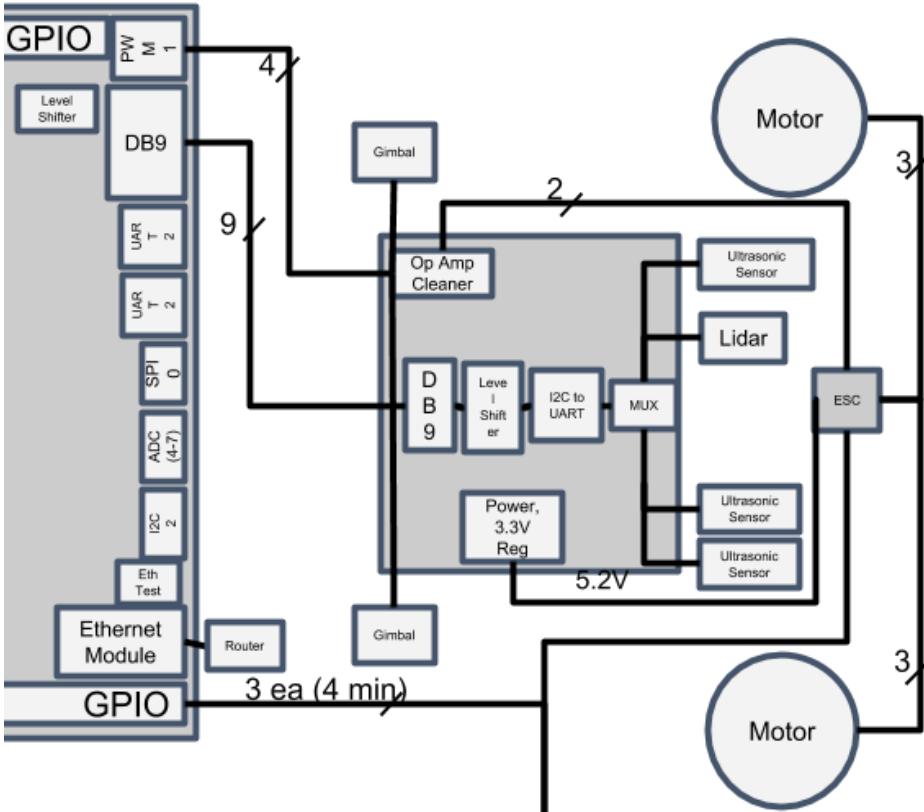
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Collecting Data

By: Cameron

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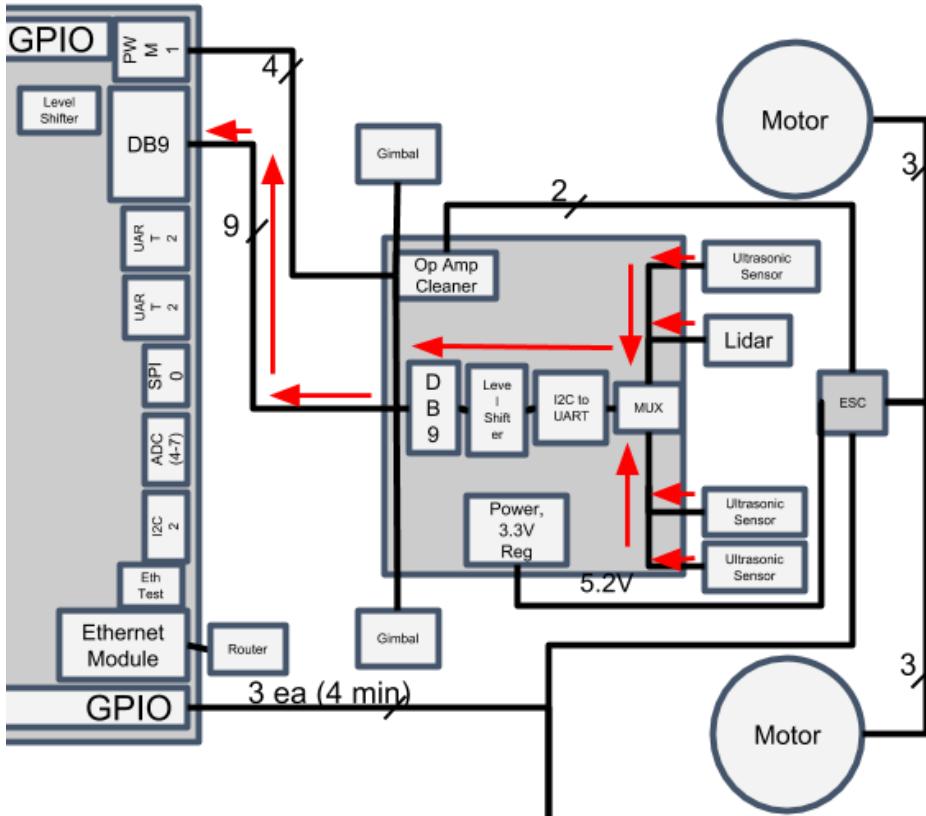
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Instructing Motors/Gimbals

By: Cameron

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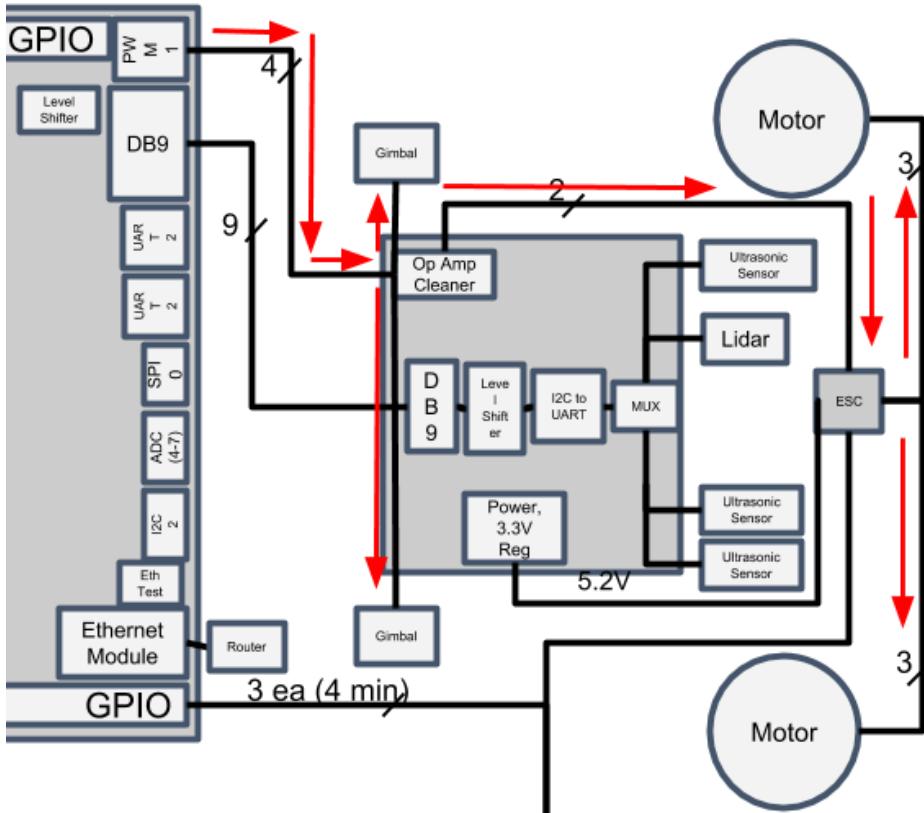
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Lidar Sensor Testing

By: Huishan

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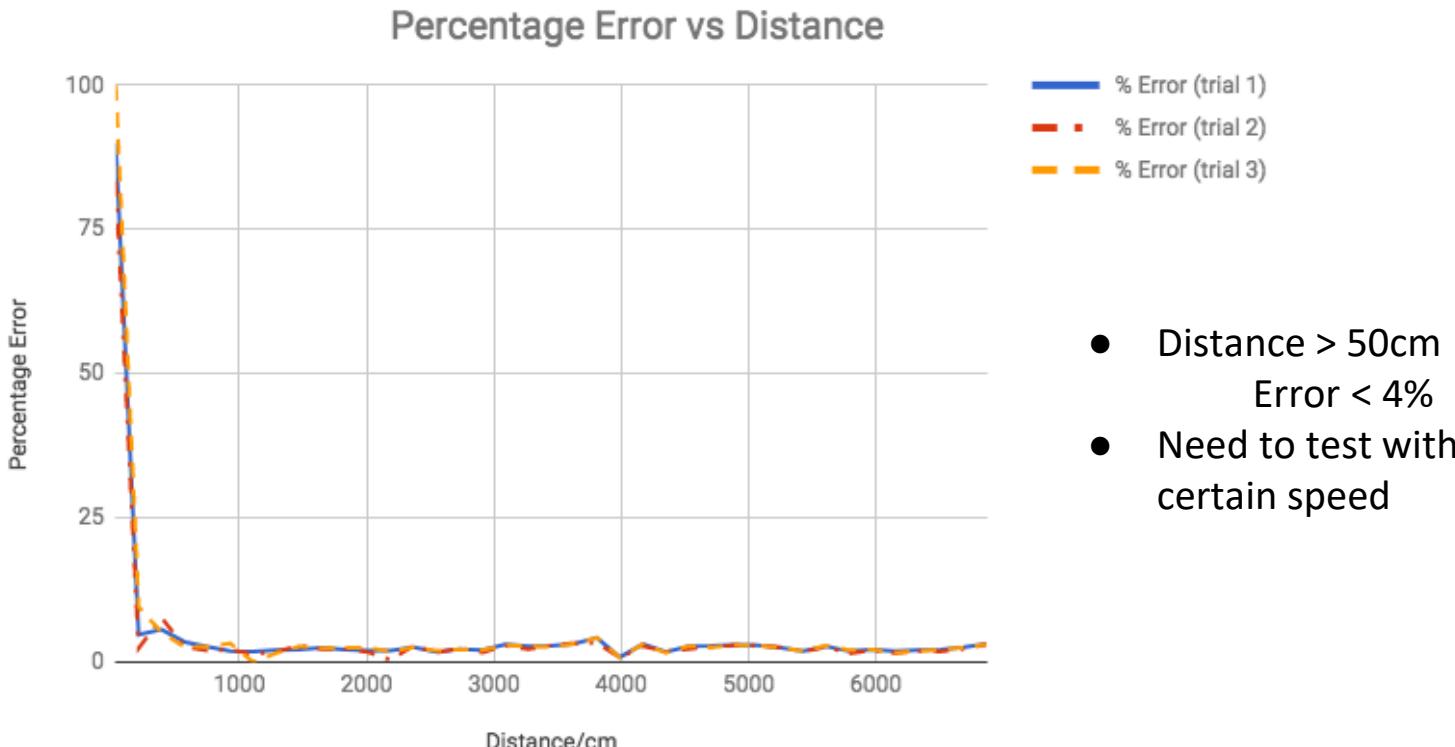
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Web App

By: Mark

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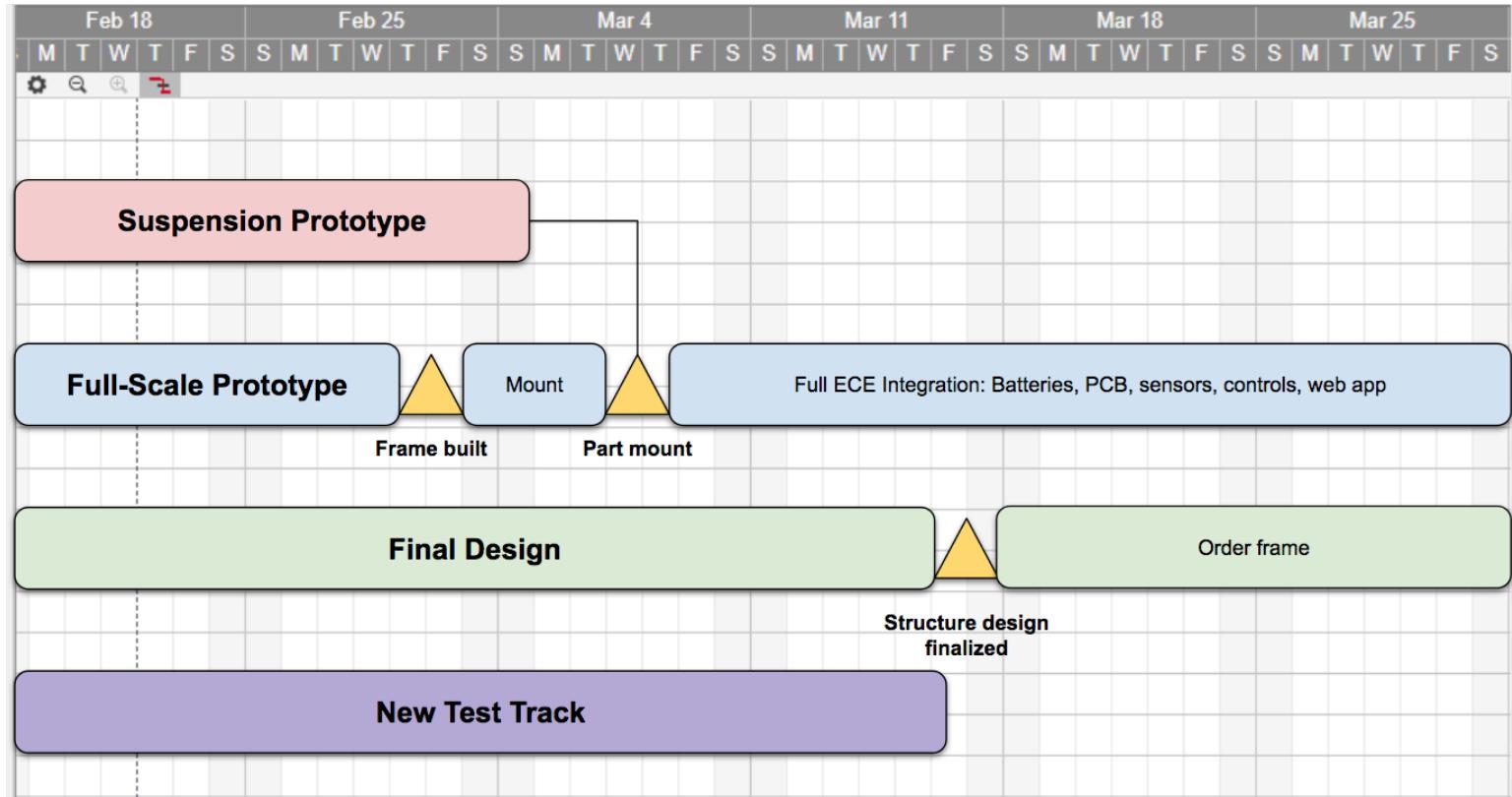
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Winter Quarter Schedule

By: Raymond



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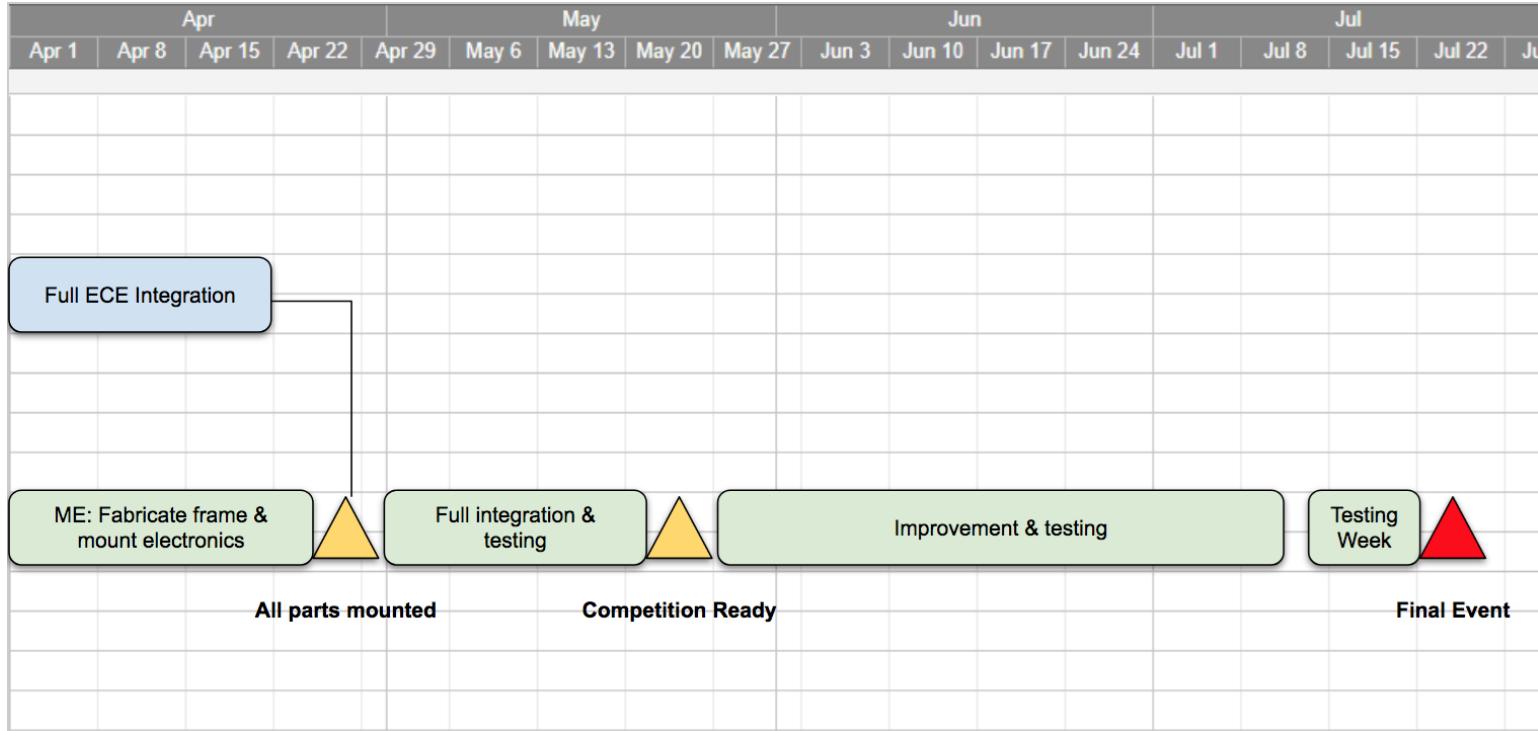
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Spring Quarter Schedule

By: Raymond



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Pod Cost Breakdown

By: Jack B.

Subteam	Item	Price
<u>Structures</u>	I-Beam Test Track	\$3,000
	Shell	\$5,000
Subsystems Cost: \$8,000	Battery Packs + BMS	\$600
	Sensors	\$560
	PCB (Manufactured by Laritech)	\$10,000 (Sponsored By Laritech)
<u>Stability and Braking</u>	Secondary Brakes	\$50
	Stability System	\$2000
Subsystems Cost: \$2050	Servos (MagLev Gimbaling)	\$1,200
	Maglev Engines	\$12,000
	Total Cost:	\$34,410

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Funds Overview

By: Jack B.

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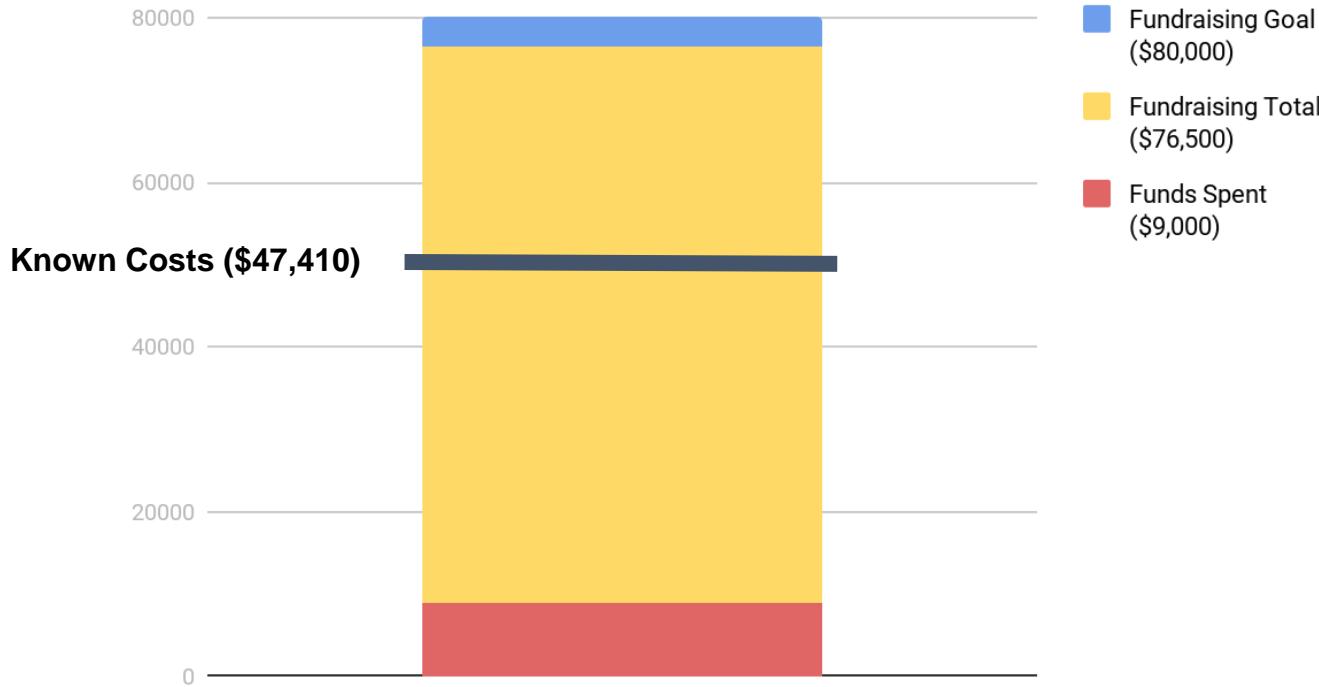
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Fundraising/Budget Total



Thank You!

By: Jack B.



Raytheon

Jonathan Siegel

Peter Carter

Rob 89' & Tracie Jupille

UC SANTA BARBARA
economics

Burrous Family

Ken and Nancy Goldsholl

John Jacobs

Paul Hoff

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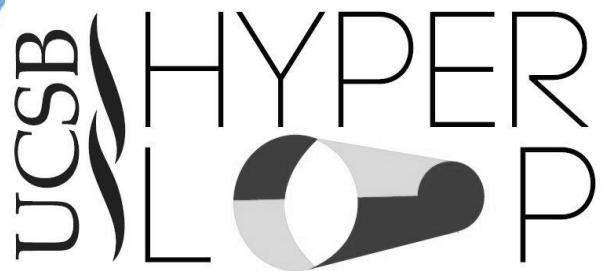
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Design Review
Feb. 23, 2018

Levitation Concept Assessment

By: Nate

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	HDK Module (small)	Arx Pax Module (large)
Motor Controller	0.03kg	1kg
Motor and Starm	0.38kg	6.8kg
Batteries	0.065kg	2.25kg
Thrust/Mass	3.53 N/kg	5.97 N/kg