O3FESA

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1 What is O3FESA?

O3FESA (Open-source Object-Oriented Finite Element Shell Analysis) is a software written in C++ using the objected-oriented programming paradigm.

2 Shell theories

2.1 First-order shear deformation theory

2.1.1 Displacement field

$$u_1(x, y, z, t) = u(x, y, t) + z\phi_x(x, y, t)$$
 (2.1a)

$$u_2(x, y, z, t) = v(x, y, t) + z\phi_y(x, y, t)$$
 (2.1b)

$$u_3(x, y, z, t) = w(x, y, t)$$
 (2.1c)

The displacement vector is defined as

$$q = \begin{cases} u \\ v \\ w \\ \phi_x \\ \phi_y \end{cases}$$
 (2.2)

2.1.2 Strain

The Green-Lagrange strain tensor is given by:

$$E_{jk} = \frac{1}{2} \left(\frac{\partial u_j}{\partial X_k} + \frac{\partial u_k}{\partial X_j} + \frac{\partial u_m}{\partial X_j} \frac{\partial u_m}{\partial X_k} \right), \qquad j, k, l = 1, 2, 3$$
 (2.3)

If the rotation of transverse normals are moderate, the strain-displacement relations in Eq. (2.3) simplifies to von Kármán strains:

$$\begin{cases}
\frac{\epsilon_{xx}}{\epsilon_{yy}} \\
\gamma_{xy}\\ \gamma_{xz}\\ \gamma_{yz}
\end{cases} = \begin{cases}
\frac{\partial u}{\partial x} + \frac{1}{2} \left(\frac{\partial w}{\partial x}\right)^{2} \\
\frac{\partial v}{\partial y} + \frac{1}{2} \left(\frac{\partial w}{\partial x}\right)^{2} \\
\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} + \frac{\partial w}{\partial x} \frac{\partial w}{\partial y}
\end{cases} + z \begin{cases}
\frac{\partial \phi_{x}}{\partial x} \\
\frac{\partial \phi_{y}}{\partial y} \\
\frac{\partial \phi_{y}}{\partial y}
\end{cases}$$

$$\frac{\partial \phi_{y}}{\partial y} + \frac{\partial \phi_{y}}{\partial x}$$

$$\frac{\partial w}{\partial x} + \phi_{x}$$

$$\frac{\partial w}{\partial x} + \phi_{y}$$

$$0$$

$$0$$

or in a compact form

$$\epsilon = \epsilon^0 + z\epsilon^1 \tag{2.5}$$

2.1.3 Principle of Virtual Work

The Principle of Virtual Work (PVW) states that the sum of internal and external virtual works must be zero:

$$\delta W^{\rm int} + \delta W^{\rm ext} = 0 \tag{2.6}$$

The virtual internal work is related to the virtual strain energy, given by

$$\delta W^{\rm int} = \int_{V} \delta \epsilon^{T} \sigma \, dV \tag{2.7}$$

The stress vector, σ , is defined as

$$\sigma = \begin{cases} \sigma_{xx} \\ \sigma_{yy} \\ \tau_{xy} \\ \tau_{xz} \\ \tau_{yz} \end{cases} = C \left(\epsilon - \epsilon_0 \right) \tag{2.8}$$

in which C is the material constitutive matrix and ϵ_0 is initial deformation vector. These relations are defined as

$$C = \begin{bmatrix} \bar{Q}_m & 0\\ 0 & \bar{Q}_s \end{bmatrix} \tag{2.9}$$

and

$$\epsilon_0 = \begin{Bmatrix} \alpha \\ 0 \end{Bmatrix} \Delta T \tag{2.10}$$

Substituting Eqs. (2.5) in Eq. (2.7) yields

$$\delta W^{\text{int}} = \int_{\Omega} \int_{-h/2}^{h/2} \left(\delta \epsilon^T \sigma + z \delta \epsilon^T \sigma \right) dz d\Omega$$
 (2.11)

Integrating (2.5) through the thickness, one obtains

$$\delta W^{\text{int}} = \int_{\Omega} \left(\delta \epsilon_m^T N + \delta \epsilon_b^T M + \delta \epsilon_s^T Q \right) d\Omega \tag{2.12}$$

where the strains components, stress and thermal resultants, respectively

$$\epsilon_{m} = \begin{cases} \frac{\partial u}{\partial x} + \frac{1}{2} \frac{\partial w^{2}}{\partial x} \\ \frac{\partial v}{\partial y} + \frac{1}{2} \left(\frac{\partial w}{\partial x} \right)^{2} \\ \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} + \frac{\partial w}{\partial x} \frac{\partial w}{\partial y} \end{cases}$$

$$\epsilon_{b} = \begin{cases} \frac{\partial \phi_{x}}{\partial x} \\ \frac{\partial \phi_{y}}{\partial y} \\ \frac{\partial \phi_{y}}{\partial y} + \frac{\partial \phi_{y}}{\partial x} \end{cases}$$

$$(2.13a)$$

$$\epsilon_{b} = \begin{cases} \frac{\partial \phi_{x}}{\partial x} \\ \frac{\partial \phi_{y}}{\partial y} \\ \frac{\partial \phi_{x}}{\partial y} + \frac{\partial \phi_{y}}{\partial x} \end{cases}$$
(2.13b)

$$\epsilon_{s} = \begin{cases} \frac{\partial w}{\partial x} + \phi_{x} \\ \frac{\partial w}{\partial x} + \phi_{y} \end{cases}$$
 (2.13c)

$$\begin{cases}
N \\
M \\
Q
\end{cases} = \begin{bmatrix}
A & B & 0 \\
B & D & 0 \\
0 & 0 & A_s
\end{bmatrix} \begin{cases}
\epsilon_m \\
\epsilon_b \\
\epsilon_s
\end{cases} - \begin{cases}
N_T \\
M_T \\
0
\end{cases} = \hat{C}\hat{\epsilon} - \hat{\Psi} \tag{2.14}$$

The membrane strain, Eq. (2.13a), can be considered as a linear and nonlinear parts

$$\epsilon_{m} = \epsilon_{m}^{l} + \epsilon_{m}^{nl} = \begin{cases} \frac{\partial u}{\partial x} \\ \frac{\partial v}{\partial y} \\ \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \end{cases} + \begin{cases} \frac{1}{2} \left(\frac{\partial w}{\partial x}\right)^{2} \\ \frac{1}{2} \left(\frac{\partial w}{\partial x}\right)^{2} \\ \frac{\partial w}{\partial x} \frac{\partial w}{\partial y} \end{cases}$$
(2.15)

Furthermore, the strains in Eq. (2.13) can written as a function of a derivative

operators, that is,

$$\epsilon_{m}^{l} = \begin{bmatrix} \frac{\partial}{\partial x} & 0 & 0 & 0 & 0 \\ 0 & \frac{\partial}{\partial y} & 0 & 0 & 0 \\ \frac{\partial}{\partial y} & \frac{\partial}{\partial x} & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} u \\ v \\ w \\ \phi_{x} \\ \phi_{y} \end{bmatrix} = \mathcal{L}_{m}q$$

$$(2.16a)$$

$$\epsilon_{m}^{nl} = \frac{1}{2} \begin{bmatrix} \frac{\partial w}{\partial x} & 0\\ 0 & \frac{\partial w}{\partial y} \\ \frac{\partial w}{\partial y} & \frac{\partial w}{\partial x} \end{bmatrix} \begin{bmatrix} 0 & 0 & \frac{\partial}{\partial x} & 0 & 0\\ 0 & 0 & \frac{\partial}{\partial y} & 0 & 0 \end{bmatrix} \begin{bmatrix} u\\ v\\ w\\ \phi_{x}\\ \phi_{y} \end{bmatrix} = \frac{1}{2} \Theta \mathcal{L}_{\theta} q \qquad (2.16b)$$

$$\epsilon_{b} = \begin{bmatrix} 0 & 0 & 0 & \frac{\partial}{\partial x} & 0 \\ 0 & 0 & 0 & 0 & \frac{\partial}{\partial y} \\ 0 & 0 & 0 & \frac{\partial}{\partial y} & \frac{\partial}{\partial x} \end{bmatrix} \begin{bmatrix} u \\ v \\ w \\ \phi_{x} \\ \phi_{y} \end{bmatrix} = \mathcal{L}_{b} q$$
 (2.16c)

$$\epsilon_{s} = \begin{bmatrix} 0 & 0 & \frac{\partial}{\partial x} & 1 & 0 \\ 0 & 0 & \frac{\partial}{\partial y} & 0 & 1 \end{bmatrix} \begin{cases} u \\ v \\ w \\ \phi_{x} \\ \phi_{y} \end{cases} = \mathcal{L}_{s} q$$

$$(2.16d)$$

With exception of Eq. (2.16b), that depends on the displacement, the first variation of Eq. (2.16) has the form:

$$\delta \epsilon_* = \mathcal{L}_* \delta q \tag{2.17}$$

while

$$\delta \epsilon_m^{nl} = \Theta \mathcal{L}_\theta \delta q \tag{2.18}$$

Now, the virtual internal work can now be written as

$$\delta W^{\rm int} = \int_{\Omega} \delta \hat{e}^T \hat{\sigma} d\Omega \tag{2.19}$$

in which

$$\delta \hat{\epsilon} = \begin{bmatrix} \mathcal{L}_m \\ \mathcal{L}_b \\ \mathcal{L}_s \end{bmatrix} \delta q + \begin{bmatrix} \Theta \mathcal{L}_\theta \\ 0 \\ 0 \end{bmatrix} \delta q$$
 (2.20a)

$$\hat{\sigma} = \begin{cases} N \\ M \\ Q \end{cases} \tag{2.20b}$$

The virtual external work can be defined as

$$\delta W^{\text{ext}} = -\left(\int_{\Omega} \delta q^T f \, d\Omega + \delta q^T g\right) \tag{2.21}$$

where the first term represents the external load due to loading acting on and area Ω and second one the work done by concentrated loading, where f and g are the distributed load vector and nodal point load vector, respectively. The PVW may be now defined as

$$\int_{\Omega} \delta \hat{\epsilon}^T \hat{\sigma} d\Omega = \int_{\Omega} \delta q^T f d\Omega + \sum_{j} \delta q^j g^j$$
 (2.22)

2.1.4 Finite element discretization

The displacement field within an element, Eq. (2.2), is given by

$$q^{\mathcal{E}} = \hat{N}\hat{q}^{\mathcal{E}} \tag{2.23}$$

where

$$\hat{N} = \begin{bmatrix} \tilde{N} & \tilde{0} & \tilde{0} & \tilde{0} & \tilde{0} \\ \tilde{0} & \tilde{N} & \tilde{0} & \tilde{0} & \tilde{0} \\ \tilde{0} & \tilde{0} & \tilde{N} & \tilde{0} & \tilde{0} \\ \tilde{0} & \tilde{0} & \tilde{0} & \tilde{N} & \tilde{0} \\ \tilde{0} & \tilde{0} & \tilde{0} & \tilde{0} & \tilde{N} \end{bmatrix}$$
(2.24)

in which $\tilde{N} = [\tilde{N}_1, \tilde{N}_2, \cdots, \tilde{N}_n]$ is the shape functions vector, $\tilde{0} = [0]_n$ is a null vector with n elements, in which n is the number of nodes. The local element

displacement vector $\hat{q}^{\mathcal{E}}$ is given by

$$\hat{q}^{\mathcal{E}} = \begin{cases} \hat{u}^{\mathcal{E}} \\ \hat{v}^{\mathcal{E}} \\ \hat{w}^{\mathcal{E}} \\ \hat{\phi}^{\mathcal{E}}_{x} \\ \hat{\phi}^{\mathcal{E}}_{y} \end{cases}$$
 (2.25)

where, for the sake of generality, $\hat{\gamma} = [\gamma_1, \gamma_2, \cdots, \gamma_n]^T$ are nodal displacements. Substituting Eq. (2.23) in Eq. (2.22) yields

$$\delta \hat{q}^{\mathcal{E}^T} \left(\int_{\Omega^{\mathcal{E}}} \mathbf{B}^{\mathcal{E}^T} \hat{\sigma}^{\mathcal{E}} \, \mathrm{d}\Omega^{\mathcal{E}} - \hat{f}^{\mathcal{E}} \right) = 0 \tag{2.26}$$

where $B^{\mathcal{E}}$ is the strain-displacement matrix

$$B^{\mathcal{E}} = \begin{bmatrix} B_m \\ B_b \\ B_s \end{bmatrix} + \begin{bmatrix} \Theta^{\mathcal{E}} B_{\theta} \\ 0 \\ 0 \end{bmatrix} = B^l + B^{nl}^{\mathcal{E}}$$
 (2.27)

 $\hat{\sigma}^{\mathcal{E}}$ is the resultant internal forces

$$\hat{\sigma}^{\mathcal{E}} = \hat{C}^{\mathcal{E}} \left(B^{l} + \frac{1}{2} B^{nl}^{\mathcal{E}} \right) \hat{q}^{\mathcal{E}} - \Psi^{\mathcal{E}}$$
 (2.28)

and $\hat{f}^{\mathcal{E}}$ is the equivalent nodal load vector

$$\hat{f}^{\mathcal{E}} = \int_{\Omega^{\mathcal{E}}} \hat{N} f^{\mathcal{E}} \, \mathrm{d}\Omega^{\mathcal{E}} \tag{2.29}$$

With regard to Eq. (2.16b), one notices that $\Theta^{\mathcal{E}}$ depends on the displacement within each element, and so the nonlinear part of the strain-displacement matrix.

The strain-displacement matrix $B^{\mathcal{E}}$ is written in terms of its membrane component B_m ,

$$B_{m} = \begin{bmatrix} \frac{\partial \tilde{N}}{\partial x} & 0 & 0 & 0 & 0 \\ 0 & \frac{\partial \tilde{N}}{\partial y} & 0 & 0 & 0 \\ \frac{\partial \tilde{N}}{\partial y} & \frac{\partial \tilde{N}}{\partial x} & 0 & 0 & 0 \end{bmatrix}$$
(2.30)

bending component B_b

$$B_{b} = \begin{bmatrix} 0 & 0 & 0 & \frac{\partial \tilde{N}}{\partial x} & 0 \\ 0 & 0 & 0 & 0 & \frac{\partial \tilde{N}}{\partial y} \\ 0 & 0 & 0 & \frac{\partial \tilde{N}}{\partial y} & \frac{\partial \tilde{N}}{\partial x} \end{bmatrix}$$
(2.31)

shear component B_s

$$\boldsymbol{B}_{s} = \begin{bmatrix} 0 & 0 & \frac{\partial \tilde{N}}{\partial x} & N & 0 \\ 0 & 0 & \frac{\partial \tilde{N}}{\partial y} & 0 & N \end{bmatrix}$$
 (2.32)

and gradient component B_{θ}

$$\boldsymbol{B}_{\theta} = \begin{bmatrix} 0 & 0 & \frac{\partial \tilde{N}}{\partial x} & 0 & 0 \\ 0 & 0 & \frac{\partial \tilde{N}}{\partial y} & 0 & 0 \end{bmatrix}$$
 (2.33)

Thus, the nonlinear equilibrium equation within the element become

$$\psi^{\mathcal{E}} = \int_{\Omega^{\mathcal{E}}} B^{\mathcal{E}^T} \hat{\sigma}^{\mathcal{E}} d\Omega^{\mathcal{E}} - \hat{f}^{\mathcal{E}} = 0$$
 (2.34)

which is also know as residual. The first term of Eq. (2.34) can be defined as

$$\int_{\Omega^{\mathcal{E}}} B^{\mathcal{E}^T} \hat{\sigma}^{\mathcal{E}} d\Omega^{\mathcal{E}} = K^{\mathcal{E}} \hat{q}^{\mathcal{E}} - \hat{\Psi}^{\mathcal{E}}$$
(2.35)

where $K^{\mathcal{E}}$ is the standard stiffness matrix

$$K^{\mathcal{E}} = \int_{\Omega^{\mathcal{E}}} \left(B^{l} + B^{nl^{\mathcal{E}}} \right)^{T} \hat{C}^{\mathcal{E}} \left(B^{l} + \frac{1}{2} B^{nl^{\mathcal{E}}} \right) d\Omega^{\mathcal{E}}$$
 (2.36)

and $\hat{\Psi}^{\mathcal{E}}$ is a thermal load vector

$$\hat{\Psi}^{\mathcal{E}} = \int_{\Omega^{\mathcal{E}}} \left(B^{l} + B^{nl}^{\mathcal{E}} \right)^{T} \Psi^{\mathcal{E}} d\Omega^{\mathcal{E}}$$
 (2.37)

The stiffness matrix can be rewritten as a sum of linear and nonlinear stiffness matrices:

$$K^{\mathcal{E}} = K^{l^{\mathcal{E}}} + K^{nl^{\mathcal{E}}} \tag{2.38}$$

where

$$K^{l^{\mathcal{E}}} = \int_{\Omega^{\mathcal{E}}} B_{m}^{T} A^{\mathcal{E}} B_{m} \, d\Omega^{\mathcal{E}} + \int_{\Omega^{\mathcal{E}}} B_{m}^{T} B^{\mathcal{E}} B_{b} \, d\Omega^{\mathcal{E}} + \int_{\Omega^{\mathcal{E}}} B_{b}^{T} B^{\mathcal{E}} B_{m} \, d\Omega^{\mathcal{E}}$$

$$+ \int_{\Omega^{\mathcal{E}}} B_{b}^{T} D^{\mathcal{E}} B_{b} \, d\Omega^{\mathcal{E}} + \int_{\Omega^{\mathcal{E}}} B_{s}^{T} A_{s}^{\mathcal{E}} B_{s} \, d\Omega^{\mathcal{E}}$$

$$(2.39)$$

and

$$K^{nl^{\mathcal{E}}} = \frac{1}{2} \int_{\Omega^{\mathcal{E}}} B_{m}^{T} A^{\mathcal{E}} \Theta^{\mathcal{E}} B_{\theta} \, d\Omega^{\mathcal{E}} + \frac{1}{2} \int_{\Omega^{\mathcal{E}}} B_{b}^{T} B^{\mathcal{E}} \Theta^{\mathcal{E}} B_{\theta} \, d\Omega^{\mathcal{E}}$$

$$+ \int_{\Omega^{\mathcal{E}}} B_{\theta}^{T} \Theta^{T^{\mathcal{E}}} A^{\mathcal{E}} B_{m} \, d\Omega^{\mathcal{E}} + \int_{\Omega^{\mathcal{E}}} B_{\theta}^{T} \Theta^{T^{\mathcal{E}}} B^{\mathcal{E}} B_{b} \, d\Omega^{\mathcal{E}}$$

$$+ \frac{1}{2} \int_{\Omega^{\mathcal{E}}} B_{\theta}^{T} \Theta^{T^{\mathcal{E}}} A^{\mathcal{E}} \Theta^{\mathcal{E}} B_{\theta} \, d\Omega^{\mathcal{E}}$$

$$(2.40)$$

In compact forms, the linear and nonlinear stiffness matrix are given by,

$$K^{l^{\mathcal{E}}} = K^{\mathcal{E}}_{pp} + K^{\mathcal{E}}_{ph} + K^{\mathcal{E}}_{hp} + K^{\mathcal{E}}_{hh} + K^{\mathcal{E}}_{ss}$$
 (2.41)

and

$$K^{nl}^{\mathcal{E}} = \frac{1}{2} K_{p\theta}^{\mathcal{E}} + \frac{1}{2} K_{b\theta}^{\mathcal{E}} + K_{\theta p}^{\mathcal{E}} + K_{\theta b}^{\mathcal{E}} + \frac{1}{2} K_{\theta \theta}^{\mathcal{E}}$$
 (2.42)

respectively. The thermal load vector can be written as

$$\hat{\Psi}^{\mathcal{E}} = \int_{\Omega^{\mathcal{E}}} B_{m}^{T} N_{T}^{\mathcal{E}} d\Omega^{\mathcal{E}} + \int_{\Omega^{\mathcal{E}}} B_{b}^{T} M_{T}^{\mathcal{E}} d\Omega^{\mathcal{E}} + \int_{\Omega^{\mathcal{E}}} B_{\theta}^{T} \Theta^{T\mathcal{E}} N_{T}^{\mathcal{E}} d\Omega^{\mathcal{E}}$$
(2.43)

The last term of Eq. (2.43) may be rewritten as

$$\int_{\Omega^{\mathcal{E}}} B_{\theta}^{T} \Theta^{T^{\mathcal{E}}} N_{T}^{\mathcal{E}} d\Omega^{\mathcal{E}} = \left(\int_{\Omega^{\mathcal{E}}} B_{\theta}^{T} \begin{bmatrix} N_{T}^{1} & N_{T}^{3} \\ N_{T}^{3} & N_{T}^{2} \end{bmatrix} B_{\theta} d\Omega^{\mathcal{E}} \right) \hat{q}^{\mathcal{E}} = K_{T}^{\mathcal{E}}$$
(2.44)

Thus, one may define the thermal load vector as

$$\hat{\Psi}^{\mathcal{E}} = \hat{f}_{N_T}^{\mathcal{E}} + \hat{f}_{M_T}^{\mathcal{E}} + K_T^{\mathcal{E}}$$
 (2.45)

The residual can now be rewritten as

$$\psi^{\mathcal{E}} = \bar{K}^{\mathcal{E}} \hat{q}^{\mathcal{E}} - \bar{f}^{\mathcal{E}} \tag{2.46}$$

where $\bar{K}^{\mathcal{E}}$ is given by

$$\bar{K}^{\mathcal{E}} = K^{l^{\mathcal{E}}} + K^{nl^{\mathcal{E}}} - K_{T}^{\mathcal{E}}$$
(2.47)

and

$$\bar{f}^{\mathcal{E}} = \hat{f}^{\mathcal{E}} + \hat{f}^{\mathcal{E}}_{N_T} + \hat{f}^{\mathcal{E}}_{M_T} \tag{2.48}$$

2.1.5 Solution to nonlinear equilibrium equations

An approximation of the residual (unbalanced forces) may be obtained by equating to zero the linearized Taylor's series expansion of $\psi^{\mathcal{E}}$ in the neighborhood of an initial estimate of the total displacement, $\hat{q}_{i}^{\mathcal{E}}$, as

$$\psi^{\mathcal{E}}(\hat{q}_{i+1}^{\mathcal{E}}) \simeq \psi^{\mathcal{E}}(\hat{q}_{i}^{\mathcal{E}}) + T^{\mathcal{E}}\Delta\hat{q}_{i}^{\mathcal{E}} = 0 \tag{2.49}$$

where $T^{\mathcal{E}}$ is known as the tangent stiffness matrix. It is evaluated as

$$T^{\mathcal{E}} = \frac{\partial \psi^{\mathcal{E}}(\hat{q}_i^{\mathcal{E}})}{\partial \hat{q}_i^{\mathcal{E}}}$$
 (2.50)

The tangent matrix may be written as

$$T^{\mathcal{E}} = \begin{bmatrix} \frac{\partial \psi_{1}^{\mathcal{E}}}{\partial \hat{q}_{1}^{\mathcal{E}}} & \frac{\partial \psi_{1}^{\mathcal{E}}}{\partial \hat{q}_{2}^{\mathcal{E}}} & \dots & \frac{\partial \psi_{1}^{\mathcal{E}}}{\partial \hat{q}_{m}^{\mathcal{E}}} \\ \frac{\partial \psi_{2}^{\mathcal{E}}}{\partial \hat{q}_{1}^{\mathcal{E}}} & \frac{\partial \psi_{2}^{\mathcal{E}}}{\partial \hat{q}_{2}^{\mathcal{E}}} & \dots & \frac{\partial \psi_{2}^{\mathcal{E}}}{\partial \hat{q}_{m}^{\mathcal{E}}} \\ \vdots & \vdots & \ddots & \vdots \\ \frac{\partial \psi_{m}^{\mathcal{E}}}{\partial \hat{q}_{1}^{\mathcal{E}}} & \frac{\partial \psi_{m}^{\mathcal{E}}}{\partial \hat{q}_{2}^{\mathcal{E}}} & \dots & \frac{\partial \psi_{m}^{\mathcal{E}}}{\partial \hat{q}_{m}^{\mathcal{E}}} \\ \frac{\partial \psi_{m}^{\mathcal{E}}}{\partial \hat{q}_{1}^{\mathcal{E}}} & \frac{\partial \psi_{m}^{\mathcal{E}}}{\partial \hat{q}_{2}^{\mathcal{E}}} & \dots & \frac{\partial \psi_{m}^{\mathcal{E}}}{\partial \hat{q}_{m}^{\mathcal{E}}} \end{bmatrix}$$

$$(2.51)$$

where m is the element number of degrees of freedom. For convenience, the tangent matrix is rewritten with respect to Eq. (2.34) as

$$T^{\mathcal{E}} = \int_{\Omega^{\mathcal{E}}} \left(B^{\mathcal{E}} \frac{\partial \hat{\sigma}^{\mathcal{E}}}{\partial \hat{q}^{\mathcal{E}}} + \frac{\partial B^{\mathcal{E}}}{\partial \hat{q}^{\mathcal{E}}} \hat{\sigma}^{\mathcal{E}} \right) d\Omega^{\mathcal{E}}$$
 (2.52)

The first term of Eq. (2.52) may be rewritten as (see Eq. (2.28))

$$\int_{\Omega^{\mathcal{E}}} B^{\mathcal{E}^{T}} \frac{\partial \hat{\sigma}^{\mathcal{E}}}{\partial \hat{q}^{\mathcal{E}}} d\Omega^{\mathcal{E}} = \int_{\Omega^{\mathcal{E}}} B^{\mathcal{E}^{T}} \hat{C}^{\mathcal{E}} \frac{\partial}{\partial \hat{q}^{\mathcal{E}}} \left[\left(B^{l} + \frac{1}{2} B^{nl^{\mathcal{E}}} \right) \hat{q}^{\mathcal{E}} - \Psi^{\mathcal{E}} \right] d\Omega^{\mathcal{E}}
= \int_{\Omega^{\mathcal{E}}} B^{\mathcal{E}^{T}} \hat{C}^{\mathcal{E}} B^{\mathcal{E}} d\Omega^{\mathcal{E}}
= K^{l^{\mathcal{E}}} + K^{nl_{2}}{}^{\mathcal{E}}$$
(2.53)

where

$$K^{nl_2}{}^{\mathcal{E}} = K^{\mathcal{E}}_{p\theta} + K^{\mathcal{E}}_{b\theta} + K^{\mathcal{E}}_{\theta\rho} + K^{\mathcal{E}}_{\theta\theta} + K^{\mathcal{E}}_{\theta\theta}$$
 (2.54)

The second term of Eq. (2.52) is written as

$$\int_{\Omega^{\varepsilon}} \frac{\partial B^{\varepsilon^{T}}}{\partial \hat{q}^{\varepsilon}} \hat{\sigma}^{\varepsilon} d\Omega^{\varepsilon} = \int_{\Omega^{\varepsilon}} B_{\theta}^{T} \frac{\partial \Theta^{\varepsilon^{T}}}{\partial \hat{q}^{\varepsilon}} N^{\varepsilon} d\Omega^{\varepsilon}
= \int_{\Omega^{\varepsilon}} B_{\theta}^{T} \begin{bmatrix} \frac{\partial}{\partial \hat{q}^{\varepsilon}} \left(\frac{\partial w^{\varepsilon}}{\partial x} \right) N_{1}^{\varepsilon} + \frac{\partial}{\partial \hat{q}^{\varepsilon}} \left(\frac{\partial w^{\varepsilon}}{\partial y} \right) N_{3}^{\varepsilon} \\ \frac{\partial}{\partial \hat{q}^{\varepsilon}} \left(\frac{\partial w^{\varepsilon}}{\partial x} \right) N_{3}^{\varepsilon} + \frac{\partial}{\partial \hat{q}^{\varepsilon}} \left(\frac{\partial w^{\varepsilon}}{\partial y} \right) N_{2}^{\varepsilon} \end{bmatrix} d\Omega^{\varepsilon}
= \int_{\Omega^{\varepsilon}} B_{\theta}^{T} \begin{bmatrix} N_{1}^{\varepsilon} & N_{3}^{\varepsilon} \\ N_{3}^{\varepsilon} & N_{2}^{\varepsilon} \end{bmatrix} \begin{cases} \frac{\partial}{\partial \hat{q}^{\varepsilon}} \left(\frac{\partial w^{\varepsilon}}{\partial x} \right) \\ \frac{\partial}{\partial \hat{q}^{\varepsilon}} \left(\frac{\partial w^{\varepsilon}}{\partial y} \right) \end{cases} d\Omega^{\varepsilon}$$

$$(2.55)$$

The derivative terms may be evaluated as

$$\frac{\partial}{\partial \hat{q}_{j}^{\mathcal{E}}} \left[\frac{\partial \left(\tilde{N}^{\alpha} \hat{w}_{\alpha}^{\mathcal{E}} \right)}{\partial \lambda} \right] = \begin{cases} \frac{\partial \tilde{N}}{\partial \lambda}, & \text{if } \hat{q}_{j}^{\mathcal{E}} = \hat{w}_{\alpha}^{\mathcal{E}} \\ 0, & \text{otherwise} \end{cases}$$
 (2.56)

which is the definition of the strain-displacement matrix B_{θ} . Thus,

$$\int_{\Omega^{\varepsilon}} \frac{\partial B^{\varepsilon^{T}}}{\partial \hat{q}^{\varepsilon}} \hat{\sigma}^{\varepsilon} = \int_{\Omega^{\varepsilon}} B_{\theta}^{T} \begin{bmatrix} N_{1}^{\varepsilon} & N_{3}^{\varepsilon} \\ N_{3}^{\varepsilon} & N_{2}^{\varepsilon} \end{bmatrix} B_{\theta} d\Omega^{\varepsilon}$$

$$= K^{\sigma \varepsilon} \tag{2.57}$$

Finally, the tangent stiffness matrix is given by

$$T^{\mathcal{E}} = K^{l^{\mathcal{E}}} + K^{nl_2^{\mathcal{E}}} + K^{\sigma \mathcal{E}}$$
 (2.58)