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			File	
Name :	sun_planet_2	_narrow		
Changed by:	Angelos	on: 25.12.2020	at: 12:51:16	

#### Important hint: At least one warning has occurred during the calculation:

1-> Mesh load factor Ky = 1.200000

This input is unususal and will result in faulty results.

Please check if you have entered it deliberately!

2-> Gear pair 1 - 2:

The dynamic factor Kv is very high.

The formulae in the standard probably do not suit this case.

3-> The circumferential speed is very high (45.0206 m/s)!

You have to take adequate action to

guarantee proper lubrication.

4-> Notice concerning gear 1:

Dimension over pins is not measurable (facewidth is too small)!

5-> Notice concerning gear 2:

Dimension over pins is not measurable (facewidth is too small)!

# Calculation of a helical-toothed cylindrical gear pair

Drawing or article number:

 Gear 1:
 0.000.0

 Gear 2:
 0.000.0

Calculation method ISO 6336:2019

----- Gear 1 ----- Gear 2 --

Power (kW) [P] 156.255 Speed (1/min) 5024.0 4220.2 [n] Torque (Nm) 297.0 353.6 [T] Application factor 1.75 [KA] Distribution factor [Kγ] 1.20 Required service life (h) [H] 10000.00 Gear driving (+) / driven (-)

Working flank gear 1: Right flank

Gear 1 direction of rotation: Clockwise

Tooth geometry and material

Geometry calculation according to ISO 21771:2007

------ Gear 1 ------ Gear 2 --

Center distance (mm) [a] 196.000

Center distance tolerance ISO 286:2010 Measure js7



Normal module (mm)	[mn]	8.	0000
Normal pressure angle (°)	[ɑn]	25.	0000
Helix angle at reference circle (°)	[β]	11.	0000
Number of teeth	[z]	21	25
Facewidth (mm)	[b]	28.00	26.00
Hand of gear	right	left	
Accuracy grade	[Q-ISO 1328:2013]	A6	A6
Inner diameter (mm)	[di]	0.00	0.00
Inner diameter of gear rim (mm)	[dbi]	0.00	0.00

#### Material

#### Gear 1

18CrNiMo7-6, Case-carburized steel, case-hardened

ISO 6336-5 Figure 9/10 (MQ), Core hardness >=25HRC Jominy J=12mm<HRC28

#### Gear 2

18CrNiMo7-6, Case-carburized steel, case-hardened

ISO 6336-5 Figure 9/10 (MQ), Core hardness >=25HRC Jominy J=12mm<HRC28

O for body		Gear 1	Gear 2	
Surface hardness	7NT 1NAT	HRC 61		HRC 61
Material treatment according to ISO 6336:2006 Normal, life factor			420.6	20
Fatigue strength, tooth root stress (N/mm²)	[σFlim]	430.00	430.0	
Fatigue strength for Hertzian pressure (N/mm²)	[σHlim]	1500.00	1500.0	
Tensile strength (N/mm²)	[σB]	1200.00	1200.0	
Yield point (N/mm²)	[σS]	850.00 206000	850.0	50
Young's modulus (N/mm²) Poisson's ratio	[E]	0.300	206000	200
	[v]			300
Roughness average value DS, flank (µm)	[RAH]	0.60	0.6	
Roughness average value DS, root (µm)	[RAF]	3.00	3.0	
Mean roughness height, Rz, flank (μm)	[RZH]	4.80	4.8	
Mean roughness height, Rz, root (μm)	[RZF]	20.00	20.0	J0
Gear reference profile  1: Reference profile Dedendum coefficient Root radius factor Addendum coefficient Tip radius factor Protuberance height coefficient Protuberance angle Tip form height coefficient Ramp angle	1.25 / 0.38 / 1.0 IS [hfP*] [pfP*] [haP*] [paP*] [hprP*] [qprP] [hFaP*] [αKP]	SO 53:1998 Pr	1.250	(ρfPmax*= 0.318)
	not topping			
Gear reference profile 2:				
Reference profile	1.25 / 0.38 / 1.0 [	SO 53:1998 Pr	ofil A	
Dedendum coefficient	[hfP*]		1.250	
Root radius factor	[ρfP*]		0.380	(ρfPmax*= 0.318)
Addendum coefficient	[haP*]		1.000	
Tip radius factor	[ρaΡ*]		0.000	
Protuberance height coefficient	[hprP*]		0.000	
Protuberance angle	[aprP]		0.000	
Tip form height coefficient	[hFaP*]		0.000	



70.000

Ramp angle  $\left[\alpha KP\right]$  0.000

not topping

# Information on final machining

Dedendum reference profile	[hfP*]	1.250	1.250
Tooth root radius Refer. profile	[ρfP*]	0.380	0.380
Addendum Reference profile	[haP*]	1.000	1.000
Protuberance height coefficient	[hprP*]	0.000	0.000
Protuberance angle (°)	[aprP]	0.000	0.000
Tip form height coefficient	[hFaP*]	0.000	0.000
Ramp angle (°)	[αKP]	0.000	0.000

Type of profile modification: none (only running-in)

Tip relief by running in ( $\mu$ m) [Ca L/R] 2.0 / 2.0 2.0 / 2.0

[TS]

Lubrication typeOil injection lubricationType of oilISO-VG 220Lubricant baseMineral-oil baseOil nominal kinematic viscosity at 40°C (mm²/s)[v40]220.00Oil nominal kinematic viscosity at 100°C (mm²/s)[v100]17.50Specific density at 15°C (kg/dm³)[p]0.895

#### Gear pair

Oil temperature (°C)

Overall transmission ratio	[itot]	-1.190
Gear ratio	[iiOt] [u]	1.190
Transverse module (mm)		8.150
,	[mt]	25.409
Transverse pressure angle (°)	[at]	
Working pressure angle (°)	[awt]	30.250
M 1: (2)	[awt.e/i]	30.262 / 30.239
Working pressure angle at normal section (°)	[ɑwn]	29.748
Helix angle at operating pitch circle (°)	[βw]	11.489
Base helix angle (°)	[βb]	9.958
Reference center distance (mm)	[ad]	187.444
Pitch on reference circle (mm)	[pt]	25.603
Base pitch (mm)	[pbt]	23.126
Transverse pitch on contact-path (mm)	[pet]	23.126
Sum of profile shift coefficients	[Σxi]	1.1672
Transverse contact ratio	[εα]	1.190
Transverse contact ratio with allowances	[εα.e/m/i]	1.192 / 1.188 / 1.185
Overlap ratio	[εβ]	0.197
Total contact ratio	[εγ]	1.387
Total contact ratio with allowances	[εγ.e/m/i]	1.389 / 1.386 / 1.382
Length of path of contact (mm)	[ga, e/i]	27.518 ( 27.564 / 27.395 )
Length T1-A (mm)	[T1A]	31.654 ( 31.608 / 31.738 )
Length T1-B (mm)	[T1B]	36.045 ( 36.045 / 36.007 )
Length T1-C (mm)	[T1C]	45.077 ( 45.056 / 45.098 )
Length T1-D (mm)	[T1D]	54.780 ( 54.734 / 54.865 )
Length T1-E (mm)	[T1E]	59.172 ( 59.172 / 59.134 )
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Length T2-A (mm)	[T2A]	67.087 ( 67.087 / 67.048 )
Length T2-B (mm)	[T2B]	62.695 ( 62.650 / 62.779 )
Length T2-C (mm)	[T2C]	53.663 ( 53.639 / 53.688 )
Length T2-D (mm)	[T2D]	43.960 ( 43.960 / 43.921 )
Length T2-E (mm)	[T2E]	39.569 ( 39.523 / 39.653 )
	[]	,
Length T1-T2 (mm)	[T1T2]	98.741 ( 98.695 / 98.786 )
	[=]	30( 30 30
Minimal length of contact line (mm)	[Lmin]	26.398
Gear 1		
Lead height (mm)	[pz]	2766.052
Axial pitch (mm)	[px]	131.717
Profile shift coefficient	[x]	0.5690
Tooth thickness, arc, in module	[sn*]	2.1015
, ,		
Tip alteration (mm)	[k*mn]	-0.781
Reference diameter (mm)	[d]	171.144
Base diameter (mm)	[db]	154.589
Tip diameter (mm)	[da]	194.686
(mm)	[da.e/i]	194.686 / 194.640
Tip diameter allowances (mm)	[Ada.e/i]	0.000 / -0.046
Tip form diameter (mm)	[dFa]	194.686
(mm)	[dFa.e/i]	194.686 / 194.640
Root diameter (mm)	[df]	160.248
Generating Profile shift coefficient	[xE.e/i]	0.5563/ 0.5496
Generated root diameter with xE (mm)		160.045 / 159.937
	[df.e/i]	
Root form diameter (mm)	[dFf]	164.495
(mm)	[dFf.e/i]	164.334 / 164.249
Involute length (mm)	[l_dFa-l_dFf]	17.537
Addendum, m <sub>n</sub> (h <sub>aP</sub> *+x+k) (mm)	[ha]	11.771
(mm)	[ha.e/i]	11.771 / 11.748
Dedendum (mm)	[hf=mn*(hfP*-x)]	5.448
(mm)	[hf.e/i]	5.550 / 5.603
Tooth height (mm)	[h]	17.219
Virtual gear no. of teeth	[zn]	22.053
Normal tooth thickness at tip circle (mm)	[san]	3.700
(mm)	[san.e/i]	3.626 / 3.536
Normal tooth thickness at tip form circle (mm)	[sFan]	3.700
(mm)	[sFan.e/i]	3.626 / 3.536
Normal space width at root circle (mm)	[efn]	3.858
(mm)	[efn.e/i]	3.888 / 3.904
Gear 2		
lood beinkt (mm)	[1	2202.040
Lead height (mm)	[pz]	3292.919
Axial pitch (mm)	[px]	131.717
Profile shift coefficient	[x]	0.5982
Tooth thickness, arc, in module	[sn*]	2.1287
Tip alteration (mm)	[k*mn]	-0.781
Reference diameter (mm)	[d]	203.743
Base diameter (mm)	[db]	184.034
Tip diameter (mm)	[da]	227.753
(mm)	[da.e/i]	227.753 / 227.707
• •		0.000 / -0.046
Tip diameter allowances (mm)	[Ada.e/i]	
Tip form diameter (mm)	[dFa]	227.753



(mm)	[dFa.e/i]	227.753 / 227.707
Root diameter (mm)	[df]	193.315
Generating Profile shift coefficient	[xE.e/i]	0.5855/ 0.5788
Generated root diameter with xE (mm)	[df.e/i]	193.111 / 193.004
Root form diameter (mm)	[dFf]	197.363
(mm)	[dFf.e/i]	197.192 / 197.102
Involute length (mm)	[l_dFa-l_dFf]	17.550
Addendum, $m_n(h_{aP}^*+x+k)$ (mm)	[ha]	12.005
(mm)	[ha.e/i]	12.005 / 11.982
Dedendum (mm)	[hf=mn*(hfP*-x)]	5.214
(mm)	[hf.e/i]	5.316 / 5.370
Tooth height (mm)	[h]	17.219
Virtual gear no. of teeth	[zn]	26.253
Normal tooth thickness at tip circle (mm)	[san]	3.918
(mm)	[san.e/i]	3.845 / 3.757
Normal tooth thickness at tip form circle (mm)	[sFan]	3.918
(mm)	[sFan.e/i]	3.845 / 3.757
Normal space width at root circle (mm)	[efn]	3.688
(mm)	[efn.e/i]	3.711 / 3.723

# Gear specific pair data Gear pair 1, Gear 1

[dw]	178.957
[dw.e/i]	178.978 / 178.936
[dNa]	194.686
[dNa.e/i]	194.686 / 194.640
[c]	2.000
[c.e/i]	2.201 / 2.079
[dNf]	167.049
[dNf.e/i]	167.114 / 167.015
[cF.e/i]	1.432 / 1.341
[vga]	13.644
[ζa]	0.438
[ζf]	-0.780
[ζm]	0.438
[Kga]	0.290
[Kgf]	-0.276
[ξdFa.e/i]	43.862 / 43.834
[ξdNa.e/i]	43.862 / 43.834
[ξdNf.e/i]	23.527 / 23.430
[ξdFf.e/i]	20.662 / 20.569
[d-B]	170.572 ( 170.572 / 170.540 )
[d-D]	189.476 ( 189.423 / 189.574 )
[ε]	0.609 ( 0.610 / 0.607 )
	[dNa] [dNa.e/i] [c] [c.e/i] [dNf] [dNf.e/i] [cF.e/i] [vga] [ζa] [ζf] [ζm] [Kga] [Kgf]  [ξdFa.e/i] [ξdNa.e/i] [ξdNf.e/i] [ξdFf.e/i] [d-B] [d-D]

# Gear specific pair data Gear pair 1, Gear 2

Operating pitch diameter (mm)	[dw]	213.044
(mm)	[dw.e/i]	213.069 / 213.019
Active tip diameter (mm)	[dNa]	227.753
(mm)	[dNa.e/i]	227.753 / 227.707
Theoretical tip clearance (mm)	[c]	2.000
Effective tip clearance (mm)	[c.e/i]	2.201 / 2.079
Active root diameter (mm)	[dNf]	200.328
(mm)	[dNf.e/i]	200.394 / 200.292
Reserve (dNf-dFf)/2 (mm)	[cF.e/i]	1.646 / 1.550



Max. sliding velocity at tip (m/s)	[vga]	12.995
Specific sliding at the tip	[ζa]	0.438
Specific sliding at the root	[ζf]	-0.780
Mean specific sliding	[ζm]	0.438
Sliding factor on tip	[Kga]	0.276
Sliding factor on root	[Kgf]	-0.290
Roll angle at dFa (°)	[ξdFa.e/i]	41.773 / 41.748
Roll angle to dNa (°)	[ξdNa.e/i]	41.773 / 41.748
Roll angle to dNf (°)	[ξdNf.e/i]	24.690 / 24.610
Roll angle at dFf (°)	[ξdFf.e/i]	22.050 / 21.972
Diameter of single contact point B (mm)	[d-B]	222.691 ( 222.640 / 222.786 )
Diameter of single contact point D (mm)	[d-D]	203.958 ( 203.958 / 203.924 )
Addendum contact ratio	[ε]	0.580 ( 0.582 / 0.578 )

#### General influence factors

		Gear 1 Gear 2
Nominal circum. force at pitch circle (N)	[Ft]	3470.8
Axial force (N)	[Fa]	674.6
Radial force (N)	[. ⊆] [Fr]	1648.7
Normal force (N)	[Fnorm]	3901.2
Nominal circumferential force per mm (N/mm)	[w]	133.49
Only as information: Forces at operating pitch circle:	[]	
Nominal circumferential force (N)	[Ftw]	3319.2
Axial force (N)	[Faw]	674.6
Radial force (N)	[Frw]	1935.7
Circumferential speed reference circle (m/s)	[v]	45.02
Circumferential speed operating pitch circle (m/s)	[v(dw)]	47.08
chounterential opera operating pitch and (1170)	[*(a**)]	11.00
Running-in value (µm)	[yp]	0.8
Running-in value (µm)	[yf]	1.0
Correction factor	[CM]	0.800
Gear blank factor	[CR]	1.000
Basic rack factor	[CBS]	1.073
Material coefficient	[E/Est]	1.000
Singular tooth stiffness (N/mm/µm)	[c']	15.836
Meshing stiffness (N/mm/µm)	[cγα]	18.092
Meshing stiffness (N/mm/µm)	[cγβ]	15.378
Reduced mass (kg/mm)	[mRed]	0.07439
Resonance speed (min-1)	[nE1]	7091
Resonance ratio (-)	[N]	0.708
Subcritical range		
Running-in value (µm)	[yα]	0.8
Bearing distance I of pinion shaft (mm)	[1]	56.000
Distance s of pinion shaft (mm)	[s]	5.600
Outside diameter of pinion shaft (mm)	[dsh]	28.000
Load in accordance with Figure 13, ISO 6336-1:2006	[-]	4
0:a), 1:b), 2:c), 3:d), 4:e)		
Coefficient K' according to Figure 13, ISO 6336-1:2006	[K']	-1.00
Without stiffening		
Tooth trace deviation (active) (µm)	[Fβy]	3.83
from deformation of shaft (µm)	[fsh*B1]	3.15
fsh ( $\mu$ m) = 3.15 , B1=1.00 , fH $\beta$ 5 ( $\mu$ m) = 6.50		
Tooth without tooth trace modification		
Position of contact pattern:	favorabl	е
from production tolerances (µm)	[fma*B2]	12.73
B2=		



1.00 Tooth trace deviation, theoretical (μm) Running-in value (μm)	[Fβx] [yβ]	4.50 0.67
Dynamic factor	[Kv]	1.453
Face load factor - flank - Tooth root - Scuffing	[ΚΗβ] [Κ <b>F</b> β] [ΚΒβ]	1.072 1.049 1.072
Transverse load factor - flank - Tooth root - Scuffing	[ΚΗα] [ΚFα] [ΚΒα]	1.000 1.000 1.000
Number of load cycles (in mio.)	[NL]	3014.400 2532.096

# Tooth root load capacity

Calculation of Tooth form coefficients according method: B			
		Gear 1	Gear 2
Calculated with generating profile shift coefficient	[xE.i]	0.5496	0.5788
Tooth form factor	[YF]	1.29	1.28
Stress correction factor	[YS]	2.19	2.22
Load application angle (°)	[αFen]	32.20	31.55
Load distribution influence factor	[fε]	0.	982
Load application diameter (mm)	[d <sub>en</sub> ]	188.664	221.902
Bending moment arm (mm)	[hF]	11.20	11.25
Tooth thickness at root (mm)	[sFn]	19.51	19.75
Tooth root radius (mm)	[ρ <b>F</b> ]	3.30	3.24
Bending moment arm (-)	[hF/mn]	1.400	1.406
Tooth thickness at root (-)	[sFn/mn]	2.438	2.469
Tooth root radius (-)	[pF/mn]	0.412	0.405
Calculation cross section diameter (mm)	[d <sub>sFn</sub> ]	162.460	195.597
Tangents on calculation cross section (°)	$[\alpha_{sFn}]$	30.000	30.000
Notch parameter	[qs]	2.958	3.047
Helix angle factor	[Υβ]	1.	038
Deep tooth factor	[YDT]	1.	000
Gear rim factor	[YB]	1.00	1.00
Effective facewidth (mm)	[beff]	28.00	26.00
Nominal stress at tooth root (N/mm²)	[σF0]	45.61	49.18
Tooth root stress (N/mm²)	[σF]	146.07	157.52
Permissible bending stress at root of Test-gear			
Notch sensitivity factor	[YdreIT]	1.004	1.005
Surface factor	[YRrelT]	0.957	0.957
Size factor, tooth root	[YX]	0.970	0.970
Finite life factor	[YNT]	0.871	0.874
$Y_{drelT}^*Y_{RrelT}^*Y_X^*Y_{NT}$		0.811	0.815
Alternating bending factor, mean stress influence coefficient			
	[YM]	1.000	1.000
Stress correction factor	[Yst]	2.	00
Yst*σFlim (N/mm²)	[σFE]	860.00	860.00
Permissible tooth root stress $\sigma FG/SFmin$ (N/mm²)	[σFP]	436.12	437.98
Limit strength tooth root (N/mm²)	[σFG]	697.79	700.77
Required safety	[SFmin]	1.60	1.60
Safety for tooth root stress	[SF=σFG/σF]	4.78	4.45
Transmittable power (kW)	[kWRating]	466.53	434.46



# Flank safety

		Gear 1	Gear 2
Zone factor	[ZH]		2.035
Elasticity factor (√N/mm²)	[ZE]	18	9.812
Contact ratio factor	[Zε]		0.958
Helix angle factor	[Zβ]		1.009
Effective facewidth (mm)	[beff]	2	6.00
Nominal contact stress (N/mm²)	[σH0]	44	7.35
Contact stress at operating pitch circle (N/mm²)	[σHw]	80	9.25
Coefficient [fZCa] 1.20 (Helical gear sets without fla	ank modifications)		
Single tooth contact factor	[ZB,ZD]	1.05	1.02
Contact stress (N/mm²)	[σHB, σHD]	846.97	825.95
Lubrication factor for NL	[ZL]	1.020	1.020
Speed factor for NL	[ZV]	1.044	1.044
Roughness factor for NL	[ZR]	0.986	0.986
Material hardening factor for NL	[ZW]	1.000	1.000
Finite life factor	[ZNT]	0.882	0.887
	[ZL*ZV*ZR*ZN	NT] 0.926	0.931
Limited pitting is permitted:	No		
Size factor (flank)	[ZX]	1.000	1.000
Permissible contact stress, σHG/SHmin (N/mm²)	[σHP]	1068.69	1074.42
Pitting stress limit (N/mm²)	[σHG]	1389.30	1396.75
Required safety	[SHmin]	1.30	1.30
Safety factor for contact stress at operating pitch circle	[SHw]	1.72	1.73
Safety against pressure, σHG/σHBD Single contact	[SHBD]	1.64	1.69
Safety regarding transmittable torque	[(SHBD)^2]	2.69	2.86
Transmittable power (kW)	[kWRating]	248.77	264.41

# Micropitting according to

ISO/TS 6336-22:2018

Calculation has not been carried out, lubricant: Load stage micropitting test not known

# Scuffing load capacity

Calculation method according to	ISO/TS 6336-20/21:2017			
Helical load factor for scuffing	[ΚΒγ]	1.	000	
Lubrication coefficient for lubrication type	[XS] 1.200			
Scuffing test and load stage	[FZGtest] FZG - Test A / 8.3 / 90 (ISO 14635 - 1)			12
Multiple meshing factor	[Xmp] 1.000			
Relative structural factor, scuffing	[XWreIT]	1.000		
Thermal contact factor (N/mm/s^.5/K)	[BM]	13.780 13.780		
Relevant tip relief (µm)	[Ca]	2.00 2.00		
Optimal tip relief (µm)	[Ceff]	15.49		
Ca taken as optimal in the calculation (0=no, 1=yes)		0	0	
Effective facewidth (mm)	[beff]	26.	000	
Applicable circumferential force/facewidth (N/mm)	[wBt]	[wBt] 436.828		
$KB\gamma = 1.000 \text{ , } wBt*KB\gamma = 436.828$				
Angle factor	[Χαβ] 1.130			

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# ε1: 0.609 , ε2: 0.580

Flash temperature-criteria		
Lubricant factor	[XL]	0.830
Tooth mass temperature (°C)	[θMi]	83.32
$\theta$ Mi = $\theta$ oil + XS*0.47*Xmp* $\theta$ flm		
Average flash temperature (°C)	[θflm]	23.61
Scuffing temperature (°C)	[ <del>0</del> S]	348.80
Γ coordinates (point of highest temperature)	[Г]	0.214
[Γ.A]= -0.298 [Γ.Ε]= 0.313		
Highest contact temp. (°C)	[θB]	125.09
Flash factor (°K*N^75*s^.5*m^5*mm)	[XM]	50.058
Approach factor	[XJ]	1.000
Load sharing factor	[XF]	1.000
Dynamic viscosity (mPa*s)	[ηM]	41.90 ( 70.0 °C)
Coefficient of friction	[µ <sub>m</sub> ]	0.032
Required safety	[SBmin]	2.000
Margin of safety for scuffing, flash temperature	[SB]	5.061
Integral temperature-criteria		
Lubricant factor	[XL]	1.000
Tooth mass temperature (°C)	[0MC]	85.47
$\theta$ MC = $\theta$ oil + XS*0.70* $\theta$ flaint		
Mean flash temperature (°C)	[θflaint]	18.41
Integral scuffing temperature (°C)	[θSint]	360.78
Flash factor (°K*N^75*s^.5*m^5*mm)	[XM]	50.058
Running-in factor, well run in	[XE]	1.000
Contact ratio factor	[Xɛ]	0.374
Dynamic viscosity (mPa*s)	[ηOil]	41.90 ( 70.0 °C)
Mean coefficient of friction	[µ <sub>m</sub> ]	0.027
Geometry factor	[XBE]	0.209
Meshing factor	[XQ]	1.000
Tip relief factor	[XCa]	1.091
Integral tooth flank temperature (°C)	[θint]	113.08
Required safety	[SSmin]	1.800
Safety factor for scuffing (intgtemp.)	[SSint]	3.190
Safety referring to transmittable torque	[SSL]	6.749

### Measurements for tooth thickness

		Gear 1 Gear 2		
Tooth thickness tolerance		DIN 3967 cd25	DIN 3967 cd25	
Tooth thickness allowance (normal section) (mm)	[As.e/i]	-0.095 /-0.145	-0.095 /-0.145	
Number of teeth spanned	[k]	4.000	5.000	
Base tangent length (no backlash) (mm)	[Wk]	88.376	112.266	
Base tangent length with allowance (mm)	[Wk.e/i]	88.289 / 88.244	112.180 /112.135	
(mm)	[ΔWk.e/i]	-0.086 / -0.131	-0.086 / -0.131	
Diameter of measuring circle (mm)	[dMWk.m]	177.358	214.643	
Theoretical diameter of ball/pin (mm)	[DM]	16.594	16.233	
Effective diameter of ball/pin (mm)	[DMeff]	18.000	18.000	
Radial single-ball measurement backlash free (mm)	[MrK]	104.806	121.493	
Radial single-ball measurement (mm)	[MrK.e/i]	104.732 /104.693	121.417 /121.377	
Diameter of measuring circle (mm)	[dMMr.m]	181.535	215.103	
Diametral measurement over two balls without clearance (mm)	[MdK]	209.077	242.542	
Diametral two ball measure (mm)	[MdK.e/i]	208.929 /208.852	242.390 /242.310	
Diametral measurement over pins without clearance (mm)	[MdR]	209.613	242.986	



Measurement over pins according to DIN 3960 (mm)  Measurement over 2 pins, free, according to AGMA 2002 (mm)  Measurement over 3 pins, axial, according to AGMA 2002 (mm)	[MdR.e/i] [dk2f.e/i]	209.465 /209.387 208.903 /208.825	242.834 /242.754 242.369 /242.289
(min)	[dk3A.e/i]	209.465 /209.387	242.834 /242.754
Chordal tooth thickness (no backlash) (mm)	[sc]	16.787	17.011
Normal chordal tooth thickness with allowance (mm)	[sc.e/i]	16.695 / 16.647	16.919 / 16.871
Reference chordal height from da.m (mm)	[ha]	12.157	12.336
Tooth thickness, arc (mm)	[sn]	16.812	17.029
(mm)	[sn.e/i]	16.717 / 16.667	16.934 / 16.884
Backlash free center distance (mm)	[aControl.e/i]	195.827 /19	5.735
Backlash free center distance, allowances (mm)	[jta]	-0.174 / -0	.265
dNf.i with aControl (mm)	[dNf0.i]	166.652	199.914
Reserve (dNf0.i-dFf.e)/2 (mm)	[cF0.i]	1.159	1.361
Tip clearance (mm)	[c0.i(aControl)]	1.836	1.836
Center distance allowances (mm)	[Aa.e/i]	0.023 / -0	.023
Circumferential backlash from Aa (mm)	[jtw_Aa.e/i]	0.027 / -0	.027
Radial backlash (mm)	[jrw.e/i]	0.288 / 0.151	
Circumferential backlash (transverse section) (mm)	[jtw.e/i]	0.336 / 0.176	
Normal backlash (mm)	[jn.e/i]	0.282 / 0.153	
Torsional angle on input with output fixed:			
Total torsional angle (°)	[j.tSys]	0.2150/0.	1124

#### **Toothing tolerances**

		Gear 1	Gear 2
According to ISO 1328-1:2013, ISO 1328-2:1997			
Accuracy grade	[Q]	A6	A6
Single pitch deviation (µm)	[fptT]	12.00	12.00
Base circle pitch deviation (µm)	[fpbT]	10.70	10.70
Sector pitch deviation over k/8 pitches (µm)	[Fpk/8T]	25.00	24.00
Profile form deviation (µm)	[ffαT]	13.00	13.00
Profile slope deviation (µm)	[fHαT]	10.00	10.00
Total profile deviation (µm)	[FαT]	17.00	17.00
Helix form deviation (µm)	[ffβT]	10.00	10.00
Helix slope deviation (µm)	[fHβT]	9.00	9.00
Total helix deviation (µm)	[FβT]	14.00	14.00
Total cumulative pitch deviation (µm)	[FpT]	36.00	37.00
Adjacent pitch difference (µm)	[fuT]	17.00	17.00
Runout (µm)	[FrT]	32.00	33.00
Single flank composite, total (µm)	[FisT]	47.00	48.00
Single flank composite, tooth-to-tooth (µm)	[fisT]	11.00	11.00
Radial composite, total (µm)	[FidT]	64.00	64.00
Radial composite, tooth-to-tooth (µm)	[fidT]	34.00	34.00

FidT (Fi"), fidT (fi") according to ISO 1328:1997 calculated with the geometric mean values for mn and d

Axis alignment tolerances (recommendation acc. to ISO TR 10064-3:1996, Quality)

 $6 \\ \label{eq:maximum} \mbox{Maximum value for deviation error of axis ($\mu m$)} \qquad [f\Sigma\beta] \qquad 15.08 \qquad (F\beta= 14.00 \ \ ) \\ \mbox{Maximum value for inclination error of axes ($\mu m$)} \qquad [f\Sigma\delta] \qquad 30.15$ 

#### Modifying and defining the tooth form

Data for the tooth form calculation :



Data not available.

Please run the calculation in the "Tooth form" tab and open the main report again.

#### Supplementary data

Mass (kg)	[m]	5.422	7.087
Total mass (kg)	[mGes]	12.50	9
Moment of inertia for system, relative to the input:			
calculation without consideration of the exact tooth shap	e		
Single gears, (da+df)/2di (kg*m²)	[J]	0.02135	0.03927
System (da+df)/2di (kg*m²)	[J]	0.04	905
Torsional stiffness at driving gear with fixed driven gear:			
Torsional stiffness (MNm/rad)	[cr]	2.28	34
Torsion when subjected to nominal torque (°)	[δcr]	0.00	)7
Mean coefficient of friction (as defined in Niemann)	[µ <sub>m</sub> ]	0.02	27
Wear sliding coef. by Niemann	[ζw]	0.52	22
Loss factor	[HV]	0.14	5
Gear power loss (kW)	[PVZ]	0.60	06
Meshing efficiency (%)	[ηz]	99.61	2
Sound pressure level according to Masuda, without conta	act analysis		
	[dB(A)]	89.6	
Oil requirement for injection lubrication (I/min)	[Voil]	1.89	00
with oil cooler, for assumed difference in temperature of	of oil (°C):		
	10		

# Service life, damage

Required safety	for tooth ro	ot		[SFmin]		1.60
Required safety	for tooth fla	ınk		[SHmin]		1.30
Service life (cale	culated with	required safeti	es):			
System service	life (h)			[Hatt]		> 1000000
Tooth root servi	ce life (h)			[HFatt]	1e+06	1e+06
Tooth flank serv	rice life (h)			[HHatt]	1e+06	1e+06
Note: The entry	1e+006 h n	neans that the	Service life > 1,000,000	) h.		
Damage calcula	ated on the b	oasis of the req	uired service life [H] (	10000.0	h)	
F1%	F2%	H1%	H2%			

0.0000

### Remarks:

0.00

 Specifications with [.e/i] imply: Maximum [e] and minimum value [i] for Taking all tolerances into account
 Specifications with [.m] imply: Mean value within tolerance

0.0000

- For the backlash tolerance, the center distance tolerances and the tooth thickness allowance are taken into account.

The maximum and minimum clearance according to the largest or smallest allowances are defined..

0.0000



The calculation is performed for the operating pitch circle.

- Calculation of Zbet according to Corrigendum 1 ISO 6336-2:2008 with Z $\beta$  = 1/(COS( $\beta$ )^0.5)
- Details of calculation method:

cγ according to Method B

Kv according to Method B

 $KH\beta$  and  $KF\beta$  according to Method C

fm $\alpha$  according to Equation 64, fsh according to 57/58, F $\beta$ x according to 52/53/54

 $KH\alpha,\,KF\alpha$  according to Method B

- The logarithmically interpolated value taken from the values for the fatigue strength and the static strength, based on the number of load cycles, is used for coefficients ZL, ZV, ZR, ZW, ZX, YdrelT, YRrelT and YX..

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