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Green's theorem and distribution of singularities for motion of an object in an unbounded fluid

Whenever we have an object undergoing oscillatory motion in a fluid, there is a portion of fluid which always moves along with the object. This should not to be understood as a result of viscosity. But rather from the point of view of inertia. As we move an object inside a fluid, the object displaces some portion of fluid. Thus the movement of an object in a fluid is associated with the movement of some fluid. This leads to the idea of added mass form. We do not feel in a ~~very~~ most simplistic form because the effect of added mass as such in air because the density of air is much less than that of water.

For a fluid like water, the inviscid flow theory for ~~other~~ elementary flows like source, sink, dipole etc becomes very useful to develop theoretical tools. It becomes useful to analyze the added mass effect and thus to develop better insights of added mass helping us to develop better insights of added mass for a given body shape.

Let us consider two scalar potentials ϕ and ψ both satisfying Laplace equation in ~~in~~ a volume V bounded by a closed surface S . Thus from divergence theorem we get,

$$\begin{aligned} \int_S \left[\phi \frac{\partial \psi}{\partial n} - \psi \frac{\partial \phi}{\partial n} \right] dS &= \int_V \nabla \cdot (\phi \frac{\partial \psi}{\partial n} - \psi \frac{\partial \phi}{\partial n}) dV \\ &= \int_V (\phi \nabla^2 \psi + \nabla \phi \cdot \nabla \psi - \psi \nabla^2 \phi - \nabla \psi \cdot \nabla \phi) dV \\ &= 0 \end{aligned}$$

This result is known as Green's theorem.

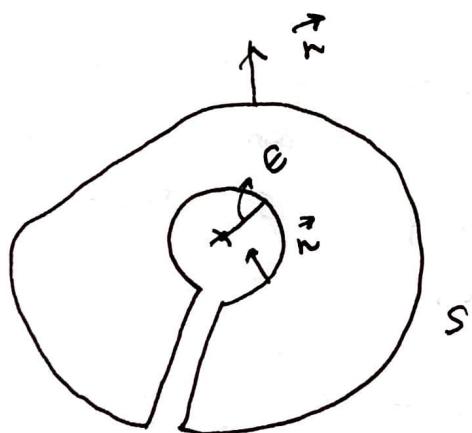
Suppose, we take the velocity potential as,

$$\text{Ans} \quad \psi = \frac{1}{4\pi r} = \left(\frac{1}{4\pi}\right) \cdot [(x - a)^2 + (y - b)^2 + (z - c)^2]^{-\frac{1}{2}}; \quad (2)$$

where, (a, b, c) is the location of the source σ in a fixed coordinate system (x, y, z) . We can show that this function, $\psi(x, y, z; a, b, c)$ satisfies Laplace equation ~~and~~ no matter we choose (x, y, z) ; or (a, b, c) coordinate system.

Furthermore, the potential ψ has a singularity at r . We say, ~~that~~ point $\vec{r}(a, b, c)$ is a source point and point $\vec{r}(x, y, z)$ is a field point.

With the ~~singular~~ velocity potential ψ defined in equation (2), we can find the other velocity potentials using Green's theorem (1). However, this requires a careful choice of the surface S . There is a ~~singularity~~ singularity at r . Thus we ~~cannot~~ construct the surface ~~as~~ as,



where we surround the singularity at r with a very small surface say S_f and connect it to the outer surface S as shown in the above picture. The normal vector \vec{n} is ~~also~~ always pointing outward from the surface enclosing the

\leftarrow find ~~at~~ position.

Thus, from equation ①, we have,

$$\frac{1}{4\pi} \int_{S+E}^P [\phi \frac{\partial}{\partial n} (\frac{1}{r}) - \frac{1}{r} \cdot \frac{\partial \phi}{\partial n}] d\sigma = 0$$

$$\text{or, } \frac{1}{4\pi} \int_S^P (\phi \frac{\partial}{\partial n} (\frac{1}{r}) - \frac{1}{r} \frac{\partial \phi}{\partial n}) d\sigma = 0 \quad \text{--- ②}$$

$$= - \frac{1}{4\pi} \int_{S_E}^P (\phi \frac{\partial}{\partial n} (\frac{1}{r}) - \frac{1}{r} \frac{\partial \phi}{\partial n}) d\sigma. \quad \text{--- ③}$$

Now, we want to examine the integral on the right hand side of equation ②, as the radius ~~of the~~ ^{sphere} r of the inner ~~area~~ tends towards zero.

If ϵ is very small, we consider ϕ to change very slowly ~~near the origin~~ and take that as almost like a constant. Thus we can take ϕ out of this ~~integral~~ first integrate on the right hand side.

$$\text{Thus, } - \frac{1}{4\pi} \int_{S_E}^P \phi \frac{\partial}{\partial n} (\frac{1}{r}) d\sigma$$

$$= - \frac{1}{4\pi} \cdot \phi \cdot \int_{S_E}^P \frac{1}{r} d\sigma \quad \text{--- ④}$$

and the second integral $= - \frac{1}{4\pi} \int_{S_E}^P (-\frac{1}{r}) \frac{\partial \phi}{\partial n} d\sigma$.

Now, since S_E can be viewed as an enclosing surface of a sphere, $\frac{\partial}{\partial n} (\frac{1}{r}) \equiv \frac{\partial}{\partial r} (\frac{1}{r}) = -\frac{1}{r^2}$.

and using idea of a solid angle,

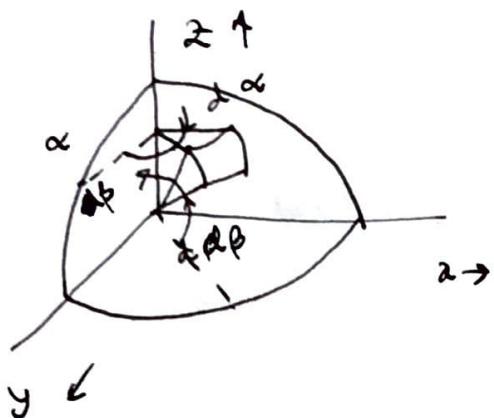
$$d\sigma = \frac{r^2}{r^2} \cdot d\Omega.$$



we have,

$$\begin{aligned} - \frac{1}{4\pi} \phi \cdot \int_{S_E}^P \frac{\partial}{\partial n} (\frac{1}{r}) d\sigma &= - \frac{1}{4\pi} \phi \int_{S_E}^P (-\frac{1}{r^2}) \cdot r^2 d\Omega \\ &= - \frac{1}{4\pi} \phi \int_{S_E}^P (-d\Omega) \end{aligned}$$

Note, α is a solid angle.



$$\alpha = (\text{solid angle}) \cdot (\text{area})$$

$$\alpha = (r d\phi) \cdot (r \sin \theta d\theta)$$

$$= r^2 \sin \theta d\phi d\theta$$

$$\therefore \cancel{d\theta} d\theta = \frac{d\sigma}{r^2}$$

$$= \sin \theta d\phi d\theta.$$

$$\therefore \oint d\theta = 8 \int_{\pi/2}^{\pi} \int_{\beta=0}^{\pi/2} \sin \theta d\phi d\theta$$

$$= -8 \int_{\pi/2}^{\pi} [\cos \theta]_{\beta=0}^{\pi/2} d\theta = -8 \cdot \frac{1}{2} = -4\pi.$$

$$\therefore \lim_{\epsilon \rightarrow 0} \left(-\frac{1}{4\pi} \oint_{S_\epsilon} \frac{\partial \Phi}{\partial n} dS \right) = -\Phi(x, y, z)$$

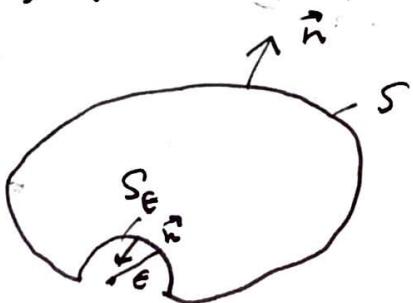
Since we take ρ as almost constant, the second integral $-\frac{1}{4\pi} \int_{S_\epsilon} (-\frac{1}{n}) \frac{\partial \Phi}{\partial n} dS$ vanishes as $\epsilon \rightarrow 0$.

Thus, we have,

$$\Phi(x, y, z) = -\frac{1}{4\pi} \int_S \left[\Phi_{,n}(\vec{r}) - \frac{1}{n} \cdot \frac{\partial \Phi}{\partial n} \right] dS$$

— (5).

Equation (5) is the representation of velocity potential if the point of evaluation is inside the surface S . On the other hand, if the point is on the surface S , we can surround the singularity by N ,



The surface S is now laid on a hemisphere. And thus, we can show that,

$$\Phi(x, y, z) = \frac{1}{2\pi} - \frac{1}{2\pi} \int_S \left[\varphi \frac{\partial \Phi}{\partial n} (\frac{1}{r}) - \frac{1}{r} \cdot \frac{\partial \Phi}{\partial n} \right] dS. \quad -⑥$$

Equation ⑥ is frequently used for calculating the velocity potential on the surface of a ship hull or a moving body. In these cases, the derivative $\frac{\partial \Phi}{\partial n}$ is known on the profile at any point and thus equation ⑥ is actually an integral equation to solve to find the velocity potential $\Phi(x, y, z)$. This integral equation can be solved using a suitable numerical technique.

For a body like a ship hull moving in an unbounded fluid, the surface S must include the body surface, the sea bed and the free surface. In this case, there are additional boundary conditions to be imposed and there is a computational advantage of using equation ⑥ to solve for the velocity potential. In this context, we refine the Green function as,

$$G(x, y, z; \alpha, \beta, \gamma) = \frac{1}{r} + H(x, y, z; \alpha, \beta, \gamma) \quad -⑦$$

where, H is any function that satisfy Laplace's equation. Thus, using ⑦ in ⑥ we have

$$\int_S \left(\Phi \frac{\partial G}{\partial n} - G \frac{\partial \Phi}{\partial n} \right) dS = \begin{cases} 0 & \text{if } \alpha = x, \beta = y, \gamma = z \\ -2\pi \Phi(x, y, z) & \\ -4\pi \Phi(x, y, z) & \end{cases}$$

if the point α - evaluation is outside the region S ; if α is on the surface S and inside the surface S respectively.

(In 2D, equation ⑤ can be shown to be as

$$\Phi(x, y, z) = -\frac{1}{2\pi} \int_{\Gamma} \left[\frac{\omega^2}{r} \left(\frac{1}{r} \right) - \frac{1}{r} \frac{\partial \Phi}{\partial n} \right] ds$$

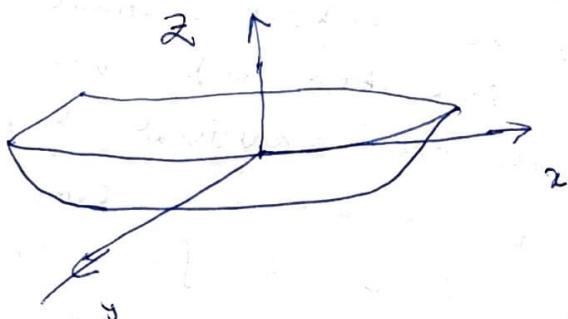
where the source potential is of the form

$\frac{1}{2\pi} \lambda_1(r)$ as we have seen earlier while studying the source flows in 2D. The point of evaluation is along the curve S .)

Assignment (20 marks)

linear wave body interaction problem

(wave diffraction + radiation)



Total relative potential, $\Phi(x, y, z, t)$

$$= \operatorname{Re} \left[\sum_{j=1}^6 w_j \Phi_j(x, y, z) + \operatorname{Re} [A \cdot \Psi_A(x, y, z) e^{i\omega t}] \right]$$

Φ_j are the radiation potential — ①

and Ψ_A is the diffracted wave potential.

Boundary conditions

$$\frac{\partial \Phi_j}{\partial n} = 0 \quad \text{when } j = 1, 2, 3 \quad \text{for } j = 1, 2, 3 \quad \left. \right\} - ②$$

$$S_B = i\omega(n + k_j) \quad \text{for } j = 4, 5, 6$$

$$\text{since, } \Psi_j(t) = \operatorname{Re} (i\omega w_j e^{i\omega t})$$

Here, \vec{n} is an unit normal vector pointing into the body from the fluid.

For wave diffraction problem,

$$* \quad \frac{\partial \Phi_A}{\partial n} = 0 \quad \text{on } S_B \quad (3)$$

Furthermore, we consider $\Phi_A = \Phi_0 + \Phi_F$, where Φ_0 is the incident wave potential and Φ_F is scattered wave potential.

$$\text{Thus, we have, } \frac{\partial \Phi_0}{\partial n} = - \frac{\partial \Phi_F}{\partial n} \text{ on } S_B. \quad (4)$$

From combine a free surface boundary condition we have,

$$- \frac{\omega^2}{g} \Phi_F + \frac{\partial \Phi_F}{\partial z} = 0 \quad \text{on } S_F \quad \text{for } j=1, 2, \dots, T. \quad (5)$$

The last boundary condition we consider is the Sommerfeld radiation boundary condition, which make the radiation potentials (Φ_j , $j=1, 2, 3, \dots, k$) make the scattered potential different from the incident potential Φ_0 ,

$$\Phi_j \propto R^{-\frac{1}{2}} e^{-ikR} \quad \text{as } R \rightarrow \infty \quad j=1, 2, 3, \dots, T. \quad (6)$$

Total hydrodynamic pressure on the body is given as,

$$P = - g \left(\frac{\partial \Phi}{\partial t} + g z \right)$$

$$= - g \rho c \left[\underbrace{\sum_{j=1}^k}_{\text{radiation}} \left(\sum_{j=1}^k \Phi_j + A (\Phi_0 + \Phi_F) \right) + i \omega \right]$$

$$- g z \quad (7)$$

The total hydrodynamic force is given as,

$$(F_m) = - \rho g \int_S (\vec{n} \times \vec{v}) z ds - g \rho c \sum_{j=1}^6 i \omega b_j e^{i \omega t}$$

$$\cdot \int_S (\vec{n} \times \vec{v}) a_j ds$$

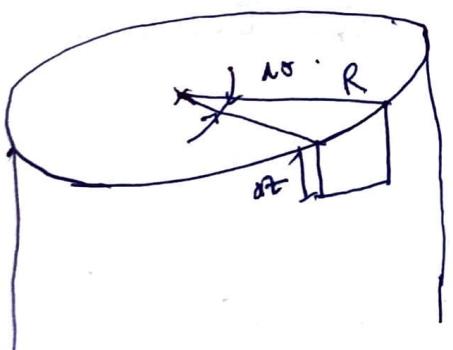
$$- g \rho c i \omega t e^{i \omega t} \int_S (\vec{n} \times \vec{v}) (a_s + a_p) ds.$$

(8)

Sommerfeld radiation boundary condition

* wave energy flux is given by $\Theta = \frac{\rho g a^2}{2}$. This is the wave energy per unit wave length in 2D. And this is energy per unit area in 3D.

Suppose there is an elemental surface ds at large distance from the source of wave generation. The total wave energy out of this elemental area is



$$dE = \left(\frac{\rho g a^2}{2}\right) \cdot ds$$

$$= \left(\frac{\rho g a^2}{2}\right) \cdot R \cdot d\theta \cdot dz$$

$$\therefore E = \int_0^{2\pi} \int_{-h}^0 \frac{\rho g a^2}{2} \cdot R \cdot d\theta \cdot dz$$

$$\theta = 0 \quad z = -h$$

$$= \pi R \cdot \frac{\rho g a^2}{2} \cdot 2\pi R \cdot h$$

If this energy is to be preserved with increasing R or ~~constant~~ constant depth h ,

then, we must have, $\alpha \propto R^{-1}$, or, $\alpha \propto R^{-\frac{1}{2}}$.
 we know from ~~kinematic~~ kinematic free surface
 boundary condition that, $\eta = -\frac{1}{2} \cdot \frac{\partial \phi}{\partial r} \Big|_{r=0}$,

$$\phi \propto R \{ \phi(x, y, z) \cdot e^{i\omega t} \}.$$

$$\Rightarrow \alpha(r, y, z) \propto n \propto \alpha \propto R^{-\frac{1}{2}}. \quad \text{--- (1)}$$

On the other hand by applying Laplace coordinate systems
 solution in cylindrical the resulting potential
 we can find the solution of $\phi(R, \theta, z)$ can be represented by Hankel function ($H_m^{(1)}(R)$)
 $\phi(R, \theta, z) = R(\theta) \cdot \theta(0) \cdot Z(z)$ and show that
 $R(r)$ can be of order m of second type. At large
 i.e., when $r \rightarrow +\infty$, $H_m^{(1)}(R)$ can be
 R , which are asymptotically 0.
 shown to be $\propto e^{-ikr}$. --- (2)

Thus, combining with (1) and (2) we have
 $\alpha(r, y, z) = \alpha(r, \theta, z) \propto -\frac{z}{r} \cdot e^{-ikr}$ and
 $r \rightarrow \infty$. This is Sommerfeld condition as $r \rightarrow \infty$.

Added mass and damping coefficients

From expression on wave radiation force, we have,

$$F_i = \operatorname{Re} \left\{ \sum_{j=1}^6 \alpha_{ij} e^{i\omega t} f_{ij} \right\}, \text{ for } i=1, 2, \dots, 6.$$

where, $f_{ij} = -g \int_{S_B} \Phi_j \frac{\partial \Phi_i}{\partial n} ds.$

Since, from boundary condition we have,

$$\left. \frac{\partial \Phi_j}{\partial n} \right|_{S_B} = i \omega h_j \text{ for } j=1, 2, 3$$

we write $F_{ij} = \omega (\alpha_{ij} - i \alpha_{bj}) \quad \text{--- (1)}$

where, $f_{ij} \in \mathbb{C}; \alpha_{ij} \in \mathbb{R}; \alpha_{bj} \in \mathbb{R}.$

Since ~~the~~ motion of the barge is proportional to $e^{i\omega t}$, we can interpret equation (1) as the overall hydrodynamic force F_{ij} is comprised of two parts. One part is in phase with the acceleration and another part is in phase with velocity.

The coefficient α_{ij} is known as added mass. Since it represents the real part of the force which is proportional to the acceleration, α_{ij} is known as damping coefficient.

The various components of added mass and damping coefficient can be interpreted as force component evaluated in 3 directions due to motion in longitudinal direction. These coefficients are specified as a function of ~~the~~ frequency or wave number.

Wave - exciting forces and moments

The first term of total wave exciting forces and moments are given as,

$$\{F_{ex}\} = - \rho g e^{i\omega t} A \cdot e^{i\omega t} \int_{S_B} (\Phi_0 + \Phi_T) \cdot \left(\frac{\vec{n}}{r} \times \vec{h} \right) dS$$

It is convenient to rewrite above expression as,

$$(F_{ex})_i = \operatorname{Re} \{ A e^{i\omega t} x_i \} \quad i = 1, 2, \dots, 6.$$

where, $x_i = \operatorname{Im} \{ \vec{A} e^{i\omega t} \int_{S_B} (\Phi_0 + \Phi_T) \left(\frac{\vec{n}_i}{r} \times \vec{h}_i \right) dS \}$

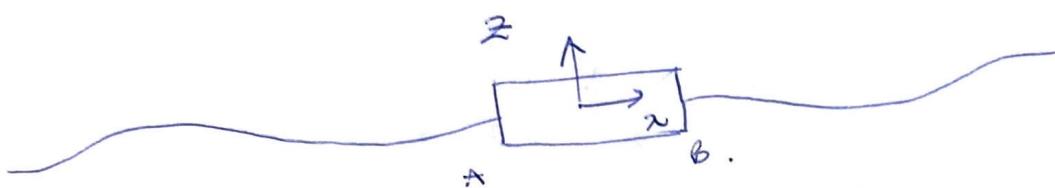
$$x_i = - i \omega \rho \int_{S_B} (\Phi_0 + \Phi_T) \left(\frac{\vec{n}_i}{r} \times \vec{h}_i \right) dS$$

is known as amplitude of exciting force or moment

& and clearly, $\{x_i\} \in \mathbb{R}$. In order to evaluate $\{x_i\}$, we must solve for the scattered wave potential Φ_T for the given boundary conditions.

The problem of finding ~~exist.~~ exciting force can be greatly simplified if we assume that the size and shape of the ship hull is such that the presence of the object does not affect the overall pressure distribution under the otherwise uninterrupted incident wave. A progressive incident wave has its own pressure distribution across the water depth and thus we can evaluate the total wave exciting force and moment on the ship hull using this pressure distribution alone provided the ship hull does not alter the incident wave. This approach is known as福兰德-基尔波恩 approximation.

Demonstration of Frigate Krylov Approach



~~Free surface force~~

$$F_z = \int_{x=x_A}^{x=x_B} p_z dx$$

$$p_z = - \rho g z + \rho g k_p(z) \cdot G(vx - ct)$$

where, $k_p(z) = \frac{Gh K(h+z)}{Gh (Kh)}$

Similar to intact mass and damping coefficients, the wave exciting force and moment amplitudes are specified as a function of wave frequency.

body response in regular wave

For a floating body at zero speed oscillations in response to an ~~incident~~ incident wave experience the following force components:

1) The inertia force component. This is given as $F_{\text{inert}} = - m \ddot{x}_{\text{re}} [\{ u_{\text{ff}} \} e^{i\omega t}]$

$$= - \cancel{m \ddot{x}_{\text{re}}} [\{ u_{\text{ff}} \}]$$

$$= - \omega^2 m \ddot{x}_{\text{re}} [\{ u_{\text{ff}} \} e^{i\omega t}]$$

2) The restoring force component given as ~~body~~ $F_{\text{rest}} = [C_{\text{re}} \{ u_{\text{ff}} \} e^{i\omega t}]$

3) The wave radiation and exciting force component ~~body~~ $F_{\text{wave}} = [F_{\text{exc}} \{ u_{\text{ff}} \} e^{i\omega t}]$

$$\omega [a] \operatorname{re} [\{g\} \cdot e^{i\omega t}] - i\omega [b] \operatorname{re} [\{g\} \cdot e^{i\omega t}] \\ + \operatorname{re} [A \cdot \{g\} \cdot e^{i\omega t}]$$

Now, balancing force components we have,

$$-\omega^2 [m] \operatorname{re} [\{g\} e^{i\omega t}] + [c] \operatorname{re} [\{g\} e^{i\omega t}] \\ = \omega^2 [a] \operatorname{re} [\{g\} e^{i\omega t}] \\ - i\omega [b] \operatorname{re} [\{g\} e^{i\omega t}] \\ + \operatorname{re} [A \cdot \{x\} e^{i\omega t}]$$

$$\therefore \{-\omega^2 (m+a) + i\omega b\} + [c] \{g\} \\ = A \{x\}$$

$$\text{or, } \{g\} = A [\alpha]^{-1} \{x\}, \text{ where,} \\ [\alpha] = -\omega^2 (m+a) + i\omega b + c$$

$$\text{or, } Z_j(\omega, \theta) = \frac{\{g_j\}}{A}, \text{ for } j=1, 2, 3, \dots, 6.$$

vector $\{g\}$ is known \Rightarrow frequency response and
the ratio $Z_j(\omega, \theta)$ is known \Rightarrow Response
amplitude operator ($A(\omega)$). It depends upon
incident wave ~~position~~ θ frequency (ω) and
direction (θ). 3

If we take only one member of the equation of motion for a specific degree of freedom say herez and consider that to be uncoupled \Rightarrow
from any other degree of freedom we may

$$\text{have } \{ -\omega^2 (m + a_{33}) + i\omega b_{33} + c_{33} \} \{g_3\} \\ = A \{x_3\}$$

This type of relationship is very alike analogous to a single degree of freedom Spring Mass Damper System. However, the ripples here lies in the fact that θ_3 , b_3 these are function of frequency ω , and therefore as it is not straightforward to solve for this the motion response θ_3 even for the case when the motion of displacement to be uncoupled.

All the other members of the state matrices depend on ship body geometry of the vessel, location of center of gravity, center of buoyancy and center of flotation.

wave encounter frequency
for a ship with a forward speed

for a ship moving with a forward speed V , the frequency of the incident waves will be affected from what the ship experiences when it is at rest. That

