

plane Couette flow (steady, $\nabla \cdot \mathbf{u} = 0$, incompressible flow).

- flow between two parallel plates.
- flow only in x -direction.
- no body force is present.

$$\vec{v} = (u, v, 0)$$

$$z = h$$

$u \rightarrow$

Navier-Stokes

equation for this case,

x -momentum equation -

$$\frac{\partial u}{\partial t} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right) \quad \text{--- (1)}$$

$$\frac{\partial u}{\partial t} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial z^2} \right) \quad \text{--- (2)}$$

for steady flow, $\frac{\partial u}{\partial t} = 0$.

$$u = u(z) \quad \therefore \frac{\partial u}{\partial x} = 0;$$

from continuity equation, $\frac{\partial w}{\partial z} = 0 \rightarrow$ variation of w across z

$$w = 0 \quad \text{at } z = 0 \quad \text{and } z = h.$$

$$\text{So, } w = 0, \quad 0 \leq z \leq h.$$

Equation (1) reduces to,

$$-\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \frac{\partial^2 u}{\partial z^2} = 0$$

$$\text{or, } \frac{1}{\rho} \frac{\partial p}{\partial x} = \nu \frac{\partial^2 u}{\partial z^2}$$

$$\text{or, } \frac{\partial p}{\partial x} = \mu \frac{\partial^2 u}{\partial z^2} \quad \text{--- (2)}$$

from y and z momentum equation we get,

$$\frac{\partial p}{\partial y} = 0; \quad \frac{\partial p}{\partial z} = 0$$

\therefore Equation (2) further reduces to,

$$\frac{dp}{dx} = \mu \frac{d^2 u}{dz^2}$$

Subjected to $u = 0$ at $z = 0$ and $u = v$ at $z = h$.



x momentum ~~equation~~ equation is the absence of any pressure ~~gradient~~ gradient, and body force

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial z} = - \frac{\partial p}{\partial x} + \nu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial z^2} \right) \quad \text{--- (1)}$$

$$\frac{\partial u}{\partial x} = 0; \quad \frac{\partial u}{\partial z} = 0;$$

then (1) becomes, $\frac{\partial u}{\partial t} = \nu \frac{\partial^2 u}{\partial z^2}$ --- (2)

Subjected to $u = 0$ at $z = 0$
and $u = U(t)$

$$= U \cos \omega t \text{ at } z = h.$$

- Assume a $u(x, t)$

$$\omega, \quad u(x, t) = \text{Re} [f(z) e^{i\omega t}]$$

- real part is in phase with $\cos(\omega t)$

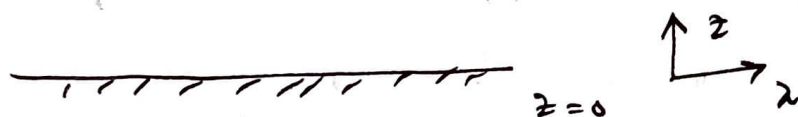
and imaginary part is in phase with $\sin(\omega t)$.

from (2) we get, $i\omega f = \nu \frac{d^2 f}{dz^2}$

or determine boundary layers:

Steady flow past a flat plate

$z \rightarrow \infty$
↑



$$0 < x < L$$

$$u = U \text{ as } z \rightarrow \infty.$$

$$u = u(x, z)$$

$$\omega = \omega(x, z)$$

$$\frac{\partial u}{\partial x} + \frac{\partial \omega}{\partial z} = 0 \quad \text{--- (1)}$$

$$u \frac{\partial u}{\partial x} + \omega \frac{\partial u}{\partial z} = - \frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial z^2} \right) \quad \text{--- (2)}$$

(we need laminar boundary layer theory because we can not obtain certain flow properties for flow past a flat plate using plain Görtler flow solution while letting $z \rightarrow \infty$).

(~~the~~ plane Couette flow ~~and~~ solutions suggest viscous effects will persist as far away from the plate).

(the unsteady motion past a flat plate allowed us to develop necessary solutions to take into account viscous diffusion as we go away from the plate).

Next, we study stream flow past a flat plate).

~~the~~ ~~total~~

$$u \frac{\partial \omega}{\partial x} + \omega \frac{\partial \omega}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial z} + \nu \left(\frac{\partial^2 \omega}{\partial x^2} + \frac{\partial^2 \omega}{\partial z^2} \right) \quad (2)$$

Subjected to, $(u, \omega) = 0$ at $z = 0$

and, $(u, \omega) = (U, 0)$ at ~~the~~ large z .

To make any further progress from this point onward we use order of magnitude analysis.

~~this~~ ~~gives~~ - boundary layer thickness is δ .

- $u \approx U$ immediately outside the boundary layer, u vary very rapidly from $u=0$ at $z=0$ to $u=U$ ~~at~~ on the other hand, ~~the~~ ω vary from 0 to 0.

- $\frac{\partial u}{\partial z}$ is $O\left(\frac{U}{\delta}\right) \Rightarrow \frac{\partial u}{\partial x}$ is $O\left(\frac{U}{L}\right)$.

- $\frac{\partial u}{\partial x} + \frac{\partial^2 u}{\partial z^2} = 0 \rightarrow \frac{\partial^2 u}{\partial z^2} \approx \frac{\partial \omega}{\partial z}$

- this gives, $\omega \approx 0$.

thus, we have,

$$u \frac{\partial u}{\partial x} + w \frac{\partial u}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \frac{\partial^2 u}{\partial z^2}$$

$$0 = -\frac{1}{\rho} \frac{\partial p}{\partial x}$$

at very large z , $\frac{\partial p}{\partial x}$ is what gives U .

$$\therefore, \text{there, } 0 + 0 = -\frac{1}{\rho} \frac{\partial p}{\partial x} + 0$$

$$\text{or, } \frac{\partial p}{\partial x} = 0 \text{ at very large } z.$$

- pressure outside the boundary layer is impressed upon the boundary layer.

- thus, ①, ② and ③ becomes,

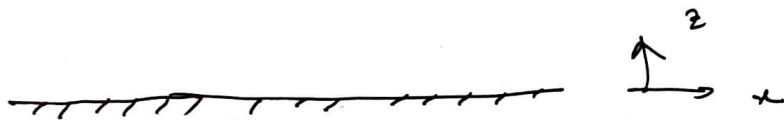
$$u \frac{\partial u}{\partial x} + w \frac{\partial u}{\partial z} - \nu \frac{\partial^2 u}{\partial z^2} = 0$$

$$\frac{\partial u}{\partial x} + \frac{\partial w}{\partial z} = 0.$$

and

unsteady motion over flat plate

$$\left(\begin{array}{l} w = 0 \text{ at } z = 0; \\ \frac{\partial w}{\partial z} = 0, \text{ given, } \\ w = 0 \end{array} \right)$$



$$\left. \begin{array}{l} u(t) \\ = U_0 \cos(\omega t) \end{array} \right\} \begin{array}{l} u = U(t) \text{ at } z = 0, \\ u = 0 \text{ as } z \rightarrow \infty. \end{array} \quad \left\{ \begin{array}{l} \text{Boundary} \\ \text{conditions.} \end{array} \right.$$

$$\frac{\partial u}{\partial x} + \frac{\partial w}{\partial z} = 0. \quad \text{①} \quad u = u(x, t)$$

$$\therefore \frac{\partial w}{\partial z} = 0.$$

$$\therefore \frac{\partial u}{\partial x} = 0.$$

$$u(x, t) = \text{Re} [f(z) \cdot e^{i\omega t}]$$

$$\begin{aligned} \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + w \frac{\partial u}{\partial z} &= -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial z^2} \right) \\ \frac{\partial u}{\partial t} &= \nu \frac{\partial^2 u}{\partial z^2}. \quad \text{②} \end{aligned}$$

-②.

$$u(z, t) = \operatorname{Re} [f(z) \cdot e^{i\omega t}] \quad - \textcircled{3} \quad \textcircled{4}$$

$$f(z) = A \cdot e^{k_1 z} + B \cdot e^{k_2 z}$$

Substitution of $f(z)$ into $\textcircled{1}$ gives,
$$\begin{cases} k_1 = (1+i) \sqrt{\frac{\omega}{2\nu}} \\ k_2 = -(1+i) \sqrt{\frac{\omega}{2\nu}} \end{cases}$$

$$u(z, t) = \operatorname{Re} [A \cdot e^{k_1 z} \cdot e^{i\omega t} + B \cdot e^{k_2 z} \cdot e^{i\omega t}]$$

$$= \operatorname{Re} [A \cdot e^{\sqrt{\frac{\omega}{2\nu}} \cdot z} \cdot e^{i \left(\sqrt{\frac{\omega}{2\nu}} z + \omega t \right)} + B \cdot e^{-\sqrt{\frac{\omega}{2\nu}} \cdot z} \cdot e^{i \left(-\sqrt{\frac{\omega}{2\nu}} z + \omega t \right)}]$$

$$= \operatorname{Re} [A \cdot e^{\sqrt{\frac{\omega}{2\nu}} \cdot z} \left\{ \cos \left(\sqrt{\frac{\omega}{2\nu}} z + \omega t \right) + i \sin \left(\sqrt{\frac{\omega}{2\nu}} z + \omega t \right) \right\} + B \cdot e^{-\sqrt{\frac{\omega}{2\nu}} \cdot z} \left\{ \cos \left(-\sqrt{\frac{\omega}{2\nu}} z + \omega t \right) + i \sin \left(-\sqrt{\frac{\omega}{2\nu}} z + \omega t \right) \right\}]$$

$$= A \cdot e^{\sqrt{\frac{\omega}{2\nu}} \cdot z} \cos \left(\sqrt{\frac{\omega}{2\nu}} \cdot z + \omega t \right) + B \cdot e^{-\sqrt{\frac{\omega}{2\nu}} \cdot z} \cos \left(-\sqrt{\frac{\omega}{2\nu}} z + \omega t \right)$$

for, $z=0$, $u = u_0 = u_0 \cos(\omega t)$.

or, $u_0 \cos(\omega t) = A \cdot (1) \cdot \cos(\omega t) + B \cdot (1) \cdot \cos(\omega t)$.

for, $z \rightarrow \infty$, $u = 0$.

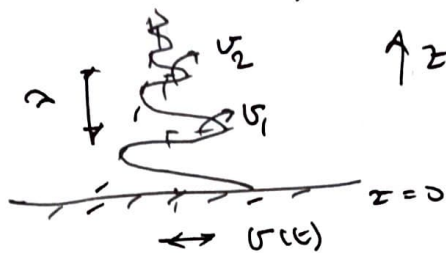
$$0 = A \cdot e^{\sqrt{\frac{\omega}{2\nu}} \cdot z} \cos \left(\sqrt{\frac{\omega}{2\nu}} \cdot z + \omega t \right) + B \cdot e^{-\sqrt{\frac{\omega}{2\nu}} \cdot z} \cos \left(-\sqrt{\frac{\omega}{2\nu}} z + \omega t \right)$$

$\therefore A = 0$, $B = u_0$.

$$\therefore u(z, t) = u_0 \cdot e^{-\sqrt{\frac{\omega}{2\nu}} \cdot z} \cos \left(\omega t - \sqrt{\frac{\omega}{2\nu}} \cdot z \right)$$

$$= u_0 \cdot e^{-\sqrt{\frac{\omega}{2\nu}} \cdot z} \cos \left(\sqrt{\frac{\omega}{2\nu}} \cdot z - \omega t \right)$$

- for water, $\nu = 10^{-6} \text{ m}^2/\text{s}$;
- the velocity profile refers to a traverse damped oscillatory motion with wave length $\frac{2\pi}{\sqrt{\frac{\omega}{2\nu}}}$ or, $2\pi \sqrt{\frac{2\nu}{\omega}}$.



$$\frac{u_2}{u_1} = e^{-\sqrt{\frac{\omega}{2\nu}} \cdot (z_2 - z_1)} \quad u_2 < u_1$$

$$\begin{aligned} \frac{u_2}{u_1} &= e^{-\sqrt{\frac{\omega}{2\nu}} \cdot \lambda} \\ &= e^{-\sqrt{\frac{\omega}{2\nu}} \cdot 2\pi \sqrt{\frac{2\nu}{\omega}}} = e^{-2\pi} \\ &= 0.002. \end{aligned}$$

for water $\nu = 10^{-6} \text{ m}^2/\text{s}$; and if $\omega = 2\pi \text{ rad/s}$;

then $\lambda = 2\pi \sqrt{\frac{2\nu}{\omega}} = 2\pi \sqrt{\frac{2 \times 10^{-6}}{2\pi}} = 2\pi \times 0.0014$

$= 2\pi \times \sqrt{\frac{2}{\pi}} \times 10^{-3} = 2\pi \sqrt{3.16 \times 10^{-3}} = 2 \times 10^{-3} \times \sqrt{0.318} = 0.56$

$=$

for water $\nu = 10^{-6} \text{ m}^2/\text{s}$ and if $\omega = 2\pi \text{ rad/s}$; then

$\lambda = 2\pi \sqrt{\frac{2\nu}{\omega}} = 2\sqrt{\frac{10^{-6}}{\pi}} \times \pi = 2\sqrt{\pi} \times 10^{-3} = 2 \times 1.77 \times 10^{-3}$

$= 3.54 \times 10^{-3} \text{ m} = 3.54 \times 10^{-3} \times 10^2 \text{ cm}$

$= 0.35 \text{ cm}; \quad \text{and} \quad \frac{u_2}{u_1} = e^{-2\pi}$

$= 0.002;$

- thus, the thickness of the layer in which the effect of the velocity of the oscillating plate is present is very small, unless, the frequency of oscillation of the plate is very small.

- For most of our hydrodynamic application the motion is ~~not~~ reasonably ~~low~~ high. Thus, the boundary layer thickness is generally very small.

- Thus the unsteady flow problem past a ~~flat~~ oscillating flat plate suggests that the boundary layer where we expect to have most of viscous effect is very thin.

- This ~~is~~ information we can get obtain from studying the solutions for plain Couette flow.

$$\begin{aligned}\tau &= \mu \frac{\partial u}{\partial z} \Big|_{z=0} = \mu \frac{\partial}{\partial z} \left[u_0 \cdot e^{-\sqrt{\frac{\omega}{2\nu}} \cdot z} \cos \left(\sqrt{\frac{\omega}{2\nu}} \cdot z - \omega t \right) \right] \\ &= \mu \left[u_0 \left(-\sqrt{\frac{\omega}{2\nu}} \right) \cdot e^{-\sqrt{\frac{\omega}{2\nu}} \cdot z} \cos \left(\sqrt{\frac{\omega}{2\nu}} \cdot z - \omega t \right) \right. \\ &\quad \left. - u_0 \cdot e^{-\sqrt{\frac{\omega}{2\nu}} \cdot z} \sin \left(\sqrt{\frac{\omega}{2\nu}} \cdot z - \omega t \right) \cdot \sqrt{\frac{\omega}{2\nu}} \right] \\ &= -\mu u_0 \cdot e^{-\sqrt{\frac{\omega}{2\nu}} \cdot z} \cdot \sqrt{\frac{\omega}{2\nu}} \left\{ \cos \left(\sqrt{\frac{\omega}{2\nu}} \cdot z - \omega t \right) \right. \\ &\quad \left. + \sin \left(\sqrt{\frac{\omega}{2\nu}} \cdot z - \omega t \right) \right\}\end{aligned}$$

- Due to oscillatory motion of plate, the direction of stress oscillates as well.

- τ is maximum at $z=0$.

Prandtl boundary layer equations

$$\left. \begin{aligned} u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial z} &= -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \frac{\partial^2 u}{\partial z^2} \\ \frac{\partial u}{\partial x} + \frac{\partial v}{\partial z} &= 0. \end{aligned} \right\} \text{--- (1)}$$

Since the order of magnitude analysis gives $\frac{\partial p}{\partial z} = 0$.

across the boundary layer, $p = p(x)$ and thus

We can substitute $\frac{\partial p}{\partial x}$ with $\frac{dp}{dx}$ in equation (1).

Using ~~equation~~ ~~equation~~ equation (1), we ~~want~~ want

to describe a boundary layer whose thickness $\delta(x)$ ~~is~~ increased ~~monotonically~~ monotonically with x . This is ~~not~~ possible if we have an adverse pressure gradient. ~~on~~ on the other hand, if the pressure gradient ϕ is favourable, then only $\delta(x)$ will increase along x . If there is an

adverse pressure gradient present inside the boundary layer, the pressure downstream ~~will~~ will try to prevent the flow ~~and~~ and also the viscous drag tries to retard the motion. In this case, boundary layer separation may take place.

Boundary layer theory does not work if there is a boundary layer separation.

~~Final~~ ~~for~~ ~~for~~ ~~for~~

$$\begin{aligned} u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial z} &= -\frac{1}{\rho} \frac{dp}{dx} + \nu \frac{\partial^2 u}{\partial z^2} \\ O\left(\frac{U^2}{L}\right) + (?) &= O\left(\nu \frac{U}{\delta^2}\right) \quad \text{--- (2)} \end{aligned}$$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial z} = 0$$

$$O\left(\frac{U}{L}\right) + O\left(\frac{V}{\delta}\right) = 0$$

$$V \sim O\left(\frac{U}{L}\right) \delta$$

$$\sim O\left(U \frac{\delta}{L}\right)$$

∴

Thus, if $\delta \ll L$;

$$V \ll U.$$

thus, order of magnitude analysis from 8 given,

$$O\left(\frac{U^2}{L}\right) + O\left(\nu \left(\frac{\delta}{L}\right) \left(\frac{U}{\delta}\right)\right) = O\left(2 \cdot \frac{\nu U}{\delta^2}\right)$$

$$\text{or, } O\left(\frac{U^2}{L}\right) = O\left(2 \cdot \frac{\nu U}{\delta^2}\right)$$

$$\text{or, } O(\delta^2) = O\left(\frac{2\nu L}{U}\right) = O\left(\frac{2\nu L}{U}\right)$$

$$Re = \frac{\rho U L}{\mu}$$

$$= O\left(\frac{L^2}{Re}\right)$$

$$Re = \frac{\rho U L}{\mu}$$

$$\text{or, } O(\delta) = O\left(Re^{-\frac{1}{2}} L\right) = \frac{L}{Re^{\frac{1}{2}}}$$

$$\text{or, } \boxed{O\left(\frac{\delta}{L}\right) = O\left(Re^{-\frac{1}{2}}\right)}$$

key assumption behind Prandtl boundary layer equations.

$$\textcircled{1} \quad \frac{\delta}{L} \ll 1 ; \quad (Re \text{ is large})$$

$\textcircled{2}$ Boundary layer separation does not take place, or pressure gradient is favourable.

- Boundary layer equations are simplified version of Navier-Stokes equation applicable only within the boundary layer when thickness of the boundary layer is very small compared to the length along x , i.e. $\delta \ll L$