

Homework ... the last one.

The goal of this worksheet is to put into practice what we learned about finite element matrices formed from linear elements on triangular meshes - and then have a bit of fun with that code

Background

The following is a quick summary of what we discussed in class about the FEM for linear elements in triangular meshes. Problem 1 will ask you to program this and test that code.

Local Mass and Stiffness Matrices

For the k^{th} triangular element in a mesh, whose vertices are

$$\mathbf{r}_1, \mathbf{r}_2, \mathbf{r}_3,$$

we can express the edges of the triangle as

$$\mathbf{E}_1 = \mathbf{r}_3 - \mathbf{r}_2$$

$$\mathbf{E}_2 = \mathbf{r}_1 - \mathbf{r}_3$$

$$\mathbf{E}_3 = \mathbf{r}_2 - \mathbf{r}_1$$

(where we use a naming convention that \mathbf{E}_i is the edge opposite \mathbf{r}_i , and oriented so the edges point clockwise around the triangle), and the area of the triangle as

$$A = \frac{1}{2} \|\mathbf{E}_1 \times \mathbf{E}_2\|_2.$$

Given these definitions, we can express the **local** stiffness and mass matricees (S^k and M^k) as

$$S^k = \frac{1}{4A} \begin{bmatrix} \mathbf{E}_1 \cdot \mathbf{E}_1 & \mathbf{E}_1 \cdot \mathbf{E}_2 & \mathbf{E}_1 \cdot \mathbf{E}_3 \\ \mathbf{E}_2 \cdot \mathbf{E}_1 & \mathbf{E}_2 \cdot \mathbf{E}_2 & \mathbf{E}_2 \cdot \mathbf{E}_3 \\ \mathbf{E}_3 \cdot \mathbf{E}_1 & \mathbf{E}_3 \cdot \mathbf{E}_2 & \mathbf{E}_3 \cdot \mathbf{E}_3 \end{bmatrix}, \quad M^k = \frac{A}{12} \begin{bmatrix} 2 & 1 & 1 \\ 1 & 2 & 1 \\ 1 & 1 & 2 \end{bmatrix} \quad (1)$$

Global Mass and Stiffness Matrices

Say that we have a list of N_v 3D vertices, V , and a list of N_t triangles, T . To get the vertices k_1, k_2, k_3 of the k^{th} triangle, and their 3D positions, in MATLAB we would do something like

```
% get vertex indices of k^th triangle as 1x3 vector of integers
tri = T(k,:);
% parse each index from 'tri'
k_1 = tri(1);
k_2 = tri(2);
k_3 = tri(3);
% get the 3D positions of the vertices
r_1 = V(k_1,:);
r_2 = V(k_2,:);
r_3 = V(k_3,);
```

The previous section tells you how to compute local mass and stiffness matrices from $\mathbf{r}_1, \mathbf{r}_2, \mathbf{r}_3$ but we still need to assemble the **global** mass and stiffness matrices from the local pieces. For instance, the $(1, 3)$ entry of the k^{th} *local* stiffness matrix contributes to the (k_1, k_3) entry of the *global* stiffness matrix - and the full *global* stiffness matrix is the **sum** of all of the local stiffness matrices. In math, this means that

$$\begin{aligned}\mathbf{S}_{global}^k(k_i, k_j) &= \mathbf{S}^k(i, j) \\ \mathbf{M}_{global}^k(k_i, k_j) &= \mathbf{M}^k(i, j)\end{aligned}$$

for $i, j = 1, 2, 3$, and

$$\begin{aligned}\mathbf{S} &= \sum_{k=1}^{N_t} \mathbf{S}_{global}^k \\ \mathbf{M} &= \sum_{k=1}^{N_t} \mathbf{M}_{global}^k\end{aligned}$$

[Note: be sure to use sparse matrices here since you'll quickly run out of memory if you allocate big matrices with lots of zeros]

Problem 1: Solving a Poisson equation on a sphere

Based on the previous section, implement a function to compute \mathbf{S} and \mathbf{M} from a $N_v \times 3$ array of vertices and an $N_t \times 3$ array of triangles.

To test this function, we'll try to solve the Poisson equation on the surface of a sphere

$$\Delta_{sphere} u = f \tag{2}$$

$$\frac{1}{\sin^2(\theta)} \frac{\partial^2 u}{\partial \phi^2} + \frac{1}{\sin(\theta)} \frac{\partial}{\partial \theta} \left(\sin(\theta) \frac{\partial u}{\partial \theta} \right) = f(\theta, \phi), \tag{3}$$

where $\theta \in [0, \pi]$ and $\phi \in [0, 2\pi]$. Specifically, we'll take

$$f = -2 \cos(\theta)$$

so that the exact solution is

$$u = \cos(\theta)$$

Note that $u = \cos(\theta) + c$ also solves the Poisson problem for any constant c , but choosing $c = 0$ gives us the solution with the smallest 2-norm.

For our finite element problem we need a good triangulation of the sphere which is provided by icosphere on Matlab's file exchange (a similar version can be pip installed with python). Further, we need basis coefficients f_i such that

$$f \approx \sum_i^{N_v} f_i \psi_i(\mathbf{x})$$

where $\psi_i(\mathbf{x})$ are the linear hat functions defined on our mesh. This approximation of f is just linear Lagrange interpolation, so we can recognize f_i as f evaluated at the position of the i^{th} vertex. In MATLAB this can be accomplished as something like

```
% V is an N_v X 3 array of vertices
f = @(th,ph) -2*cos(th);
[phi, theta, ~] = cart2sph(V(:, 1), V(:, 2), V(:, 3));
theta = theta + pi/2; %put theta in [0,pi]
f_i = f(theta,phi);
```

Now we can find a weak solution to our Poisson problem by solving

$$\mathbf{S}\mathbf{u} = \mathbf{M}\mathbf{f}$$

for $\mathbf{u} = [u_i]$, such that

$$u^{FEM} \approx \sum_i^{N_v} u_i \psi_i(\mathbf{x}).$$

But you will see that \mathbf{S} is singular!. This is because the original PDE does not have a unique solution either (only determined up to a constant) so a vector of all 1s is in the nullspace of \mathbf{S} . To get around this we seek a solution with minimal norm (like we did with the PDE) and this is accomplished by solving instead

$$(\mathbf{S} + \epsilon \mathbf{I}) \mathbf{u} = \mathbf{M}\mathbf{f}$$

where ϵ is some small number (just take $\epsilon = 10^{-8}$). This approach is equivalent to simulating heat flow over a long time since the Poisson equation represents the steady state.

We can evaluate the accuracy of our solution using the exact coefficients u_i^{exact} , computed similar to f_i as

```
% V is an N_v X 3 array of vertices
u_ex = @(th,ph) cos(th);
[phi, theta, ~] = cart2sph(V(:, 1), V(:, 2), V(:, 3));
theta = theta + pi/2; %put theta in [0,pi]
u_ex_i = u_ex(theta,phi);
```

the L^2 error in the solution is computed as

$$\begin{aligned} \|u^{\text{exact}} - u^{FEM}\|_{L^2}^2 &= \int_{S^2} \left(\sum_i^{N_v} [u_i^{\text{exact}} - u_i^{FEM}] \psi_i(\mathbf{x}) \right)^2 \\ &= \int_{S^2} \sum_j^{N_v} [u_j^{\text{exact}} - u_j^{FEM}] \psi_j(\mathbf{x}) \left(\sum_i^{N_v} [u_i^{\text{exact}} - u_i^{FEM}] \psi_i(\mathbf{x}) \right) \\ &= \sum_j^{N_v} [u_j^{\text{exact}} - u_j^{FEM}] \left(\sum_i^{N_v} [u_i^{\text{exact}} - u_i^{FEM}] \left\{ \int_{S^2} \psi_j(\mathbf{x}) \psi_i(\mathbf{x}) \right\} \right) \\ &= [\mathbf{u}^{\text{exact}} - \mathbf{u}^{FEM}]^T \mathbf{M} [\mathbf{u}^{\text{exact}} - \mathbf{u}^{FEM}] \end{aligned}$$

Show that the error decreases quadratically as you refine the mesh by factors of 2 using a log-log plot.

I know this was a lot of background, but the crux of the question is simple:

1. Define a mesh and compute the smallest edge length (smallest edge length is easy to find but in general it's a bad measure of the mesh size)

```
(a) % N defines the level of refinement for the sphere.  
    % Increasing N by 1 roughly halves the average edge length  
    N = 0;  
    [V,T] = icosphere(N);  
    h_mesh(N+1) = min(pdist(V));
```

2. Compute \mathbf{S}, \mathbf{M}

```
(a) [S,M] = get_FEM_Mats(V,T);
```

3. Compute the RHS and exact solution vectors u_i^{exact}, f_i

```
(a) u_ex = @(th,ph) cos(th);  
f = @(th,ph) -2*cos(th);  
[phi, theta, ~] = cart2sph(V(:, 1), V(:, 2), V(:, 3));  
theta = theta + pi/2; %put theta in [0,pi]  
u_ex_i = u_ex(theta,phi);  
f_i = f(theta,phi);
```

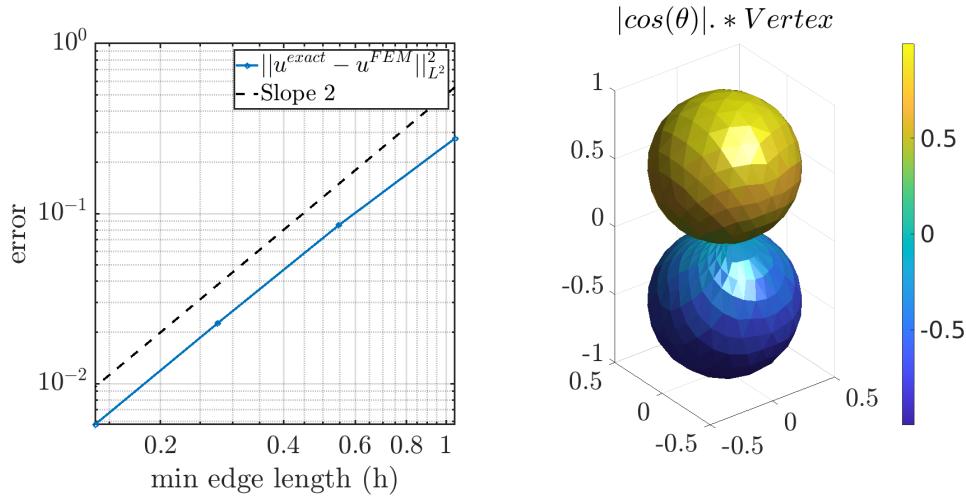
4. Solve the linear system

$$(\mathbf{S} + \epsilon \mathbf{I}) \mathbf{u}^{FEM} = \mathbf{M} \mathbf{f}$$

5. compute and save the L^2 error

$$\text{Error}(N) = \sqrt{[\mathbf{u}^{\text{exact}} - \mathbf{u}^{FEM}]^T \mathbf{M} [\mathbf{u}^{\text{exact}} - \mathbf{u}^{FEM}]}$$

6. Refine the mesh by increasing N by 1 and repeat from step 1



Problem 2

Do something fun with the new code that you have! The following are a few suggestions, but feel free to do whatever you want here. I'm excited to see what you come up with!

1. Compute the eigen vectors of the discrete Laplacian $Lap = M^{-1}S$ and plot some of them using e.g

```
f = abs(repmat(e_vec,1,3)).*V;
C = V;
h = trisurf(tri, f(:,1), f(:,2), f(:,3),C);
set(h,'edgecolor','none')
daspect([1 1 1])
```

these eigen vectors are discrete approximations to the *eigen functions* of the Laplacian on a sphere and they're called *spherical harmonics*

2. Solve the heat or wave equation on sphere. For example, the heat equation can be discretized using e.g backward euler

$$\mathbf{M} (\mathbf{u}^{n+1} - \mathbf{u}^n) = \mathbf{S} \mathbf{u}^{n+1}$$

3. Solve the heat equation but the unknowns are the vertices themselves! One can show that this is equivalent to what's called mean curvature flow and it's discretized very simply as

$$\mathbf{M} (\mathbf{V}^{n+1} - \mathbf{V}^n) = \mathbf{S} \mathbf{V}^{n+1}$$

where \mathbf{V}^n is the $N_v \times 3$ array of vertices. **This looks best on a mesh that isn't a sphere, try 'the stanford bunny'**