

Project 9: Coupled Systems

Abstract

In this project, we were investigating eigenvalues and eigenvectors in a coupled mass system. The physical context was the oscillation of three equal masses coupled by four springs of equal spring constants. We used the python "eig" function to compute the eigenvalues and eigenvectors for this system. Three eigenvalues were found for this system because this system involves three coupled masses. The eigenvalues depends on our values for k and m, which are 10 Nm^{-1} and 1kg respectively. The eigenvalues of this system were computed to be: $-\omega^2 = [-34.142, -20.0, -5.858]$ We also found out that one of the normal mode frequencies was the square root of one of the eigenvalues.

Description

An eigenvector or characteristic vector of a linear transformation is a nonzero vector that changes at most by a scalar factor when that linear transformation is applied to it. The corresponding eigenvalue is the factor by which the eigenvector is scaled. They are a special set of scalars associated with a linear system of equations (i.e., a matrix equation) that are sometimes also known as characteristic roots, characteristic values, proper values, or latent roots.

Let A be a linear transformation represented by a matrix A . If there is a vector $x \in \mathbb{R}^n \neq 0$ such that $Ax = \lambda x$ for some scalar λ , then λ is called the eigenvalue of A with corresponding (right) eigenvector x .

Letting A be a $k \times k$ square matrix

$$\begin{bmatrix} a_{11} & a_{12} & \cdots & a_{1k} \\ a_{21} & a_{22} & \cdots & a_{2k} \\ \vdots & \vdots & \ddots & \vdots \\ a_{k1} & a_{k2} & \cdots & a_{kk} \end{bmatrix}$$

with eigenvalue λ , then the corresponding eigenvectors satisfy

$$\begin{bmatrix} a_{11} & a_{12} & \cdots & a_{1k} \\ a_{21} & a_{22} & \cdots & a_{2k} \\ \vdots & \vdots & \ddots & \vdots \\ a_{k1} & a_{k2} & \cdots & a_{kk} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_k \end{bmatrix} = \lambda \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_k \end{bmatrix}$$

which is equivalent to the homogeneous system

$$\begin{bmatrix} a_{11} - \lambda & a_{12} & \cdots & a_{1k} \\ a_{21} & a_{22} - \lambda & \cdots & a_{2k} \\ \vdots & \vdots & \ddots & \vdots \\ a_{k1} & a_{k2} & \cdots & a_{kk} - \lambda \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_k \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 0 \end{bmatrix}$$

Equation above can be written compactly as

$$(A - \lambda I)x = 0$$

where I is the identity matrix. As shown in Cramer's rule, a linear system of equations has nontrivial solutions if the determinant vanishes, so the solutions of equation above are given by

$$\det(A - \lambda I) = 0$$

This equation is known as the characteristic equation of A , and the left-hand side is known as the characteristic polynomial.

Algorithm and Discussion

Consider a system as shown in the figure below



Using Hooke's Law we can find the force acting on each mass. We compare this to Newton's definition of force. We put all these in matrix forms and we then find out that the eigenvalue is $-\omega^2$. We using eig function to find the eigenvalues and use the RK4 approach to validate that the eigenvalues computed are correct.

$$\begin{aligned} k_1 &= k_2 = k_3 = k_4 = k \\ m_1 &= m_2 = m_3 = m \\ F_1 &= -kx_1 + k(x_2 - x_1) \\ F_2 &= -k(x_2 - x_1) + k(x_3 - x_2) \\ F_3 &= -k(x_3 - x_2) + k(-x_3) \\ \hat{a}(t) &= -\omega^2 \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} e^{-i\omega t} \\ \hat{F}(x) &= \begin{bmatrix} -2k & k & 0 \\ k & -2k & k \\ 0 & k & -2k \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} e^{-i\omega t} \\ -m\omega^2 \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} e^{-i\omega t} &= \begin{bmatrix} -2k & k & 0 \\ k & -2k & k \\ 0 & k & -2k \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} e^{-i\omega t} \\ &= \begin{bmatrix} \frac{2k}{m} & \frac{-k}{m} & 0 \\ \frac{-k}{m} & \frac{2k}{m} & \frac{-k}{m} \\ 0 & \frac{-k}{m} & \frac{2k}{m} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} e^{-i\omega t} \\ &= \omega^2 \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} e^{-i\omega t} \\ \begin{bmatrix} \frac{2k}{m} - \omega^2 & \frac{-k}{m} & 0 \\ \frac{-k}{m} & \frac{2k}{m} - \omega^2 & \frac{-k}{m} \\ 0 & \frac{-k}{m} & \frac{2k}{m} - \omega^2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} &= 0 \\ \begin{vmatrix} \frac{2k}{m} - \omega^2 & \frac{-k}{m} & 0 \\ \frac{-k}{m} & \frac{2k}{m} - \omega^2 & \frac{-k}{m} \\ 0 & \frac{-k}{m} & \frac{2k}{m} - \omega^2 \end{vmatrix} &= 0 \end{aligned}$$

Implementation and Code

We use the eig function to find eigenvalues and eigenvectors then we use to Rk4 to validate the values we computed. We also investigate the motion of each masses with subplots.

```
In [7]: %matplotlib inline
import pandas as pd
import matplotlib.pyplot as plt
import numpy as np
import sympy as sp
from numpy import linalg as LA

k = 10
m = 1.0
Mat = np.array([[-(2*k)/m, k/m, 0], [k/m, -(2*k)/m, k/m], [0, k/m, -(2*k)/m]])

vals, vecs = LA.eig(Mat)

print("values:")
print(np.around(vals,3))
print("vector:")
print(np.around(vecs, 3))
```

```
values:
[-34.142 -20.      -5.858]
vector:
[[ 0.5   0.707  0.5 ]
 [-0.707  0.     0.707]
 [ 0.5  -0.707  0.5 ]]
```

Using the Rk4 to validate the eigenvalues. We start out our masses at the eigenvectors. We state our initial conditions to be the eigenvectors

```
In [8]: L = 10
k = 10
m = 1.0
dt = 0.03
t = 0.0

x1i = 0.5*L/10 # initial displacement from equil, m1
v1i = 0.0 # initial vel, m1
x2i = 0.71*L/10 # initial displacement from equil, m2
v2i = 0.0 # initial vel, m2
x3i = 0.5*L/10 # initial displacement from equil, m3
v3i = 0.0 # initial vel, m3

s = np.array([x1i, v1i, x2i, v2i, x3i, v3i]) # initial state

def derivs_3m(s, t):
    x1=s[0] # get the variables from the state
    v1=s[1]
    x2=s[2]
    v2=s[3]
    x3=s[4]
    v3=s[5]
    a1 = -(2*k*x1) + k*x2)/m
    a2 = -(2*k*x2 + k*x1 + k*x3)/m
    a3 = (k*x2 - (2*k)*x3)/m
    return np.array([v1, a1, v2, a2, v3, a3])

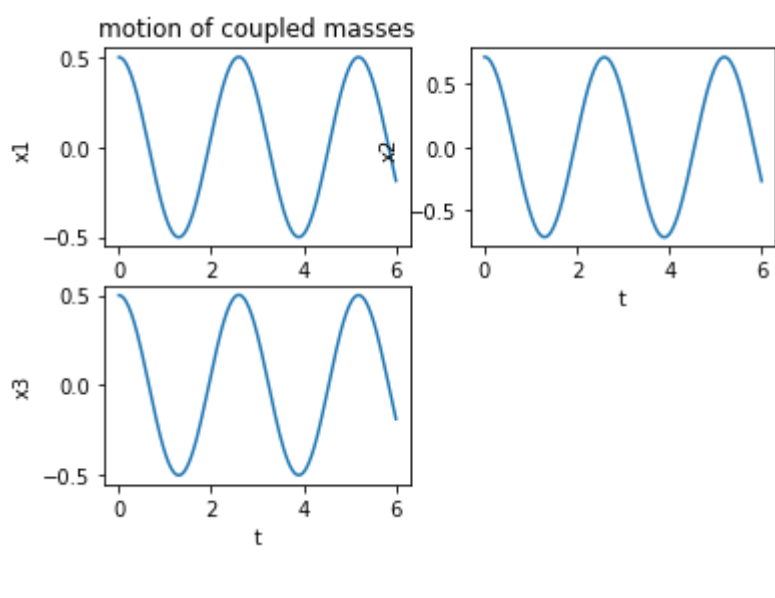
def RK4Step(s, dt, t, derivs):
    """
    Take a single RK4 step.
    """
    f1 = derivs(s, t)
    f2 = derivs(s+f1*dt/2.0, t+dt/2.0)
    f3 = derivs(s+f2*dt/2.0, t+dt/2.0)
    f4 = derivs(s+f3*dt, t+dt)
    return s + (f1+2*f2+2*f3+f4)*dt/6.0
```

```
In [9]: x1list=[s[0]]
x2list=[s[2]]
x3list=[s[4]]
tlist=[0.0]
t=0.0

while t<6:
    s = RK4Step(s, dt, t, derivs_3m)
    t += dt
    x1list.append(s[0])
    x2list.append(s[2])
    x3list.append(s[4])
    tlist.append(t)

plt.subplot(221)
plt.ylabel("x1")
plt.title("motion of coupled masses")
plt.plot(tlist,x1list,label="x1")
plt.subplot(222)
plt.ylabel("x2")
plt.plot(tlist,x2list,label="x2")
plt.subplot(223)
plt.ylabel("x3")
plt.plot(tlist,x3list,label="x3")
```

Out[9]: <matplotlib.lines.Line2D at 0x21f19d5be88>



We use curvefitting function to find one out of the three of the normal mode frequencies. I curvefitted for all three masses just to confirm that they did oscillate with the same frequency

```
In [10]: from scipy.optimize import curve_fit

def cosFit(t, A, omega, phi):
    """
    Function def for a cosine fit
    """
    return A*np.cos(omega*t+phi)

x1a=np.array(x1list)
ta=np.array(tlist)
popt, pcov = curve_fit(cosFit, ta, x1a, p0=(0.5, np.sqrt(5.858), 0.0))

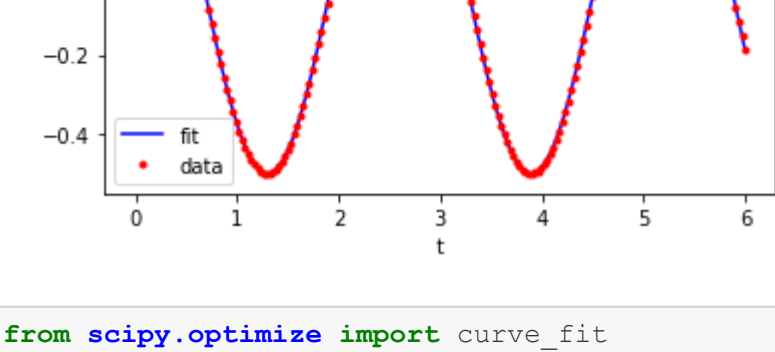
A=popt[0]
omega=popt[1]
phi=popt[2]

print("A =>", A)
print("omega**2 =>", omega**2)
print("phi =>", phi)

plt.title('Fit to find frequency.')
plt.xlabel('t')
plt.ylabel('x1')
plt.plot(ta, cosFit(ta, A, omega, phi), 'b-', label='fit')
plt.plot(ta, x1a, 'r.', label='data')
plt.legend()
```

```
A => 0.5009755240452879
omega**2 => 5.857865857639822
phi => -0.00011479642670829052
```

Out[10]: <matplotlib.legend.Legend at 0x21f19e56c08>



```
In [11]: from scipy.optimize import curve_fit

def cosFit(t, A, omega, phi):
    """
    Function def for a cosine fit
    """
    return A*np.cos(omega*t+phi)

x2a=np.array(x2list)
ta=np.array(tlist)
popt, pcov = curve_fit(cosFit, ta, x2a, p0=(0.7071, np.sqrt(5.858), 0.0))

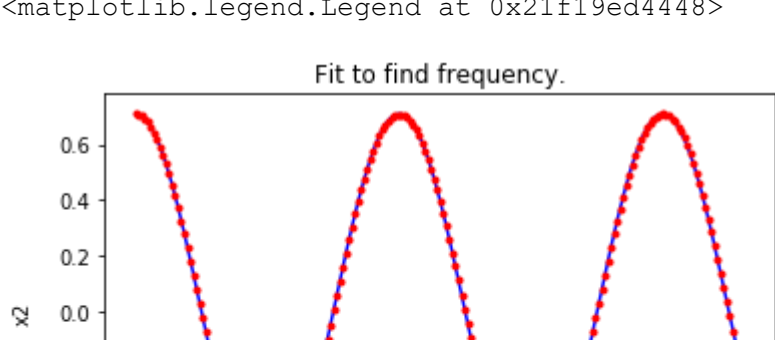
A=popt[0]
omega=popt[1]
phi=popt[2]

print("A =>", A)
print("omega**2 =>", omega**2)
print("phi =>", phi)

plt.title('Fit to find frequency.')
plt.xlabel('t')
plt.ylabel('x2')
plt.plot(ta, cosFit(ta, A, omega, phi), 'b-', label='fit')
plt.plot(ta, x2a, 'r.', label='data')
plt.legend()
```

```
A => 0.7086202744686608
omega**2 => 5.8578658571069845
phi => 0.00011501275702344706
```

Out[11]: <matplotlib.legend.Legend at 0x21f19ed4448>



```
In [12]: from scipy.optimize import curve_fit

def cosFit(t, A, omega, phi):
    """
    Function def for a cosine fit
    """
    return A*np.cos(omega*t+phi)

x3a=np.array(x3list)
ta=np.array(tlist)
popt, pcov = curve_fit(cosFit, ta, x3a, p0=(0.7071, np.sqrt(5.858), 0.0))

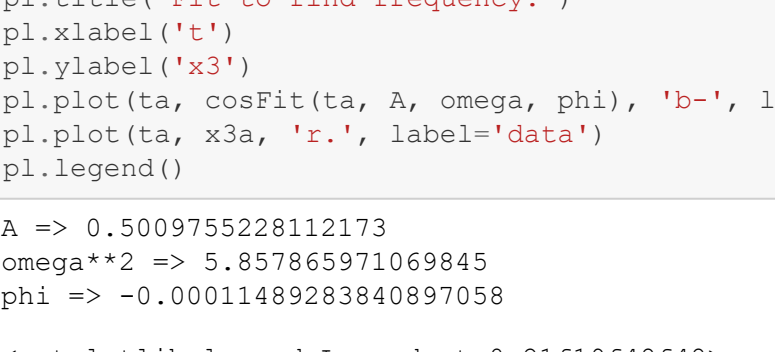
A=popt[0]
omega=popt[1]
phi=popt[2]

print("A =>", A)
print("omega**2 =>", omega**2)
print("phi =>", phi)

plt.title('Fit to find frequency.')
plt.xlabel('t')
plt.ylabel('x3')
plt.plot(ta, cosFit(ta, A, omega, phi), 'b-', label='fit')
plt.plot(ta, x3a, 'r.', label='data')
plt.legend()
```

```
A => 0.5009755228112173
omega**2 => 5.8578658571069845
phi => -0.00011489283840897058
```

Out[12]: <matplotlib.legend.Legend at 0x21f19f48f48>



After this fit, we found out that one of the normal mode frequency was the squareroot of one of the eigenvalues.

Conclusion

In this project, we were investigating eigenvectors and eigenvalues of a coupled system of three masses connected with four springs of equal spring constants. The eigenvalues of this system were computed to be: $-\omega^2 = [-34.142, -20.0, -5.858]$ We used Rk4 approach to validate one of the eigenvalues and we found out that one of the normal mode frequency of the system was the squareroot of one of the eigenvalues.