

Develop and Test of an Improved 4-node Axisymmetric (ring) Element for Nearly Incompressible Materials

PART I Formulation

AFEM Term Project

Reza Behrou, Farhad Shahabi

Contents

Problem Statement



Motivation



Finite Element Analysis



Hybrid Variational Principle

Problem Statement

✓ Axisymmetric Problems

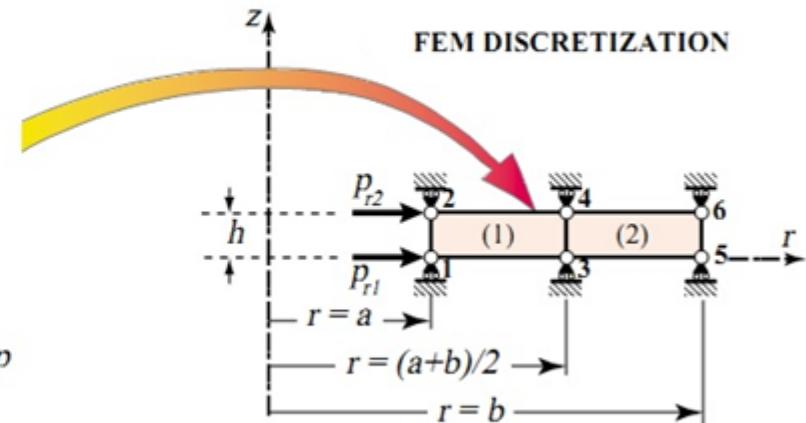
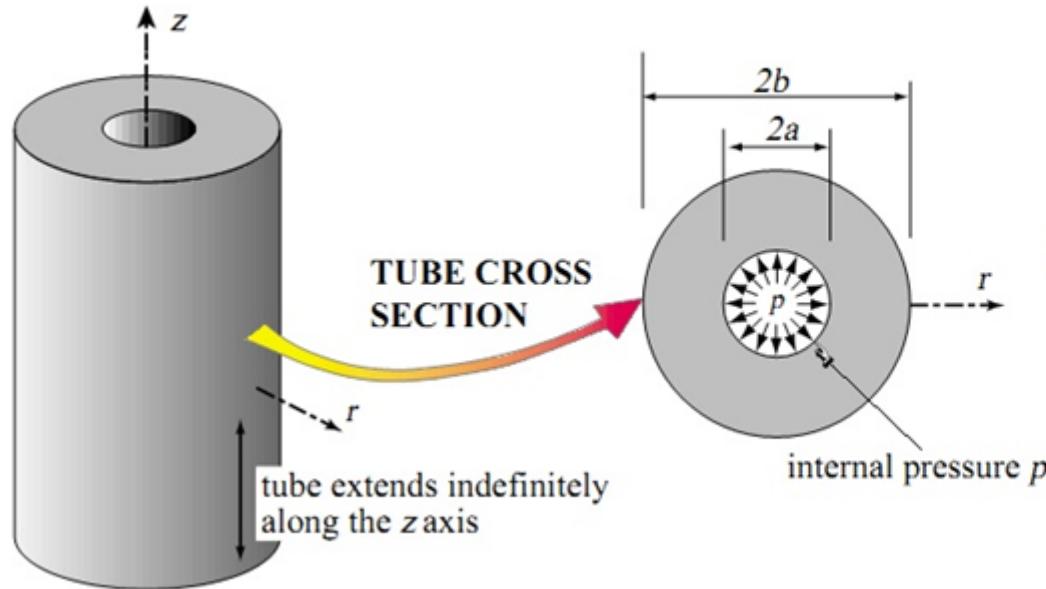
The axisymmetric problem deals with the analysis of structures of revolution under axisymmetric loading

Fabrication: Axisymmetric bodies are usually easier to manufacture than bodies with more complex geometries.

Strength: axisymmetric configurations are often optimal in terms of strength to weight ratio

Problem Statement

✓ Benchmark problem: a thick cylinder under internal pressure

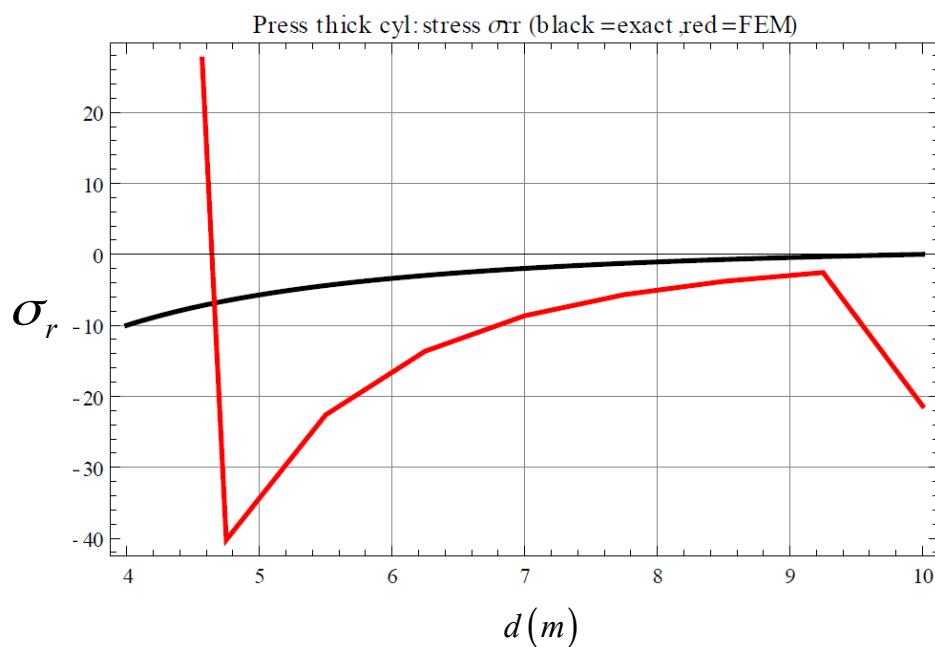
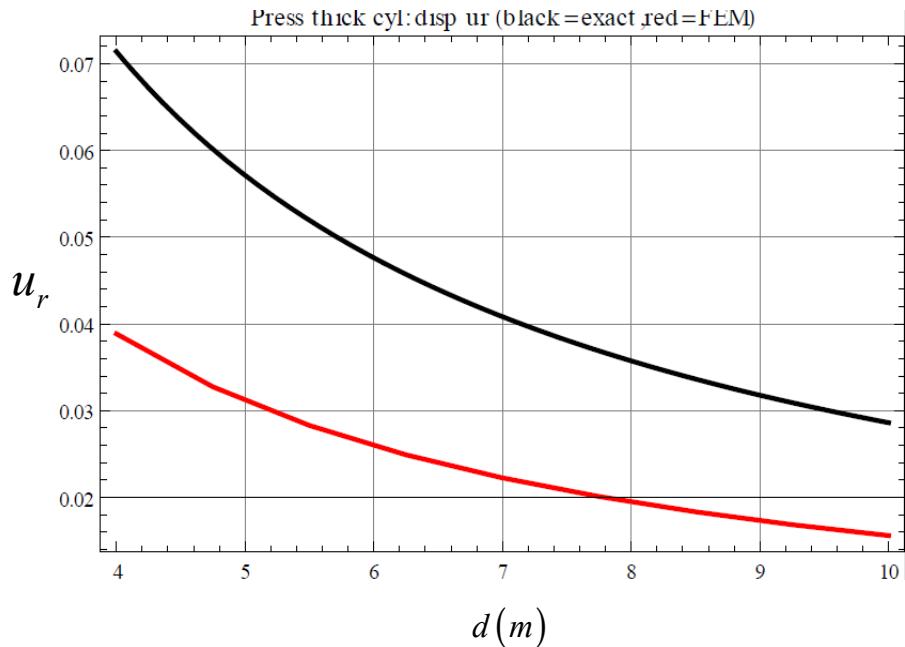


A thick cylinder under internal pressure discretized by the 4-node quadrilateral axisymmetric elements

Motivation

- ✓ The performance of standard 4-node isoparametric quadrilateral Element

$$v = 0.499$$

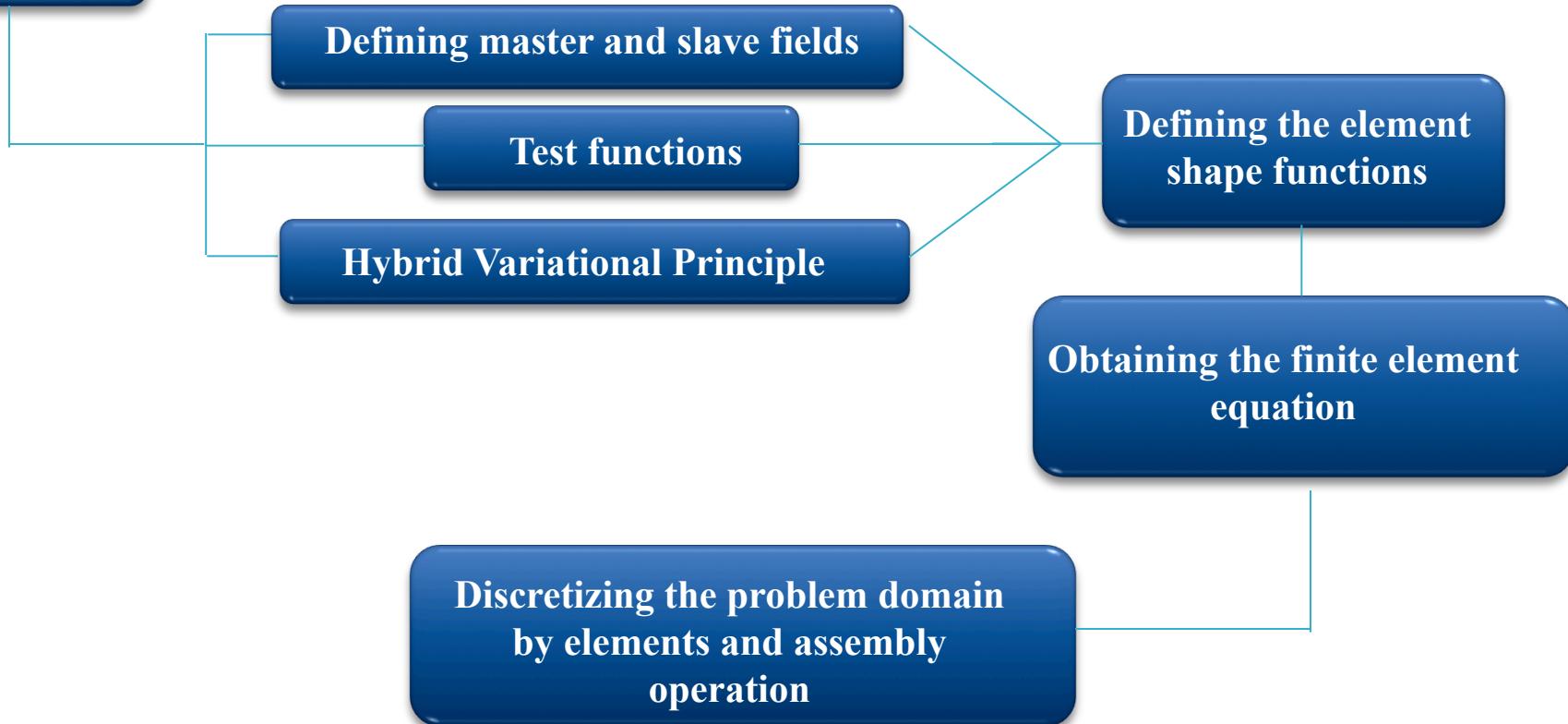


The accuracy of displacement and stress fields predicted by the standard 4-node quadrilateral for the near incompressible materials

Finite Element Analysis

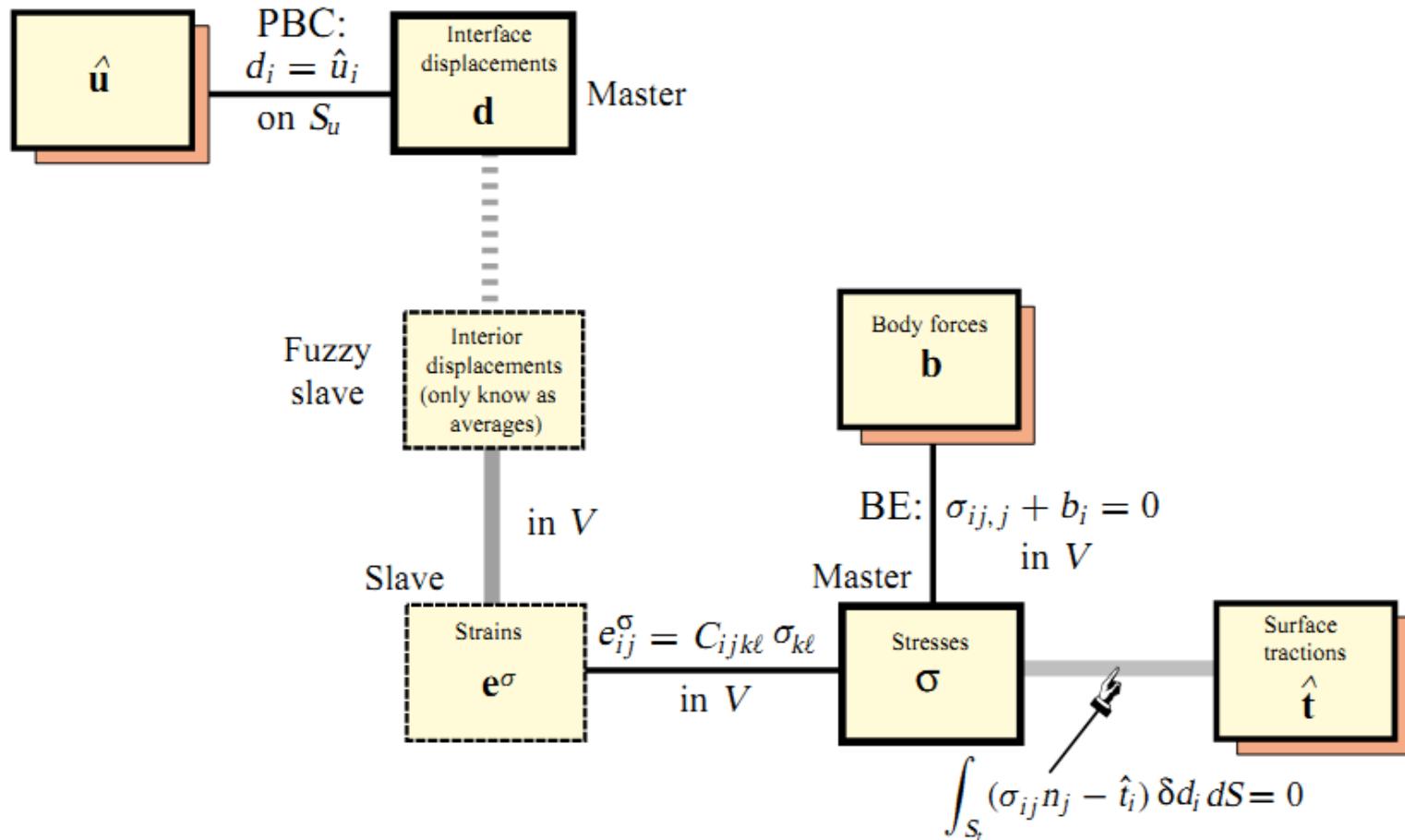
✓ The Finite Element Solution Chart

Finite Element Method



Hybrid Variational Principle

✓ Tonti Diagram



Schematics of Weak Form of the
equilibrium-stress-hybrid principle.

Hybrid Variational Principle

✓ Hybrid Variational Principle Formulation

$$\Pi_{Hybrid} = \sum_n \left(\int_{V_n} \frac{1}{2} C_{ijkl} \sigma_{ij} \sigma_{kl} dv - \int_s T_i u_i ds + \int_{s_T} \bar{T}_i u_i ds \right)$$

Assumed Stress Field

$$\{\sigma\} = [P] \{\beta\} + [P_F] \{\beta_F\} \quad \sigma_{ij,j} + b_i = \rho \ddot{u}_i$$

Balance of Linear momentum

Assumed Displacement Field

$$\{U\} = L \{u^e\}$$

Traction

$$\{T\} = [\sigma] \{n\}$$

Assumed Traction

$$\{T\} = [R] \{\beta\} + [R_F] \{\beta_F\}$$

$$\begin{aligned} \Pi_{Hybrid} &= \sum_n \left(\int_{V_n} \frac{1}{2} (\{\beta\}^T [H] \{\beta\} + \{\beta_F\}^T [H_{FF}] \{\beta_F\}) dv + \{\beta\}^T [H_F] \{\beta_F\} dv \right) \\ &\quad - \sum_n \left(- \int_s (\{\beta\}^T [R]^T L + \{\beta_F\}^T [R_F]^T L) \{u^e\} ds + \int_{s_T} \bar{T}_i L \{u^e\} ds \right) \end{aligned}$$

Hybrid Variational Principle

✓ Hybrid Variational Principle Formulation

$$H = \int_{V_n} [P]^T [C][P] dV \quad H_F = \int_{V_n} [P]^T [C][P_F] dV \quad H_{FF} = \int_{V_n} [P_F]^T [C][P_F] dV$$

Simplifying the problem by solving for the static case with the negligible body force

$$\beta_F = P_F = R_F = 0$$

$$\Pi_{Hybrid} = \sum_n \left(\int_{V_n} \frac{1}{2} (\{\beta\}^T [H] \{\beta\}) dV \right) - \sum_n \left(\int_s (\{\beta\}^T [R]^T L) \{u_i^e\} ds + \int_{S_T} \bar{T}_i L \{u_i^e\} ds \right)$$

$$\begin{aligned} \int_s (\{\beta\}^T [R]^T L) \{u_i^e\} ds &= \int_s T_i u_i ds \Rightarrow \int_s \sigma_{ij} u_i n_j ds \\ \Rightarrow \int_{V_n} \sigma_{ij,j} u_i dV + \int_{V_n} \sigma_{ij} u_{i,j} dV &= \int_{V_n} \sigma_{ij} \varepsilon_{ij} dV = \int_{V_n} [\sigma]^T [\varepsilon] dV \end{aligned}$$

Hybrid Variational Principle

✓ Hybrid Variational Principle Formulation

$$\Pi_{Hybrid} = \sum_n \left(\int_{V_n} \frac{1}{2} \left(\{\beta\}^T [H] \{\beta\} \right) dV \right) - \sum_n \left(\int_s \underbrace{\left(\{\beta\}^T [P]^T [B] \right)}_{\downarrow} \{u_i^e\} dV + \int_{S_T} \bar{T}_i L \{u_i^e\} ds \right)$$

$$G = \int_{V_n} [P]^T [B] dV$$

Taking the variation of the above functional WRT β

$$[H][\beta] - [G]\{u_i^e\} = 0 \Rightarrow [\beta] = [H]^{-1}[G]\{u_i^e\}$$

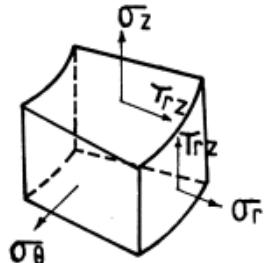
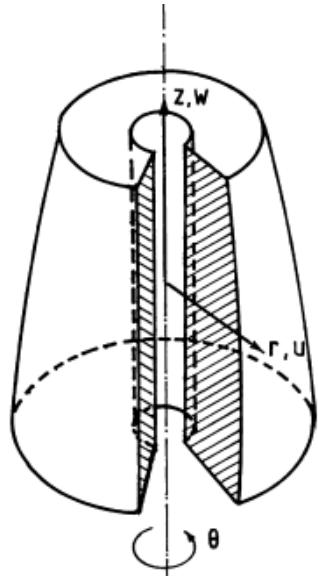
$$\Pi_{Hybrid} = - \sum_n \left(\int_{V_n} \frac{1}{2} \left(\{u_i^e\}^T [G]^T [H]^{-T} [G] \{u_i^e\} \right) dV \right) + \int_{S_T} \bar{T}_i L \{u_i^e\} ds$$

Taking the variation of the above functional WRT u_i^e

$$K = [G]^T [H]^{-T} [G]$$

Hybrid Variational Principle

✓ Hybrid Variational Principle Formulation for the Axisymmetric Problem



The schematic of axisymmetric element

$$\varepsilon_r = \frac{\partial u}{\partial r}, \quad \varepsilon_\theta = \frac{u}{r} \quad \text{and} \quad \varepsilon_z = \frac{\partial w}{\partial z}$$

$$\gamma_{rz} = \frac{\partial u}{\partial z} + \frac{\partial w}{\partial r}$$

$$\boldsymbol{\varepsilon} = C \boldsymbol{\sigma}$$

$$\boldsymbol{\sigma} = \begin{Bmatrix} \sigma_z & \sigma_\theta & \sigma_r & \tau_{rz} \end{Bmatrix}^T \quad \boldsymbol{\varepsilon} = \begin{Bmatrix} \varepsilon_z & \varepsilon_\theta & \varepsilon_r & \gamma_{rz} \end{Bmatrix}^T$$

$$u = \sum_{i=1}^4 N_i(\xi, \eta) u_i$$

$$r = \sum_{i=1}^4 N_i(\xi, \eta) r_i$$

$$w = \sum_{i=1}^4 N_i(\xi, \eta) w_i$$

$$z = \sum_{i=1}^4 N_i(\xi, \eta) z_i$$

$$N_i = \frac{1}{4} (1 + \xi \xi_i) (1 + \eta \eta_i)$$

Hybrid Variational Principle

✓ Hybrid Variational Principle Formulation for the Axisymmetric Problem

$$\frac{\partial \sigma_r}{\partial r} + \frac{\partial \tau_{rz}}{\partial z} + \frac{\sigma_r - \sigma_\theta}{r} = 0$$

$$\frac{\partial \sigma_z}{\partial z} + \frac{\partial \tau_{rz}}{\partial r} + \frac{\tau_{rz}}{r} = 0$$

$$\nabla^2 (\sigma_r + \sigma_\theta + \sigma_z) = \left(\frac{\partial^2}{\partial r^2} + \frac{1}{r} \frac{\partial}{\partial r} + \frac{\partial^2}{\partial z^2} \right) (\sigma_r + \sigma_\theta + \sigma_z) = 0$$

The Assumed stress functions

$$\sigma_z = \beta_6 + \beta_7 z - 3\beta_3 r - \beta_4 \frac{1}{r} - \beta_5 \frac{z}{r}$$

$$\sigma_\theta = \beta_1 + \beta_2 z + 2\beta_3 r$$

$$\sigma_r = \beta_1 + \beta_2 z + \beta_3 r + \beta_4 \frac{1}{r} + \beta_5 \frac{z}{r}$$

$$\tau_{rz} = \beta_5 - \frac{1}{2} \beta_7 r$$

$$n_\beta \geq n_{DOF} - n_{RB}$$

The Number of Stress Parameters

Hybrid Variational Principle

✓ Hybrid Variational Principle Formulation for the Axisymmetric Problem

$$\begin{bmatrix} \varepsilon_r \\ \varepsilon_z \\ \varepsilon_\theta \\ \gamma_{rz} \end{bmatrix} = \begin{bmatrix} \frac{\partial N_1^e}{\partial r} & 0 & \frac{\partial N_2^e}{\partial r} & 0 & \frac{\partial N_3^e}{\partial r} & 0 & \frac{\partial N_4^e}{\partial r} & 0 \\ 0 & \frac{\partial N_1^e}{\partial z} & 0 & \frac{\partial N_2^e}{\partial z} & 0 & \frac{\partial N_3^e}{\partial z} & 0 & \frac{\partial N_4^e}{\partial z} \\ \frac{N_1^e}{r} & 0 & \frac{N_2^e}{r} & 0 & \frac{N_3^e}{r} & 0 & \frac{N_4^e}{r} & 0 \\ \frac{\partial N_1^e}{\partial z} & \frac{\partial N_1^e}{\partial r} & \frac{\partial N_2^e}{\partial z} & \frac{\partial N_2^e}{\partial r} & \frac{\partial N_3^e}{\partial z} & \frac{\partial N_3^e}{\partial r} & \frac{\partial N_4^e}{\partial z} & \frac{\partial N_4^e}{\partial r} \end{bmatrix} \begin{bmatrix} u_i^e \end{bmatrix} = [B] \begin{bmatrix} u_i^e \end{bmatrix}$$

$$H = 2\pi \int_{-1}^1 \int_{-1}^1 P^T C P r \det|J| d\xi d\eta$$

$$[P] = \begin{bmatrix} 1 & z & r & \cancel{1/r} & \cancel{z/r} & 0 & 0 \\ 0 & 0 & -3r & -\cancel{1/r} & -\cancel{z/r} & 1 & z \\ 1 & z & 2r & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 & -\cancel{1/2r} \end{bmatrix}$$

$$G = 2\pi \int_{-1}^1 \int_{-1}^1 P^T B r \det|J| d\xi d\eta$$

$$K = [G]^T [H]^{-T} [G]$$

Thanks
For Your
Attention