

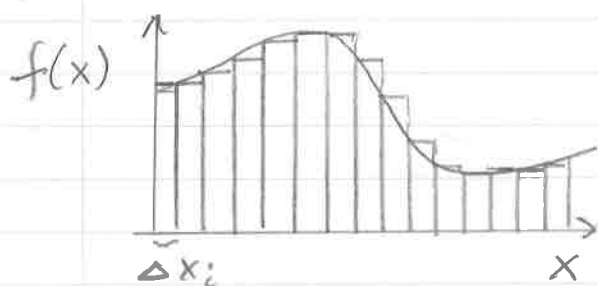
(Forbes 4.4)

Dynamics of a Continuous Rigid Body (CRB)

In reality, rigid bodies are not made up of discrete particles (unless we go to the atomic level). Therefore, DRBs are in reality approximations of continuous rigid bodies.

Def: A continuous rigid body (CRB) is a continuum in which the distance between any two points on the body is constant.

Recall how to calculate the area under a curve in Calculus



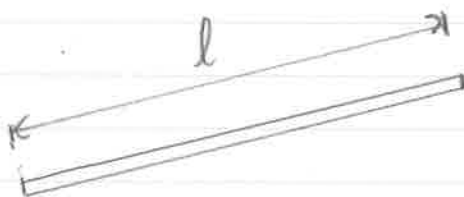
$$\sum_{i=1}^N f(x_i) \Delta x_i$$

As $\Delta x_i \rightarrow 0$,

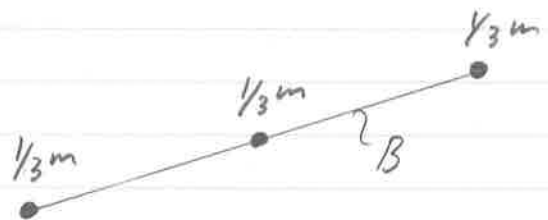
$$\sum_{i=1}^N f(x_i) \Delta x_i \rightarrow \int f(x) dx$$

The same concept applies to DRBs / CRBs.

Consider a slender bar. In reality the bar is a continuous rigid body, but we can approximate it as a DRB.



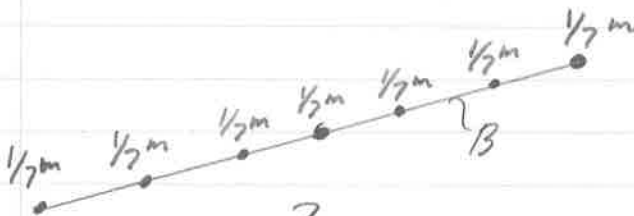
Slender bar
mass m



3 particle DRB model of
slender bar

$$m_B = \sum_{i=1}^3 \frac{1}{3}m = m$$

Let's improve our DRB by increasing number of particles

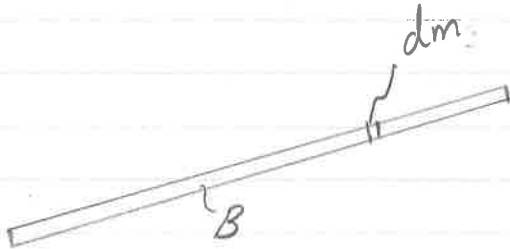


$$m_B = \sum_{i=1}^7 \frac{1}{7}m = m$$

Our DRB approximation of the bar improves as we increase the number of particles, and proportionally decrease the mass of each particle.

What if we let l , the number of particles, approach infinity?

If we still want $m_B = m$, then the mass of each particle must approach zero.

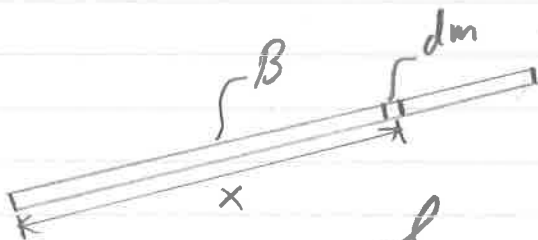


Def: Consider a CRB B . The zeroth moment of mass or mass of B is

$$m_B = \int_B dm = \int_B \sigma dV$$

where σ is the volumetric density of B , which in general does not need to be constant over the body.

For our slender bar, let ρ be the density per unit length (i.e., $m = \rho l$)



$$dm = \rho dx$$

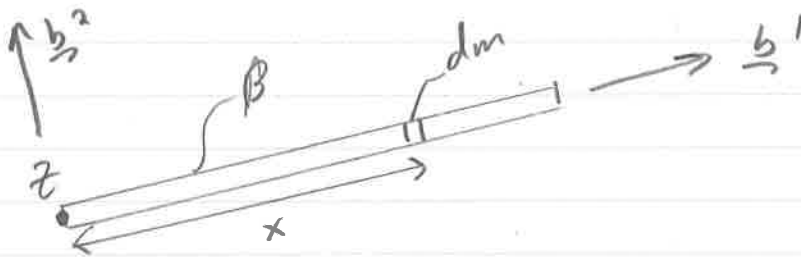
$$m_B = \int_0^l \rho dx = \rho x \Big|_0^l = \rho l = m$$

Def: Consider a CRB B and point z . The position of the center of mass of B relative to z is

$$\int_B^{\text{cm}} z = \frac{1}{m_B} \int_B \int_B^{\text{cm}} dm z = \frac{1}{m_B} \int_B \sigma \int_B^{\text{cm}} dV,$$

where σ is the volumetric density of B .

For slender bar



Again, $dm = \rho dx$, $m_B = m = \rho l$

$$\int_B^{\text{cm}} dm z = \int_B^{\text{cm}} \begin{bmatrix} x \\ 0 \\ 0 \end{bmatrix}$$

$$\int_B^{\text{cm}} z = \frac{1}{\rho l} \int_0^l \begin{bmatrix} x \\ 0 \\ 0 \end{bmatrix} dx$$

$$= \frac{1}{l} \begin{bmatrix} x \\ 0 \\ 0 \end{bmatrix} \Big|_0^l = \frac{1}{l} \begin{bmatrix} \frac{1}{2} l^2 \\ 0 \\ 0 \end{bmatrix}$$

$$= \frac{1}{l} \begin{bmatrix} \frac{1}{2} l^2 \\ 0 \\ 0 \end{bmatrix} = \begin{bmatrix} \frac{1}{2} l \\ 0 \\ 0 \end{bmatrix}$$

Def: Consider a CRB B and point z . The first moment of mass of B relative to z is

$$\underline{c}_{B,z} = \int_B \underline{r}^{dz} dm = \int_B \sigma \underline{r}^{dz} dV = m_B \underline{r}^{dz}$$

where σ is the volumetric density of B .

For slender bar

$$\underline{c}_{B,z} = \int_0^l \rho \underline{r}_b^T \begin{bmatrix} x \\ 0 \\ 0 \end{bmatrix} dx$$

$$= \rho \underline{r}_b^T \int_0^l \begin{bmatrix} x \\ 0 \\ 0 \end{bmatrix} dx = \rho \underline{r}_b^T \begin{bmatrix} \frac{1}{2} x^2 \\ 0 \\ 0 \end{bmatrix} \bigg|_0^l$$

$$= \underline{r}_b^T \begin{bmatrix} \frac{1}{2} \rho l^2 \\ 0 \\ 0 \end{bmatrix} \quad \swarrow \text{Since } m = \rho l$$

$$= \underline{r}_b^T \begin{bmatrix} \frac{1}{2} ml \\ 0 \\ 0 \end{bmatrix}$$

$$= m \underline{r}^{dz}$$

Def: Consider a CRB B and point z . The second moment of mass of B relative to z , resolved in \underline{F}_b is

$$\begin{aligned}\underline{J}_b^{Bz} &= - \int_B \underline{r}_b^{dmz^x} \underline{r}_b^{dmz^x} dm \\ &= - \int_B \sigma \underline{r}_b^{dmz^x} \underline{r}_b^{dmz^x} dV,\end{aligned}$$

where σ is the volumetric density of B .

For slender bar,

$$\begin{aligned}\underline{J}_b^{Bz} &= - \int_0^l \rho \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & -x \\ 0 & x & 0 \end{bmatrix} \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & -x \\ 0 & x & 0 \end{bmatrix} dx \\ &= -\rho \int_0^l \begin{bmatrix} 0 & 0 & 0 \\ 0 & -x^2 & 0 \\ 0 & 0 & -x^2 \end{bmatrix} dx = -\rho \begin{bmatrix} 0 & 0 & 0 \\ 0 & -\frac{x^3}{3} & 0 \\ 0 & 0 & -\frac{x^3}{3} \end{bmatrix} \bigg|_0^l \\ &= \begin{bmatrix} 0 & 0 & 0 \\ 0 & \frac{1}{3}\rho l^3 & 0 \\ 0 & 0 & \frac{1}{3}\rho l^3 \end{bmatrix} \stackrel{m=\rho l}{=} \begin{bmatrix} 0 & 0 & 0 \\ 0 & \frac{1}{3}ml^2 & 0 \\ 0 & 0 & \frac{1}{3}ml^2 \end{bmatrix}\end{aligned}$$

If we want \underline{I}_b^{BC} , we need \underline{r}_b^{dmc}

$$\underline{r}_b^{dmc} = \underline{r}^{dmc} - \underline{r}^{c2}$$

$$= \underline{F}_b^T \left(\begin{bmatrix} x \\ 0 \\ 0 \end{bmatrix} - \begin{bmatrix} \frac{1}{2}l \\ 0 \\ 0 \end{bmatrix} \right) = \underline{F}_b^T \begin{bmatrix} x - \frac{1}{2}l \\ 0 \\ 0 \end{bmatrix}$$

$$\underline{I}_b^{BC} = - \int_0^l \rho \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & -x + \frac{l}{2} \\ 0 & x - \frac{l}{2} & 0 \end{bmatrix} \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & -x + \frac{l}{2} \\ 0 & x - \frac{l}{2} & 0 \end{bmatrix} dx$$

$$= -\rho \int_0^l \begin{bmatrix} 0 & 0 & 0 \\ 0 & -x^2 + xl - \frac{l^2}{4} & 0 \\ 0 & 0 & -x^2 + xl - \frac{l^2}{4} \end{bmatrix} dx$$

$$= -\rho \left[\begin{array}{ccc} 0 & 0 & 0 \\ 0 & -\frac{x^3}{3} + \frac{1}{2}xl^2 - x\frac{l^2}{4} & 0 \\ 0 & 0 & -\frac{x^3}{3} + \frac{1}{2}xl^2 - x\frac{l^2}{4} \end{array} \right] \bigg|_0^l$$

$$= -\rho \left[\begin{array}{ccc} 0 & 0 & 0 \\ 0 & -\frac{l^3}{3} + \frac{1}{2}l^3 - \frac{l^3}{4} & 0 \\ 0 & 0 & -\frac{l^3}{3} + \frac{1}{2}l^3 - \frac{l^3}{4} \end{array} \right]$$

$$= \begin{bmatrix} 0 & 0 & 0 \\ 0 & \frac{1}{12}\rho l^3 & 0 \\ 0 & 0 & \frac{1}{12}\rho l^3 \end{bmatrix} \stackrel{m=\rho l}{=} \begin{bmatrix} 0 & 0 & 0 \\ 0 & \frac{1}{12}ml^2 & 0 \\ 0 & 0 & \frac{1}{12}ml^2 \end{bmatrix}$$

Def: Consider frame \mathcal{F}_a , point w , and CRB B .
 The translational momentum of B relative to w wrt \mathcal{F}_a is

$$\mathcal{P}_B^{Bw/a} = \int_B d\mathcal{P}_B^{Bw/a} = \int_B \underline{v}^{Bw/a} dm$$

Since $\underline{v}^{Bw/a} = \underline{r}^{Bw/a} \cdot \underline{\omega}^a$

$$\begin{aligned} \mathcal{P}_B^{Bw/a} &= \int_B \underline{r}^{Bw/a} dm \\ &= \left(\int_B \underline{r}^{Bw/a} dm \right) \cdot \underline{\omega}^a \\ &= m_B \underline{r}^{Bw/a} = m_B \underline{v}^{Bw/a} \end{aligned}$$

Therefore,

$$\boxed{\mathcal{P}_B^{Bw/a} = m_B \underline{v}^{Bw/a}}$$

Euler's First Law (E1L)

Consider inertial frame \mathcal{I}_a , unforced particle w , CRB B , and an external force per unit volume applied to mass element dm of \underline{df}^{dm} . Then,

$$\underline{f}^B = \rho^{Bw/a} \cdot \underline{a},$$

where

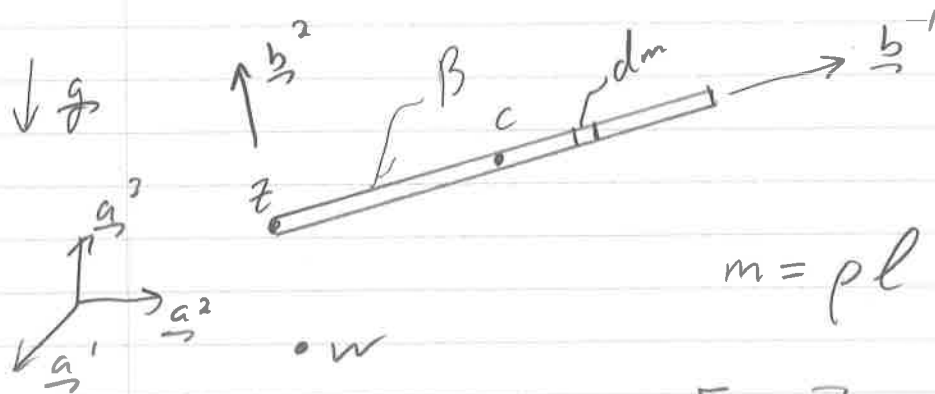
$$\underline{f}^B = \int_B d\underline{f}^{dm}.$$

If m_B is constant, then

$$\underline{f}^B = (m_B \underline{v}^{cw/a}) \cdot \underline{a} = m_B \underline{v}^{cw/a} \cdot \underline{a}$$

$$\underline{f}^B = m_B \underline{a}^{cw/a}$$

Return to slender bar



$$\text{Let } \underline{r}^{cw} = \underline{F}_a^T \begin{bmatrix} x_a \\ y_a \\ z_a \end{bmatrix}$$

$$\underline{v}^{cw/a} = \underline{F}_a^T \begin{bmatrix} \dot{x}_a \\ \dot{y}_a \\ \dot{z}_a \end{bmatrix}$$

$$\underline{a}^{cw/a} = \underline{F}_a^T \begin{bmatrix} \ddot{x}_a \\ \ddot{y}_a \\ \ddot{z}_a \end{bmatrix}$$

$$\underline{p}^{Bw/a} = m_B \underline{v}^{cw/a} = \underline{F}_a^T \begin{bmatrix} m \dot{x}_a \\ m \dot{y}_a \\ m \dot{z}_a \end{bmatrix}$$

Gravitational force on dm :

$$d\underline{f}^{dm} = g dm = \underline{F}_a^T \begin{bmatrix} 0 \\ 0 \\ -g \end{bmatrix} dm$$

$$\underline{f}^B = \int_B d\underline{f}^{dm} = \int_B \underline{F}_a^T \begin{bmatrix} 0 \\ 0 \\ -g \end{bmatrix} dm = \underline{F}_a^T \begin{bmatrix} 0 \\ 0 \\ -g \end{bmatrix} \int_0^l \rho dx$$

$$= \underline{F}_a^T \begin{bmatrix} 0 \\ 0 \\ -g \end{bmatrix} \rho x \Big|_0^l = \underline{F}_a^T \begin{bmatrix} 0 \\ 0 \\ -\rho l g \end{bmatrix} = \underline{F}_a^T \begin{bmatrix} 0 \\ 0 \\ -mg \end{bmatrix}$$

Using EIL:

$$\underline{f}^B = m_B \underline{a} \quad \text{c.w.l.a.l.a}$$

$$\underline{J}_a^T \begin{bmatrix} 0 \\ 0 \\ -mg \end{bmatrix} = \underline{J}_a^T \begin{bmatrix} m \ddot{x}_a \\ m \ddot{y}_a \\ m \ddot{z}_a \end{bmatrix}$$

EoMs:

$$\begin{aligned} m \ddot{x}_a &= 0 \\ m \ddot{y}_a &= 0 \\ m(\ddot{z}_a + g) &= 0 \end{aligned}$$

Def! Consider frame \mathcal{F}_a , point z , and CRB B .
The angular momentum of B relative to z w.r.t \mathcal{F}_a is

$$\begin{aligned}\underline{h}^{Bz/a} &= \int_B d\underline{h}^{dmz/a} = \int_B \underline{r}^{dmz} \times d\underline{p}^{dmz/a} \\ &= \int_B \underline{r}^{dmz} \times \underline{v}^{dmz/a} dm\end{aligned}$$

Recall $\underline{v}^{dmz/a} = \underline{v}^{dmz^{*a}} = \underline{v}^{dmz^{*b}} + \underline{\omega}^{ba} \times \underline{r}^{dmz}$

If z is a point on the CRB and frame \mathcal{F}_b is fixed to the CRB, then $\underline{v}^{dmz^{*b}} = \underline{0}$. Therefore,

$$\underline{v}^{dmz/a} = \underline{\omega}^{ba} \times \underline{r}^{dmz} = - \underline{r}^{dmz} \times \underline{\omega}^{ba}$$

and

$$\begin{aligned}\underline{h}^{Bz/a} &= - \int_B \underline{r}^{dmz} \times (\underline{r}^{dmz} \times \underline{\omega}^{ba}) dm \\ &= \underline{I}_b^T \left(- \int_B \underline{r}_b^{dmz} \underline{r}_b^{dmz^T} dm \underline{\omega}_b^{ba} \right) \\ &\quad \underline{J}_b^{Bz}\end{aligned}$$

$$= \underline{I}_b^T \underline{J}_b^{Bz} \underline{\omega}_b^{ba} \quad \text{or} \quad \underline{h}_b^{Bz/a} = \underline{J}_b^{Bz} \underline{\omega}_b^{ba}$$

Euler's Second Law (E2L)

Consider inertial frame \mathcal{F}_a , unforced particle w , point z , CRB β , and an external force per unit volume applied to mass element dm of $d\mathbf{f}^{dm}$. Then,

$$\boxed{\underline{\mathbf{h}}^{Bz/a \cdot a} + \underline{\mathbf{c}}^{Bz} \times \underline{\mathbf{v}}^{zw/a \cdot a} = \underline{\mathbf{m}}^{Bz}}$$

where

$$\underline{\mathbf{m}}^{Bz} = \int_B \underline{\mathbf{r}}^{dmz} \times d\mathbf{f}^{dm}$$

If $z=c$, then $\underline{\mathbf{c}}^{Bc} = \underline{\mathbf{0}}$ and

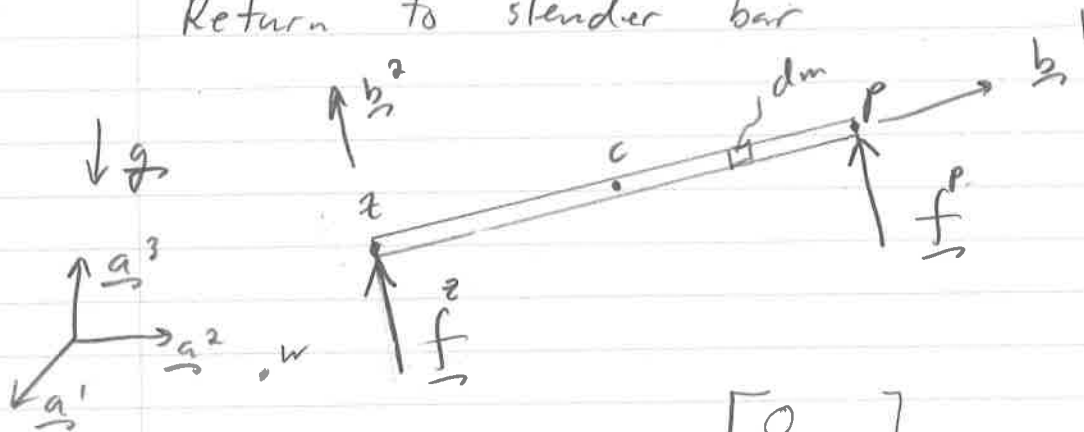
$$\boxed{\underline{\mathbf{h}}^{Bc/a \cdot a} = \underline{\mathbf{m}}^{Bc}}$$

Since $\underline{\mathbf{h}}^{Bc/a} = \underline{\mathbf{J}}_b^T \underline{\mathbf{J}}_b^{Bc} \underline{\omega}_b^{b_c}$, E2L in \mathcal{F}_b is

Euler's Equation :

$$\boxed{\underline{\mathbf{J}}_b^{Bc, ba} \underline{\omega}_b^{ba} + \underline{\omega}_b^{ba} \times \underline{\mathbf{J}}_b^{Bc} \underline{\omega}_b^{ba} = \underline{\mathbf{m}}_b^{Bc}}$$

Return to slender bar



where $\underline{f}^z = \underline{J}_b^T \begin{bmatrix} 0 \\ +f_{b2}^z \\ 0 \end{bmatrix}$, $\underline{f}^P = \underline{J}_b^T \begin{bmatrix} 0 \\ f_{b2}^P \\ 0 \end{bmatrix}$

Recall $\underline{J}_b^{Bc} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & \frac{1}{12}ml^2 & 0 \\ 0 & 0 & \frac{1}{12}ml^2 \end{bmatrix}$

$$\underline{h}^{Bc/a} = \underline{J}_b^T \underline{J}_b^{Bc} \underline{\omega}_b^{ba} = \underline{J}_b^T \begin{bmatrix} 0 & 0 & 0 \\ 0 & \frac{1}{12}ml^2 & 0 \\ 0 & 0 & \frac{1}{12}ml^2 \end{bmatrix} \begin{bmatrix} \omega_{b1}^{ba} \\ \omega_{b2}^{ba} \\ \omega_{b3}^{ba} \end{bmatrix}$$

$$= \underline{J}_b^T \begin{bmatrix} 0 \\ \frac{1}{12}ml^2 \omega_{b2}^{ba} \\ \frac{1}{12}ml^2 \omega_{b3}^{ba} \end{bmatrix}$$

$$\underline{m}^{Bc} = \int_B \underline{r}^{dmc} \times d\underline{f}^{dm}$$

$$= \int_B \underline{r}^{dmc} \times d\underline{f}^{dmg} + \underline{r}^{pc} \times \underline{f}^P + \underline{r}^{zc} \times \underline{f}^z$$

Notice point forces appear without integral.

Recall $\int_{\gamma} f^{dm_{\gamma}} = \int_{\gamma} g \, dm = \int_{ba} g_a \, dm$

$$\int_B \int_{\gamma}^{dm_c} x d\int_{\gamma}^{dm_{\gamma}} = \int_0^l p \, \vec{F}_b^T \begin{bmatrix} x - \frac{l}{2} \\ 0 \\ 0 \end{bmatrix} dx \, \int_{ba} g_a$$

$$= p \, \vec{F}_b^T \int_0^l \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & -x + \frac{l}{2} \\ 0 & x - \frac{l}{2} & 0 \end{bmatrix} dx \, \int_{ba} g_a$$

$$= p \, \vec{F}_b^T \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & -\frac{1}{2}x^2 + \frac{x}{2}l \\ 0 & \frac{1}{2}x^2 - \frac{x}{2}l & 0 \end{bmatrix} \Big|_0^l \int_{ba} g_a$$

$$= p \, \vec{F}_b^T \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & -\frac{1}{2}l^2 + \frac{1}{2}l^2 \\ 0 & \frac{1}{2}l^2 - \frac{1}{2}l^2 & 0 \end{bmatrix} \int_{ba} g_a$$

$$= \underline{0}$$

$$\underline{m}^{Bc} = \underline{0} + \vec{F}_b^T (\int_b^{Pc} \underline{f}_b^P + \int_b^{Zc} \underline{f}_b^Z)$$

$$= \vec{F}_b^T \left(\begin{bmatrix} l/2 \\ 0 \\ 0 \end{bmatrix} \begin{bmatrix} 0 \\ f_{b2}^P \\ 0 \end{bmatrix} + \begin{bmatrix} -l/2 \\ 0 \\ 0 \end{bmatrix} \begin{bmatrix} 0 \\ +f_{b2}^Z \\ 0 \end{bmatrix} \right)$$

$$= \vec{F}_b^T \left(\begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & -l/2 \\ 0 & l/2 & 0 \end{bmatrix} \begin{bmatrix} 0 \\ f_{b2}^P \\ 0 \end{bmatrix} + \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & l/2 \\ 0 & -l/2 & 0 \end{bmatrix} \begin{bmatrix} 0 \\ f_{b2}^Z \\ 0 \end{bmatrix} \right)$$

$$\underline{m}^{Bc} = \underline{f}_b^T \begin{bmatrix} 0 \\ 0 \\ \frac{l}{2}(f_{b2}^P - f_{b2}^Z) \end{bmatrix}$$

$$E2L: \quad \underline{h}^{Bc/a'a} = \underline{m}^{Bc}$$

$$\text{or} \quad \underline{J}_b^{Bc} \underline{\dot{w}}_b^{ba} + \underline{\dot{w}}_b^{baT} \underline{J}_b^{Bc} \underline{w}_b^{ba} = \underline{m}_b^{Bc}$$

$$\begin{bmatrix} 0 & 0 & 0 \\ 0 & \frac{1}{12}ml^2 & 0 \\ 0 & 0 & \frac{1}{12}ml^2 \end{bmatrix} \begin{bmatrix} \dot{w}_{b1}^{ba} \\ \dot{w}_{b2}^{ba} \\ \dot{w}_{b3}^{ba} \end{bmatrix}$$

$$+ \begin{bmatrix} 0 & -w_{b3}^{ba} & w_{b2}^{ba} \\ w_{b3}^{ba} & 0 & -w_{b1}^{ba} \\ -w_{b2}^{ba} & w_{b1}^{ba} & 0 \end{bmatrix} \begin{bmatrix} 0 \\ \frac{1}{12}ml^2 w_{b2}^{ba} \\ \frac{1}{12}ml^2 w_{b3}^{ba} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ \frac{l}{2}(f_{b2}^P - f_{b2}^Z) \end{bmatrix}$$

$$\begin{bmatrix} 0 \\ \frac{1}{12}ml^2 \dot{w}_{b2}^{ba} \\ \frac{1}{12}ml^2 \dot{w}_{b3}^{ba} \end{bmatrix} + \begin{bmatrix} \frac{1}{12}ml^2 (-w_{b3}^{ba} w_{b2}^{ba} + w_{b2}^{ba} w_{b3}^{ba}) \\ -\frac{1}{12}ml^2 w_{b1}^{ba} w_{b3}^{ba} \\ \frac{1}{12}ml^2 w_{b1}^{ba} w_{b2}^{ba} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ \frac{l}{2}(f_{b2}^P - f_{b2}^Z) \end{bmatrix}$$

$$\frac{1}{12}ml^2 (\dot{w}_{b2}^{ba} - w_{b1}^{ba} w_{b3}^{ba}) = 0$$

$$\frac{1}{12}ml^2 (\dot{w}_{b3}^{ba} + w_{b1}^{ba} w_{b2}^{ba}) = \frac{l}{2} (f_{b2}^P - f_{b2}^Z)$$

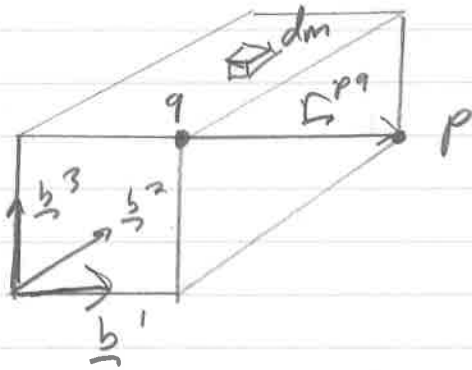
$$\dot{w}_{b2}^{ba} - w_{b1}^{ba} w_{b3}^{ba} = 0$$

$$\frac{1}{6}ml (\dot{w}_{b3}^{ba} + w_{b1}^{ba} w_{b2}^{ba}) = f_{b2}^P - f_{b2}^Z$$

Parallel Axis Theorem

Given $\underline{J}_b^{Bp} = - \int_B \underline{r}_b^{dmp^x} \underline{r}_b^{dmp^x} dm$,

can we find $\underline{J}_b^{Bq} = - \int_B \underline{r}_b^{dmq^x} \underline{r}_b^{dmq^x} dm$?



$$\underline{J}_b^{Bq} = - \int_B \underline{r}_b^{dmq^x} \underline{r}_b^{dmq^x} dm$$

$$= - \int_B (\underline{r}_b^{dmp} + \underline{r}_b^{pq})^x (\underline{r}_b^{dmp} + \underline{r}_b^{pq})^x dm$$

$$= - \int_B (\underline{r}_b^{dmp^x} \underline{r}_b^{dmp^x} + \underline{r}_b^{dmp^x} \underline{r}_b^{pq^x} + \underline{r}_b^{pq^x} \underline{r}_b^{dmp^x} + \underline{r}_b^{pq^x} \underline{r}_b^{pq^x}) dm$$

$$= - \int_B \underline{r}_b^{dmp^x} \underline{r}_b^{dmp^x} dm - \int_B \underline{r}_b^{dmp^x} dm \underline{r}_b^{pq^x}$$

$$- \underline{r}_b^{pq^x} \int_B \underline{r}_b^{dmp^x} dm - \int_B dm \underline{r}_b^{pq^x} \underline{r}_b^{pq^x}$$

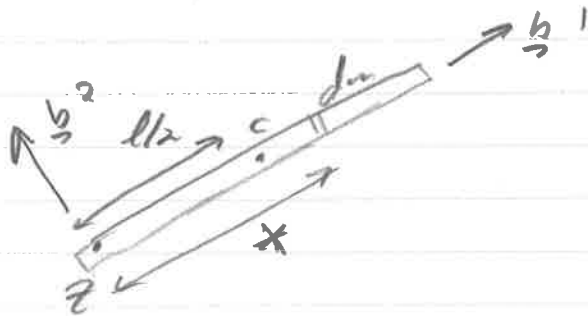
$$\underline{J}_b^{Bq} = \underline{J}_b^{Bp} - \underline{C}_b^{Bp^x} \underline{r}_b^{pq^x} - \underline{r}_b^{pq^x} \underline{C}_b^{Bp^x} - m_B \underline{r}_b^{pq^x} \underline{r}_b^{pq^x}$$

↳ Parallel Axis Theorem

If $p=c$, then $\underline{C}_b^{Bc} = \underline{0}$ and

$$\underline{J}_b^{Bq} = \underline{J}_b^{Bc} - m_B \underline{r}_b^{cq^x} \underline{r}_b^{cq^x}$$

Return to our slender bar example:



We previously calculated

$$\underline{J}_b^{Bz} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & \frac{1}{3}ml^2 & 0 \\ 0 & 0 & \frac{1}{3}ml^2 \end{bmatrix}$$

$$\underline{J}_b^{Bc} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & \frac{1}{12}ml^2 & 0 \\ 0 & 0 & \frac{1}{12}ml^2 \end{bmatrix}$$

Notice that $\underline{J}_b^{Bc} = \underline{0}$ and $\underline{r}^{cz} = \underline{J}_b^T \begin{bmatrix} l/2 \\ 0 \\ 0 \end{bmatrix}$

$$m_B \underline{r}_b^{czx} \underline{r}_b^{czx} = m \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & -l/2 \\ 0 & l/2 & 0 \end{bmatrix} \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & -l/2 \\ 0 & l/2 & 0 \end{bmatrix}$$

$$= m \begin{bmatrix} 0 & 0 & 0 \\ 0 & -l/4 & 0 \\ 0 & 0 & -l/4 \end{bmatrix}$$

$$\underline{J}_b^{Bc} - m_B \underline{r}_b^{czx} \underline{r}_b^{czx} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & \frac{1}{2}ml^2 & 0 \\ 0 & 0 & \frac{1}{2}ml^2 \end{bmatrix} + \begin{bmatrix} 0 & 0 & 0 \\ 0 & \frac{1}{4}ml^2 & 0 \\ 0 & 0 & \frac{1}{4}ml^2 \end{bmatrix}$$

$$= \begin{bmatrix} 0 & 0 & 0 \\ 0 & \frac{4}{12}ml^2 & 0 \\ 0 & 0 & \frac{4}{12}ml^2 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & \frac{1}{3}ml^2 & 0 \\ 0 & 0 & \frac{1}{3}ml^2 \end{bmatrix}$$

J_6^{Bz}

3 Steps to Success for a CRB (Newton-Euler Approach to Dynamics)

1) Kinematics

- i) Frames and DCMs
- ii) Angular Velocity
- iii) Positions
 - position of c (and z) (\underline{r}^{cw}) (\underline{r}^{zw})
 - position of dm relative to c or z (\underline{r}^{dmc} or \underline{r}^{dmz})
- iv) Velocity
 - velocity of c (and z) ($\underline{v}^{cw/a}$) ($\underline{v}^{zw/a}$)
- v) Acceleration
 - acceleration of c ($\underline{a}^{cw/a}$)

2) FBD of CRB

- forces
- moments about c or z (\underline{m}^{Bc} or \underline{m}^{Bz})

3) Newton's / Euler's Laws

- i) Angular momentum about c or z

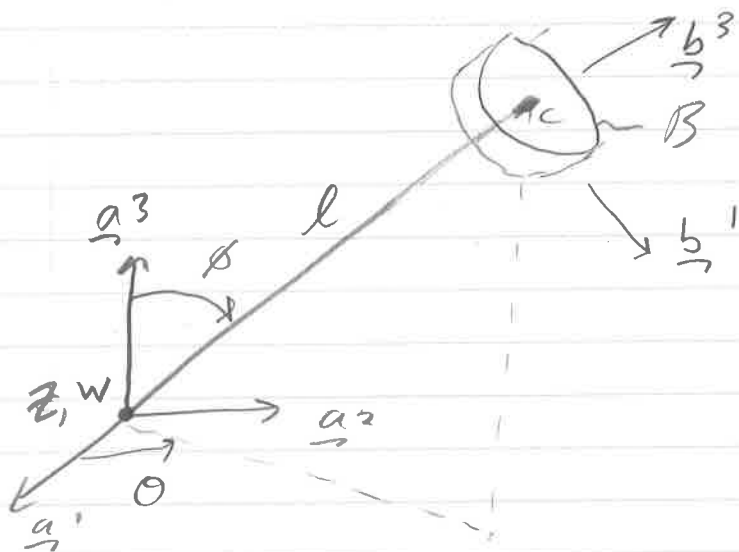
$$(\underline{h}^{Bc/a} = \underline{J}_b^T \underline{J}_b^{Bc} \underline{\omega}_b^{ba} \text{ or } \underline{h}^{Bz/a} = \underline{J}_b^T \underline{J}_b^{Bz} \underline{\omega}_b^{ba})$$
- ii) N2L/E1L ($\underline{f}^B = m_B \underline{a}^{cw/a}$)
- iii) N2LR/E2L ($\underline{h}^{Bc/a} = \underline{m}^{Bc}$

$$\text{or } \underline{h}^{Bz/a} + \underline{c}^{Bz} \times \underline{v}^{zw/a} = \underline{m}^{Bz})$$

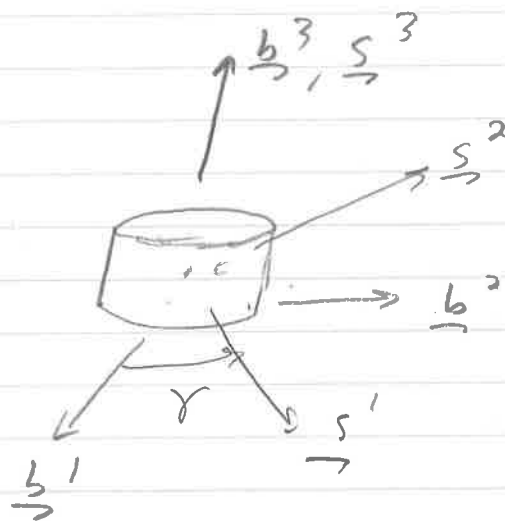
Euler's Equations ($\underline{J}_b^{Bc} \cdot \underline{\omega}_b^{ba} + \underline{\omega}_b^{ba} \times \underline{J}_b^{Bc} \underline{\omega}_b^{ba} = \underline{m}_b^{Bc}$)

$$(\underline{J}_b^{Bz} \cdot \underline{\omega}_b^{ba} + \underline{\omega}_b^{ba} \times \underline{J}_b^{Bz} \underline{\omega}_b^{ba} + \underline{c}_b^{Bz} \times \underline{v}_b^{zw/a} + \underline{c}_b^{Bz} \times \underline{\omega}_b^{ba} \times \underline{v}_b^{zw/a} = \underline{m}_b^{Bz})$$

Example: Consider a gyro pendulum.



Close up of gyro:



$$\mathcal{F}_a \xrightarrow{\mathcal{L}_2(\theta)} \mathcal{F}_q \xrightarrow{\mathcal{L}_2(\theta)} \mathcal{F}_b \xrightarrow{\mathcal{L}_3(\gamma)} \mathcal{F}_s$$

$\dot{\gamma}$ is constant spin rate of gyro.

Find EoMs of gyro pendulum.

1) Kinematics

$$i) \underline{C}_{sa} = \underline{C}_3(r) \underline{C}_2(\theta) \underline{C}_3(\phi)$$

$$ii) \underline{\omega}^{sa} = \underline{\omega}^{sb} + \underline{\omega}^{bq} + \underline{\omega}^{qa}$$

$$= \underline{F}_s^T \begin{bmatrix} 0 \\ 0 \\ \dot{r} \end{bmatrix} + \underline{F}_b^T \begin{bmatrix} 0 \\ \dot{\theta} \\ 0 \end{bmatrix} + \underline{F}_q^T \begin{bmatrix} 0 \\ 0 \\ \dot{\phi} \end{bmatrix}$$

$$= \underline{F}_s^T \left(\begin{bmatrix} 0 \\ 0 \\ \dot{r} \end{bmatrix} + \underline{C}_{sb} \begin{bmatrix} 0 \\ \dot{\theta} \\ 0 \end{bmatrix} + \underline{C}_{bq} \begin{bmatrix} 0 \\ 0 \\ \dot{\phi} \end{bmatrix} \right)$$

$\underline{C}_3(r) \underline{C}_2(\theta)$

$$= \underline{F}_s^T \left(\begin{bmatrix} 0 \\ 0 \\ \dot{r} \end{bmatrix} + \begin{bmatrix} c_r & s_r & 0 \\ -s_r & c_r & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} 0 \\ \dot{\theta} \\ 0 \end{bmatrix} \right.$$

$$\left. + \begin{bmatrix} c_r & s_r & 0 \\ -s_r & c_r & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} c_\theta & 0 & -s_\theta \\ 0 & 1 & 0 \\ s_\theta & 0 & c_\theta \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ \dot{\phi} \end{bmatrix} \right)$$

$$= \underline{F}_s^T \left(\begin{bmatrix} 0 \\ 0 \\ \dot{r} \end{bmatrix} + \begin{bmatrix} s_r \dot{\theta} \\ c_r \dot{\theta} \\ 0 \end{bmatrix} + \begin{bmatrix} c_r & s_r & 0 \\ -s_r & c_r & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} -s_\theta \dot{\phi} \\ 0 \\ c_\theta \dot{\phi} \end{bmatrix} \right)$$

$$= \underline{F}_s^T \begin{bmatrix} s_r \dot{\theta} - c_r s_\theta \dot{\phi} \\ c_r \dot{\theta} + s_r s_\theta \dot{\phi} \\ \dot{r} + c_\theta \dot{\phi} \end{bmatrix} \quad \text{or} \quad \underline{\omega}^{sa} = \underline{F}_b^T \begin{bmatrix} -s_\theta \dot{\phi} \\ \dot{\theta} \\ c_\theta \dot{\phi} + \dot{r} \end{bmatrix}$$

iii) N/A

$$\int dm \vec{z} = \int dm \vec{c} + \int l \vec{e}$$

$$= \vec{F}_s^T \begin{bmatrix} p_{s1} \\ p_{s2} \\ p_{s3} \end{bmatrix} + \vec{F}_b^T \begin{bmatrix} 0 \\ 0 \\ l \end{bmatrix}$$

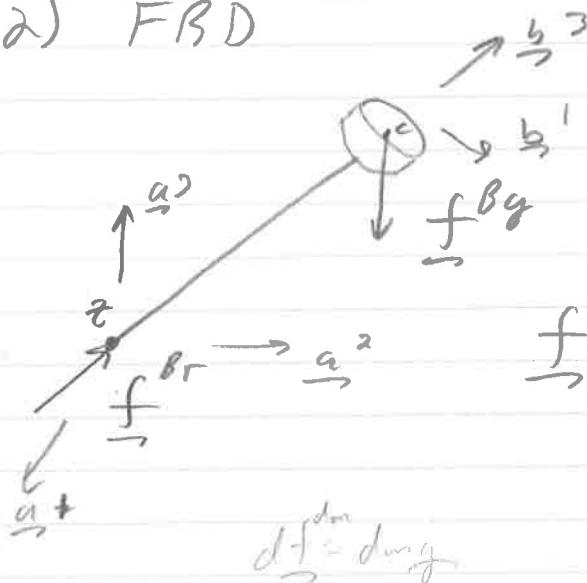
$$= \vec{F}_s^T \left(\begin{bmatrix} p_{s1} \\ p_{s2} \\ p_{s3} \end{bmatrix} + \underline{C}_3(\gamma) \begin{bmatrix} 0 \\ 0 \\ l \end{bmatrix} \right)$$

$$= \vec{F}_s^T \begin{bmatrix} p_{s1} \\ p_{s2} \\ p_{s3} + l \end{bmatrix}$$

$$\int \vec{z}^w = \underline{0}$$

i), v) N/A

2) FRD



$$\vec{f}^{Br} = \vec{F}_b^T \begin{bmatrix} f_{b1}^{Br} \\ f_{b2}^{Br} \\ f_{b3}^{Br} \end{bmatrix}$$

$$\begin{aligned} \vec{f}^{Bg} &= \int_B d\vec{f}^{dmg} = \vec{F}_c^T \begin{bmatrix} 0 \\ 0 \\ -mg \end{bmatrix} \\ &= \vec{F}_b^T \begin{bmatrix} \sin \alpha mg \\ 0 \\ -\cos \alpha mg \end{bmatrix} \end{aligned}$$

$$\underline{f}^B = \underline{f}^{Br} + \underline{f}^{Bs} = \underline{F}_b^T \begin{bmatrix} f_{b1}^{Br} + s_{\text{spring}} \\ f_{b2}^{Br} \\ f_{b3}^{Br} - c_{\text{spring}} \end{bmatrix}$$

$$\begin{aligned} \underline{m}^{Bz} &= \underline{r}^{cz} \times \underline{f}^{Bs} \\ &= \underline{F}_b^T \underline{r}_b^{cz} \times \underline{f}_b^{Bs} = \underline{F}_b^T \begin{bmatrix} 0 & -l & 0 \\ l & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} s_{\text{spring}} \\ 0 \\ -c_{\text{spring}} \end{bmatrix} \\ &= \underline{F}_b^T \begin{bmatrix} 0 \\ l m g s_{\text{spring}} \\ 0 \end{bmatrix} \end{aligned}$$

$$3) i) \underline{h}^{Bz/a} = \underline{F}_s^T \underline{J}_s^{Bz} \underline{\omega}_s^{sa}$$

Use Parallel Axis Theorem

$$\underline{J}_s^{Bz} = \underline{J}_s^{Bc} - m_B \underline{r}_s^{cz} \times \underline{r}_s^{cz}$$

We are given

$$\underline{J}_s^{Bc} = \begin{bmatrix} \frac{1}{12} m (3r^2 + h^2) & 0 & 0 \\ 0 & \frac{1}{12} m (3r^2 + h^2) & 0 \\ 0 & 0 & \frac{1}{2} m r^2 \end{bmatrix}$$

$$\text{For brevity let } J_{s11} = \frac{1}{12} (3r^2 + h^2)$$

$$J_{s22} = \frac{1}{12} (3r^2 + h^2)$$

$$J_{s33} = \frac{1}{2} m r^2$$

$$\begin{aligned}
 \underline{J}_S^{Bc} &= \begin{bmatrix} J_{S11} & 0 & 0 \\ 0 & J_{S22} & 0 \\ 0 & 0 & J_{S33} \end{bmatrix} - m \begin{bmatrix} 0 & -l & 0 \\ l & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} 0 & -l & 0 \\ l & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} \\
 &= \begin{bmatrix} J_{S11} & 0 & 0 \\ 0 & J_{S22} & 0 \\ 0 & 0 & J_{S33} \end{bmatrix} - m \begin{bmatrix} -l^2 & 0 & 0 \\ 0 & -l^2 & 0 \\ 0 & 0 & 0 \end{bmatrix} \\
 &= \begin{bmatrix} J_{S11} + ml^2 & 0 & 0 \\ 0 & J_{S22} + ml^2 & 0 \\ 0 & 0 & J_{S33} \end{bmatrix}
 \end{aligned}$$

Therefore,

$$\begin{aligned}
 \underline{h}^{Bz/a} &= \underline{F}_S^T \begin{bmatrix} J_{S11} + ml^2 & 0 & 0 \\ 0 & J_{S22} + ml^2 & 0 \\ 0 & 0 & J_{S33} \end{bmatrix} \begin{bmatrix} s\ddot{\phi} - c\dot{\phi}\dot{\phi} \\ c\ddot{\phi} + s\dot{\phi}\dot{\phi} \\ \ddot{\gamma} + c\dot{\phi}\dot{\phi} \end{bmatrix} \\
 &= \underline{F}_S^T \begin{bmatrix} (J_{S11} + ml^2)(s\ddot{\phi} - c\dot{\phi}\dot{\phi}) \\ (J_{S22} + ml^2)(c\ddot{\phi} + s\dot{\phi}\dot{\phi}) \\ J_{S33}(\ddot{\gamma} + c\dot{\phi}\dot{\phi}) \end{bmatrix}
 \end{aligned}$$

ii) E1L

Not translating, so not interested

iii) E2L

$$\begin{aligned}
 \underline{h}^{Bz/a} + \underline{c}^{Bz} \times \underline{v}^{Z/a} &= \underline{m}^{Bz} \\
 \underline{h}^{Bz/a} + \underline{\omega}^{Sc} \times \underline{h}^{Bz/a} &= \underline{m}^{Bz}
 \end{aligned}$$

$$\underline{F}_s^T (\underline{J}_s^{Bz} \cdot s_a + \underline{\omega}_s^{sa} \times \underline{J}_s^{Bz} \underline{\omega}_s^{sa}) = \underline{F}_s^T \underline{m}_s^{Bz}$$

$$\underline{J}_s^{Bz} \cdot s_a + \underline{\omega}_s^{sa} \times \underline{J}_s^{Bz} \underline{\omega}_s^{sa} = \underline{C}_{sb} \underline{m}_b^{Bz}$$

In first-order state-space form

$$\dot{\underline{\omega}}_s^{sa} = \underline{J}_s^{Bz^{-1}} \left(\underline{C}_{sb} \underline{m}_b^{Bz} - \underline{\omega}_s^{sa} \times \underline{J}_s^{Bz} \underline{\omega}_s^{sa} \right)$$

$$\underline{f}(\underline{\omega}_s^{sa})$$

To make our derivation slightly easier, we will make some approximations. We could derive the equations of motion without the approximations, but the equations would be more complicated.

Assumption 1: $\dot{\gamma}$ is much larger than $\dot{\phi}$ and $\dot{\theta}$

Therefore $J_{s33}(\dot{\gamma} + c_{\phi}\dot{\theta}) \approx J_{s33}\dot{\gamma}$

Also,
$$\underbrace{J_{s11}(s_{\gamma}\dot{\phi} - c_{\gamma}s_{\phi}\dot{\theta})}_{\text{Assume } \approx 0} \ll J_{s33}(\dot{\gamma} + c_{\phi}\dot{\theta})$$

$$\underbrace{J_{s22}(c_{\gamma}\dot{\phi} + s_{\gamma}s_{\phi}\dot{\theta})}_{\text{Assume } \approx 0} \ll J_{s33}(\dot{\gamma} + c_{\phi}\dot{\theta})$$

Therefore

$$\underline{h}^{Bela} \approx \underline{F}_s^T \begin{bmatrix} ml^2(s_{\gamma}\dot{\phi} - c_{\gamma}s_{\phi}\dot{\theta}) \\ ml^2(c_{\gamma}\dot{\phi} + s_{\gamma}s_{\phi}\dot{\theta}) \\ J_{s33}\dot{\gamma} \end{bmatrix}$$

$$= \underline{F}_b^T \underline{C}_{sb}^T [\dots]$$

$$= \underline{F}_b^T \begin{bmatrix} -ml^2 s_{\phi}\dot{\theta} \\ ml^2 \dot{\phi} \\ J_{s33}\dot{\gamma} \end{bmatrix}$$

Return to E2L

$$\underline{h}^{Bz/a'a} = \underline{m}^{Bz}$$

$$\underline{h}^{Bz/a'b} + \underline{\omega}^{ba} \times \underline{h}^{Bz/a'a} = \underline{m}^{Bz}$$

$$\underline{F}_b^i (\underline{\dot{h}}_b^{Bz/a'a} + \underline{\omega}_b^{ba} \times \underline{h}_b^{Bz/a'a}) = \underline{F}_b^i \underline{m}_b^{Bz}$$

$$\begin{bmatrix} -ml^2(c_\phi \ddot{\phi} + s_\phi \ddot{\theta}) \\ ml^2 \ddot{\theta} \\ 0 \end{bmatrix} + \begin{bmatrix} 0 & -c_\phi \dot{\theta} & \dot{\phi} \\ c_\phi \dot{\theta} & 0 & s_\phi \dot{\theta} \\ -\dot{\phi} & -s_\phi \dot{\theta} & 0 \end{bmatrix} \begin{bmatrix} -ml^2 s_\phi \dot{\theta} \\ ml^2 \dot{\phi} \\ T_{S33} \dot{\theta} \end{bmatrix} = \begin{bmatrix} 0 \\ mgl s_\phi \\ 0 \end{bmatrix}$$

$$\begin{bmatrix} -ml^2(c_\phi \ddot{\phi} + s_\phi \ddot{\theta}) - ml^2 \dot{\phi} \dot{\theta} c_\phi + T_{S33} \dot{\phi} \dot{\theta} \\ ml^2 \ddot{\theta} - ml^2 s_\phi c_\phi \dot{\theta}^2 + T_{S33} \dot{\theta} s_\phi \dot{\theta} \\ ml^2 s_\phi \dot{\theta} \dot{\phi} - ml^2 s_\phi \dot{\phi} \dot{\theta} \end{bmatrix} = \begin{bmatrix} 0 \\ mgl s_\phi \\ 0 \end{bmatrix}$$

$$\text{EoMs: } ml^2 s_\phi \dot{\theta} + 2ml^2 c_\phi \dot{\phi} \dot{\theta} = \dot{\phi} T_{S33} \dot{\theta}$$

$$ml^2 \ddot{\theta} - ml^2 s_\phi c_\phi \dot{\theta}^2 = mgl s_\phi - s_\phi \dot{\theta} T_{S33} \dot{\theta}$$

When $\theta = 0$, $\phi = 90^\circ$, $\dot{\theta} = \dot{\phi} = 0$

$$ml^2 \ddot{\theta} = 0$$

$$ml^2 \ddot{\phi} = mgl$$

ϕ increases, pendulum swings down and

$$ml^2 \ddot{\theta} \approx \dot{\phi} J_{33} \dot{\phi}$$

$$ml^2 \ddot{\phi} \approx mgl$$

$\dot{\theta}$ will increase, pendulum will precess, and eventually

$$ml^2 s_\phi \ddot{\theta} + 2ml^2 c_\phi \dot{\phi} \dot{\theta} = \dot{\phi} J_{33} \dot{\phi}$$

$$ml^2 \ddot{\theta} - ml^2 s_\phi c_\phi \dot{\theta}^2 = s_\phi (mgl - \dot{\theta} J_{33} \dot{\phi})$$

Larger $\dot{\phi}$ leads to slower precession (smaller $\dot{\theta}$) and greater cancellation of the mgl term

Energy and Work of a Continuous Rigid Body

Just like for a DRB, the kinetic energy associated with a CRB is

$$T_{Bw/c} = \frac{1}{2} m_B \underline{v}_a^{c w/c} \underline{v}_a^{c w/c} + \frac{1}{2} \underline{\omega}_b^{ba} \underline{J}_b^{Bc} \underline{\omega}_b^{ba}$$

The gravitational potential associated with CRB B is

$$V_{Bw} = -m_B \underline{g} \cdot \underline{r}_a^{cw}$$

All of the energy and work methods we saw for particles also apply to CRBs

For example, we can derive a Work-Energy Theorem for rigid bodies:

$$\begin{aligned} \frac{d}{dt} (T_{Bw/c}) &= \frac{d}{dt} \left(\frac{1}{2} m_B \underline{v}_a^{c w/c} \underline{v}_a^{c w/c} + \frac{1}{2} \underline{\omega}_b^{ba} \underline{J}_b^{Bc} \underline{\omega}_b^{ba} \right) \\ &= \frac{1}{2} m_B \left(\dot{\underline{v}}_a^{c w/c} \underline{v}_a^{c w/c} + \underline{v}_a^{c w/c} \dot{\underline{v}}_a^{c w/c} \right) \\ &\quad + \frac{1}{2} \left(\dot{\underline{\omega}}_b^{ba} \underline{J}_b^{Bc} \underline{\omega}_b^{ba} + \underline{\omega}_b^{ba} \underline{J}_b^{Bc} \dot{\underline{\omega}}_b^{ba} \right) \\ &= m_B \dot{\underline{v}}_a^{c w/c} \underline{v}_a^{c w/c} + \underline{\omega}_b^{ba} \underline{J}_b^{Bc} \dot{\underline{\omega}}_b^{ba} \end{aligned}$$

Knowing that $\underline{f}_a^B = m_B \underline{\dot{v}}_a^{cw/a}$

$$\underline{J}_b^{Bc} \underline{\omega}_b^{ba} + \underline{\omega}_b^{ba \times} \underline{J}_b^{Bc} \underline{\omega}_b^{ba} = \underline{m}_b^{Bc}$$

gives

$$\begin{aligned} \frac{d}{dt} (T_{Bw/a}) &= \underline{f}_a^{BT} \underline{\dot{v}}_a^{cw/a} + \underline{\omega}_b^{baT} (\underline{m}_b^{Bc} - \underline{\omega}_b^{ba \times} \underline{J}_b^{Bc} \underline{\omega}_b^{ba}) \\ &= \underline{f}_a^B \cdot \underline{\dot{v}}_a^{cw/a} + \underline{\omega}_b^{ba} \cdot \underline{m}_b^{Bc} \quad (\text{recall } \underline{\omega}_b^{baT} \underline{\omega}_b^{ba \times} = 0) \end{aligned}$$

Therefore

$$\boxed{\frac{d}{dt} (T_{Bw/a}) = \underline{f}_a^B \cdot \underline{\dot{v}}_a^{cw/a} + \underline{m}_b^{Bc} \cdot \underline{\omega}_b^{ba}}$$

↳ Work-Energy Theorem for a rigid body (DRB or CRB)

Total energy of rigid body:

$$E_{Bw/a} = T_{Bw/a} + V_{Bw}$$

If all forces are conservative, then

$$E_{Bw/a}(t_1) = E_{Bw/a}(t_2) \quad \text{or} \quad \dot{E}_{Bw/a} = 0$$

↳ Conservation of Energy.

We can also describe impulses acting on a rigid body.

$$\underline{\hat{f}}^B = \int_{t_1}^{t_2} \underline{f}^B dt = \int_{t_1}^{t_2} \underline{p}^{Bw/a} dt$$

$$\boxed{\underline{\hat{f}}^B = \underline{p}^{Bw/a}(t_2) - \underline{p}^{Bw/a}(t_1)}$$

$$\underline{\hat{m}}^{Bz} = \int_{t_1}^{t_2} \underline{m}^{Bz} dt = \int_{t_1}^{t_2} \left(\underline{h}^{Bz/a} + \underline{c}^{Bz} \times \underline{v}^{zw/a} \right) dt$$

$$\boxed{\underline{\hat{m}}^{Bz} = \underline{h}^{Bz/a}(t_2) - \underline{h}^{Bz/a}(t_1) + \int_{t_1}^{t_2} \underline{c}^{Bz} \times \underline{v}^{zw/a} dt}$$

If $z = c$, then $\underline{c}^{Bz} = 0$

$$\boxed{\underline{\hat{m}}^{Bc} = \underline{h}^{Bc/a}(t_2) - \underline{h}^{Bc/a}(t_1)}$$

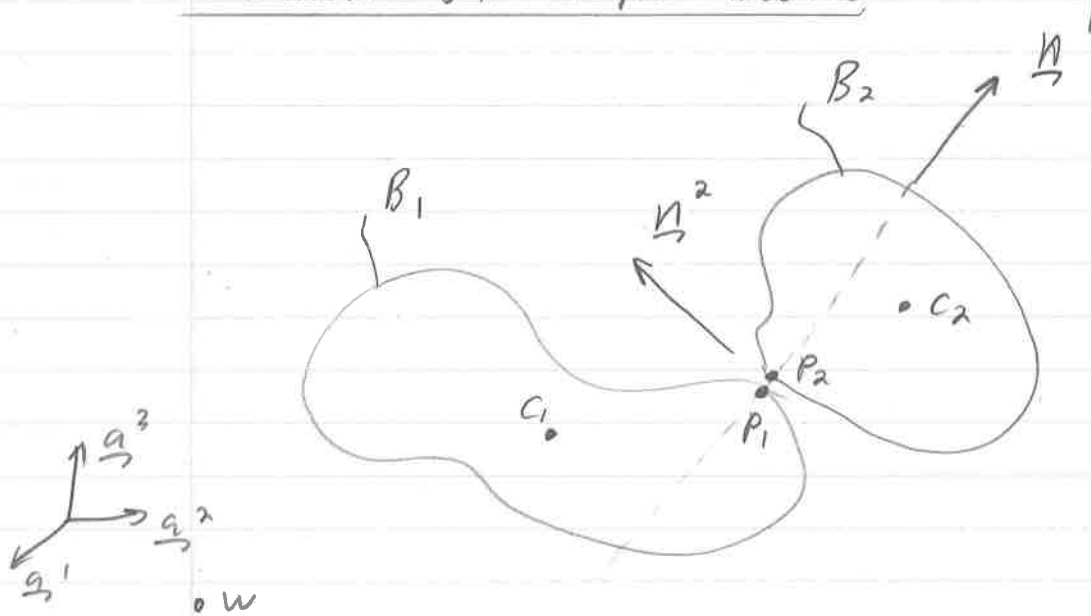
If $\underline{\hat{f}}^B = 0$, then $\underline{p}^{Bw/a}(t_1) = \underline{p}^{Bw/a}(t_2)$

↳ Conservation of translational momentum

If $\underline{\hat{m}}^{Bc} = 0$, then $\underline{h}^{Bc/a}(t_1) = \underline{h}^{Bc/a}(t_2)$

↳ conservation of angular momentum.

Collision of Rigid Bodies



$\underline{v}^{C_1 w/a}$: velocity of center of mass of B_1 rel. to w wrt J_a

$\underline{v}^{C_2 w/a}$: velocity of center of mass of B_2 rel. to w wrt J_a .

$\underline{v}^{P_1 w/a}$: velocity of impact point p_1 rel. to w wrt J_a

$\underline{v}^{P_2 w/a}$: velocity of impact point p_2 rel. to w wrt J_a .

$$\text{(OR!)} \quad e = \frac{(\underline{v}^{P_2 w/a}(t_2) - \underline{v}^{P_1 w/a}(t_2)) \cdot \underline{n}_1}{(\underline{v}^{P_1 w/a}(t_1) - \underline{v}^{P_2 w/a}(t_1)) \cdot \underline{n}_1}$$

Conservation of Translational Momentum of System

$$m_{B_1} \underline{v}^{C_1 W/a}(t_1) + m_{B_2} \underline{v}^{C_2 W/a}(t_1) \\ = m_{B_1} \underline{v}^{C_1 W/a}(t_2) + m_{B_2} \underline{v}^{C_2 W/a}(t_2)$$

Conservation of Angular Momentum of System

$$\underline{h}^{B_1 C_1/a}(t_1) + \underline{h}^{B_2 C_2/a}(t_1) = \underline{h}^{B_1 C_1/a}(t_2) + \underline{h}^{B_2 C_2/a}(t_2)$$