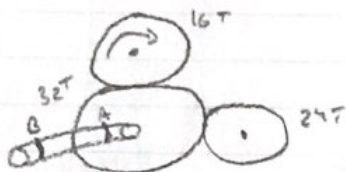


Machine elements

HW7

March 24, 2020

Problem 2:

Given: $P = 8$, $\phi = 20^\circ$, $E = 100\%$

$$\dot{W} = 1500 \text{ rpm} \cdot \frac{2\pi}{60} = 157.08 \text{ rad/s}$$

$$T = 90 \text{ lb}\cdot\text{in}$$

$$a. F_r = F_t \tan \phi$$

$$F_t = \frac{T}{r_p}$$

$$r_p = \frac{N_p}{2P} = \frac{16}{2(1)} = 1$$

$$F_t = \frac{90}{1} = 90 \text{ lbs.}$$

$$F_r = 90 \tan 20^\circ = 32.76 \text{ lbs.}$$

$$90 + 32.76 = 122.76 \text{ lbs.} \rightarrow \sum M_B = (N_{\text{net}})5 - R_A(4) = 0$$

 $\hookrightarrow N_{\text{net}}$

$$R_A = 217 \text{ lbs}$$

$$\sum F_y = N_{\text{net}} + R_B - R_A = 0$$

$$R_B = 43.4 \text{ lbs}$$

b. now spinning in opp. Direction so.

$$F_r = 90 - 32.76 = 57.24 \text{ lbs.}$$

$$N_{\text{net}} = \sqrt{2 \cdot 57.24^2} = 80.95$$

$$\sum M_B = (N_{\text{net}})5 - R_A(4) = 0$$

$$R_A = 101.2 \text{ lbs.}$$

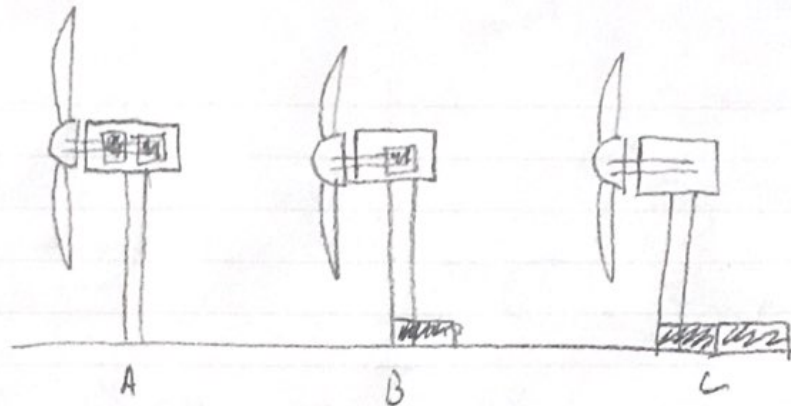
$$\sum F_y = N_{\text{net}} + R_B - R_A = 0$$

$$R_B = 20.23 \text{ lbs.}$$

c. They are diff. b/c of the way the gear is rotating. ~~the~~ When spinning counter clockwise the force goes against the ~~the~~ forces and visa-versa.

d. The 16 Tooth gear will fail first because it would have to go through more rotations than the other gears causing more wear on the parts.

Problem 3



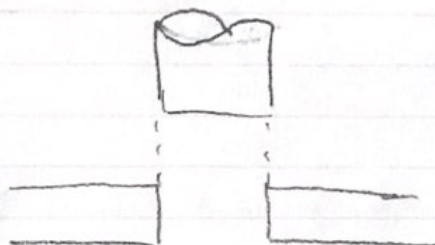
| Part 1: | A | B | C |
|---------------------------------|--|---|---|
| Stress on Tower | - Heavy on TOP and cause bending/ Stress in Tower | - Less heavy on TOP so Less Stress than A | - Least Stress of The 3 Towers |
| Convenience of Maintenance | NOT convenient X | Half convenient - | Most convenient ✓ |
| Performance | would be best at converting wind into electricity. | Okay at converting wind into energy | Worst at converting wind into energy |
| Bearing / shaft placement | placement is good for converting Energy. | placement would lose energy | placement would lose energy. |

part 2: The Load on The Vertical Shaft would cause Axial Loading and ~~be~~ bending. Also The rotation of The blades could cause Torsion.

part 3: A would be The best choice if looking at converting Wind Energy To Electricity.

Problem 4

Find hole / shaft limits for a Sliding fit using a basic hole size of 1 in.



Sliding fit \rightarrow H7/g6

Hole \rightarrow 1 in and Tolerance of 0.0008 from chart and H7.

$$\begin{aligned} \text{So } D_{\max} &= 1 + 0.0008 = 1.0008 \text{ in} \\ D_{\min} &= 1 - 0.0008 = 0.9992 \text{ in} \end{aligned}$$

Shaft \rightarrow 1 in and Tolerance of 0.0003 from g6.

$$\begin{aligned} D_{\max} &= 1 \text{ in} \\ D_{\min} &= 1 - 0.0003 \text{ in} = 0.9997 \text{ in} \end{aligned}$$

Problem 5

Flexible Shaft Couplings

Product: High-misalignment precision Flexible Shaft Coupling

Part #: 54125 K 88

Cost: \$ 94.68 each

Description: allows compensation in mis-aligned parts



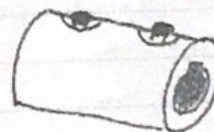
Rigid Shaft Couplings

Product: Set Screw Shaft Coupling

Part #: 6412 K 8

Cost: \$ 16.05 each

Description: hold couplings in place with screws.



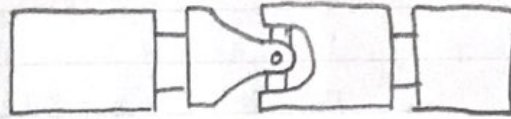
U-Joint

Product: Singl - U - Joint

Part #: 82841K72

Cost: \$151.15 each

Description: Needle bearings allow high shaft speeds



Flexible Shafts

Product: Heavy Duty Flexible Shaft w/ Female & Female Fittings.

Part #: 6426K101

Cost: \$179.28 each

Description: handles high torque applications



Shaft Adapters

Product: Step-down Shaft adapter

Part #: 9283T11

Cost: \$69.87 each

Description: Reduces shaft diameter while still increasing length.



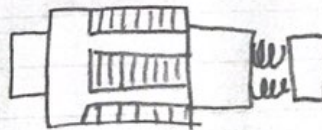
Torque-Limiting Shaft Couplings

Product: Torque-Limiting Coupler

Part #: 9132K32

Cost: \$128.25 each

Description: ~~the~~ stops load when too much torque is applied.



Machinable Shaft End

Product: 7/8" Diameter keyed shafts,

Part #: 3463N91

Cost: \$103.50 each

Description: allows you to clamp parts to shaft

