

# Lagrangian Dynamics

Dominik Szablonski

November 27, 2024

# Contents

<b>1</b>	<b>Lagrangian Mechanics</b>	<b>3</b>
1.1	Introduction . . . . .	3
1.2	Review of classical mechanics and mathematical preliminaries . . . . .	3
1.2.1	Potential confusion . . . . .	4
1.3	Lagrangian and Lagrange's Equation . . . . .	4
1.3.1	Generalised approach to solving a dynamical system . . . . .	5
1.3.2	Generalised Coordinates . . . . .	5
<b>2</b>	<b>The Calculus of Variations</b>	<b>6</b>
2.1	The Principle of Least Action . . . . .	6
<b>3</b>	<b>Hamiltonian Mechanics</b>	<b>8</b>
3.1	Euler-Lagrange Equations of the Second Kind . . . . .	9
3.2	Theory of (Galilean) Relativity . . . . .	9
3.2.1	Time Dependence . . . . .	10
3.2.2	Spatial Dependence . . . . .	10
3.2.3	Spatial Orientation . . . . .	11
3.2.4	Independence of Reference Frame . . . . .	11
3.2.5	Other Symmetries of the Lagrangian and the Hamiltonian . . . . .	12
<b>4</b>	<b>Normal Modes for Lagrangian Dynamics</b>	<b>13</b>
4.1	Informal Method . . . . .	13
4.2	More Formal Method . . . . .	14
4.3	Formal Matrix Treatment . . . . .	14
4.3.1	Lagrangian in Matrix Form . . . . .	15
4.3.2	Diagonalisation of the equations of motion . . . . .	15
4.3.3	Zero-Point Modes . . . . .	15
4.3.4	Degenerate Normal Modes . . . . .	15
<b>A</b>	<b>Misc. Formulae</b>	<b>16</b>
A.1	Total Derivative . . . . .	16

# Chapter 1

## Lagrangian Mechanics

### 1.1 Introduction

We have up until now solved dynamical problems using Newton's laws. However, they are not applicable to all scenarios. The laws are the following,

1. A body acted on with no forces moves with uniform velocity.
  - Einstein's general relativity is a counterexample to this law.
2. The rate of change of momentum of a body is give by the total force acting on that body.
  - In quantum mechanics, momentum is probabilistic.
3. Every action force has an equal and opposite reaction force.
  - This is not true in collective phenomenon, e.g. confinement of quarks in a proton.

Lagrangian/Hamiltonian mechanics is able to solve all systems where Newtonian mechanics is applicable<sup>1</sup>, as well as alleviating Newtonian mechanics of its shortcomings.

### 1.2 Review of classical mechanics and mathematical preliminaries

Consider a particle of mass  $m$  moving in 1 dimension in an arbitrary, explicitly time-independent potential  $V(x)$ . The particle will feel a force  $F$  due to the potential such that,

$$F = -\frac{\partial V}{\partial x} \quad (1.1)$$

and has an acceleration such that,

$$F = m\ddot{x}. \quad (1.2)$$

The kinetic energy of the particle is given by,

$$T = \frac{1}{2}m\dot{x}^2 \quad (1.3)$$

and the total energy,

$$E = T + V \quad (1.4)$$

is conserved. The momentum,

$$p = m\dot{x} \quad (1.5)$$

is not conserved.

---

<sup>1</sup>Except for cases with friction, which will be omitted in this course.

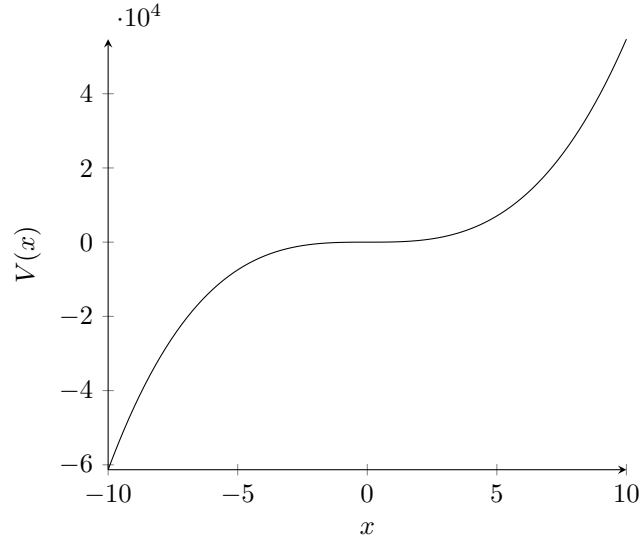


Figure 1.1: Arbitrary potential

### 1.2.1 Potential confusion

We have stated that the potential is *explicitly* time independent, however, we may have reason to suspect this is not the case. There are two ways to look at this,

1. **NO** → The potential does not depend on  $t$  as we have not specified it to be so. Mathematically, we can express this via the partial derivative,

$$\frac{\partial V}{\partial t} = 0. \quad (1.6)$$

2. **YES** → The potential does depend on time, as the particle may move to a different position, and thus be in an area of different potential, i.e. the time dependence is *implicit*. Mathematically, this is expressed by considering the total derivative,

$$\begin{aligned} \frac{dV}{dt} &= \frac{\partial V}{\partial t} + \frac{\partial V}{\partial x} \frac{dx}{dt} + \frac{\partial V}{\partial \dot{x}} \frac{d\dot{x}}{dt} \\ &= \frac{\partial V}{\partial x} \frac{dx}{dt} = \frac{\partial V}{\partial x} \dot{x}. \end{aligned} \quad (1.7)$$

We can extend this concept further to kinetic energy  $T$  and total energy  $E$ , whose total derivatives with time are,

$$\frac{dT}{dt} = m\dot{x}\ddot{x} \quad (1.8)$$

$$\frac{dE}{dt} = \frac{d}{dt}(T + V) = \dot{x} \left( \frac{\partial V}{\partial x} + m\ddot{x} \right). \quad (1.9)$$

We can get some very interesting physics from eq. (1.9), and where *Lagrange's approach* to mechanics falls in. Lagrange's method says to assume two solutions to eq. (1.9),

$$\dot{x} = 0 \quad \frac{\partial V}{\partial x} + m\ddot{x} = 0 \quad (1.10)$$

where the latter is what we wish to investigate.

## 1.3 Lagrangian and Lagrange's Equation

Let us define the *Lagrangian*,

$$\boxed{L = T - V}. \quad (1.11)$$

In the 1 dimensional case,

$$L(x, \dot{x}) = \frac{1}{2}m\dot{x}^2 - V(x)$$

$$\frac{\partial L}{\partial x} = \underbrace{-\frac{\partial V}{\partial x}}_{\text{Newton's second law}} = m\ddot{x} = \frac{d}{dt}(m\dot{x}) = \frac{d}{dt}\left(\frac{\partial L}{\partial \dot{x}}\right). \quad (1.12)$$

From eq. (1.12), we obtain *Lagrange's equation*,

$$\boxed{\frac{d}{dt}\left(\frac{\partial L}{\partial \dot{x}}\right) = \frac{\partial L}{\partial x}}. \quad (1.13)$$

### 1.3.1 Generalised approach to solving a dynamical system

1. Work out the degrees of freedom of a system, that is,

$$\{q_i\} \quad \text{Set of generalised coordinates} \quad (1.14)$$

$$\{\dot{q}_i\} \quad \text{Set of generalised velocities.} \quad (1.15)$$

2. Write down the Lagrangian,

$$L(\{q_i\}, \{\dot{q}_i\}, t) = T - V. \quad (1.16)$$

3. Derive the equations of motion using Lagrange's equation,

$$\underbrace{\frac{d}{dt}\left(\frac{\partial L}{\partial \dot{q}_i}\right)}_{\text{Generalised Momentum}} = \underbrace{\frac{\partial L}{\partial q_i}}_{\text{Generalised Force}} \quad (1.17)$$

### 1.3.2 Generalised Coordinates

Before we proceed to talking about generalised coordinates, let's find the Lagrangian in plane polar coordinates. We require the following velocities to compute the Lagrangian,

$$v_r = \dot{r} \quad v_\theta = r\dot{\theta} \quad v^2 = \dot{r}^2 + r^2\dot{\theta}^2. \quad (1.18)$$

The kinetic energy is then,

$$T = \frac{1}{2}m(\dot{r}^2 + r^2\dot{\theta}^2), \quad (1.19)$$

and the Lagrangian,

$$L = \frac{1}{2}m(\dot{r}^2 + r^2\dot{\theta}^2) - V(r, \theta). \quad (1.20)$$

Since there are two coordinates in the Lagrangian, we have 2 equations of motion, given by,

$$\frac{d}{dt}\left(\frac{\partial L}{\partial \dot{r}}\right) = \frac{\partial L}{\partial r}, \quad (1.21)$$

$$\frac{d}{dt}\left(\frac{\partial L}{\partial \dot{\theta}}\right) = \frac{\partial L}{\partial \theta}. \quad (1.22)$$

We find that each coordinate (degree of freedom) is treated democratically. If we were to do this in spherical polar coordinates, we would find that we would get 3 equations of motion, etc. Instead of being restricted to a set coordinate system, let us describe the lagrangian for a *generalised coordinate system*. Below are the generalised quantities we should be aware of:

$$\text{Generalised coordinates} \quad q_i(t) \quad (1.23)$$

$$\text{Generalised velocities} \quad \dot{q}_i(t) \quad (1.24)$$

$$\text{Generalised momenta} \quad p_i(t) = \frac{\partial L}{\partial \dot{q}_i} \quad (1.25)$$

for which the generalised Lagrangian equation is given by eq. (1.17).

## Chapter 2

# The Calculus of Variations

### 2.1 The Principle of Least Action

We define the action  $S$ ,

$$S = \int L \, dt \quad (2.1)$$

and we assert that a system evolved over time such that the action is minimised. This is a fundamental law of physics. We have that  $S = S[x(t)]$ , indicating it is a function of a function.

Our general problem is that we wish to find a function  $x(t)$  which minimises,

$$S[x(t)] = \int_{t_1}^{t_2} L(x(t), \dot{x}(t), t) \, dt \quad (2.2)$$

subject to the boundary conditions,

$$x(t_1) = x_1 \quad (2.3)$$

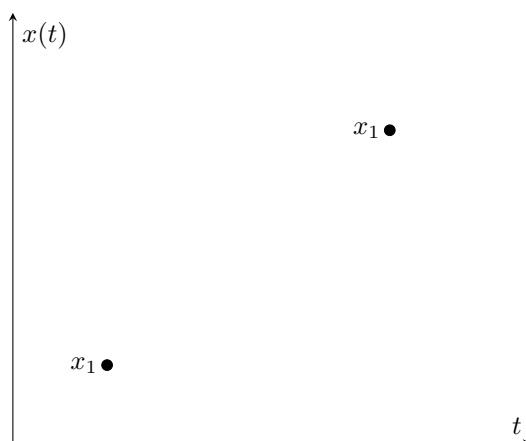
$$x(t_2) = x_2 \quad (2.4)$$

for single values functions of time.

Let us suppose a function  $x(t)$  minimises  $S$ . Let us now consider a small displacement  $\delta x(t)$  on  $x(t)$ , such that we have a modified path  $x(t) + \delta x(t)$ . This modified path must obey the boundary conditions, so

$$\delta x(t_1) = 0 \quad (2.5)$$

$$\delta x(t_2) = 0. \quad (2.6)$$



We must find an  $x(t)$  such that  $\delta x(t)$  does not change the value of  $S$ . We can write the modified Lagrangian,

$$L\left(x(t) + \delta x(t), \dot{x}(t) + \frac{d}{dt}\delta x(t), t\right) = L + \delta L. \quad (2.7)$$

We can then calculate the modified action,

$$\delta S = \int_{t_1}^{t_2} \delta L dt = \int_{t_1}^{t_2} \left[ \frac{\partial L}{\partial x} \delta x(t) + \frac{\partial L}{\partial \dot{x}} \frac{d}{dt}(\delta x(t)) \right] dt. \quad (2.8)$$

We can integrate the second term of eq. (2.8) by parts,

$$\begin{aligned} \int_{t_1}^{t_2} \frac{\partial L}{\partial \dot{x}} \frac{d}{dt}(\delta x(t)) dt &= \underbrace{\left[ \frac{\partial L}{\partial \dot{x}} \delta x(t) \right]_{t_1}^{t_2}}_{0 \text{ by BCs}} - \int_{t_1}^{t_2} \delta x(t) \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{x}} \right) dt \\ &= - \int_{t_1}^{t_2} \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{x}} \right) \delta x(t) dt \\ \implies \delta S &= \int_{t_1}^{t_2} \left[ \frac{\partial L}{\partial x} - \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{x}} \right) \right] \delta x(t) dt = 0. \end{aligned} \quad (2.9)$$

We can interpret eq. (2.9) in 2 ways,

1.  $\delta x(t)$  is carefully chosen such that the integrand is non-zero, but integrates to 0. However, since  $\delta x(t)$  is arbitrary, we can choose it to not be so;
2. The integrand is 0 everywhere.

Given the second conclusion, we have that,

$$\delta x(t) = 0 \quad \boxed{\frac{\partial L}{\partial x} - \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{x}} \right) = 0} \quad (2.10)$$

and we have thus derived the Euler-Lagrange equations.

A further conclusion is that the principle of least action directly implies that  $x(t)$  obeys Lagrange's equation, which directly implies Newton's laws.

### Principle of Least Action - More Generally

Any function defined,

$$f = f(y_i(x), y'_i(x), x) \quad (2.11)$$

with an integral,

$$S = \int_{x_1}^{x_2} f dx \quad (2.12)$$

is minimised by,

$$\frac{d}{dx} \left( \frac{\partial f}{\partial y'_i} \right) = \frac{\partial f}{\partial y_i}. \quad (2.13)$$

## Chapter 3

# Hamiltonian Mechanics

We are motivated to reformulate our approach to mechanics to directly include the conservation of momentum. However, because of the nature of partial derivatives, simply rewriting the Lagrangian to be  $L = L(q, p, t)$  violates the principle of least action. We require a new quantity, which we will call the Hamiltonian, which we write  $H = H(\{q_i\}, \{p_i\}, t)$  which we can obtain by the *Legendre Transform*.

Let us begin by writing,

$$dL = \frac{\partial L}{\partial q} dq + \frac{\partial L}{\partial \dot{q}} d\dot{q} + \frac{\partial L}{\partial t} dt. \quad (3.1)$$

We want to find a function  $H(q, p, t)$  where,

$$p \equiv \frac{\partial L}{\partial \dot{q}} \quad (3.2)$$

where,

$$dH = \frac{\partial H}{\partial q} dq + \frac{\partial H}{\partial p} dp + \frac{\partial H}{\partial t} dt. \quad (3.3)$$

We have that the Legendre transformation of  $L$  is  $H$ , so,

$$H = p\dot{q} - L. \quad (3.4)$$

From eq. (3.4),

$$dH = p d\dot{q} + \dot{q} dp - dL. \quad (3.5)$$

and by eq. (3.1),

$$\begin{aligned} dH &= p d\dot{q} + \dot{q} dp - \frac{\partial L}{\partial q} dq - \underbrace{\frac{\partial L}{\partial \dot{q}} d\dot{q}}_{p d\dot{q}} - \frac{\partial L}{\partial t} dt \\ &= \dot{q} dp - \frac{\partial L}{\partial q} dq - \frac{\partial L}{\partial t} dt. \end{aligned} \quad (3.6)$$

By comparing the coefficients of eqs. (3.3) and (3.6), we obtain *Hamilton's equations*,

$$\frac{\partial H}{\partial p} = \dot{q} \quad \frac{\partial H}{\partial q} = -\dot{p} \quad \frac{\partial H}{\partial t} = -\frac{\partial L}{\partial t}. \quad (3.7)$$

$$(3.8)$$

Most generally, for a system of  $N$  degrees of freedom  $\{q_i\}$ , and Lagrangian,  $L(\{q_i\}, \{\dot{q}_i\}, t)$ , the hamiltonian is defined,

$$H(\{q_i\}, \{p_i\}, t) = \sum_i p_i \dot{q}_i - L(\{q_i\}, \{\dot{q}_i\}, t) \quad (3.9)$$

for,

$$p_i = \frac{\partial L}{\partial \dot{q}_i}, \quad (3.10)$$



and Hamilton's equations,

$$\boxed{\frac{\partial H}{\partial p_i} = \dot{q}_i} \quad (3.11)$$

$$\boxed{\frac{\partial H}{\partial q_i} = -\dot{p}_i} \quad (3.12)$$

$$\boxed{\frac{\partial H}{\partial t} = -\frac{\partial L}{\partial t}}, \quad (3.13)$$

which produce  $2N$  1<sup>st</sup> order differential equations, in contrast to the  $N$  2<sup>nd</sup> order differential equations produces by Lagrange's method. Let us further state that,

$$\frac{\partial H}{\partial t} = \frac{dH}{dt}. \quad (3.14)$$

### 3.1 Euler-Lagrange Equations of the Second Kind

Equations eq. (2.10) are known as the Euler-Lagrange equations of the first kind. Let us consider a general function  $f(y(x), y'(x), x)$  and its total derivative in  $x$ ,

$$\frac{df}{dx} = \frac{\partial f}{\partial x} + \frac{\partial f}{\partial y'} y' + \frac{\partial f}{\partial y''} y''. \quad (3.15)$$

If  $y$  is a solution of the Euler-Lagrange equations, then eq. (2.13) holds, and,

$$\begin{aligned} \frac{df}{dx} &= \frac{\partial f}{\partial x} + \frac{d}{dx} \left( \frac{\partial f}{\partial y'} \right) y' + \frac{\partial f}{\partial y''} y'' \\ &= \frac{\partial f}{\partial x} + \frac{d}{dx} \left( y' \frac{\partial f}{\partial y'} \right) \end{aligned} \quad (3.16)$$

where the latter terms were reduced by the reverse product rule. By trivial rearranging, eq. (3.16) becomes the Euler-Lagrange equations of the second kind,

$$\boxed{\frac{\partial f}{\partial x} = \frac{d}{dx} \left( f - y' \frac{\partial f}{\partial y'} \right)}. \quad (3.17)$$

Applying eq. (3.17) to the Lagrangian,

$$\begin{aligned} \frac{d}{dt} \underbrace{\left( L - \dot{x} \frac{\partial L}{\partial \dot{x}} \right)}_{-H} &= \underbrace{\frac{\partial L}{\partial t}}_{-\frac{\partial H}{\partial t}} \\ \implies \frac{dH}{dt} &= \frac{\partial H}{\partial t}. \end{aligned} \quad (3.18)$$

### 3.2 Theory of (Galilean) Relativity

The postulates of Galilean relativity state that there is no preferred,

1. Origin of time;
2. Position in space;
3. Inertial frame of reference;
4. Orientation in space;

which hold in classical physics. Let us analyse the consequences of these postulates in the Hamiltonian. Let us first define our system:

Consider an isolated system of  $N$  particles of mass  $m_j$  with vector coordinates  $\mathbf{r}_j$  and momenta  $\mathbf{p}_j$ .

For a system of  $N$  particles in 3 dimensions, we have  $3N$  degrees of freedom and  $6N$  independent variables. The total momentum can be defined,

$$\mathbf{P} = \sum_j \mathbf{p}_j. \quad (3.19)$$

We can then summarise the consequences of the postulates,

1. Equations of motion are unchanged by a displacement in time, i.e.,  $t \rightarrow t + \delta t$ . This requires the Hamiltonian to be explicitly time independent,

$$\frac{\partial H}{\partial t} = 0 = \frac{dH}{dt} \quad (3.20)$$

implying  $H$  is a conserved quantity.

2. Equations of motion are unchanged by a displacement of the entire system in space. We must then require  $H(q_i, p_i)$  to only depend on relative position,

$$\mathbf{r}_j - \mathbf{r}_k = \mathbf{r}_{jk} \equiv \mathbf{r}_j \quad (3.21)$$

4. Equations of motion are invariant under spacial rotations. We require  $H$  to only depend on scalar products of  $\mathbf{r}_{jk}$ .

We will analyse these in more detail in the sections below.

### 3.2.1 Time Dependence

Consider a general functions,  $F(q_i, p_i, t)$ . The general equation for change of the function with time is,

$$\begin{aligned} \frac{dF}{dt} &= \sum_i \left( \frac{\partial F}{\partial q_i} \dot{q}_i + \frac{\partial F}{\partial p_i} \dot{p}_i \right) + \frac{\partial F}{\partial t} \\ \text{Hamilton's Equations} \implies &= \underbrace{\sum_i \left( \frac{\partial F}{\partial q_i} \frac{\partial H}{\partial p_i} - \frac{\partial F}{\partial p_i} \frac{\partial H}{\partial q_i} \right)}_{[F, H]} + \frac{\partial F}{\partial t} \end{aligned} \quad (3.22)$$

Above, we have defined the *Poisson bracket*, denoted as  $[F, H]$ . Let us note that  $[F, G] = -[G, F]$  for any function  $F$  and  $G$ . Let us then rewrite eq. (3.22) fully,

$$\boxed{\frac{\partial F}{\partial t} = [F, H] + \frac{\partial F}{\partial t}} \quad (3.23)$$

$\implies$  The Hamiltonian generates a displacement in time..

If  $\frac{\partial F}{\partial t} = 0$ , and  $[F, H] = 0$ , then we say  $F$  commutes and is a *conserved quantity*. Furthermore, the Poisson bracket for the set  $\{\{q_i\}, \{p_i\}\}$  is given by,

$$[q_\alpha, p_\beta] = \delta_{\alpha\beta}. \quad (3.24)$$

### 3.2.2 Spatial Dependence

Let us denote the generalised coordinates  $q_i$  as  $q_{sj}$ , where  $s$  represents the spacial direction, and  $j$  denotes the particle. Let us consider a displacement of  $N$  particles in the  $x$  direction,

$$q_{xj} \rightarrow q_{xj} + \delta x, \forall j \in \mathbb{N} \quad (3.25)$$

Our function  $F$  then undergoes a displacement,

$$F \rightarrow F + \delta F \quad (3.26)$$

where we can approximate  $\delta F$  by Taylor expansion,

$$\delta F = \sum_i \frac{\partial F}{\partial q_{xi}} \delta x. \quad (3.27)$$

Let us assume that this displacement is generated by the total  $x$  momentum,  $P_x = \sum_k p_{xk}$ . Let us study the behaviour of the Poisson bracket,

$$\begin{aligned} [F, P_x] &= \sum_j \sum_s \left( \frac{\partial F}{\partial q_{sj}} \underbrace{\frac{\partial P_x}{\partial p_{sj}}}_{\delta_{sx}} - \frac{\partial F}{\partial p_{sj}} \underbrace{\frac{\partial P_x}{\partial q_{sj}}}_0 \right) \\ &= \sum_j \frac{\partial F}{\partial q_{xj}} \\ \implies \delta F &= [F, P_x] \delta x \end{aligned} \quad (3.28)$$

which in words, states,  $P_x$  generates a displacement in the  $x$ -direction of the entire system.

$P_x$  can be applied to any observable, so let us apply it to the Hamiltonian. We have,

$$\Delta H = [H, P_x] \delta x \quad (3.29)$$

Spatial displacements leave the Hamiltonian unchanged, so we have,

$$[H, P_x] = 0. \quad (3.30)$$

We furthermore have the relation in eq. (3.23), when applied to  $P_x$ ,

$$\frac{dP_x}{dt} = [P_x, H] = -[H, P_x] = 0 \quad (3.31)$$

which reveals linear momentum is a conserved quantity.

### 3.2.3 Spatial Orientation

Generator of rotations about an axis is the component of angular momentum along that axis. Angular momentum is conserved.

### 3.2.4 Independence of Reference Frame

A boost is a change of inertial frame by a constant velocity. We wish to find the generator of boosts. Let us call this quantity  $B$ ,

$$B = \sum_i (m_i x_i - t p_i) \quad (3.32)$$

which is the position of where the centre of mass was at  $t = 0$ , multiplied by the total mass,  $M = \sum_i m_i$ . We wish to use this quantity as it is conserved. Let us consider the Poisson bracket of an arbitrary function  $F$  with  $B$ ,

$$\begin{aligned} [F, B] &= \sum_i \left( \frac{\partial F}{\partial q_i} \frac{\partial B}{\partial p_i} - \frac{\partial B}{\partial q_i} \frac{\partial F}{\partial p_i} \right) \\ &= \sum_i m_i \left( \frac{\partial F}{\partial q_i} (-t) - \frac{\partial F}{\partial p_i} (m_i) \right) \end{aligned} \quad (3.33)$$

Let us now set  $F = q_\alpha$ . A small displacement to this generalised coordinate can be written as,  $\delta q_k = -t\varepsilon$ , where  $\varepsilon$  is a small velocity. The transformation caused by the generator  $B$  in the  $x$ -coordinate is then,

$$x_\alpha \rightarrow x_\alpha - \varepsilon t. \quad (3.34)$$

Similarly, let us consider  $F = p_\alpha$ . The small displacement in this generalised momentum is,  $p_\alpha = -m_\alpha \varepsilon$ . So, the transformation caused by the generator  $B$  in the  $x$  component of momentum is,

$$p_{\alpha x} \rightarrow p_{\alpha x} - m_\alpha \varepsilon \quad (3.35)$$

In summary,  $B$  generates boosts in the  $x$  direction of velocity  $\varepsilon$ .

### Boost generator behaviour with the Hamiltonian

Given that  $B$  is a conservative quantity, we must write,

$$\frac{dB}{dt} = [H, B] + \frac{\partial B}{\partial t} = 0. \quad (3.36)$$

Rearranging this,

$$[H, B] = -\frac{\partial B}{\partial t} = \sum_i p_i = P. \quad (3.37)$$

In order for eq. (3.37) to hold, we require  $H \propto P^2$ , i.e., kinetic energy depending quadratically on momenta is a consequence of the principle of relativity.

### 3.2.5 Other Symmetries of the Lagrangian and the Hamiltonian

1.  $L$  is invariant under the adding of a constant independent of  $q_i, \dot{q}_i, t$ .
2.  $L$  is invariant under scaling, i.e.,  $L$  is invariant under a transformation  $L' = KL$  where  $K \neq K(q_i, \dot{q}_i, t)$ .
3. Invariant under the addition of a term of the form  $\frac{df}{dt}$  where  $f = f(q_i, t)$ . I.e., invariant under transformation  $L' = L + \frac{df}{dt}$ .

*Proof.* Consider the action,

$$\begin{aligned} S' &= \int_{t_1}^{t_2} L'(q_i, \dot{q}_i, t) dt = \int_{t_1}^{t_2} L(q_i, \dot{q}_i, t) dt + \int_{t_1}^{t_2} \frac{df}{dt} dt \\ &= S + [f]_{t_1}^{t_2} \\ &= S + f(q_i(t_1), t_1) - f(q_i(t_2), t_2) \end{aligned} \quad (3.38)$$

and the latter terms only depend on the end points of the action. These points are fixed, which imply that the function  $f$  does not depend on the path. So we have that  $\delta S' = \delta S = 0$ , and the action after the transformation corresponds to the same path as before the transformation. So the equations of motion remain unchanged.  $\square$

## Chapter 4

# Normal Modes for Lagrangian Dynamics

Normal modes are the modes of the system where all parts of the system are oscillating at the same frequency. "Normal" refers to orthogonality of the modes, meaning that there is no coupling between the modes. If a system oscillates in one of its normal modes, it will stay in that mode.

We will consider 3 different methods of finding normal modes by analysing a simple coupled pendulum. Let us first state the lagrangian for this system,

$$L = \frac{1}{2}m(\dot{x}_1^2 + \dot{x}_2^2) - \frac{1}{2}k(x_2 - x_1)^2 - \frac{1}{2}\frac{mg}{l} \quad (4.1)$$

with the corresponding equations of motion,

$$m\ddot{x}_1 = -k(x_1 - x_2) - \frac{mg}{l}x_1 \quad (4.2)$$

$$m\ddot{x}_2 = +k(x_1 - x_2) - \frac{mg}{l}x_2 \quad (4.3)$$

### 4.1 Informal Method

Let us add eq. (4.2) and eq. (4.3), and define  $X_1 = x_1 + x_2$ ,

$$m\ddot{X}_1 = -m\frac{g}{l}X_1 \quad (4.4)$$

from which we can clearly identify SHM in  $X_1$  with  $\omega^2 = \frac{g}{l}$ , corresponding to the first normal mode.

Let us now subtract the two equations of motion and define  $X_2 = x_1 - x_2$ ,

$$\ddot{X}_2 = -\left(\frac{2k}{m} + \frac{g}{l}\right)X_2 \quad (4.5)$$

which is clearly SHM in  $X_2$  with  $\omega^2 = \frac{2k}{m} + \frac{g}{l}$ .

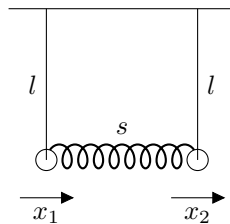


Figure 4.1

## 4.2 More Formal Method

In normal modes, all components oscillate with the same frequency. We can then assume a general solution,

$$x_j = C_j e^{i\omega_n t} \implies \ddot{x}_j = -\omega_n^2 C_j e^{i\omega_n t}. \quad (4.6)$$

Doing this transforms a simultaneous differential equation into a purely algebraic form. Substituting this into our equations of motion and dividing through by  $e^{i\omega_n t}$ ,

$$-m\omega_n^2 C_1 = -k(C_1 - C_2) - m\frac{g}{l}C_1 \quad (4.7)$$

$$-m\omega_n^2 C_2 = +k(C_1 - C_2) - m\frac{g}{l}C_2. \quad (4.8)$$

We then obtain two solutions,

$$C_2 = C_1 \qquad C_2 = -C_1 \quad (4.9)$$

which correspond to the pendula moving in the same direction with equal amplitude, and moving in opposite directions with equal amplitude. Our solutions for the normal modes are as in the previous section.

## 4.3 Formal Matrix Treatment

Let us consider the equations of motion again, assuming that all parts of the system are performing simple harmonic motion,

$$m\ddot{x}_1 = -m\omega_n^2 x_1 = -\left(\frac{mg}{l} + k\right)x_1 - (-k)x_2 \quad (4.10)$$

$$m\ddot{x}_2 = -m\omega_n^2 x_2 = -(-k)x_1 - \left(\frac{mg}{l} + k\right)x_2 \quad (4.11)$$

$$(4.12)$$

We can clearly rewrite these equations as a single matrix equation,

$$M\ddot{X} = -\omega_n^2 M X = -K X \quad (4.13)$$

if we define,

$$X = \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} \qquad M = \begin{pmatrix} m & 0 \\ 0 & m \end{pmatrix} \qquad K = \begin{pmatrix} \frac{mg}{l} + k & -k \\ -k & \frac{mg}{l} + k \end{pmatrix} \quad (4.14)$$

where  $X$  is the vector of our degrees of freedom,  $M$  is our mass matrix, and  $K$  encodes the restoring forces acting on the system. We are further able to identify the equation,

$$K X = \omega_n^2 M X. \quad (4.15)$$

Multiplying eq. (4.15) by  $M^{-1}$ ,

$$\boxed{M^{-1} K X = \omega_n^2 X} \quad (4.16)$$

which is the general for motion *close to an equilibrium* in matrix form, which is an eigen equation. The eigenvalues of this equation represent the normal frequencies, i.e., solving,

$$\det\{M^{-1} K - \omega_n^2 I\} = 0. \quad (4.17)$$

Evaluating this produces the normal frequencies as in the previous sections. By further finding the eigenvectors of the eigenequation, we are able to identify the normal modes of the system. For our particular system with the coupled pendula, we find that the eigenvectors are,

$$X_1 = \frac{1}{\sqrt{2}} \begin{pmatrix} 1 \\ 1 \end{pmatrix} C \qquad X_2 = \frac{1}{\sqrt{2}} \begin{pmatrix} 1 \\ -1 \end{pmatrix} C \quad (4.18)$$

where  $C$  is the amplitude of the oscillation. We can interpret the first normal mode corresponds to the two masses moving in the same direction, while the second normal mode corresponds to the two masses

moving in opposite directions with equal amplitude.

In order to write the general displacement of a mass  $m_j$  in the  $n$ th normal mode, we must take into account the amplitude and phase,

$$x_j^{(n)} = X_n^{(j)} C_n e^{i(\omega_n t)} \quad (4.19)$$

and to get the total displacement of the mass  $m_j$ , we must sum over all normal modes,

$$x_j = \sum_n \left( Q_n^{(j)} C_n e^{i(\omega_n t)} \right) \quad (4.20)$$

where we determine  $C_n$  from initial conditions.  $C_n$  can be complex in order to encode the phase difference. NOTE: We require  $2N$  conditions for  $N$  degrees of freedom.

### 4.3.1 Lagrangian in Matrix Form

The Lagrangian in matrix form is given by,

$$L = \frac{1}{2} \dot{X}^T M \dot{X} - \frac{1}{2} X^T K X \quad (4.21)$$

### 4.3.2 Diagonalisation of the equations of motion

We can diagonalise our equations of motion. If eigenvectors of our equations of motion are orthogonal, such that,

$$\mathbf{X}^{(j)} \cdot \mathbf{X}^{(k)} = \delta_{ij} \quad (4.22)$$

then we can construct an orthogonal matrix, whose columns are the eigenvectors of the system,

$$B_{ij} = \mathbf{X}_i^{(j)}. \quad (4.23)$$

From this we can diagonalise our equation of motion, i.e., eq. (4.16),

$$\underbrace{(B^{-1} M^{-1} K B)}_{\text{Diagonal Matrix}} \underbrace{(B^{-1} \mathbf{X})}_Q = \omega_n^2 \underbrace{(B^{-1} X)}_Q \quad (4.24)$$

which produces equations of motion which are diagonal in the normal modes of our system, or viewing this another way, in the normal coordinates of the system. We can further diagonalise the Lagrangian,

$$L = \frac{1}{2} \dot{Q}^T B^T M B \dot{Q} - \frac{1}{2} \dot{Q}^T B^T K B Q \quad (4.25)$$

This allows for the normal modes of the system to be decoupled.

### 4.3.3 Zero-Point Modes

Zero point modes occur for eigenvalues of  $\lambda = 0$ . In a system close to equilibrium, this means  $\omega^2 = 0$ . We have,

$$\ddot{\mathbf{X}} = \omega^2 \mathbf{X} \implies \ddot{\mathbf{X}} = 0. \quad (4.26)$$

The normal mode is then,

$$\dot{x}_1 = \dot{x}_2 = \dot{x}_3 = \text{Const.} \quad (4.27)$$

which is known as free motion.

### 4.3.4 Degenerate Normal Modes

Degenerate modes occur when we have a solution to the eigenvector equation such that,

$$x_1 + x_2 + x_3 = 0 \quad (4.28)$$

which implies that there are infinitely many eigenvectors, so long that they satisfy eq. (4.28). Furthermore, the linear of two degenerate normal modes always produces another degenerate normal mode which satisfied eq. (4.28).

# Appendix A

## Misc. Formulæ

### A.1 Total Derivative

$$\frac{df}{dt} = \sum_i \frac{\partial f}{\partial r_i} \frac{dr_i}{dt} \tag{A.1}$$

for  $r_i = r_i(t)$ .