

Supplementary Material:
A Spatially-Lifted SCvx NMPC for Executing
SIPP-Generated
Time-Corridor Trajectories on an Articulated
Loader

Thanh Binh Do and François Guérin

Scope

This document provides the detailed mathematical proofs omitted from the main ECC paper [1], following the same notation and equation numbering. The spatial dynamics are

$$z' = f_a(z, s) + G(z, s)w,$$

with f_a, G as in (13)–(14), admissible state set Z_s defined by Assumption 1, and lifted control set \mathcal{W} by (16). We give constructive proofs for Lemma 1 (well-posedness), Theorem 2 (recursive feasibility with explicit slack bounds), and Theorem 3 (local practical stability).

1 Proof of Lemma 1 (Well-Posedness)

The system

$$f(z, w, s) = f_a(z, s) + G(z, s)w$$

has

$$f_a(z, s) = \begin{bmatrix} (1 - d_f \kappa_{\text{ref}}) \tan e_\theta \\ \frac{(1 - d_f \kappa_{\text{ref}}) \sin \gamma}{(L_f \cos \gamma + L_r) \cos e_\theta} - \kappa_{\text{ref}}(s) \\ 0 \\ 0 \\ 0 \end{bmatrix}, \quad G(z, s) = \begin{bmatrix} 0 & 0 & 0 \\ \frac{L_r}{L_f \cos \gamma + L_r} & 0 & 0 \\ 1 & 0 & 0 \\ 0 & 0 & 1 \\ 0 & 1 & 0 \end{bmatrix}.$$

Assumption 1 ensures that all denominators remain bounded away from zero on the compact set Z_s . Define

$$C_w := (1 + \omega_{\max} + a_{\max}) w_{2,\max}, \quad \text{so that } \|w\| \leq C_w \text{ for all } w \in \mathcal{W}.$$

To establish existence and uniqueness, it suffices to show that f is globally Lipschitz in z on Z_s and satisfies Carathéodory's conditions.

Bounded Jacobians. Let $D(\gamma) = L_f \cos \gamma + L_r \geq D_{\min} > 0$. The first component $f_{a,1} = (1 - d_f \kappa_{\text{ref}}) \tan e_\theta$ has derivatives

$$\frac{\partial f_{a,1}}{\partial d_f} = -\kappa_{\text{ref}} \tan e_\theta, \quad \frac{\partial f_{a,1}}{\partial e_\theta} = (1 - d_f \kappa_{\text{ref}}) \sec^2 e_\theta.$$

Since $|e_\theta| \leq \frac{\pi}{2} - \varepsilon_\theta$, both $\tan e_\theta$ and $\sec e_\theta$ are bounded. Similar arguments applied to

$$f_{a,2} = \frac{(1 - d_f \kappa_{\text{ref}}) \sin \gamma}{D(\gamma) \cos e_\theta} - \kappa_{\text{ref}}(s)$$

show that every partial derivative of f_a remains bounded on Z_s . Thus, there exists $L_a > 0$ such that

$$\left\| \frac{\partial f_a}{\partial z}(z, s) \right\| \leq L_a \quad \forall (z, s) \in Z_s \times [0, s_f]. \quad (1)$$

For $G(z, s)$, only $G_{21} = L_r/D(\gamma)$ depends on z , and

$$\frac{\partial G_{21}}{\partial \gamma} = \frac{L_r L_f \sin \gamma}{D(\gamma)^2}$$

is also bounded. Hence there exists $L_G > 0$ such that

$$\left\| \frac{\partial g_i}{\partial z}(z, s) \right\| \leq L_G, \quad i = 1, 2, 3. \quad (2)$$

Lipschitz continuity. For any $z_1, z_2 \in Z_s$, $w \in \mathcal{W}$, and $s \in [0, s_f]$,

$$\|f_a(z_1, s) - f_a(z_2, s)\| \leq L_a \|z_1 - z_2\|, \quad \|G(z_1, s)w - G(z_2, s)w\| \leq L_G \|w\| \|z_1 - z_2\|.$$

Therefore,

$$\|f(z_1, w, s) - f(z_2, w, s)\| \leq L \|z_1 - z_2\|, \quad L := L_a + L_G C_w. \quad (3)$$

Carathéodory conditions. (i) $f(\cdot, w, s)$ is continuous in z since f_a, G are smooth. (ii) $f(z, w, \cdot)$ is measurable because $\kappa_{\text{ref}}(s)$ is measurable. (iii) On the compact domain, $\|f(z, w, s)\| \leq m_{\max}$, so $m(s) \equiv m_{\max}$ is integrable.

Hence, by the Carathéodory theorem [2, 3], the ODE

$$\frac{dz}{ds} = f(z(s), w(s), s), \quad z(0) = z_0,$$

admits a unique absolutely continuous solution for any $z_0 \in Z_s$ and measurable bounded $w(\cdot) \in L^\infty([0, s_f]; \mathcal{W})$, with continuous dependence on $(z_0, w(\cdot))$. This proves Lemma 1. \square

2 Proof of Theorem 2 (Recursive Feasibility)

Let $\{(z_{k,j}^*, w_{k,j}^*)\}$ be the optimal solution of the SCvx subproblem (23) at step k , with nominal (\bar{z}_j, \bar{w}_j) . At step $k+1$, the standard shift–append candidate is

$$\tilde{z}_0 = z_{k+1}, \quad \tilde{w}_j = \bar{w}_{j+1} \quad (j < N_p - 1), \quad \tilde{w}_{N_p-1} = K_f \tilde{x}_{N_p-1},$$

and the nonlinear update

$$\tilde{z}_{j+1} = \tilde{z}_j + \Delta s f(\tilde{z}_j, \tilde{w}_j, s_j).$$

Define $\delta z_j := \tilde{z}_j - \bar{z}_{j+1}$. From (3) and bounded $\|\tilde{w}_j - \bar{w}_{j+1}\| \leq \rho_{\max}$,

$$\|\delta z_{j+1}\| \leq (1 + L\Delta s)\|\delta z_j\| + C_G \rho_{\max} \Delta s, \quad (4)$$

which yields, via the discrete Grönwall inequality,

$$\|\delta z_j\| \leq e^{Lj\Delta s} \|\delta z_0\| + \frac{C_G \rho_{\max}}{L} (e^{Lj\Delta s} - 1).$$

Under $L N_p \Delta s \leq 1$, $e^{Lj\Delta s} \leq e$, leading to

$$\|\delta z_j\| \leq C_z \rho_{\max} N_p \Delta s. \quad (5)$$

For the consistency constraint $\psi(z)$, the Taylor remainder

$$\tilde{\nu}_j = \psi(\tilde{z}_j) - \psi(\bar{z}_j) - \nabla \psi(\bar{z}_j)^\top (\tilde{z}_j - \bar{z}_j) = \frac{1}{2} (\tilde{z}_j - \bar{z}_j)^\top \nabla^2 \psi(\xi_j) (\tilde{z}_j - \bar{z}_j)$$

satisfies $\|\tilde{\nu}_j\|_\infty \leq \frac{1}{2} H_\psi (C_z \rho_{\max} N_p \Delta s)^2 =: \epsilon_\nu$. Thus $\|\nu_j^*\|_\infty \leq \epsilon_\nu$.

The temporal deviation $\delta t_j = \tilde{t}_j - \bar{t}_{j+1}$ follows

$$\delta t_{j+1} = \delta t_j + \Delta s (\tilde{w}_{2,j} - \bar{w}_{2,j+1}),$$

and by Lipschitz continuity of ψ , $|\tilde{w}_{2,j} - \bar{w}_{2,j+1}| \leq \tilde{C} \rho_{\max} N_p \Delta s$, leading to

$$|\delta t_j| \leq C_\sigma \rho_{\max} N_p^2 \Delta s^2 =: \epsilon_\sigma.$$

Consequently, the time-corridor slacks satisfy $\|\sigma_j^{\pm, *}\|_\infty \leq \epsilon_\sigma$. This establishes the explicit bounds (27)–(29) and recursive feasibility. \square

3 Proof of Theorem 3 (Local Practical Stability)

Linearizing the terminal dynamics $(A_{\text{lin}}, B_{\text{lin}})$ and choosing a stabilizing feedback K_f , let P_f solve the Riccati equation with $Q_f \succ 0$. Then for $V(x) = x^\top P_f x$,

$$V(x^+) - V(x) = -x^\top Q_f x$$

in the linear case. For the nonlinear dynamics $x^+ = A_{\text{cl}}x + \phi(x)$ with $\|\phi(x)\| \leq c_\phi \|x\|^2$, one obtains

$$V(x^+) - V(x) \leq -\lambda_{\min}(Q_f)\|x\|^2 + 2c_A c_\phi \|x\|^3 + c_P c_\phi^2 \|x\|^4.$$

Choosing $r > 0$ such that $2c_A c_\phi r + c_P c_\phi^2 r^2 \leq \frac{1}{2}\lambda_{\min}(Q_f)$ ensures

$$V(x^+) - V(x) \leq -\frac{1}{2}\lambda_{\min}(Q_f)\|x\|^2$$

for $\|x\| \leq r$. The terminal set

$$X_f = \{x : x^\top P_f x \leq \rho_f\} \subseteq \{x : \|x\| \leq r\}$$

thus guarantees invariance and local decrease of V .

For the overall MPC value function V_k , let (z^*, w^*) be the optimal solution at step k , (\tilde{z}, \tilde{w}) the feasible candidate at step $k+1$. By optimality,

$$V_{k+1} \leq J_{\text{nl}}(\tilde{z}, \tilde{w}),$$

and standard MPC arguments yield

$$V_{k+1} - V_k \leq -\alpha_3(\|x_k\|) + (\beta_{\text{lin}} + \beta_{\text{slack}})\epsilon,$$

where $\epsilon = \max\{\epsilon_\nu, \epsilon_\sigma\}$,

$$\beta_{\text{lin}} = (N_p L_\ell + L_\Phi) C_z N_p \Delta s, \quad \beta_{\text{slack}} = N_p (\rho_\nu + 2\rho_\sigma).$$

Hence

$$V_{k+1} - V_k \leq -\alpha_3(\|x_k\|) + \beta\epsilon, \quad \beta = \beta_{\text{lin}} + \beta_{\text{slack}}.$$

By standard Lyapunov arguments [2, 4], the closed-loop error x_k converges to a neighborhood of the origin of radius $\mathcal{O}(\sqrt{\epsilon})$, establishing local practical stability. \square

4 Kinematic Model (Front-Axle-Centric)

4.1 State and Control Inputs

We define the state vector \mathbf{q} based on the pose of the front axle, and the control input vector \mathbf{u} relative to the front axle.

The state vector is:

$$\mathbf{q} = \begin{bmatrix} x_f \\ y_f \\ \theta_f \\ \gamma \end{bmatrix}$$

where (x_f, y_f) is the position of the front axle center, θ_f is the orientation of the front frame, and γ is the articulation angle.

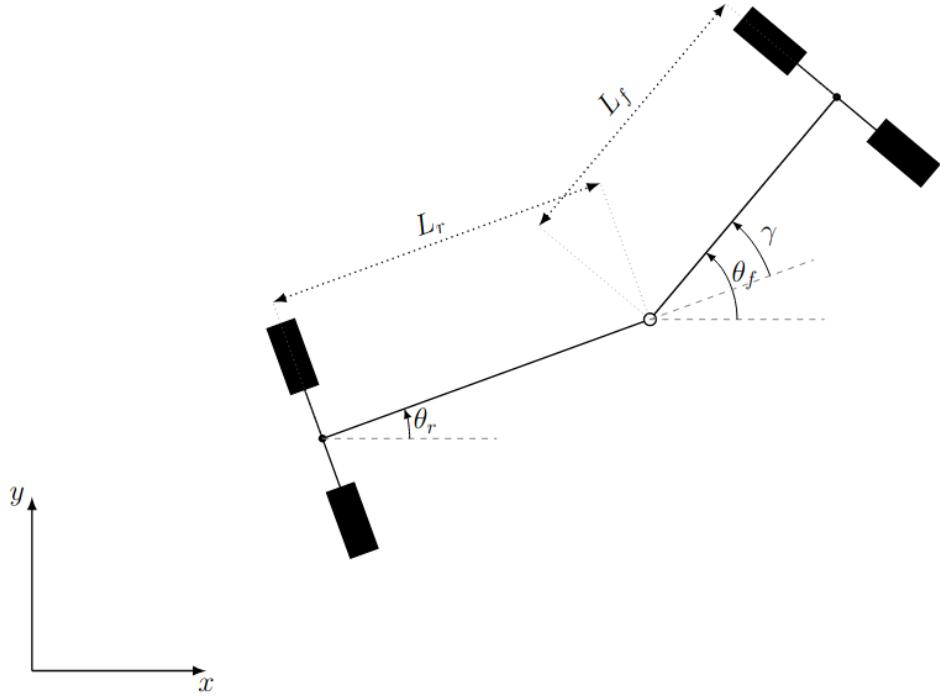


Figure 1: Robot Caterpillar 996

The control input vector is:

$$\mathbf{u} = \begin{bmatrix} v_f \\ \omega_s \end{bmatrix}$$

where v_f is the linear velocity of the front axle and $\omega_s = \dot{\gamma}$ is the articulation (steering) rate.

4.2 Geometric Constraints

The pose of the rear axle is defined algebraically based on the state \mathbf{q} . The parameters L_f and L_r are the lengths from the pivot joint to the front and rear axles, respectively.

The rear frame orientation θ_r is:

$$\theta_r = \theta_f - \gamma$$

The rear axle position (x_r, y_r) is reconstructed as:

$$x_r = x_f - L_f \cos(\theta_f) - L_r \cos(\theta_r) \quad (6)$$

$$y_r = y_f - L_f \sin(\theta_f) - L_r \sin(\theta_r) \quad (7)$$

Substituting θ_r , we get the full reconstruction:

$$x_r = x_f - L_f \cos(\theta_f) - L_r \cos(\theta_f - \gamma) \quad (8)$$

$$y_r = y_f - L_f \sin(\theta_f) - L_r \sin(\theta_f - \gamma) \quad (9)$$

4.3 Non-Holonomic (No-Slip) Constraints

4.3.1 Front Axle Constraint

The velocity of the front axle is our control input v_f in the direction θ_f . This directly gives two state derivatives:

$$\dot{x}_f = v_f \cos(\theta_f) \quad (10)$$

$$\dot{y}_f = v_f \sin(\theta_f) \quad (11)$$

4.3.2 Rear Axle Constraint

The velocity of the rear axle (\dot{x}_r, \dot{y}_r) must be in its forward direction θ_r . This side-slip constraint is used to find the rest of the model:

$$\dot{y}_r \cos(\theta_r) - \dot{x}_r \sin(\theta_r) = 0$$

4.4 Derivation of the State-Space Model

We need to find $\dot{\theta}_f$. We take the time derivative of the rear axle position (x_r, y_r) from the reconstruction equations:

$$\dot{x}_r = \dot{x}_f + L_f \sin(\theta_f) \dot{\theta}_f + L_r \sin(\theta_r) (\dot{\theta}_f - \dot{\gamma}) \quad (12)$$

$$\dot{y}_r = \dot{y}_f - L_f \cos(\theta_f) \dot{\theta}_f - L_r \cos(\theta_r) (\dot{\theta}_f - \dot{\gamma}) \quad (13)$$

Substituting these into the rear no-slip constraint yields:

$$(\dot{y}_f - L_f \cos(\theta_f) \dot{\theta}_f - \dots) \cos(\theta_r) - (\dot{x}_f + L_f \sin(\theta_f) \dot{\theta}_f + \dots) \sin(\theta_r) = 0$$

Grouping terms by $\dot{x}_f, \dot{y}_f, \dot{\theta}_f$, and $\dot{\gamma}$ simplifies the equation to:

$$(\dot{y}_f \cos \theta_r - \dot{x}_f \sin \theta_r) - \dot{\theta}_f (L_f \cos \gamma + L_r) + L_r \dot{\gamma} = 0$$

Now, substitute the inputs: $\dot{x}_f = v_f \cos \theta_f$, $\dot{y}_f = v_f \sin \theta_f$, and $\dot{\gamma} = \omega_s$. The first term simplifies:

$$v_f \sin(\theta_f) \cos(\theta_r) - v_f \cos(\theta_f) \sin(\theta_r) = v_f \sin(\theta_f - \theta_r) = v_f \sin(\gamma)$$

The full equation becomes:

$$v_f \sin(\gamma) - \dot{\theta}_f (L_f \cos \gamma + L_r) + L_r \omega_s = 0$$

Solving for $\dot{\theta}_f$:

$$\dot{\theta}_f = \frac{v_f \sin(\gamma) + L_r \omega_s}{L_f \cos(\gamma) + L_r}$$

4.5 Final Kinematic Model

The complete front-axle-centric kinematic model is:

$$\dot{\mathbf{q}} = \begin{bmatrix} \dot{x}_f \\ \dot{y}_f \\ \dot{\theta}_f \\ \dot{\gamma} \end{bmatrix} = \begin{bmatrix} v_f \cos(\theta_f) \\ v_f \sin(\theta_f) \\ \frac{v_f \sin(\gamma) + L_r \omega_s}{L_f \cos(\gamma) + L_r} \\ \omega_s \end{bmatrix}$$

5 Extension: Bucket Forward Kinematics

The position of the bucket's center of mass (x_b, y_b) can be found using forward kinematics. This requires adding the bucket angle δ to the state. The top-down projected length of the bucket arm is $L_b \cos(\delta)$.

The position of the bucket is the position of the front axle plus the vector from the axle to the bucket:

$$x_b = x_f + (L_b \cos(\delta)) \cos(\theta_f) \quad (14)$$

$$y_b = y_f + (L_b \cos(\delta)) \sin(\theta_f) \quad (15)$$

Substituting the full expressions for x_f , y_f , and θ_f gives the complete forward kinematics for the bucket:

$$x_b = x_r + L_r \cos(\theta_r) + (L_f + L_b \cos(\delta)) \cos(\theta_r + \gamma) \quad (16)$$

$$y_b = y_r + L_r \sin(\theta_r) + (L_f + L_b \cos(\delta)) \sin(\theta_r + \gamma) \quad (17)$$

References

References

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