

## ATTENTION!

**This Page Is the Cover Page of Your Project, Do Not Put Anything Before this Page**

### MEEN-414-646 Elements of Turbomachinery , Module M 2 Design of Subsonic Turbine M2-T and Compressor Blades M2-C

No. of students involved: 2 for 1 for M-2T and 1 for M-2C

Student 1: Last Name, Initial

Student 2: Last Name, Initial

Distributed: 01/27/15

**Final Date: 02/03/15**

#### M-2T Subsonic Turbine Blade Design Instruction

For the turbine design project, develop a design software that enables you to generate subsonic turbine blades.

- 1) Read Chapter 10, turbine and compressor part, **you may omit the conformal transformation part.**
- 2) Generate a family of profiles that have  $\alpha_1 = 90^\circ$ ,  $\alpha_2 = 160^\circ$ ,  $\alpha_1 = 45^\circ$ ,  $\alpha_2 = 160^\circ$ ,
- 3) Superpose the turbine base profile on the camber using the code you already developed

To design subsonic turbine blades, a step-by-step design procedure is given, see also Fig. 1

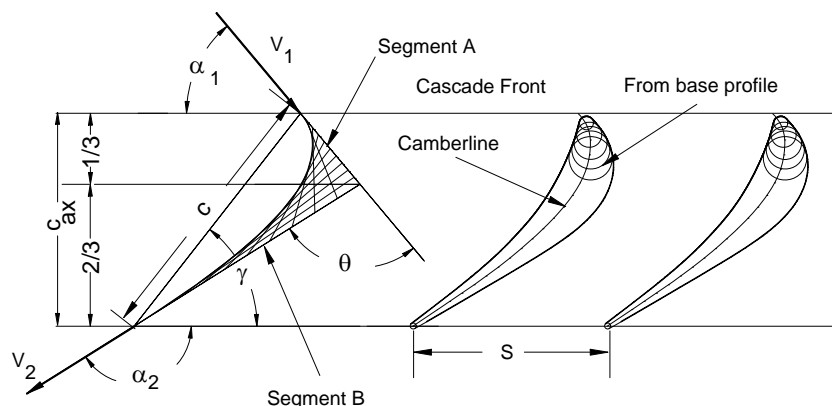
##### Step1

- 1) Draw two parallel lines that are apart by an axial chord.
- 2) Choose an axial chord  $C_{ax}$  of any length you wish.
- 3) Draw at 1/3 of the  $C_{ax}$  a parallel line to the cascade front.
- 4) Draw the inlet flow velocity vector with the given angle  $\alpha_1$  and intersect it with the 1/3 line.
- 5) At the point of intersection draw exit velocity vector line at an angle  $\alpha_2$ .
- 6) Intersect the above line with the cascade rear front line.
- 7) Connect the front intersect with the rear, you find the stagger angle  $\gamma$
- 8) With the tangent at inlet and exit as Fig 1. Construct the camberline using **Bezier function** as detailed in Chapter 10. You may use instead of 1/3 a one half distance (item 3)
- 9) Once you have the camberline, superimpose the base profile as detailed in Chapter 10. The base profile is in *ecampus* as a data file.

##### Step 2: Manual design of camber line

- 1) Divide segment A and B into equal segments as shown in Fig.1.
- 2) Connect the point 2 of A with point n-1 of B, 3 with n-2, etc to design a camber envelope.
- 3) Tangent the camber to the envelope.
- 4) You may curve fit the camber or extract discrete points.
- 5) Use the camber in conjunction with a given base line. Use the superposition principle, Chap 10 .

A Simple Method for Designing Subsonic Turbine Blades



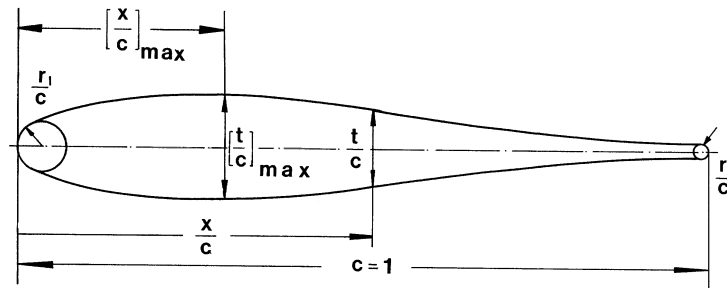
**Figure 1**

MEEN-414-646- Elements of Turbomachinery, Module 2  
M2C- Design of Compressor Blades

**M2C Module: Subsonic Compressor Design Using NACA Camberline Equation**

For the compressor design project, develop a design software that enables you to generate NACA compressor blade profiles

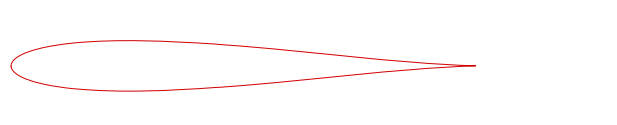
- 1) Read Chapter 10
  - 2) Follow instructions and arrive at the equation for blade camber line for compressor
  - 3) For compressor blades, the higher the blade camber height, the higher is the lift coefficient  $C_l$   
Vary  $C_l$  from 0 to 1.0 to get a feeling about the camber.
  - 4) Superimpose the following base profile and get the compressor blade
  - 5) Further instruction will be given to you in the class.
- For generation of suction and pressure side you may spline function or any other interpolation function.



**Figure 2:** Base profile

|                                     |      |      |      |      |      |      |      |      |      |      |
|-------------------------------------|------|------|------|------|------|------|------|------|------|------|
| $\left[\frac{x}{c}\right]_{\max\%}$ | 5    | 10   | 15   | 20   | 25   | 30   | 35   | 40   | 45   | 50   |
| $\left[\frac{t}{c}\right]_{\max\%}$ | 8.4  | 11.1 | 13.0 | 14.4 | 14.9 | 15.0 | 14.4 | 13.5 | 12.4 | 11.2 |
| $\left[\frac{x}{c}\right]_{\max\%}$ | 55   | 60   | 65   | 70   | 75   | 80   | 85   | 90   | 95   | 100  |
| $\left[\frac{t}{c}\right]_{\max\%}$ | 10.1 | 8.9  | 7.7  | 6.7  | 5.7  | 4.7  | 3.8  | 3.0  | 2.1  | 0.0  |

Also: The following base profile generated by Joukowsky transformation will be inserted into a special folder in *ecampus* that you have access to. Once you have the file take Column 1 and 2 only.



**Figure 3:** Base profile generated by conformal transformation

**Module II:**  
**Design of Subsonic Turbine Blade M2-T**  
**& Compressor Blade M2-C**

*Prepared by:*

Dung Tran

Jitaditya Mondal

Submission Date: 02/17/2016

## Contents

|      |                                                 |    |
|------|-------------------------------------------------|----|
| I.   | Introduction .....                              | 1  |
| II.  | Design Details.....                             | 1  |
| A.   | Nomenclature .....                              | 1  |
| B.   | Turbine Blade Design .....                      | 2  |
| B.1. | Camberline equation .....                       | 2  |
| B.2. | Suction and pressure side equation .....        | 4  |
| B.3. | Base profile (thickness) .....                  | 4  |
| C.   | Compressor Blade Design .....                   | 5  |
| C.1. | Camberline equation .....                       | 5  |
| C.2. | Suction and pressure side equation.....         | 5  |
| C.3. | Base profile (thickness) .....                  | 5  |
| III. | Results.....                                    | 6  |
| A.   | Turbine Blade design.....                       | 6  |
| B.   | Compressor Blade design.....                    | 10 |
| IV.  | Discussion and conclusion .....                 | 12 |
|      | REFERENCES .....                                | 12 |
|      | APPENDIX .....                                  | 13 |
| A.1  | Zone 1 Base-line profile .....                  | 13 |
| A.2  | Zone 1- Baseline profile: $t/c$ vs. $x/c$ ..... | 13 |
| B.1  | Zone 3 Base-line profile .....                  | 14 |
| B.2  | Zone 3- Baseline profile: $t/c$ vs. $x/c$ ..... | 14 |
| C.   | Matlab code .....                               | 15 |

## List of Figures

|                                                                                                                                      |    |
|--------------------------------------------------------------------------------------------------------------------------------------|----|
| Figure 1: Parameters of the blade .....                                                                                              | 2  |
| Figure 2: Superimposition of base profile on camberline.....                                                                         | 4  |
| Figure 3: Turbine Blade Design (Single Blade) - Zone 3 (Thick) Baseline Profile, $\alpha_1 = 45^\circ$ , $\alpha_2 = 20^\circ$ ..... | 6  |
| Figure 4: Turbine Blade Design (2 Blades) - Zone 3 (Thick) Baseline Profile, $\alpha_1 = 45^\circ$ , $\alpha_2 = 20^\circ$ .....     | 6  |
| Figure 5: Turbine Blade Design (Single Blade) - Zone 3 (Thick) Baseline Profile, $\alpha_1 = 90^\circ$ , $\alpha_2 = 20^\circ$ ..... | 7  |
| Figure 6: Turbine Blade Design (2 Blades) - Zone 3 (Thick) Baseline Profile, $\alpha_1 = 90^\circ$ , $\alpha_2 = 20^\circ$ .....     | 7  |
| Figure 7: Turbine Blade Design (Single Blade) - Zone 1 (Thin) Baseline Profile, $\alpha_1 = 45^\circ$ , $\alpha_2 = 20^\circ$ .....  | 8  |
| Figure 8: Turbine Blade Design (2 Blades) - Zone 1 (Thin) Baseline Profile, $\alpha_1 = 45^\circ$ , $\alpha_2 = 20^\circ$ .....      | 8  |
| Figure 9: Turbine Blade Design (Single Blade) - Zone 1 (Thin) Baseline Profile, $\alpha_1 = 90^\circ$ , $\alpha_2 = 20^\circ$ .....  | 9  |
| Figure 10: Turbine Blade Design (2 Blades) - Zone 1 (Thin) Baseline Profile, $\alpha_1 = 90^\circ$ , $\alpha_2 = 20^\circ$ .....     | 9  |
| Figure 11: Compressor Blade Design (Single Blade) - Zone 1 (Thin) Baseline Profile, $CL=1.0$ .....                                   | 10 |
| Figure 12: Compressor Blade Design (2 Blades) - Zone 1 (Thin) Baseline Profile, $CL=1.0$ .....                                       | 10 |
| Figure 13: Compressor Blade Design (Single Blade) - Zone 1 (Thin) Baseline Profile, $CL=0.5$ .....                                   | 11 |
| Figure 14: Compressor Blade Design (2 Blades) - Zone 1 (Thin) Baseline Profile, $CL=0.5$ .....                                       | 11 |

## I. Introduction

The module provides an introduction about axial turbine and axial compressor blades design. For the turbine blade, a Bezier function is utilized for a camberline construction, while in compressor blade, the author uses NACA Camberline equation. A given set of data of base profile is used to plot the suction and pressure lines. Matlab code for plotting those blades are presented in the Appendix.

## II. Design Details

### A. Nomenclature

|            |                           |
|------------|---------------------------|
| $V_1$      | Inlet velocity            |
| $V_2$      | Outlet velocity           |
| $\alpha_1$ | Inlet angle               |
| $\alpha_2$ | Outlet angle              |
| $\phi$     | Auxiliary angle           |
| $\gamma$   | Stagger angle             |
| $c$        | Chord length              |
| $v$        | Camber line tangent angle |
| $x_c, y_c$ | Camber line coordinates   |
| $x_p, y_p$ | Pressure side coordinates |
| $x_s, y_s$ | Suction side coordinates  |
| $C_L^*$    | Lift coefficient          |

## B. Turbine Blade Design

### B.1. Camberline equation

a. Stagger angle based on  $\alpha_1$ ,  $\alpha_2$ ,  $C$  – blade chord

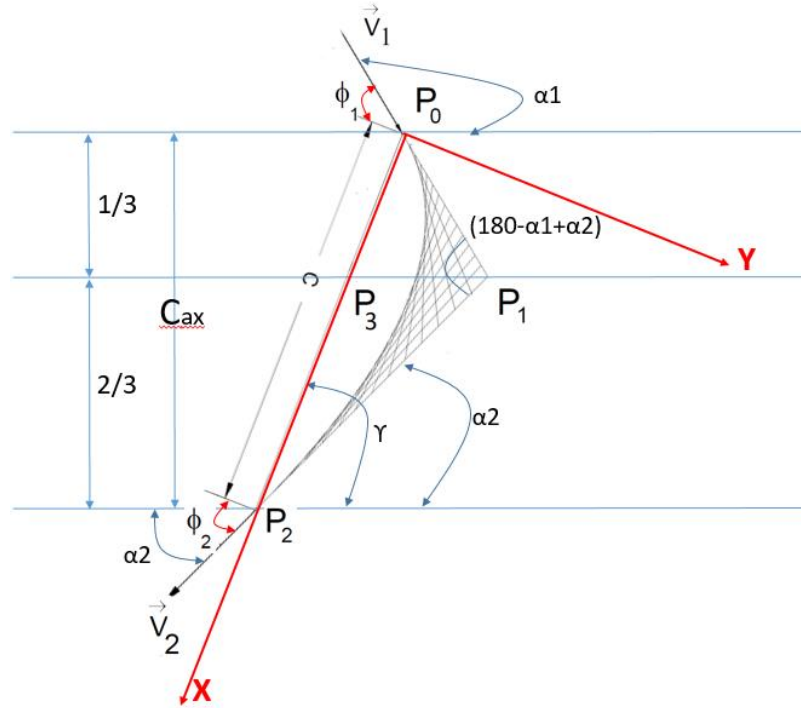


Figure 1: Parameters of the blade

Consider the triangle  $\Delta P_0 P_1 P_2$

Calculate stagger angle given inlet and outlet flow angles

$\Delta P_0 P_1 P_2$

$$\frac{P_0 P_2}{\sin(180 - \alpha_1 + \alpha_2)} = \frac{P_0 P_1}{\sin(\gamma - \alpha_2)}$$

$$\text{where: } P_0 P_2 = C, \quad P_0 P_1 = \frac{1}{3} * C_{ax} / \sin(180 - \alpha_1)$$

$$\rightarrow \frac{C_{ax}}{C} = \frac{3 * \sin(\alpha_1) * \sin(\gamma - \alpha_2)}{\sin(\alpha_1 - \alpha_2)}$$

$$\text{Additionally } \sin(\gamma) = \frac{C_{ax}}{C}$$

$$\rightarrow \tan(\gamma) = \frac{\sin(\alpha_2)}{-\frac{1}{3} * \frac{\sin(\alpha_1 - \alpha_2)}{\sin(\alpha_1)} + \cos(\alpha_2)}$$

b. *Bezier function*

Determine the coordinates of P0, P1 and P2

- P0(x\_p0,y\_p0),

$$x_{p0} = y_{p0} = 0$$

- P1(x\_p1,y\_p1),

Method 1:

Calculate the area of triangle P1P0P3 using

$$P1P0 = 1/3 * C / \sin(\alpha_1) * \sin(\gamma)$$

$$P1P3 = 1/3 * C / \sin(\alpha_1) * \sin(\pi - \alpha_1 - \gamma)$$

$$P0P3 = 1/3 * C$$

Then,

$$y_{p1} = 2 * \text{area} / (P0P3);$$

$$x_{p1} = y_{p1} / \tan(\pi - \alpha_1 - \gamma)$$

Method2:

$$\phi_1 = \alpha_1 + \gamma - \pi/2$$

$$\phi_2 = \frac{3\pi}{2} - \alpha_1 - \gamma$$

Then,

$$y_{p1} = C * (\cot(\phi_1) / (1 + \cot(\phi_1) / \cot(\phi_2))) ;$$

$$x_{p1} = C * (1 / (1 + \cot(\phi_1) / \cot(\phi_2))) ;$$

- P2(x\_p2,y\_p2)

$$x_{p2} = C, y_{p2} = 0$$

c. *Camberline equation*

$$x_{cam(i)} = (1 - \zeta)^2 * x_{p0} + 2 * (1 - \zeta) * \zeta * x_{p1} + \zeta^2 * x_{p2};$$

$$y_{cam(i)} = (1 - \zeta)^2 * y_{p0} + 2 * (1 - \zeta) * \zeta * y_{p1} + \zeta^2 * y_{p2};$$

where  $\zeta = [0..1]$

Camberline tangent angle

$$v_{cam(i)} = \text{atan} \left( \frac{-2 * (1 - \zeta) * y_{p0} + 2 * (1 - 2 * \zeta) * y_{p1} + 2 * \zeta * y_{p2}}{-2 * (1 - \zeta) * x_{p0} + 2 * (1 - 2 * \zeta) * x_{p1} + 2 * \zeta * x_{p2}} \right);$$



The diagram illustrates the geometric construction of a camberline and its associated surfaces (Suction surface and Pressure surface) using a series of circles. The camberline is the mean line, and the surfaces are defined by the upper and lower envelopes of the circles. Key points S, C, and P are marked on the camberline, and their coordinates  $(x_S, y_S)$ ,  $(x_C, y_C)$ , and  $(x_P, y_P)$  are indicated. The angle  $\theta$  is also shown.

A given base profile was superimposed on the camber line to obtain the profile for the suction side and pressure side. The equation used for the suction side was:

$$y_s = y_c + \left(\frac{t}{2}\right) \cos(v)$$

$$x_p = x_c + \left(\frac{t}{2}\right) \sin(v)$$

$$y_p = y_c - \left(\frac{t}{2}\right) \cos(v)$$

Given in Appendix A.2

## C. Compressor Blade Design

### C.1. Camberline equation

For designing NACA compressor blades, the camber line equation is first derived, which is

$$\frac{y_c(x)}{c} = -\frac{C_L^*}{4 * \pi} \left[ \left(1 - \frac{x_c}{c}\right) \ln \left(1 - \frac{x_c}{c}\right) + \frac{x_c}{c} \ln \left(\frac{x_c}{c}\right) \right]$$

And camber line angle is given by

$$\frac{dy(x)}{dx} = \tan(v) = \frac{C_L^*}{4 * \pi} \ln \left( \frac{1 - \frac{x_c}{c}}{\frac{x_c}{c}} \right)$$

### C.2. Suction and pressure side equation

The formula is given in section B.2. for both turbine and compressor designs.

### C.3. Base profile (thickness)

Given in Appendix A.2

### III. Results

#### A. Turbine Blade design

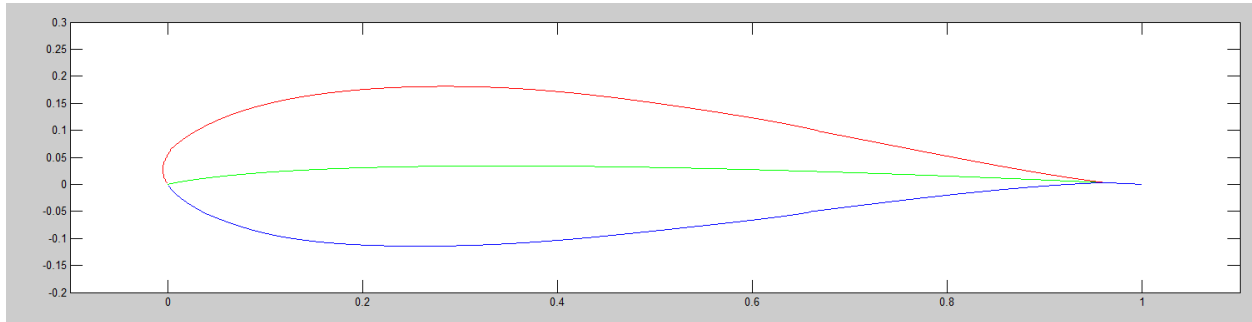


Figure 3: Turbine Blade Design (Single Blade) - Zone 3 (Thick) Baseline Profile,  $\alpha_1 = 45\text{deg}$ ,  $\alpha_2 = 20\text{deg}$

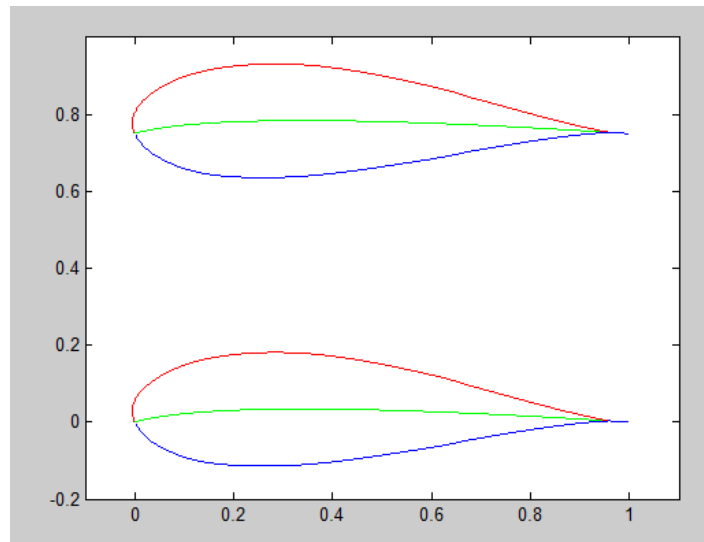


Figure 4: Turbine Blade Design (2 Blades) - Zone 3 (Thick) Baseline Profile,  $\alpha_1 = 45\text{deg}$ ,  $\alpha_2 = 20\text{deg}$

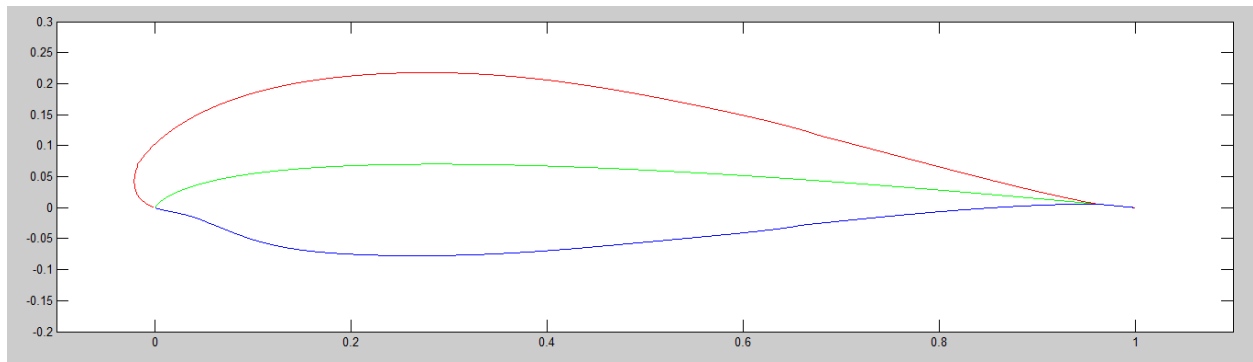


Figure 5: Turbine Blade Design (Single Blade) - Zone 3 (Thick) Baseline Profile,  $\alpha_1 = 90\text{deg}$ ,  $\alpha_2 = 20\text{deg}$

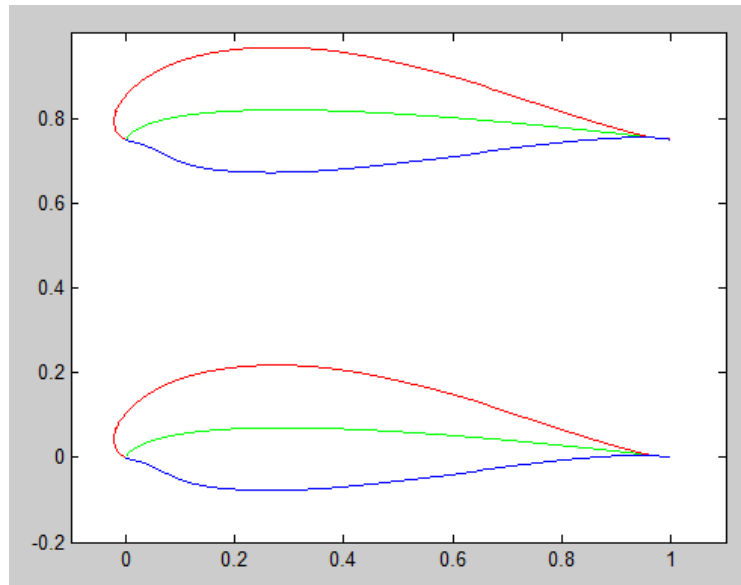


Figure 6: Turbine Blade Design (2 Blades) - Zone 3 (Thick) Baseline Profile,  $\alpha_1 = 90\text{deg}$ ,  $\alpha_2 = 20\text{deg}$

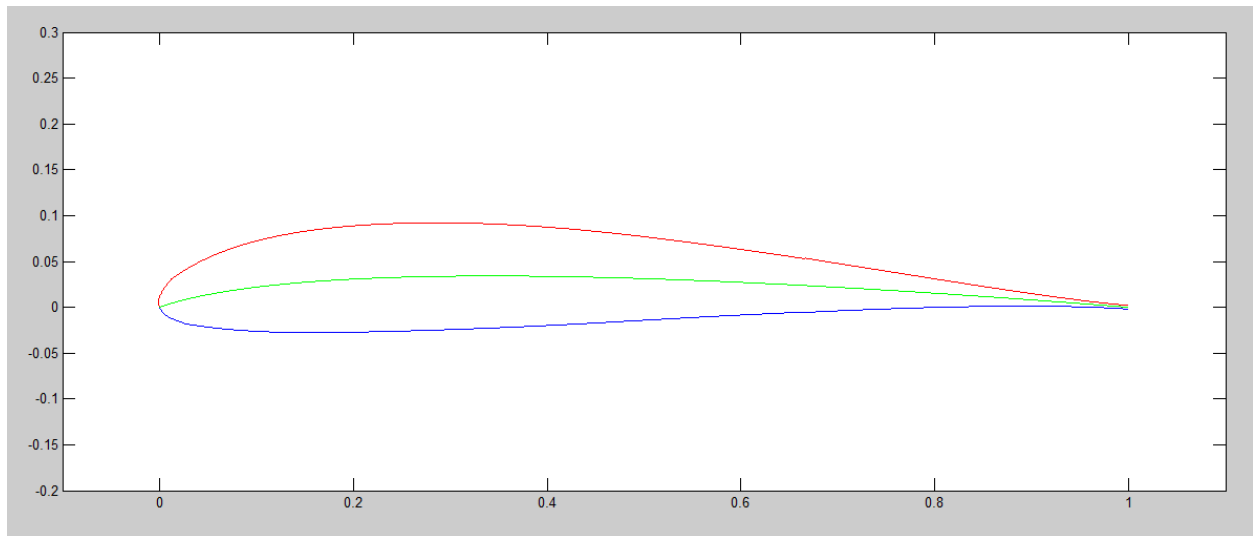


Figure 7: Turbine Blade Design (Single Blade) - Zone 1 (Thin) Baseline Profile,  $\alpha_1 = 45\text{deg}$ ,  $\alpha_2 = 20\text{deg}$

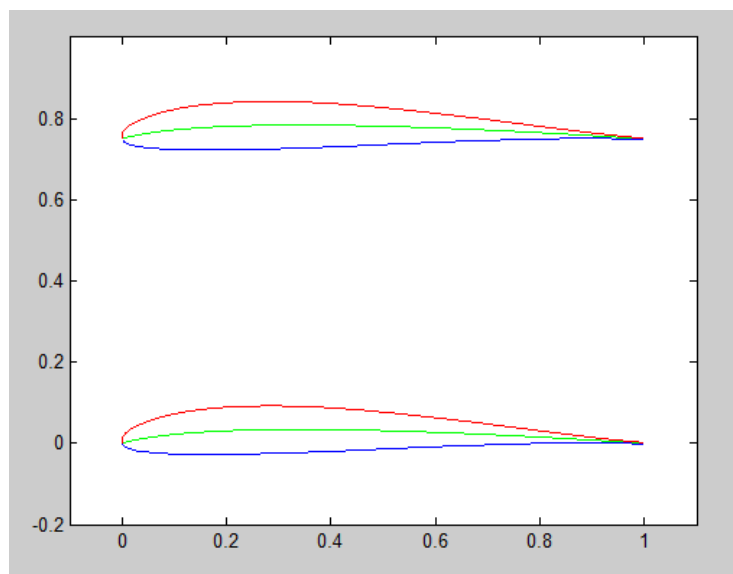


Figure 8: Turbine Blade Design (2 Blades) - Zone 1 (Thin) Baseline Profile,  $\alpha_1 = 45\text{deg}$ ,  $\alpha_2 = 20\text{deg}$

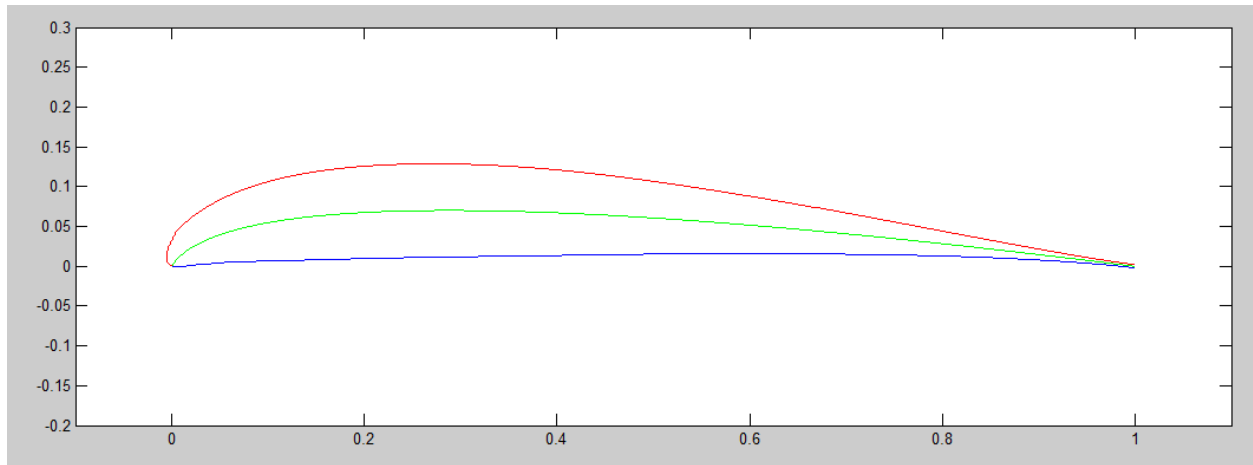


Figure 9: Turbine Blade Design (Single Blade) - Zone 1 (Thin) Baseline Profile,  $\alpha_1 = 90\text{deg}$ ,  $\alpha_2 = 20\text{deg}$

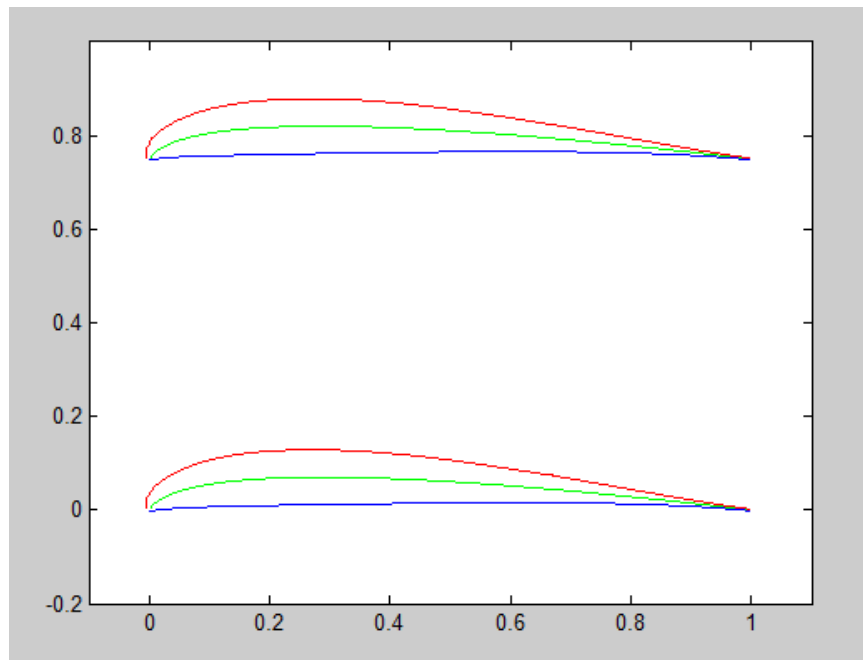


Figure 10: Turbine Blade Design (2 Blades) - Zone 1 (Thin) Baseline Profile,  $\alpha_1 = 90\text{deg}$ ,  $\alpha_2 = 20\text{deg}$

## B. Compressor Blade design

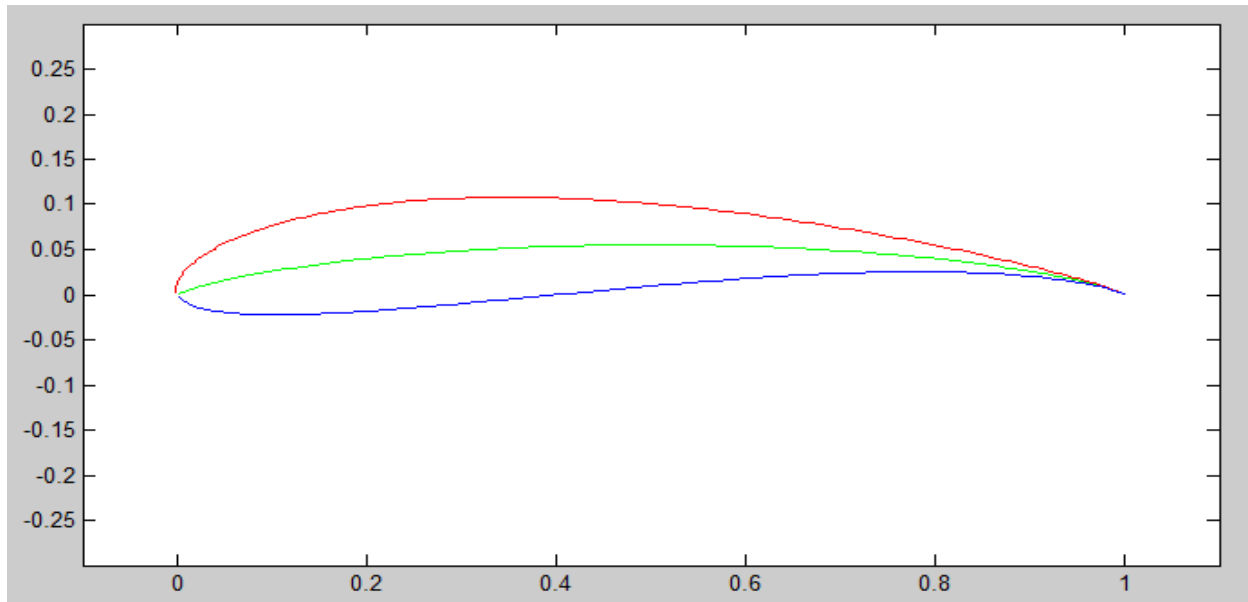


Figure 11: Compressor Blade Design (Single Blade) - Zone 1 (Thin) Baseline Profile,  $CL=1.0$

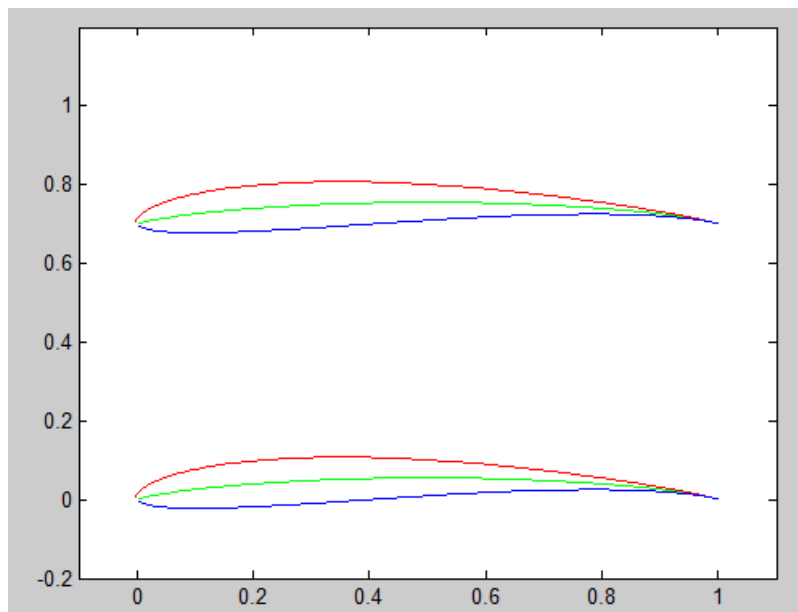


Figure 12: Compressor Blade Design (2 Blades) - Zone 1 (Thin) Baseline Profile,  $CL=1.0$

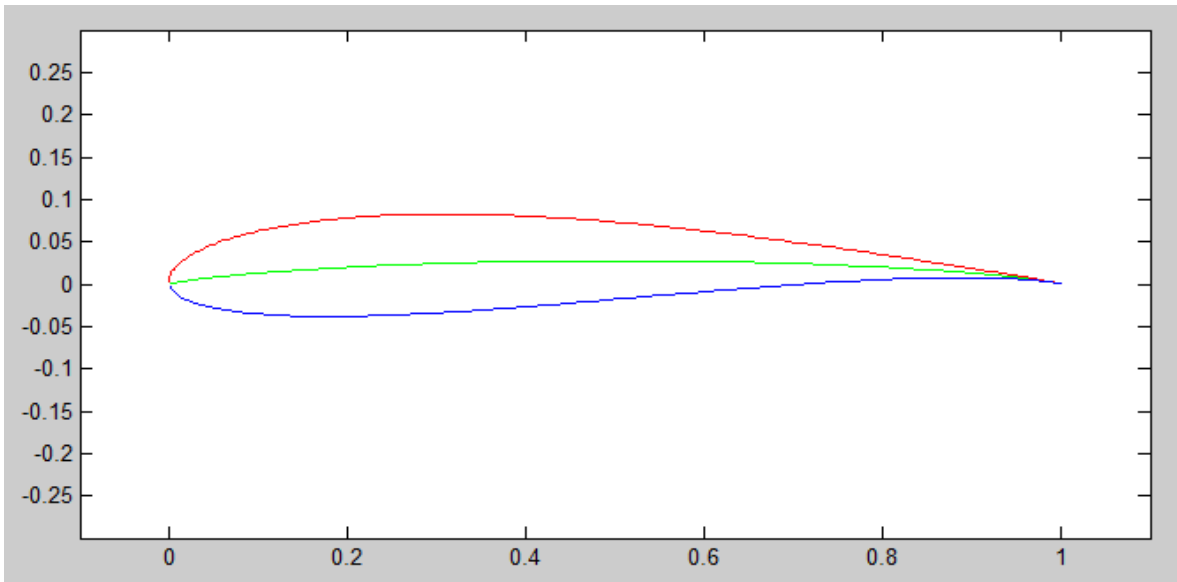


Figure 13: Compressor Blade Design (Single Blade) - Zone 1 (Thin) Baseline Profile,  $CL=0.5$

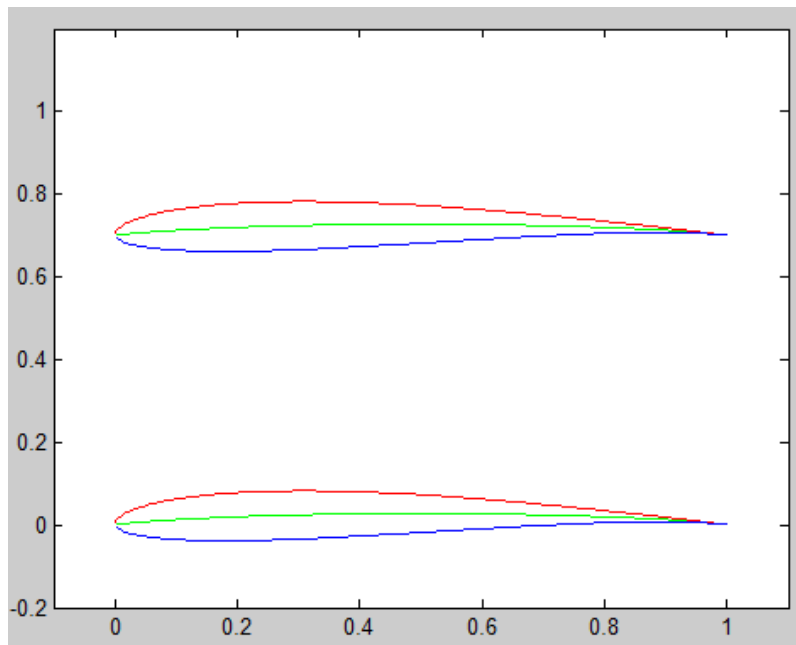


Figure 14: Compressor Blade Design (2 Blades) - Zone 1 (Thin) Baseline Profile,  $CL=0.5$



#### IV. Discussion and conclusion

- In turbine, increasing deflection, increased the curvature of the camber line.
- In compressor, it was noted that increasing  $C_L^*$  increased the curvature of the camber line.  
Increased curvature can produce more lift

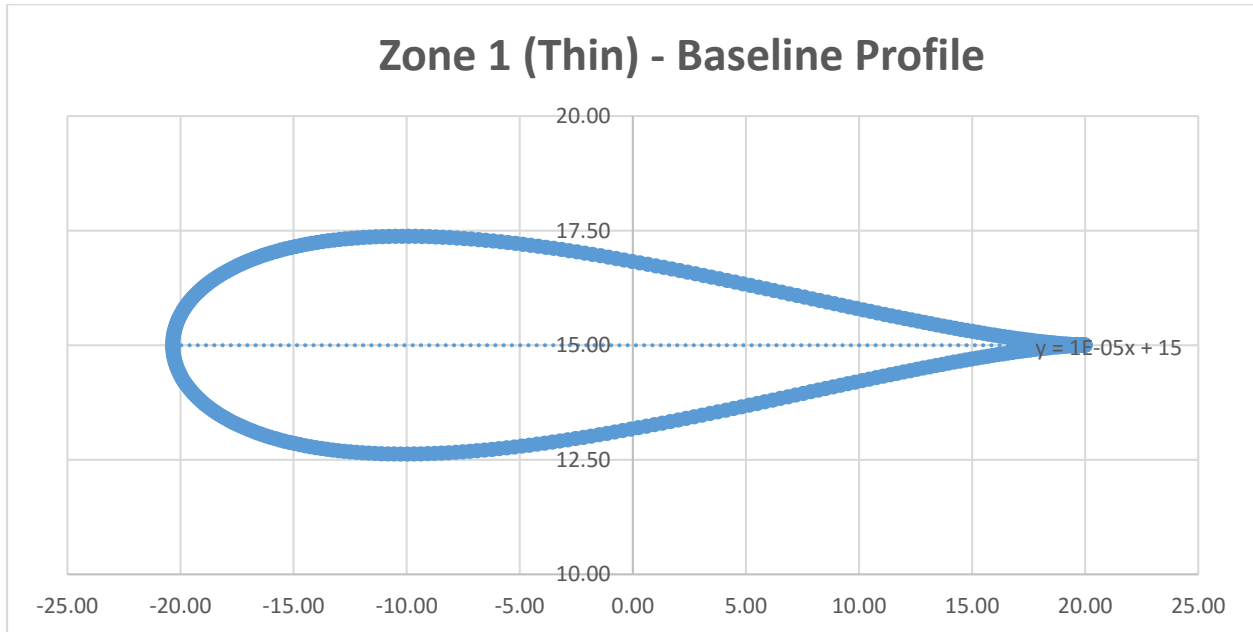
Thus, this module provided an introduction to students in designing of simple blades for turbines and compressors.

#### REFERENCES

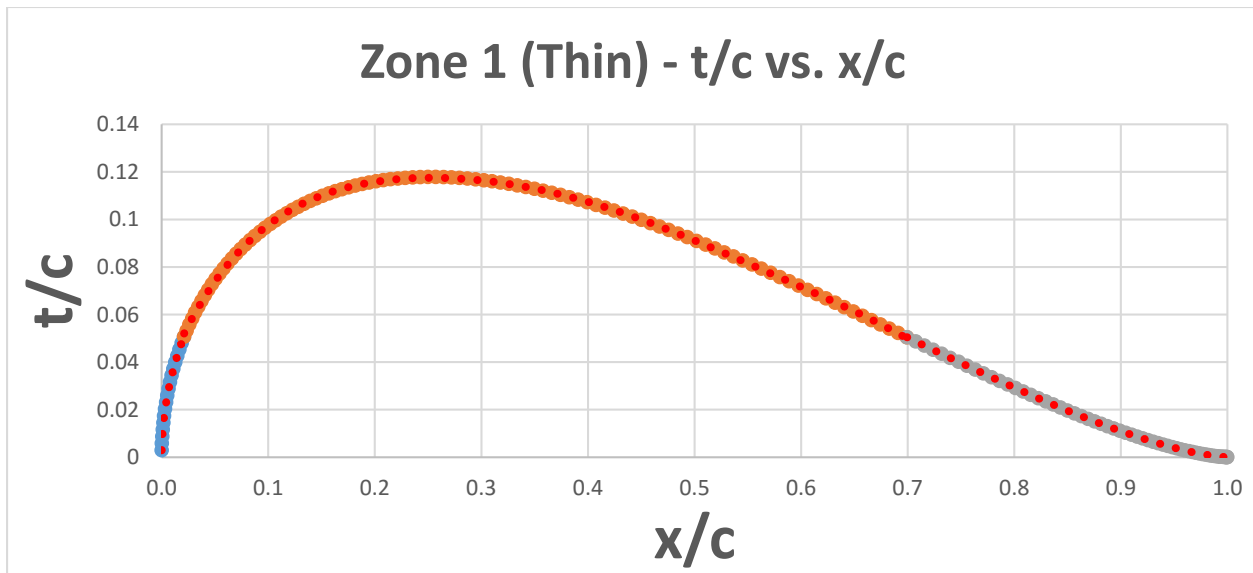
[1] Schobeiri, Meinhard T., *Turbomachinery Flow Physics and Dynamic Performance – Chapter 10*, 2<sup>nd</sup> Ed. Springer 2012. Print & Online

## APPENDIX

### A.1 Zone 1 Base-line profile

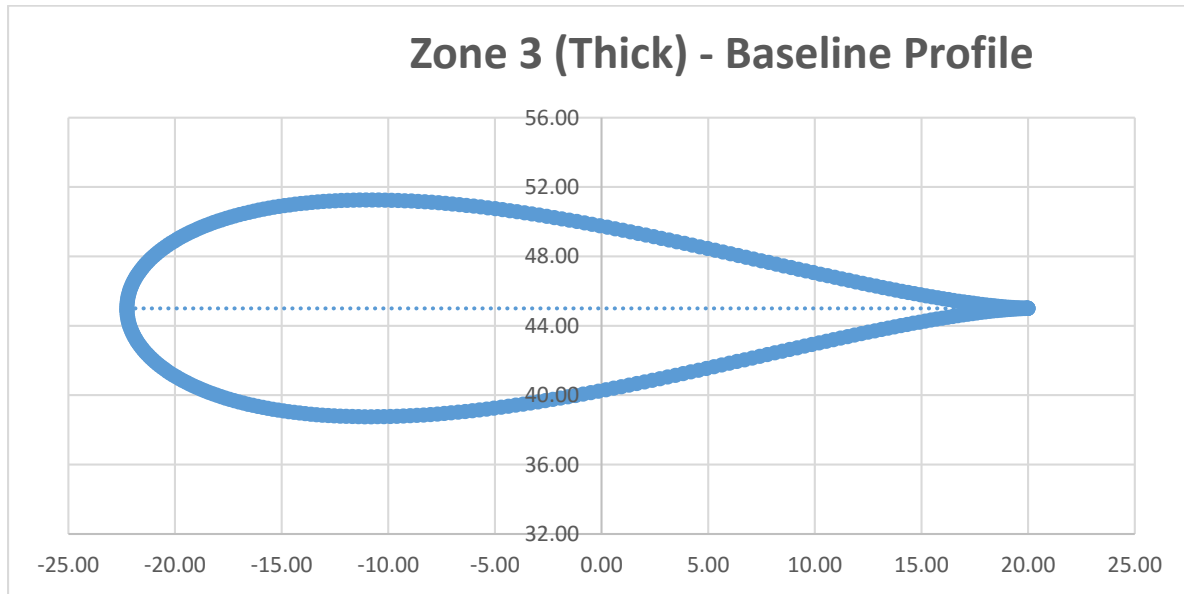


### A.2 Zone 1- Baseline profile: $t/c$ vs. $x/c$

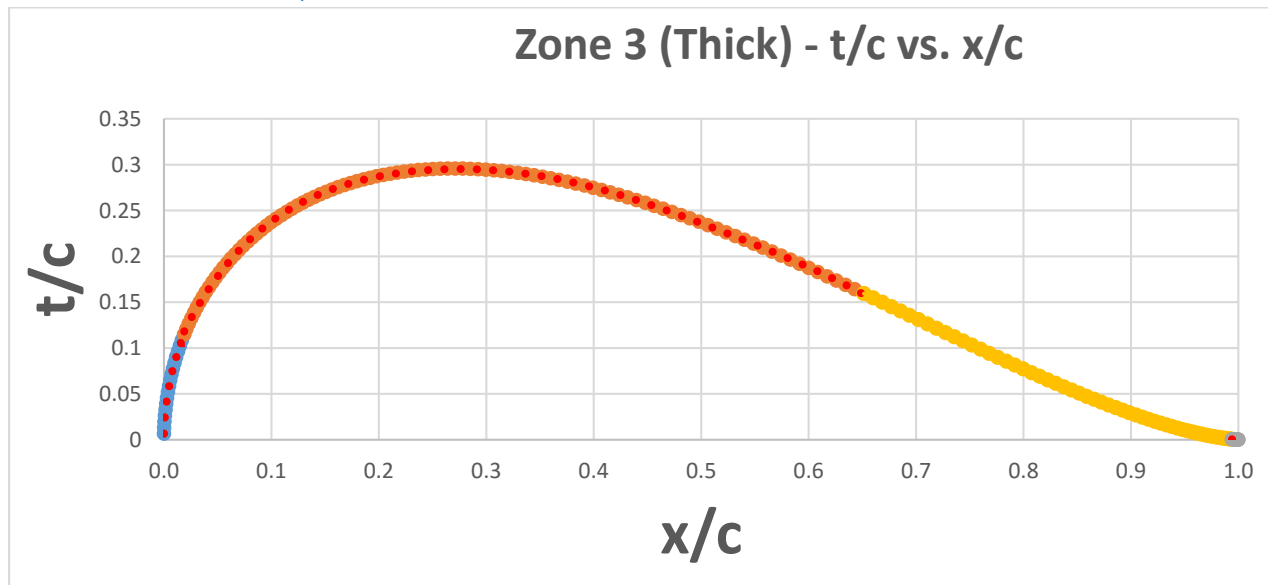


| $x/c$                         | $t/c$ as function of $x/c$                                                         |
|-------------------------------|------------------------------------------------------------------------------------|
| $0 \sim 190/10,000$           | $y = 0.3419x^{0.4929}$                                                             |
| $191/10,000 \sim 6550/10,000$ | $y = -15.631x^6 + 38.563x^5 - 38.22x^4 + 19.934x^3 - 6.2802x^2 + 1.1333x + 0.0307$ |
| $6551/10,000 \sim 1$          | $y = 75.656x^6 - 375.15x^5 + 774.1x^4 - 850.22x^3 + 524.07x^2 - 172.08x + 23.628$  |

## B.1 Zone 3 Base-line profile



## B.2 Zone 3- Baseline profile: $t/c$ vs. $x/c$



| $x/c$                          | $t/c$ as function of $x/c$                                                          |
|--------------------------------|-------------------------------------------------------------------------------------|
| $0 \sim 190/10,000$            | $y = 0.8232x^{0.4941}$                                                              |
| $191/10,000 \sim 6550/10,000$  | $y = -56.476x^6 + 129.12x^5 - 118.24x^4 + 56.666x^3 - 16.456x^2 + 2.8703x + 0.0696$ |
| $6551/10,000 \sim 9955/10,000$ | $y = 65.209x^6 - 309.36x^5 + 610.82x^4 - 641.17x^3 + 376.88x^2 - 118.1x + 15.713$   |
| $9956/10,000 \sim 1$           | $y = 7.1679x^2 - 14.367x + 7.1992$                                                  |

```
% MEEN-646: Module 2
% Design of Subsonic Compressor Blade M2-C
```

```
% Objective:
% Develop a design software that enables you to generate subsonic compressor blades
% using NACA Camberline Equation
% Given Parameters
% - Base profile
% - Blade chord C
% - Lift coefficient
%
% Instruction:
% Input: C, s, CL, iZone selection
```

```
clc;
```

```
ln = @(x)(log(x));
```

```
%input:
iZone=3; % input 1, 2, 3 (1 for thinnest profile and 3 for thickest profile)
```

```
%input parameters
CL = 1.0; %lift coefficient
C = 1.0; %blade chord
s = 0.5; %spacing
%n_iter = 10000; %number of iteration
```

```
n_iter = 10000;
```

```
n_iter_b1 = 190;
n_iter_b2 = 6650;
n_iter_b3 = 9955; %for zone3 only
x_cam1=n_iter_b1/n_iter*C;
x_cam2=n_iter_b2/n_iter*C;
x_cam3=n_iter_b3/n_iter*C;%for zone3 only
```

```
%NACA Camberline Equation
for i=1:1:n_iter
    x_cam(i) = i/n_iter*C;
    y_cam(i) = -C*CL/(4*pi)*((1-i/n_iter)*ln(1-i/n_iter)+i/n_iter*ln(i/n_iter));
    %camber line tangent angle
    v_cam(i)=atan(CL/(4*pi)*ln((1-i/n_iter)/(i/n_iter)));
```

```
%for the 2nd camberline
y_cam1(i) = y_cam(i)+s;
end
```

```
%blade thickness
if iZone == 1
    for i=1:1:n_iter_b1
        x(i)=i/n_iter;
        t(i)=C*(0.3419*x(i)^0.4929);%zone1
    end
```

```
for i=(n_iter_b1+1):1:n_iter_b2
    x(i)=i/n_iter;
    t(i)=C*(-15.631*x(i)^6 + 38.563*x(i)^5 - 38.22*x(i)^4 + 19.934*x(i)^3 - 6.2802*x(i)^2 + 1.1333*x(i) + 0.0307);%zone1
end
```

```

for i=(n_iter_b2+1):1:n_iter
    x(i)=i/n_iter;
    t(i)=C*(75.656*x(i)^6 - 375.15*x(i)^5 + 774.1*x(i)^4 - 850.22*x(i)^3 + 524.07*x(i)^2 - 172.08*x(i) + 23.628);%zone1
end

elseif iZone == 2
    for i=1:1:n_iter_b1
        x(i)=i/n_iter;
        t(i)=C*(0.6128*x(i)^0.4937);%zone2
    end

    for i=(n_iter_b1+1):1:n_iter_b2
        x(i)=i/n_iter;
        t(i)=C*(-35.559*x(i)^6 + 83.97*x(i)^5 - 79.529*x(i)^4 + 39.519*x(i)^3 - 11.876*x(i)^2 + 2.0934*x(i) + 0.0531);%zone2
    end

    for i=(n_iter_b2+1):1:n_iter
        x(i)=i/n_iter;
        t(i)=C*(93.702*x(i)^6 - 455.68*x(i)^5 + 921.82*x(i)^4 - 991.96*x(i)^3 + 598.52*x(i)^2 - 192.34*x(i) + 25.931);%zone2
    end

    if t(i)<0
        t(i)=0;
    end

elseif iZone == 3
    for i=1:1:n_iter_b1
        x(i)=i/n_iter;
        t(i)=C*(0.8232*x(i)^0.4941);%zone3
    end

    for i=(n_iter_b1+1):1:n_iter_b2
        x(i)=i/n_iter;
        t(i)=C*(-56.476*x(i)^6 + 129.12*x(i)^5 - 118.24*x(i)^4 + 56.666*x(i)^3 - 16.456*x(i)^2 + 2.8703*x(i) + 0.0696);%zone3
    end

    for i=(n_iter_b2+1):1:n_iter_b3
        x(i)=i/n_iter;
        t(i)=C*(65.209*x(i)^6 - 309.36*x(i)^5 + 610.82*x(i)^4 - 641.17*x(i)^3 + 376.88*x(i)^2 - 118.1*x(i) + 15.713);%zone3
    end

    for i=(n_iter_b2+1):1:n_iter
        x(i)=i/n_iter;
        t(i)=C*(7.1679*x(i)^2 - 14.367*x(i) + 7.1992);%zone3
    end

    if t(i)<0
        t(i)=0;
    end

end

% suction side coordinate
for i=1:1:n_iter
    x_S(i) = x_cam(i) - (t(i)/2)*sin(v_cam(i));
    y_S(i) = y_cam(i) + (t(i)/2)*cos(v_cam(i));
    y_S1(i) = y_S(i) + s;
end

```

```
%pressure side coordinate
for i=1:1:n_iter
    x_P(i) = x_cam(i) + (t(i)/2)*sin(v_cam(i));
    y_P(i) = y_cam(i) - (t(i)/2)*cos(v_cam(i));
    y_P1(i) = y_P(i) + s;
end
```

```
%plot the blade
figure(1);
plot(x_cam,y_cam,'g')
hold on
plot(x_S,y_S,'r')
hold on
plot(x_P,y_P,'b')
hold on
%for i=1:(n_iter-1)
%   th = 0:pi/50:2*pi;
%   xunit = t(i)/2 * cos(th) + x_cam(i);
%   yunit = t(i)/2 * sin(th) + y_cam(i);
%   h = plot(xunit, yunit,'r');
%   hold on
%end
```

```
plot(x_cam,y_cam1,'g')
hold on
plot(x_S,y_S1,'r')
hold on
plot(x_P,y_P1,'b')
hold on
%for i=500:500:(n_iter-500)
%   th = 0:pi/50:2*pi;
%   xunit = t(i)/2 * cos(th) + x_cam(i);
%   yunit = t(i)/2 * sin(th) + y_cam1(i);
%   h = plot(xunit, yunit,'r');
%   hold on
%end
```

```
xlim([-1 1.1])
%ylim([-0.2 0.8])
ylim([-0.2 0.2])
```

```

%           MEEN-646: Module 2
%           Design of Subsonic Turbine Blade M2-T

% Objective:
%   Develop a design software that enables you to generate subsonic turbine blades
%
% Given Parameters
%   - Generate a family of profile (alpha1=90, alpha2=160), (alpha1=45, alpha2=160)
%   - Blade chord C
%
% Instruction:
%   Input: alpha1, alpha2 & iZone

clc;

iZone=1;

%input:
alpha1 = 45; %in degree
alpha2 = 20; %in degree
%convert to rad
alpha1 = (alpha1/180)*pi;
alpha2 = (alpha2/180)*pi;

C = 1.00; %chord
s = 0.75; %spacing
n_iter = 10000; %number of iteration for camberline

n_iter_b1 = 190;
n_iter_b2 = 6650;
n_iter_b3 = 9955; %for zone3 only
x_cam1=n_iter_b1/n_iter*C;
x_cam2=n_iter_b2/n_iter*C;
x_cam3=n_iter_b3/n_iter*C;%for zone3 only

%cascade stagger angle
gamma = atan(sin(alpha2)/(-1/3*sin(alpha1-alpha2)/sin(alpha1)+cos(alpha2)));
C_ax = C*sin(gamma);

%define camber line equation:
x_p0 = 0; y_p0 = 0;
x_p2 = C; y_p2 = 0;

%determine P1 coordinates by consider triangle P0P1P3
%a_1 = 1/3*C/sin(alpha1)*sin(gamma); %P1P0 length
%b_1 = 1/3*C/sin(alpha1)*sin(pi-alpha1-gamma); %P1P3 length
%c_1 = 1/3*C; %P0P3 length
%p = (a_1+b_1+c_1)/2;
%area = sqrt(p*(p-a_1)*(p-b_1)*(p-c_1));

%y_p1 = 2*area/(1/3*C);
%x_p1 = y_p1/tan(pi-alpha1-gamma);

%determine P1 coordinates, formula given in the book (equation 10.40)
phi1=pi/2-alpha1+gamma;
phi2=pi/2+alpha2-gamma;
y_p1 = C*(cot(phi1)/(1+cot(phi1)/cot(phi2)));
x_p1 = C*(1/(1+cot(phi1)/cot(phi2)));

```

```
%Bezier Curve
```

```
for i=1:1:n_iter
    zeta(i) = i/n_iter;
    x_cam(i) = (1-zeta(i))^2*x_p0 + 2*(1-zeta(i))*zeta(i)*x_p1 + zeta(i)^2*x_p2;
    y_cam(i) = (1-zeta(i))^2*y_p0 + 2*(1-zeta(i))*zeta(i)*y_p1 + zeta(i)^2*y_p2;
```

```
%camber line tangent angle
```

```
v_cam(i)=atan((-2*(1-zeta(i))*y_p0 + 2*(1-2*zeta(i))*y_p1 + 2*zeta(i)*y_p2)/(-2*(1-zeta(i))*x_p0 + 2*(1-2*zeta(i))*x_p1 + 2*zeta(i)*x_p2));
```

```
%for the 2nd camberline
```

```
y_cam1(i) = y_cam(i)+s;
```

```
end
```

```
%blade thickness
```

```
if iZone == 1
```

```
    for i=1:1:n_iter
        x(i)=x_cam(i)/C;
        if (x_cam(i)<x_cam1)
            t(i)=C*(0.3419*x(i)^0.4929);%zone1
        elseif (x_cam(i)>x_cam1) && (x_cam(i)<x_cam2)
            t(i)=C*(-15.631*x(i)^6 + 38.563*x(i)^5 - 38.22*x(i)^4 + 19.934*x(i)^3 - 6.2802*x(i)^2 + 1.1333*x(i) + 0.0307);%zone1
        elseif (x_cam(i)>x_cam2)
            t(i)=C*(75.656*x(i)^6 - 375.15*x(i)^5 + 774.1*x(i)^4 - 850.22*x(i)^3 + 524.07*x(i)^2 - 172.08*x(i) + 23.628);%zone1
        end
    end
```

```
elseif iZone == 2
```

```
    for i=1:1:n_iter
        x(i)=x_cam(i)/C;
        if (x_cam(i)<x_cam1)
            t(i)=C*(0.6128*x(i)^0.4937);%zone2
        elseif (x_cam(i)>x_cam1) && (x_cam(i)<x_cam2)
            t(i)=C*(-35.559*x(i)^6 + 83.97*x(i)^5 - 79.529*x(i)^4 + 39.519*x(i)^3 - 11.876*x(i)^2 + 2.0934*x(i) + 0.0531);%zone2
        elseif (x_cam(i)>x_cam2)
            t(i)=C*(93.702*x(i)^6 - 455.68*x(i)^5 + 921.82*x(i)^4 - 991.96*x(i)^3 + 598.52*x(i)^2 - 192.34*x(i) + 25.931);%zone2
        end
```

```
        if t(i)<0
```

```
            t(i)=0;
```

```
        end
```

```
    end
```

```
elseif iZone == 3
```

```
    for i=1:1:n_iter
        x(i)=x_cam(i)/C;
        if (x_cam(i)<x_cam1)
            t(i)=C*(0.8232*x(i)^0.4941);%zone3
        elseif (x_cam(i)>x_cam1) && (x_cam(i)<x_cam2)
            t(i)=C*(-56.476*x(i)^6 + 129.12*x(i)^5 - 118.24*x(i)^4 + 56.666*x(i)^3 - 16.456*x(i)^2 + 2.8703*x(i) + 0.0696);%zone3
        elseif (x_cam(i)>x_cam2) && (x_cam(i)<x_cam3)
            t(i)=C*(65.209*x(i)^6 - 309.36*x(i)^5 + 610.82*x(i)^4 - 641.17*x(i)^3 + 376.88*x(i)^2 - 118.1*x(i) + 15.713);%zone3
        elseif (x_cam(i)>x_cam3)
            t(i)=C*(7.1679*x(i)^2 - 14.367*x(i) + 7.1992);%zone3
        end
```

```
        if t(i)<0
```

```
            t(i)=0;
```

```
        end
```

```
    end
```

```
end
```



```
%suction side coordinate
```

```
for i=1:1:n_iter
    x_S(i) = x_cam(i) - (t(i)/2)*sin(v_cam(i));
    y_S(i) = y_cam(i) + (t(i)/2)*cos(v_cam(i));
    y_S1(i) = y_S(i) + s;
end
```

```
%pressure side coordinate
```

```
for i=1:1:n_iter
    x_P(i) = x_cam(i) + (t(i)/2)*sin(v_cam(i));
    %x_P_test(i) = x_cam(i) - (t(i)/2)*sin(v_cam(i));
    y_P(i) = y_cam(i) - (t(i)/2)*cos(v_cam(i));
    y_P1(i) = y_P(i) + s;
end
```

```
%plot the blade
```

```
figure(1);
plot(x_cam,y_cam,'g')
hold on
plot(x_S,y_S,'r')
hold on
plot(x_P,y_P,'b')
hold on
%plot(x_P_test,y_P,'y')
%hold on
%for i=1:1000:(n_iter)
%   th = 0:pi/50:2*pi;
%   xunit = t(i)/2 * cos(th) + x_cam(i);
%   yunit = t(i)/2 * sin(th) + y_cam(i);
%   h = plot(xunit, yunit,'r');
%   hold on
%end
```

```
plot(x_cam,y_cam1,'g')
hold on
plot(x_S,y_S1,'r')
hold on
plot(x_P,y_P1,'b')
hold on
%plot(x_P_test,y_P1,'y')
%hold on
%for i=1000:1000:(n_iter-1000)
%   th = 0:pi/50:2*pi;
%   xunit = t(i)/2 * cos(th) + x_cam(i);
%   yunit = t(i)/2 * sin(th) + y_cam1(i);
%   h = plot(xunit, yunit,'r');
%   hold on
%end
```

```
xlim([-0.1 1.1])
%ylim([-0.2 1])
ylim([-0.2 0.3])
```