

Problem Set 8 – Solutions

11/07/21

Problem 1 Yoyo (1)

String is wrapped around yoyo with radius $R \Rightarrow v = \dot{x} = \omega R \Rightarrow \omega = \frac{\dot{x}}{R}$ (1)

(Note: (1) corresponds to no-slip condition of rolling disk with radius R)

Total kinetic energy of yoyo, using (1) and $I = \frac{1}{2}mR^2$:

$$\begin{aligned} T &= \frac{1}{2}m\dot{x}^2 + \frac{1}{2}I\omega^2 = \frac{1}{2}m\dot{x}^2 + \frac{1}{2}\left(\frac{1}{2}mR^2\right)\left(\frac{\dot{x}}{R}\right)^2 \\ &= \frac{1}{2}m\dot{x}^2 + \frac{1}{4}m\dot{x}^2 = \frac{3}{4}m\dot{x}^2 \end{aligned} \quad (2)$$

Lagrange function for adapted coordinate x :

$$\mathcal{L}(x, \dot{x}) = T - U = \frac{3}{4}m\dot{x}^2 + mgx \quad (3)$$

Euler-Lagrange equation: $\frac{d}{dt} \frac{\partial \mathcal{L}}{\partial \dot{x}} = \frac{\partial \mathcal{L}}{\partial x} \Rightarrow \frac{3}{2}m\ddot{x} = mg \Rightarrow \ddot{x} = \frac{2}{3}g$ (4)

Downward acceleration is more slowly than free-fall acceleration because some of the gravitational force is used to accelerate the rotation of the yoyo.

Problem 2 Yoyo (2)

- a) As stated in the hint in the question, the Lagrange function $\mathcal{L}(z, \dot{z}, \dot{\phi})$ of the yoyo without the constraint is given by (keeping in mind that z is directed vertically downward)

$$\mathcal{L}(z, \dot{z}, \dot{\phi}) = T - U = \frac{1}{2} m \dot{z}^2 + \frac{1}{2} I \dot{\phi}^2 + m g z . \quad (1)$$

- b) The constraint is given by the fact that the yoyo is connected to the string, which itself is suspended at time-dependent height $z_0(t)$. This constraint can be expressed in the following way. Assume that the string has constant total length L . One portion of the string is wrapped around the yoyo of radius R . The length of this portion is $(-\phi R)$ (see figure in question). The remaining portion of the string is equal to the vertical distance $z - z_0$ between the yoyo and the suspension point z_0 . Thus

$$-\phi R + z - z_0(t) = L . \quad (2)$$

Eq. (2) corresponds to a constraint that couples the variables z and ϕ . The constraint is time-dependent because of the time-dependence of $z_0(t)$. The corresponding constraint function is

$$c(z, \phi; t) = z - z_0(t) - \phi R - L = 0 . \quad (3)$$

c) z - equation: $\frac{d}{dt} \frac{\partial \mathcal{L}}{\partial \dot{z}} - \frac{\partial \mathcal{L}}{\partial z} = \lambda \frac{\partial c}{\partial z}$ (4)

with $\mathcal{L}(z, \dot{z}, \dot{\phi})$ from (1) and $c(z, \phi; t)$ from (3). This gives: $m\ddot{z} - mg = \lambda$. (5)

ϕ - equation: $\frac{d}{dt} \frac{\partial \mathcal{L}}{\partial \dot{\phi}} - \frac{\partial \mathcal{L}}{\partial \phi} = \lambda \frac{\partial c}{\partial \phi}$. Using (1) and (3) this gives: $I\ddot{\phi} = -\lambda R$. (6)

$I\ddot{\phi} = \Gamma$ where Γ is the torque $\Rightarrow \lambda = -\frac{\Gamma}{R}$

d) Combining (5) and (6) to eliminate λ gives: $m\ddot{z} - mg = -\frac{I}{R}\ddot{\phi}$ (7)

Using the constraint in (2) to express ϕ in terms of z :

$$(2) \Rightarrow \phi = \frac{1}{R}(z - z_0 - L) \Rightarrow \ddot{\phi} = \frac{1}{R}(\ddot{z} - \ddot{z}_0) \quad (8)$$

$$\text{Inserting (8) in (7)} \Rightarrow m\ddot{z} - mg = -\frac{I}{R^2}(\ddot{z} - \ddot{z}_0) . \quad (9)$$

$$\text{Using } I = \frac{1}{2}mR^2 \Rightarrow m\ddot{z} - mg = -\frac{1}{2}m(\ddot{z} - \ddot{z}_0) = -\frac{1}{2}m\ddot{z} + \frac{1}{2}m\ddot{z}_0$$

$$\Rightarrow \ddot{z} = \frac{1}{3}\ddot{z}_0 + \frac{2}{3}g \quad (10)$$

e) Without external vertical driving, i.e., $z_0 = 0$, (10) reproduces the solution (4) of Problem 1.

Remark (not required in homework):

General solution of (10):

$$z(t) = \frac{1}{3}z_0(t) + \frac{1}{3}gt^2 + at + b \quad \text{with constants } a, b .$$

For example, $z_0(t) = A \sin(\omega_0 t)$ sinusoidal driving with frequency ω_0 and amplitude A , and assuming $a = b = 0$:

$$z(t) = \frac{A}{3} \sin(\omega_0 t) + \frac{1}{3}gt^2$$

= overall downward acceleration $2g/3$ with sinusoidal modulation of amplitude $A/3$.

Problem 7.35 (page 287) "Figure 7.16 is a bird's-eye view ... "

As pointed out in the lecture notes, the kinetic energy of a bead is

$$T = \frac{1}{2} m \left(\frac{d\vec{r}}{dt} \cdot \frac{d\vec{r}}{dt} \right)$$

where $\vec{r}(t)$ is the time-dependent position vector relative to a stationary origin O .

Choose the origin O to be stationary point A shown in Figure 7.16 in the question.

(The origin O cannot be chosen as the center C of the hoop because C is moving!)

Thus the position vector \vec{r} relative to $O = A$ can be decomposed as $\vec{r} = \vec{c} + \vec{r}'$ where

\vec{c} is the vector from A to C (the center of the hoop) and \vec{r}' is the vector from C to \vec{r} .

Thus:

$$\frac{d\vec{r}}{dt} = \frac{d\vec{c}}{dt} + \frac{d\vec{r}'}{dt} . \quad (1)$$

The center C of the hoop is turning about A with angular velocity ω . So, the velocity of C is $\vec{v}_C = \frac{d\vec{c}}{dt} = R\omega \hat{v}_C$ where R is the radius of the hoop (= half the diameter AB).

For *constant* angle ϕ , the bead is rotating about C with angular velocity ω (following the rotation of the hoop). Thus, for non-constant angle ϕ the bead is rotating about C with total angular velocity $\omega + \dot{\phi}$. The velocity of the vector \vec{r}' from C to \vec{r} is thus

$$\vec{v}' = \frac{d\vec{r}'}{dt} = R(\omega + \dot{\phi}) \hat{v}' .$$

The angle between \hat{v}_C and \hat{v}' is equal to ϕ (see figure below).

Thus: $\hat{v}_C \cdot \hat{v}' = \cos(\phi)$.

Using the above results for $\bar{\mathbf{v}}_C$ and $\bar{\mathbf{v}}'$ one finds for the kinetic energy of the bead:

$$\begin{aligned}
 T &= \frac{1}{2}m \left(\frac{d\bar{\mathbf{r}}}{dt} \cdot \frac{d\bar{\mathbf{r}}}{dt} \right) = \frac{1}{2}m \left(\frac{d\bar{\mathbf{c}}}{dt} + \frac{d\bar{\mathbf{r}}'}{dt} \right) \cdot \left(\frac{d\bar{\mathbf{c}}}{dt} + \frac{d\bar{\mathbf{r}}'}{dt} \right) \\
 &= \frac{1}{2}m (\bar{\mathbf{v}}_C + \bar{\mathbf{v}}') \cdot (\bar{\mathbf{v}}_C + \bar{\mathbf{v}}') = \frac{1}{2}m (v_C^2 + v'^2 + 2\bar{\mathbf{v}}_C \cdot \bar{\mathbf{v}}') \\
 &= \frac{1}{2}m \left[R^2\omega^2 + R^2(\omega + \dot{\phi})^2 + 2R^2\omega(\omega + \dot{\phi})\cos(\phi) \right] \\
 &= \frac{1}{2}mR^2 \left[\omega^2 + (\omega + \dot{\phi})^2 + 2\omega(\omega + \dot{\phi})\cos(\phi) \right]
 \end{aligned} \tag{2}$$

Since there is no potential energy, the Lagrange function for the adapted coordinate ϕ is given by the last line in (2):

$$\mathcal{L}(\phi, \dot{\phi}) = T = \frac{1}{2}mR^2 \left[\omega^2 + (\omega + \dot{\phi})^2 + 2\omega(\omega + \dot{\phi})\cos(\phi) \right] \tag{3}$$

Euler-Lagrange equation: $\frac{d}{dt} \frac{\partial \mathcal{L}}{\partial \dot{\phi}} = \frac{\partial \mathcal{L}}{\partial \phi}$

$$\frac{\partial \mathcal{L}}{\partial \dot{\phi}} = \frac{1}{2}mR^2 \left[2(\omega + \dot{\phi}) + 2\omega\cos(\phi) \right]$$

$$\frac{d}{dt} \frac{\partial \mathcal{L}}{\partial \dot{\phi}} = \frac{1}{2}mR^2 \left[2\ddot{\phi} - 2\omega\sin(\phi)\dot{\phi} \right]$$

$$\frac{\partial \mathcal{L}}{\partial \phi} = -mR^2\omega(\omega + \dot{\phi})\sin(\phi)$$

$$\Rightarrow \frac{1}{2}mR^2 \left[2\ddot{\phi} - 2\omega\sin(\phi)\dot{\phi} \right] = -mR^2\omega(\omega + \dot{\phi})\sin(\phi)$$

$$2\ddot{\phi} - 2\omega\sin(\phi)\dot{\phi} = -2\omega(\omega + \dot{\phi})\sin(\phi)$$

$$\ddot{\phi} - \omega\sin(\phi)\dot{\phi} = -\omega^2\sin(\phi) - \omega\dot{\phi}\sin(\phi) \quad \text{terms shown in blue cancel}$$

$$\ddot{\phi} = -\omega^2\sin(\phi)$$

The last equation is exactly the equation of a simple pendulum with g/l replaced by ω^2 oscillating about the point $\phi = 0$ (that is, B); compare Taylor, Section 7.2, Eq. (7.33).

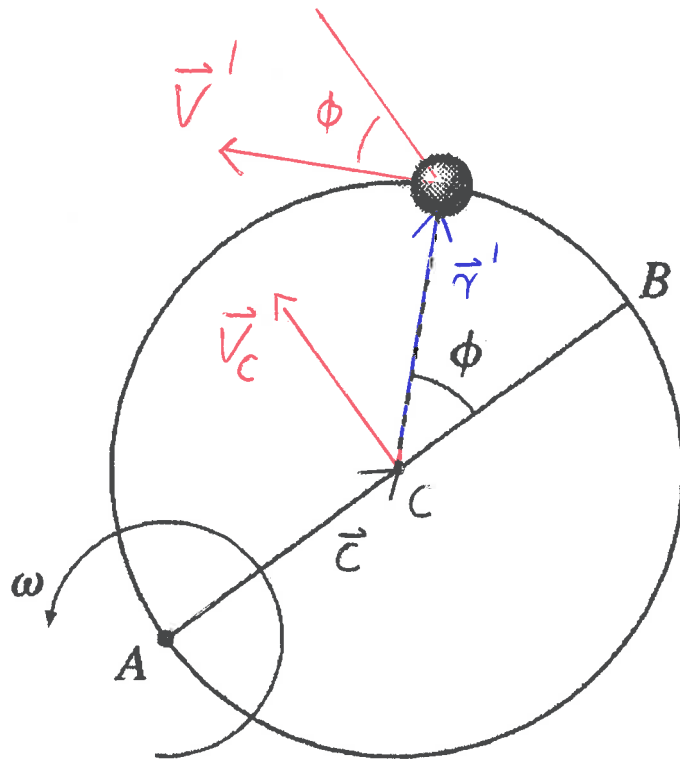


Figure 7.16 Problem 7.35

7.21 ★ If we use two-dimensional polar coordinates, the bead's velocity is $\mathbf{v} = (\dot{r}, r\dot{\phi}) = (\dot{r}, r\omega)$, where ω is the fixed angular velocity with which the rod is forced to rotate. Thus $\mathcal{L} = T - U = \frac{1}{2}mv^2 = \frac{1}{2}m(\dot{r}^2 + r^2\omega^2)$. (U is a constant, which we may as well take to be zero.) The Lagrange equation is $\ddot{r} = \omega^2 r$, the general solution of which is $r(t) = Ae^{\omega t} + Be^{-\omega t}$. If $r(0) = \dot{r}(0) = 0$, then $A = B = 0$ and the bead stays put; that is, $r = 0$ is an equilibrium point (though unstable, as we'll see). If $r(0) = r_o \neq 0$, but $\dot{r}(0) = 0$, then $A = B = r_o/2$ and

$$r(t) = \frac{1}{2}r_o(e^{\omega t} + e^{-\omega t}) \rightarrow \frac{1}{2}r_o e^{\omega t}$$

as $t \rightarrow \infty$. As seen in the rotating frame of the rod, there is an outward “centrifugal force” $m\omega^2 r$. This causes the bead to accelerate outward, and as r increases, the acceleration increases in proportion — hence the exponential growth of r .

7.23 ★ The small cart's KE is $T = \frac{1}{2}mv^2 = \frac{1}{2}m(\dot{x} + \dot{X})^2 = \frac{1}{2}m(\dot{x} - A\omega \sin \omega t)^2$, and $U = \frac{1}{2}kx^2$. Thus $\partial\mathcal{L}/\partial\dot{x} = m(\dot{x} - A\omega \sin \omega t)$ and Lagrange's equation reads

$$-kx = m\ddot{x} - mA\omega^2 \cos \omega t \quad \text{or} \quad \ddot{x} + \omega_o^2 x = B \cos \omega t$$

where I have replaced k/m by ω_o^2 and renamed $A\omega^2$ as B .