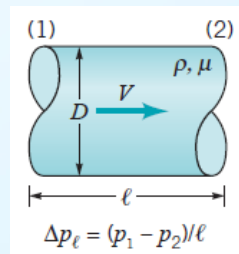




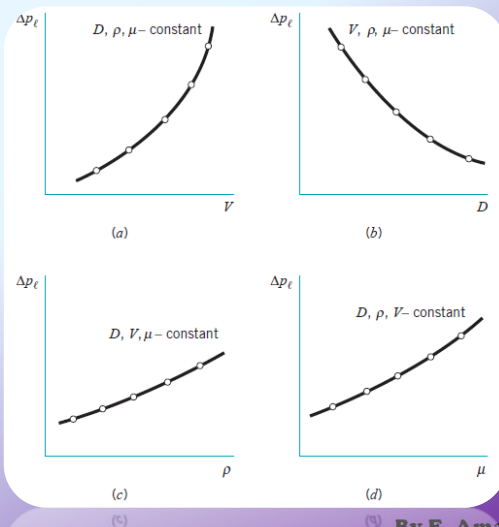
Dimensional analysis

● Application: experimental correlations

flow in smooth pipes ▼



$$\Delta p_\ell = f(D, \rho, \mu, V)$$



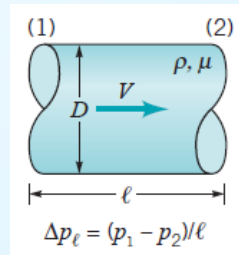
Chapter 8

By E. Amani

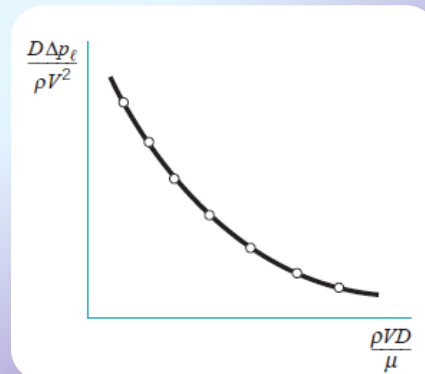
Dimensional analysis

- **Application: experimental correlations**

flow in smooth pipes ▼



$$\frac{D \Delta p_\ell}{\rho V^2} = \phi \left(\frac{\rho V D}{\mu} \right)$$



Chapter 8

By E. Amani

Buckingham Pi theorem

- **Dimensionless group or parameter:** The product of several dimensional quantities with dimension of 1

height

$$\pi_1 = \frac{h}{D}$$

$$\left[\frac{h}{D} \right] = \frac{L}{L} = 1$$

diameter

- **Theorem:**

- A phenomenon involving n variables
- with r is the minimum number of reference dimensions (products of basic dimensions)
- Can be described by a relation among $n - r$ dimensionless parameters

Chapter 8

By E. Amani

Dimensional analysis

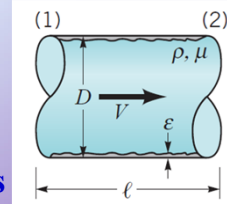
Steps:

1. Finding all design variables governing an objective variable.

Example: Pressure drop in incompressible pipe flow

Pressure drop $\Delta p = f(\rho, \mu, V, \ell, D, \varepsilon)$ $n = 7$ (1.8)

viscosity μ
density ρ
Bulk velocity V
roughness ε



2. Listing the dimensions of all variables (using Table 1.1 FLT and MLT).

(FLT)	(MLT)		
$[\Delta p] = FL^{-2} = MT^{-2}L^{-1}$	$[V] = LT^{-1}$	$[\varepsilon] = L$	
$[\rho] = FL^{-3} = ML^{-3}$	$[\ell] = L$		(2.8)
$[\mu] = FL^{-2}T = ML^{-1}T^{-1}$	$[D] = L$		

Chapter 8

By E. Amani

Dimensional analysis

Steps:

3. Determine the minimum reference dimensions.

- The minimum dimensional groups for constructing all dimensions
- Start from simplest dimensions
- Check r using both FLT and MLT

(FLT)	(MLT)		
$[\Delta p] = FL^{-2} = MT^{-2}L^{-1}$	$[V] = L\cancel{T}^{-1}$	$[\varepsilon] = L$	
$[\rho] = FL^{-3} = ML^{-3}$	$[\ell] = L$		
$[\mu] = \cancel{F}L^{-2}T = \cancel{M}L^{-1}T^{-1}$	$[D] = L$	$r = 3: L, T, M$	

4. Choose r design variables to construct the reference dimensions

- D, V, ρ (should involve all reference dimensions)

Chapter 8 $L \rightarrow D \quad T \rightarrow D/V \quad M \rightarrow \rho D^3 \quad (3.8) \quad \text{By E. Amani}$

Dimensional analysis

Steps:

5. Determine the dimensionless variables:

- Dividing the remaining variables, not used in step 4, to their dimensions $L \rightarrow D \quad T \rightarrow D/V \quad M \rightarrow \rho D^3$
- Replace reference dimensions by equivalent variables from step 4

$$\begin{aligned}\pi_1 &= \frac{\Delta p}{MT^{-2}L^{-1}} = \frac{\Delta p}{\rho D^3 (D/V)^{-2} D^{-1}} = \frac{\Delta p}{\rho V^2} \\ \pi_2 &= \frac{\mu}{ML^{-1}T^{-1}} = \frac{\mu}{\rho D^3 D^{-1} (D/V)^{-1}} = \frac{\mu}{\rho V D} \\ \pi_3 &= \frac{\ell}{L} = \frac{\ell}{D} \\ \pi_4 &= \frac{\varepsilon}{L} = \frac{\varepsilon}{D} \\ \pi_1 &= f(\pi_2, \pi_3, \pi_4) \quad (5.8) \quad \frac{\Delta p}{\rho V^2} = f\left(\frac{\mu}{\rho V D}, \frac{\ell}{D}, \frac{\varepsilon}{D}\right)\end{aligned} \quad (4.8)$$

Chapter 8

By E. Amani

Dimensional analysis

Steps:

6. Combining dimensionless groups (if desired)

$$\begin{aligned}\pi_1 &\rightarrow \pi_1^\alpha \\ \pi_1, \pi_2 &\rightarrow \pi_1, \pi_1 \pi_2 \text{ or } \pi_2, \pi_1 \pi_2 \\ \pi_1, \pi_2 &\rightarrow \pi_1, \pi_1 / \pi_2 \text{ or } \pi_2, \pi_1 / \pi_2 \\ \pi_1, \pi_2 &\rightarrow \pi_1, \pi_1^\alpha \pi_2^\beta \text{ or } \pi_2, \pi_1^\alpha \pi_2^\beta\end{aligned}$$

Example:

$$\begin{aligned}\frac{\Delta p}{\rho V^2} &= f_2\left(\frac{\mu}{\rho V D}, \frac{\ell}{D}, \frac{\varepsilon}{D}\right) \\ \pi_2 &= \frac{\mu}{\rho V D} \rightarrow \pi_2^{-1} = \frac{\rho V D}{\mu} \\ \text{Euler number (Eu)} &\leftarrow \frac{\Delta p}{\rho V^2} = f_3\left(\frac{\rho V D}{\mu}, \frac{\ell}{D}, \frac{\varepsilon}{D}\right) \quad \text{Reynolds number (Re)} \quad (6.8)\end{aligned}$$

Chapter 8

By E. Amani

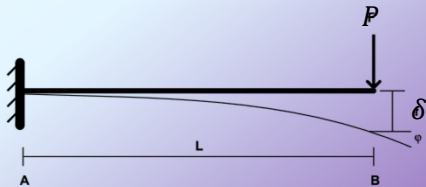
Dimensional analysis

- **Corollary:** When there is a single dimensionless parameter

$$\pi_1 = f(.) = cte$$

- **Example:** Perform the dimensional analysis for the deflection of a beam

$$\delta = f(P, \ell, E, I)$$



Lecture Notes

Chapter 8

By E. Amani

Dimensional analysis

- **Important Dimensionless Groups (parameters) in Fluid Mechanics**

TABLE 7.1

Some Common Variables and Dimensionless Groups in Fluid Mechanics

Variables: Acceleration of gravity, g ; Bulk modulus, E_v ; Characteristic length, ℓ ; Density, ρ ; Frequency of oscillating flow, ω ; Pressure, p (or Δp); Speed of sound, c ; Surface tension, σ ; Velocity, V ; Viscosity, μ

Dimensionless Groups	Name	Interpretation (Index of Force Ratio Indicated)	Types of Applications
$\frac{\rho V \ell}{\mu}$	Reynolds number, Re	$\frac{\text{inertia force}}{\text{viscous force}}$	Generally of importance in all types of fluid dynamics problems
$\frac{V}{\sqrt{g \ell}}$	Froude number, Fr	$\frac{\text{inertia force}}{\text{gravitational force}}$	Flow with a free surface
$\frac{p}{\rho V^2}$	Euler number, Eu	$\frac{\text{pressure force}}{\text{inertia force}}$	Problems in which pressure, or pressure differences, are of interest
$\frac{\rho V^2}{E_v}$	Cauchy number, ^a Ca	$\frac{\text{inertia force}}{\text{compressibility force}}$	Flows in which the compressibility of the fluid is important

Chapter 8

By E. Amani

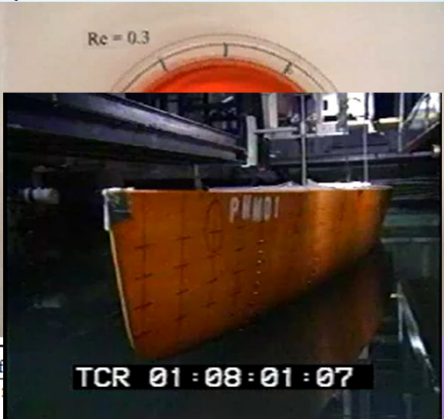
Dimensional analysis

- Important Dimensionless Groups in Fluid Mechanics

TABLE 7.1
Some Common Variables and Dimensionless Groups

Variables: Acceleration of gravity, g ; Bulk modulus, E_v ; oscillating flow, ω ; Pressure, p (or Δp); Speed of sound

Dimensionless Groups	Name	Interpretation	Force Ratio
$\frac{\rho V \ell}{\mu}$	Reynolds number, Re	inertia force	viscous force
$\frac{V}{\sqrt{g \ell}}$	Froude number, Fr	inertia force	gravitational force
$\frac{p}{\rho V^2}$	Euler number, Eu	pressure force	inertia force
$\frac{\rho V^2}{E_v}$	Cauchy number, ^a Ca	inertia force	compressibility force



Chapter 8

By E. Amani

Dimensional analysis

- Important Dimensionless Groups (parameters) in Fluid Mechanics

$\frac{V}{c}$	Mach number, ^a Ma	inertia force	compressibility force	Flows in which the compressibility of the fluid is important
$\frac{\omega \ell}{V}$	Strouhal number, St	inertia (local) force	inertia (convective) force	Unsteady flow with a characteristic frequency of oscillation
$\frac{\rho V^2 \ell}{\sigma}$	Weber number, We	inertia force	surface tension force	Problems in which surface tension is important

^aThe Cauchy number and the Mach number are related and either can be used as an index of the relative effects of inertia and compressibility. See accompanying discussion.

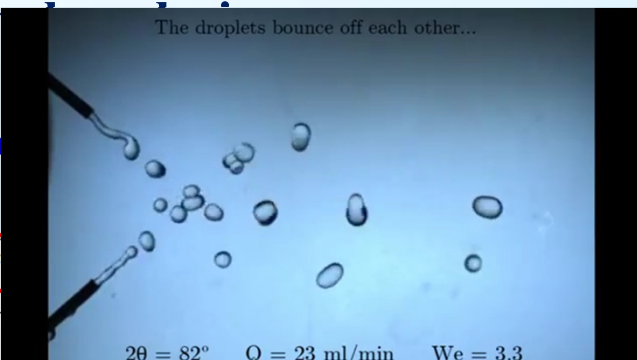
Chapter 8

By E. Amani

Dimensional Analysis

- **Important dimensionless numbers in Fluid Mechanics**

$\frac{V}{c}$
 Mach number



The droplets bounce off each other...

$2\theta = 82^\circ$ $Q = 23 \text{ ml/min}$ $We = 3.3$

$\frac{\omega \ell}{V}$
 $\frac{\rho V^2 \ell}{\sigma}$
 The Cauchy number and its importance. See accompanying document.

Chapter 8

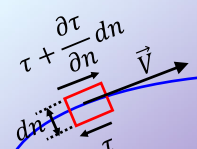
By E. Amani

Dimensional analysis

- **The Reynolds number: Physical interpretation**
 - For the sake of simplicity, steady flow

⇒ **Lecture Notes**

$$\frac{\text{inertial force}}{\text{viscous force}} \propto \frac{\rho V_0 \ell}{\mu} \equiv Re \quad (8.8)$$



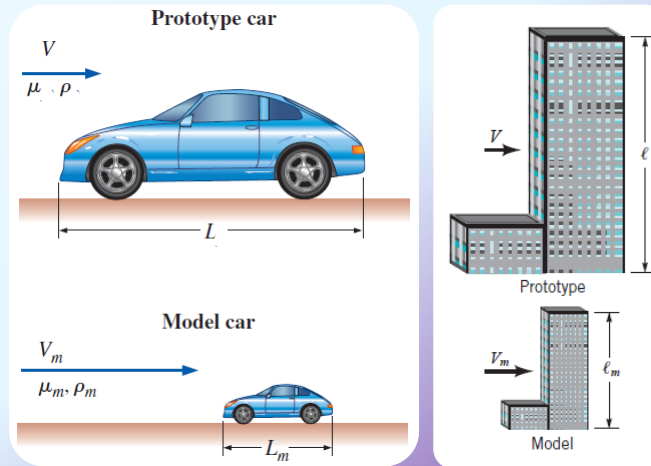
- **Exercise:** Similarly prove the physical meanings of the Euler and Froude numbers.
- **Note:** A dimensionless parameter, representing two forces ratio, is important if and only if none of the forces is negligible

Chapter 8

By E. Amani

Modeling and similitude

● Prototype and model

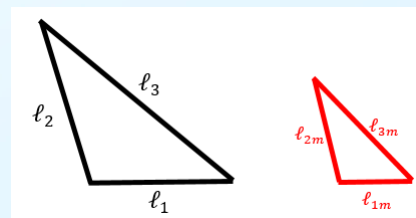


Chapter 8

By E. Amani

Similitude

● Geometrical similitude



$$\frac{\ell_{1m}}{\ell_1} = \frac{\ell_{2m}}{\ell_2} = \frac{\ell_{3m}}{\ell_3}$$

Chapter 8

By E. Amani

Similitude

- **Geometrical similitude**

$$\frac{\ell_{im}}{\ell_i} = \text{const} = \lambda_\ell; \quad i = 1, \dots$$

Length scale

constant for
all
dimensions

- **Kinematic similitude**

$$\vec{V}_m = \lambda_V \vec{V} \quad \text{or} \quad \frac{V_m}{V} = \lambda_V, \quad \vec{V}_m \parallel \vec{V}$$

Velocity scale

- **Dynamic similitude**

$$\vec{F}_{\alpha,m} = \lambda_F \vec{F}_\alpha \quad \text{or} \quad \frac{F_{\alpha,m}}{F_\alpha} = \lambda_F, \quad \vec{F}_{\alpha,m} \parallel \vec{F}_\alpha; \quad \alpha = 1, \dots$$

Force scale

constant for
all forces

Chapter 8

By E. Amani

Similitude

- **Design of the model**

- The dimensional variables are not the same for the prototype and model
- It is sufficient that all but one of the dimensionless parameters are designed to be the same for the prototype and model
- The last dimensionless parameter would be the same since

$$\pi_2 = \pi_{2m}, \pi_3 = \pi_{3m}, \dots \xrightarrow{\pi_1 = f(\pi_2, \pi_3, \dots)} \pi_1 = \pi_{1m} \quad (12.8)$$

Chapter 8

By E. Amani

Similitude

- **Example: Pressure drop in pipes** $\frac{\Delta p}{\rho V^2} = f_3 \left(Re, \frac{\ell}{D}, \frac{\varepsilon}{D} \right)$

Design: $Re_m = \frac{\rho_m V_m D_m}{\mu_m} = \frac{\rho V D}{\mu} = Re \quad (13.8)$

$$\frac{\ell_m}{D_m} = \frac{\ell}{D}, \frac{\varepsilon_m}{D_m} = \frac{\varepsilon}{D} \quad (14.8) \quad \rightarrow \quad \frac{\Delta p_m}{\rho_m V_m^2} = \frac{\Delta p}{\rho V^2} \quad (15.8)$$

- **Note 1: Eq. (15.8) can be used to calculate the pressure drop for the prototype using the one measured for model** $\xrightarrow{(15.8)} \Delta p = \frac{\rho V^2}{\rho_m V_m^2} \Delta p_m$

- **Note 2: The model and prototype have geometrical similarity (14.8)** $\xrightarrow{(14.8)} \frac{\ell_m}{\ell} = \frac{D_m}{D}, \frac{\varepsilon_m}{\varepsilon} = \frac{D_m}{D} \rightarrow \frac{\ell_m}{\ell} = \frac{\varepsilon_m}{\varepsilon} = \frac{D_m}{D} = cte$

- **Note 3: The model and prototype have dynamic similarity (13.8)** $\xrightarrow{(13.8)} \frac{F_{Im}}{F_{\tau m}} = \frac{F_I}{F_{\tau}} \rightarrow \frac{F_{Im}}{F_I} = \frac{F_{\tau m}}{F_{\tau}} = cte$

Chapter 8

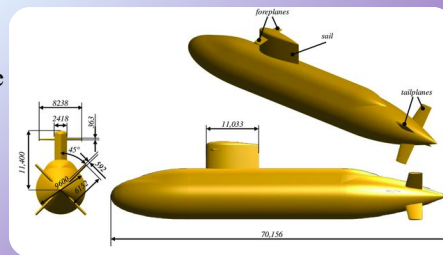
By E. Amani

Similitude

- **Sample problem**

A 1:20 scale model of a submarine is tested in a wind tunnel under air absolute pressure of 300 lb/in² and temperature of 120 °F. The speed of the prototype is 15 kn in the ocean water with a kinematic viscosity of 1.121 ft³/s.

- Determine the required air speed in the wind tunnel. The air is assumed as an ideal gas with a gas constant of 1716 ft.lb/(slug.R).
- What is the ratio of the model to prototype drag forces.
- In spite of the wind tunnel high pressure, why is the incompressible flow assumption valid?
- Is the similitude between the model and prototype violated if the submarine moves near the ocean surface?



➡ **Lecture Notes**

Chapter 7

By E. Amani

Equation non-dimensionalization

● Sample problem ➡ Lecture Notes

A viscous fluid flows through a vertical, square channel as shown in below figure. The velocity w can be expressed as

$$w = f(x, y, b, \mu, \gamma, V, \partial p / \partial z)$$

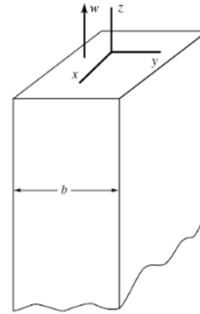
where μ is the fluid viscosity, γ the fluid specific weight, V the mean velocity, and $\partial p / \partial z$ the pressure gradient in the z direction.

(a) Use dimensional analysis to find a suitable set of dimensionless variables and parameters for this problem.

(b) The differential equation governing the fluid motion for this problem is

$$\frac{\partial p}{\partial z} = -\gamma + \mu \left(\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} \right)$$

Write this equation in a suitable dimensionless form, and show that the similarity requirements obtained from this analysis are the same as those resulting from the dimensional analysis of part (a).



● The governing dimensionless parameters of a phenomenon appear in its governing equations or boundary conditions

Chapter 8

By E. Amani

The end of chapter 8

Chapter 8

By E. Amani