



Open-channel flow applications

- Canals



irrigation canal ▲

Open-channel flow applications

- Canals
- Culverts



Chapter 8

By E. Amani

Open-channel flow applications

- Canals
- Culverts
- Spillways



Chapter 8

Bell-mouth spillway ▲

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Bell-mouth spillway ▲

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Ogee spillway ▲

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Open-channel flow applications

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- Weirs



Triangular sharp-crested weir ▲

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Open-channel flow applications

- Canals
- Culverts
- Spillways
- Weirs
- Sluice gates



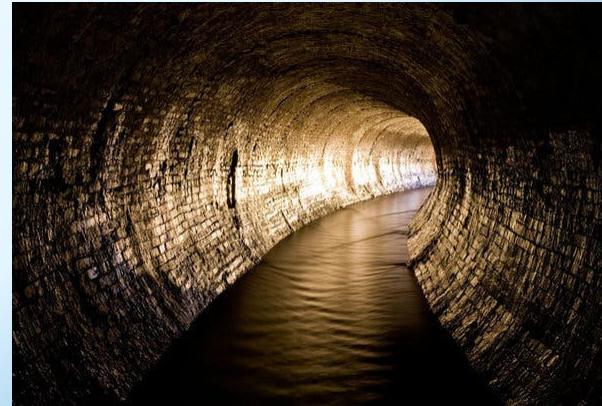
Dam sluice gates ▲

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Open-channel flow applications

- Canals
- Culverts
- Spillways
- Weirs
- Sluice gates
- Sewers



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Open-channel flow applications

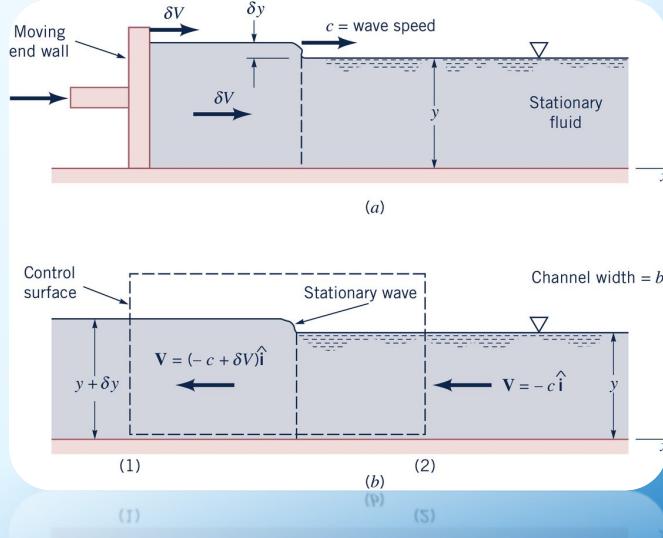
- Canals
- Culverts
- Spillways
- Weirs
- Sluice gate
- Sewers
- Dams
- ...



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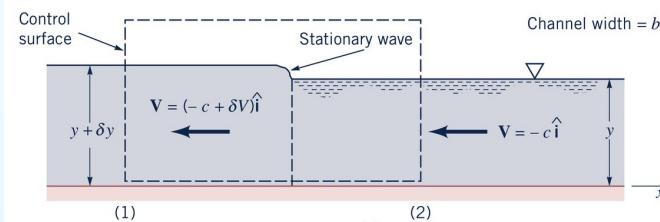
Gravity wave and Froude number



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Gravity wave and Froude number



• 1D flow assumption + rectangular channel:

• Mass: $\rho c b y = \rho(c - \delta V)(y + \delta y)b$

• Momentum: $-\frac{1}{2}\rho g b[(y + \delta y)^2 - y^2] = \rho c b y(c - \delta V - c)$

- Exercise: Using Bernoulli equation normal to streamline, show that the pressure hydrostatically changes

- Removing δV and solving for c :

$$c = \sqrt{gy \left(1 + \frac{\delta y}{y}\right) \left(1 + \frac{1}{2} \frac{\delta y}{y}\right)}$$

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Gravity wave and Froude number

- Large-amplitude single-wave:

$$c = \sqrt{gy \left(1 + \frac{\delta y}{y}\right) \left(1 + \frac{1}{2} \frac{\delta y}{y}\right)}$$

$\delta y \uparrow \Rightarrow c \uparrow$

- Small-amplitude elementary single-wave:

$$c = \sqrt{gy}$$

- Froude number:

$$Fr \equiv \frac{V}{c} = \frac{V}{\sqrt{gL}}$$

↑ For rectangular channel
L = y

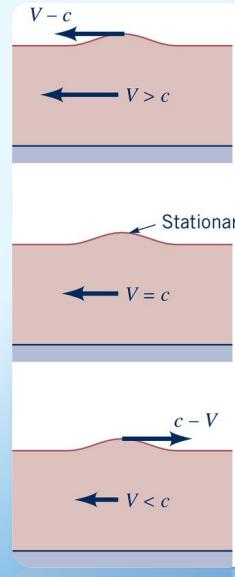
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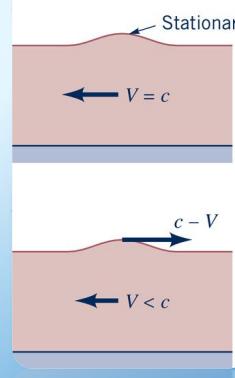
Gravity wave and Froude number

- Regimes:

- Supercritical ($Fr > 1$):



- Critical ($Fr = 1$):



- Subcritical ($Fr < 1$):



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Forces

- Gravity ✓
- Inertia ✓
- Friction ✓ \times
- Surface tension ✓ \times
- pressure \times

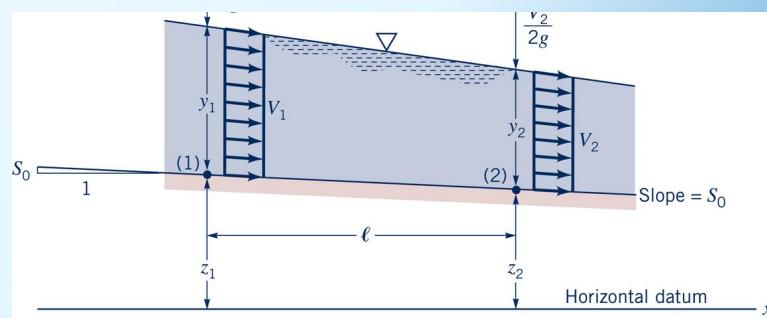
$$Fr \sim \frac{\text{Inertia}}{\text{Gravity}}$$

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Classification

- Based on depth variation:
 - 1. Uniform Flow (UF): $dy/dx = 0, y = y_n$
 - 2. Gradually Varying Flow (GVF): $dy/dx \ll 1$
 - 3. Rapidly Varying Flow (RVF): $dy/dx = O(1)$



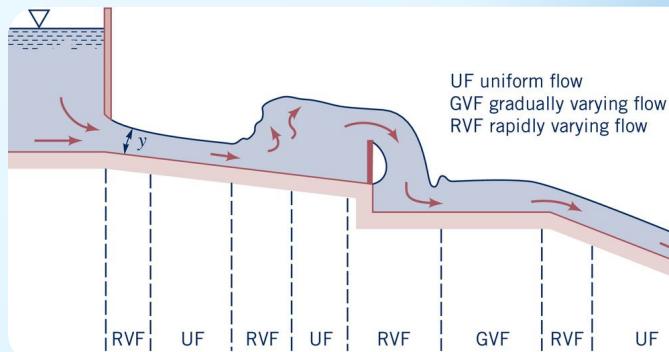
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- **Laminar/turbulent:**

1. Laminar ($Re < 500$)
2. Turbulent ($Re > 12500$)

Hydraulic radius

$$Re = VR_h/v$$

Flow of

50 °F water ($v = 1.41 \times 10^{-5} \text{ ft/s}$), $V = 1 \text{ ft/s}$, and $R_h = 10 \text{ ft}$
 $\Rightarrow Re = 7.1 \times 10^5$

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Classification

- **Based on depth variation:**

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- **Laminar/turbulent:**

1. Laminar ($Re < 500$)
2. Turbulent ($Re > 12500$)

- **Wave propagation characteristics:**

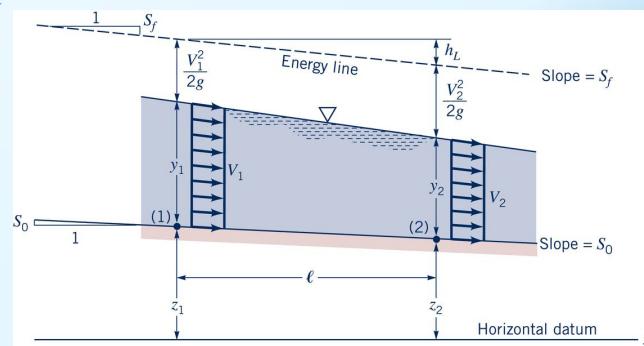
1. Subcritical ($Fr < 1$)
2. Critical ($Fr = 1$)
3. Supercritical ($Fr > 1$)

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Specific energy diagram

- **Assumptions:**



- 1D flow
- Steady flow
- Constant bottom slope ($S_0 = (z_1 - z_2)/\ell$)

$$\frac{p_1}{\gamma} + \frac{V_1^2}{2g} + z_1 = \frac{p_2}{\gamma} + \frac{V_2^2}{2g} + z_2 + h_L \quad p_1/\gamma = y_1, p_2/\gamma = y_2$$

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Specific energy diagram

- Leads to:

$$y_1 + \frac{V_1^2}{2g} + S_0\ell = y_2 + \frac{V_2^2}{2g} + h_L$$

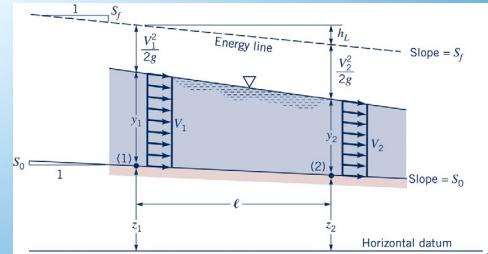
- The specific energy (head) definition:

$$E \equiv y + \frac{V^2}{2g}$$

- and friction slope
($S_f = h_L/\ell$) results in:

$$E_1 = E_2 + (S_f - S_0)\ell$$

For non-constant bottom slope $\downarrow (z_2 - z_1) + h_L$

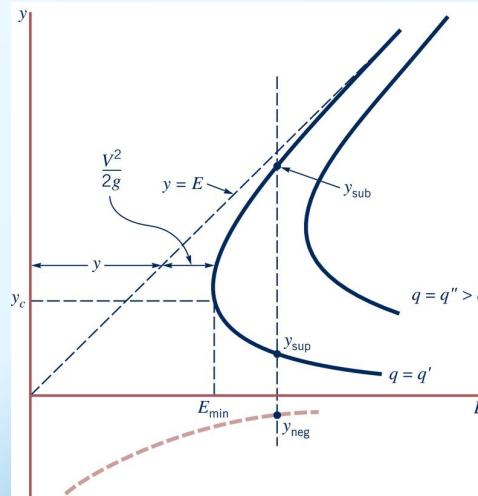


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Specific energy diagram

- diagram



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Specific energy diagram

- For rectangular channel of width b

$(q = Q/b = Vy_b/b = Vy)$:

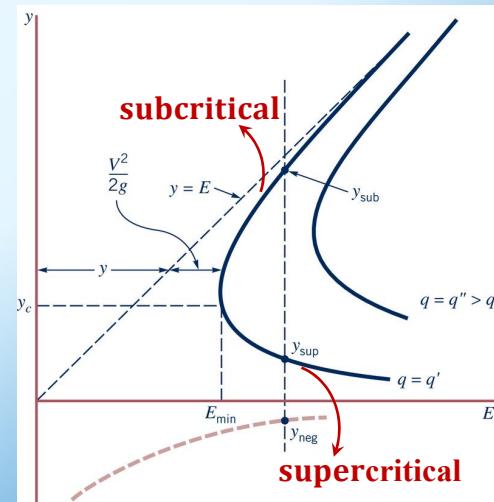
$$E = y + \frac{q^2}{2gy^2}$$

$$\rightarrow y^3 - Ey^2 + (q^2/2g) = 0$$

$$dE/dy = 0 \rightarrow y_c = \left(\frac{q^2}{g}\right)^{1/3}$$

$$E_{\min} = \frac{3y_c}{2}$$

$$V_c = \frac{q}{y_c} = \sqrt{gy_c} \rightarrow \text{Fr}_c = \frac{V_c}{c} = 1$$



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Specific energy diagram

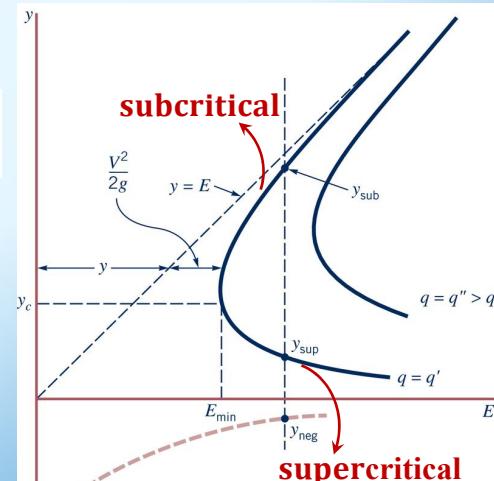
- For non-rectangular channel,
for a given Q , $A(y)$,
and $b_0(y)$ [2]:

$$A(y_c) = \left(\frac{b_0(y_c)Q^2}{g}\right)^{1/3}$$

Should be solved by an trial and error approach for y_c and:

$$V_c = c = \frac{Q}{A_c} = \left(\frac{gA(y_c)}{b_0(y_c)}\right)^{1/2}$$

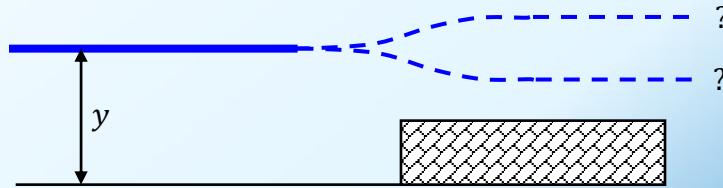
$$\text{Fr} = \frac{V}{c} = \frac{V}{\sqrt{gL}} \rightarrow L = \frac{A(y_c)}{b_0(y_c)}$$



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Flow over a bump

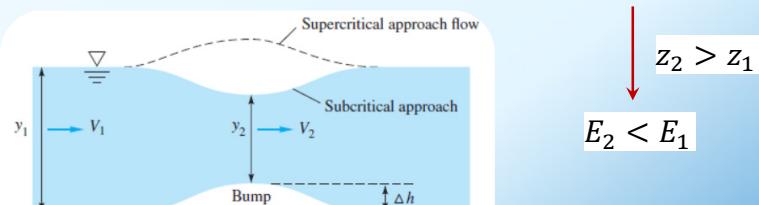


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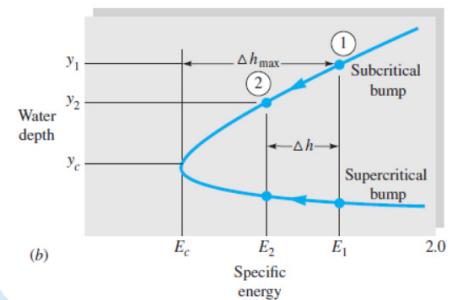
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Flow over a bump

$$E_1 = E_2 + (z_2 - z_1) + h_L$$



(a)



(b)

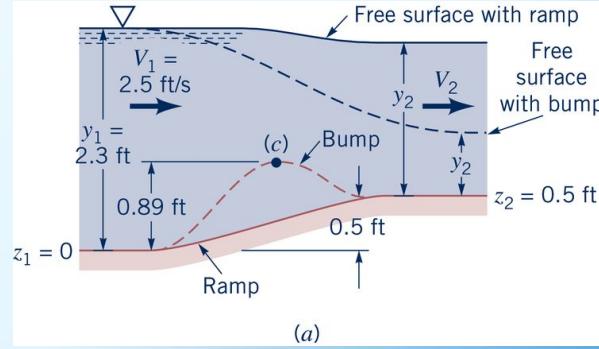
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Example

- Rectangular channel, water, neglecting friction, $y_2 = ?$



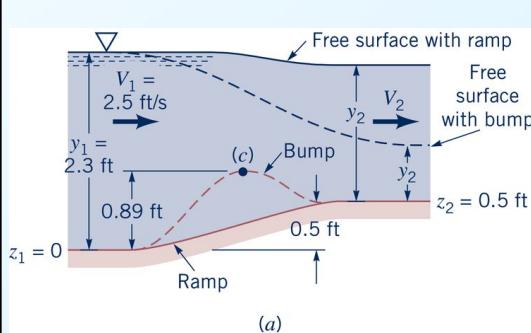
→ Notes: VIII.1

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Example

- Rectangular channel, water, neglecting friction, $y_2 = ?$

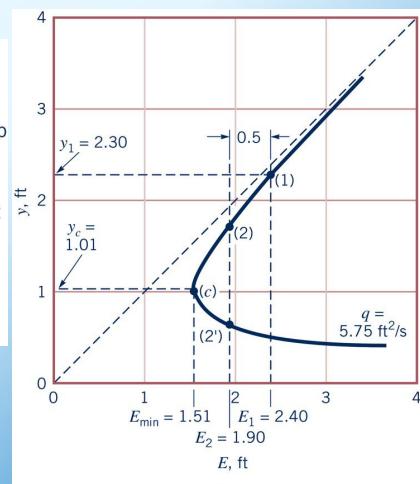


$$y_c = \left(\frac{q^2}{g} \right)^{1/3} = 1.01 \text{ ft}$$

$$E_{\min} = \frac{3y_c}{2} = 1.51 \text{ ft}$$

$$E_1 = E_{\min} + z_c - z_1$$

$$\rightarrow z_c - z_1 = 0.89 \text{ ft}$$



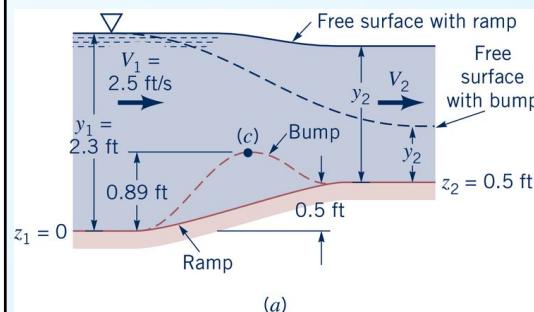
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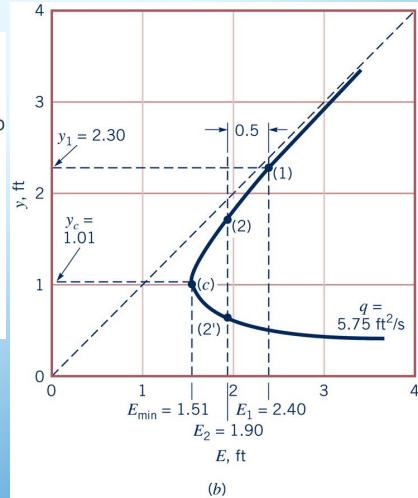
Question

- What happens if the bump level gets higher than 0.89 ft?

- Choking



(a)



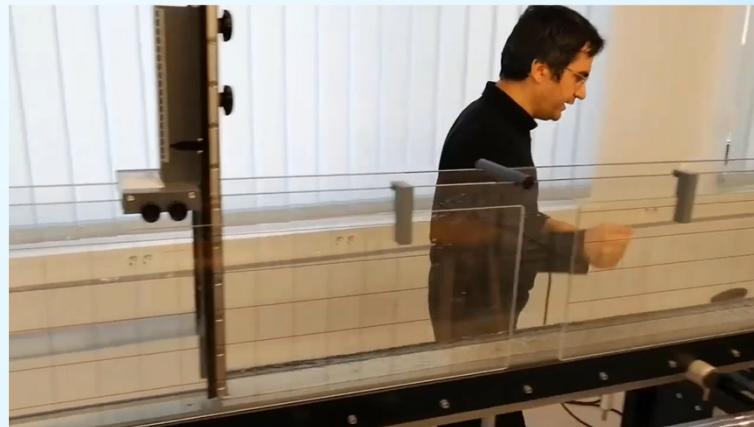
(b)

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Question

- Choking



- See also:

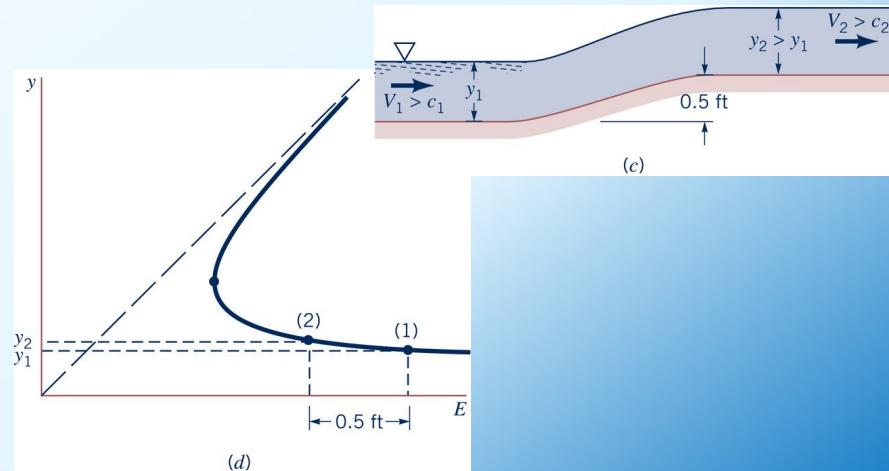
https://en.wikiversity.org/wiki/Classic_energy_problem_in_open-channel_flow

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Example

- Rectangular channel, neglecting friction, $y_2 = ?$



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Uniform flow

- Bottom slope cancels out friction slope (balance between gravity and friction)
- For constant area and shape channels:

$$E_1 = E_2 + (S_f - S_0)\ell \quad \text{if } S_0 = S_f$$

$$E_1 = E_2 \quad A = \text{cte}$$

$$E = y + \frac{Q^2}{2gA^2} \quad y = \text{cte}$$

$$V = Q/A = \text{cte} \quad \frac{dy}{dx} = 0$$

(a) A diagram showing a channel of constant width 'a' with a free surface. The discharge is indicated as 'Q'.

(b) A diagram of a trapezoidal channel. The width at the bottom is labeled 'a'. The free surface is horizontal. The discharge is indicated as 'Q'.

Open-channel flow counterpart of fully-developed internal flow

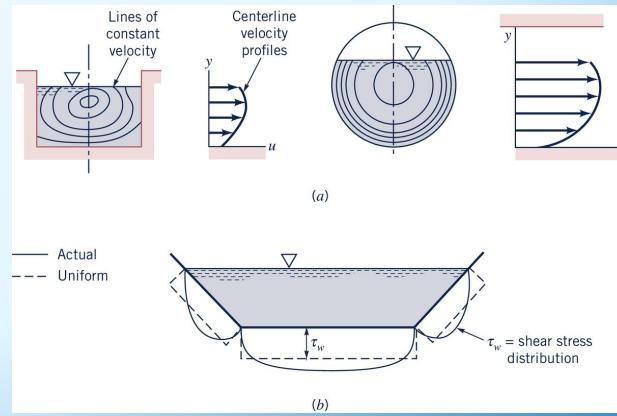
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Uniform flow

- Assumptions:

- Steady flow
- 1D flow (uniform velocity and shear stress profiles)
- Constant-area and -shape channel



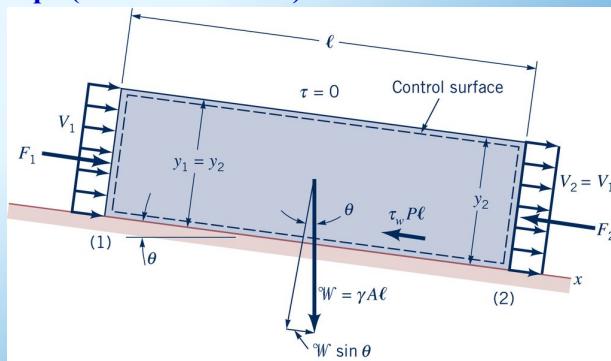
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Uniform flow

- Assumptions:

- Steady flow
- 1D flow (uniform velocity and shear stress profiles)
- Constant-area and -shape channel
- Small bottom slope ($\theta < 6^\circ$ or 0.1 rad)



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Uniform flow

- Momentum balance**

$$F_1 - F_2 - \tau_w P\ell + \mathcal{W} \sin \theta = 0 \rightarrow \tau_w = \frac{\mathcal{W} \sin \theta}{P\ell} = \frac{\mathcal{W} S_0}{P\ell}$$

$$\tau_w = \frac{\gamma A \ell S_0}{P\ell} = \gamma R_h S_0$$

$R_h \equiv \frac{A}{P} = \frac{1}{4} D_h$

Hydraulic radius **Wetted perimeter**

$\sin \theta \approx \tan \theta = S_0, \quad \mathcal{W} = \gamma A \ell,$

$\tau_w = \frac{\mathcal{W} \sin \theta}{P\ell} = \frac{\mathcal{W} S_0}{P\ell}$

Equal pressure distributions

Control surface

(1) y_1 V_1 F_1 θ $\tau_w P\ell$ $\mathcal{W} = \gamma A \ell$ $\mathcal{W} \sin \theta$

(2) y_2 $V_2 = V_1$ F_2

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Uniform flow

- Momentum balance**

$$F_1 - F_2 - \tau_w P\ell + \mathcal{W} \sin \theta = 0 \rightarrow \tau_w = \frac{\mathcal{W} \sin \theta}{P\ell} = \frac{\mathcal{W} S_0}{P\ell}$$

$$\tau_w = \frac{\gamma A \ell S_0}{P\ell} = \gamma R_h S_0$$

(31.4) $\tau_w = \frac{1}{8} \rho V^2 f$

Assuming turbulent fully-developed pipe flow for wholly turbulent regime

$f = f(\varepsilon/D)$

$\sin \theta \approx \tan \theta = S_0, \quad \mathcal{W} = \gamma A \ell,$

$\tau_w = \frac{\mathcal{W} \sin \theta}{P\ell} = \frac{\mathcal{W} S_0}{P\ell}$

$C = (8g/f)^{1/2}$

$V = C \sqrt{R_h S_0}$

Equal pressure distributions

Control surface

(1) y_1 V_1 F_1 θ $\tau_w P\ell$ $\mathcal{W} = \gamma A \ell$ $\mathcal{W} \sin \theta$

(2) y_2 $V_2 = V_1$ F_2

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Uniform flow

Manning equation

Manning equation Experiment

$$V = C \sqrt{R_h S_0}$$

Chezy equation

where $\kappa = 1$ if SI units are used,
and $\kappa = 1.49$ if BG units are used.

$$V = \frac{\kappa}{n} R_h^{2/3} S_0^{1/2}$$

$$Q = \frac{\kappa}{n} A R_h^{2/3} S_0^{1/2}$$

Manning resistance coefficient (dimensional!!) for water

Values of the Manning Coefficient, n (Ref. 6)

Wetted Perimeter	n	Wetted Perimeter	n
A. Natural channels		D. Artificially lined channels	
Clean and straight	0.030	Glass	0.010
Sluggish with deep pools	0.040	Brass	0.011
Major rivers	0.035	Steel, smooth	0.012
B. Floodplains		E. Excavated earth channels	
Pasture, farmland	0.035	Steel, painted	0.014
Light brush	0.050	Steel, riveted	0.015
Heavy brush	0.075	Cast iron	0.013
Trees	0.15	Concrete, finished	0.012
C. Excavated earth channels		F. Rubble masonry	
Clean	0.022	Planned wood	0.012
Gravelly	0.025	Clay tile	0.014
Weedy	0.030	Brickwork	0.015
Stony, cobbles	0.035	Asphalt	0.016
		Corrugated metal	0.022
		Rubble masonry	0.025

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Uniform flow

- Normal depth (y_N):**

$$Q = \frac{\kappa}{n} A \left(\frac{A}{P} \right)^{2/3} S_0^{1/2} = \frac{\kappa}{n} \frac{A^{5/3} S_0^{1/2}}{P^{2/3}} \rightarrow A(y_N) = \left(\frac{nQ}{\kappa S_0^{1/2}} \right)^{3/5} P^{2/5}(y_N)$$

solved for y_N

- Best hydraulic cross section (P_{\min} or A_{\min}) for given Q , S_0 , and material (n):**

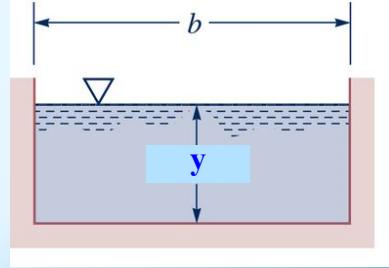
$$\frac{dP}{dy} = 0 = \frac{dA}{dy}$$

- Best hydraulic cross section gives the maximum flow rate for given A , S_0 , and material (n)**

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Example

- For a rectangular cross-section open-channel, given Q , S_0 , and material (n): a) Determine the best hydraulic cross section. b) Calculate the normal depth in this channel as a function of other parameters.



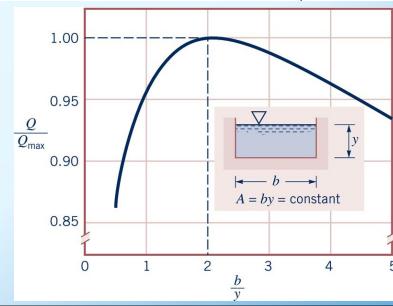
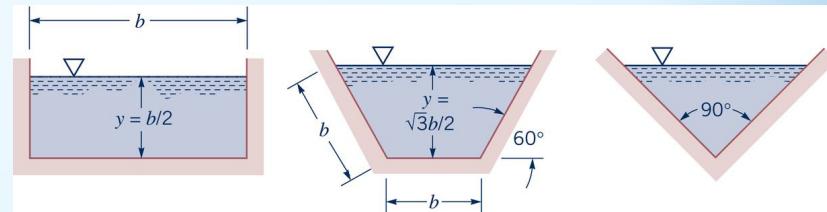
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Uniform flow

- Best hydraulic cross section (P_{\min}) for given Q , S_0 , and material (n):



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Gradually Varied Flow (GVF)

Factors leading to GVF:

- Varying bottom slope ($S_0 \neq cte$)
- Varying cross-sectional area or shape ($A \neq cte$)
- Obstruction (developing flow)

Governing equation:

- Momentum (Bernoulli) + continuity [2]:

$$\frac{dy}{dx} = \frac{S_0 - S_f}{1 - Fr^2}, Fr^2 = \frac{V^2 b_0}{gA}$$

$$V = \frac{\kappa}{n} R_h^{2/3} S_{0n}^{1/2} \rightarrow S_f = S_{0n} = \frac{n^2}{\kappa^2} \frac{V^2}{R_h^{4/3}}$$

Assuming Manning
equation locally

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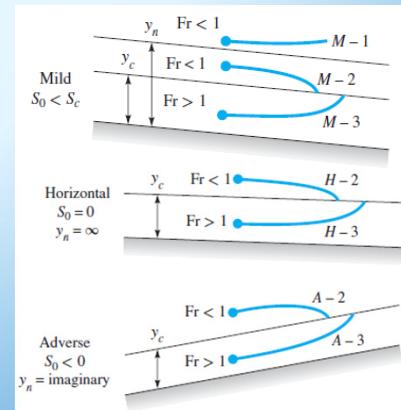
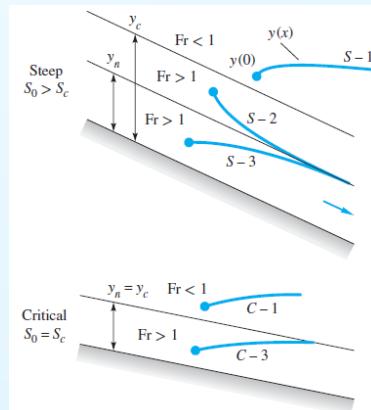
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Gradually Varied Flow (GVF)

12 basic solution curves:

Should be solved numerically

$$\frac{dy}{dx} = \frac{S_0 - S_f}{1 - Fr^2}, Fr^2 = \frac{Q^2 b_0}{gA^3}, S_f = \frac{n^2}{\kappa^2} \frac{Q^2}{R_h^{4/3} A^2}$$

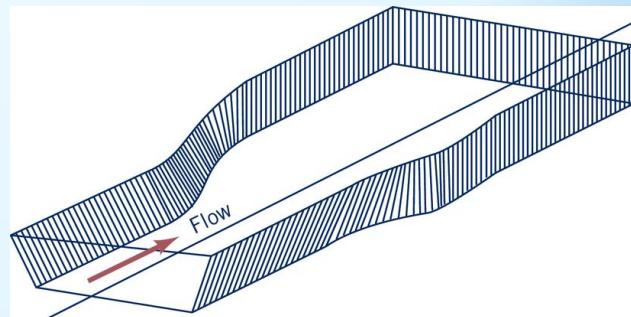


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Rapidly Varied Flow (RVF)

- 2D or 3D

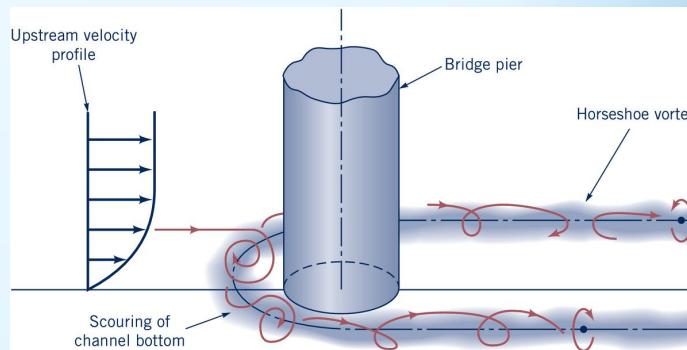


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Rapidly Varied Flow (RVF)

- 2D or 3D



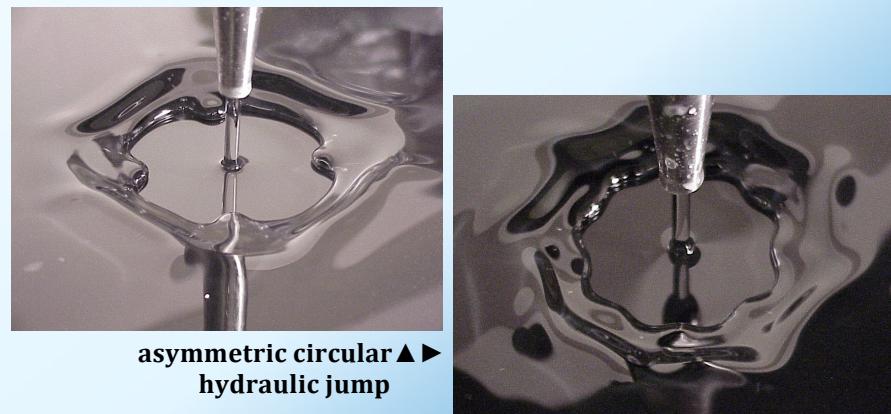
The flow around a bridge pier ▲

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Rapidly Varied Flow (RVF)

- 2D or 3D

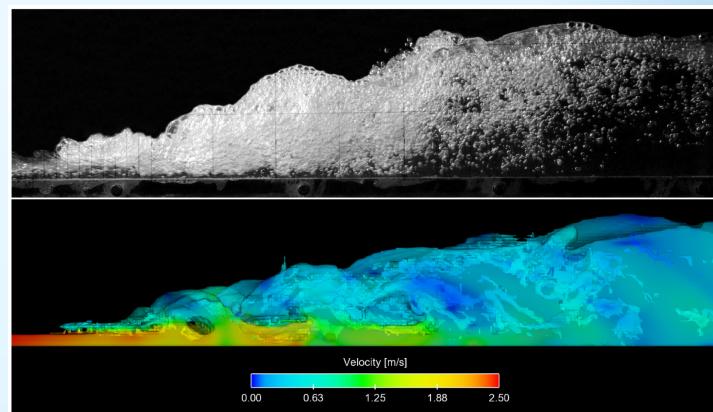


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Rapidly Varied Flow (RVF)

- 2D or 3D
- 1D



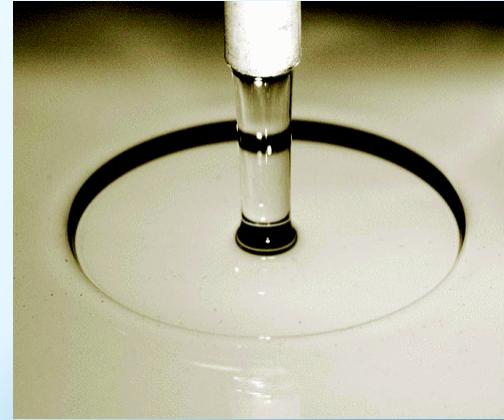
Planar hydraulic jump ▲

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Rapidly Varied Flow (RVF)

- 2D or 3D
- 1D



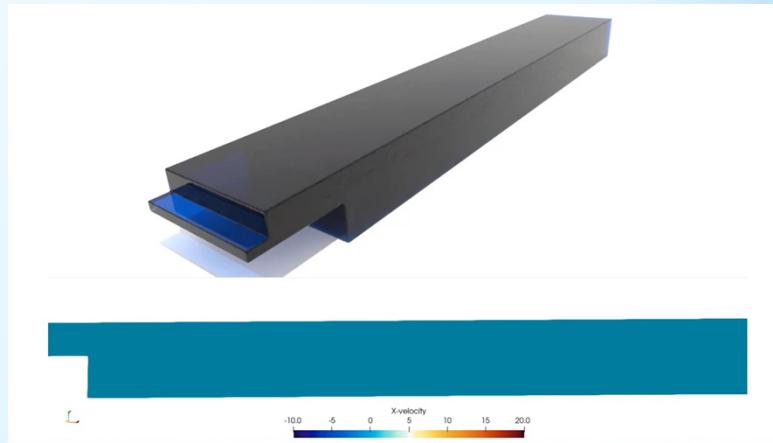
Symmetric circular hydraulic jump ▲

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Rapidly Varied Flow (RVF)

- CFD

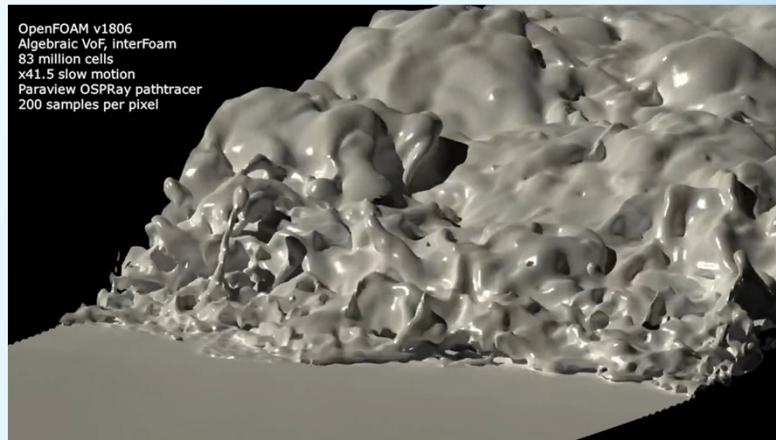


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Rapidly Varied Flow (RVF)

- CFD



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Hydraulic jump

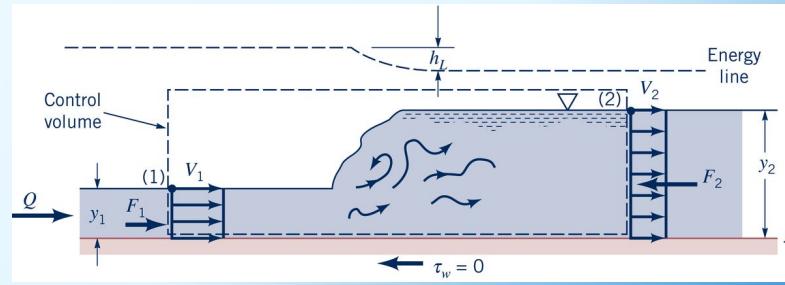


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Hydraulic jump

- A mechanism for supercritical upstream flow transition to subcritical downstream flow

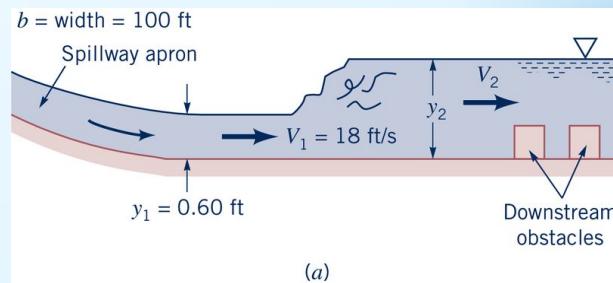


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Hydraulic jump

- Situations leading to hydraulic jump:
- Spillway + obstacle



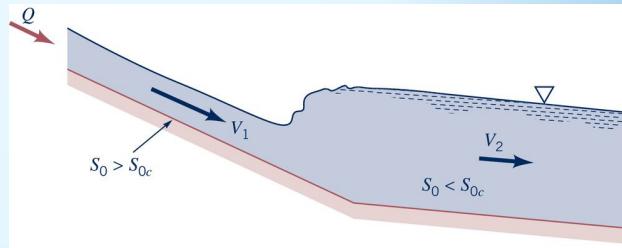
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Hydraulic jump

• Situations leading to hydraulic jump:

- Spillway + obstacle
- Sudden change in channel slope



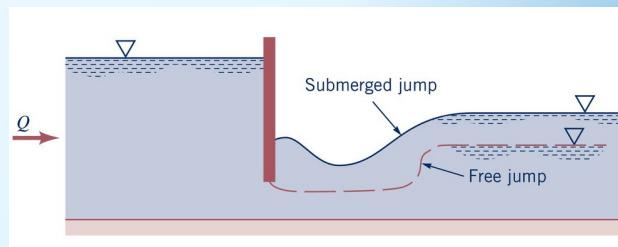
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Hydraulic jump

• Situations leading to hydraulic jump:

- Spillway + obstacle
- Sudden change in channel slope
- Sluice gate
- ...



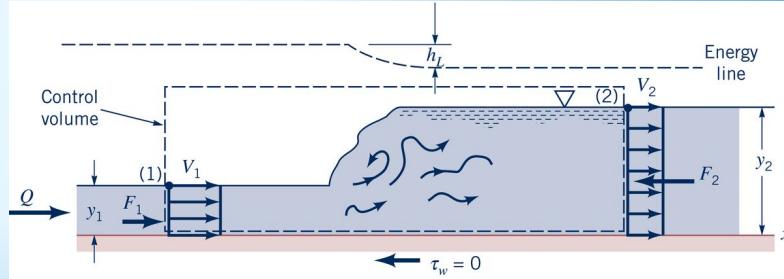
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1D hydraulic jump

- Assumptions:

- Sections 1 and 2: Steady and 1D flow
- Rectangular channel of constant width (b)



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1D hydraulic jump

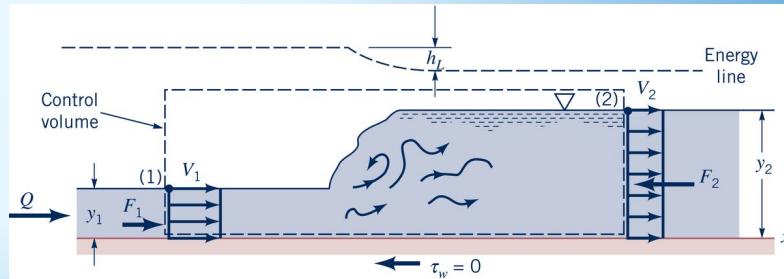
- Finite control volume approach:

- Momentum:

$$F_1 - F_2 = \rho Q(V_2 - V_1) = \rho V_1 y_1 b(V_2 - V_1)$$

$$F_1 = p_{c1}A_1 = \gamma y_1^2 b/2 \text{ and } F_2 = p_{c2}A_2 = \gamma y_2^2 b/2, \text{ where } p_{c1} = \gamma y_1/2 \text{ and } p_{c2} = \gamma y_2/2$$

$$\frac{y_1^2}{2} - \frac{y_2^2}{2} = \frac{V_1 y_1}{g} (V_2 - V_1)$$



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1D hydraulic jump

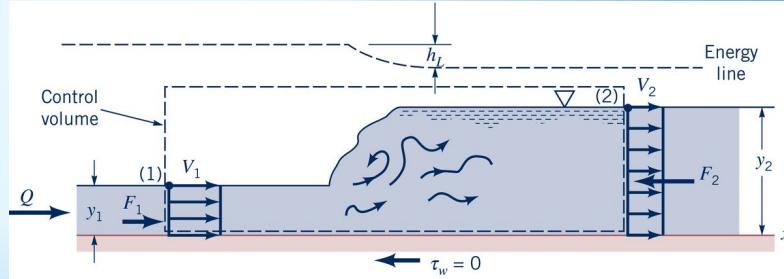
- Finite control volume approach:

Momentum: $\frac{y_1^2}{2} - \frac{y_2^2}{2} = \frac{V_1 y_1}{g} (V_2 - V_1)$

Mass: $y_1 b V_1 = y_2 b V_2 = Q$

Energy: $y_1 + \frac{V_1^2}{2g} = y_2 + \frac{V_2^2}{2g} + h_L$

- 3 unknowns: y_2, V_2, h_L



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1D hydraulic jump

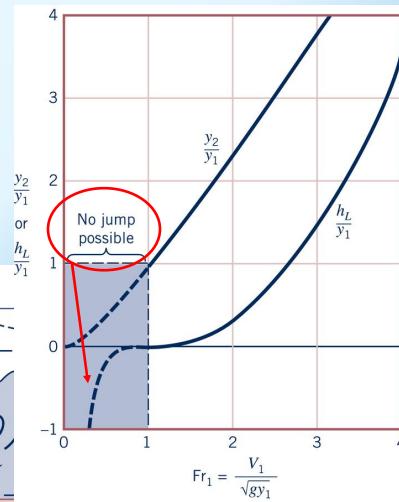
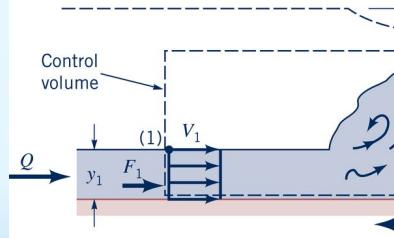
- 2 solutions:

No jump: $y_1 = y_2, V_1 = V_2$, and $h_L = 0$.

Jump:

$$\frac{y_2}{y_1} = \frac{1}{2}(-1 + \sqrt{1 + 8Fr_1^2}) \quad Fr_1 = V_1 / \sqrt{gy_1}$$

$$\frac{h_L}{y_1} = 1 - \frac{y_2}{y_1} + \frac{Fr_1^2}{2} \left[1 - \left(\frac{y_1}{y_2} \right)^2 \right]$$



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1D hydraulic jump

- Exercise: Show that the Hydraulic jump relation can be written as a function of upstream Froude number as:

$$Fr_1 = V_1 / \sqrt{gy_1}$$

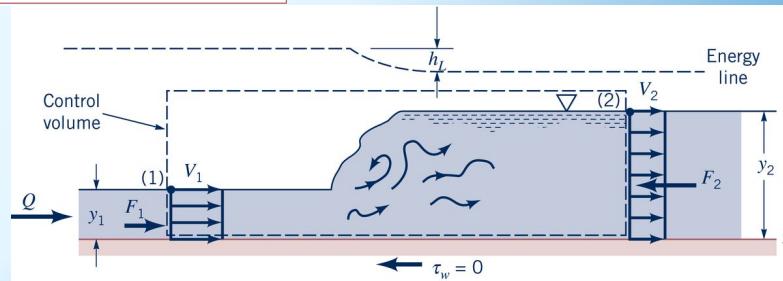
$$\frac{y_2}{y_1} = \frac{1}{2}(-1 + \sqrt{1 + 8Fr_1^2})$$

$$\frac{h_L}{y_1} = 1 - \frac{y_2}{y_1} + \frac{Fr_1^2}{2} \left[1 - \left(\frac{y_1}{y_2} \right)^2 \right]$$

$$Fr_2 = V_2 / \sqrt{gy_2}$$

$$\frac{y_2}{y_1} = \frac{1}{4} \left(\frac{1 + \sqrt{1 + 8Fr_2^2}}{Fr_2^2} \right)$$

$$\frac{h_L}{y_1} = 1 - \frac{y_2}{y_1} + \frac{Fr_2^2}{2} \left[\left(\frac{y_2}{y_1} \right)^3 - \frac{y_2}{y_1} \right]$$

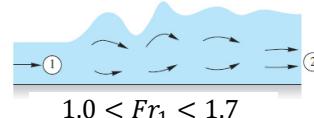


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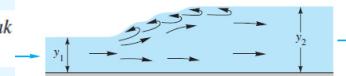
Hydraulic jump classification

Standing-wave or *undular jump* about $4y_2$ long; low dissipation, less than 5 percent.

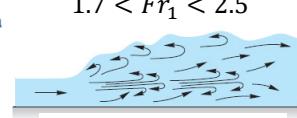


$$1.0 < Fr_1 < 1.7$$

Smooth surface rise with small rollers, known as a *weak jump*; dissipation 5 to 15 percent.

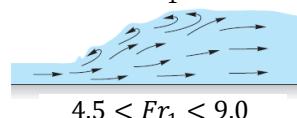


Unstable, *oscillating jump*; each irregular pulsation creates a large wave that can travel downstream for miles, damaging earth banks and other structures. Not recommended for design conditions. Dissipation 15 to 45 percent.



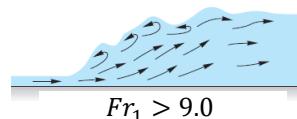
$$2.5 < Fr_1 < 4.5$$

Stable, well-balanced, *steady jump*; best performance and action, insensitive to downstream conditions. Best design range. Dissipation 45 to 70 percent.



$$4.5 < Fr_1 < 9.0$$

Rough, somewhat intermittent *strong jump*, but good performance. Dissipation 70 to 85 percent.



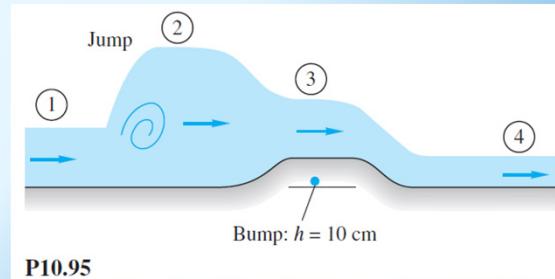
$$Fr_1 > 9.0$$

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Example

P10.95 A 10-cm-high bump in a wide horizontal water channel creates a hydraulic jump just upstream and the flow pattern in Fig. P10.95. Neglecting losses except in the jump, for the case $y_3 = 30$ cm, estimate (a) V_4 , (b) y_4 , (c) V_1 , and (d) y_1 .



P10.95

→ Notes: VIII.3

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Analogy between Compressible and Open-Channel Flows

Open-channel flow

Small-amplitude
gravity waves

Froude number $Fr = V/c_{oc}$

$$ybV = cte$$

depth y

width b

Compressible flow

Sound waves

Mach number $Ma = V/c$

$$\rho AV = cte$$

density ρ

area A

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The end of chapter 8

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