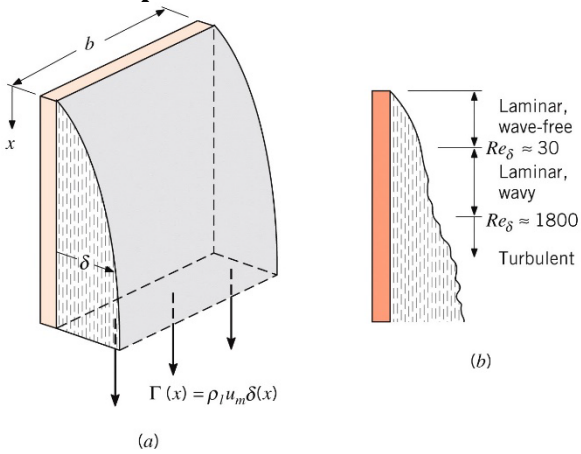
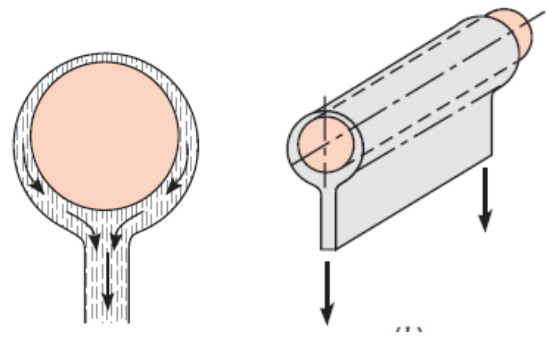
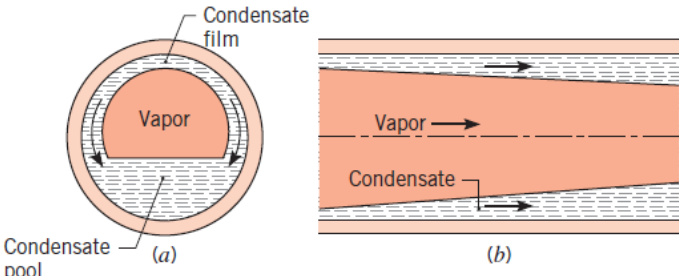


## Chapter 5

Table 1: Film condensation <sup>a</sup>

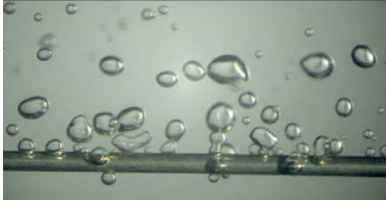

Geometry	Recommended Correlation	Restrictions
<b>1-Vertical plate <sup>b</sup></b>  <p>Figure 10.12 © John Wiley &amp; Sons, Inc. All rights reserved.</p>	$h'_{fg} = h_{fg} + 0.68c_{p,l}(\bar{T}_{sat} - T_s)$ <b>Vertical</b> (10.32)  (10.42)-(10.45)  <b>Inclined</b> above + ( $g \rightarrow g \cos \theta$ )	Laminar, $Ja < 0.1$ , $1 \leq Pr \leq 100$ , $\rho_l \gg \rho_v$ , $\rho_l \gg \rho_v$ .
<b>2-Spheres &amp; cylinders (external condensation)</b> 	(10.46) $h'_{fg} = h_{fg} + 0.68c_{p,l}(\bar{T}_{sat} - T_s)$ Sphere: $C = 0.826$ Cylinder: $C = 0.729$  <b>vertical tier of N horizontal unfinned tubes</b> (10.49), $n = -1/6$  <b>Inclined</b> above + ( $g \rightarrow g \cos \theta$ )  <b>Finned tube</b> (10.48)	$(L/D > 1.8 \tan \theta)$
<b>3-Horizontal tube (internal condensation)</b> 	(10.50)  <b>a) low vapor velocity</b> (10.46), $C = 0.555$ $h'_{fg} = h_{fg} + 0.375c_{p,l}(T_{sat} - T_s)$  <b>b) high vapor velocity</b> (10.51) <sup>c</sup>	$Re_{v,i} < 35000$  $\dot{m}/A > 500 \text{ kg/s.m}^2$

<sup>a</sup> Fluid properties at film temperature ( $T_f = (T_s + T_{sat})/2$ ), and  $h_{fg}$  and vapor properties ( $\rho_v$ ) at  $T_{sat}$

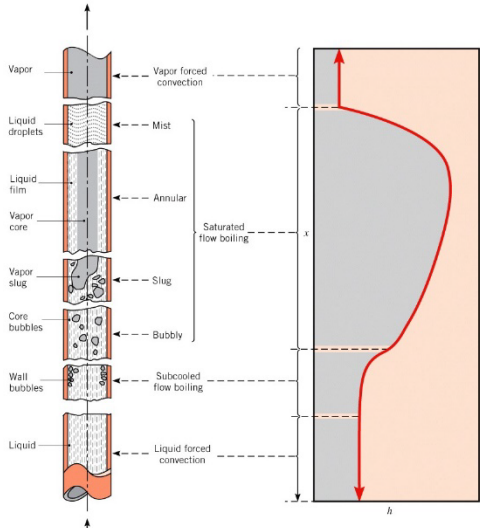
<sup>b</sup> May be used for condensation on the inner or outer surface of a vertical tube of radius  $R \gg \delta$ .

<sup>c</sup> All properties at  $T_{sat}$

**Table 2: Pool boiling**

Geometry	Recommended Correlation	Restrictions
<b>1-Free convection boiling</b> ( $\Delta T_e \leq \Delta T_{e,A}$ ) $\Delta T_{e,A} \sim 5^\circ\text{C}$	<b>Natural convection relations</b>	
<b>2-Nucleate boiling<sup>a</sup></b> ( $\Delta T_{e,A} \leq \Delta T_e \leq \Delta T_{e,C}$ ) $\Delta T_{e,C} \sim 30^\circ\text{C}$ 	(10.5) and Table 10.1  <b>Critical heat flux</b> (10.6) Large horizontal plate: $C = 0.149$ Sphere, cylinder, and others: $C = 0.131$  <b>Minimum heat flux</b> (10.7) $C = 0.09$	Clean surface  $Bo^{-1/2} > 0.2$  moderate pressures
<b>3-Film boiling<sup>b</sup></b> ( $\Delta T_e \geq \Delta T_{e,D}$ ) $\Delta T_{e,D} \sim 120^\circ\text{C}$ 	(10.8) $h'_{fg} = h_{fg} + 0.80c_{p,v}(T_s - T_{sat})$ Horizontal cylinder: $C = 0.62$ Sphere: $C = 0.67$  <b>Including Radiation (<math>T_s \geq 300^\circ\text{C}</math>)</b> (10.9)-(10.11)	

<sup>a</sup> All properties at  $T_{sat}$ <sup>b</sup> Vapor properties at system pressure and film temperature ( $T_f = (T_s + T_{sat})/2$ ), and  $\rho_l$  and  $h_{fg}$  at  $T_{sat}$ **Table 3: Forced convection boiling**

Geometry	Recommended Correlation	Restrictions
<b>1- Flow over a cylinder in cross-flow</b>	critical heat flux: (10.12)-(10.14)	
<b>2-Flow in tubes<sup>a</sup></b> 	<b>single-phase forced convection region</b> (8.62) <sup>a</sup>  <b>saturated flow boiling region</b> max (10.15a, 10.15b), (10.16) and Table 10.2 $Fr \lesssim 0.04$ , $f(Fr) = 2.63 Fr^{0.3}$ and otherwise, $f(Fr) = 1$ $Fr = (\dot{m}''/\rho_l)^2/gD$ $h_{sp}$ from (8.62) <sup>a</sup>	turbulent flow  turbulent flow $Co = \sqrt{\sigma/(g[\rho_l - \rho_v])}/\tilde{D}_h \lesssim 1/2$ smooth tubes

<sup>a</sup> All properties at  $T_{sat}$