

Massachusetts Institute of Technology
2.004 Dynamics and Control II

Lab Section: Tues 3-5pm

Name: Elissa Ito

Collaborators (if any):

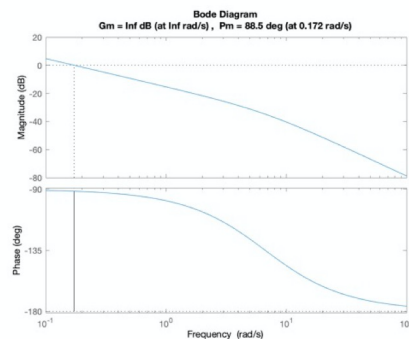
Lab 7 – Frequency Domain Control Design

Prelab

$$T = 0.15 \text{ s} \quad K_{dc} \approx 8.78 \quad K = 0.172$$

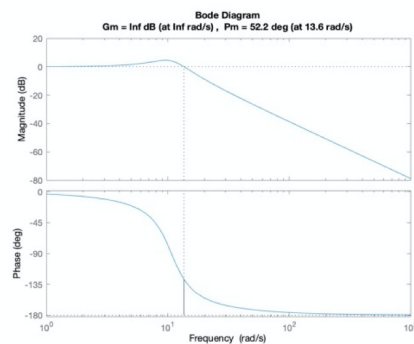
$$G(s) = \frac{\Theta(s)}{V_c(s)} = \frac{K}{\tau s^2 + s} = \frac{0.172}{0.15s^2 + s}$$

a)



phase margin = 88.5 deg
crossover freq. = 0.172 rad/sec

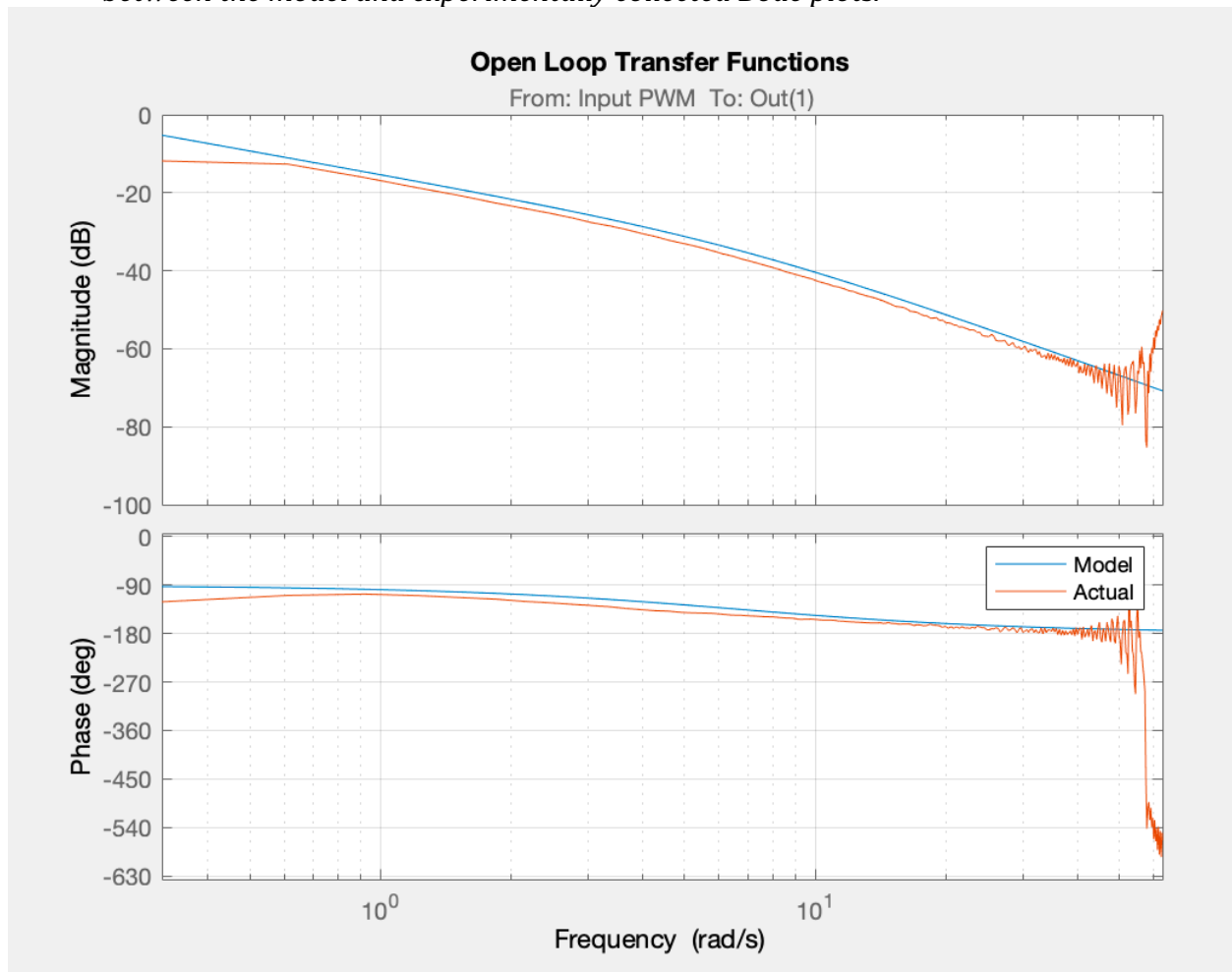
$$b) \quad G_{cl}(s) = \frac{\frac{0.172}{0.15s^2 + s} \cdot 100}{1 + \frac{0.172}{0.15s^2 + s} \cdot 100} = \frac{17.2}{17.2 + 0.15s^2 + s}$$



bandwidth = 15.4765

Experiment #1: Open and Closed Loop Bode Plots:

1. *Attach open loop Bode plots and locate the plant poles. Explain the discrepancies between the model and experimentally collected Bode plots.*



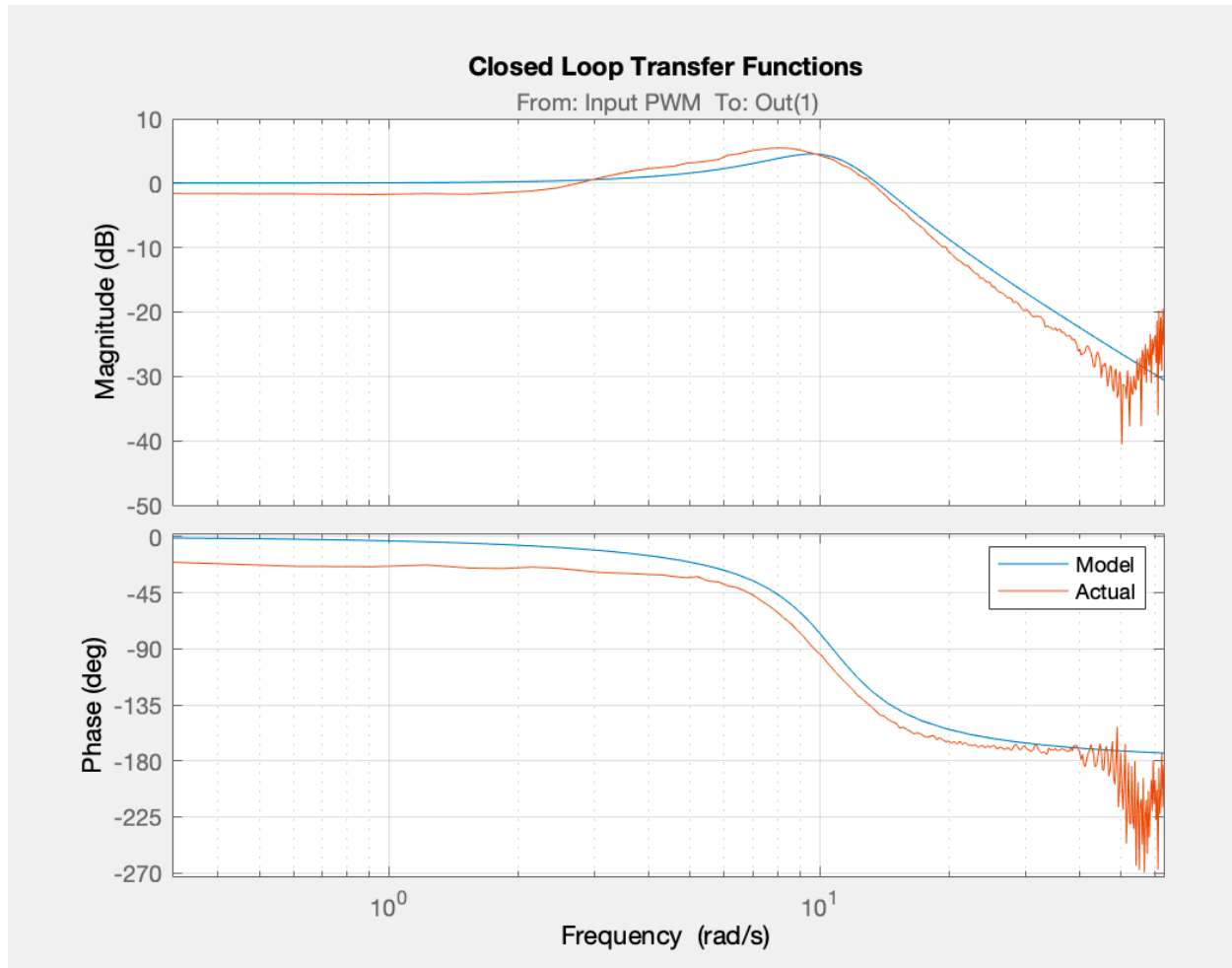
$$0.172/0.15s^2+s$$

Poles at $s=0$, -6.6667 (where phase @-90, @-135)

Discrepancy at high and low frequencies – low frequency is deadband and high frequency is oversaturated.

2. *Attach closed loop Bode plots with a proportional controller with $K_p = 100$. Explain the discrepancies between the model and measured bode plots.*

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Discrepancy at high and low frequencies – low frequency is deadband and high frequency is oversaturated.

3. When the experiment and model do not agree, is this due to model, or the way we collected data? Explain your reasoning.

The way we collected data; the friction in the flywheel causes deadband and saturation and low and high frequencies respectively.

Experiment #2. Controller Design:

Time domain step response specifications: overshoot 7% or less, and 2% settling time of 0.4 seconds or less.

$$\%OS \leq 7 \Rightarrow \zeta \geq 0.646$$

$$PM \approx 100 * \zeta = 64.6$$

$$0.4 = 4 / (\zeta * \omega_n) \Rightarrow \omega_n = 15.48$$

$$\omega_c \approx \omega_n = 15.48$$

$$\phi_{max} = 64.6 - 27.5 = 37.1$$

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1. What values of ϕ_m and ω_c should we use to achieve the required time domain performances?

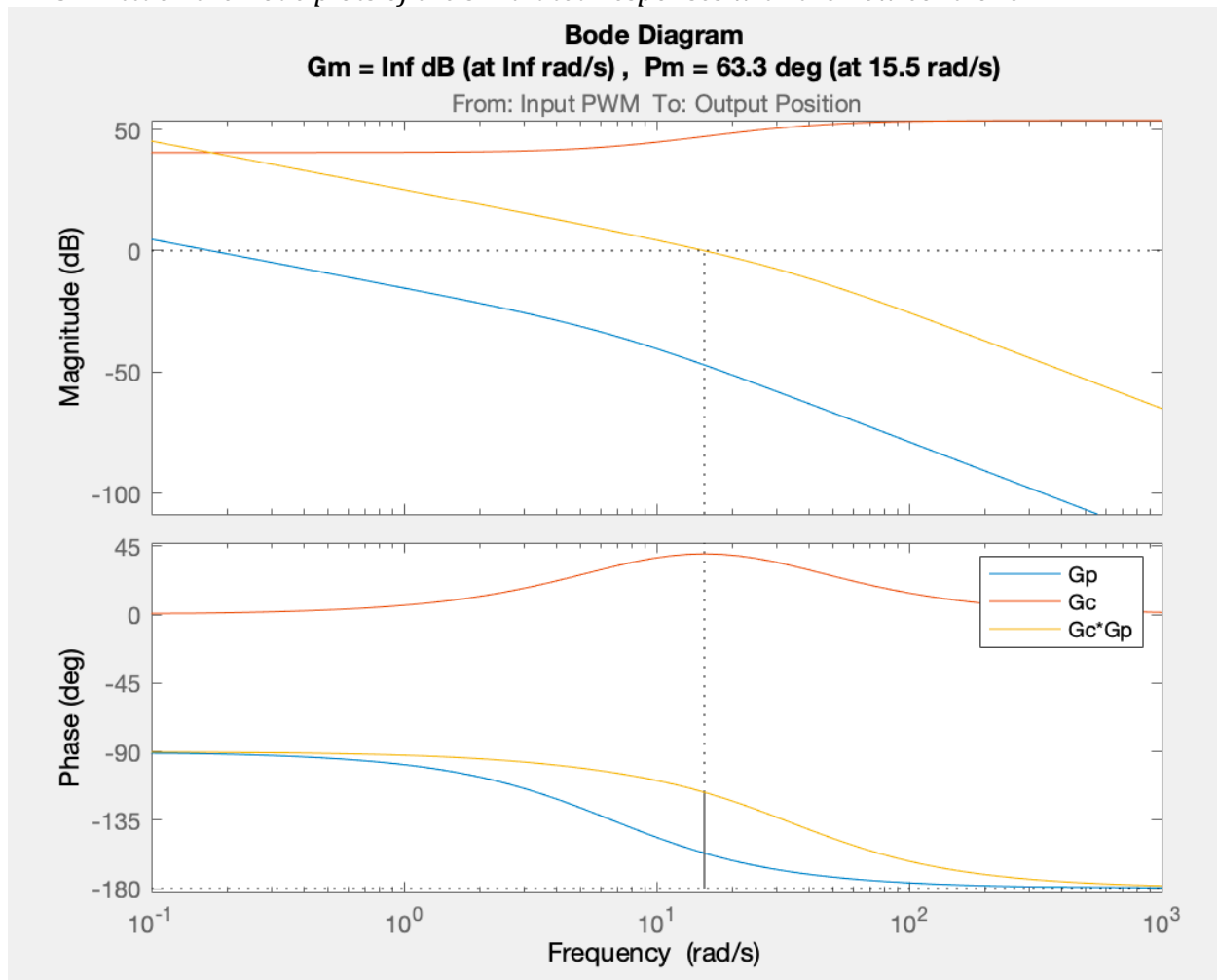
$$PM = 64.6$$

$$\omega_c = 15.48$$

2. Transfer function of your lead controller:

$$G_c(s) = 487.51(s+7.218)/(s+33.2) = 106.01(0.14s+1)/(0.03s+1)$$

3. Attach the Bode plots of the simulated responses with the new controller.

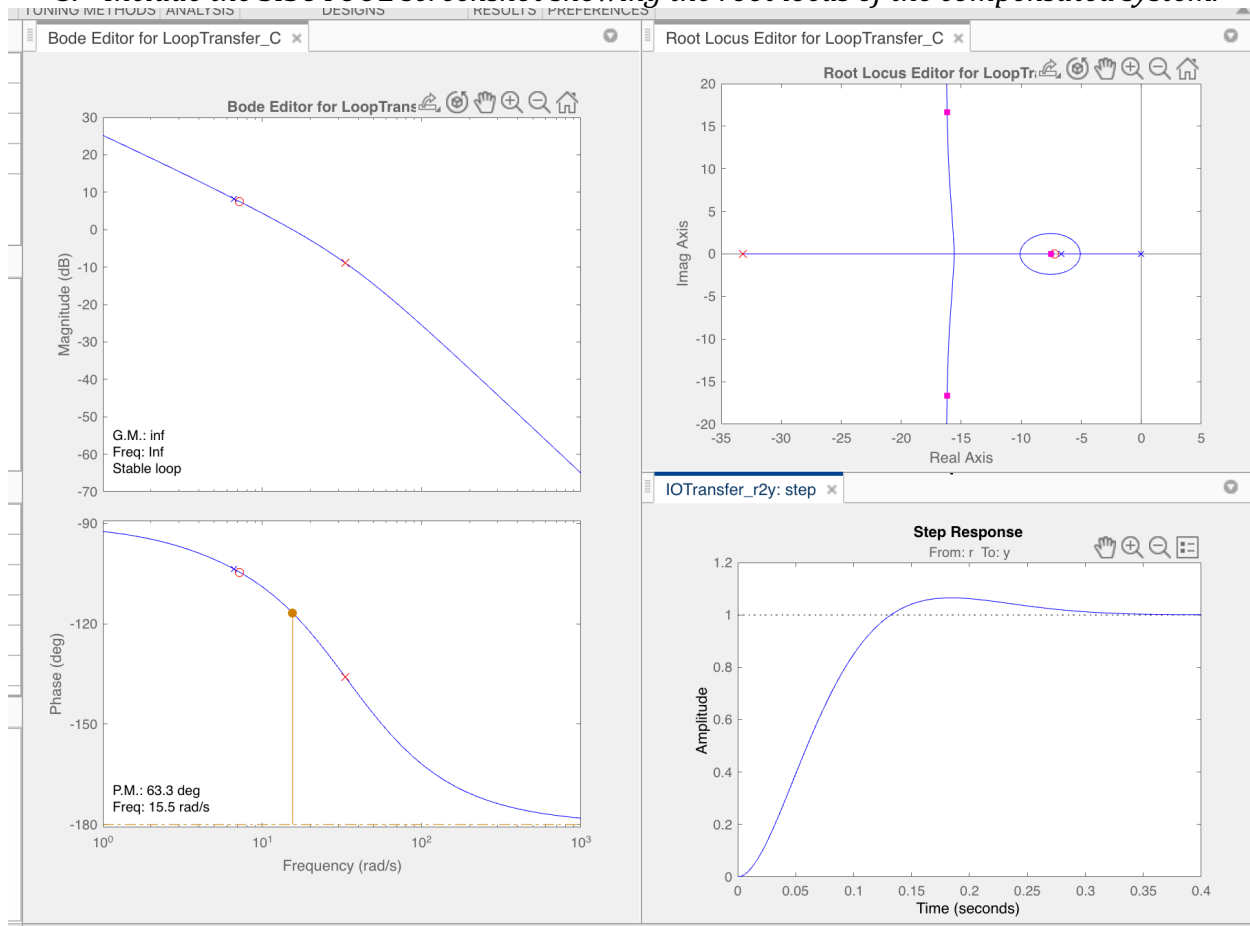


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4. Find the closed-loop step response characteristics by running the MATLAB command:
`stepinfo(feedback(Gc*Gp,1))`

RiseTime: 0.0872
SettlingTime: 0.2777
SettlingMin: 0.9138
SettlingMax: 1.0651
Overshoot: 6.5078
Undershoot: 0
Peak: 1.0651
PeakTime: 0.1850

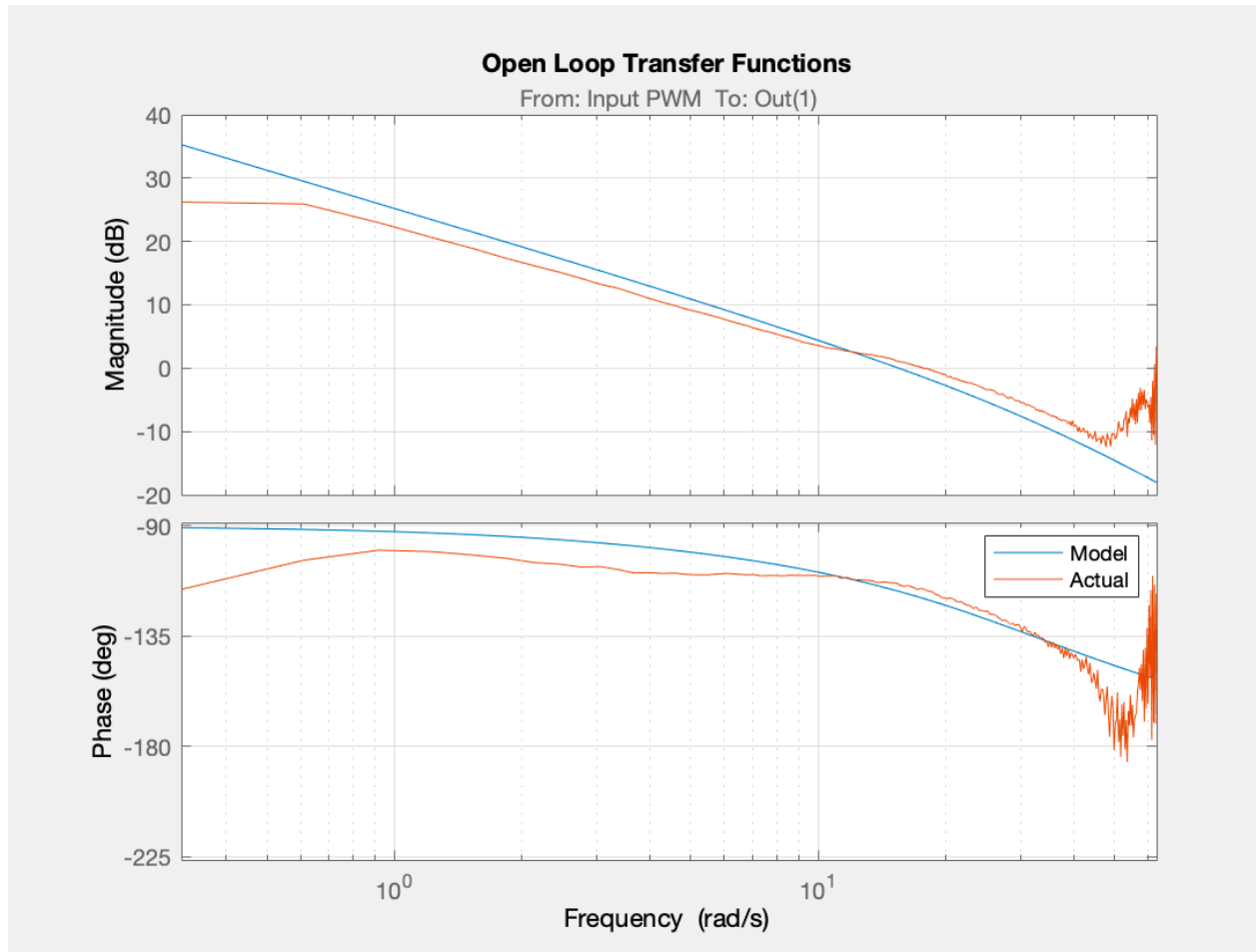
5. Include the SISOTOOL screenshot showing the root locus of the compensated system.



Experiment #3. Controller Testing:

1. Attach open loop Bode plots for the compensated system, $(G_c G_p)$ and determine the new crossover frequency and phase margin from the actual data.

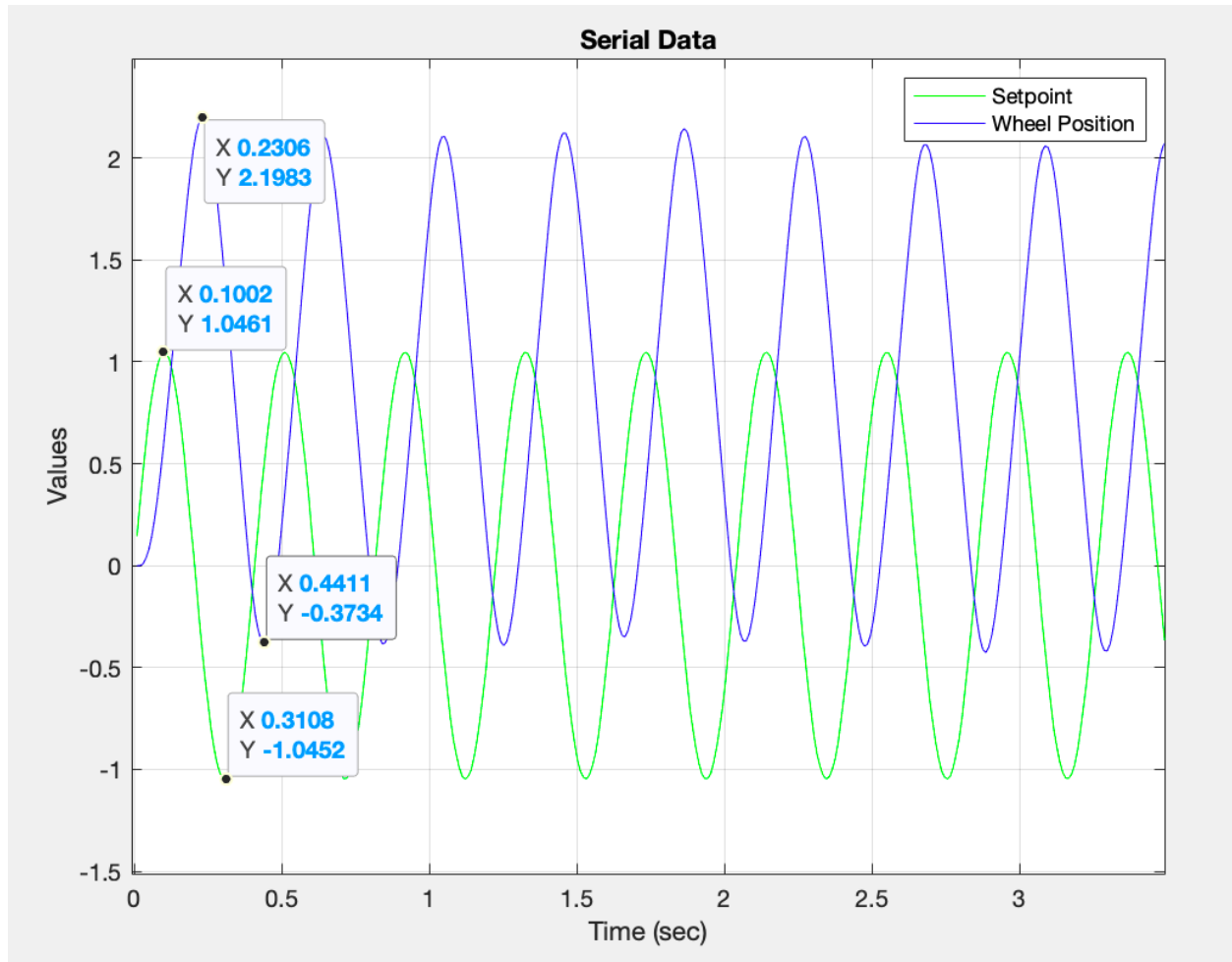
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Crossover Freq = 15.4
Phase Margin = 64

2. Attach the sine wave plot when driven by a frequency equals to the crossover frequency, and include the phase and magnitude information for the crossover frequency. Do they match with the values determined from the open loop Bode plots?

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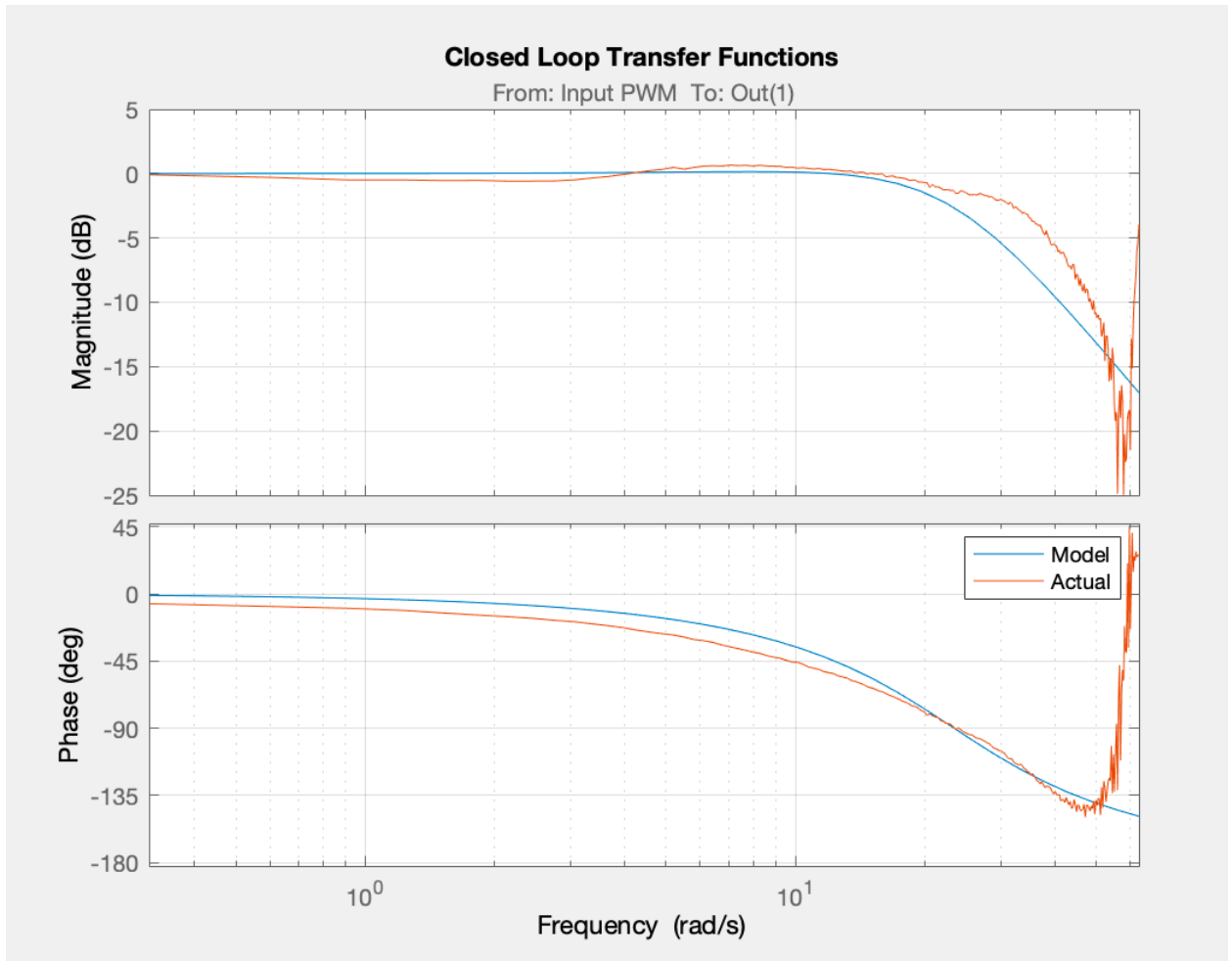


$$\text{Gain} = (2.1983 + 0.3734) / (1.0461 + 1.0452) = 1.23 = 1.798\text{dB}$$

$$\text{Phase} = 360 * (0.1002 - 0.2306) * 2.451 = -115.06\text{deg}$$

3. Attach the closed loop Bode plots and estimate the closed loop bandwidth.

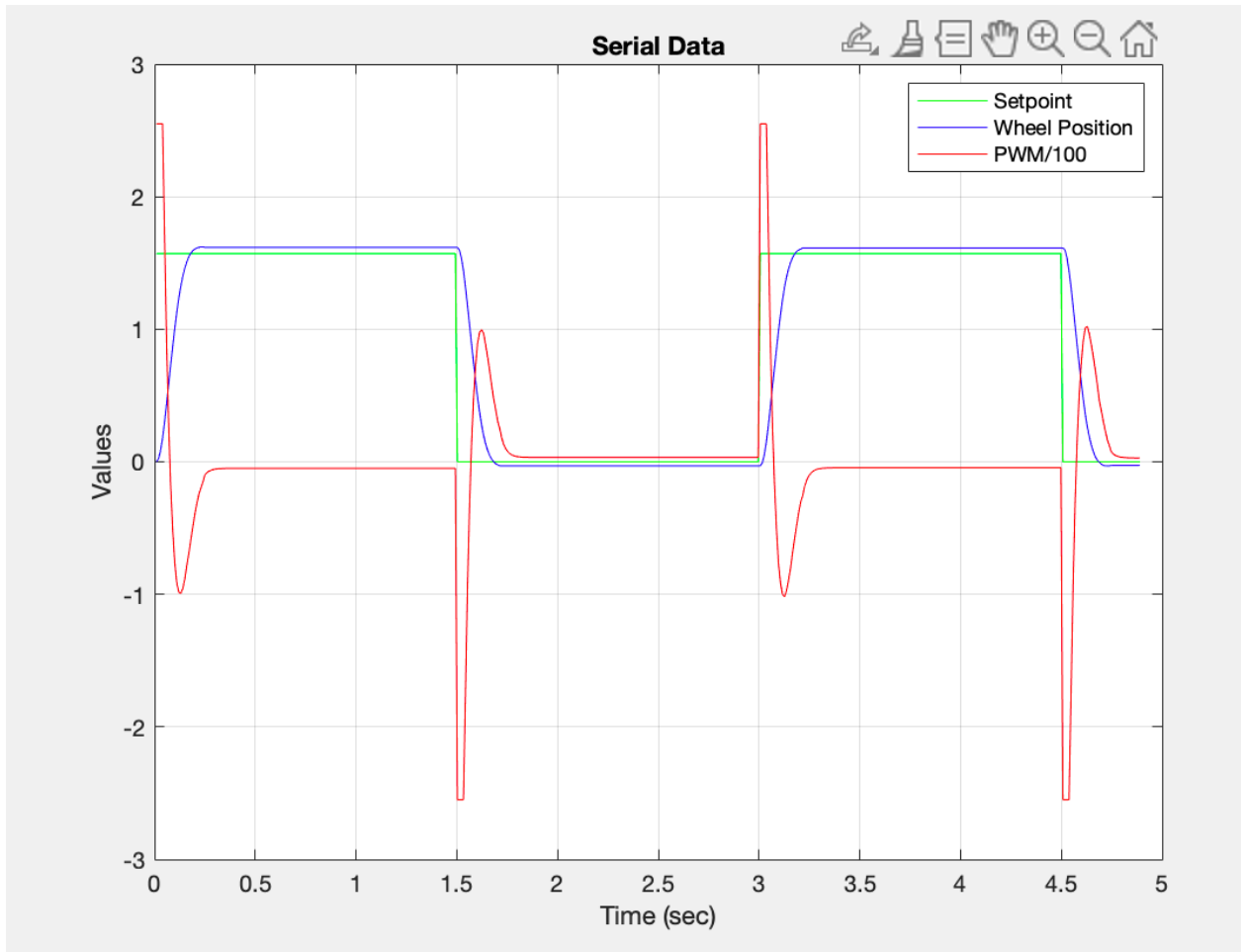
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Bandwidth = 33.5

4. *Attach the step response of the actual closed loop system. Measure the actual percent overshoot and settling time.*

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$$\%OS = 3.7$$

$$T_s = 0.22s$$

5. Describe the effects of incorporating the lead controller onto the system.

Incorporating the lead controller caused the step response to reach the transient response a lot faster. It caused a greater overshoot.