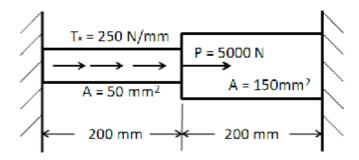
## Problem 5

Consider the bar loaded as shown below:



Assume  $E = 200 \,\text{GPa}$  and that the bar is fixed at both ends.

- a) Construct a 1D linear bar finite element model of the bar. Use two elements in each section of the bar (4 elements in total). Label all nodes and elements.
- b) Write the global system of equations  $[K]\{d\} = \{R\}.$
- c) Apply the boundary conditions to this global system of equations and solve for  $\{d\}$ .
- d) Plot the displacements u(x) vs. x for the entire bar.
- e) What are the reaction forces at the two ends?

Note: All calculations performed in MATLAB. See appendix?? for details.

(a) Figure 1 shows the 1D linear bar finite element model of the bar given in the problem statement. The model has 4 elements and 5 nodes, with the known tractions, forces, and boundary conditions shown in the figure.

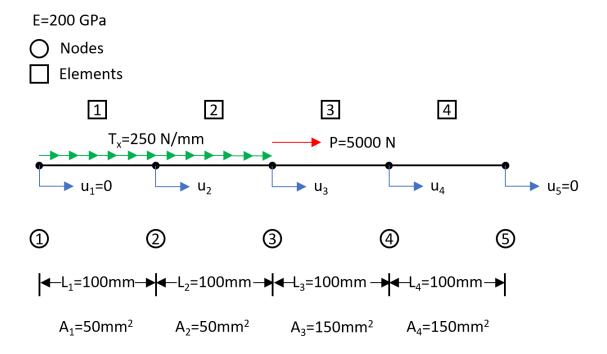


Figure 1: 1D linear bar finite element model of bar.

(b) Let  $k_i = \frac{EA_i}{L_i}$ . We assemble a local stiffness matrix  $[K_i]$  for each element i as shown below:

$$[K_{1}] = \begin{bmatrix} k_{1} & -k_{1} \\ -k_{1} & k_{1} \end{bmatrix}$$

$$[K_{2}] = \begin{bmatrix} k_{2} & -k_{2} \\ -k_{2} & k_{2} \end{bmatrix}$$

$$[K_{3}] = \begin{bmatrix} k_{3} & -k_{3} \\ -k_{3} & k_{3} \end{bmatrix}$$

$$[K_{4}] = \begin{bmatrix} k_{4} & -k_{4} \\ -k_{4} & k_{4} \end{bmatrix}$$

Through superposition we combine the local stiffness matrices into a global stiffness matrix, [K]:

Next, we assemble our displacement vector, noting that  $d_1 = d_5 = 0$ :

$$\{d\} = \begin{cases} d_1 \\ d_2 \\ d_3 \\ d_4 \\ d_5 \end{cases} = \begin{cases} 0 \\ d_2 \\ d_3 \\ d_4 \\ 0 \end{cases}$$

Then, the vector of applied forces:

$$\{r\} = \begin{cases} F_1 \\ F_2 \\ F_3 \\ F_4 \\ F_5 \end{cases}$$

 $F_1$  and  $F_5$  are unknown because we know that the displacements at both boundaries are 0. The applied forces at nodes 2-4 are outlined below:

$$F_2 = T_x * L_1 + P$$

$$F_3 = T_x * (L_1 + L_2) + P$$

$$F_A = P$$

$$\{r\} = \begin{Bmatrix} F_1 \\ T_x * L_1 \\ T_x * (L_1 + L_2) + P \\ P \\ F_5 \end{Bmatrix} = \begin{Bmatrix} F_1 \\ 30000 \\ 55000 \\ 5000 \\ F_5 \end{Bmatrix}$$

The global system of equations  $[K]\{d\} = \{r\}$ :

$$\begin{bmatrix} k_1 & -k_1 & 0 & 0 & 0 \\ -k_1 & k_1 + k_2 & -k_2 & 0 & 0 \\ 0 & -k_2 & k_2 + k_3 & -k_3 & 0 \\ 0 & 0 & -k_3 & k_3 + k_4 & -k_4 \\ 0 & 0 & 0 & -k_4 & k_4 \end{bmatrix} \begin{pmatrix} 0 \\ d_2 \\ d_3 \\ d_4 \\ 0 \end{pmatrix} = \begin{pmatrix} F_1 \\ 30000 \\ 55000 \\ 5000 \\ F_5 \end{pmatrix}$$

(c) Substituting in given values and solving in MATLAB yields the following for  $\{d\}$ :

$$\begin{cases} d_1 \\ d_2 \\ d_3 \\ d_4 \\ d_5 \end{cases} = \begin{cases} 0 \\ 0.3313 \\ 0.3625 \\ 0.1896 \\ 0 \end{cases}$$
 mm

(d) Following a similar solution procedure as in Problem 2 we solve for displacement, u(x), in MATLAB. Note, values are not shown for legibility:

$$u = a_1 + a_2 x + a_3 x^2 + a_4 x^3 + a_5 x^4$$

$$\begin{pmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \\ u_5 \end{pmatrix} = [A] \begin{pmatrix} 1 \\ x \\ x^2 \\ x^3 \\ x^4 \end{pmatrix}$$

$$A = \begin{bmatrix} 1 & x_1 & x_1^2 & x_1^3 & x_1^4 \\ 1 & x_2 & x_2^2 & x_2^3 & x_2^4 \\ 1 & x_3 & x_3^2 & x_3^3 & x_3^4 \\ 1 & x_4 & x_4^2 & x_4^3 & x_4^4 \\ 1 & x_5 & x_5^2 & x_5^3 & x_5^4 \end{bmatrix}$$

$$[N_1 \quad N_2 \quad N_3 \quad N_4 \quad N_5] = [1 \quad x \quad x^2 \quad x^3 \quad x^4] [A]^{-1}$$

$$\{u\} = [N] \{d\}$$

Figure 2 shows the displacement u across the finite element bar model. Nodes 1-5 are marked with a red star. The curve obeys the given boundary conditions with  $u(x_1) = u(x_5) = 0$ .

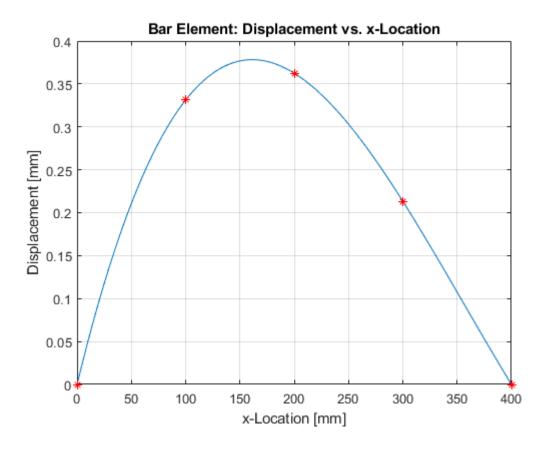


Figure 2: Displacement u vs. x location

Finite Element Analysis I

(e) Solving the global  $[K]\{d\} = \{r\}$  equation in MATLAB yields the following for  $\{r\}$ :

$$\{r\} = \begin{cases} F_1 \\ F_2 \\ F_3 \\ F_4 \\ F_5 \end{cases} = \begin{cases} -33125 \\ 30000 \\ 55000 \\ 5000 \\ -56875 \end{cases}$$
 N

The reaction forces at the ends are given by  $F_1$  and  $F_5$ :

$$F_1 = -33125 \,\mathrm{N}$$
  $F_5 = -56875 \,\mathrm{N}$