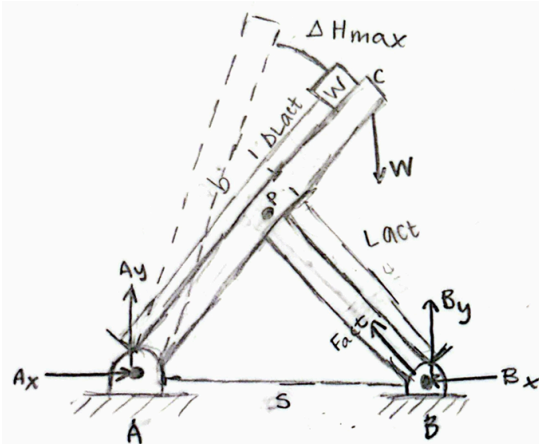


ENGRD 2020 Portfolio: Mechanism Including an Actuator

Problem Definition: The goal of my mechanism is to lift the maximum possible weight (W) to the highest vertical displacement using a linear actuator, within a fixed 2D working space of 150 cm width, and 50cm height. The system contains two ground-mounted pinned supports at points A and B, a rigid lifting bar, and a single rod-type linear actuator. All supports and the actuator are assumed rigid.

Sketch of design:



Design objectives:

- Maximize vertical lift height (ΔH)
- Maximize allowable lifting weight
- Satisfy actuator force limits (Rod-Style Actuator 58kN)
- Operate within spatial constraints.

Constraints:

- Design window: 150cm x 50cm
- Actuator maximum force: 58kN
- Horizontal distance between supports: $s = 0.3\text{m}$
- Bar length: $b = 1\text{m}$
- Actuator stroke: 1.5m

Degrees of Freedom:

The system behaves as a single degree of freedom linkage controlled by actuator stroke. The supports are pinned and the actuator imposes one displacement input, therefore only one geometric parameter controls lifting height.

Static Analysis:

Treating the member as a rigid bar, a vertical lifting reaction is produced by the actuator force projected vertically:

$$W_{\max} = F_{\text{act}} \sin(\Phi).$$

With $F_{\max} = 58\text{kN}$ and geometry $b = 1\text{m}$, $s = 0.3\text{m}$:

$$\Phi = \tan^{-1}(b/s) = \tan^{-1}(3.33) = 73^\circ$$

$$W_{\max} = (58\text{kN})(\sin(73^\circ)) = 55\text{kN}$$

Therefore, the rigid-body upper bound load is approximately 55kN.

Step 2:

The bar is now treated as a beam, not perfectly rigid. It deflects under the payload weight and the actuator induced transverse load.

Assumptions:

- Linear elastic Euler-Bernoulli beam
- Small deflections
- Transverse loading only
- Simply supported between A-B
- Point load at distance $b=1\text{m}$
- Material: steel, $E = 200\text{ GPa}$

Analysis:

For a simply supported beam of length b under mid-span point load:

$$\delta_{\max} = Wb^3/48EI$$

Assuming $W=55\text{kN}$ and the rectangular section has a width of 20mm and a height of 40mm :

$$I = bh^3/12 = (0.02)(0.04)^3/12 = 1.07 \times 10^{-7} \text{ m}^4$$

$$\delta_{\max} = 6.7\text{mm}$$

This maximum deflection meets the requirement $\delta_{\max} < 0.02b = 20\text{mm}$.

Final Beam Design:

Rectangular steel beam made of structural steel:

- $E=200\text{ GPa}$
- $b=1\text{m}$
- Cross-section: 40mm height x 20mm width

The heavier dimension is oriented vertically to maximize the area moment of inertia and reduce deflection.

Sketch:

