

The shallow water equations

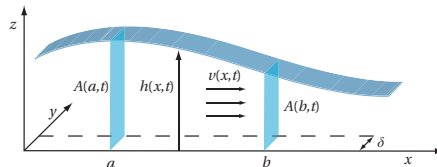
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The shallow water equations

- Examples:
 - Coastal flows; effects of storm surges e. g.
 - Flows in dams and rivers
 - Tsunami modeling
- Water-flow case in which **vertical dimensions** \ll **horizontal dimensions**
- Here: shallow and narrow **channel** of water; constant density ρ

Assumptions



- (a) **Narrow, straight channel; width δ constant.** Horizontal movement only in x direction. Everything constant in y direction. Vertical cross section area:

$$A(x, t) = h(x, t)\delta \quad (1)$$

- (b) **Shallow channel:** Horizontal dynamics dominates. Vertical dynamics ignored:

- Ignoring vertical velocity: $\mathbf{v}(\mathbf{x}, t) = (v(x, t), 0, 0)$
- $v(x, t)$ represents a **depth-averaged** velocity
- Effects of air pressure ignored
- Hydrostatic water pressure in the vertical direction:

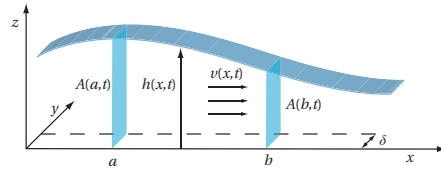
$$p(x, z, t) = \rho g(h(x, t) - z) \quad \text{for } z \leq h \quad (2)$$

- (c) Friction forces at bottom and walls ignored

The shallow water equations

- The shallow water equations: a system of 2 conservation laws in $h(x, t)$ and $v(x, t)$
- If not narrow channel, 1 additional conservation law for additional horizontal velocity component
- These equations derived from conservation of mass + conservation of momentum + relation (2).

Mass conservation



- Water mass between a and b :

$$m(t) = \int_a^b \rho A(x, t) dx \quad (3)$$

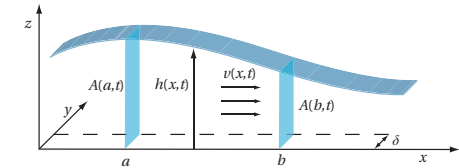
- Mass flux at $x = a$ in positive x direction:

$$\rho v(a, t) A(a, t) \quad [\text{kg/s}] \quad (4)$$

- Mass conservation and expressions (3), (4):

$$\begin{aligned} \frac{d}{dt} m(t) &= \frac{d}{dt} \int_a^b \rho A(x, t) dx \\ &= \rho v(a, t) A(a, t) - \rho v(b, t) A(b, t) = - \int_a^b \frac{\partial}{\partial x} (\rho v A) dx \\ \Rightarrow \int_a^b \left(\frac{\partial}{\partial t} (\rho A) + \frac{\partial}{\partial x} (\rho v A) \right) &= 0 \end{aligned}$$

Mass conservation



$$\int_a^b \left(\frac{\partial}{\partial t} (\rho A) + \frac{\partial}{\partial x} (\rho v A) \right) = 0$$

Recall (1): $A(x, t) = h(x, t)\delta$ and that ρ, δ constant \Rightarrow

$$\int_a^b \left(\frac{\partial}{\partial t} h + \frac{\partial}{\partial x} (vh) \right) = 0$$

$a < b$ arbitrarily \Rightarrow

$$\frac{\partial}{\partial t} h + \frac{\partial}{\partial x} (vh) = 0 \quad (5)$$

First conservation law done!

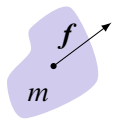
Momentum balance

Momentum balance for rigid bodies = Newton's 2nd law:

$$m\mathbf{a} = \mathbf{f}$$

Generalization for changing mass:

$$\frac{d}{dt} (m\mathbf{v}) = \mathbf{f}$$

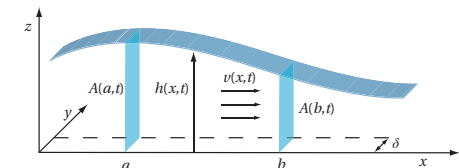


- Momentum $\mathbf{r} = m\mathbf{v}$ is a **vector**
- An applied force changes momentum (direction and/or magnitude)
- In 1D, the momentum r can be positive or negative
- (Compare the above with mass and energy conservation!)

Force on a body V with boundary ∂V from an exterior pressure field p :

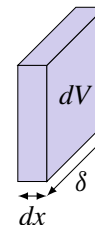
$$\mathbf{f} = - \int_{\partial V} p \mathbf{n} dS = - \int_{\partial V} p d\mathbf{S}$$

Momentum balance



Momentum balance law for region $a < x < b$:

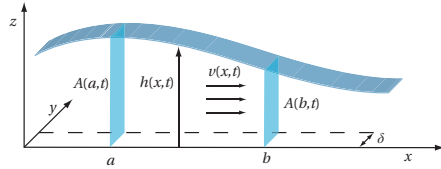
$$\frac{dr}{dt} = [\text{mom. inflow at } a] - [\text{mom. outflow at } b] + [\text{force on region}]$$



- Infinitesimal volume $dV = A(x, t) dx = h(x, t)\delta dx$
- Momentum in dV : $dr = \rho v A dx = \rho v h \delta dx$
- Momentum in region $a < x < b$:

$$r(t) = \int_a^b dr = \int_a^b \rho v A dx = \int_a^b \rho v h \delta dx \quad (6)$$

Momentum flux



Momentum flux is $\underbrace{\text{momentum density}}_{\frac{[\text{momentum}]}{\text{m}^3}} \times \underbrace{\text{volume flux}}_{\frac{\text{m}^3}{\text{s}}}$

$$\begin{aligned} [\text{mom. inflow at } a] &= \underbrace{\rho v(a,t)}_{\frac{[\text{momentum}]}{\text{m}^3}} \underbrace{A(a,t)v(a,t)}_{\frac{\text{m}^3}{\text{s}}} = \underbrace{\rho v(a,t)^2 A(a,t)}_{\frac{[\text{momentum}]}{\text{s}}} \\ &= \rho v(a,t)^2 h(a,t)\delta \end{aligned} \quad (7a)$$

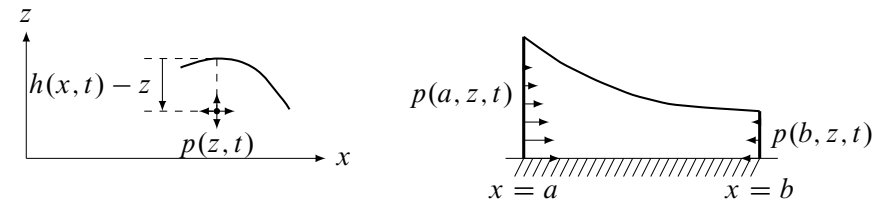
$$[\text{mom. outflow at } b] = \rho v(b,t)^2 A(b,t) = \rho v(b,t)^2 h(b,t)\delta \quad (7b)$$

Forces

- Only forces from water pressure (ignoring friction)
- Hydrostatic water pressure in vertical direction:

$$p(x, z, t) = \rho g(h(x, t) - z)$$

- Forces on region $a < x < b$ from pressures at $x = a$ and $x = b$



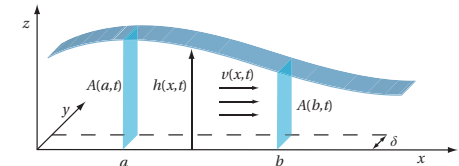
Forces

Hydrostatic water pressure: $p(x, z, t) = \rho g(h(x, t) - z)$

Integrated pressure forces on vertical surface $A(x, t) = h(x, t)\delta$:

$$\begin{aligned} \int_0^\delta \int_0^h p(x, z, t) dz dy &= \delta \int_0^h p(x, z, t) dz = \delta \int_0^h \rho g(h - z) dz \\ &= -\delta \rho g \frac{(h - z)^2}{2} \Big|_{z=0}^{z=h} = \delta \rho g \frac{h(x, t)^2}{2} \end{aligned} \quad (8)$$

Momentum balance



Recall:

$$\frac{dr}{dt} = [\text{mom. inflow at } a] - [\text{mom. outflow at } b] + [\text{force on region}]$$

Equations (6), (7), (8) \Rightarrow

$$\begin{aligned} \frac{d}{dt} \int_a^b \rho v h \delta dx &= \rho v^2(a, t) h(a, t) \delta - \rho v^2(b, t) h(b, t) \delta \\ &\quad + \delta \rho g \frac{h(a, t)^2}{2} - \delta \rho g \frac{h(b, t)^2}{2} \\ &= - \int_a^b \left[\frac{\partial}{\partial x} (\rho v^2 h \delta) + \frac{\delta \rho g}{2} \frac{\partial}{\partial x} h^2 \right] dx \end{aligned}$$

The final system

δ, ρ constant \Rightarrow

$$\int_a^b \left[\frac{\partial}{\partial t} v h + \frac{\partial}{\partial x} v^2 h + \frac{g}{2} \frac{\partial}{\partial x} h^2 \right] dx = 0$$

$a < b$ arbitrary \Rightarrow conservation law

$$\frac{\partial}{\partial t} v h + \frac{\partial}{\partial x} v^2 h + \frac{g}{2} \frac{\partial}{\partial x} h^2 = 0$$

Recall mass balance (5):

$$\frac{\partial}{\partial t} h + \frac{\partial}{\partial x} (v h) = 0$$

Defining $q = v h$ (δq is the volume flux), the system can be written

$$\frac{\partial}{\partial t} \begin{pmatrix} h \\ q \end{pmatrix} + \frac{\partial}{\partial x} \begin{pmatrix} h \\ q^2/h + g h^2/2 \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \end{pmatrix}$$

Is the system hyperbolic?

$$\frac{\partial}{\partial t} \begin{pmatrix} h \\ q \end{pmatrix} + \frac{\partial}{\partial x} \begin{pmatrix} h \\ q^2/h + g h^2/2 \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \end{pmatrix}$$

Can be written

$$\mathbf{u}_t + \mathbf{f}_x = \mathbf{0},$$

where

$$\mathbf{u} = \begin{pmatrix} h \\ q \end{pmatrix}, \quad \mathbf{f} = \begin{pmatrix} f_1 \\ f_2 \end{pmatrix} = \begin{pmatrix} h \\ q^2/h + g h^2/2 \end{pmatrix}$$

Hyperbolic if the Jacobian

$$\mathbf{Df} = \begin{pmatrix} \frac{\partial f_1}{\partial h} & \frac{\partial f_1}{\partial q} \\ \frac{\partial f_2}{\partial h} & \frac{\partial f_2}{\partial q} \end{pmatrix}$$

is **diagonalizable** with **real eigenvalues**

Is the system hyperbolic?

$$f_1 = q$$

$$f_2 = \frac{q^2}{h} + \frac{g}{2} h^2$$

$$\mathbf{Df} = \begin{pmatrix} \frac{\partial f_1}{\partial h} & \frac{\partial f_1}{\partial q} \\ \frac{\partial f_2}{\partial h} & \frac{\partial f_2}{\partial q} \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ -q^2/h^2 + gh & 2q/h \end{pmatrix}$$

Characteristic equation for eigenvalues λ :

$$\begin{aligned} |\lambda \mathbf{I} - \mathbf{Df}| &= \begin{vmatrix} \lambda & -1 \\ q^2/h^2 - gh & \lambda - 2q/h \end{vmatrix} \\ &= \lambda \left(\lambda - \frac{2q}{h} \right) + \frac{q^2}{h^2} - gh = 0 \end{aligned}$$

$$\lambda = \frac{q}{h} \pm \sqrt{\frac{q^2}{h^2} - \frac{q^2}{h^2} + gh} = \frac{q}{h} \pm \sqrt{gh} \quad (9)$$

Is the system hyperbolic?

Recall $q = v h \Rightarrow$

$$\lambda = v \pm \sqrt{gh}$$

$h \geq 0 \Rightarrow$ real eigenvalues!

$h > 0 \Rightarrow$ separate eigenvalues \Rightarrow two linearly independent eigenvectors \Rightarrow diagonalizable **Df**

Thus, system is hyperbolic for $h > 0$