## TTK4190 Guidance and Control of Boats

## Assignment 2, Pt. 3

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### Problem 1 - Propeller Revolution and Speed Control

**a**)

The propeller thrust and torque coefficients  $K_T$  and  $K_Q$  can be computed by using the wageningen function found in the MSS toolbox.

```
1 Ja = 0; % advance number
2 PD = 1.5; % pitch/diameter ratio
3 AEAO = 0.65; % blade area ratio
4 z = 4; % number of propeller blades
5 [KT,KQ] = wageningen(Ja,PD,AEAO,z);
```

#### b)/c)

 $\tau=0$  and looking at the figure  $Y=\frac{Q_d}{K_m}$  Converting equation to time domain

$$\frac{Q_m}{Y} = \frac{K_m}{T_m s + 1} e^{-\tau s}$$

$$\mathcal{L}^{-1}(Q_m T_m s + Q_m = Q_d)$$

$$\dot{Q}_m T_m + Q_m = Q_d$$

$$\dot{Q}_m = -\frac{Q_m}{T_m} + \frac{Q_d}{T_m}$$

$$= -\frac{1}{T_m} * Q_m + \frac{K_m}{T_m} * Y$$

which is implemented in ship.m as follows

```
% Part 3, 1b,1c) compute torque-based shaft dynamics here
         = 10^5;
         = 0.6;
                                         % (s^-1)
  Km
         = 10;
  \operatorname{Tm}
                                         % (s)
         = 0;
                                         % time delay (s)
  thr = rho * Dia^4 * KT * abs(n) * n;
                                     % thrust command (N)
  Q = rho * Dia^5 * KQ * abs(n) * n;
                                     % torque command
9
  persistent Qm
11
      if isempty(Qm)
12
         Qm = 0;
      end
14
15
  Qd = rho*Dia^5*KQ*abs(n_c)*n_c;
16
  Qm_dot = (Qd - Qm)/Tm;
17
18
  Y = Qd/Km;
                                       % control input
19
20
21
  A = -1/Tm;
                                       % transfer function to state space
  B = Km/Tm;
22
23
  Qm\_dot = A * Qm + B * Y;
24
25
  Qm = euler2(Qm_dot,Qm,h);
27
         = (1/Im) * (Qm - Q);
  n_dot
28
```

Thus, the formula for the shaft speed dynamics  $\dot{n}$  is found.

d)

The linear surge equation for the surge subsystem from [1] can be expressed as

$$U = \frac{(t-1)T}{X_u}$$

through this derivation:

$$(m - X_{\dot{u}})\dot{u}_r - X_u u_r = -X_{\delta\delta}\delta^2 + (1 - t)T$$

$$\dot{u}_r = \delta = 0$$

$$\downarrow$$

$$-X_u u_r = (1 - t)T$$

$$u_r = U$$

$$\downarrow$$

$$U = \frac{(t - 1)T}{X_u}$$

The assumptions are that the Boat must not accelerate and the rudder angle must be zero ( $\dot{u}_r = 0, \, \delta = 0$ ).

**e**)

The desired thrust  $T_d$  can be computed from d) as

$$T_d = \frac{U_{ref} X_u}{(t-1)} \tag{1}$$

and propeller revolution  $n_d$  as

$$n_d = sgn(T_d)\sqrt{\frac{|T_d|}{\rho D^4 K_T(0)}}$$
 (2)

```
1 function n_c = open_loop_speed_control(U_ref)
   % Ship variables
                        = 17.0677e6;
                                             % mass (kg)
                        = 17.067/e6; % mass (kg)
= -8.9830e5; % added mass in surge
  Xudot
                        = 20;
                                             % linear damping time constant
                        = - (m-Xudot) /T1;
                                           % linear damping in surge
   % Propeller variables
                        = 3.3;
                                             % propeller diameter (m)
  Dia
10 rho
                        = 1025;
                                             % density of water (m/s^3)
                        = 0;
= 1.5;
11
   Ja
                                             % advance number
                                             % pitch/diameter ratio
12 PD
13 AEAO
                        = 0.65;
                                             % blade area ratio
                        = 4; % numbe
= wageningen(Ja,PD,AEAO,z);
                                             % number of propeller blades
14
15
   [KT, \neg]
16
                        = 0.05;
                                             % thrust deduction number p. 164 in Fossen
17
18
   % Combining the equation in task d) and inserting T (eq. 9.7 in Fossen)
19
                = U_ref * Xu / ((t-1) *rho * Dia^4 * KT);
   n_c_squared
20
                        = sign(n_c_squared) * sqrt(abs(n_c_squared));
   end
```

The Boat does not reach the desired velocity when maintaining a constant heading angle, as we are not utilizing closed-loop control to regulate speed.

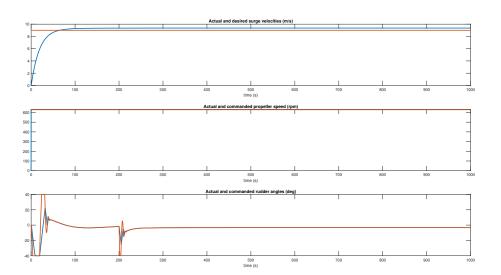


Figure 1: The result of open-loop speed control

# References

[1] T. Fossen, Handbook of Marine Craft Hydrodynamics and Motion Control. John Wiley & Sons, 2011.