${\bf Description:}$

Natural frequency of a square plate is analyzed and compared.

${\bf Reference:}$

NAFEMS Manual. Solution Retrieved from Ansys verification problem (VMP09-T12).

Material and Geometric data:

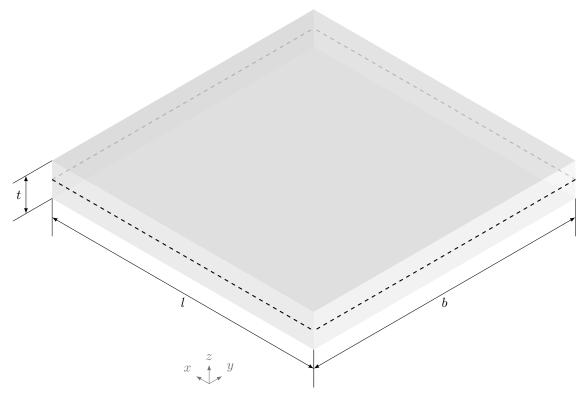


Figure 1: VMP09

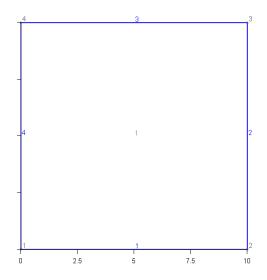
Table 1: Input Data

| Material Property | | Geometric Data | | Loading Data | |
|--------------------------|---------------|-----------------|----------|--------------|--|
| Young's Modulus (E) | 2E11 pa | Length (l) | 10 m | | |
| Poission's Ratio (ν) | 0.3 | Breath (b) | $10 \ m$ | Nil | |
| Density (ρ) | $8000~Kg/m^3$ | Thickness (t) | 0.05~m | | |

Mesh and boundary condition :

Table 2: FEM and Boundary condition data

| Direchlet Boundary | | | Neumann Boundary | | | | |
|--------------------|------|------------|------------------|-------------|-------|-------|-------|
| Geo -Entity | w | θ_x | θ_y | Geo -Entity | F_z | M_x | M_y |
| line {1,2,3,4} | Free | Free | Free | Nil | | | |



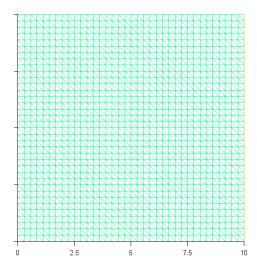


Figure 2: Geomentry and Mesh of TIM68

Analytically solution:

Retrieved Natural frequencies from reference manuals are

 $\mathrm{Mode}\ 4 = 1.622\ Hz$

Mode $5 = 2.360 \; Hz$

Mode $6 = 2.922 \; Hz$

 $\mathrm{Mode}\ 7 = 4.233\ Hz$

Mode $8 = 4.233 \; Hz$

 $\mathrm{Mode}\ 9 = 7.416\ Hz$

Result and error analysis:

The Natural modes obtained are plotted in the below figures.

For each natural frequency the error percentage is

Mode 4=0.0018~%

 $Mode\ 5=0.0042\ \%$

Mode 6 = 0.061 %

Mode 7 = 0.911 %

Mode 8 = 0.902 %

 $\mathrm{Mode}\ 9 = 0.645\ \%$

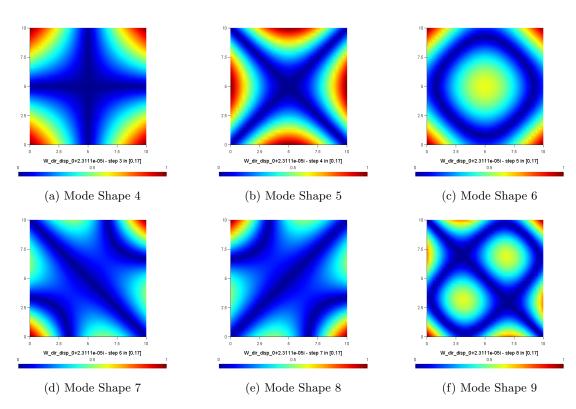


Figure 3: Natural Modes of a Square Plate