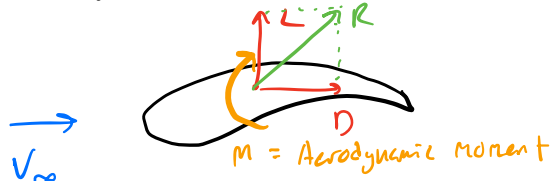


Lecture 4

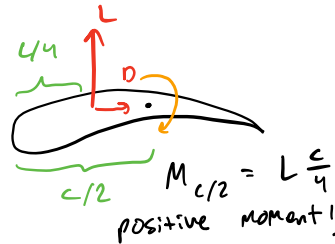
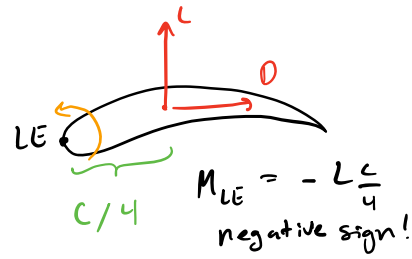
Aerodynamic Moments



Sign convention:

⤵ + positive moment
wing tends to pitch up

⤵ - negative moment
wing tends to pitch down

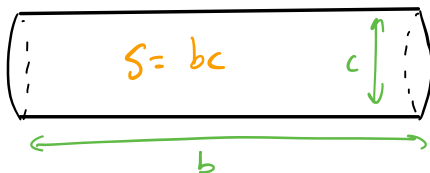
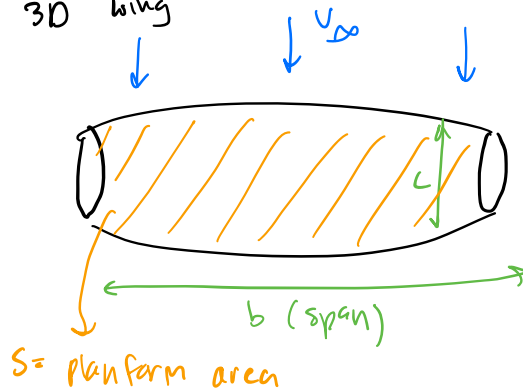


- neglect moment along chord line (Drag)

Lecture 5

Non-dimensional forces

3D wing



If no taper, rectangular wing

Lift, Drag: Newtons = N = $\text{kg} \cdot \frac{\text{m}}{\text{s}^2}$

Moment: Newton-meter = N·m

What affects aerodynamic forces & moments?

- Surface area, S
as $S \uparrow$, Forces \uparrow
- Velocity, V_∞
as $V_\infty \uparrow$, Forces \uparrow
- Density, ρ_∞
as $\rho_\infty \uparrow$, Forces \uparrow

3D wing coefficients:

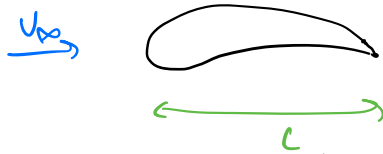
$$C_L = \frac{L}{\frac{1}{2} \rho_\infty V_\infty^2 S} \quad \text{lift coefficient}$$

$$C_D = \frac{D}{\frac{1}{2} \rho_\infty V_\infty^2 S} \quad \text{drag coefficient}$$

$$C_M = \frac{M}{\frac{1}{2} \rho_\infty V_\infty^2 S c} \quad \text{moment coefficient}$$

$$q_\infty = \frac{1}{2} \rho_\infty V_\infty^2 = \text{dynamic pressure}$$

2D wing



no planform area!

no span dimension

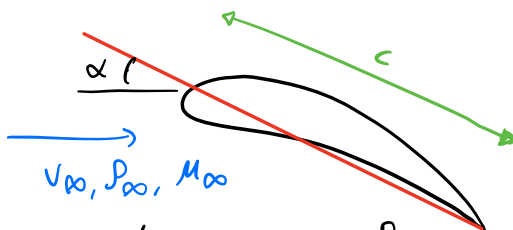
Forces defined "per unit span"

Lift / Drag: N/m

Moment: Nm/m = N

$$C_L = \frac{L}{\frac{1}{2} \rho_\infty V_\infty^2 c} ; C_D = \frac{D}{\frac{1}{2} \rho_\infty V_\infty^2 c}$$

$$C_m = \frac{M}{\frac{1}{2} \rho_\infty V_\infty^2 c^2}$$



$$C_L = \frac{L}{\frac{1}{2} \rho_\infty V_\infty^2 c} \quad C_D = \frac{D}{\frac{1}{2} \rho_\infty V_\infty^2 c}$$

$$C_L \text{ (or } C_D) = f(\underbrace{\text{airfoil shape, } \alpha, Re, M}_{\text{All non-dimensional}})$$

What parameters influence C_L & C_D ?

- Shape of airfoil: t/c , camber, LE shape, TE shape
- Angle of attack, α
- Viscosity of fluid = μ [kg/m.s]

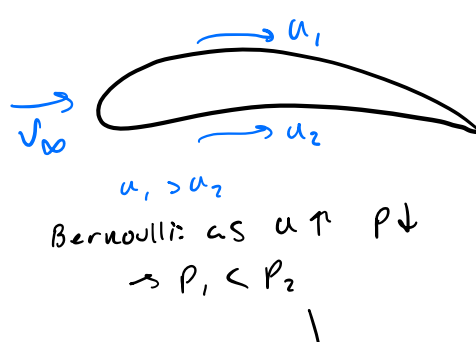
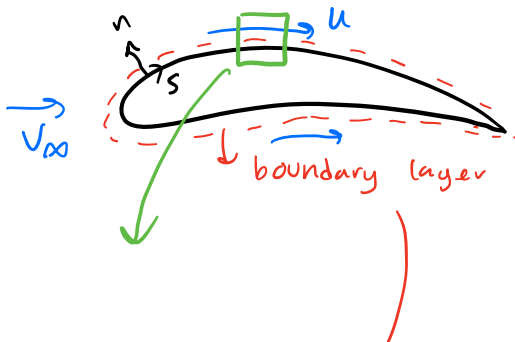
ND: Reynolds Number = $Re = \frac{\rho_\infty V_\infty c}{\mu_\infty}$

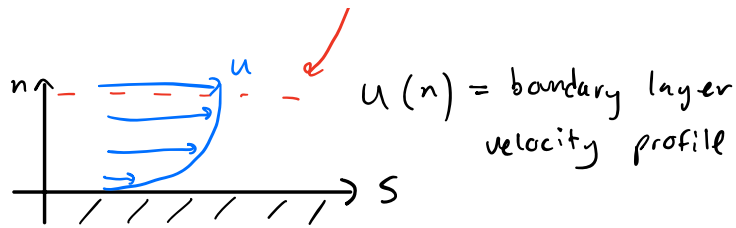
- speed of sound = a_∞

ND: Mach number = $M = \frac{V_\infty}{a_\infty}$

Lecture 6: Pressure / Shear

what creates aerodynamic forces?





↓
same units as shear stress:
pressure × area = Force
shear × area = Force

No-slip condition: $u(n=0) = 0$

Boundary layer (BL) has viscosity

BL has friction, fluid friction = shear

$$\tau_{\text{wall}} = \text{shear stress at wall} = \mu \left. \frac{\partial u}{\partial n} \right|_{\text{wall}}$$



$\tau(s)$
- always tangent to surface

$p(s)$
- always acts normal to surface

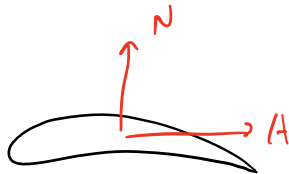
skin friction coeff.

$$C_f = \frac{\tau}{q_\infty}$$

pressure coefficient

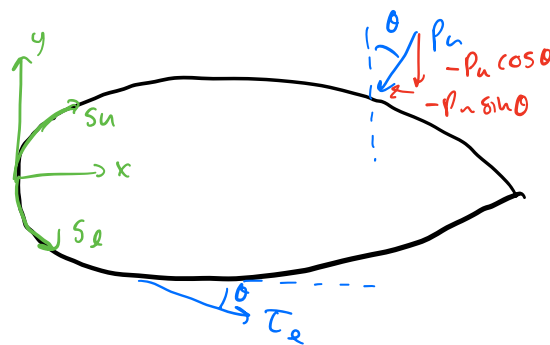
$$C_p = \frac{p - p_\infty}{q_\infty}$$

$$q_\infty = \frac{1}{2} \rho_\infty U_\infty^2$$



$$N = - \int_{LE}^{TE} (p_n \cos \theta + \tau_u \sin \theta) dS_u + \int_{LE}^{TE} (p_l \cos \theta - \tau_l \sin \theta) dS_l$$

$$A = \int_{LE}^{TE} (-p_u \sin \theta + \tau_u \cos \theta) dS_u + \int_{LE}^{TE} (p_l \sin \theta + \tau_l \cos \theta) dS_l$$



Lecture 7: Fluid flow governing equations

Navier - Stokes eqn. govern all fluid flow.

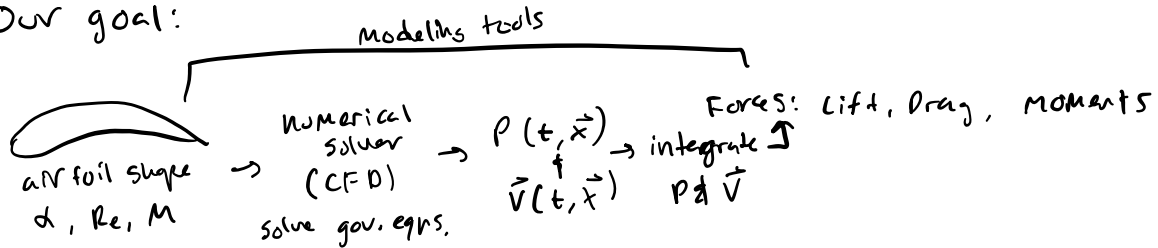
we want: $\vec{V}(t, \vec{x})$ & $P(t, \vec{x})$

Field variables

\vec{x} = position ; t = time

contour plot

Our goal:



Governing eqn. 1: cons. mass (continuity)

$$\frac{d\rho}{dt} + \nabla \cdot (\rho \vec{V}) = 0 \quad \text{full eqn w/ no assumptions}$$

$$\frac{d\rho}{dt} = 0, \text{ if flow incompressible: } \nabla \cdot \vec{V} = 0$$

If flow is 2D & planar, $\vec{V} = u \hat{i} + v \hat{j}$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

If flow is 2D & cylindrical, $\vec{V} = u_r \hat{r} + u_\theta \hat{\theta}$

$$\frac{\partial(r u_r)}{\partial r} + \frac{\partial u_\theta}{\partial \theta} = 0$$

Governing eqn. 2: cons. momentum (Navier-Stokes)

$\sum \vec{F} = m \vec{a}$ applied to a fluid particle

skip derivation

$$\sum \vec{F} = m \frac{d\vec{v}}{dt} = \frac{d(m\vec{v})}{dt} = \text{rate of change of linear momentum}$$

$$\rho \left\{ \frac{\partial u}{\partial t} + \nabla \cdot (u \vec{V}) \right\} = - \frac{\partial P}{\partial x} + \mu \nabla^2 u \Rightarrow \text{momentum in x-dir}$$

$$\rho \left\{ \frac{\partial \vec{v}}{\partial t} + \nabla \cdot (\vec{v} \vec{v}) \right\} = - \frac{\partial p}{\partial y} + \mu \nabla^2 \vec{v} \Rightarrow \text{momentum in } y\text{-dir}$$

Newtonian fluid, incompressible, 2D, planar

→ solve with continuity eqn → 3 eq, 3 unknowns (u, v, p)

Lecture 8: Stream function

Stream function, Ψ "psi" only defined for steady 2D flows

For an incompressible flow,

$$\text{stream function defined as: } \boxed{u = \frac{\partial \Psi}{\partial y} \text{ and } v = - \frac{\partial \Psi}{\partial x}}$$

$\frac{\partial [\cdot]}{\partial t} = 0$
 ↓
 planar or cylindrical

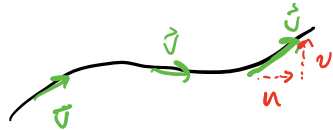
starting w/ continuity:

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0, \text{ substitute } \Psi \text{ definition} \rightarrow \frac{\partial}{\partial x} \left(\frac{\partial \Psi}{\partial y} \right) - \frac{\partial}{\partial y} \left(\frac{\partial \Psi}{\partial x} \right) = 0$$

$$\frac{\partial^2 \Psi}{\partial x \partial y} - \frac{\partial^2 \Psi}{\partial x \partial y} = 0$$

Stream function satisfies continuity:

Stream line:



tangent to velocity vectors

$$\frac{\partial \Psi}{\partial x} = \frac{v}{u}$$

$$\text{Expand } d\Psi = \underbrace{\frac{\partial \Psi}{\partial x}}_{-v} dx + \underbrace{\frac{\partial \Psi}{\partial y}}_u dy = -v dx + u dy = 0$$

If $d\Psi = 0$, then $\Psi = \text{const}$, and $v dx = u dy$
or $\frac{dy}{dx} = \frac{v}{u}$

Lines of constant Ψ are stream lines

Stream function exists for 2D incompressible flow

- check incompressible continuity = 0