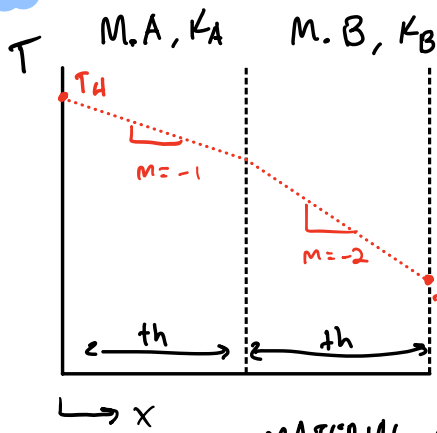


# 1) SKETCH TEMP PROFILES

## a) PLANE WALL



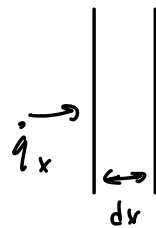
$$T(x=0) = T_h, \quad T(x=2l_h) = T_c$$

$$K_A = 2K_B$$

$$\text{FOURIER'S: } \dot{q}'' = -K \nabla T$$

$$(1) \quad \dot{q} = -K \cdot A_c \cdot \frac{dT}{dx} \quad \text{ID, AREA}$$

MATERIAL A:



$$\dot{q}_{x+dx} \therefore \dot{q}_x = \dot{q}_{x+dx}$$

$$\dot{q}_{x+dx} = \dot{q}_x + \frac{d\dot{q}_x}{dx} dx$$

TAYLOR EXPANSION

$$\therefore \frac{d\dot{q}_x}{dx} = 0 \rightarrow \dot{q}_x = \text{CONST}$$

$$\rightarrow \text{FROM EQ (1): IF } \dot{q} = \text{CONST}, A_c \text{ CONST, } \rightarrow \frac{dT}{dx} \propto \frac{1}{K}$$

$$\text{SINCE } K_A = 2 \cdot K_B,$$

$$\therefore \left. \frac{dT}{dx} \right|_A = \frac{1}{2} \cdot \left. \frac{dT}{dx} \right|_B$$

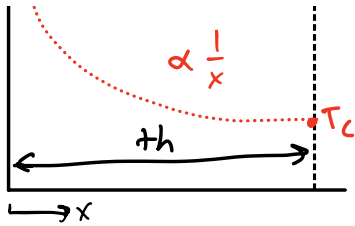
$$\frac{dT}{dx} = \text{CONST}$$

## b) PLANE WALL



$$\text{COND. VARIES, } K(x) = C_1 x + C_2 \quad | \quad C_1, C_2 > 0$$

FROM FOURIER'S LAW:



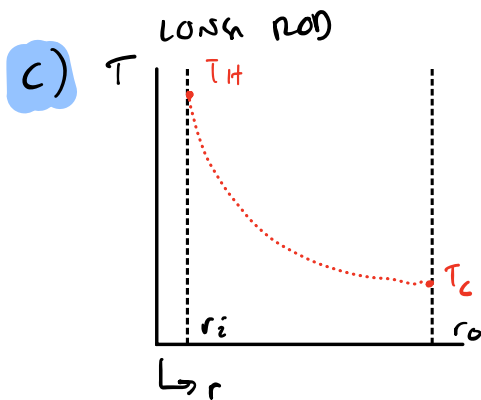
$$\dot{q} = -k(x) A_c \frac{dT}{dx} \quad (1)$$

BY EARLIER PROOF,  $\frac{d\dot{q}_x}{dx} = 0 \rightarrow \dot{q}_x = \text{CONST}$

EQ (1):  $\frac{dT}{dx} = -\frac{\dot{q}}{k(x) A_c}$

SINCE  $\dot{q}, A_c$  CONST,  $\frac{dT}{dx} \propto -\frac{1}{k(x)}$

$$\rightarrow \frac{dT}{dx} \propto -\frac{1}{c_1 x + c_2}$$



$k = \text{CONST}$

FOURIER'S:

$$\dot{q} = -k A_c \frac{dT}{dr} \quad (1)$$

BY EARLIER PROOF,  $\frac{d\dot{q}_r}{dr} = 0 \rightarrow \dot{q}_r = \text{CONST}$

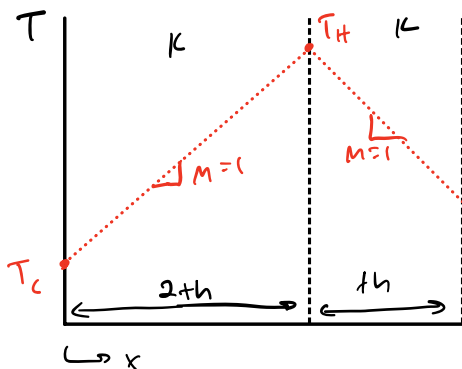
EQ (1):  $\frac{dT}{dr} = -\frac{\dot{q}_r}{k A_c}$

$A_c = 2\pi r L$

$\dot{q}_r, k, L$  CONST

$$\rightarrow \frac{dT}{dr} \propto -\frac{1}{r}$$

d) PLANE WALL



FOURIER'S:

$$\dot{q} = -k A_c \frac{dT}{dx}$$

EARLIER PROOF:  $\frac{d\dot{q}}{dx} = 0$

$$0 = \frac{d}{dx} \left( -k A_c \frac{dT}{dx} \right)$$

$$\int 0 dx = -k A_c \int \frac{1}{dx^2} T dx$$

$\int -C_1 = \int dT$

$R(1) \setminus T = T_c$

$$\int \frac{dx}{kA_c} \int \frac{dx}{kA_c}$$

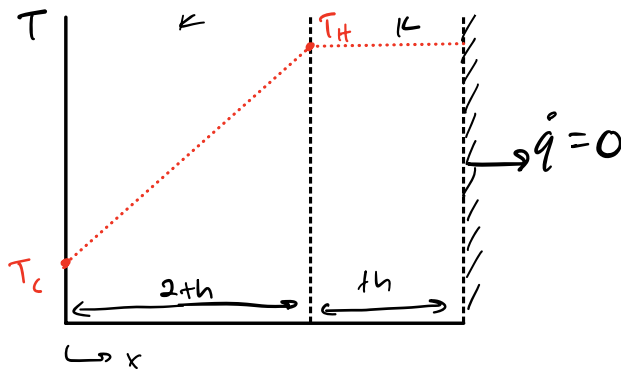
$$T(x) = -\frac{C_1 x}{kA_c} + C_2$$

$$T(x=0) = C_2 = T_c$$

$$T(x=2+h) = -\frac{C_1(2+h)}{kA_c} + T_c = T_H \rightarrow C_1 = \frac{(T_c - T_H)kA_c}{2+h}$$

$$\Rightarrow T(x) = \frac{(T_H - T_c)}{2+h} x + T_c$$

e) PLANE WALL



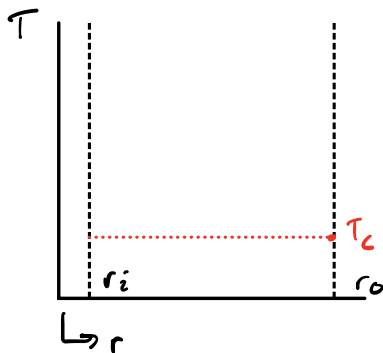
ALONG WALL,  $\dot{q}_{IN} = \dot{q}_{OUT} = 0$

$$\rightarrow \dot{q} = 0$$

$$\dot{q} = -kA_c \frac{dT}{dx} = 0$$

$$\therefore \frac{dT}{dx} = 0 \rightarrow T = \text{CONST}$$

f) ROD



$$\dot{q}_{IN} = \dot{q}_{OUT} = 0$$

$$\dot{q} = -kA_c \frac{dT}{dr} = 0$$

$$\rightarrow \frac{dT}{dr} = 0$$

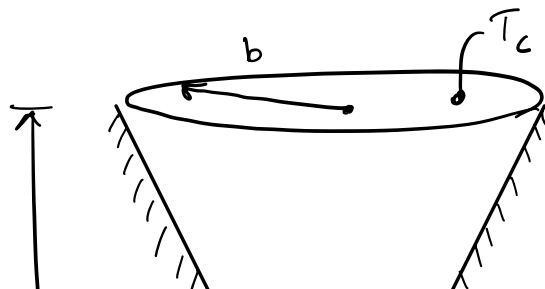
2)

$$b = 5 \text{ cm} = 0.05 \text{ [m]}$$

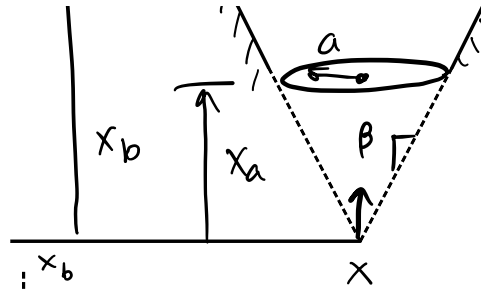
$$x_a = 1 \text{ cm} = 0.01 \text{ [m]}$$

$$x_b = 4 \text{ cm} = 0.04 \text{ [m]}$$

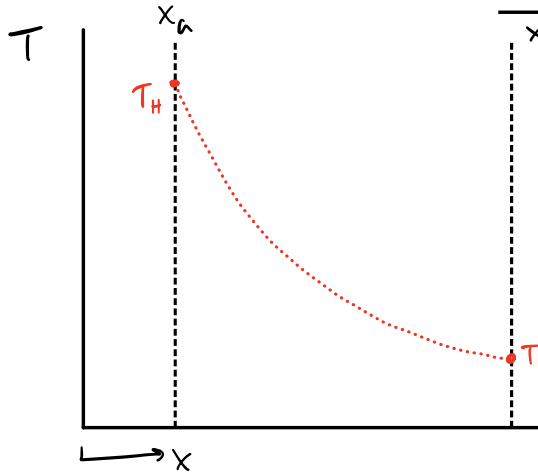
$$a = b/2$$



$$T_c = 80 \text{ [K]} \quad T_H = 285 \text{ [K]}$$

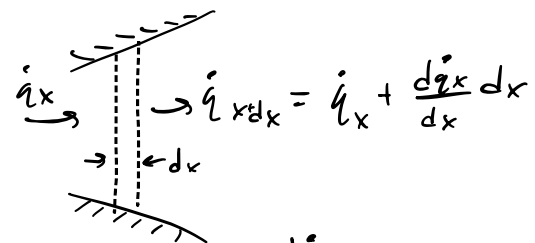


a)



FOURIER'S:  $\dot{q}'' = -k \frac{dT}{dx}$

$$\rightarrow \dot{q} = -k A_c \frac{dT}{dx} \quad (1)$$



$$\dot{q}_x = \dot{q}_{x+dx}, \therefore \frac{d\dot{q}_x}{dx} = 0 \quad (2)$$

(1) & (2):  $0 = \frac{d}{dx} \left( -k A_c \frac{dT}{dx} \right)$

$$A_c = \pi (r)^2 \quad r = x \beta$$

$$A_c = \pi \beta^2 x^2$$

$$\rightarrow 0 = -k \pi \beta^2 \frac{d}{dx} \left( x^2 \frac{dT}{dx} \right) = x^2 \frac{d^2 T}{dx^2} + 2x \frac{dT}{dx} = 0$$

LOSE  $x=0$  AS SOLUTION  
→ NOT NEEDED

$$0 = x^2 \frac{d^2 T}{dx^2} + 2x \frac{dT}{dx} \rightarrow \int 0 dx = \int \frac{d}{dx} \left( x^2 \frac{dT}{dx} \right) dx$$

$$C_1 = x^2 \frac{dT}{dx} \rightarrow \int \frac{dT}{dx} dx = \int \frac{C_1}{x^2} dx$$

c)  $T(x) = -\frac{C_1}{x} + C_2$

d)

BC1:  $T(x=x_a) = T_H$

BC2:  $T(x=x_b) = T_c$

FROM BC1:  $T(x_a) = -\frac{C_1}{x_a} + C_2 = T_H \quad (1)$

FROM BC2:  $T(x_b) = -\frac{C_1}{x_b} + C_2 = T_c \quad (2)$

FROM (1):  $285 [K] = -\frac{C_1}{0.01 [m]} + C_2 = -100 C_1 + C_2 = 285 \quad (3)$

FROM (2):  $80 [K] = -\frac{C_1}{0.04 [m]} + C_2 = -25 C_1 + C_2 = 80 \quad (4)$

FROM (3):  $C_2 = 285 + 100 C_1$

SUB INTO (4):  $-25 C_1 + (285 + 100 C_1) = 80$

$\rightarrow 75 C_1 = -205$

$\rightarrow C_1 = -\frac{41}{15} \approx -2.733$

SUB KNOWN  $C_1$  INTO

$\rightarrow C_2 = 285 + 100 \left( -\frac{41}{15} \right) = \frac{35}{3} \approx 11.67$

$\therefore T(x) = \frac{2.733}{x} + 11.667$

e)

$\dot{q} = -k A_c \frac{dT}{dx}$

$k$ : EES LOOKUP

$A_c = \pi \beta^2 x^2$

$\frac{dT}{dx} = -\frac{C_1}{x^2}$

f)

IN EES

g)

$x = x_a$  CONVECTION  $\rightarrow \bar{h}, T_h$

BC EQUATION:  $\dot{q}_{conv} = \bar{h} \cdot A_s \cdot (T_s - T_\infty)$

$\Rightarrow \dot{q}_{conv} = \bar{h} \cdot (\pi \beta^2 a^2) \cdot (T_s - T_h)$

$\uparrow$   
 $T_s = T(x = x_a)$

\$Load Incompressible

\$UNITSYS SI K Pa J mass rad

*"Kyle Adler ME364 HW01"**"givens"*

$$b = 0.05 \text{ [m]}$$

$$x\_b = 0.04 \text{ [m]}$$

$$x\_a = 0.01 \text{ [m]}$$

$$\beta = b/x\_b$$

$$T\_h = 285 \text{ [K]}$$

$$T\_c = 80 \text{ [K]}$$

$$k = \text{conductivity}(\text{Stainless\_AISI310}, T = \text{average}(T\_h, T\_c))$$

*"calculated by hand"*

$$C\_1 = -41/15 \text{ [1/K-m]}$$

$$C\_2 = 35/3 \text{ [1/K]}$$

*"area and dTdx equations"*

$$A = \pi \beta^2 x^2$$

$$dTdx = -C\_1/x^2$$

*"value for x, qdot does not change with respect to x so any nonzero real value yields same answer"*

$$x = x\_a$$

*"commented for part f"**"e) heat transfer through support"*

$$q\_dot = -k * A * dTdx$$

*"f) plot of temp as function of x between x=x\_a and x=x\_b"*

$$T = -C\_1/x + C\_2$$

SOLUTION

**Unit Settings: SI K Pa J mass rad**

$$A = 0.0004909 \text{ [m}^2\text{]}$$

$$b = 0.05 \text{ [m]}$$

$$\beta = 1.25 \text{ [-]}$$

$$C_1 = -2.733 \text{ [K-m]}$$

$$C_2 = 11.67 \text{ [K]}$$

$$dTdx = 27333 \text{ [K/m]}$$

$$k = 9.073 \text{ [W/m-K]}$$

$$q = -121.7 \text{ [W]}$$

$$T = 285 \text{ [K]}$$

$$T_c = 80 \text{ [K]}$$

$$T_h = 285 \text{ [K]}$$

$$x = 0.01 \text{ [m]}$$

$$x_a = 0.01 \text{ [m]}$$

$$x_b = 0.04 \text{ [m]}$$

No unit problems were detected.

KEY VARIABLES

$$\dot{q} = -121.7 \text{ [W]}$$

*e) heat transfer through support*

