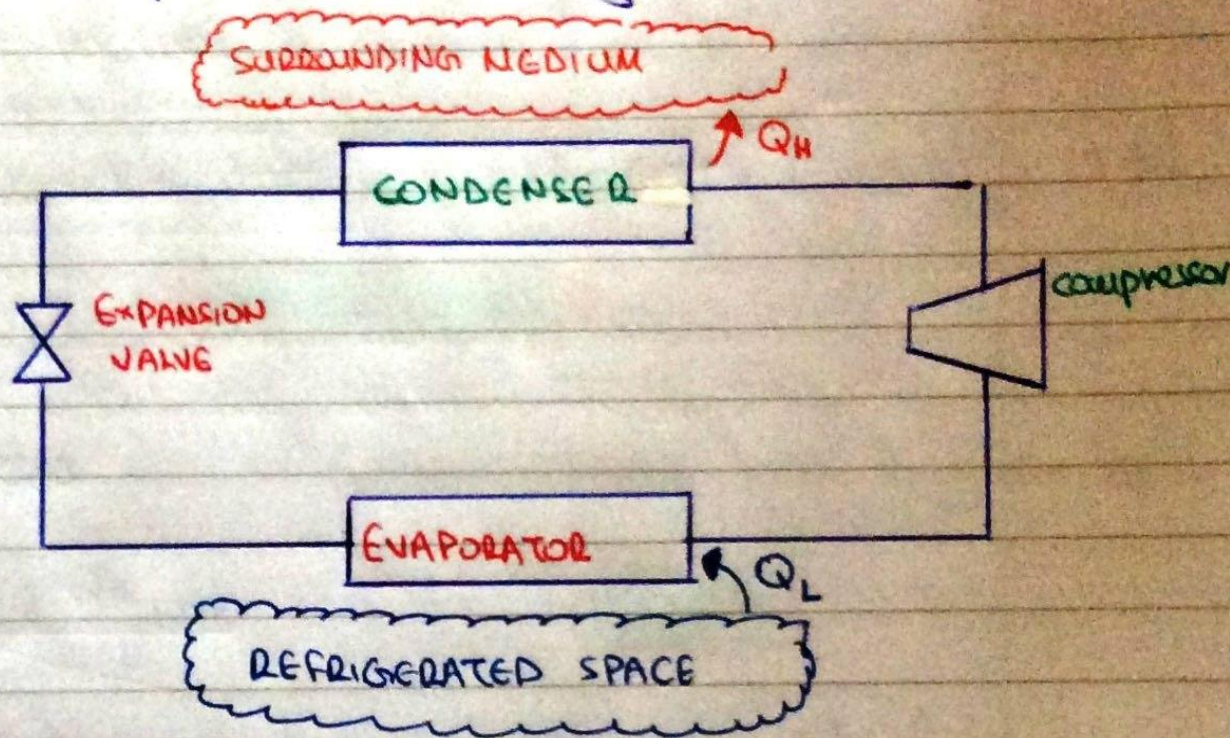


Question 1:

a) The basic components of a refrigerator system are presented below:



b) By using the definition of the  $COP_R$ , the power input to the refrigerator is determined as:

$$COP_R = \frac{\text{Desired Output}}{\text{Required Input}} = \frac{\dot{Q}_L}{\dot{W}_{\text{net},in}} \Rightarrow COP_R = \frac{\dot{Q}_L}{\dot{W}_{\text{net},in}} \quad [\text{Note form}]$$

$$\dot{W}_{\text{net},in} = \frac{\dot{Q}_L}{COP_R} = \frac{85 \text{ [kJ/min]}}{1.4} = \Delta \quad \dot{W}_{\text{net},in} = 60.7 \text{ kJ/min}$$

c) From the energy balance will have the following:

$$\dot{Q}_H = \dot{Q}_L + \dot{W}_{\text{net},in} = 85 \text{ [kJ/min]} + 60.7 \text{ [kJ/min]} = 145.7 \text{ [kJ/min]}$$



Question 2:

$$\dot{W}_{\text{net}, \text{in}} = 450 \text{ W}$$

$$\text{COP}_R = 2.5$$

$$m_{\text{tot}} = 5 \cdot 10 = 50 \text{ kg}$$

$$T_L = 8^\circ \text{C}$$

$$T_{\text{in}} = 20^\circ \text{C}$$

$$C_p = 4.2 \text{ kJ/kg}\cdot\text{K}$$

The total amount of heat that needs to be removed from the water tanks is

$$Q_L = [m \cdot C \cdot \Delta T] = 50 \text{ [kg]} \cdot 4.2 \frac{\text{kJ}}{\text{kg}\cdot\text{K}} \cdot 12 \text{ [K]}$$

$$\# Q_L = 2520 \text{ kJ} \#$$

The rate that the refrigerator removes heat is given by

$$\dot{Q}_L = (\text{COP}_R) \cdot \dot{W}_{\text{net}, \text{in}} = 2.5 \cdot 450 = 1125 \text{ W}$$

The total time required to removed 2520 kJ of heat

$$\Delta t = \frac{Q_L}{\dot{Q}_L} = \frac{2520 \text{ kJ}}{1.125 \frac{\text{kJ}}{\text{s}}} \Rightarrow \Delta t = 37.3 \text{ min}$$

Question 3:

$$Q_H = 700 \text{ kJ}$$

$$T_H = 500 \text{ K}$$

$$W_{\text{net}} = 300 \text{ kJ}$$

$$T_L = 290 \text{ K}$$

The maximum efficiency of a heat engine is given by

$$\eta_{\text{th}, \text{max}} = 1 - \frac{T_L}{T_H} = 1 - \frac{290}{500} = 0.42 \text{ [42\%]}$$

The actual thermal efficiency is given by the following expression

$$\eta_{\text{th}} = \frac{W_{\text{net}}}{Q_H} = \frac{300 \text{ kJ}}{700 \text{ kJ}} = 42.9\%$$

The claim is false as  $\eta_{\text{th}} > \eta_{\text{th}, \text{max}}$ .



Question 4:

$$T = 160^\circ\text{C}$$

$$\dot{m} = 440 \text{ kg/s}$$

$$\dot{W}_{\text{net, out}} = 22 \text{ MW}$$

$$T_L = 25^\circ\text{C}$$

$$\left. \begin{array}{l} T_{\text{source}} = 160^\circ\text{C} \\ x_{\text{source}} = 0 \end{array} \right\} h_{\text{source}} = 675,47 \text{ kJ/kg}$$

$$\left. \begin{array}{l} T_{\text{sink}} = 25^\circ\text{C} \\ x_{\text{sink}} = 0 \end{array} \right\} h_{\text{sink}} = 104,83 \text{ kJ/kg}$$

a) The rate of heat input to the plant is taken as the enthalpy difference between the source and the sink

$$\dot{Q}_{\text{in}} = \dot{m}_g \cdot [h_{\text{source}} - h_{\text{sink}}] = 440 [\text{kg/s}] \cdot 570,64 [\text{kJ/kg}]$$

$$\dot{Q}_{\text{in}} = 251081,6 \left[ \frac{\text{kJ}}{\text{s}} \right] = \boxed{251.082 \text{ [kW]}}$$

So, the actual thermal efficiency:  $\eta_{\text{th}} = \frac{\dot{W}_{\text{net, out}}}{\dot{Q}_{\text{in}}} = \frac{22 \text{ MW}}{251.2 \text{ MW}} \Rightarrow$

$$\boxed{\eta_{\text{th}} = 8,8\%}$$

b) The maximum thermal efficiency is given by the following:

$$\eta_{\text{th, max}} = 1 - \frac{T_L}{T_H} = 1 - \frac{(25+273) \text{ K}}{(160+273) \text{ K}} \Rightarrow \boxed{\eta_{\text{th, max}} = 31,2\%}$$

c) The heat rejection is  $\dot{Q}_{\text{out}} = \dot{Q}_{\text{in}} - \dot{W}_{\text{net, out}} = 229,1 \text{ MW}$

Question 5:

The COP of the heat pump will be maximum when the heat pump operates in a reversible manner. So, it should be

$$\text{COP}_{\text{HP, rev}} = \frac{1}{1 - \frac{T_L}{T_H}} = \frac{1}{1 - \left[ \frac{283}{243} \right]} = 29,3 \text{ for outdoor: } 10^\circ\text{C}$$

$$\text{COP}_{\text{HP, rev}} = \frac{1}{1 - \frac{T_L}{T_H}} = \frac{1}{1 - \left[ \frac{268}{243} \right]} = 11,7 \text{ for outdoor: } -5^\circ\text{C}$$



Finally,  $COP_{HP, rev} = \frac{1}{1 - \frac{T_L}{T_H}} = \frac{1}{1 - \left(\frac{243}{293}\right)} = 5,86$   
for outdoor:  $-30^{\circ}C$