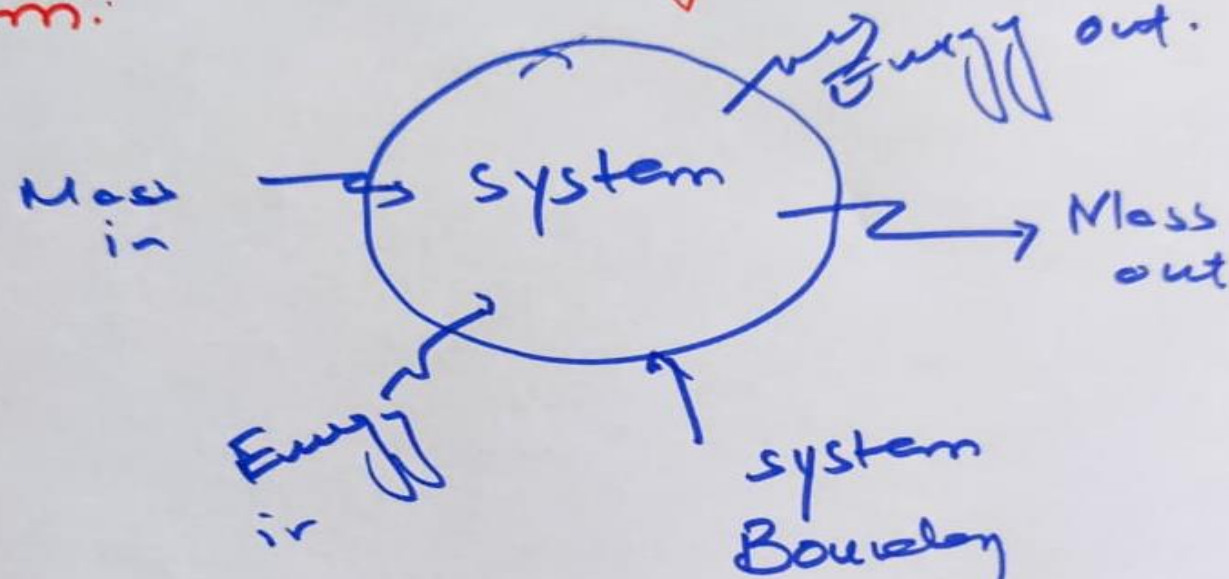


1st Law for Open Systems

First Law of thermodynamics for Open system:



Open system.

Rate of Mass transfer

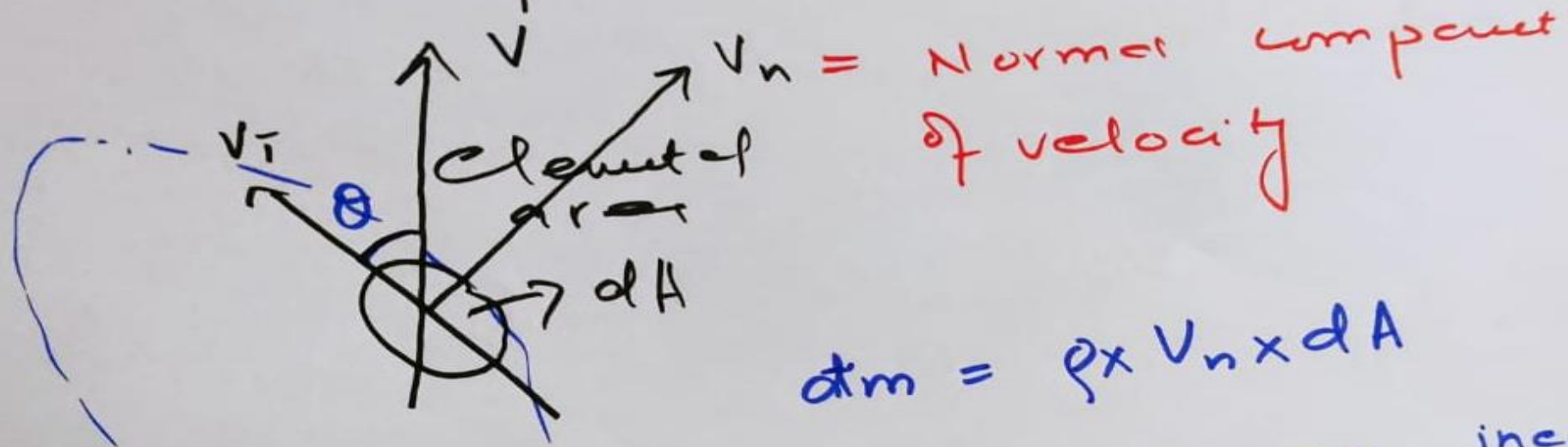
Rate of Energy transfer

'Mass Flow rate'

1

Mass Flow rate:-

Amount of mass flowing through
a section per unit time.



$$dm = \rho \times V_n \times dA$$

Work transfer
Heat transfer
Mass transfer

Path mass flow rate
Function differential.
is an inexact

Mass Flow Rate
is a path
function. (2)

$\int_{m_1}^{m_2} dm = ? \times$

$$\Rightarrow \int \rho \mathbf{v} \cdot d\mathbf{A} = \int \rho v_n dA$$

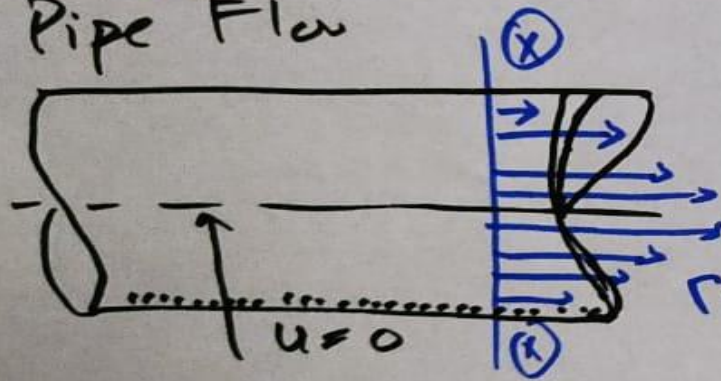
$\therefore v_n = \text{Velocity normal to the plane.}$

$$\dot{m} = \int \rho v_n dA$$

Valid for

Incompressible / Compressible Fluids.

e.g. Pipe Flow



At wall of Pipe velocity $u=0$

u is maximum at center.

velocity Distribution Profile.

$$V_{avg} = \frac{1}{A} \int v_n dA$$

$$\dot{m} = \rho V_{avg} A$$

$$\dot{m} = \rho A V_{avg}$$

$$V = V_{avg}$$

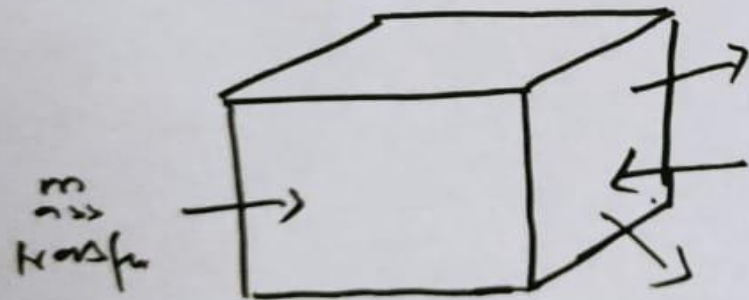
14/5

Open Systems : Control Volume Approach.

↔ Also known as
Eulerian Approach in Fluid Mechanics

Closed System : Control Mass Approach

Also called as Lagrangian approach
in Fluid Mechanics



Conservation of mass

"mass cannot be created or destroyed only change one form to other"

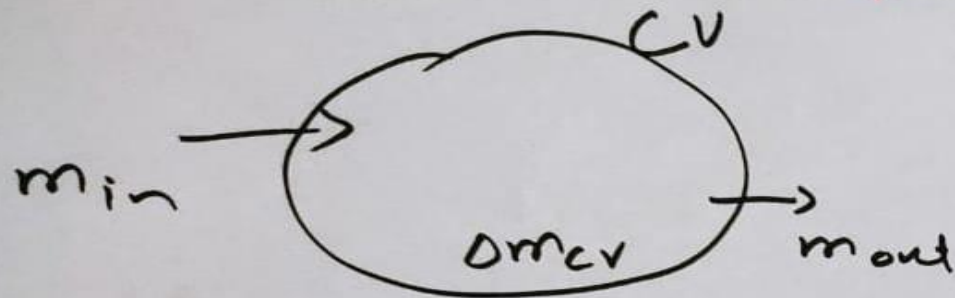
eg 1 kg of water

1 kg of steam

(4)

Conservation of mass

(Mathematically)



$$m_{in} > m_{out} ?$$

Δm_{cv}
= mass
accumulated

$$m_{in} - m_{out} = \Delta m_{cv}$$

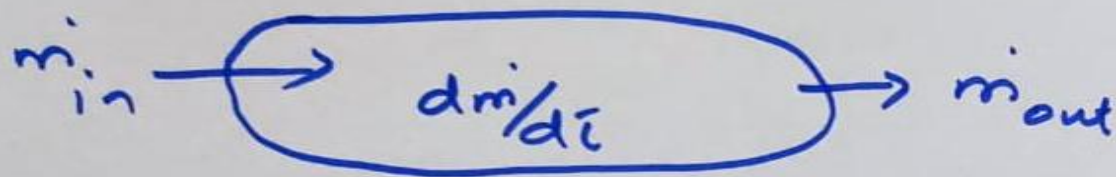
Time Rate \Rightarrow $m_{in}^{\cdot} - m_{out}^{\cdot} = \frac{dm}{dt}$

dt = time interval.

$$\frac{dm}{dt} = \text{Rate of mass accumulation}$$

(5)

For Steady Flow



$$\frac{dm}{dt} = 0$$

Rte of accumulation
 $= 0$

Mass:-

Mass accumulated within the system
with respect to time will remain
constant

$$| m_{cv} = \text{constant} |$$

$$| \frac{dm}{dt} = 0 |$$

Conservation of mass For Steady Flow

$$\frac{m_{in} = m_{out} = 0}{m_{in} = m_{out}}$$

(6)

$$\rho A_1 v_1 = \rho A_2 v_2$$

Continuity Equation

Valid for compressible as well as incompressible fluids.

Mass Balance

$$\dot{m}_{in} - \dot{m}_{out} = \frac{dm}{dt}$$

Unsteady Flow

Mass Balance

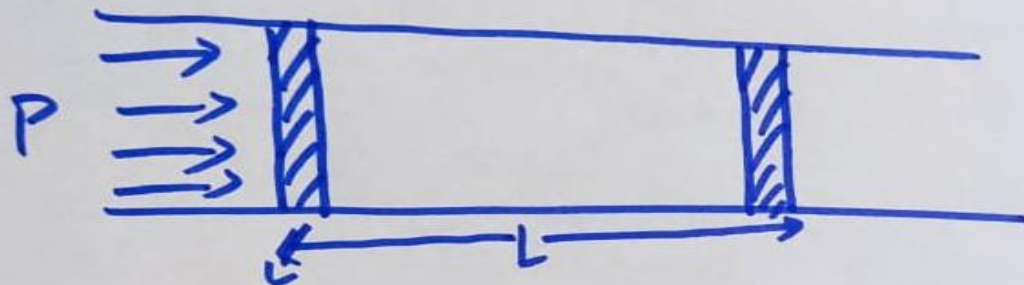
$$\dot{m}_{in} = \dot{m}_{out}$$

Steady Flow

(7)

Flow Work:- Work Required to maintain the flow

Consider a passage



Area of Element
+ to P

$$W = F \times \text{displacement}$$

$$= P \times A \times L$$

$$\boxed{W = PV}$$

Total Energy:-

$$E = \text{Internal Energy} + \text{Kinetic Energy} + \text{Potential Energy}$$

$$\underline{E = U + \frac{1}{2}mv^2 + mgz} \quad \text{KJ / J}$$

Specific Total Energy :-

$$E/m = \frac{U}{m} + \frac{1}{2} \frac{mv^2}{m} + \frac{mgz}{m}$$

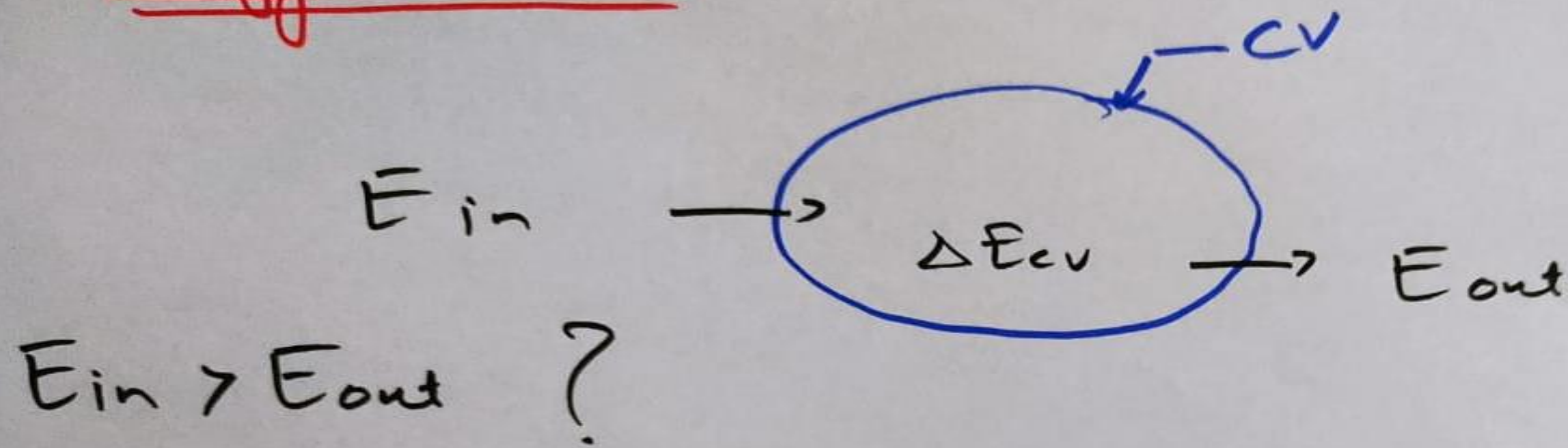
$$\boxed{e = u + \frac{1}{2}v^2 + gz} \quad \text{KJ/kg, J/kg}$$

Energy Balance :-

P.T.O

(9)

Energy Balance



$$E_{in} - E_{out} = \Delta E_{cv}$$

ΔE_{cv} = Energy
Accumulated
inside the
control volume.

Time Rate:-

$$E_{in} - E_{out} = \frac{dE}{dt}$$

(10)

For Steady Flow

$\Delta E_{cv} \rightarrow$ Constant

$$\frac{dE_{cv}}{dt} = 0$$

$$\boxed{\dot{E}_{in} - \dot{E}_{out} = 0}$$

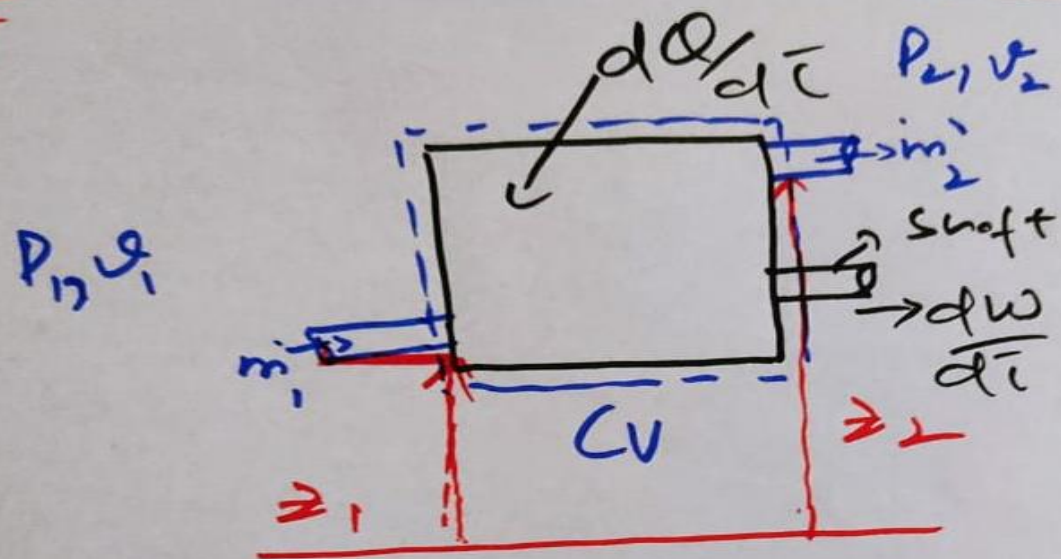
$$\boxed{\dot{E}_{in} = \dot{E}_{out}}$$

Steady
Flow

Now to derive tu
SF EE

11

SFEE



Steady
Flow Device

$$\frac{dm}{dt} = 0$$

No mass
& energy

$$\frac{dE}{dt} = 0$$

accumulation

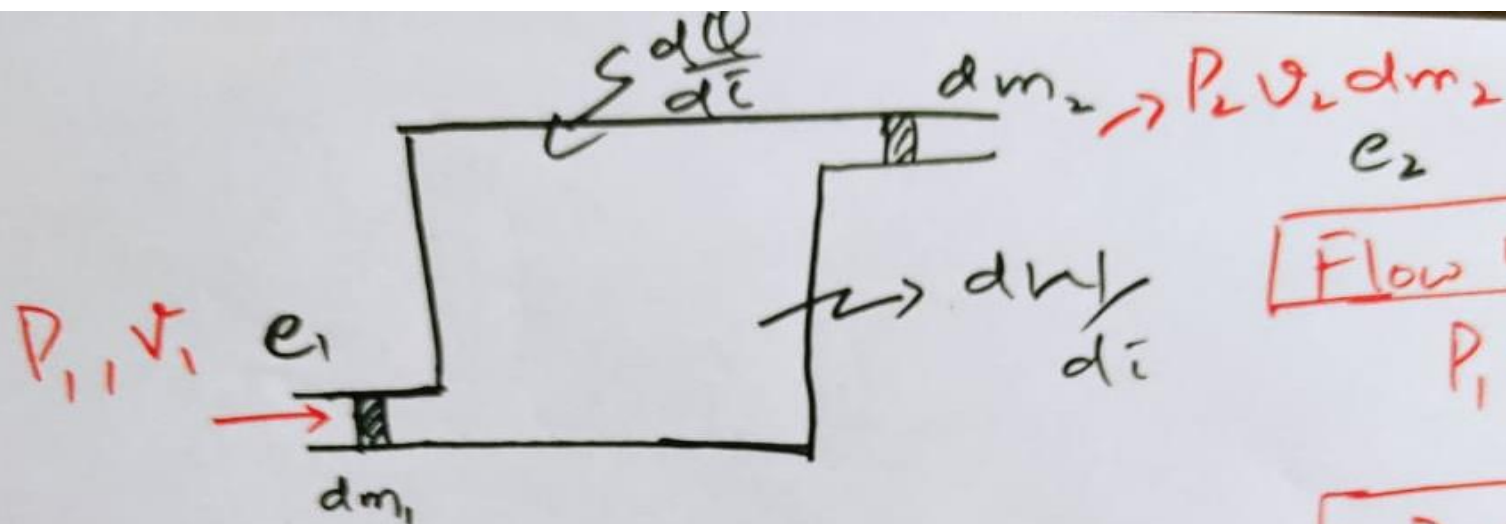
In a Steady Flow system
there are two types of work transfer

→ External Work
 $\left\{ \begin{array}{l} \text{Shaft Work} \\ \text{Electric Work} \end{array} \right.$

→ Flow Work

(12)

(B)



Flow work per unit mass
 $P_1 v_1$

Flow work
 $P_1 v_1 dm_1$

Total work transfer

$$W = W_x - P_1 v_1 dm_1 + P_2 v_2 dm_2$$

External work

Work added into the system

Work done by the system

Time Rate

$$\frac{dW}{dt} = \frac{dW_x}{dt} - \frac{P_1 v_1 dm_1}{dt} + \frac{P_2 v_2 dm_2}{dt}$$

$$\frac{dm_1}{d\bar{t}} = \text{Inlet mass Flow Rate} = \dot{m}_1$$

$$\frac{dm_2}{d\bar{t}} = \text{Outlet mass Flow rate} = \dot{m}_2$$

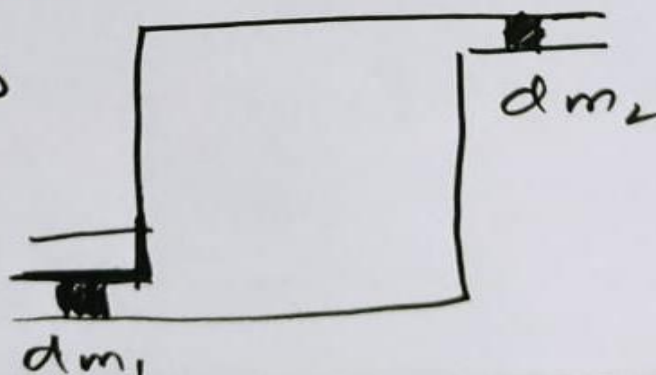
$$\frac{dw}{d\bar{t}} = \frac{dw_{ku}}{d\bar{t}} - P_1 v_1 \dot{m}_1 + P_2 v_2 \dot{m}_2$$

Rate of work transfer (shaft work)

External Rate of work transfer

P_2, v_2, \dot{m}_2

$c_1 = \text{specific}$
 Total energy
 P_1, v_1, \dot{m}_1



(14)

$$m_1 e_1 + \frac{dQ}{d\bar{t}} = m_2 e_2 \frac{dW}{d\bar{t}}$$

$$e = u + \frac{1}{2}v^2 + gz$$

$$m_1 \left(u_1 + \frac{v_1^2}{2} + gz_1 \right) + \frac{dQ}{d\bar{t}}$$

$$= m_2 \left(u_2 + \frac{v_2^2}{2} + gz_2 \right) + \frac{dW}{d\bar{t}}$$

$$m_1 \left(u_1 + \frac{v_1^2}{2} + gz_1 \right) + \frac{dQ}{d\bar{t}} = m_2 \left(u_2 + \frac{v_2^2}{2} + gz_2 \right)$$

$$+ \frac{dW}{d\bar{t}} - P_1 v_1 m_1 + P_2 v_2 m_2 \quad (15)$$

$$m_1 \left(u_1 + \frac{v_1^2}{2} + g z_1 \right) + \frac{dQ}{dt} = m_2 \left(u_2 + \frac{v_2^2}{2} + g z_2 \right) + \frac{dW_x}{dt} - P_1 v_1 m_1 + P_2 v_2 m_2$$

$$m_1 \left(\underline{u_1} + \frac{v_1^2}{2} + g z_1 + \underline{P_1 v_1} \right) + \frac{dQ}{dt} = m_2 \left(u_2 + \frac{v_2^2}{2} + g z_2 + P_2 v_2 \right)$$

$$h = u + p v$$

$$+ \left(\frac{dW_x}{dt} \right)$$

→ External work per unit time

Steady Flow

$$m_1 \left(h_1 + \frac{v_1^2}{2} + g z_1 \right) + \frac{dQ}{dt} = m_2 \left(h_2 + g z_2 + \frac{v_2^2}{2} \right) + \frac{dW_x}{dt}$$

Steady Flow

$$m_1 = m_2$$

(16)

$m_1 = m_2 = m = \frac{dm}{dt} \therefore$ Steady Flow

SFEE on time basis

$$\frac{dm}{dt} \left(h_1 + \frac{v_1^2}{2} + g z_1 \right) + \frac{dQ}{dt} = \frac{dm}{dt} \left(h_2 + \frac{v_2^2}{2} + g z_2 \right)$$

Dividing by $\frac{dm}{dt}$ both sides

$$h_1 + \frac{v_1^2}{2} + g z_1 + \frac{dQ}{dm} = h_2 + \frac{v_2^2}{2} + g z_2 + \frac{dW}{dm}$$

$\frac{dQ}{dm}$

$\frac{dW}{dm}$

$10 \frac{J}{kg} \cdot \frac{J}{kg}$

$10 \frac{J}{kg} \cdot \frac{J}{kg}$

SFEE on mass basis

(17)

Bernoulli Equation is a special
case of SFEE.

Bernoulli Equation	SFEE
Valid for only Incompressible Fluids	Valid for both incompressible compressible Fluids.

SFEE \rightarrow First Law of thermodynamics
for open system.