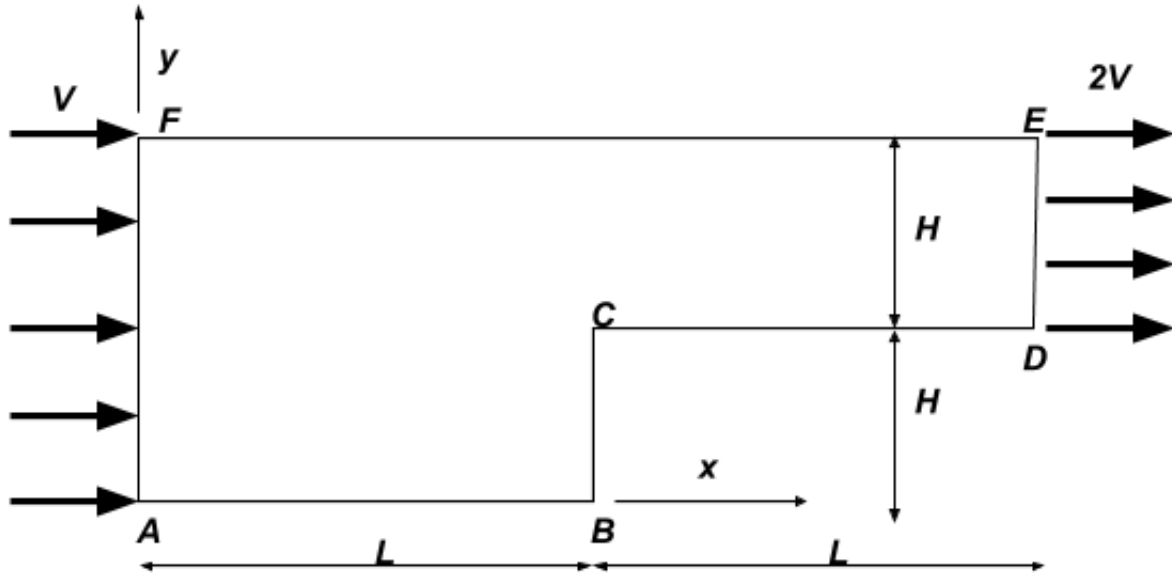


Homework 4

AE370 - Spring 2018
Emilio R. Gordon

Problem: Steady-state fluid problem

Use the finite difference method to solve the following incompressible/inviscid fluid flow problem:



The flow problem can be described by the following GDE:

$$\frac{\partial^2 \Phi}{\partial x^2} + \frac{\partial^2 \Phi}{\partial y^2} = 0$$

where the streamline function Φ is related to the x and y components of the velocity vector through:

$$V_x = \frac{\partial \Phi}{\partial y} \quad V_y = -\frac{\partial \Phi}{\partial x}$$

The boundary conditions are:

$$\begin{aligned} \Phi &= V y && \text{along AF} \\ \Phi &= 2VH && \text{along FE} \\ \Phi &= 2V(y - H) && \text{along DE} \\ \Phi &= 0 && \text{along ABCD} \end{aligned}$$

Use a second-order central difference scheme to solve the problem using N_x grid spacings to discretize L (i.e. $\Delta x = L/N_x$) and N_y grid spacings to discretize H (i.e., $\Delta y = H/N_y$).

- (a) Derive the discretized form of the PDE
- (b) Choose a numbering of the grid points (Describe it thoroughly in your report!) and put together the matrix equation.
- (c) Implement the boundary conditions.
- (d) Write a Matlab code that solves this problem. As output you should create the following three plots:
 - (a) A contour plot for the streamline function (using the command `CONTOUR`)
 - (b) A vector plot for the velocity field (using the central difference approximation for the first derivatives of the streamline function and the command `QUIVER`)
 - (c) A x-y plot of the pressure distribution along the edge boundary condition, where the pressure is obtained by $P = \rho(V_x^2 + V_y^2)$ where ρ is the fluid density.
- (e) Solve the problem for $L=1$ [m], $H = 0.2$ [m], $V = 1$ [m/s] and $\rho = 1[kg/m^3]$. Perform a convergence study to determine the appropriate values of N_x and N_y . Comment on your solution and especially on the computed solution in the vicinity of the corner.

Solution:

(a) Derive the discretized form of the PDE

Before beginning the problem, it is important to classify the PDE's character according to the directions along which information can travel. From class, we learned it is possible to determine a PDE's direction by...

Determining PDE Directions

$$a \frac{\partial^2 T}{\partial x^2} + b \frac{\partial^2 T}{\partial x \partial y} + c \frac{\partial^2 T}{\partial y^2} + d \frac{\partial T}{\partial x} + e \frac{\partial T}{\partial y} + g T + h = 0$$

Such that the slope (dx/dy) is controlled by the sign of $(b^2 - 4ac)$. In other words, If...

$(b^2 - 4ac) < 0 \rightarrow$ the slope is imaginary (all directions)
 \rightarrow **Elliptic PDE**

$(b^2 - 4ac) = 0 \rightarrow$ There is only one slope (information uniformly in one direction)
 \rightarrow **Parabolic PDE**

$(b^2 - 4ac) > 0 \rightarrow$ There are two slopes (information in two paths)
 \rightarrow **Hyperbolic PDE**

The general differential equation given $\frac{\partial^2 \Phi}{\partial x^2} + \frac{\partial^2 \Phi}{\partial y^2} = 0$ classifies as an elliptical PDE meaning information travels in all directions.

In part b, a structured grid will be created. For a structured grid, using the second-order central difference approach for x-derivatives and indexing i values for a given y-location (j), we have

$$\left(\frac{\partial^2 \Phi}{\partial x^2} \right)_{i,j} = \frac{\Phi_{i-1,j} - 2\Phi_{i,j} + \Phi_{i+1,j}}{\Delta x^2} + O(\Delta x^2)$$

For a structured grid, using the second-order central difference approach for y-derivatives and indexing j values for a given x-location (i), we have

$$\left(\frac{\partial^2 \Phi}{\partial y^2} \right)_{i,j} = \frac{\Phi_{i,j-1} - 2\Phi_{i,j} + \Phi_{i,j+1}}{\Delta y^2} + O(\Delta y^2)$$

Combining these two expressions into our PDE, we get the discretized form of the PDE.

$$\frac{\Phi_{i,j-1} - 2\Phi_{i,j} + \Phi_{i,j+1}}{\Delta y^2} + \frac{\Phi_{i-1,j} - 2\Phi_{i,j} + \Phi_{i+1,j}}{\Delta x^2} = 0$$

Expanding

$$\frac{\Phi_{i,j-1}}{\Delta y^2} - \frac{2\Phi_{i,j}}{\Delta y^2} + \frac{\Phi_{i,j+1}}{\Delta y^2} + \frac{\Phi_{i-1,j}}{\Delta x^2} - \frac{2\Phi_{i,j}}{\Delta x^2} + \frac{\Phi_{i+1,j}}{\Delta x^2} = 0$$

Simplifying

$$2\Phi_{i,j}(\Delta x^2 + \Delta y^2) = \Delta x^2(\Phi_{i,j-1} + \Phi_{i,j+1}) + \Delta y^2(\Phi_{i-1,j} + \Phi_{i+1,j})$$

Divide by Δy^2 and define $\eta = \frac{\Delta x}{\Delta y}$ such that

$$2\Phi_{i,j}(1 + \eta^2) = \eta^2(\Phi_{i,j-1} + \Phi_{i,j+1}) + (\Phi_{i-1,j} + \Phi_{i+1,j})$$

Our final discretized GDE therefore is...

$$0 = \eta^2 \Phi_{i,j-1} + \eta^2 \Phi_{i,j+1} - 2\Phi_{i,j}(1 + \eta^2) + \Phi_{i-1,j} + \Phi_{i+1,j}$$

Where

$$\eta = \frac{\Delta x}{\Delta y}$$

(b) Choose a numbering of the grid points and put together the matrix equation.

From the code provided, it is easy to extrapolate the Global Equation Number (GEN) by the problem. The shape of the duct adds some complexities to calculating the GEN however, it is simple enough to look at in sections. NOTE: The figure in the code provided differs from the problem statement figure. The figure provided in the code was used in this analysis.

```

1  %      B                      E                      G
2  %      -----
3  %      |                      |
4  %      |                      |
5  %      |                      |
6  %      |                      |
7  %      |                      |
8  %      |                      | F
9  %      |                      -----
10 %      |                      |
11 %      |                      |
12 %      |                      |
13 %      |                      |
14 %      |                      |
15 %      |                      |
16 %      -----
17 %      A                      C
18 %
19 % Dimensions:    AC = DF = BE = EG = L
20 %              CD = FG = DE = AB/2 = H
21 % The domain is discretized with Nx grid spacings along AC and DF
22 % and NY grid spacings along CD and FG.
23 % Index convention:
24 %      Local indices (i,j)          Global index (q)
25 %      A      (1,1)                  qA=1
26 %      B      (1,2*Ny+1)             qB=2*Ny+1
27 %      C      (Nx+1,1)               qC=Nx*(2*Ny+1)+1
28 %      D      (Nx+1,Ny+1)            qD=qC+Ny
29 %      E      (Nx+1,2*Ny+1)          qE=qD+Ny
30 %      F      (2*Nx+1,Ny+1)          qF=qE+(Nx-1)*(Ny+1)+1
31 %      G      (2*Nx+1,2*Ny+1)        qG=qF+Ny

```

Figure 1: Problem Diagram and Index Convention

The Duct can be divided into two sections: ABEDC and DEGF. The only notable difference is the lack of a bottom half on the second section. In addition, we already know the GEN of the corners given N_x and N_y . The order at which the GEN are defined will be as follows:

- (1) Start at the A corner, $GEN = 1$
- (2) Move upward till we reach point B, $(i=1, j=2N_y+1)$
- (3) Shift to the bottom of the next column and repeat.

In class, we were given the equation $GEN = i + (j - 1)n$, where i and j are the indices and n is the column number. Building on this, a GEN for section ABEDC can be made understanding that the section ends when $i = Nx + 1$. As a result, it can be said that so long as $i \leq Nx + 1$ then $q = (i - 1)(2Ny + 1) + j$ essentially applying a GEN as described above.

The above GEN equation only applies when $i \leq Nx + 1$, section ABEDC. The second section, section DEGF, has the physical constraint of not existing for any points below $Ny + 1$. Keeping with the GEN ordering system defined above for the second section, it can be said that so long as $j \geq Ny + 1$ then $q = (Nx + 1)(2Ny + 1) + (i - Nx - 2)(Ny + 1) + j - Ny$.

Any other possible value for the global equation number falls outside of the shape and is set to 0.

To summarize, calculating the Global Equation Number is done by the following function.

```

1 function q=compute_q(i,j,Nx,Ny)
2 q = 0;
3     if(i <= Nx +1)
4         q = (i-1)*(2*Ny+1)+j;
5     elseif(j >= Ny + 1)
6         q = (Nx + 1)*(2*Ny+1)+(i-Nx-2)*(Ny+1)+j-Ny;
7     end
8 end

```

Figure 2: Subroutine to Compute the Global Equation Number Corresponding to Grid

With this model in place, we can now start forming the linear system. Keeping with the form of $Au = b$ the following code was used to establish an A matrix, the matrix of our knowns.

From part 1, we arrived at a formula for the discretized PDE of

$$0 = \eta^2 \Phi_{i,j-1} + \eta^2 \Phi_{i,j+1} - 2 \Phi_{i,j} (1 + \eta^2) + \Phi_{i-1,j} + \Phi_{i+1,j}$$

Where

$$\eta = \frac{\Delta x}{\Delta y}$$

From this, a five-point computational stencil for each node (i,j) is constructed.

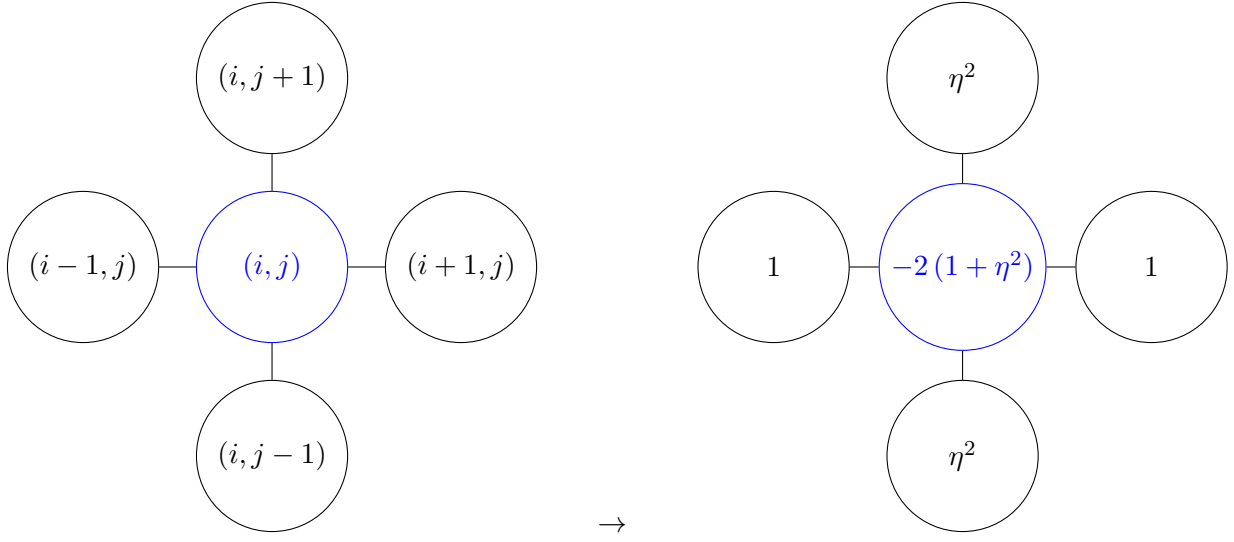


Figure 3: Five-Point Computational Stencil

It is easier to start forming a linear system by filling in the interior. The code in Figure 4 does this by

1. First establishing a zero "sparse" matrix with dimensions $m \times n$ where m and n are both the number of nodes in the system.
2. Step 1 is set as a placeholder along the diagonal for all degrees of freedom. These are later overwritten by the interior grid points for the appropriate cells.
3. We are only focusing on the interior. Therefore, for the ABEDC section, we parse over
 - $i = 2 : Nx$
 - 2: Starting at the second column
 - Nx : Ending right before the boundary condition $Nx + 1$
 - $j = 2 : 2Ny$
 - 2: Starting at the second row
 - $2Ny$: Ending right before the boundary condition $2Ny + 1$
4. Again, we are only focusing on the interior. Therefore, for the DEGF section, we parse over
 - $i = Nx + 1 : 2Nx$
 - $j = Ny + 2 : 2Ny$

```

1 % Set up matrix (Amat) and vector (bvec) dimensions
2 Amat=sparse(Numeq,Numeq);
3 bvec=zeros(Numeq,1);
4 %
5 % Build linear system
6 for i=1:Numeq;           % place 1 along diagonal for all DOF then overwrite the interior grid
   points
7     Amat(i,i)=1;
8 end
9 for i=2:Nx % loop over interior grid points — left half of domain
10     for j=2:2*Ny
11         qij=compute_q(i,j,Nx,Ny); % compute equation number q for(i,j) grid point
12         Amat(qij,qij)=-2*(1+eta^2);%Grid Point
13         q=compute_q(i-1,j,Nx,Ny); %Grid Point to the left
14         Amat(qij,q)= 1;
15         q=compute_q(i+1,j,Nx,Ny); %Grid Point to the right
16         Amat(qij,q)= 1;
17         q=compute_q(i,j-1,Nx,Ny); % Grid point below
18         Amat(qij,q)= eta^2;
19         q=compute_q(i,j+1,Nx,Ny); %Grid Point above
20         Amat(qij,q)= eta^2;
21     end
22 end
23 for i=Nx+1:2*Nx % loop over interior grid points — right half of domain
24     for j=Ny+2:2*Ny
25         qij=compute_q(i,j,Nx,Ny);
26         Amat(qij,qij)= -2*(1+eta^2);
27         q=compute_q(i-1,j,Nx,Ny);
28         Amat(qij,q)= 1;
29         q=compute_q(i+1,j,Nx,Ny);
30         Amat(qij,q)= 1;
31         q=compute_q(i,j-1,Nx,Ny);
32         Amat(qij,q)= eta^2;
33         q=compute_q(i,j+1,Nx,Ny);
34         Amat(qij,q)= eta^2;
35     end
36 end

```

Figure 4: Building the A Matrix for the Linear System

Implement the boundary conditions.

Implementing the boundary conditions poses a set of challenges. We will tackle this by focusing on them individually.

We start with

$$\Phi = 2VH \quad \text{along FE}$$

To implement this, it is required to define the nodes it is applied to. Simply put, a boundary condition along FE means that for the condition applies for $\Phi(x, 2H)$. In terms of indices

- $i = 2 : 2 Nx$
 - Starting at the second column
 - Boundary condition applies till the second to last column
- $j = 2 Ny + 1$
 - Constant
 - Fixed at the top edge of the boundary.

```
1 for i=2:2*Nx
2     j=2*Ny+1;
3     q=compute_q(i,j,Nx,Ny);
4     bvec(q)= 2*V*H; %Boundary Condition
5 end
```

Figure 5: Implementing Boundary Conditions Over the Top Edge

Next we implement

$$\Phi = 2V(y - H) \quad \text{along DE}$$

To implement this, it is required to define the nodes it is applied to. Simply put, a boundary condition along DE means that for the condition applies for $\Phi(2L, y)$. In terms our indices

- $i = 2Nx + 1$
 - Constant
 - Fixed at the right edge of the boundary.
- $j = Ny + 1 : 2Ny + 1$
 - Starting from the bottom of section DEGF
 - Ending at the top of section DEGF

```

1 for j=Ny+1:2*Ny+1
2     i=2*Nx+1;
3     q=compute_q(i,j,Nx,Ny);
4     y = (j-1)*dy;
5     bvec(q)= 2*V*(y-H); %Boundary Condition
6 end

```

Figure 6: Implementing Boundary Conditions Over the Right Edge

Finally, the last non-zero boundary condition is

$$\Phi = V y \quad \text{along AF}$$

To implement this, it is required to define the nodes it is applied to. Simply put, a boundary condition along AF, means that for the condition applies for $\Phi(1, y)$. In terms our indices

- $j = 1 : 2Ny + 1$
 - Starting from the bottom of line AF
 - Ending at the top of section AF

```

1 for j=1:2*Ny+1
2     bvec(j)= V*(j-1)*dy;
3 end

```

Figure 7: Implementing Boundary Conditions Over the Left Edge

```

1 %
2 % Build right-hand-side vector (imposed Psi BC)
3 for j=1:2*Ny+1 % loop over the left edge:
4     bvec(j)= V*(j-1)*dy;
5 end
6 for i=2:2*Nx % loop over top edge:
7     j=2*Ny+1;
8     q=compute_q(i,j,Nx,Ny);
9     bvec(q)= 2*V*H;
10 end
11 for j=Ny+1:2*Ny+1 % loop over right edge:
12     i=2*Nx+1;
13     q=compute_q(i,j,Nx,Ny);
14     y = (j-1)*dy;
15     bvec(q)= 2*V*(y-H);
16 end
17 % Remainder of boundary has Psi=0

```

Figure 8: Building the B Matrix for the Linear System

Write a Matlab code that solves this problem and output the following three plots:

The complete code can be found on the final pages of this report.

A contour plot for the streamline function is shown in Figure . Here, the contours represent the computed function, $\Phi(x, y)$. Note that the sharper gradient on the right side of the domain. This is an indication of acceleration of the fluid flow which is expected for incompressible flow.

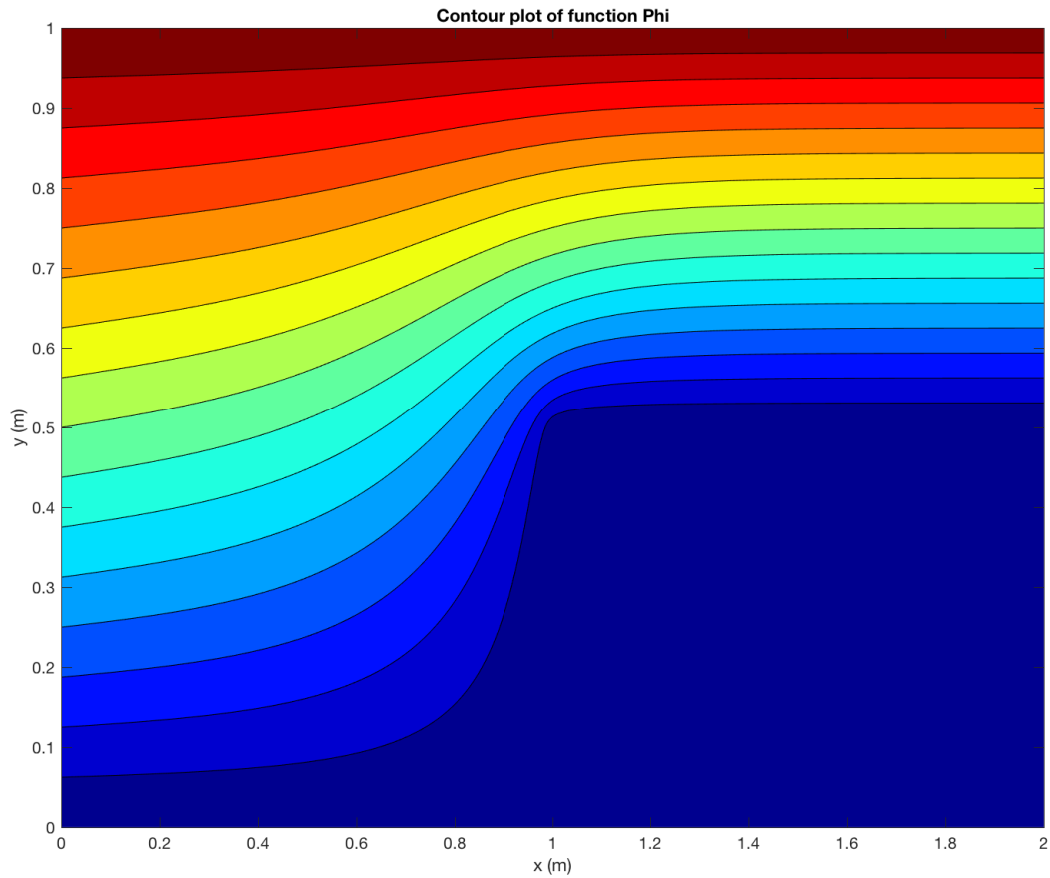


Figure 9: Contour Plot of the Streamline Function $\Phi(x, y)$

A vector plot for the velocity field using the central difference approximation for the first derivatives of the streamline function is shown in Figure . Differentiating the streamline function field, $\Phi(x, y)$ is done by using the central difference scheme for the interior nodes and then the forward finite difference scheme for the boundary nodes. The results show, as suggested from before, an acceleration of the flow past the step. In addition, the velocity field also shows high velocity at the corner of the step.

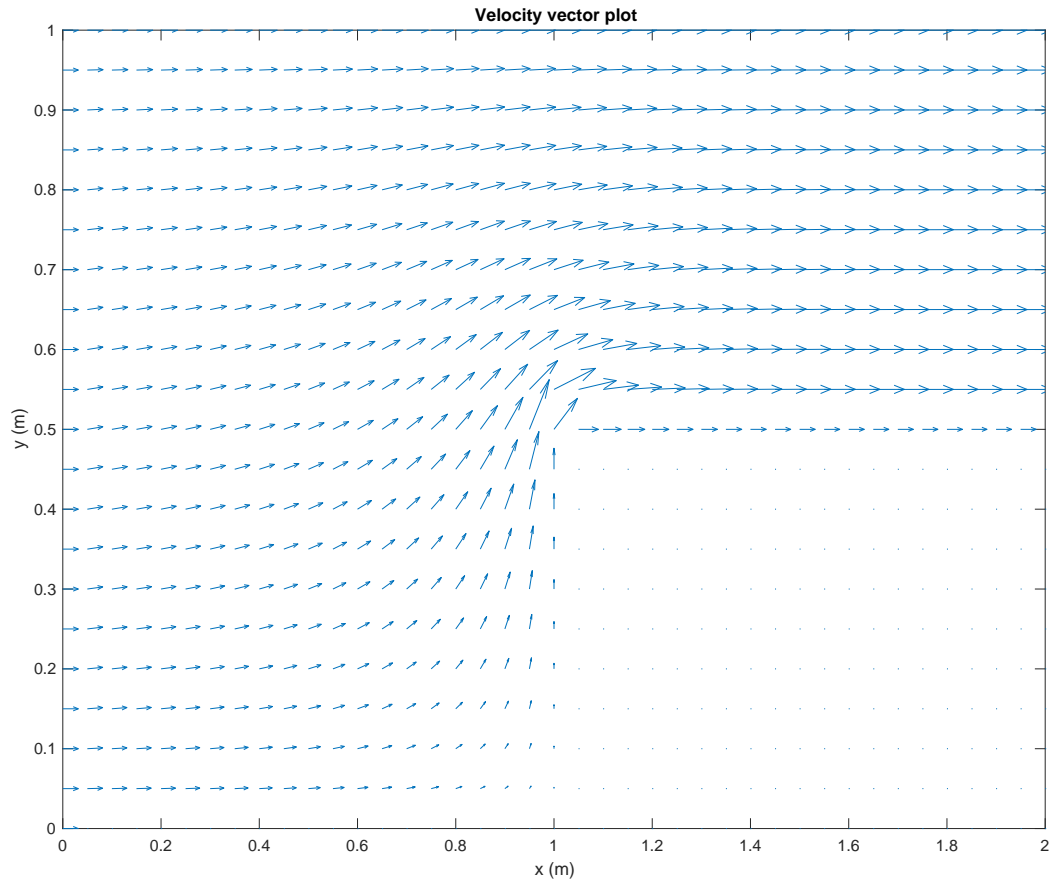


Figure 10: Velocity Vector Field for the Streamline Function $\Phi(x, y)$

The dynamic pressure field along the edge boundary condition is obtained by $P = \frac{1}{2} \rho (V_x^2 + V_y^2)$ where ρ is the fluid density and is shown in Figure . It becomes clear that there is high pressure at the corner which is expected do to the higher velocity. Increasing N_y to create a finer discretization (noting that N_x is always $2N_y$).

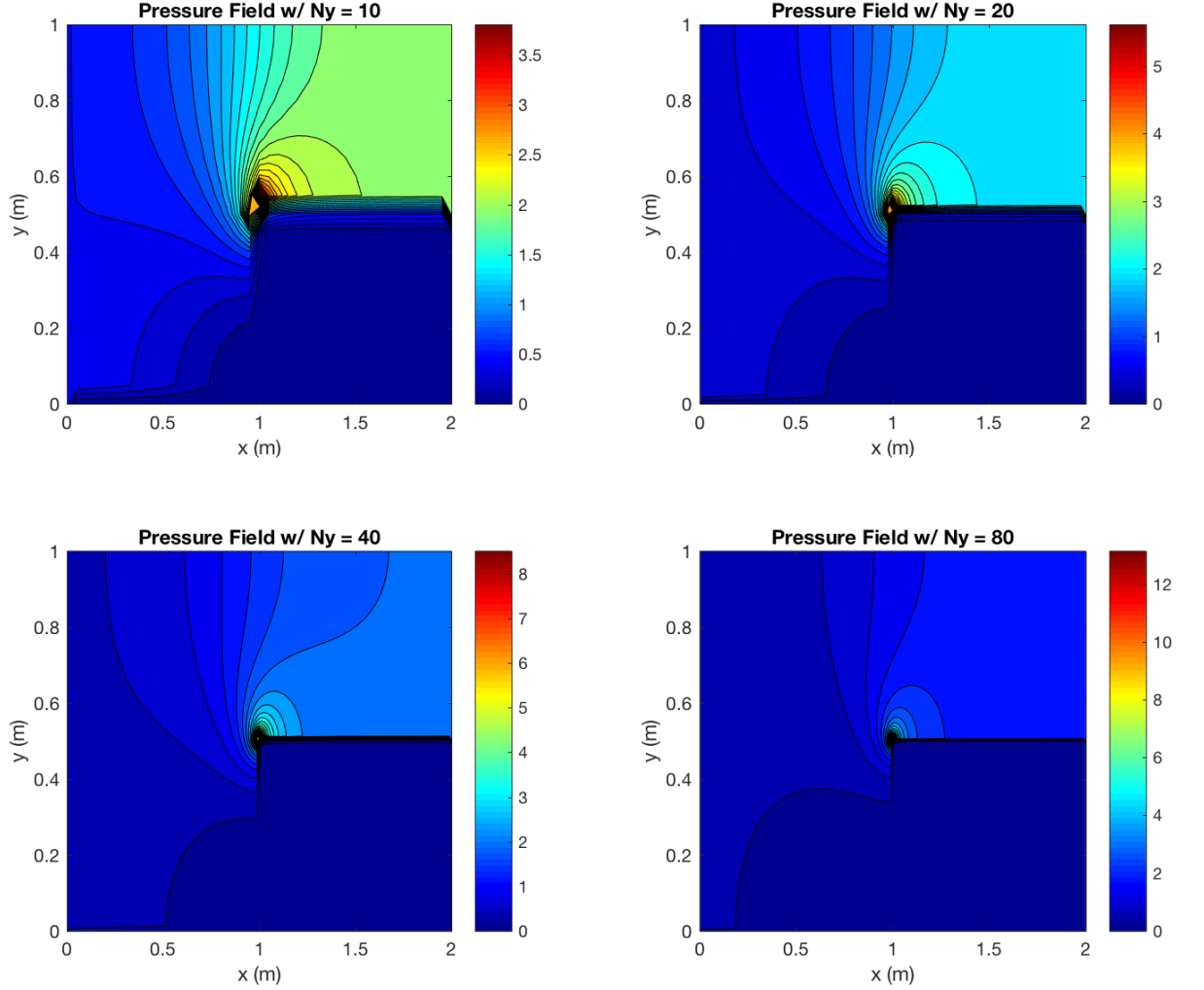


Figure 11: Pressure Field for Various Values of N_y . Note $N_x = 2 N_y$

This becomes even more clear when zooming in on the corner as shown in Figure

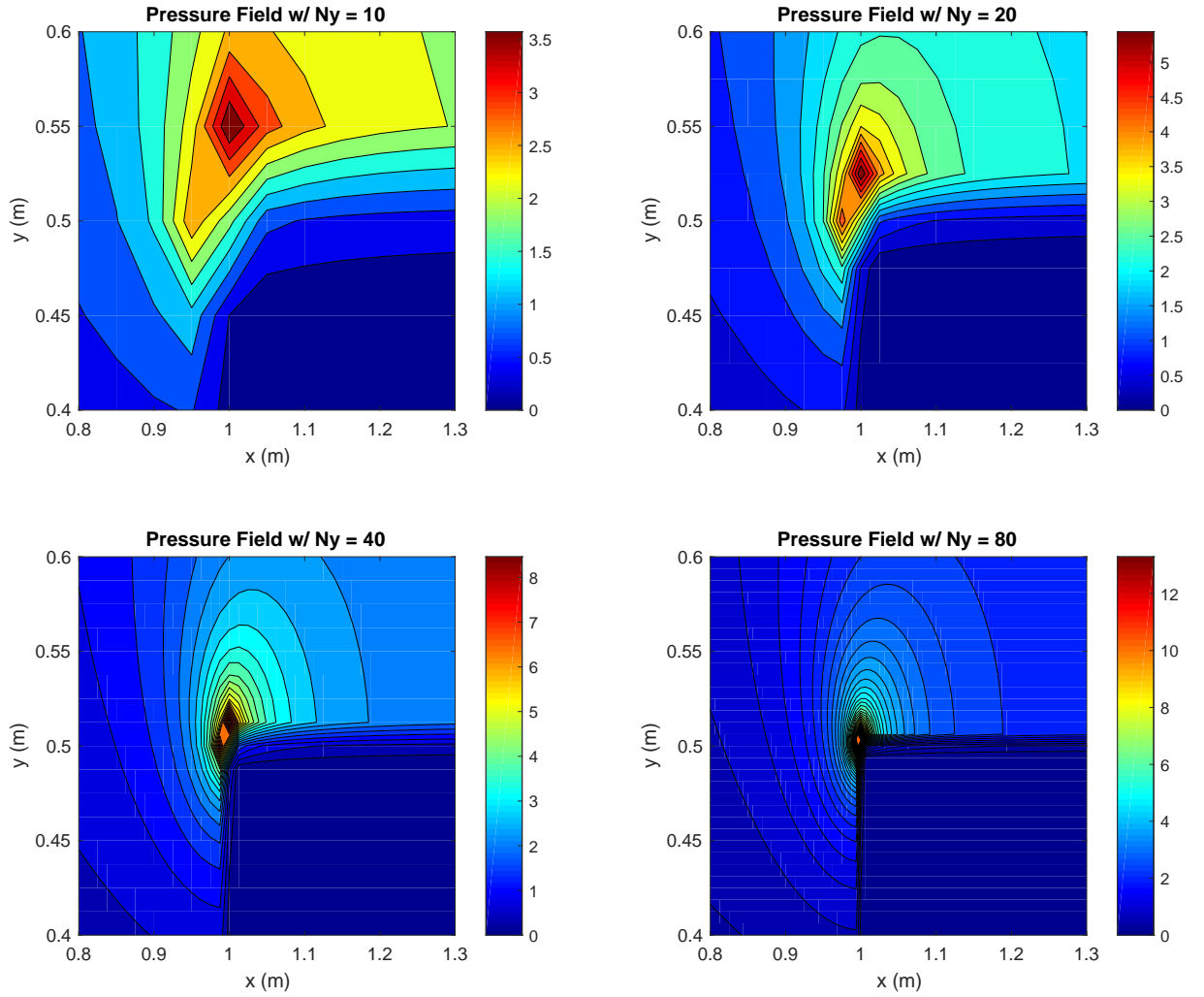


Figure 12: Pressure Field Zoomed Corner View

The extremely volatile pressure gradient seen at the corner attributes to convergence issues at the corner.

Solve the problem for $L=1$ [m], $H = 0.2$ [m], $V = 1$ [m/s] and $\rho = 1[kg/m^3]$. Perform a convergence study to determine the appropriate values of N_x and N_y . Comment on your solution and especially on the computed solution in the vicinity of the corner.

Finally, the evolution of the pressure along the vertical segment (side CD) is shown in the figures below for four different discretizations.

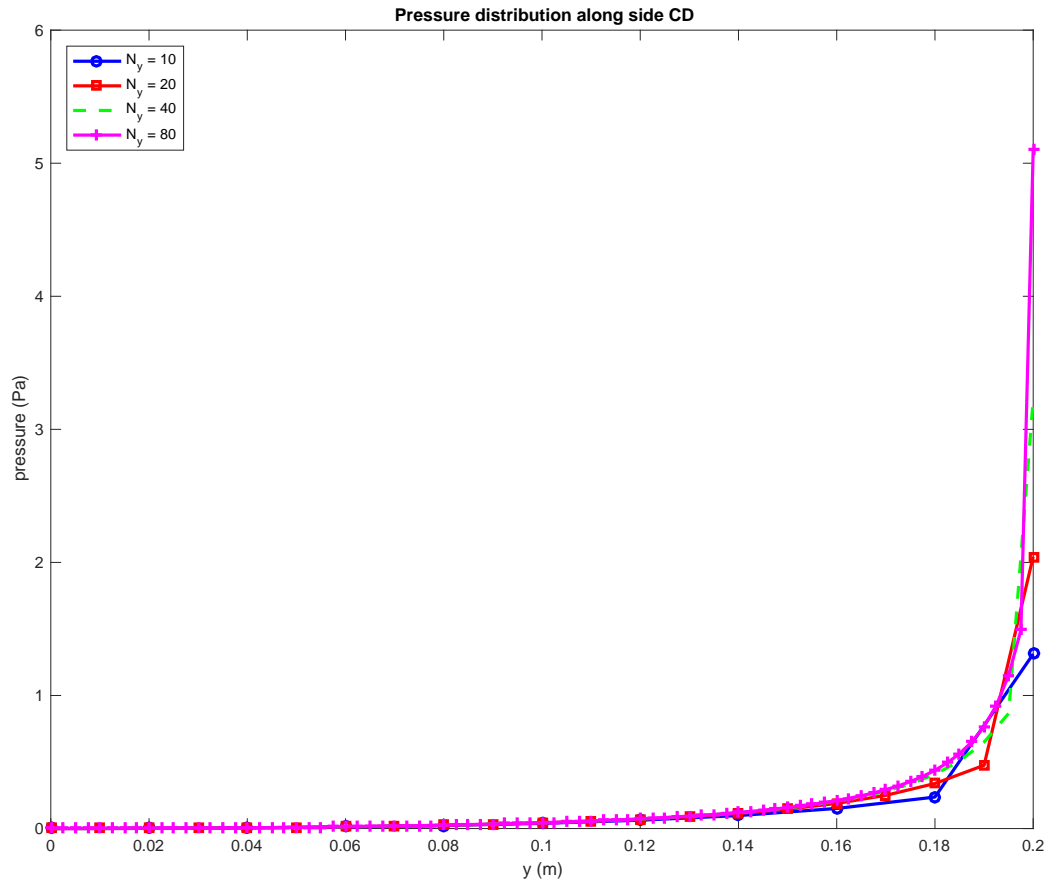


Figure 13: Pressure Values along CD

From these plots, it can be shown how the solution is convergent along the vertical step, diverging only when approaching the vicinity of the corner. This is a result of the extremely high gradient present at the corner. Figure 14 below examines more closely how the divergence is more apparent in the vicinity of the corner.

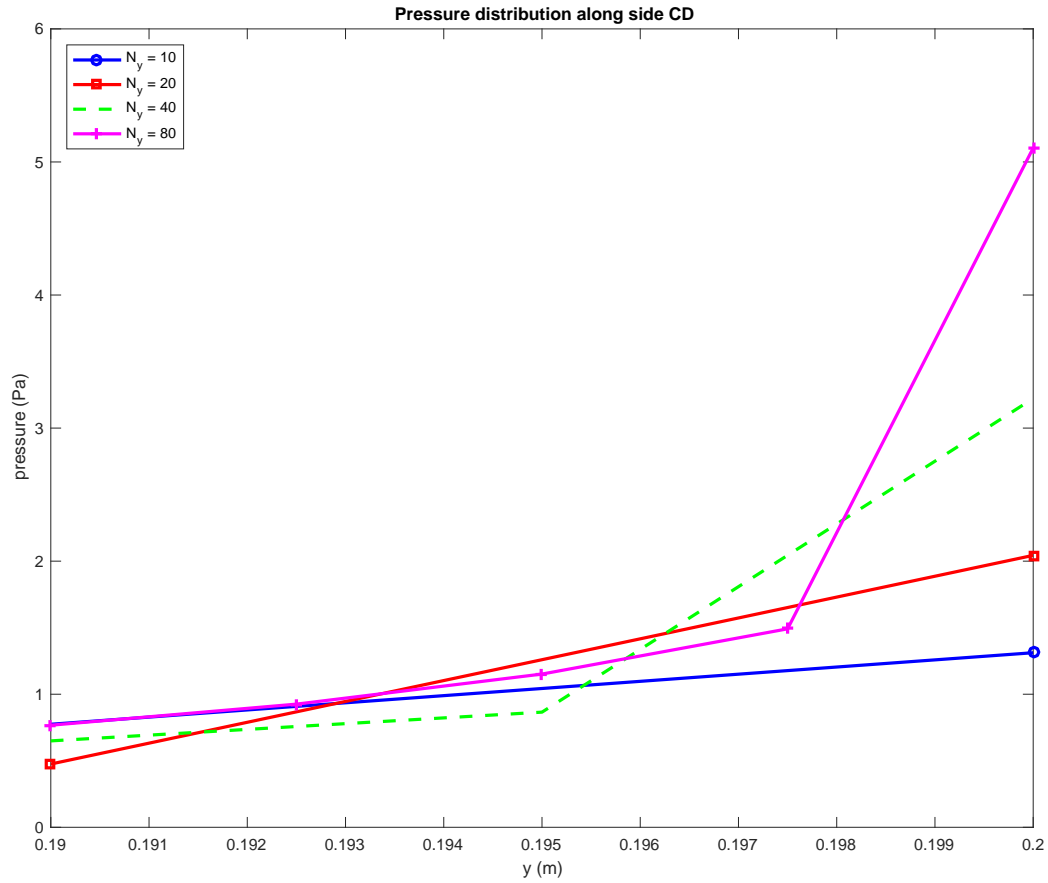


Figure 14: Zoomed Pressure Values along CD

Another way to solve for convergence of the entire system is to examine information regarding the exit velocity. It is understood that at the exit plane, the velocity at the top will be 2 m/s and at the lower end, 1 m/s. This means that the average output velocity is 1.5m/s. Using this information, we can compute the velocity for different subdivisions of N_y (keeping $N_x = 2 N_y$), compute the mean exit plane velocity and calculated the relative error, knowing that the exact average velocity must be 1.5.

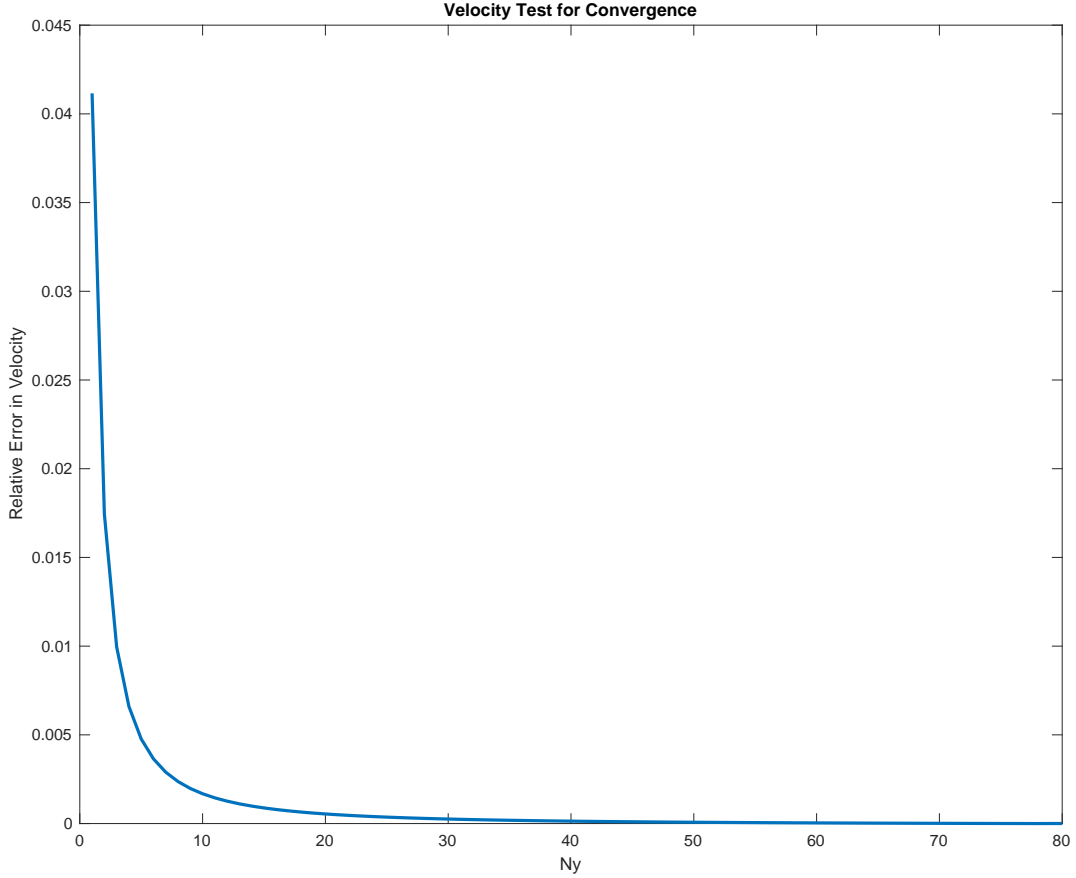


Figure 15: Relative Velocity Error at Exit Plane

From the relative velocity error study, an estimate for a good N_x and N_y values are:

$$(N_x, N_y) = (80, 40)$$

These values are also supported by the pressure study from Figure 12 and Figure 13. Though increasing the N_y and N_x values will produced better results, it can be shown that the estimated results from $(N_x, N_y) = (80, 40)$ is sufficient enough to provide accurate data at reasonable computation speeds.

Complete Code

```

1 % Program StepFlow
2 % Finite difference code to solve the problem of an inviscid,
3 % incompressible flow going over a step.
4 %
5 %
6 %      B                      E                      G
7 %      -----
8 %      |                      |
9 %      |                      |
10 %     |                      |
11 %     |                      |
12 %     |                      |
13 %     |                      D                      | F
14 %     |                      -----
15 %     |                      |
16 %     |                      |
17 %     |                      |
18 %     |                      |
19 %     |                      |
20 %     |                      |
21 %     -----
22 %     A                      C
23 %
24 % Dimensions:    AC = DF = BE = EG = L
25 %               CD = FG = DE = AB/2 = H
26 % The domain is discretized with Nx grid spacings along AC and DF
27 % and NY grid spacings along CD and FG.
28 % Index convention:
29 %      Local indices (i,j)          Global index (q)
30 %      A      (1,1)                  qA=1
31 %      B      (1,2*Ny+1)             qB=2*Ny+1
32 %      C      (Nx+1,1)               qC=Nx*(2*Ny+1)+1
33 %      D      (Nx+1,Ny+1)            qD=qC+Ny
34 %      E      (Nx+1,2*Ny+1)          qE=qD+Ny
35 %      F      (2*Nx+1,Ny+1)          qF=qE+(Nx-1)*(Ny+1)+1
36 %      G      (2*Nx+1,2*Ny+1)        qG=qF+Ny
37 %
38 function StepFlow
39 close all; clear all; clc;
40 % Key parameters
41 L = 1; %x-dimension of domain (m)
42 H = 0.2; %y-direction of step (m)
43
44 Ny1= 20; %input(' Enter number of grid spacings in y (Ny) ')
45 Nx1= 2*Ny1; %input(' Enter number of grid spacings in x (Nx) ')
46 Ny2 = 40;
47 Nx2 = 2*Ny2;
48 Ny3 = 80;
49 Nx3 = 2*Ny3;
50 Ny4 = 160;
51 Nx4 = 2*Ny4;
52

```

```

53 dx1=L/Nx1; % grid spacing in x direction
54 dy1=H/Ny1; % grid spacing in y direction
55 dx2=L/Nx2;
56 dy2=H/Ny2;
57 dx3=L/Nx3;
58 dy3=H/Ny3;
59 dx4=L/Nx4;
60 dy4=H/Ny4;
61
62 V=1; % imposed inflow velocity (in m/s) (the outflow velocity = 2V)
63 rho=1; % fluid density (in kg/m^3) (to compute the dynamic pressure)
64
65 Psimat1 = computeStream(Nx1, Ny1,L,H,V,dx1,dy1);
66 Psimat2 = computeStream(Nx2, Ny2,L,H,V,dx2,dy2);
67 Psimat3 = computeStream(Nx3, Ny3,L,H,V,dx3,dy3);
68 Psimat4 = computeStream(Nx4, Ny4,L,H,V,dx4,dy4);
69
70 plotContour(Psimat4,Nx4,Ny4,L,H)
71
72 figure(2)
73 plotVectorField(Psimat1,Nx1,Ny1,L,H,V,dx1,dy1)
74
75 figure(3)
76 subplot(2, 2,1);
77 plotContourPressures(Psimat1,Nx1,Ny1,L,H,V,dx1,dy1,rho)
78 subplot(2, 2,2);
79 plotContourPressures(Psimat2,Nx2,Ny2,L,H,V,dx2,dy2,rho)
80 subplot(2, 2,3);
81 plotContourPressures(Psimat3,Nx3,Ny3,L,H,V,dx3,dy3,rho)
82 subplot(2, 2,4);
83 plotContourPressures(Psimat4,Nx4,Ny4,L,H,V,dx4,dy4,rho)
84
85 figure(5)
86 subplot(2, 2,1);
87 plotContourPressuresZoomed(Psimat1,Nx1,Ny1,L,H,V,dx1,dy1,rho)
88 subplot(2, 2,2);
89 plotContourPressuresZoomed(Psimat2,Nx2,Ny2,L,H,V,dx2,dy2,rho)
90 subplot(2, 2,3);
91 plotContourPressuresZoomed(Psimat3,Nx3,Ny3,L,H,V,dx3,dy3,rho)
92 subplot(2, 2,4);
93 plotContourPressuresZoomed(Psimat4,Nx4,Ny4,L,H,V,dx4,dy4,rho)
94
95 figure(4)
96 plotPressures(Psimat1,Psimat2, Psimat3, Psimat4,Nx1,Ny1,Nx2,Ny2,Nx3,Ny3,Nx4,Ny4,H,V,dx1,dy1,
    dx2,dy2,dx3,dy3,dx4,dy4,rho)
97
98 end
99
100 function [Psimat] = computeStream(Nx, Ny, L, H,V,dx,dy)
101 eta=dx/dy; % grid spacing ratio
102 rho=1; % fluid density (in kg/m^3) (to compute the dynamic pressure)
103
104 % Compute global equation number of corners (see schematic above)
105 qA=1;

```

```

106 qB=2*Ny+1;           % N
107 qC=Nx*(2*Ny+1)+1; % Nx*(qB)+1
108 qD=qC+Ny;
109 qE=qD+Ny;
110 qF=qE+(Nx-1)*(Ny+1)+1;
111 qG=qF+Ny;
112 Numeq=qG;           % number of equations
113 %
114 % Set up matrix (Amat) and vector (bvec) dimensions
115 Amat=sparse(Numeq,Numeq);
116 bvec=zeros(Numeq,1);
117 % Build linear system
118 for i=1:Numeq;       % place 1 along diagonal for all DOF then overwrite the interior grid
    points
119     Amat(i,i)=1;
120 end
121 for i=2:Nx % loop over interior grid points — left half of domain
122     for j=2:2*Ny
123         qij=compute_q(i,j,Nx,Ny); % compute equation number q for(i,j) grid point
124         Amat(qij,qij)=-2*(1+eta^2);%Grid Point
125         q=compute_q(i-1,j,Nx,Ny); %Grid Point to the left
126         Amat(qij,q)= 1;
127         q=compute_q(i+1,j,Nx,Ny); %Grid Point to the right
128         Amat(qij,q)= 1;
129         q=compute_q(i,j-1,Nx,Ny); % Grid point below
130         Amat(qij,q)= eta^2;
131         q=compute_q(i,j+1,Nx,Ny); %Grid Point above
132         Amat(qij,q)= eta^2;
133     end
134 end
135 for i=Nx+1:2*Nx % loop over interior grid points — right half of domain
136     for j=Ny+2:2*Ny
137         qij=compute_q(i,j,Nx,Ny);
138         Amat(qij,qij)= -2*(1+eta^2);
139         q=compute_q(i-1,j,Nx,Ny);
140         Amat(qij,q)= 1;
141         q=compute_q(i+1,j,Nx,Ny);
142         Amat(qij,q)= 1;
143         q=compute_q(i,j-1,Nx,Ny);
144         Amat(qij,q)= eta^2;
145         q=compute_q(i,j+1,Nx,Ny);
146         Amat(qij,q)= eta^2;
147     end
148 end
149
150
151 %
152 % Build right-hand-side vector (imposed Psi BC)
153 for j=1:2*Ny+1 % loop over the left edge:
154     bvec(j)= V*(j-1)*dy;
155 end
156 for i=2:2*Nx % loop over top edge:
157     j=2*Ny+1;
158     q=compute_q(i,j,Nx,Ny);

```

```

159     bvec(q)= 2*V*H;
160 end
161 for j=Ny+1:2*Ny+1 % loop over right edge:
162     i=2*Nx+1;
163     q=compute_q(i,j,Nx,Ny);
164     y = (j-1)*dy;
165     bvec(q)= 2*V*(y-H);
166 end
167 % Remainder of boundary has Psi=0
168 %
169 % Solve linear system
170 Psivec=Amat\bvec;
171 % Build Psi array for visualization (fill lower right with zero
172 Psimat=zeros(2*Nx+1,2*Ny+1);
173 for i=1:2*Nx+1 % loop over all points in domain
174     for j=1:2*Ny+1
175         q=compute_q(i,j,Nx,Ny);
176         if q == 0 % if inside step region, assign Phi=0
177             Psimat(i,j)=0;
178         else % if inside computational domain, extra Phi value from solution vector
179             Psimat(i,j)=Psivec(q);
180         end
181     end
182 end
183 end
184
185 function q=compute_q(i,j,Nx,Ny)
186 % Subroutine to compute the global equation number corresponding to grid
187 % point (i,j)
188 q = 0;
189 if(i <= Nx +1)
190     q = (i-1)*(2*Ny+1)+j;
191 elseif(j >= Ny + 1)
192     q = (Nx + 1)*(2*Ny+1)+(i-Nx-2)*(Ny+1)+j-Ny;
193 end
194 end
195
196 function plotContour(Psimat,Nx,Ny,L,H)
197
198 xvec=linspace(0,2*L,2*Nx+1); % vector with 2*Nx+1 values of x
199 yvec=linspace(0,2*H,2*Ny+1); % vector with 2*Ny+1 values of y
200 contourf(xvec,yvec,Psimat',15) % create filled contour plots of phi field
201 colormap(jet);
202 xlabel('x (m)');
203 ylabel('y (m)');
204 title('Contour plot of function Phi');
205 set(gcf,'paperorientation','landscape');
206 set(gcf,'paperunits','normalized');
207 set(gcf,'paperposition',[0 0 1 1]);
208 print(gcf,'-dpdf','contourStream.pdf');
209 end
210
211 function [vx,vy] = Velocity(Psimat,Nx,Ny,V,dx,dy)
212 vx=zeros(2*Nx+1,2*Ny+1); % x-component of velocity at each grid point

```

```

213 vy=zeros(2*Nx+1,2*Ny+1);    % y-component of velocity at each grid point
214
215 for j=1:2*Ny+1    % left edge
216     vx(1,j)=V;
217     vy(1,j)=0;
218 end
219 for j=Ny+1:2*Ny+1    % right edge
220     vx(2*Nx+1,j)=2*V;
221     vy(2*Nx+1,j)=0;
222 end
223 for i=2:2*Nx    % top edge (backward difference)
224     vx(i,2*Ny+1)=(Psimat(i,2*Ny+1)-Psimat(i,2*Ny))/dy;
225     vy(i,2*Ny+1)=0;
226 end
227 for i=2:Nx    % bottom edge (forward difference)
228     vx(i,1)=(Psimat(i,2)-Psimat(i,1))/dy;
229 end
230 for i=2:2*Nx    % interior nodes
231     for j=2:2*Ny
232         vx(i,j)=(Psimat(i,j+1)-Psimat(i,j-1))/2/dy;
233         vy(i,j)=-(Psimat(i+1,j)-Psimat(i-1,j))/2/dx;
234     end
235 end
236 end
237
238
239 function plotVectorField(Psimat,Nx,Ny,L,H,V,dx,dy)
240 % Compute and display velocity vector field
241
242 xvec=linspace(0,2*L,2*Nx+1);    % vector with 2*Nx+1 values of x
243 yvec=linspace(0,2*H,2*Ny+1);
244
245 [vx,vy]=Velocity(Psimat,Nx,Ny,V,dx,dy);
246
247 [x,y]=meshgrid(xvec,yvec);
248 quiver(x',y',vx,vy);    % create vector plot
249 xlabel('x (m)');
250 ylabel('y (m)');
251 axis([0 2 0 1])
252 title(' Velocity vector plot');
253 set(gcf,'paperorientation','landscape');
254 set(gcf,'paperunits','normalized');
255 set(gcf,'paperposition',[0 0 1 1]);
256 print(gcf,'-dpdf','vfield.pdf');
257 end
258
259
260 function plotContourPressures(Psimat,Nx,Ny,L,H,V,dx,dy,rho)
261 [vx,vy] = Velocity(Psimat,Nx,Ny,V,dx,dy);
262 % Compute and display pressure field
263 xvec=linspace(0,2*L,2*Nx+1);    % vector with 2*Nx+1 values of x
264 yvec=linspace(0,2*H,2*Ny+1);
265
266 pressure=0.5*rho*(vx.^2+vy.^2);    % pressure array

```

```

267 contourf(xvec,yvec,pressure',30)      % filled contour plot of pressure field
268 colormap(jet);
269 colorbar
270 xlabel('x (m)');
271 ylabel('y (m)');
272 title(sprintf('Pressure Field w/ Ny = %d',Ny));
273 set(gcf,'paperorientation','landscape');
274 set(gcf,'paperunits','normalized');
275 set(gcf,'paperposition',[0 0 1 1]);
276 print(gcf,'-dpdf','contourPressure.pdf');
277 end
278
279 function plotContourPressuresZoomed(Psimat,Nx,Ny,L,H,V,dx,dy,rho)
280 [vx,vy] = Velocity(Psimat,Nx,Ny,V,dx,dy);
281 % Compute and display pressure field
282 xvec=linspace(0,2*L,2*Nx+1);      % vector with 2*Nx+1 values of x
283 yvec=linspace(0,2*H,2*Ny+1);
284
285 for i = 1:length(xvec)
286     if xvec(i) == 0.8
287         xLow = i
288     end
289     if xvec(i) == 1.3
290         xHigh = i
291     end
292 end
293 for i = 1:length(yvec)
294     if yvec(i) == 0.4
295         yLow = i
296     end
297     if yvec(i) == 0.6
298         yHigh = i
299     end
300 end
301 pressure=0.5*rho*(vx.^2+vy.^2);      % pressure array
302 contourf(xvec(xLow:xHigh),yvec(yLow:yHigh),pressure(xLow:xHigh,yLow:yHigh)',(Ny/2)+5)      %
    % filled contour plot of pressure field
303 colormap(jet);
304 colorbar
305 xlabel('x (m)');
306 ylabel('y (m)');
307 title(sprintf('Pressure Field w/ Ny = %d',Ny));
308 set(gcf,'paperorientation','landscape');
309 set(gcf,'paperunits','normalized');
310 set(gcf,'paperposition',[0 0 1 1]);
311 print(gcf,'-dpdf','contourPressureZoomed.pdf');
312 end
313
314 function plotPressures(Psimat1,Psimat2, Psimat3, Psimat4,Nx1,Ny1,Nx2,Ny2,Nx3,Ny3,Nx4,Ny4,H,V
    ,dx1,dy1,dx2,dy2,dx3,dy3,dx4,dy4,rho)
315 [vx1,vy1] = Velocity(Psimat1,Nx1,Ny1,V,dx1,dy1);
316 [vx2,vy2] = Velocity(Psimat2,Nx2,Ny2,V,dx2,dy2);
317 [vx3,vy3] = Velocity(Psimat3,Nx3,Ny3,V,dx3,dy3);
318 [vx4,vy4] = Velocity(Psimat4,Nx4,Ny4,V,dx4,dy4);

```



```

319
320 pressure1=0.5*rho*(vx1.^2+vy1.^2)
321 pressure2=0.5*rho*(vx2.^2+vy2.^2);
322 pressure3=0.5*rho*(vx3.^2+vy3.^2);
323 pressure4=0.5*rho*(vx4.^2+vy4.^2);
324
325 yvalues1=linspace(0,H,Ny1+1);      % vector with Ny+1 values of y along CD
326 yvalues2=linspace(0,H,Ny2+1);
327 yvalues3=linspace(0,H,Ny3+1);
328 yvalues4=linspace(0,H,Ny4+1);
329
330 pvalues1(1:Ny1+1)=pressure1(Nx1+1,1:Ny1+1) % extract pressure values along CD from pressure
      array
331 pvalues2(1:Ny2+1)=pressure2(Nx2+1,1:Ny2+1);
332 pvalues3(1:Ny3+1)=pressure3(Nx3+1,1:Ny3+1);
333 pvalues4(1:Ny4+1)=pressure4(Nx4+1,1:Ny4+1);
334
335 plot(yvalues1,pvalues1,'bo-', 'linewidth',2)
336 hold on
337 plot(yvalues2,pvalues2,'rs-', 'linewidth',2)
338 hold on
339 plot(yvalues3,pvalues3,'g—', 'linewidth',2)
340 hold on
341 plot(yvalues4,pvalues4,'m+-', 'linewidth',2)
342 xlabel('y (m)');
343 ylabel('pressure (Pa)');
344 %axis([0.19 0.2 0 6])
345 title(' Pressure distribution along side CD');
346 legend(sprintf('N_y = %d',Ny1),sprintf('N_y = %d',Ny2),sprintf('N_y = %d',Ny3),sprintf('N_y
      = %d',Ny4), 'Location', 'northwest')
347 set(gcf, 'paperorientation', 'landscape');
348 set(gcf, 'paperunits', 'normalized');
349 set(gcf, 'paperposition', [0 0 1 1]);
350 print(gcf, '-dpdf', 'PressurePlots.pdf');
351 end

```