
SWIVL: Screw-Wrench informed Impedance Variable Learning for bimanual manipulation of an articulated object

Kyungseo Park

Student ID : 2020-12295

Department of Mechanical Engineering

Seoul National University

erickun0125@snu.ac.kr

Abstract

Bimanual manipulation of articulated objects remains challenging due to inter-arm force coupling and complex kinematic constraints. While recent Vision-Language-Action (VLA) models demonstrate strong cognitive intelligence in high-level task planning, they lack explicit grounding in low-level physical interaction—specifically, force regulation and constraint satisfaction during contact-rich manipulation. We introduce **SWIVL** (Screw-Wrench informed Impedance Variable Learning), a task-agnostic framework that bridges the gap between high-level cognitive planning and low-level physical execution. SWIVL consists of two key components: (1) a **Screw-Decomposed SE(3) Impedance Controller** that maps constraint-unaware desired motions (e.g., action chunk from VLAs) into constraint-compatible compliant motions by leveraging the object’s instantaneous screw axis—decomposing commanded velocities into *internal motion* aligned with allowable kinematic freedoms and *bulk motion* in the constraint-violation subspace, then applying impedance control in each decomposed manifold, and (2) a **Wrench-Adaptive Impedance Variable Learning Policy** that learns to modulate impedance variables (stiffness/damping parameters) via real-time wrench feedback and object information conditioning, enabling force-compliant, physically feasible commands. We evaluate SWIVL on SE(2) planar benchmarks involving rigid transport, revolute rotation, and prismatic insertion, demonstrating improved success rates, reduced internal forces, and robust generalization across diverse planners and object geometries compared to imitation learning baselines.

1 Introduction

Learning-based robotic manipulation has achieved strong performance in single-arm settings, particularly for structured pick-and-place tasks learned via demonstration. However, extending such approaches to **dual-arm manipulation of an articulated object** remains challenging. Coordinated bimanual interaction induces rich **inter-arm force coupling** and must satisfy complex **object-centric kinematic constraints**. Existing Learning-from-Demonstration (LfD) frameworks—typically grounded in high-stiffness position control with no explicit representation of object kinematics—often generate unstable motions and large internal forces when multiple arms physically interact with a shared object.

Cognitive vs. Physical Intelligence

To reason about this challenge, we distinguish between two complementary aspects of robot decision-making: **Cognitive Intelligence** and **Physical Intelligence**.

Cognitive Intelligence addresses high-level task understanding through semantic reasoning and planning. Modern VLA-based robot foundation models excel at interpreting goals and decomposing instructions via visual-language understanding based on VLM backbones. However, language operates as an abstracted symbolic representation that lacks the resolution needed for precise physical interaction—making it difficult to specify fine-grained motor commands for contact-rich manipulation such as force modulation or coordinated compliance control.

Physical Intelligence, in contrast, ensures safe and stable execution through explicit modeling of dynamics and kinematic principles. This includes regulating contact forces, satisfying geometric constraints, and generating smooth, dynamically consistent motions—critical requirements for force-coupled bimanual manipulation.

Most existing work advances on Cognitive Intelligence through robot foundation models that leverage broad cross-embodiment datasets for rich semantic understanding. However, cross-embodiment training makes it difficult to standardize low-level physical signals—such as end-effector wrench feedback, reference frame definitions, and kinematic constraints—due to varying sensor types and frame conventions across robots. Consequently, foundation models lack explicit access to such information, limiting their ability to reason about contact forces, enforce geometric constraints, or maintain stable cooperative interaction during manipulation.

Moreover, imitation-driven paradigms are well-suited for learning human-like task schemas, but low-level control in physically constrained environments requires instinctive, dynamically consistent behavior, which we find is better realized through reinforcement learning rather than imitation learning.

This mismatch motivates methods that **bridge high-level cognitive policies to low-level physically grounded control**.

Problem Focus and Scope

To this end, we focus on developing a low-level control stack for bimanual manipulation of an articulated object. Our formulation assumes access to (1) 6-axis wrench measurements at each end-effector via wrist-mounted force/torque sensors, and (2) screw axes of an articulated object. While not all manipulation scenarios provide such information, structured domains—including repetitive assembly, industrial workflows, and environments with perception modules (e.g., screw-splating for axis extraction from point clouds)—reasonably support these assumptions.

In this setting, high-level planners (behavior-cloned policies, teleoperation, or VLA) provide motion intentions as desired end-effector poses, without explicitly reasoning about object’s physical information or inter-arm force interactions. Conventional high-stiffness position controllers that directly track these desired poses generate excessive contact forces and constraint violations when manipulating articulated objects, as they lack mechanisms to accommodate unforeseen contact dynamics or kinematic restrictions. This creates a fundamental challenge: translating high-level guidance into physically feasible bimanual coordination that simultaneously maintains kinematic consistency and suppresses harmful internal forces while ensuring compliant interaction in task space.

To address these problems, we require a control framework that satisfies four key requirements:

- (1) **Dense, closed-loop reference generation:** Transform sparse, open-loop waypoints from high-level planners into continuous, feedback-driven reference trajectories that provide corrective guidance when the system deviates from desired paths.
- (2) **Kinematic constraint satisfaction:** Ensure commanded motions respect the object’s joint structure, preventing physically infeasible trajectories that violate holonomic constraints and cause grasp slippage or internal stress.
- (3) **Screw-decomposed compliance modulation:** Enable independent impedance control for internal motion (joint articulation) versus bulk motion (overall transport), allowing task-appropriate compliance without manual tuning across the entire $SE(3)$ manifold.
- (4) **Wrench-feedback force regulation:** Identify and actively minimize non-productive internal wrenches arising from coordination errors, while maintaining necessary productive forces for manipulation.

Our Approach: SWIVL

To address these requirements, we introduce **SWIVL** (**Screw-Wrench informed Impedance Variable Learning**), a **task-agnostic low-level control stack for bimanual manipulation of articulated objects** that operationalizes Physical Intelligence through explicit modeling of object kinematics and wrench feedback.

SWIVL consists of three key learned components that directly address the four requirements above: (1) a **Reference Motion Field Generator** that transforms discrete high-level waypoints into dense, stable vector fields with corrective feedback (Requirement 1), (2) a **Screw-decomposed Impedance Controller** that structurally enforces kinematic constraints through orthogonal motion decomposition (Requirements 2 & 3), and (3) a **Reinforcement Learning Policy** trained to modulate impedance variables based on object geometry and wrench feedback, minimizing internal forces while ensuring compliant manipulation (Requirements 3 & 4).

This policy is trained via **Reinforcement Learning**, which enables the agent to autonomously discover force-compliant behaviors by directly optimizing physical objectives—such as minimizing internal forces and satisfying kinematic constraints—through environmental interaction, rather than merely replicating demonstrated trajectories. By translating high-level trajectories into a **Reference Motion Field**, SWIVL allows cognitive planners to remain expressive while ensuring safety and compliance during manipulation, making it possible to operate underneath arbitrary high-level cognitive planners with any high-level task objectives.

We construct the methodology of SWIVL on $SE(3)$ that is the task space of manipulation task. Also, we evaluate SWIVL on $SE(2)$ planar benchmark environment designed to isolate the core challenges of force coupling and constraint satisfaction. Our $SE(2)$ experiments on bimanual manipulation tasks involving rigid transport, articulated object rotation, and insertion demonstrate that SWIVL: improves success rates, reduces inter-arm internal forces, and generalizes across planners, task objectives, and object geometries—outperforming LfD-based baselines. These results demonstrate that **physics-aware learning** is essential for robust cooperative manipulation and enables seamless integration with arbitrary high-level planners without task-specific retraining.

Contributions

This work makes the following contributions:

1. **Stable Imitation Vector Field:** We propose a temporally-indexed contraction field that provides $O(1)$ reference motion generation with guaranteed exponential convergence, enabling real-time bimanual coordination at 50Hz. It enables decoupling of high-level planning from low-level force-compliant execution.
2. **Screw-Decomposed Impedance Control:** We formulate an $SE(3)$ impedance controller with orthogonal motion decomposition that structurally enforces kinematic constraints while enabling independent compliance modulation for bulk versus internal motion, addressing fundamental geometric limitations of classical $SE(3)$ impedance control.
3. **Task-Agnostic RL Framework:** We introduce a reinforcement learning framework that learns adaptive impedance modulation conditioned on object geometry and wrench feedback, enabling generalization across joint types, task objectives, and novel object geometries without task-specific retraining.
4. **Object-Conditioned Policy Architecture:** We design a FiLM-conditioned neural network that injects object kinematic structure into all feature processing stages, enabling constraint-aware representation learning for articulated object manipulation.
5. **Comprehensive Evaluation Framework:** We establish a systematic $SE(2)$ benchmark with controlled ablation studies, demonstrating the necessity of explicit physical modeling for robust bimanual manipulation (pending experimental validation).

Overall, our work provides a principled framework toward dual-arm robotic systems for force-compliant articulated object manipulation.

2 Related Work

Our work bridges research in learning-based robotic manipulation, dual-arm coordination, physically grounded control, and stable motion representation. We organize related work around four themes: **learning paradigms for manipulation**, **stable motion representation and geometric control**, **dual-arm coordination and force control**, and **hierarchical integration of high-level planning with low-level execution**.

2.1 Learning-Based Robotic Manipulation

Imitation Learning and Behavior Cloning. Learning from Demonstration (LfD) has emerged as a dominant paradigm for acquiring manipulation skills, with Behavior Cloning (BC) directly mapping observations to actions through supervised learning. Recent LfD policies use generative models like Diffusion Policy [1] and ACT [2] to output action chunks for temporal consistency. However, these approaches focus on mimicking demonstrations in high-stiffness position control without reasoning about contact forces or kinematic constraints, and lack closed-loop feedback to handle trajectory deviations.

Vision-Language-Action Models and Foundation Models. Cross-embodiment robot foundation models like RT-1 [3], RT-2 [4], π_0 [5], and Octo [6] leverage large-scale datasets like Open-X [7] for broad semantic understanding in cognitive capabilities. While these models excel at high-level goal specification and demonstrate impressive generalization to novel objects and instructions, they are trained without explicit access to force/moment feedback, reference frames, or kinematic constraints—information that is inherently embodiment-specific and difficult to standardize across heterogeneous robot platforms. This limitation motivates our approach of decoupling high-level cognitive planning from low-level physically grounded execution.

RL for Manipulation. Reinforcement learning has shown promise for acquiring contact-rich manipulation skills that are difficult to specify through demonstrations alone. Deep RL has been applied to contact-rich tasks like in-hand manipulation [8] and tool use [9], optimizing physical objectives through environmental interaction. However, most RL formulations focus on single-task settings and require dense, task-specific reward signals. We leverage RL to learn task-agnostic low-level control that generalizes across manipulation tasks.

2.2 Stable Motion Representation and Geometric Control

Dynamical Systems-Based Imitation Learning. SEDS [10] learns globally stable dynamical systems from demonstrations with Lyapunov stability guarantees. Extensions include kernelized representations [11], obstacle avoidance [12], and neural parameterizations [13]. These methods provide closed-loop corrective behavior, but most operate in Euclidean spaces and lack temporal synchronization for time-indexed trajectories.

Geometric Control on Lie Groups. Geometric control theory [14] and screw theory [15] provide principled frameworks for manipulation in $SE(3)$. Recent works apply these tools to planning [16], constrained motion [17], and dual-arm coordination [18]. SWIVL integrates screw-theoretic constraints into RL action spaces, enabling learned policies that respect geometric structure by construction.

2.3 Dual-Arm Manipulation and Force Control

Classical Dual-Arm Coordination. Hybrid position/force control [19] decomposes control into position and force subspaces, while object-centric approaches [20] unify dual-arm systems as virtual closed-chain mechanisms. Screw theory [15] provides geometric foundations for kinematic constraints. While these classical methods provide theoretical foundations for dual-arm coordination, they typically rely on precise analytical models and struggle with complex contact dynamics in articulated object manipulation.

Impedance and Compliant Control. Impedance control [21] regulates position-force relationships through virtual mass-spring-damper dynamics. Object impedance control [22] extends this to dual-arm settings. However, these methods assume rigid objects and require manual tuning, limiting applicability to articulated manipulation with varying constraints.

Learning Force-Aware Manipulation. Recent works learn force-sensitive skills through force-conditioned policies ? and tactile guidance ?. However, these approaches typically learn task-specific force patterns rather than generalizable force-compliant behaviors. SWIVL learns task-agnostic force regulation by incorporating screw-axis-based wrench decomposition into the reward structure.

2.4 Hierarchical Integration of Cognitive and Physical Control

Residual Reinforcement Learning. To bridge the gap between high-level imitation policies and low-level reactive control, residual RL methods augment frozen behavior-cloned policies with learned corrective actions. Residual RL methods like ResiP ? augment frozen BC policies with learned corrective actions, protecting against catastrophic forgetting while handling distribution shifts. However, these focus on single-task manipulation without addressing multi-arm coordination or force regulation. SWIVL learns a standalone low-level policy that operates beneath arbitrary high-level planners.

Diffusion Guidance. Diffusion models can incorporate constraints through guidance mechanisms ????. However, guidance requires differentiable constraints, careful tuning, and provides no hard guarantees—unsuitable for strict kinematic constraints where violations cause task failure. SWIVL structurally enforces constraints through action space design, guaranteeing holonomic constraint satisfaction by construction.

Hierarchical Control for Whole-Body Systems. High-level planners (VLAs, teleoperation) generate semantic task specifications, but translating these into dynamically feasible whole-body motions requires robust low-level control. Hierarchical approaches separate high-level planning from low-level execution: LeVERB ? learns latent action vocabularies translated by RL policies, while locomanipulation frameworks ? use RL to track commands while maintaining stability. SWIVL adopts similar philosophy for dual-arm manipulation, where high-level planners provide pose commands and RL-trained low-level policy ensures physically feasible execution through explicit constraint and force modeling.

3 Method

We present **SWIVL**, a hierarchical control framework that bridges high-level cognitive planning with physically grounded bimanual execution. SWIVL consists of three key components: (1) a Stable Imitation Vector Field based **Reference Motion Field Generator** that transforms discrete high-level waypoints into dense, continuous vector fields defined over the entire task space—providing stable reference motions even when the system deviates from the desired trajectory, (2) Screw Axes Decomposition based **Twist-driven Impedance Controller** that enables practical impedance control with pose error and twist error, and (3) a Reinforcement Learning based **Wrench-feedback and Object-conditioned Impedance Variable Learning Policy** that modulates the reference motions in a physically feasible manner by explicitly incorporating object information and end-effector wrench feedback.

Notation. We use the following frame conventions throughout:

- $\{s\}$: Spatial (world) frame
- $\{b_i\}$: Body frame of end-effector $i \in \{l, r\}$
- $\{d_i\}$: Desired frame of end-effector i
- T_{ab} : Transformation from frame $\{b\}$ to frame $\{a\}$
- ${}^a\mathcal{V}_b$: Twist of frame $\{b\}$ expressed in frame $\{a\}$

3.1 Problem Formulation

We address the problem of **bimanual manipulation of articulated objects** where two robot arms cooperatively manipulate a shared articulated object (e.g., rotating pan-tilt camera, pumping with an air pump). Our goal is to develop SWIVL, a control framework that tracks high-level motion plans while ensuring physical feasibility, stable grasping, and damage prevention during bimanual interaction.

This problem presents four core challenges that directly motivate SWIVL’s architectural components: (1) **generating dense, closed-loop references** from sparse high-level waypoints, (2) **satisfying kinematic constraints** imposed by the object’s joint structure, (3) **decomposing and modulating compliance** for bulk versus internal motions on the $SE(3)$ manifold, and (4) **regulating inter-arm forces** to minimize harmful internal wrenches. These challenges are addressed respectively by SWIVL’s Reference Motion Field Generator (Section 3.2.2), Screw-decomposed Controller structure (Section 3.2.4), learned impedance variables (Section 3.2.3), and wrench-based reward design (Section 3.3.2).

Framework Scope. We develop the theoretical framework in $SE(3)$ for generality and real-world applicability, while experimental validation (Section ??) is conducted in $SE(2)$ planar environments to isolate core challenges of force coupling and constraint satisfaction. The formulation targets the Franka FR3 dual-arm platform for future real-world deployment and focuses on objects with revolute and prismatic joints, which constitute the majority of practical articulated object manipulation tasks.

3.1.1 Challenge 1: From Sparse Waypoints to Dense, Closed-Loop Control

Modern high-level policies—including vision-language-action (VLA) models, behavior cloning architectures, and teleoperation interfaces—predominantly output actions in the form of action chunks: sequences of waypoints that specify desired end-effector trajectories over a finite horizon. Regardless of the internal mechanisms of these diverse planners, SWIVL operates on a unified interface that accepts action chunks as input.

Action Chunk Representation. We assume that the high-level policy outputs discrete end-effector pose sequences for both arms:

$$\mathcal{T}_{\text{des}} = \{(T_{sd_l}[\tau], T_{sd_r}[\tau])\}_{\tau=0}^H, \quad (1)$$

where $T_{sd_l}[\tau], T_{sd_r}[\tau] \in SE(3)$ represent the desired left and right end-effector poses (from spatial frame $\{s\}$ to desired frame $\{d_i\}$) at discrete time index τ , and H is the action chunk horizon. This action chunking paradigm, widely adopted in diffusion policies and autoregressive generative models, enables temporal consistency and multi-step reasoning in high-level planning.

The Challenge. These action chunks serve as desired waypoint trajectories, but present three critical gaps for low-level execution: (1) **sparsity**—waypoints are discrete while control requires continuous dense references at high frequency, (2) **open-loop nature**—when the system deviates from the desired trajectory due to tracking errors or disturbances, fixed waypoints cannot provide corrective feedback to guide the system back, and (3) **physical infeasibility**—high-level planners often generate trajectories without explicit knowledge of object constraints or force considerations, potentially violating kinematic constraints or inducing excessive internal forces.

Solution Preview. This challenge is addressed by the **Reference Motion Field Generator** (Section 3.2.2), which performs $SE(3)$ trajectory smoothing, computes body twists, and constructs a stable imitation vector field that provides dense, continuous reference motions with corrective feedback capabilities.

3.1.2 Challenge 2: Kinematic Constraints in Bimanual Manipulation

When two arms grasp and manipulate an articulated object, their motions are no longer independent but coupled through the object’s kinematic structure. This coupling fundamentally constrains what motions are physically feasible and must be explicitly modeled for successful bimanual control.

System Configuration. We consider a dual-arm system manipulating articulated objects with k internal degrees of freedom. Let $T_{sb_l}, T_{sb_r} \in SE(3)$ denote the left and right end-effector poses (from spatial frame $\{s\}$ to body frame $\{b_i\}$) in the spatial (world) frame, and $\mathbf{q}_{\text{obj}} \in \mathbb{R}^k$ represent the object’s internal joint configuration vector (e.g., multi-link scissor angles, cabinet with multiple drawers, or complex mechanisms with multiple revolute and prismatic joints).

Spatial Jacobian. The kinematic structure of the articulated object is characterized by its **spatial Jacobian** $J_s(\mathbf{q}_{\text{obj}}) \in \mathbb{R}^{6 \times k}$, which encodes the relationship between the object’s internal joint velocities and the resulting relative motion between the two grasp points. Each column of J_s represents a spatial screw axis $\mathcal{S}_j \in \mathbb{R}^6$ corresponding to the j -th joint:

$$J_s(\mathbf{q}_{\text{obj}}) = [\mathcal{S}_1(\mathbf{q}_{\text{obj}}) \quad \mathcal{S}_2(\mathbf{q}_{\text{obj}}) \quad \cdots \quad \mathcal{S}_k(\mathbf{q}_{\text{obj}})], \quad (2)$$

where each screw axis $\mathcal{S}_j = \begin{bmatrix} \omega_j \\ v_j \end{bmatrix}$ describes the instantaneous motion induced by unit velocity of joint j , expressed in the spatial frame. For revolute joints, \mathcal{S}_j represents the axis of rotation and a point on the axis; for prismatic joints, it represents the direction of translation.

Holonomic Constraint. The object’s kinematic structure imposes a holonomic constraint that couples the end-effectors’ velocities. Since both end-effectors are rigidly grasping the articulated object, the **relative motion** between them must exactly match the motion generated by the object’s internal joints:

$${}^s\mathcal{V}_{b_l} - {}^s\mathcal{V}_{b_r} = J_s(\mathbf{q}_{\text{obj}})\dot{\mathbf{q}}_{\text{obj}}, \quad (3)$$

where ${}^s\mathcal{V}_{b_l}, {}^s\mathcal{V}_{b_r} \in \mathbb{R}^6$ are the spatial twists (6D velocities) of each end-effector, and $\dot{\mathbf{q}}_{\text{obj}} \in \mathbb{R}^k$ is the object’s joint velocity vector. This constraint specifies that the relative motion between the two arms must lie in the range space of J_s , aligning precisely with the object’s allowable internal motions along its kinematic chain.

The Challenge. Any control policy for bimanual manipulation must ensure that commanded motions satisfy this holonomic constraint. Violating the constraint leads to physically infeasible trajectories that cause grasp slippage, internal stress accumulation, and potential task failure.

Solution Preview. This challenge is addressed by the **Screw-decomposed Controller** (Section 3.2.4), which uses orthogonal projection operators based on the object Jacobian to structurally decompose commanded motions into components that lie in the range of J_i (constraint-satisfying internal motion) and its orthogonal complement (bulk motion), guaranteeing kinematic feasibility by construction.

3.1.3 Challenge 3: Decomposing and Modulating Compliance on SE(3)

To enable task-agnostic control, the low-level policy must interpret high-level intentions directly from trajectory structure without task-specific annotations. We observe that bimanual manipulation of articulated objects inherently involves two distinct motion intentions: moving the object as a whole through space, and actuating its internal joint.

Semantic Motion Decomposition. For task semantics, we want to decompose each end-effector’s reference motion into two semantically meaningful components:

- **Bulk motion** (orthogonal to screw axis): Drives the object’s **overall motion** through space—transport, reorientation, and gross positioning. The term is borrowed from fluid dynamics, where “bulk motion” refers to macroscopic flow of the entire fluid body.
- **Internal motion** (parallel to screw axis): Drives the object’s **joint articulation**—opening, closing, rotating, or extending the internal degree of freedom. This represents productive motion that directly actuates the object’s constrained joint.

By explicitly providing both components to the policy, we enable it to infer task semantics from trajectory structure: tasks emphasizing bulk motion (e.g., transporting a closed scissor) vs. tasks emphasizing internal motion (e.g., cutting with stationary scissors) vs. coordinated tasks (e.g., simultaneously moving and actuating). This decomposition enables generalization across diverse manipulation objectives without explicit task labels.

Orthogonality and Metric Definition. Performing this decomposition requires separating parallel and orthogonal components relative to the screw axis, which necessitates defining an inner product on the Lie algebra $\mathfrak{se}(3)$. However, in a purely kinematic sense, there is no natural inner product that defines orthogonality between two twists in terms of rigid body motion. This is fundamentally because purely kinematic formulations lack a natural physical scale to compare rotations (dimensionless) with translations (length), making any choice of inner product physically arbitrary. Standard candidates—Euclidean product on \mathbb{R}^6 (mixes incompatible units, not frame-invariant), reciprocal

product (requires dual space pairing), Killing form (degenerate for SE(3))—all present difficulties for physically meaningful rigid body motion decomposition (see Appendix ?? for detailed analysis).

SE(3) Impedance Control Limitations. Impedance control—regulating the dynamic relationship between motion and force—is a natural paradigm for compliant bimanual manipulation. However, implementing impedance control on the SE(3) manifold presents fundamental mathematical challenges. Classical Cartesian impedance control derives elastic wrenches from potential energy via differentiation, which for SE(3) introduces a nonlinear Jacobian-like operator $J_{\mathcal{E}}$ (detailed in Section 3.1.5 below). This nonlinearity fundamentally limits the design freedom of the stiffness matrix K : achieving desired compliance behavior in one region of the configuration space may produce unintended, potentially unstable behavior elsewhere due to the varying geometric structure.

The Challenge. The core challenge arises when attempting to integrate impedance control with task-semantic decomposition. For effective bimanual manipulation, we require **independent compliance modulation** along bulk versus internal motion directions. However, the nonlinear coupling in $J_{\mathcal{E}}$ prevents straightforward decomposition-aware stiffness design. Additionally, the choice of metric for orthogonal decomposition directly affects which motions are considered "parallel" or "orthogonal" to the constraint, yet no single choice is universally optimal across all tasks and object geometries.

Solution Preview. These challenges are addressed through three coupled mechanisms: (1) the **Learned Impedance Variables** (Section 3.2.3), where the policy outputs separate damping coefficients $d_{i,\parallel}$ and $d_{i,\perp}$ for independent compliance modulation, and critically, a **learnable characteristic length scale** α that adaptively defines the metric tensor $G = \text{diag}(\alpha^2 I_3, I_3)$, allowing the policy to discover task-appropriate notions of orthogonality; (2) the **Screw-decomposed Controller** (Section 3.2.4), which constructs projection operators using this learned metric to decompose control actions into bulk and internal components; and (3) the **Reference Motion Field** (Section 3.2.2), which incorporates the stability term $k_{p_i} \mathcal{E}_i$ directly into the reference twist, sidestepping the explicit nonlinear Jacobian $J_{\mathcal{E}}$ while maintaining SE(3) impedance behavior.

3.1.4 Challenge 4: Force Coupling and Internal Wrenches

In bimanual manipulation, when both arms rigidly grasp the same object, their force interactions become coupled. Any coordination error that violates the kinematic constraint generates **internal wrenches**—forces and torques that stress the object and grasps without contributing to productive manipulation.

Internal Wrenches. Let $\mathcal{F}_l, \mathcal{F}_r \in \mathbb{R}^6$ denote body wrenches measured at the end-effectors: $\mathcal{F}_i = \begin{bmatrix} m_i \\ f_i \end{bmatrix}$ (moment and force). When the two arms' motions deviate from the kinematic constraint (Eq. 3), they "fight" each other, generating internal wrenches that: (1) increase contact stress and risk grasp failure, (2) waste energy through non-productive internal loading, and (3) potentially damage the object or robot hardware.

The Challenge. For compliant manipulation, we must distinguish between force components that contribute to desired object motion versus those that only create internal stress. The control framework needs to identify and actively minimize non-productive internal forces while maintaining necessary productive forces for manipulation. This requires decomposing measured wrenches into components aligned with (productive) and orthogonal to (internal) the object's allowable motion direction.

Solution Preview. This challenge is addressed through the **Wrench-based Reward Design** (Section 3.3.2), which decomposes measured wrenches using the transpose of twist projection operators $\mathcal{F}_{i,\perp} = P_{i,\perp}^T \mathcal{F}_i$ to identify internal wrenches orthogonal to the object's allowable motion, and explicitly penalizes $\|\mathcal{F}_{i,\perp}\|_2^2$ to train the policy to minimize non-productive forces. This wrench decomposition is fully consistent with the twist decomposition framework, exploiting the duality between twist and wrench spaces under the reciprocal product (virtual power).

3.1.5 Mathematical Background: SE(3) Impedance Control Geometry

For completeness, we provide the geometric details underlying Challenge 3. Impedance control—regulating the dynamic relationship between motion and force—is a natural paradigm for compliant

bimanual manipulation. However, implementing impedance control on the SE(3) manifold presents fundamental mathematical challenges that motivate our learning-based approach.

Kinetic Energy Metric and Distance Problem. Unlike Euclidean spaces, SE(3) is a manifold where defining meaningful error metrics is non-trivial. The most physically natural metric on SE(3) is the **kinetic energy-based Riemannian metric** induced by the spatial inertia matrix $M \in \mathbb{R}^{6 \times 6}$. For tracking control, we need to measure the "distance" between the current pose T and desired pose T_{des} , typically represented by the error pose $T_{\text{err}} = T_{\text{des}}^{-1}T$. However, SE(3) lacks a **bi-invariant Riemannian metric**—a metric that remains consistent under both left and right group actions. The exponential map follows screw motions (constant-twist trajectories), while geodesics under the kinetic energy metric follow more complex, inertia-weighted paths. This mismatch means that the natural logarithm map $\log(T_{\text{err}})^\vee \in \mathfrak{se}(3)$, commonly used to define pose errors, does not correspond to minimal-energy paths under the physically meaningful metric.

Nonlinear Stiffness and Design Constraints. In classical Cartesian impedance control, elastic wrenches are derived from potential energy via differentiation. For SE(3), a nonlinear Jacobian-like operator emerges:

$$\mathcal{F}_{\text{elastic}} = -J_{\mathcal{E}}^\top K \mathcal{E}, \quad (4)$$

where $J_{\mathcal{E}}$ is given by:

$$J_{\mathcal{E}} = \begin{pmatrix} \alpha J_l^{-1}(e_R) & 0_{3 \times 3} \\ -[e_p] & I_3 \end{pmatrix} \in \mathbb{R}^{6 \times 6}. \quad (5)$$

Here, $J_l(\theta)$ is the left Jacobian of SO(3):

$$J_l(\theta) = I + \frac{1 - \cos \|\theta\|}{\|\theta\|^2} [\theta] + \frac{\|\theta\| - \sin \|\theta\|}{\|\theta\|^3} [\theta]^2. \quad (6)$$

$J_{\mathcal{E}}$ is configuration-dependent and couples rotational and translational components in complex, state-dependent ways. This nonlinearity fundamentally limits the design freedom of the stiffness matrix K : achieving desired compliance behavior in one region of the configuration space may produce unintended, potentially unstable behavior elsewhere due to the varying geometric structure. These mathematical constraints motivate our learning-based approach described in Challenge 3 above.

3.2 SWIVL Architecture

SWIVL adopts a **four-layer hierarchical architecture** that directly addresses the four challenges identified above. The architecture decouples high-level reasoning from low-level physical interaction:

- **Layer 1 (High-Level Policy):** Provides task-level guidance through sparse waypoint generation
- **Layer 2 (Reference Motion Field Generator):** Addresses Challenge 1 by transforming sparse waypoints into dense, closed-loop reference trajectories
- **Layer 3 (Impedance Variable Modulation Policy):** Addresses Challenges 3 & 4 by learning adaptive compliance modulation based on object geometry and wrench feedback
- **Layer 4 (Screw-decomposed Controller):** Addresses Challenges 2 & 3 by structurally enforcing kinematic constraints through orthogonal motion decomposition

The core innovation lies in the tight integration of Layers 2-4, which together enable physically grounded, force-compliant bimanual manipulation underneath arbitrary high-level planners.

3.2.1 Layer 1: High-Level Policy

The top layer generates goal-directed behavior in action chunk. This layer can be instantiated by:

- **Vision-Language-Action (VLA) models**
- **Behavior cloning policies**
- **Teleoperation interfaces**

Interface: Outputs discrete end-effector pose waypoints $\{T_{sd_l}[\tau], T_{sd_r}[\tau]\}_{\tau=0}^H$ at low frequency.

3.2.2 Layer 2: Reference Motion Field Generator

This layer addresses Challenge 1 by bridging the gap between discrete high-level waypoints and continuous low-level control. High-level policies provide sparse waypoints at low frequency ($\sim 10\text{Hz}$), while the low-level controller requires smooth, dense reference trajectories at high frequency ($\sim 50\text{Hz}$) with corrective feedback capabilities. The Reference Motion Field Generator performs three steps to achieve this transformation:

Step 1: SE(3) Trajectory Smoothing. To address the sparsity gap, we perform smooth interpolation in $SE(3)$ to obtain dense desired trajectories at the Low-Level Policy frequency $\Delta t_{LL} \ll \Delta t_{HL}$:

$$\{T_{sd_l}(t), T_{sd_r}(t)\}_{t=0}^{H_{LL}}, \quad (7)$$

where H_{LL} is the smoothed trajectory horizon. We use SLERP for rotations and cubic splines for translations; see Appendix ?? for details. Both interpolation schemes are differentiable in time, so they induce smooth position and rotation trajectories $p_i^{\text{des}}(t)$ and $R_i^{\text{des}}(t)$ for each end-effector.

Step 2: Body Twist Computation. For a smooth trajectory $T_{sd_i}(t) = \begin{bmatrix} R_{sd_i}(t) & p_{sd_i}(t) \\ 0 & 1 \end{bmatrix}$, we directly compute the desired twist by differentiating the pose and expressing the resulting velocities in the instantaneous desired frame. Let $R_{sd_i}(t)$ denote the rotation from the desired body frame $\{d_i\}$ to the spatial frame $\{s\}$, and $p_{sd_i}(t)$ the position of the desired body frame origin in the spatial frame. The corresponding desired-frame angular and linear velocities are given by

$$\mathcal{V}_i^{\text{des}}(t) = \begin{bmatrix} \omega_i^{\text{des}}(t) \\ v_i^{\text{des}}(t) \end{bmatrix}, \quad [\omega_i^{\text{des}}(t)]_{\times} = R_{sd_i}(t)^{\top} \dot{R}_{sd_i}(t), \quad v_i^{\text{des}}(t) = R_{sd_i}(t)^{\top} \dot{p}_{sd_i}(t). \quad (8)$$

Here $[\omega]_{\times}$ is the skew-symmetric matrix representation of the angular velocity vector ω , and both $\omega_i^{\text{des}}(t)$ and $v_i^{\text{des}}(t)$ are expressed in the desired frame $\{d_i\}$ by construction.

Step 3: Stable Imitation Vector Field Design. As execution progresses within an action chunk, the actual end-effector poses may deviate significantly from the desired trajectory due to tracking errors, disturbances, and model mismatch. To address the open-loop nature of waypoints and provide robustness when the system deviates from the desired trajectory, we construct a vector field that balances **imitation** of demonstrated motions and **stability** for error correction. We employ a stable imitation vector field that combines two components: an imitation term that mimics the demonstrated velocity profile at the temporally synchronized point, and a stability term that provides corrective feedback via a term proportional to the $SE(3)$ pose error for convergence to the desired trajectory:

$$\mathcal{V}_i^{\text{ref}}(t, T_{sb_i}) = \text{Ad}_{T_{b_i d_i}} \mathcal{V}_i^{\text{des}}(t) + k_{p_i} \mathcal{E}, \quad (9)$$

where Ad_T denotes the adjoint transformation that maps twists between frames. Since the desired twist $\mathcal{V}_i^{\text{des}}(t)$ is computed in the desired frame $\{d_i\}$ (Eq. ??), we must transform it to the current body frame $\{b_i\}$ where the controller operates. The transformation $T_{b_i d_i} = T_{b_i s} T_{s d_i} = (T_{sb_i})^{-1} T_{sd_i}$ represents the relative transformation from the desired frame to the current body frame, and $\text{Ad}_{T_{b_i d_i}}$ performs the corresponding twist transformation. The pose error term is given by:

$$\begin{aligned} \mathcal{E} &= \begin{pmatrix} \alpha e_R \\ e_p \end{pmatrix} \in \mathbb{R}^6 \\ e_{R_i} &= \log(R_{sb_i}^{\top} R_{sd_i})^{\vee} \in \mathbb{R}^3 \quad (\text{rotation error in body frame}) \\ e_{p_i} &= R_{sb_i}^{\top} (p_{sd_i} - p_{sb_i}) \in \mathbb{R}^3 \quad (\text{translation error in body frame}) \\ k_{p_i} &\in \mathbb{R} \quad (\text{scalar proportional gain}) \end{aligned} \quad (10)$$

Here, α represents a characteristic length that weights the rotational cost relative to translation, where $G = \text{diag}(\alpha^2 I_3, I_3)$ is the metric tensor defining an inner product $\langle \cdot, \cdot \rangle_G$ on $\mathfrak{se}(3)$:

$$\langle \mathcal{V}_1, \mathcal{V}_2 \rangle_G = \frac{\alpha^2}{2} \text{tr}([\omega_1]^{\top} [\omega_2]) + v_1^{\top} v_2 = \mathcal{V}_1^{\top} G \mathcal{V}_2, \quad (11)$$

where $\mathcal{V}_i = \begin{bmatrix} \omega_i \\ v_i \end{bmatrix}$. This time-based approach offers computational efficiency ($O(1)$ lookup) while maintaining strong tracking performance for smooth bimanual trajectories. The term $k_p \mathcal{E}$, as will be described later, plays a role in creating an elastic force for impedance control without using the nonlinear Jacobian matrix to create a reference twist.

3.2.3 Layer 3: Impedance Variable Modulation Policy

This layer addresses Challenges 3 and 4 by learning adaptive impedance modulation. The Reinforcement Learning Policy $\pi_\theta : \mathcal{O} \rightarrow \mathcal{A}$ modulates impedance variables to enable physically feasible motions while accounting for object constraints and inter-arm force interactions. By explicitly conditioning on object geometry (screw axes) and wrench feedback, the policy learns to independently modulate compliance for bulk versus internal motions and minimize harmful internal forces.

Observation Space \mathcal{O} . The policy receives:

1. **Reference twists:** $\{\mathcal{V}_l^{\text{ref}}, \mathcal{V}_r^{\text{ref}}\} \in \mathfrak{se}(3) \times \mathfrak{se}(3)$ These are the reference motions computed by the Reference Motion Field Generator (Layer 2) at the current time t and current end-effector poses T_{sb_l}, T_{sb_r} .
2. **Object constraints:** $\{\mathcal{B}_l, \mathcal{B}_r\} \in \mathfrak{se}(3) \times \mathfrak{se}(3)$ These encode the kinematic constraint of the manipulated object. $\mathcal{B}_l, \mathcal{B}_r$ are the body-frame screw axes defining the object’s allowable internal motion directions at each end-effector. For articulated objects, these correspond to the columns of the object Jacobian J_i , related to the spatial screw axis via $\mathcal{B}_i = \text{Ad}_{T_{b_i s}} \mathcal{S}$ where $\text{Ad}_{T_{b_i s}}$ transforms spatial twists to body frame.
3. **Wrench feedback:** $\{\mathcal{F}_l, \mathcal{F}_r\} \in \mathfrak{se}(3)^* \times \mathfrak{se}(3)^*$ These are 6-dimensional wrench measurements (3D moment + 3D force) obtained from 6-axis force-torque sensors mounted at each end-effector’s wrist. Raw sensor data may be filtered (e.g., low-pass filtering or exponential smoothing) when necessary to reduce measurement noise while preserving force feedback responsiveness for compliant control.
4. **Proprioception:** $\{T_{sb_l}, T_{sb_r}, \mathcal{V}_l, \mathcal{V}_r\}$ Task-space states including end-effector poses $T_{sb_l}, T_{sb_r} \in \text{SE}(3)$ and body twists $\mathcal{V}_l, \mathcal{V}_r \in \mathfrak{se}(3)$.

Action Space \mathcal{A} . We propose an impedance variable action space that structurally enforces object’s kinematic constraints by motion decomposition. These variables parameterize the low-level controller (detailed in Section ??), allowing the policy to modulate compliance behavior dynamically:

$$a_t = (d_{l,\parallel}, d_{r,\parallel}, d_{l,\perp}, d_{r,\perp}, k_{p_l}, k_{p_r}, \alpha) \in \mathbb{R}^7, \quad (12)$$

where $d_{i,\parallel}, d_{i,\perp}, k_{p_i}, \alpha \in \mathbb{R}^+$ are positive scalar gains, and:

- $d_{i,\parallel}$: Damping coefficient for internal motion (parallel to screw axis), regulating compliance along the object’s degree of freedom.
- $d_{i,\perp}$: Damping coefficient for bulk motion (orthogonal to screw axis), regulating compliance for the object’s overall transport.
- k_{p_i} : Stiffness gain for the stability term in the vector field, determining the strength of correction towards the desired trajectory.
- α : **Learnable characteristic length scale** that adaptively weights rotational error relative to translational error in the $\text{SE}(3)$ metric tensor $G = \text{diag}(\alpha^2 I_3, I_3)$. By learning α , the policy discovers task-appropriate metric structures for orthogonal decomposition and compliance modulation.

These policy outputs parameterize the low-level controller (detailed in Layer 4 below), enabling adaptive, context-dependent impedance modulation.

3.2.4 Layer 4: Screw-decomposed Twist-driven Impedance Controller

This layer addresses Challenges 2 and 3 by executing low-level control that structurally enforces kinematic constraints while enabling independent compliance modulation. It tracks the reference motion $\mathcal{V}_i^{\text{ref}}$ from Layer 2 using the impedance parameters from Layer 3. The key innovation of

this controller is its ability to provide **two complementary views**: (1) it behaves as a geometrically consistent SE(3) impedance controller, and (2) it explicitly decomposes control actions into bulk and internal motion spaces, enabling task-aware compliance modulation.

Orthogonal Decomposition via Object Jacobian. To enable motion decomposition, we first establish projection operators that separate motions into components parallel and orthogonal to the object's kinematic constraints. Let $J_i(\mathbf{q}_{\text{obj}}) \in \mathbb{R}^{6 \times k}$ denote the body Jacobian of the object for end-effector $i \in \{l, r\}$, which encodes how the object's joint velocities $\dot{\mathbf{q}}_{\text{obj}}$ manifest as end-effector body twists. The body Jacobian relates to the spatial Jacobian (Eq. 2) via the adjoint transformation: $J_i = \text{Ad}_{T_{b_i s}} J_s$, where $T_{b_i s} = (T_{s b_i})^{-1}$ transforms spatial frame quantities to the body frame.

Using the inner product $\langle \mathcal{V}_1, \mathcal{V}_2 \rangle_G = \mathcal{V}_1^T G \mathcal{V}_2$ with metric tensor $G = \text{diag}(\alpha^2 I_3, I_3)$ on $\mathfrak{se}(3)$, we construct orthogonal projection operators:

$$P_{i,\parallel} = J_i(J_i^T G J_i)^{-1} J_i^T G, \quad P_{i,\perp} = I - P_{i,\parallel}, \quad (13)$$

where $P_{i,\parallel}$ projects onto the internal motion subspace (range of J_i) and $P_{i,\perp}$ projects onto the bulk motion subspace (orthogonal complement). These projectors satisfy $P_{i,\parallel}^T G = G P_{i,\parallel}$ and $P_{i,\perp}^T G = G P_{i,\perp}$, ensuring geometric consistency under the chosen metric. Importantly, since α is learned by the policy (Layer 3), the metric tensor G and hence the projection operators adapt dynamically to task requirements, enabling context-dependent orthogonal decomposition.

Controller Formulation. With the learned impedance variables $a_t = (d_{l,\parallel}, d_{r,\parallel}, d_{l,\perp}, d_{r,\perp}, k_{p_l}, k_{p_r}, \alpha)$ from Layer 3, we construct a damping matrix that respects the motion decomposition:

$$K_{d_i} = G(P_{i,\parallel} d_{i,\parallel} + P_{i,\perp} d_{i,\perp}), \quad (14)$$

This structure allows independent damping modulation for internal motion (via $d_{i,\parallel}$) and bulk motion (via $d_{i,\perp}$), enabling the policy to adaptively regulate compliance based on task requirements and force feedback.

The commanded wrench is then computed as:

$$\begin{aligned} \mathcal{F}_{\text{cmd},i} &= K_{d_i}(\mathcal{V}_i^{\text{ref}} - \mathcal{V}_i) + \mu_{b,i} + \gamma_{b,i} \\ &= K_{d_i}(\text{Ad}_{T_{b_i d_i}} \mathcal{V}_i^{\text{des}} - \mathcal{V}_i + k_{p_i} \mathcal{E}_i) + \mu_{b,i} + \gamma_{b,i}, \end{aligned} \quad (15)$$

where $\mathcal{V}_i^{\text{ref}} = \text{Ad}_{T_{b_i d_i}} \mathcal{V}_i^{\text{des}} + k_{p_i} \mathcal{E}_i$ is the reference twist from Layer 2, $\mu_{b,i}$ accounts for nonlinear dynamics (Coriolis and centrifugal terms), and $\gamma_{b,i}$ provides gravity compensation (omitted in planar SE(2) settings where gravity is orthogonal to the motion plane).

The commanded wrench is then mapped to joint-space motor torques via the manipulator Jacobian:

$$\tau_{\text{cmd},i} = J_i(\theta_i)^T \mathcal{F}_{\text{cmd},i}, \quad (16)$$

where $J_i(\theta_i) \in \mathbb{R}^{6 \times n}$ is the geometric Jacobian of the i -th manipulator mapping joint velocities to end-effector twist, and $\tau_{\text{cmd},i} \in \mathbb{R}^n$ is the commanded joint torque vector that serves as the final robot control command.

We now provide two complementary interpretations that reveal the dual nature of this controller.

Interpretation 1: SE(3) Impedance Control Structure.

The first interpretation reveals that our controller naturally implements SE(3) impedance control. Classical impedance control on SE(3) designs a virtual dynamical system with desired impedance characteristics:

$$M\dot{\xi} + D\xi + J_{\mathcal{E}}^T K \mathcal{E} = \mathcal{F}_{\text{ext}}, \quad (17)$$

where $\xi = {}^b\mathcal{V}_d - {}^b\mathcal{V}_b$ is the velocity error, M is the desired inertia, D is damping, and $J_{\mathcal{E}}^\top K\mathcal{E}$ is the nonlinear stiffness term arising from SE(3) geometry (Section ??). The corresponding impedance controller takes the form:

$$\mathcal{F}_{\text{cmd}} = \Lambda_b M^{-1}(D\xi + J_{\mathcal{E}}^\top K\mathcal{E}) + \Lambda_b {}^b\dot{\mathcal{V}}_d + \mu_b + \gamma_b + (I - \Lambda_b M^{-1})\mathcal{F}_{\text{ext}}, \quad (18)$$

where Λ_b is the operational space inertia matrix. Under the common simplifications $M = \Lambda_b$ (match desired and actual inertia) and ${}^b\dot{\mathcal{V}}_d = 0$ (constant reference velocity), this reduces to:

$$\mathcal{F}_{\text{cmd}} = D\xi + J_{\mathcal{E}}^\top K\mathcal{E} + \mu_b + \gamma_b. \quad (19)$$

Our controller in Eq. (??) follows this exact structure. Defining the velocity error as $\xi = \text{Ad}_{T_{b_i d_i}} \mathcal{V}_i^{\text{des}} - \mathcal{V}_i$, we can rewrite:

$$\begin{aligned} \mathcal{F}_{\text{cmd},i} &= K_{d_i}(\text{Ad}_{T_{b_i d_i}} \mathcal{V}_i^{\text{des}} - \mathcal{V}_i + k_{p_i} \mathcal{E}_i) + \mu_{b,i} + \gamma_{b,i} \\ &= K_{d_i} \xi + K_{d_i} k_{p_i} \mathcal{E}_i + \mu_{b,i} + \gamma_{b,i} \\ &\approx D\xi + J_{\mathcal{E}}^\top K\mathcal{E} + \mu_b + \gamma_b, \end{aligned} \quad (20)$$

where the correspondence is: $D \leftrightarrow K_{d_i}$ (learned damping) and the term $K_{d_i} k_{p_i} \mathcal{E}_i$ plays the role of $J_{\mathcal{E}}^\top K\mathcal{E}$ (stiffness). Critically, our approach **sidesteps the explicit nonlinear Jacobian** $J_{\mathcal{E}}$ by incorporating the $k_{p_i} \mathcal{E}_i$ term directly into the reference twist (Layer 2), avoiding the geometric complications discussed in Section ?? while maintaining impedance behavior.

Interpretation 2: Explicit Bulk-Internal Motion Decomposition.

The second interpretation reveals how the controller naturally decomposes control actions into semantically meaningful components aligned with task requirements. By decomposing both reference and actual twists into components parallel (internal) and orthogonal (bulk) to the object's kinematic constraints:

$$\mathcal{V}_{i,\parallel}^{\text{ref}} = P_{i,\parallel} \mathcal{V}_i^{\text{ref}}, \quad \mathcal{V}_{i,\parallel} = P_{i,\parallel} \mathcal{V}_i \quad (\text{internal motion}), \quad (21)$$

$$\mathcal{V}_{i,\perp}^{\text{ref}} = P_{i,\perp} \mathcal{V}_i^{\text{ref}}, \quad \mathcal{V}_{i,\perp} = P_{i,\perp} \mathcal{V}_i \quad (\text{bulk motion}), \quad (22)$$

we can expand the control law to reveal independent regulation of each motion component:

$$\begin{aligned} \mathcal{F}_{\text{cmd},i} &= K_{d_i}(\mathcal{V}_i^{\text{ref}} - \mathcal{V}_i) + \mu_{b,i} + \gamma_{b,i} \\ &= G(P_{i,\parallel} d_{i,\parallel} + P_{i,\perp} d_{i,\perp})(\mathcal{V}_i^{\text{ref}} - \mathcal{V}_i) + \mu_{b,i} + \gamma_{b,i} \\ &= \underbrace{d_{i,\parallel} G(\mathcal{V}_{i,\parallel}^{\text{ref}} - \mathcal{V}_{i,\parallel})}_{\text{internal motion control}} + \underbrace{d_{i,\perp} G(\mathcal{V}_{i,\perp}^{\text{ref}} - \mathcal{V}_{i,\perp})}_{\text{bulk motion control}} + \mu_{b,i} + \gamma_{b,i}. \end{aligned} \quad (23)$$

This decomposition provides three critical properties:

1. **Independent compliance modulation:** The policy can independently adjust $d_{i,\parallel}$ and $d_{i,\perp}$ to achieve task-specific compliance—high stiffness for bulk motion during transport, high compliance for internal motion during articulation, or vice versa.
2. **Decoupled Power Generation:** The feedback wrenches for internal and bulk motions are **reciprocally orthogonal** to the opposing motion subspaces, ensuring zero interference in terms of virtual power:

$$\mathcal{F}_{\text{cmd,fb},i,\parallel} = d_{i,\parallel} G(\mathcal{V}_{i,\parallel}^{\text{ref}} - \mathcal{V}_{i,\parallel}) \quad (24)$$

$$\mathcal{F}_{\text{cmd,fb},i,\perp} = d_{i,\perp} G(\mathcal{V}_{i,\perp}^{\text{ref}} - \mathcal{V}_{i,\perp}) \quad (25)$$

$$(\mathcal{F}_{\text{cmd,fb},i,\parallel})^\top (\mathcal{V}_{i,\perp}^{\text{ref}} - \mathcal{V}_{i,\perp}) = 0 \quad (26)$$

$$(\mathcal{F}_{\text{cmd,fb},i,\perp})^\top (\mathcal{V}_{i,\parallel}^{\text{ref}} - \mathcal{V}_{i,\parallel}) = 0. \quad (27)$$

This orthogonality follows directly from the projection properties: $P_{i,\parallel}^T G P_{i,\perp} = 0$, ensuring that control actions for each motion type do not interfere with each other.

3. **Constraint satisfaction:** The internal motion component $\mathcal{V}_{i,\parallel}$ automatically lies in the range of J_i , ensuring that commanded motions respect the object’s kinematic constraints and minimize harmful internal forces.

Together, these two interpretations demonstrate that our controller simultaneously achieves geometrically consistent SE(3) impedance behavior while enabling explicit, learning-based modulation of task-semantic motion components—a capability that would be intractable to design analytically given the geometric constraints discussed in Section ??.

3.3 Learning Framework

3.3.1 Reinforcement Learning Formulation

We formulate the Low-Level Policy learning as a Partially Observable Markov Decision Process (POMDP) $\mathcal{M} = (\mathcal{S}, \mathcal{O}, \mathcal{A}, P, r, \gamma)$, where:

- \mathcal{S} : State space (full environment state)
- \mathcal{O} : Observation space (partial observations available to the policy)
- \mathcal{A} : Action space
- $P(s_{t+1}|s_t, a_t)$: Transition dynamics
- $r : \mathcal{S} \times \mathcal{A} \rightarrow \mathbb{R}$: Reward function (Section ??)
- γ : Discount factor

The objective is to maximize expected return:

$$J(\theta) = \mathbb{E}_{\tau \sim \pi_\theta} \left[\sum_{t=0}^T \gamma^t r(s_t, a_t) \right]. \quad (28)$$

3.3.2 Reward Function Design

The reward function balances three objectives for stable, force-compliant manipulation:

$$r_t = r_{\text{track}} + r_{\text{safety}} + r_{\text{reg}}. \quad (29)$$

Motion Tracking (r_{track}). Ensures accurate tracking of reference trajectories generated by the motion field:

$$r_{\text{track}} = -w_{\text{track}} \sum_{i \in \{l, r\}} \|\mathcal{V}_i - \mathcal{V}_i^{\text{ref}}\|_G^2 = -w_{\text{track}} \sum_{i \in \{l, r\}} (\mathcal{V}_i - \mathcal{V}_i^{\text{ref}})^T G (\mathcal{V}_i - \mathcal{V}_i^{\text{ref}}), \quad (30)$$

where $G = \text{diag}(\alpha^2 I_3, I_3)$ is the learned metric tensor. This reward encourages each end-effector to follow its reference twist, where $\mathcal{V}_l, \mathcal{V}_r$ are the actual body twists and $\mathcal{V}_l^{\text{ref}}, \mathcal{V}_r^{\text{ref}}$ are the reference twists from the motion field. Using the G-metric ensures that tracking error is measured consistently with the impedance control framework, with adaptive weighting between rotational and translational components via the learned parameter α .

Safety (r_{safety}). Ensures safe operation and minimizes non-productive internal forces:

$$r_{\text{safety}} = -w_{\text{int}} \sum_{i \in \{l, r\}} \|\mathcal{F}_{i,\perp}\|_2^2, \quad (31)$$

where $\mathcal{F}_i = \begin{bmatrix} m_i \\ f_i \end{bmatrix}$ is the measured wrench at each end-effector i .

This reward addresses the internal wrench problem identified in Section ?? by minimizing internal wrenches—wrench components orthogonal to the object’s allowable motion direction. To decompose measured wrenches consistently with the twist decomposition framework in Layer 4, we seek wrench components $\mathcal{F}_{i,\parallel}$ and $\mathcal{F}_{i,\perp}$ such that they are orthogonal to complementary twist subspaces under the reciprocal product (virtual power). Specifically, we require:

$$\langle \mathcal{F}_{i,\parallel}, \mathcal{V} \rangle = \mathcal{F}_{i,\parallel}^T \mathcal{V} = 0 \quad \forall \mathcal{V} \in \text{range}(P_{i,\perp}), \quad (32)$$

$$\langle \mathcal{F}_{i,\perp}, \mathcal{V} \rangle = \mathcal{F}_{i,\perp}^T \mathcal{V} = 0 \quad \forall \mathcal{V} \in \text{range}(P_{i,\parallel}), \quad (33)$$

where $\langle \mathcal{F}, \mathcal{V} \rangle = \mathcal{F}^T \mathcal{V}$ is the reciprocal product representing virtual power. This orthogonality condition is naturally satisfied by projecting the measured wrench using the transpose of the twist projection operators, exploiting the dual relationship between twist and wrench spaces:

$$\mathcal{F}_{i,\parallel} = P_{i,\parallel}^T \mathcal{F}_i, \quad \mathcal{F}_{i,\perp} = P_{i,\perp}^T \mathcal{F}_i = (I - P_{i,\parallel})^T \mathcal{F}_i. \quad (34)$$

To verify orthogonality, for any $\mathcal{V} \in \text{range}(P_{i,\perp})$, we have $\mathcal{V} = P_{i,\perp} \mathcal{V}'$ for some \mathcal{V}' , and:

$$\mathcal{F}_{i,\parallel}^T \mathcal{V} = (P_{i,\parallel}^T \mathcal{F}_i)^T (P_{i,\perp} \mathcal{V}') = \mathcal{F}_i^T P_{i,\parallel} P_{i,\perp} \mathcal{V}' = 0, \quad (35)$$

where the last equality follows immediately from the orthogonality of the twist projectors ($P_{i,\parallel} P_{i,\perp} = 0$). Similarly, it can be shown that $\mathcal{F}_{i,\perp}^T \mathcal{V} = 0$ for all $\mathcal{V} \in \text{range}(P_{i,\parallel})$.

The parallel component $\mathcal{F}_{i,\parallel}$ represents productive wrench that performs work along the object’s internal degree of freedom, contributing to desired joint motion. The orthogonal component $\mathcal{F}_{i,\perp}$ represents **internal wrench** that:

- Does not contribute to desired object motion along the screw axis (zero virtual power along $\text{range}(P_{i,\parallel})$)
- Arises from coordination errors between the two arms
- Represents constraint forces (bearing loads, friction, etc.) unrelated to joint actuation
- Increases unnecessary contact stress and grasp instability
- Wastes energy and risks hardware damage

By penalizing $\|\mathcal{F}_{i,\perp}\|_2^2$, the policy learns to minimize non-productive forces while maintaining necessary productive forces for manipulation. This wrench decomposition is fully consistent with the twist decomposition framework, utilizing the duality of the learned kinematic structure.

Regularization (r_{reg}). Encourages smooth motion:

$$r_{\text{reg}} = -w_{\text{reg}} \sum_{i \in \{l, r\}} \|\dot{\mathcal{V}}_i\|^2. \quad (36)$$

This reduces energy consumption (torque magnitude), joint jerkiness (joint acceleration), and Cartesian jerkiness (twist acceleration), promoting natural and efficient movements.

Termination Conditions. To ensure grasp stability, episodes terminate early (task failure) if grasp drift exceeds safety thresholds:

$$\text{Terminate if: } \exists i \in \{l, r\} \text{ such that } \left\| \left[\log \left((T_{\text{grip},i}^{\text{init}})^{-1} T_{\text{grip},i} \right) \right]^\vee \right\|_2 > d_{\text{max}}, \quad (37)$$

where $T_{\text{grip},i}^{\text{init}}$ is the initial grasp pose, $T_{\text{grip},i}$ is the current grasp pose, and d_{max} is the maximum allowable drift threshold. This geodesic distance on SE(3) captures both translational and rotational drift from the initial grasp configuration. When this threshold is exceeded, the episode terminates immediately with a failure signal, encouraging the policy to maintain stable grasps throughout manipulation without explicit reward shaping.

3.3.3 Policy Network Architecture

The Low-Level Policy $\pi_\theta : \mathcal{S} \rightarrow \mathcal{A}$ is implemented as a neural network with **object-conditioned multi-stream architecture**. The network employs Feature-wise Linear Modulation (FiLM) to inject object geometric structure into all feature processing stages, enabling constraint-aware representation learning.

For detailed architecture specifications, see Appendix ??.

3.3.4 Training Procedure

We employ **Proximal Policy Optimization (PPO)** with standard hyperparameters for policy gradient updates.

PPO Objective:

$$L^{CLIP}(\theta) = \mathbb{E}_t \left[\min \left(r_t(\theta) \hat{A}_t, \text{clip}(r_t(\theta), 1 - \epsilon, 1 + \epsilon) \hat{A}_t \right) \right], \quad (38)$$

where $r_t(\theta) = \pi_\theta(a_t|o_t)/\pi_{\theta_{\text{old}}}(a_t|o_t)$ and advantages are computed via Generalized Advantage Estimation (GAE):

$$\hat{A}_t = \sum_{l=0}^{\infty} (\gamma\lambda)^l \delta_{t+l}, \quad \delta_t = r_t + \gamma V_\phi(o_{t+1}) - V_\phi(o_t). \quad (39)$$

where $V_\phi : \mathcal{O} \rightarrow \mathbb{R}$ is the value function approximated by a separate critic network with architecture identical to the policy encoder (shared encoders, separate value head).

Algorithm Summary: The complete training procedure is summarized in Algorithm ??.

4 Experiments

We evaluate SWIVL on bimanual manipulation of a **single-joint articulated object** in an SE(2) planar benchmark environment. While the Method section develops a general SE(3) formulation for k -DoF articulated objects, our experiments specialize this framework to the planar, 1-DoF setting where the object Jacobians collapse to fixed body-frame screw axes for each gripper. Our experiments address three key questions: (Q1) Does SWIVL improve task success and reduce internal forces compared to imitation learning baselines? (Q2) How do design choices in the reference motion decomposition and impedance modulation affect performance? (Q3) Can SWIVL generalize across diverse high-level planners and novel object instances within this SE(2) benchmark?

Why SE(2) and 1-DoF? Our SE(2) planar environment with single-joint articulated objects is a deliberate design choice that enables controlled, rigorous evaluation of SWIVL’s core methodology while maintaining full mathematical consistency with the general framework. This simplification strategy follows established scientific practice of validating fundamental principles in tractable settings before scaling to full complexity.

Rationale for SE(2) planar restriction: By limiting to 2D workspace, we achieve: (1) *Phenomenon isolation*—the fundamental challenges SWIVL addresses (force coupling, kinematic constraint satisfaction, compliant coordination) manifest identically in planar and spatial settings. Internal forces and constraint violations arise from the same geometric and dynamic principles regardless of ambient dimension, allowing focused study without confounding factors from 6-DoF control complexity or visual perception challenges. (2) *Experimental tractability*—reduced dimensionality enables large-scale systematic evaluation (100 trials \times 9 objects \times multiple ablations) with statistical significance that would be prohibitive in full SE(3). (3) *Analytical clarity*—planar settings eliminate gravitational effects orthogonal to the motion plane ($\gamma_{b,i} = 0$), simplifying force analysis and enabling direct interpretation of contact force patterns. (4) *Established precedent*—influential robotics work validates methods in 2D before 3D deployment, demonstrating scientific value of dimensional reduction for concept validation.

Algorithm 1 SWIVL Training

Require: Pre-trained High-Level Policy π_{HL} , object set \mathcal{O}_{obj}

- 1: Initialize: Low-Level Policy parameters θ , value function parameters ϕ
- 2: **for** episode = 1 to $N_{episodes}$ **do**
- 3: Sample object with task $o \sim \mathcal{O}_{obj}$
- 4: Initialize robot, environment, and action chunk buffer
- 5: **for** $t = 1$ to H **do**
- 6: **High-Level Policy**
- 7: **if** $t \bmod f_{HL}^{-1} == 0$ **then**
- 8: Generate action chunk: $\{T_{sd_i}[\tau]\}_{\tau=0}^{H_{chunk}} \leftarrow \pi_{HL}$
- 9: **end if**
- 10: **Reference Motion Field Generator**
- 11: Interpolate action chunk \rightarrow dense trajectory $T_{sd_i}(t)$
- 12: Compute desired body twists $\mathcal{V}_i^{des}(t)$ via Eq. ??
- 13: Apply stable vector field \rightarrow reference twists \mathcal{V}_i^{ref} via Eq. ??
- 14: Decompose $\mathcal{V}_i^{ref} \rightarrow$ internal motion $\mathcal{V}_{i,\parallel}^{ref}$ and bulk motion $\mathcal{V}_{i,\perp}^{ref}$
- 15: **Low-Level Policy**
- 16: Observe $o_t = (\mathcal{V}_i^{ref}, \mathcal{B}_i, \mathcal{F}_i, T_{sb_i}, \mathcal{V}_i)$
- 17: Sample action $a_t = (d_{i,\parallel}, d_{i,\perp}, k_{p_i}, \alpha) \sim \pi_\theta(\cdot|o_t)$
- 18: **Controller**
- 19: Compute commanded wrench $\mathcal{F}_{cmd,i}$ via Eq. ??
- 20: Execute joint torque $\tau_{cmd,i} = J_i(\theta_i)^T \mathcal{F}_{cmd,i}$ via Eq. ??
- 21: **Collect experience**
- 22: Observe o_{t+1} , compute reward r_t via Eq. ??
- 23: Store transition (o_t, a_t, r_t, o_{t+1})
- 24: **end for**
- 25: **PPO Update**
- 26: Compute advantages via GAE (Eq. ??)
- 27: **for** epoch = 1 to 10 **do**
- 28: Sample mini-batches from buffer
- 29: Update θ via Eq. ??
- 30: Update ϕ via MSE loss on value targets
- 31: **end for**
- 32: **end for**

Rationale for 1-DoF object restriction: Single-joint articulated objects provide the minimal yet complete structure to evaluate SWIVL’s constraint-aware design: (1) *Mathematical completeness*—a 1-DoF joint fully instantiates the holonomic constraint ${}^s\mathcal{V}_l - {}^s\mathcal{V}_r = \mathcal{S} \dot{q}_{obj}$ (Eq. 3 specialized to $k = 1$), exercising the screw-decomposed controller’s projection operators, bulk–internal motion decomposition, and wrench-based force regulation. All key architectural components—projection $P_{i,\parallel}, P_{i,\perp}$, metric tensor $G(\alpha)$, and impedance modulation d_{\parallel}, d_{\perp} —are actively engaged even with $k = 1$. (2) *Simplified parameter space*—constant body-frame screw axes $\mathcal{B}_l, \mathcal{B}_r \in \mathbb{R}^3$ (independent of q_{obj} for single-joint mechanisms with fixed grasps) eliminate configuration-dependent Jacobian variations present in multi-DoF chains, isolating the core challenge of coordinated force-compliant manipulation from kinematic complexity. This allows unambiguous attribution of performance to SWIVL’s impedance learning rather than object-specific kinematic modeling. (3) *Diverse coverage*—our 9-object benchmark spans three joint types (revolute, prismatic, fixed) with varying geometry, mass distribution, and inertia, providing sufficient diversity to assess generalization across constraint structures while maintaining experimental control. (4) *Direct scalability path*—the framework naturally extends to k -DoF: screw axes $\mathcal{B}_i \in \mathbb{R}^{3 \times k}$ become time-varying matrices, projections use $(\mathcal{B}_i^T G \mathcal{B}_i)^{-1} \in \mathbb{R}^{k \times k}$, and the policy observes configuration-dependent Jacobians. The SE(2), 1-DoF validation ensures correctness of the fundamental projection-based decomposition before addressing multi-DoF kinematic coupling.

Importantly, our simplifications preserve the essential geometric structure: the SE(2) Lie group, screw theory, adjoint transformations, holonomic constraints, reciprocal product duality between twists and wrenches, and learned metric tensors all reduce consistently from the general SE(3),

k -DoF formulation in Section ?? . This ensures experimental findings directly inform the design principles applicable to full SE(3), multi-DoF deployment. Complete mathematical reduction is detailed in Appendix ??.

4.1 Experimental Setup

Environment. We use an SE(2) planar workspace with dual 3-DoF planar end-effectors operating under **direct body wrench control** $\mathcal{F}_i = [m_{z,i}, f_{x,i}, f_{y,i}]^\top \in \mathbb{R}^3$. The effective manipulation space is bounded with walls at the workspace boundaries. Each end-effector provides 3-axis force-torque sensing synchronized with the low-level control frequency, enabling real-time wrench feedback for impedance modulation. The hierarchical control architecture combines vision-based high-level planning (10 Hz) with SWIVL’s low-level SE(2) control (50 Hz), with grippers maintained closed throughout episodes. Complete workspace specifications and physics configuration are in Appendix ??.

extbfTasks. We evaluate on 9 planar articulated objects spanning three joint categories: fixed (rigid transport), revolute (angular articulation), and prismatic (linear articulation), with 3 object variants per category. Each object is modeled as a **single-DoF articulated body** whose kinematic constraint is captured by a fixed screw axis in each end-effector frame. Concretely, for object joint configuration $q_{obj} \in \mathbb{R}$, the general SE(3) formulation in Section ?? reduces to the SE(2) holonomic constraint

$${}^s\mathcal{V}_l - {}^s\mathcal{V}_r = \mathcal{S} \dot{q}_{obj}, \quad {}^s\mathcal{V}_i \in \mathbb{R}^3, \mathcal{S} \in \mathbb{R}^3, \dot{q}_{obj} \in \mathbb{R}, \quad (40)$$

where $\mathcal{S} = [s_\omega, s_{v,x}, s_{v,y}]^\top$ is the spatial-frame screw axis of the joint. In our planar setting, the corresponding body-frame screw axes $\mathcal{B}_l, \mathcal{B}_r \in \mathbb{R}^3$ at each grasp are **textbf{constant}** in time and independent of q_{obj} , reflecting fixed grasp points on a 1-DoF mechanism. For each object, we specify a fixed goal configuration $({}^sT_o^{goal}, q_{obj}^{goal})$ visually marked in the workspace. Episodes initialize with objects at random configurations $({}^sT_o^{init}, q_{obj}^{init})$ within safe regions, spawning uniformly to ensure diverse initial conditions. The task objective is to manipulate the object from its initial configuration to the goal through coordinated bimanual control. Each object is tested over 100 trials with randomized initialization. Success requires: object position error < 10 pixels, orientation error $< 5^\circ$, joint error $< 5^\circ$ (revolute) or < 5 pixels (prismatic), and maintained grasp throughout. Detailed task specifications and evaluation protocol are in Appendix ??.

Implementation. The Low-Level Policy uses a multi-stream architecture with FiLM conditioning to modulate feature processing based on object structure, instantiated with the SE(2) specialization described in Appendix ??.

Observation Space (\mathbb{R}^{30}): Following Method Section 3.2.3, the SE(2) policy observes: (1) *Reference twists*—individual end-effector SE(2) reference twists $\mathcal{V}_l^{\text{ref}}, \mathcal{V}_r^{\text{ref}} \in \mathbb{R}^3$ (comprising $[\omega_z, v_x, v_y]^\top$) computed by the Reference Motion Field Generator (Layer 2) from the stable imitation vector field (Eq. (??)); (2) *Object constraints*—time-invariant body-frame screw axes $\mathcal{B}_l, \mathcal{B}_r \in \mathbb{R}^3$ that encode the 1-DoF joint structure (constant due to fixed grasp points on single-joint mechanisms); (3) *Wrench feedback*—3-axis force-torque measurements $\mathcal{F}_l, \mathcal{F}_r \in \mathbb{R}^3$ (comprising $[m_z, f_x, f_y]^\top$) from each end-effector; (4) *Proprioception*—end-effector SE(2) poses $T_{sb_l}, T_{sb_r} \in \text{SE}(2)$ (represented as $[x, y, \theta]^\top \in \mathbb{R}^3$ each) and body twists $\mathcal{V}_l, \mathcal{V}_r \in \mathbb{R}^3$. The policy network internally computes bulk-internal decomposition of reference twists and measured wrenches using projection operators.

Action Space (\mathbb{R}^7): Following Method Section 3.2.3, the SE(2) policy outputs impedance modulation variables: $a_t = (d_{l,\parallel}, d_{r,\parallel}, d_{l,\perp}, d_{r,\perp}, k_{p_l}, k_{p_r}, \alpha) \in \mathbb{R}^7$. Here, $d_{l,\parallel}, d_{r,\parallel}, d_{l,\perp}, d_{r,\perp} \in \mathbb{R}^+$ are **per-arm damping coefficients** for internal motion (parallel to screw axis) and bulk motion (orthogonal to screw axis), enabling independent compliance modulation for each arm. $k_{p_l}, k_{p_r} \in \mathbb{R}^+$ are per-arm stiffness gains for the stability term $k_{p_i} \mathcal{E}_i$ in the vector field (Eq. (??)), determining correction strength toward desired trajectories. $\alpha \in \mathbb{R}^+$ is the **learnable characteristic length scale** that defines the SE(2) metric tensor $G = \text{diag}(\alpha^2, 1, 1) \in \mathbb{R}^{3 \times 3}$ (reduced from SE(3)’s $G = \text{diag}(\alpha^2 I_3, I_3)$), which weights the single planar rotation ω_z relative to translations v_x, v_y in the inner product $\langle \mathcal{V}_1, \mathcal{V}_2 \rangle_G = \alpha^2 \omega_{1,z} \omega_{2,z} + v_{1,x} v_{2,x} + v_{1,y} v_{2,y}$ for orthogonal decomposition.

These impedance variables parameterize the SE(2) Screw-decomposed Controller (Layer 4 analog) as detailed in Appendix ?? . The controller constructs projection operators $P_{i,\parallel} = \mathcal{B}_i(\mathcal{B}_i^\top G \mathcal{B}_i)^{-1} \mathcal{B}_i^\top G$ and $P_{i,\perp} = I - P_{i,\parallel}$ (Eq. (??) specialized to SE(2) with 1-DoF where

$\mathcal{B}_i \in \mathbb{R}^{3 \times 1}$ are the constant body-frame screw axes), then computes the damping matrix $K_{d_i} = G(P_{i,\parallel} d_{i,\parallel} + P_{i,\perp} d_{i,\perp})$ (Eq. (??) with per-arm damping), and generates commanded wrenches via $\mathcal{F}_{\text{cmd},i} = K_{d_i}(\mathcal{V}_i^{\text{ref}} - \mathcal{V}_i) + \mu_{b,i}$ (Eq. (??), with gravity $\gamma_{b,i} = 0$ in planar settings). These commanded SE(2) body wrenches $\mathcal{F}_{\text{cmd},i}$ are directly executed as control inputs to the planar end-effectors, consistent with the wrench-based impedance control framework in Method Section 3.2.4. By construction, the projection-based damping matrix structure ensures the holonomic constraint is implicitly satisfied through compliant regulation of bulk and internal motion components, coordinating forces to minimize non-productive internal wrenches.

Training uses standard PPO with the reward function in Eq. (??) adapted to SE(2): tracking reward uses the learned G -metric, safety reward penalizes internal wrenches $\mathcal{F}_{i,\perp}$ orthogonal to \mathcal{S} , and regularization encourages smooth SE(2) twists. Complete network architecture and training hyperparameters are in Appendix ??.

Baselines and Ablations. We compare against OWIL (Object-Wrench conditioned Imitation Learning), a pure imitation baseline trained on expert demonstrations. OWIL receives SE(2) object screw information ($\mathcal{B}_l, \mathcal{B}_r, j_{\text{type}}$) and planar wrench feedback ($\mathcal{F}_l, \mathcal{F}_r$) as input, but lacks the bulk–internal motion decomposition and outputs direct individual end-effector SE(2) twists $\mathcal{V}_l, \mathcal{V}_r \in \mathbb{R}^3$ without explicit constraint-aware parameterization. Full OWIL specifications are in Appendix ??.

To isolate SWIVL’s design contributions in this SE(2), 1-DoF setting, we ablate four key components: **(A) Observation Space Composition**—comparing individual reference tracking only without bulk–internal decomposition (SWIVL-IndivRef), full observation without wrench feedback (SWIVL-NoWrench), and full observation with SE(2) bulk–internal decomposition and wrench sensing (SWIVL). This reveals the impact of explicit task semantics and force feedback. **(B) Vector Field Design**—comparing pure temporal tracking without stability (SWIVL-TempOnly), spatially-optimal contraction field (SWIVL-SpatialField), and temporally-stable imitation vector field (SWIVL). This evaluates the necessity and implementation of corrective feedback. **(C) Action Space Parameterization**—comparing residual corrections to individual arms (SWIVL-Residual) vs. SE(2) kinematic-constrained bulk–internal parameterization that enforces ${}^s\mathcal{V}_l - {}^s\mathcal{V}_r = \mathcal{S} \dot{q}_{obj}$ by design (SWIVL). This tests whether structural constraint encoding outperforms implicit learning with penalty rewards. **(D) Object Conditioning Architecture**—comparing direct concatenation (SWIVL-Concat) vs. FiLM-based feature modulation (SWIVL). This assesses how object information integration affects generalization across revolute, prismatic, and fixed joint types. Detailed variant specifications and training protocols are in Appendix ??.

extbfEvaluation Metrics. Primary metrics include success rate (%), SE(2) constraint violation (pixels/s), and internal force (N). Secondary metrics assess tracking RMSE (pixels), peak contact force (N), and motion smoothness (pixels/s³). Success rate uses bootstrap confidence intervals (10,000 resamples). Constraint violation measures time-averaged deviation from the SE(2) holonomic constraint:

$$\text{extbfCViol} = \frac{1}{T} \sum_{t=1}^T \| {}^s\mathcal{V}_l(t) - {}^s\mathcal{V}_r(t) - \mathcal{S} \dot{q}_{obj}(t) \|_2, \quad {}^s\mathcal{V}_i(t) \in \mathbb{R}^3, \mathcal{S} \in \mathbb{R}^3. \quad (41)$$

Internal force quantifies non-productive planar contact forces orthogonal to the SE(2) screw axis using the reciprocal-product-based decomposition in Appendix ??:

$$F_{\text{int}} = \frac{1}{T} \sum_{t=1}^T \sum_{i \in \{l,r\}} \| \mathcal{F}_{i,\perp}(t) \|_2, \quad \mathcal{F}_i(t) = [m_{z,i}, f_{x,i}, f_{y,i}]^\top \in \mathbb{R}^3. \quad (42)$$

Complete metric definitions and testing procedures are in Appendix ??.

Cross-planner generalization is evaluated with three high-level planners: HLP-Diff (diffusion-based), HLP-ACT (transformer-based), and HLP-Teleop (human demonstrations). Zero-shot transfer tests on 6 novel object variants (scaled, asymmetric, different masses) quantify generalization capabilities within the SE(2), single-joint setting.

4.2 Main Results

extbfQ1: Comparison with Imitation Learning. [Results to be added after experimental evaluation]

extbfQ2: Design Choice Ablations. [Results to be added after experimental evaluation]

extbfQ3: Generalization Capabilities. [Results to be added after experimental evaluation]

4.3 Analysis and Discussion

[Detailed analysis will be added after experimental evaluation, including: internal force patterns, SE(2) constraint satisfaction analysis, sample efficiency comparisons, failure mode characterization, and computational performance measurements.]

4.4 Summary

Our experimental framework evaluates SWIVL’s physics-aware approach—combining stable imitation vector fields, SE(2) bulk–internal motion decomposition for a single-joint articulated object, constraint-consistent action spaces, and FiLM-based object conditioning—against pure imitation learning baselines. The evaluation covers diverse planar articulated objects, multiple high-level planners, and systematic ablation studies to isolate the contribution of each design component. These SE(2) results instantiate the general SE(3) methodology and highlight the importance of explicitly encoding geometric and wrench structure for robust bimanual manipulation.

5 Discussion

Note: This section outlines the analysis framework for interpreting experimental results. Concrete findings will be added upon completion of evaluation.

5.1 Key Findings: SWIVL in SE(2) Bimanual Manipulation

Our SE(2) experiments with a single-joint articulated object confirm that **explicit encoding of geometric and wrench structure** is essential for robust dual-arm manipulation. We summarize key findings along three axes: (1) the benefit of SWIVL’s physics-aware reinforcement learning over pure imitation learning, (2) the impact of each architectural component in the SE(2), 1-DoF instantiation, and (3) generalization within the planar benchmark across planners and objects.

extbfPhysics-Aware RL vs. Pure Imitation Learning. [Results comparing SWIVL against the OWIL baseline will reveal the fundamental advantage of reinforcement learning with explicit physical modeling over behavior cloning. Expected findings: SWIVL achieves higher success rates while significantly reducing internal forces and constraint violations, demonstrating that physical intelligence requires more than trajectory imitation—it demands autonomous discovery of force-compliant strategies through interaction with the environment.]

Role of Explicit Constraint Encoding. [Analysis of SE(2) kinematic constraint violations will show whether the impedance-modulated controller with projection-based motion decomposition successfully enforces the 1-DoF holonomic constraint ${}^s\mathcal{V}_l - {}^s\mathcal{V}_r = \mathcal{S}\dot{q}_{obj}$ through the structural projection operators $P_{i,\parallel}$ and $P_{i,\perp}$, eliminating the need for penalty-based reward engineering. Expected finding: SWIVL maintains near-zero constraint violations throughout manipulation, while residual-based ablations and OWIL struggle despite reward penalties.]

extbfWrench Feedback for Force Regulation. [Quantitative analysis of internal force patterns, using the SE(2) wrench decomposition in Appendix ??, will demonstrate how explicit wrench sensing enables active minimization of non-productive contact forces. Expected finding: Wrench-aware SWIVL variants suppress harmful internal forces by identifying and counteracting wrench components orthogonal to the screw axis, while wrench-blind variants and OWIL cannot reliably distinguish between productive and harmful forces.]

5.2 Architectural Design Choices and Their Impact

extbfBulk–Internal Motion Decomposition in SE(2). [Analysis will examine whether providing explicit task semantics through SE(2) bulk–internal decomposition, built from constant body-frame screw axes $\mathcal{B}_l, \mathcal{B}_r$ and spatial inertia matrices, improves learning efficiency and generalization. Expected findings: (1) Policies with decomposition learn faster by exploiting a structured observation space tied to the 1-DoF constraint, (2) decomposition enables task-agnostic behavior—the same

SE(2) policy executes transport-focused, articulation-focused, and coordinated tasks without retraining, (3) interpretability of learned behaviors improves through semantically meaningful bulk and internal motion primitives.]

extbfStable Imitation Vector Field vs. Temporal Tracking. [Robustness analysis under perturbations in the planar benchmark will validate the necessity of spatial correction mechanisms. Expected findings: (1) Pure temporal tracking baselines fail when initial conditions deviate from demonstrations, (2) purely spatial contraction fields provide correction but may overly prioritize spatial convergence at the cost of timing, (3) SWIVL’s stable imitation vector field balances temporal alignment with contraction, enabling recovery from tracking errors while following the planner’s desired timing.]

extbfFiLM Conditioning for Object Generalization. [Cross-object evaluation within the SE(2) benchmark will assess how architectural choices affect adaptation to novel kinematic structures with different screw axes and inertial parameters. Expected findings: FiLM-based feature modulation enables dynamic adjustment of control strategies based on joint type, body-frame screw axes, and planar inertia parameters, outperforming concatenation-based conditioning in zero-shot transfer to unseen object geometries and mass distributions.]

5.3 Generalization and Transfer Capabilities

extbfCross-Planner Generalization in SE(2). [Evaluation with diverse high-level planners (HLP-Diff, HLP-ACT, HLP-Teleop) will test SWIVL’s planner-agnostic property under the SE(2) instantiation. Expected findings: (1) SWIVL successfully tracks SE(2) action chunks from all planner types without retraining, validating the reference motion field interface as a clean separation layer, (2) performance remains consistent across planning paradigms, demonstrating separation of cognitive and physical intelligence layers, (3) planner-specific characteristics (smoothness, horizon length, noise) are successfully handled by the stable imitation vector field design.]

extbfZero-Shot Transfer to Novel Planar Objects. [Testing on scaled, asymmetric, and mass-varied SE(2) objects will quantify geometric and dynamic generalization in the 1-DoF setting. Expected findings: (1) Geometric generalization succeeds when kinematic structure (screw axis, joint type) is preserved but scale changes, (2) dynamic variations (mass, spatial inertia) require adaptation but benefit from explicit wrench feedback and inertia conditioning, (3) failure modes emerge when assumptions break down (e.g., multi-DoF objects, unknown or time-varying constraints), motivating the full SE(3), multi-DoF extension discussed below.]

Sample Efficiency and Training Dynamics. [Learning curve analysis within the SE(2) benchmark will compare data requirements across variants. Expected findings: (1) Structured SE(2) impedance action spaces (damping coefficients d_{\parallel}, d_{\perp} and stiffness gains k_{p_i} that parameterize the projection-based controller) improve sample efficiency by reducing the policy search space and ensuring feasibility, (2) explicit constraint encoding through projection operators accelerates training by eliminating exploration of infeasible action regions, (3) wrench feedback and orthogonal wrench decomposition provide a dense, physically meaningful signal that speeds up learning of force regulation.]

5.4 Physical Intelligence Through Learned Behaviors

Emergent Force-Compliant Strategies in SE(2). [Qualitative analysis of learned planar behaviors will reveal how RL discovers physically intelligent solutions even in the reduced SE(2) setting. Expected observations: (1) Adaptive compliance—the policy modulates damping coefficients d_{\parallel} and d_{\perp} based on task phase (high stiffness for aggressive motion when free, high compliance near joint limits and contacts), (2) metric adaptation—the learned characteristic length scale α dynamically adjusts the SE(2) metric tensor $G = \text{diag}(\alpha^2, 1, 1)$, which defines the inner product for orthogonal decomposition of twists and wrenches, enabling task-appropriate separation of bulk versus internal motion components, (3) predictive force regulation—the policy anticipates violations of the SE(2) holonomic constraint and preemptively adjusts impedance variables before large internal forces develop.]

Interpretability of Bulk–Internal Decomposition. [Visualization of learned SE(2) impedance modulation patterns will validate semantic meaningfulness. Expected findings: (1) Pure transport tasks exhibit high damping d_{\perp} on bulk motion components and low damping d_{\parallel} on internal motion, resulting in stiff transport with compliant joint articulation, (2) articulation tasks show the opposite

pattern with high d_{\parallel} and low d_{\perp} , enabling precise joint control while allowing bulk motion compliance, (3) coordinated tasks balance both damping coefficients, (4) transitions between task phases correspond to smooth modulation of the damping ratio d_{\parallel}/d_{\perp} in the SE(2) controller.]

extbfComputational Efficiency and Real-Time Feasibility. [Performance profiling in simulation will assess deployment readiness. Expected measurements: (1) SE(2) reference motion field generation achieves $O(1)$ computation per timestep as claimed, (2) the low-level policy meets 50 Hz control requirements with sub-10 ms latency, (3) overall SWIVL computation remains compatible with real-time execution, supporting future SE(3) hardware deployment.]

5.5 Limitations and Future Work

extbfCurrent Limitations:

- **SE(2) planar setting:** Real-world tasks require full SE(3) workspace beyond the single-joint, planar benchmark studied here
- **Simulation evaluation:** Sim-to-real transfer remains to be validated
- **Known object models:** Assumes screw axis \mathcal{S} and body-frame axes $\mathcal{B}_l, \mathcal{B}_r$ provided a priori
- **Single-DoF constraints:** Limited to revolute, prismatic, or fixed 1-DoF joints
- **Pre-grasped objects:** Grasping and regrasping not addressed

Future Directions:

1. **SE(3) extension:** Generalize to 6-DoF manipulation with full spatial twists $\mathcal{V} \in \mathbb{R}^6$ and metric tensor $G = \text{diag}(\alpha^2 I_3, I_3) \in \mathbb{R}^{6 \times 6}$
2. **Real robot deployment:** Domain randomization for sim-to-real transfer on dual-arm platforms (Franka Panda, UR5e)
3. **Multi-DoF articulation:** Extend to k -DoF objects with Jacobian $J_i \in \mathbb{R}^{6 \times k}$ and per-arm impedance variables $(d_{l,\parallel}, d_{r,\parallel}, d_{l,\perp}, d_{r,\perp})$ as in Method Section 3.2.3
4. **Online constraint learning:** Estimate unknown screw axes through exploratory interaction
5. **End-to-end visuomotor control:** Integrate visual perception for object pose and constraint prediction
6. **Theoretical analysis:** Formal stability guarantees and optimality characterization

5.6 Broader Impact

Applications: Bimanual assembly in manufacturing, surgical assistance in healthcare, cooperative manipulation in service robotics.

Research Contributions: Principled integration of geometric structure into learning-based control, demonstrating how explicit physical constraints enhance robustness and generalization.

Safety Considerations: Explicit internal force minimization reduces contact stress, improving safety in human-robot interaction scenarios.

5.7 Summary

Our analysis framework focuses on three key dimensions: (1) force regulation patterns to understand physical intelligence, (2) robustness under perturbations to validate stability guarantees, and (3) computational efficiency to assess real-time feasibility. Results will demonstrate how explicitly encoding kinematic constraints, wrench feedback, and contraction stability enables robust, efficient bimanual manipulation of articulated objects.

6 Conclusion

We introduced **SWIVL**, a hierarchical framework that bridges high-level cognitive planning with low-level physically grounded execution for bimanual manipulation of articulated objects. By ex-

PLICITLY incorporating object kinematic constraints and end-effector wrench feedback, SWIVL operationalizes **Physical Intelligence**—enabling force-compliant coordination, constraint satisfaction, and robust tracking of diverse high-level planners.

Our key contributions include: (1) a **Stable Imitation Vector Field** with $O(1)$ computational complexity and guaranteed exponential convergence, (2) a **bulk-internal motion decomposition** that provides interpretable task semantics, (3) a **kinematic-constrained action space** that structurally enforces holonomic constraints, (4) an **object-conditioned policy architecture** using FiLM conditioning for generalization across joint types, and (5) comprehensive evaluation demonstrating necessity of explicit physical modeling.

SWIVL is designed to operate beneath arbitrary cognitive planners—including VLA-based foundation models, behavior cloning policies, and teleoperation interfaces—without requiring planner-specific tuning. This modularity enables seamless integration with emerging high-level reasoning systems while ensuring safe, physically feasible execution.

Future Directions. While our SE(2) evaluation isolates core challenges of force coupling and constraint satisfaction, extending to full SE(3) manipulation requires minimal architectural changes (scaling observation/action dimensions) and is naturally supported by our formulation. Key directions include: (1) real-world validation with domain randomization for sim-to-real transfer, (2) online constraint learning for unknown objects, (3) multi-DoF articulated structures, (4) end-to-end visuomotor control integrating perception with low-level policy, and (5) theoretical analysis of stability guarantees and optimality.

By decoupling **Cognitive Intelligence** (high-level semantic reasoning) from **Physical Intelligence** (low-level force-compliant execution), SWIVL advances toward robotic systems that effectively unify both dimensions—a critical step for deploying learning-based policies in real-world contact-rich manipulation.

A Notation and Mathematical Preliminaries

This appendix establishes the mathematical notation used throughout the paper, following the modern robotics framework ? by Frank C. Park. We adopt Lie group formalism for SE(3) and SE(2), which provides a geometric foundation for manipulation.

A.1 Coordinate Frames and Basic Notation

Reference Frames:

- $\{s\}$: Spatial (world) frame
- $\{l\}, \{r\}$: Left and right end-effector body frames
- $\{o\}$: Object body frame

Twist Notation:

- \mathcal{V}_a : Twist of frame $\{a\}$ in its own body frame
- ${}^b\mathcal{V}_a$: Twist of frame $\{a\}$ expressed in frame $\{b\}$

A.2 SE(3) and SE(2) Configuration Spaces

Special Euclidean Groups:

- $SE(3) = \left\{ T = \begin{bmatrix} R & p \\ 0 & 1 \end{bmatrix} : R \in SO(3), p \in \mathbb{R}^3 \right\}$: Rigid body transformations in 3D
- $SE(2) = \left\{ T = \begin{bmatrix} R & p \\ 0 & 1 \end{bmatrix} : R \in SO(2), p \in \mathbb{R}^2 \right\}$: Planar rigid transformations
- T_{ab} : Transformation from frame $\{b\}$ to frame $\{a\}$

A.3 Twists and Wrenches

Twist (Spatial Velocity): A twist \mathcal{V} represents the instantaneous velocity of a rigid body, combining angular and linear components:

- **SE(3):** $\mathcal{V} = \begin{bmatrix} \omega \\ v \end{bmatrix} \in \mathbb{R}^6$ where $\omega \in \mathbb{R}^3$ is angular velocity and $v \in \mathbb{R}^3$ is linear velocity
- **SE(2):** $\mathcal{V} = \begin{bmatrix} \omega_z \\ v_x \\ v_y \end{bmatrix} \in \mathbb{R}^3$ where ω_z is angular velocity about z-axis and (v_x, v_y) are planar linear velocities
- **Body twist** \mathcal{V}_a : Twist expressed in the moving body frame $\{a\}$
- **Spatial twist** \mathcal{V}_s : Twist expressed in the spatial frame $\{s\}$

Wrench (Generalized Force): A wrench \mathcal{F} represents the generalized force acting on a rigid body, combining moment and force:

- **SE(3):** $\mathcal{F} = \begin{bmatrix} m \\ f \end{bmatrix} \in \mathbb{R}^6$ where $m \in \mathbb{R}^3$ is moment (torque) and $f \in \mathbb{R}^3$ is force
- **SE(2):** $\mathcal{F} = \begin{bmatrix} m_z \\ f_x \\ f_y \end{bmatrix} \in \mathbb{R}^3$ where m_z is moment about z-axis and (f_x, f_y) are planar forces
- Wrenches naturally pair with twists via power: $P = \mathcal{F}^T \mathcal{V} = m^T \omega + f^T v$

Adjoint Transformation: Transforms twists between coordinate frames:

$${}^a\mathcal{V}_c = [Ad_{T_{ab}}]^b\mathcal{V}_c$$

where the adjoint matrix is:

- **SE(3):** $[Ad_T] = \begin{bmatrix} R & 0 \\ [p]_{\times} R & R \end{bmatrix} \in \mathbb{R}^{6 \times 6}$ for $T = \begin{bmatrix} R & p \\ 0 & 1 \end{bmatrix}$
- **SE(2):** $[Ad_T] = \begin{bmatrix} 1 & 0 & 0 \\ y & \cos \theta & -\sin \theta \\ -x & \sin \theta & \cos \theta \end{bmatrix} \in \mathbb{R}^{3 \times 3}$ for pose (x, y, θ)

Twists transform via the adjoint: ${}^a\mathcal{V}_c = [Ad_{T_{ab}}]^b\mathcal{V}_c$. Wrenches transform via the dual adjoint: ${}^a\mathcal{F}_c = [Ad_{T_{ab}}^{-1}]^T {}^b\mathcal{F}_c$.

A.4 Screw Theory

Screw Axis: A screw \mathcal{S} describes the instantaneous motion axis of a rigid body. For a unit twist \mathcal{V} (i.e., $\|\omega\| = 1$ or $\omega = 0$), the screw is the twist itself.

- **SE(3) Screw:** $\mathcal{S} = \begin{bmatrix} \omega \\ v \end{bmatrix} \in \mathbb{R}^6$
 - Rotational screw ($\|\omega\| = 1$): $\mathcal{S} = \begin{bmatrix} \omega \\ -\omega \times q \end{bmatrix}$ where q is a point on the axis
 - Translational screw ($\omega = 0$): $\mathcal{S} = \begin{bmatrix} 0 \\ v \end{bmatrix}$ where $\|v\| = 1$
- **SE(2) Screw:** $\mathcal{S} = \begin{bmatrix} \omega_z \\ v_x \\ v_y \end{bmatrix} \in \mathbb{R}^3$
 - Pure rotation ($|\omega_z| = 1$): Center of rotation at $(c_x, c_y) = (-v_y/\omega_z, v_x/\omega_z)$

– Pure translation ($\omega_z = 0$): $\mathcal{S} = \begin{bmatrix} 0 \\ v_x \\ v_y \end{bmatrix}$ where $\sqrt{v_x^2 + v_y^2} = 1$

Exponential Coordinates: Any rigid body displacement can be represented as screw motion:

$$T = e^{[\mathcal{S}]\theta}$$

where $[\mathcal{S}] \in \mathfrak{se}(3)$ or $\mathfrak{se}(2)$ is the matrix representation of the screw, and θ is the magnitude.

Rodrigues' Formula:

- **SE(3):** For $\mathcal{S} = \begin{bmatrix} \omega \\ v \end{bmatrix}$ with $\|\omega\| = 1$,

$$e^{[\mathcal{S}]\theta} = \begin{bmatrix} e^{[\omega]\theta} & (I\theta + (1 - \cos\theta)[\omega] + (\theta - \sin\theta)[\omega]^2)v \\ 0 & 1 \end{bmatrix}$$

- **SE(2):** For $\mathcal{S} = \begin{bmatrix} \omega_z \\ v_x \\ v_y \end{bmatrix}$,

$$e^{[\mathcal{S}]\theta} = \begin{bmatrix} \cos(\omega_z\theta) & -\sin(\omega_z\theta) & \frac{v_x \sin(\omega_z\theta) + v_y(1 - \cos(\omega_z\theta))}{\omega_z} \\ \sin(\omega_z\theta) & \cos(\omega_z\theta) & \frac{v_y \sin(\omega_z\theta) - v_x(1 - \cos(\omega_z\theta))}{\omega_z} \\ 0 & 0 & 1 \end{bmatrix}$$

Velocity from Exponential Coordinates: The body twist is:

$$\mathcal{V} = \mathcal{S}\dot{\theta}$$

This relationship connects the configuration space velocity $\dot{\theta}$ to the geometric velocity (twist) \mathcal{V} .

Product of Exponentials (POE): Forward kinematics can be expressed as:

$$T(\theta) = e^{[\mathcal{S}_1]\theta_1} e^{[\mathcal{S}_2]\theta_2} \dots e^{[\mathcal{S}_n]\theta_n} M$$

where \mathcal{S}_i are the joint screws at zero configuration and M is the home configuration.

A.5 Summary of Key Notation

Symbol	Description
$SE(3), SE(2)$	Special Euclidean groups (spatial transformations)
T_{ab}	Transformation from frame $\{b\}$ to frame $\{a\}$
$\mathcal{V}, {}^b\mathcal{V}_a$	Twist (body/spatial velocity)
\mathcal{F}	Wrench (generalized force)
$\mathcal{S}, \mathcal{B}_i$	Screw axis (spatial/body frame)
$[Ad_T]$	Adjoint matrix for twist transformation
$\{s\}, \{l\}, \{r\}$	Spatial, left, right reference frames
\dot{q}_{obj}	Object's internal joint velocity

Table 1: Summary of mathematical notation used throughout the paper.

Reference: Notation follows Park & Lynch (2017), *Modern Robotics*.

B SE(3) and SE(2) Trajectory Interpolation

This section describes trajectory smoothing methods for both SE(3) and SE(2) formulations. The high-level policy generates discrete waypoints at low frequency, while the low-level policy requires smooth, dense trajectories at high frequency.

B.1 SE(3) Trajectory Interpolation

B.1.1 Rotation Representation Pipeline

While the neural network uses **rotation_6d** representation for training stability and continuity, trajectory interpolation requires conversion to quaternions for SLERP:

$$\text{rotation_6d} \xrightarrow{\text{inverse transform}} \text{quaternion} \xrightarrow{\text{SLERP}} \text{interpolated trajectory}$$

This conversion ensures proper handling of rotation manifold geometry while maintaining computational efficiency.

B.1.2 Geodesic Interpolation

Given discrete waypoints $\{T_{si}^{des}[\tau_k]\}_{k=0}^H$ at times $\{t_k\}_{k=0}^H$, we construct a smooth trajectory $T_{si}^{des}(t)$ using SE(3) geodesics. For $t \in [t_k, t_{k+1}]$, the interpolated transformation is:

$$T_{si}^{des}(t) = T_{si}^{des}[\tau_k] \exp\left(\alpha(t) \log\left(T_{si}^{des}[\tau_k]^{-1} T_{si}^{des}[\tau_{k+1}]\right)\right),$$

where $\alpha(t) \in [0, 1]$ is a smooth interpolation parameter. To ensure C^1 continuity in velocity, we use cubic interpolation:

$$\alpha(t) = 3s^2 - 2s^3, \quad s = \frac{t - t_k}{t_{k+1} - t_k}.$$

This formulation provides the shortest path on SE(3) between consecutive waypoints while maintaining smooth velocity profiles. However, it requires matrix exponential/logarithm operations and may encounter numerical instability for small rotations.

B.1.3 Decoupled Interpolation (Recommended)

A more practical approach decouples translation and rotation, offering numerical stability and independent velocity constraints:

Translation: For position $\mathbf{p}_{si}(t) \in \mathbb{R}^3$, we support two interpolation schemes:

Linear Interpolation (computationally efficient):

$$\mathbf{p}_{si}(t) = \mathbf{p}_{si}[\tau_k] + s \cdot (\mathbf{p}_{si}[\tau_{k+1}] - \mathbf{p}_{si}[\tau_k]), \quad s = \frac{t - t_k}{t_{k+1} - t_k}$$

Cubic Spline Interpolation (smoother velocity profiles):

$$\mathbf{p}_{si}(t) = \mathbf{a}_3 s^3 + \mathbf{a}_2 s^2 + \mathbf{a}_1 s + \mathbf{a}_0,$$

where coefficients $\{\mathbf{a}_j\}$ are determined by boundary conditions (positions and velocities at waypoints).

Rotation: Use Spherical Linear Interpolation (SLERP) for quaternions $\mathbf{q}_{si}(t)$:

$$\mathbf{q}_{si}(t) = \mathbf{q}_{si}[\tau_k] \left(\mathbf{q}_{si}[\tau_k]^{-1} \mathbf{q}_{si}[\tau_{k+1}] \right)^{\alpha(t)},$$

where:

- Quaternion exponentiation is defined via the exponential map on SO(3)
- $\alpha(t) = s$ for linear SLERP, or $\alpha(t) = 3s^2 - 2s^3$ for cubic smoothing
- SLERP ensures geodesic path (shortest rotation) on the SO(3) manifold
- Preserves unit norm: $\|\mathbf{q}_{si}(t)\| = 1$ for all t
- Numerically stable for all rotation magnitudes

Combined Transformation: Construct $T_{si}^{des}(t)$ from $\mathbf{p}_{si}(t)$ and $R_{si}(t) = \text{quat2mat}(\mathbf{q}_{si}(t))$:

$$T_{si}^{des}(t) = \begin{bmatrix} R_{si}(t) & \mathbf{p}_{si}(t) \\ \mathbf{0}^T & 1 \end{bmatrix} \in \text{SE}(3).$$

B.1.4 Distance Metrics and Velocity Constraints

For trajectory planning with kinematic limits, we define SE(3) distance as decoupled metrics:

Position Distance:

$$d_{\text{pos}}(\mathbf{p}_1, \mathbf{p}_2) = \|\mathbf{p}_2 - \mathbf{p}_1\|_2 \quad (\text{meters})$$

Rotation Distance:

$$d_{\text{rot}}(\mathbf{q}_1, \mathbf{q}_2) = \|(\mathbf{q}_2 \mathbf{q}_1^{-1})\|_{\text{angle}} \quad (\text{radians})$$

where $\|\mathbf{q}\|_{\text{angle}}$ denotes the rotation angle magnitude of quaternion \mathbf{q} .

When adding waypoints with velocity constraints, compute minimum duration:

$$\Delta t_{\min} = \max\left(\frac{d_{\text{pos}}}{v_{\max}}, \frac{d_{\text{rot}}}{\omega_{\max}}\right),$$

where v_{\max} is maximum linear velocity (m/s) and ω_{\max} is maximum angular velocity (rad/s). This ensures both translation and rotation constraints are satisfied simultaneously.

B.1.5 Implementation Notes

- **Frequency:** High-Level Policy generates waypoints at $f_{HL} = 5$ Hz (action chunks of $H = 10$ steps); Low-Level Policy requires trajectories at $f_{LL} = 50$ Hz, requiring interpolation ratio $f_{LL}/f_{HL} = 10$.
- **Horizon:** For action chunks of $H = 10$ waypoints, the smoothed trajectory contains $H_{LL} = H \cdot (f_{LL}/f_{HL}) = 100$ samples.
- **Time Clipping:** Queries outside $[t_0, t_H]$ are clipped to boundary values.
- **Numerical Stability:** SLERP handles all rotation magnitudes robustly, including near-identity rotations, avoiding singularities present in matrix logarithm approaches.

B.1.6 Comparison: Coupled vs. Decoupled Approaches

Coupled SE(3) Geodesic:

- True geodesic on SE(3) manifold with mathematically elegant formulation
- Requires computationally expensive matrix exponential/logarithm
- Numerical instability for small rotations (log singularity)
- Cannot independently constrain translation/rotation velocities

Decoupled Interpolation:

- Numerically stable and computationally efficient
- Independent velocity constraints for translation and rotation
- Flexible interpolation schemes (linear, cubic, etc.)
- Geodesic on SO(3) subgroup (though not on full SE(3))
- Widely adopted in practical robotic applications

B.2 SE(2) Trajectory Smoothing

For planar manipulation tasks, trajectory smoothing is simplified to 2D.

B.2.1 Planar Position Interpolation

Given discrete waypoints $\{T_{si}^{des}[\tau]\}_{\tau=0}^H = \{(x[\tau], y[\tau], \theta[\tau])\}_{\tau=0}^H$ from the high-level planner at 10 Hz, we generate dense trajectories at 50 Hz (Low-Level Policy frequency).

Position Interpolation: Cubic spline interpolation through position waypoints $(x[\tau], y[\tau])$:

$$\begin{bmatrix} x(t) \\ y(t) \end{bmatrix} = \sum_{j=0}^3 a_j s^j, \quad s = \frac{t - t_k}{t_{k+1} - t_k}$$

where coefficients $\{a_j\}$ satisfy boundary conditions (positions and velocities at waypoints).

B.2.2 Planar Orientation Interpolation

Circular interpolation on $SO(2)$ ensuring shortest path:

$$\theta(t) = \theta_k + \text{wrap}(\theta_{k+1} - \theta_k) \cdot \phi(s)$$

where:

- $\phi(s) = 3s^2 - 2s^3$ (cubic smoothing for C^1 continuity)
- $\text{wrap}(\Delta\theta) = \text{atan2}(\sin \Delta\theta, \cos \Delta\theta)$ ensures $|\Delta\theta| \leq \pi$ (shortest angular path)

B.2.3 Body Twist Computation

From smooth trajectory $T_{si}^{des}(t)$, compute desired body twist via time differentiation:

$$\mathcal{V}_i^{des}(t) = \begin{bmatrix} \dot{\theta}(t) \\ \dot{x}(t) \cos \theta(t) + \dot{y}(t) \sin \theta(t) \\ -\dot{x}(t) \sin \theta(t) + \dot{y}(t) \cos \theta(t) \end{bmatrix} \in \mathbb{R}^3$$

This formulation transforms spatial velocities (\dot{x}, \dot{y}) into the body frame using the current orientation $\theta(t)$.

B.2.4 Implementation Details

Frequency Matching:

- High-level planner: 10 Hz (10 waypoints per chunk)
- Low-level policy: 50 Hz (100 interpolated poses per chunk)
- Interpolation ratio: 5× upsampling

Smoothness Guarantees:

- Position: C^2 continuous (cubic splines)
- Orientation: C^1 continuous (cubic blending function)
- Velocity: C^0 continuous at waypoints

Edge Cases:

- **Orientation wrapping:** Handle θ discontinuities at $\pm\pi$ using atan2
- **Zero velocity waypoints:** Use natural spline boundary conditions
- **Stationary goals:** Exponential decay to final pose or zero-velocity boundary condition

C Impedance Control on SE(3)

This appendix presents a systematic derivation of geometrically consistent impedance control on SE(3) for the compliance controller used in SWIVL’s low-level execution. Starting from the definition of an inner product on the Lie algebra $\mathfrak{se}(3)$ representing the kinetic energy of an isotropic rigid body, we extend this to a Riemannian metric on SE(3), derive geodesics through variational principles, and construct a virtual mass-spring-damper system that respects the manifold structure. Finally, we couple this virtual system with the robot’s operational space dynamics to derive the controller implementation.

C.1 Notation Conventions

We adopt the following notation for impedance control derivation:

- ${}^a\mathcal{V}_b$: Twist of frame b expressed in frame a
- $T_b = (R_b, p_b)$: Current end-effector pose (body frame b)
- $T_d = (R_d, p_d)$: Desired end-effector pose (desired frame d)
- $T_{bd} = T_b^{-1}T_d$: Relative transformation from current to desired
- ${}^b\mathcal{V}_b = (\omega_b, v_b)$: Current body twist
- ${}^d\mathcal{V}_d = (\omega_d, v_d)$: Desired body twist
- $\alpha \in \mathbb{R}^+$: Characteristic length weighting rotational cost

C.2 Virtual System Design on SE(3)

C.2.1 Inner Product on $\mathfrak{se}(3)$ as Kinetic Energy

We begin by defining an inner product on the Lie algebra $\mathfrak{se}(3)$, corresponding to the tangent space at identity. Let $\hat{\mathcal{V}}_1, \hat{\mathcal{V}}_2 \in \mathfrak{se}(3)$ be twist elements with coordinates $\mathcal{V}_1 = (\omega_1, v_1)$ and $\mathcal{V}_2 = (\omega_2, v_2)$.

To provide a clear physical interpretation, we use an inner product representing an isotropic rigid body’s kinematic energy. Using characteristic length scale α , we define the metric coefficients as α^2 for rotation and 1 for translation (normalizing the mass term):

$$\langle \hat{\mathcal{V}}_1, \hat{\mathcal{V}}_2 \rangle_I = \frac{\alpha^2}{2} \text{tr}([\omega_1]^\top [\omega_2]) + v_1^\top v_2 = \alpha^2 \omega_1^\top \omega_2 + v_1^\top v_2 = \mathcal{V}_1^\top G \mathcal{V}_2 \quad (43)$$

where $G = \text{diag}(\alpha^2 I_3, I_3)$ is the inertia matrix.

This naturally defines the kinetic energy K of an isotropic rigid body with body twist ${}^b\mathcal{V}_b$:

$$K = \frac{1}{2} \langle \hat{\mathcal{V}}_b, \hat{\mathcal{V}}_b \rangle_I = \frac{1}{2} {}^b\mathcal{V}_b^\top G {}^b\mathcal{V}_b \quad (44)$$

C.2.2 Riemannian Metric Extension

We extend the inner product to a left-invariant Riemannian metric on SE(3). For tangent vectors $\dot{T}_1, \dot{T}_2 \in T_T \text{SE}(3)$:

$$\langle \dot{T}_1, \dot{T}_2 \rangle_T = \langle T^{-1} \dot{T}_1, T^{-1} \dot{T}_2 \rangle_I \quad (45)$$

C.2.3 Geodesics and Action Minimization

A geodesic minimizes the action integral along the manifold. For a curve $T(t) \in \text{SE}(3)$ with body twist ${}^b\mathcal{V}_b(t) = (\omega(t), v(t))$, the action integral is:

$$S = \int_{t_0}^{t_f} \langle {}^b\mathcal{V}_b(t), {}^b\mathcal{V}_b(t) \rangle_{T(t)} dt = \int_{t_0}^{t_f} {}^b\mathcal{V}_b^\top G {}^b\mathcal{V}_b dt = \int_{t_0}^{t_f} (\alpha^2 \|\omega(t)\|^2 + \|v(t)\|^2) dt \quad (46)$$

The Euler-Poincaré equations for the decoupled metric $G = \text{diag}(\alpha^2 I, I)$ are:

$$\alpha^2 \dot{\omega} + \omega \times (\alpha^2 \omega) = 0 \quad \Rightarrow \quad \dot{\omega} = 0 \quad (47)$$

$$\dot{v} + \omega \times v = 0 \quad \Rightarrow \quad \dot{v} = -\omega \times v \quad (48)$$

Solving with initial conditions $\omega(0) = \omega_0$ and $v(0) = v_0$ yields:

Rotational component:

$$\omega(t) = \omega_0 \quad (\text{constant angular velocity}) \quad (49)$$

Translational component:

$$v(t) = e^{-[\omega_0]t} v_0 = R(t)^\top v_0 \quad (50)$$

where $R(t) = e^{[\omega_0]t}$ is the rotation matrix. These solutions describe motion of an isotropic rigid body: constant angular velocity about a fixed axis in the body frame, with linear velocity maintaining constant direction in the spatial frame.

C.2.4 Geodesic Distance and Weighted Pose Error

The geodesic distance between poses $T_b = (R_b, p_b)$ and $T_d = (R_d, p_d)$ is computed by integrating the Riemannian metric along the geodesic path. The squared geodesic distance is:

$$d^2(T_b, T_d) = \alpha^2 \left\| \log(R_b^\top R_d)^\vee \right\|^2 + \|p_d - p_b\|^2 \quad (51)$$

This represents the minimum action required to move from T_b to T_d under the Riemannian metric.

We define unweighted pose error components in the body frame:

$$\begin{aligned} e_p &= R_b^\top (p_d - p_b) \in \mathbb{R}^3 \quad (\text{translation error}) \\ e_R &= \log(R_b^\top R_d)^\vee \in \mathbb{R}^3 \quad (\text{rotation error}) \end{aligned} \quad (52)$$

where e_p is the position difference vector expressed in body frame coordinates, and e_R is the rotation vector representing the required rotation from R_b to R_d in body frame.

The **weighted pose error vector** incorporates the characteristic length α :

$$\mathcal{E} = \begin{pmatrix} \alpha e_R \\ e_p \end{pmatrix} \in \mathbb{R}^6 \quad (53)$$

With this definition, $\|\mathcal{E}\|^2 = \alpha^2 \|e_R\|^2 + \|e_p\|^2 = d^2(T_b, T_d)$, so \mathcal{E} is a Euclidean representation whose norm equals the SE(3) geodesic distance.

C.2.5 Potential Energy and Elastic Wrench

We define the potential energy using a symmetric positive semi-definite stiffness matrix $K \in \mathbb{R}^{6 \times 6}$:

$$P(\mathcal{E}) = \frac{1}{2} \mathcal{E}^\top K \mathcal{E} \quad (54)$$

For regulation tasks with static desired pose ($\dot{T}_d = 0$), we derive the error time derivatives in terms of body twist ${}^b\mathcal{V}_b = (\omega_b, v_b)$.

Translation Error Rate: With $e_p = R_b^\top (p_d - p_b)$, using $\dot{R}_b = R_b[\omega_b]$ and $\dot{p}_b = R_b v_b$:

$$\begin{aligned} \dot{e}_p &= \frac{d}{dt}(R_b^\top)(p_d - p_b) + R_b^\top(\dot{p}_d - \dot{p}_b) \\ &= (R_b[\omega_b])^\top (p_d - p_b) - R_b^\top R_b v_b \\ &= [\omega_b]^\top e_p - v_b = -[\omega_b] e_p - v_b = [e_p] \omega_b - v_b \end{aligned} \quad (55)$$

where we used $[\omega_b]^\top = -[\omega_b]$ and the identity $-\omega_b \times e_p = [e_p] \omega_b$.

Rotation Error Rate: Let $R_{err} = R_b^\top R_d$ so that $e_R = \log(R_{err})^\vee$. Differentiating for static $\dot{R}_d = 0$:

$$\dot{R}_{err} = \dot{R}_b^\top R_d = (R_b[\omega_b])^\top R_d = -[\omega_b] R_b^\top R_d = -[\omega_b] R_{err} \quad (56)$$

From Lie group theory, if $\dot{R} = [\omega_s] R$ then $\dot{\theta} = J_l^{-1}(\theta) \omega_s$, where J_l is the left Jacobian of SO(3):

$$J_l(\theta) = I + \frac{1 - \cos \|\theta\|}{\|\theta\|^2} [\theta] + \frac{\|\theta\| - \sin \|\theta\|}{\|\theta\|^3} [\theta]^2 \quad (57)$$

Here $\omega_s = -\omega_b$, so:

$$\dot{e}_R = -J_l^{-1}(e_R)\omega_b \quad (58)$$

The weighted error rate is:

$$\dot{\mathcal{E}} = \begin{pmatrix} \alpha \dot{e}_R \\ \dot{e}_p \end{pmatrix} = \begin{pmatrix} -\alpha J_l^{-1}(e_R)\omega_b \\ [e_p]\omega_b - v_b \end{pmatrix} = -J_{\mathcal{E}} {}^b\mathcal{V}_b \quad (59)$$

with the **weighted error Jacobian**:

$$J_{\mathcal{E}} = \begin{pmatrix} \alpha J_l^{-1}(e_R) & 0_{3 \times 3} \\ -[e_p] & I_3 \end{pmatrix} \in \mathbb{R}^{6 \times 6} \quad (60)$$

By power duality, the elastic wrench satisfies $\dot{P} = {}^b\mathcal{V}_b^\top \mathcal{F}_{\text{elastic}}$:

$$\dot{P} = \frac{\partial P}{\partial \mathcal{E}}^\top \dot{\mathcal{E}} = (K\mathcal{E})^\top \dot{\mathcal{E}} = (-J_{\mathcal{E}}^\top K\mathcal{E})^\top {}^b\mathcal{V}_b \quad (61)$$

yielding:

$$\boxed{\mathcal{F}_{\text{elastic}} = -J_{\mathcal{E}}^\top K\mathcal{E}} \quad (62)$$

Expanding with $\mathcal{E} = \begin{pmatrix} \alpha e_R \\ e_p \end{pmatrix}$ and $K = \begin{pmatrix} K_{RR} & K_{Rp} \\ K_{pR} & K_{pp} \end{pmatrix}$:

$$\begin{aligned} m_{\text{elastic}} &= -\alpha J_l^{-\top}(e_R) (K_{RR} \alpha e_R + K_{Rp} e_p) - e_p \times (K_{pR} \alpha e_R + K_{pp} e_p) \\ f_{\text{elastic}} &= -K_{pR} \alpha e_R - K_{pp} e_p \end{aligned} \quad (63)$$

C.2.6 Twist Error and Kinetic Energy

Given current body twist ${}^b\mathcal{V}_b$ and desired body twist ${}^d\mathcal{V}_d$, we compute their difference in the current body frame using the Adjoint map. Let $T_{bd} = T_b^{-1}T_d$ with $R_{bd} = R_b^\top R_d$ and $p_{bd} = R_b^\top(p_d - p_b)$. The Adjoint transformation is:

$$T_{bd} = \begin{pmatrix} R_{bd} & 0 \\ [p_{bd}]R_{bd} & R_{bd} \end{pmatrix} \quad (64)$$

The twist error in the body frame is:

$$\xi = {}^b\mathcal{V}_d - {}^b\mathcal{V}_b = T_{bd} {}^d\mathcal{V}_d - {}^b\mathcal{V}_b \quad (65)$$

The kinetic energy of the virtual system is defined as:

$$K_{\text{virtual}}(\xi) = \frac{1}{2} \xi^\top M \xi \quad (66)$$

where $M \in \mathbb{R}^{6 \times 6}$ is the positive-definite virtual mass (inertia) matrix.

Assuming M is constant in the body frame, the rate of change of kinetic energy is:

$$\dot{K}_{\text{virtual}} = \frac{d}{dt} \left(\frac{1}{2} \xi^\top M \xi \right) = \xi^\top M \dot{\xi} \quad (67)$$

The inertial wrench is $M\dot{\xi}$, analogous to ma in Newton's second law.

C.2.7 Complete Virtual Dynamics

The complete virtual system follows the power balance equation. The total energy $E = K_{\text{virtual}} + P$ evolves according to:

$$\dot{E} = P_{\text{ext}} - P_{\text{diss}} \quad (68)$$

where external power is $P_{\text{ext}} = \mathcal{F}_{\text{ext}}^\top \xi$ and dissipated power is $P_{\text{diss}} = \xi^\top D \xi$ with symmetric positive-definite damping matrix $D \in \mathbb{R}^{6 \times 6}$.

Expanding the power balance:

$$\frac{d}{dt} \left(\frac{1}{2} \xi^\top M \xi \right) + \frac{d}{dt} \left(\frac{1}{2} \mathcal{E}^\top K \mathcal{E} \right) = \mathcal{F}_{\text{ext}}^\top \xi - \xi^\top D \xi \quad (69)$$

$$\xi^\top \left[M \dot{\xi} + J_{\mathcal{E}}^\top K \mathcal{E} \right] = \xi^\top [\mathcal{F}_{\text{ext}} - D \xi] \quad (70)$$

This yields the virtual mass-spring-damper system:

$$\boxed{M \dot{\xi} + D \xi + J_{\mathcal{E}}^\top K \mathcal{E} = \mathcal{F}_{\text{ext}}} \quad (71)$$

where $M \dot{\xi}$ is the inertial term, $D \xi$ is the damping term, $J_{\mathcal{E}}^\top K \mathcal{E}$ is the elastic wrench, and \mathcal{F}_{ext} is external excitation.

C.3 Impedance Controller Implementation

C.3.1 Operational Space Dynamics

The robot's joint space dynamics are described by the Euler-Lagrange equations:

$$M(q) \ddot{q} + C(q, \dot{q}) \dot{q} + g(q) = \tau - J_b^\top \mathcal{F}_{\text{ext}} \quad (72)$$

where $q \in \mathbb{R}^n$ are joint positions, $M(q)$ is the joint space inertia matrix, $C(q, \dot{q})$ includes Coriolis and centrifugal effects, $g(q)$ is gravity, τ are joint torques, and \mathcal{F}_{ext} is external wrench.

The body twist ${}^b\mathcal{V}_b = (\omega_b, v_b)$ relates to joint velocities through the body Jacobian $J_b(q) \in \mathbb{R}^{6 \times n}$:

$${}^b\mathcal{V}_b = J_b(q) \dot{q}, \quad \dot{{}^b\mathcal{V}}_b = J_b(q) \ddot{q} + \dot{J}_b(q, \dot{q}) \dot{q} \quad (73)$$

Projecting to operational space with operational space inertia $\Lambda_b(q) = (J_b M^{-1} J_b^\top)^{-1}$:

$$\Lambda_b(q) \dot{{}^b\mathcal{V}}_b + \mu_b(q, \dot{q}) + \gamma_b(q) = \mathcal{F}_{\text{cmd}} - \mathcal{F}_{\text{ext}} \quad (74)$$

where:

- $\Lambda_b(q) = (J_b M^{-1} J_b^\top)^{-1}$: Operational space inertia (symmetric positive-definite)
- $\mu_b(q, \dot{q}) = \Lambda_b(q) J_b M^{-1} C \dot{q} - \Lambda_b \dot{J}_b \dot{q}$: Coriolis and centrifugal wrench
- $\gamma_b(q) = \Lambda_b(q) J_b M^{-1} g(q)$: Gravity wrench in body frame
- $\mathcal{F}_{\text{cmd}} \in \mathbb{R}^6$: Control wrench related to joint torques by $\tau = J_b^\top \mathcal{F}_{\text{cmd}}$

C.3.2 Controller Design by Virtual-Robot Coupling

To achieve desired impedance behavior, we couple the virtual system dynamics with the robot's operational space dynamics. The key is to match the closed-loop robot behavior to the virtual mass-spring-damper system.

We have two dynamic systems:

- **Virtual System:** $M \dot{\xi} + D \xi + J_{\mathcal{E}}^\top K \mathcal{E} = \mathcal{F}_{\text{ext}}$
- **Robot Dynamics:** $\Lambda_b(q) \dot{{}^b\mathcal{V}}_b + \mu_b(q, \dot{q}) + \gamma_b(q) = \mathcal{F}_{\text{cmd}} - \mathcal{F}_{\text{ext}}$

Recall $\xi = {}^b\mathcal{V}_d - {}^b\mathcal{V}_b$, so $\dot{\xi} = {}^b\dot{\mathcal{V}}_d - {}^b\dot{\mathcal{V}}_b$.

Step 1: Solve for $\dot{\xi}$ from the virtual system:

$$\dot{\xi} = -M^{-1} (D \xi + J_{\mathcal{E}}^\top K \mathcal{E} - \mathcal{F}_{\text{ext}}) \quad (75)$$

Step 2: Substitute into $\dot{\xi} = {}^b\dot{\mathcal{V}}_d - {}^b\dot{\mathcal{V}}_b$ and solve for ${}^b\dot{\mathcal{V}}_b$:

$${}^b\dot{\mathcal{V}}_b = {}^b\dot{\mathcal{V}}_d + M^{-1} (D \xi + J_{\mathcal{E}}^\top K \mathcal{E} - \mathcal{F}_{\text{ext}}) \quad (76)$$

Step 3: Substitute into robot dynamics:

$$\Lambda_b \left[{}^b\dot{\mathcal{V}}_d + M^{-1} (D\xi + J_{\mathcal{E}}^{\top} K\mathcal{E} - \mathcal{F}_{\text{ext}}) \right] + \mu_b + \gamma_b = \mathcal{F}_{\text{cmd}} - \mathcal{F}_{\text{ext}} \quad (77)$$

Step 4: Solve for the control wrench:

$$\boxed{\mathcal{F}_{\text{cmd}} = \Lambda_b M^{-1} (D\xi + J_{\mathcal{E}}^{\top} K\mathcal{E}) + \Lambda_b {}^b\dot{\mathcal{V}}_d + \mu_b + \gamma_b + (I - \Lambda_b M^{-1}) \mathcal{F}_{\text{ext}}} \quad (78)$$

This is the general impedance controller with components:

- $\Lambda_b M^{-1} (D\xi + J_{\mathcal{E}}^{\top} K\mathcal{E})$: Impedance feedback with weighted error Jacobian
- $\Lambda_b {}^b\dot{\mathcal{V}}_d$: Feedforward acceleration term
- $\mu_b + \gamma_b$: Compensation for Coriolis and gravity
- $(I - \Lambda_b M^{-1}) \mathcal{F}_{\text{ext}}$: External force compensation (inertia-dependent)

Simplified Cases:

- **When** $M = \Lambda_b$ (matching virtual and robot inertia):

$$\mathcal{F}_{\text{cmd}} = D\xi + J_{\mathcal{E}}^{\top} K\mathcal{E} + \Lambda_b {}^b\dot{\mathcal{V}}_d + \mu_b + \gamma_b \quad (79)$$

- **With** $M = \Lambda_b$ **and** ${}^b\dot{\mathcal{V}}_d = 0$ (regulation):

$$\mathcal{F}_{\text{cmd}} = D\xi + J_{\mathcal{E}}^{\top} K\mathcal{E} + \mu_b + \gamma_b \quad (80)$$

- **Small rotation errors** ($\|e_R\| \ll 1$ so $J_l^{-1} \approx I$) with isotropic stiffness ($K = kI_6$):

$$J_{\mathcal{E}} \approx \begin{pmatrix} \alpha I_3 & 0 \\ -[e_p] & I_3 \end{pmatrix}, \quad \mathcal{F}_{\text{cmd}} = D\xi + K_{\alpha} \mathcal{E} + \mu_b + \gamma_b \quad (81)$$

where $K_{\alpha} = \text{diag}(\alpha k I_3, k I_3)$ recovers the familiar linear impedance form.

This completes the geometrically consistent impedance controller derivation for SE(3) manipulation tasks.

D SWIVL Instantiation in SE(2)

While the SWIVL framework presented in Section ?? is formulated for general bimanual manipulation of k -DoF articulated objects in SE(3), our experimental evaluation in Section ?? focuses on SE(2) planar tasks with a single internal joint ($k = 1$). This design choice allows systematic study of force coupling and constraint satisfaction while controlling for the additional complexity of full 3D manipulation. Here we detail how the SE(3) formulation naturally reduces to SE(2) in this 1-DoF setting and how each component of SWIVL is instantiated.

D.1 SE(2) Geometric Formulation (1-DoF Object)

D.1.1 Configuration Space

In SE(2), poses are represented as $(x, y, \theta) \in \mathbb{R}^2 \times SO(2)$, where (x, y) is planar position and θ is orientation around the vertical z-axis. The homogeneous transformation matrix:

$$T \in SE(2) : \quad T = \begin{bmatrix} \cos \theta & -\sin \theta & x \\ \sin \theta & \cos \theta & y \\ 0 & 0 & 1 \end{bmatrix}$$

D.1.2 Twist Space

The Lie algebra $\mathfrak{se}(2)$ consists of planar twists:

$$\mathcal{V} = \begin{bmatrix} \omega_z \\ v_x \\ v_y \end{bmatrix} \in \mathbb{R}^3$$

where $\omega_z \in \mathbb{R}$ is angular velocity around z-axis and $(v_x, v_y) \in \mathbb{R}^2$ is linear velocity in the plane.

D.1.3 Screw Axis in SE(2)

For planar articulated objects with a

textbfsingle kinematic joint, the screw axis $\mathcal{S} = \begin{bmatrix} s_\omega \\ s_v \end{bmatrix}$ reduces to:

$$\mathcal{S} = \begin{bmatrix} s_\omega \\ s_{v,x} \\ s_{v,y} \end{bmatrix} \in \mathbb{R}^3$$

Joint Type Examples:

- **Revolute joint** (rotation around z-axis): $\mathcal{S} = \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix}$ (pure rotation)
- **Prismatic joint** (translation along direction \hat{d}): $\mathcal{S} = \begin{bmatrix} 0 \\ d_x \\ d_y \end{bmatrix}$ where (d_x, d_y) defines sliding direction
- **Fixed joint** (rigid transport): $\mathcal{S} = \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix}$ (no relative motion allowed)

In the general SE(3) formulation (Section ??), the object Jacobian $J_s(\mathbf{q}_{obj}) \in \mathbb{R}^{6 \times k}$ relates internal joint velocities to relative end-effector motion. In our SE(2), 1-DoF setting, this reduces to a single spatial screw axis $\mathcal{S} \in \mathbb{R}^3$ and the kinematic constraint becomes:

$${}^s\mathcal{V}_l - {}^s\mathcal{V}_r = \mathcal{S} \dot{q}_{obj}, \quad {}^s\mathcal{V}_i \in \mathbb{R}^3, \quad \mathcal{S} \in \mathbb{R}^3, \quad \dot{q}_{obj} \in \mathbb{R}.$$

For each grasp, the corresponding **body-frame joint screw axes** $\mathcal{B}_l, \mathcal{B}_r \in \mathbb{R}^3$ are obtained by transforming \mathcal{S} into the left and right end-effector frames via the SE(2) adjoint (Appendix ??). Because the object has a single joint and grasps remain fixed, these body-frame screw axes are **constant in time and independent of the joint configuration** q_{obj} :

$$\mathcal{B}_i = [Ad_{T_{ib}}]^b \mathcal{S}, \quad i \in \{l, r\}, \quad \mathcal{B}_i \text{ fixed for a given object.}$$

Thus, the object Jacobians in each body frame collapse to

$$J_l(q_{obj}) = \mathcal{B}_l \in \mathbb{R}^{3 \times 1}, \quad J_r(q_{obj}) = \mathcal{B}_r \in \mathbb{R}^{3 \times 1},$$

which no longer depend on q_{obj} .

D.1.4 Wrench Space

Forces and moments in SE(2) are dual to twists:

$$\mathcal{F} = \begin{bmatrix} m_z \\ f_x \\ f_y \end{bmatrix} \in \mathbb{R}^3$$

where m_z is moment around z-axis and (f_x, f_y) are planar forces.

D.1.5 Adjoint Representation

The adjoint transformation for frame changes in SE(2):

$$[Ad_T] = \begin{bmatrix} 1 & 0 & 0 \\ y & \cos \theta & -\sin \theta \\ -x & \sin \theta & \cos \theta \end{bmatrix} \in \mathbb{R}^{3 \times 3}$$

Twist transformation between frames:

$${}^s\mathcal{V} = [Ad_{T_{si}}]\mathcal{V}_i, \quad \mathcal{V}_i = [Ad_{T_{si}^{-1}}]{}^s\mathcal{V}$$

D.2 Reference Motion Field Generator in SE(2)

D.2.1 SE(2) Trajectory Smoothing

Given discrete waypoints $\{T_{si}^{des}[\tau]\}_{\tau=0}^H = \{(x[\tau], y[\tau], \theta[\tau])\}_{\tau=0}^H$ from the high-level planner at 5 Hz, we generate dense trajectories at 50 Hz (Low-Level Policy frequency).

Position Interpolation: Cubic spline interpolation through position waypoints $(x[\tau], y[\tau])$:

$$\begin{bmatrix} x(t) \\ y(t) \end{bmatrix} = \sum_{j=0}^3 a_j s^j, \quad s = \frac{t - t_k}{t_{k+1} - t_k}$$

where coefficients $\{a_j\}$ satisfy boundary conditions (positions and velocities at waypoints).

Orientation Interpolation: Circular interpolation on SO(2) ensuring shortest path:

$$\theta(t) = \theta_k + \text{wrap}(\theta_{k+1} - \theta_k) \cdot \phi(s)$$

where $\phi(s) = 3s^2 - 2s^3$ (cubic smoothing), and wrap ensures $|\theta_{k+1} - \theta_k| \leq \pi$.

D.2.2 Body Twist Computation

From smooth trajectory $T_{si}^{des}(t)$, compute desired body twist via time differentiation:

$$\mathcal{V}_i^{des}(t) = \begin{bmatrix} \dot{\theta}(t) \\ \dot{x}(t) \cos \theta(t) + \dot{y}(t) \sin \theta(t) \\ -\dot{x}(t) \sin \theta(t) + \dot{y}(t) \cos \theta(t) \end{bmatrix} \in \mathbb{R}^3$$

D.2.3 Stable Imitation Vector Field

Following Method Eq. (??), the reference twist combines imitation and stability components:

$$\mathcal{V}_i^{\text{ref}}(t, T_{sb_i}) = \text{Ad}_{T_{b_i d_i}} \mathcal{V}_i^{\text{des}}(t) + k_{p_i} \mathcal{E}_i$$

where Ad_T denotes the SE(2) adjoint transformation that maps twists between frames. Since the desired twist $\mathcal{V}_i^{\text{des}}(t)$ is computed in the desired frame $\{d_i\}$, we must transform it to the current body frame $\{b_i\}$ where the controller operates. The transformation $T_{b_i d_i} = T_{b_i s} T_{s d_i} = (T_{s b_i})^{-1} T_{s d_i}$ represents the relative transformation from the desired frame to the current body frame.

For SE(2), the adjoint transformation is:

$$\text{Ad}_{T_{b_i d_i}} = \begin{bmatrix} 1 & 0 & 0 \\ \Delta y & \cos \Delta \theta & -\sin \Delta \theta \\ -\Delta x & \sin \Delta \theta & \cos \Delta \theta \end{bmatrix} \in \mathbb{R}^{3 \times 3}$$

where $(\Delta x, \Delta y, \Delta \theta)$ are the components of $T_{b_i d_i}$.

The pose error term $\mathcal{E}_i \in \mathbb{R}^3$ is given by:

SE(2) Logarithm Map: For pose error $\Delta T = T_{si}^{des}(t^*)^{-1}T_{si}$:

$$[\log(\Delta T)]^\vee = \begin{bmatrix} \Delta\theta \\ \Delta x \cos \theta_{des} + \Delta y \sin \theta_{des} \\ -\Delta x \sin \theta_{des} + \Delta y \cos \theta_{des} \end{bmatrix}$$

where $\Delta x = x - x_{des}$, $\Delta y = y - y_{des}$, $\Delta\theta = \text{wrap}(\theta - \theta_{des})$.

D.3 Bulk-Internal Decomposition via Projection Operators in SE(2)

Following Method Section 3.2.4, SWIVL uses projection operators based on the learned metric tensor $G = \text{diag}(\alpha^2, 1, 1)$ to decompose twists into bulk and internal motion components. This approach enables independent impedance modulation for each component.

D.3.1 Metric Tensor and Inner Product

The SE(2) inner product on $\mathfrak{se}(2)$ is defined using the metric tensor $G \in \mathbb{R}^{3 \times 3}$:

$$\langle \mathcal{V}_1, \mathcal{V}_2 \rangle_G = \mathcal{V}_1^\top G \mathcal{V}_2 = \alpha^2 \omega_{1,z} \omega_{2,z} + v_{1,x} v_{2,x} + v_{1,y} v_{2,y}$$

where $\alpha \in \mathbb{R}^+$ is the **learnable characteristic length scale** (part of the RL action space) that weights rotational versus translational components. By learning α , the policy discovers task-appropriate notions of orthogonality for separating bulk versus internal motions.

D.3.2 Projection Operators

For each end-effector $i \in \{l, r\}$ with constant body-frame screw axis $\mathcal{B}_i \in \mathbb{R}^{3 \times 1}$ (1-DoF object), we construct orthogonal projection operators:

$$\begin{aligned} P_{i,\parallel} &= \mathcal{B}_i (\mathcal{B}_i^\top G \mathcal{B}_i)^{-1} \mathcal{B}_i^\top G \in \mathbb{R}^{3 \times 3} \quad (\text{project onto internal motion}), \\ P_{i,\perp} &= I_3 - P_{i,\parallel} \in \mathbb{R}^{3 \times 3} \quad (\text{project onto bulk motion}). \end{aligned}$$

These operators satisfy:

- $P_{i,\parallel}^\top G = G P_{i,\parallel}$ (G-self-adjoint for internal projection)
- $P_{i,\perp}^\top G = G P_{i,\perp}$ (G-self-adjoint for bulk projection)
- $P_{i,\parallel} + P_{i,\perp} = I_3$ (partition of identity)
- $P_{i,\parallel} P_{i,\perp} = 0$ (orthogonal subspaces under G-metric)

D.3.3 Twist Decomposition

Given reference body twist $\mathcal{V}_i^{\text{ref}} \in \mathbb{R}^3$, decompose into bulk and internal components:

$$\begin{aligned} \mathcal{V}_{i,\parallel}^{\text{ref}} &= P_{i,\parallel} \mathcal{V}_i^{\text{ref}} \in \mathbb{R}^3 \quad (\text{internal motion: range of } \mathcal{B}_i), \\ \mathcal{V}_{i,\perp}^{\text{ref}} &= P_{i,\perp} \mathcal{V}_i^{\text{ref}} \in \mathbb{R}^3 \quad (\text{bulk motion: orthogonal complement}). \end{aligned}$$

This decomposition satisfies G-orthogonality: $\langle \mathcal{V}_{i,\parallel}^{\text{ref}}, \mathcal{V}_{i,\perp}^{\text{ref}} \rangle_G = 0$.

Physical Interpretation:

- $\mathcal{V}_{i,\parallel}^{\text{ref}}$: Motion component aligned with the object's kinematic constraint (drives joint articulation)
- $\mathcal{V}_{i,\perp}^{\text{ref}}$: Motion component orthogonal to constraint (drives overall object transport/reorientation)

The Low-Level Policy receives both the full reference twists $\{\mathcal{V}_l^{\text{ref}}, \mathcal{V}_r^{\text{ref}}\}$ and their decomposed components, enabling it to learn task semantics from trajectory structure.

D.3.4 Wrench Decomposition

By duality, wrenches decompose using transposed projection operators. For measured wrench $\mathcal{F}_i \in \mathbb{R}^3$:

$$\begin{aligned}\mathcal{F}_{i,\parallel} &= P_{i,\parallel}^\top \mathcal{F}_i \in \mathbb{R}^3 \quad (\text{productive wrench}), \\ \mathcal{F}_{i,\perp} &= P_{i,\perp}^\top \mathcal{F}_i \in \mathbb{R}^3 \quad (\text{internal wrench}).\end{aligned}$$

Internal wrench $\mathcal{F}_{i,\perp}$ represents non-productive contact forces that stress the grasp without contributing to joint motion. The RL reward explicitly penalizes $\|\mathcal{F}_{i,\perp}\|_2^2$ to minimize harmful internal forces (Section 3.3.2).

Output Summary: The Reference Motion Field Generator produces at each timestep (50 Hz):

$$\{\mathcal{V}_l^{\text{ref}}, \mathcal{V}_r^{\text{ref}}, \mathcal{B}_l, \mathcal{B}_r\}$$

where individual reference twists $\mathcal{V}_i^{\text{ref}} = \text{Ad}_{T_{b_i d_i}} \mathcal{V}_i^{\text{des}} + k_{p_i} \mathcal{E}_i$ are computed from the stable imitation vector field (Eq. (??) in Method Section 3.2.2). The Low-Level Policy applies projection operators $P_{i,\parallel}$ and $P_{i,\perp}$ (parameterized by learned α) to decompose these into bulk and internal components. Constant screw axes $\mathcal{B}_l, \mathcal{B}_r$ encode the 1-DoF constraint structure.

D.4 Low-Level Policy Architecture for SE(2)

D.4.1 Observation Space

Following Method Section 3.2.3, the SE(2) policy observes:

1. Reference Twists (\mathbb{R}^6):

- $\mathcal{V}_l^{\text{ref}}, \mathcal{V}_r^{\text{ref}} \in \mathbb{R}^3$: Reference motions computed by the Reference Motion Field Generator (Layer 2) at the current time t and current end-effector poses T_{sb_l}, T_{sb_r} (6-dim)

2. Object Constraints (\mathbb{R}^6):

- $\mathcal{B}_l, \mathcal{B}_r \in \mathbb{R}^3$: Body-frame screw axes defining the object’s allowable internal motion directions at each end-effector (6-dim)

3. Wrench Feedback (\mathbb{R}^6):

- $\mathcal{F}_l, \mathcal{F}_r \in \mathbb{R}^3$: Body wrenches measured at the end-effectors (6-dim)

4. Proprioception (\mathbb{R}^{12}):

- End-effector poses: $T_{sb_l}, T_{sb_r} \in \text{SE}(2)$, represented as $(x_i, y_i, \theta_i) \in \mathbb{R}^3 \times 2$ (6-dim)
- End-effector body twists: $\mathcal{V}_l, \mathcal{V}_r \in \mathbb{R}^3$, represented as $(\omega_{z,i}, v_{x,i}, v_{y,i}) \in \mathbb{R}^3 \times 2$ (6-dim)

Total: $o_t \in \mathbb{R}^{30}$ (6+6+6+12=30-dim)

Note: The policy receives the core physical observations that directly parameterize the impedance controller. The policy network internally computes bulk-internal decomposition of reference twists and measured wrenches using projection operators $P_{i,\parallel}$ and $P_{i,\perp}$ parameterized by the learned metric tensor $G = \text{diag}(\alpha^2, 1, 1)$.

D.4.2 Action Space

Following the SE(3) formulation in Method Section 3.2.3, the SE(2) policy outputs impedance modulation variables adapted for the planar, 1-DoF setting:

Action Space:

$$a_t = (d_{l,\parallel}, d_{r,\parallel}, d_{l,\perp}, d_{r,\perp}, k_{p_l}, k_{p_r}, \alpha) \in \mathbb{R}^7$$

where:

- $d_{l,\parallel}, d_{r,\parallel} \in \mathbb{R}^+$: Per-arm damping coefficients for internal motion (parallel to screw axis)
- $d_{l,\perp}, d_{r,\perp} \in \mathbb{R}^+$: Per-arm damping coefficients for bulk motion (orthogonal to screw axis)
- $k_{p_l}, k_{p_r} \in \mathbb{R}^+$: Per-arm stiffness gains for the stability term $k_{p_i} \mathcal{E}_i$ in the reference vector field (Eq. (??))
- $\alpha \in \mathbb{R}^+$: Learnable characteristic length scale that defines the metric tensor $G = \text{diag}(\alpha^2, 1, 1)$ for the SE(2) inner product, enabling task-appropriate orthogonal decomposition of twists and wrenches

Note: This maintains the full SE(3) action space structure $(d_{l,\parallel}, d_{r,\parallel}, d_{l,\perp}, d_{r,\perp}, k_{p_l}, k_{p_r}, \alpha) \in \mathbb{R}^7$, preserving the ability to independently modulate compliance for each arm. Gripper commands are omitted as grippers remain closed throughout episodes.

D.4.3 SE(2) Screw-decomposed Controller

Following the SE(3) controller formulation in Method Section 3.2.4 (Eq. (??)–(??)), the impedance variables parameterize an SE(2) twist-driven impedance controller:

Orthogonal Projection Operators:

Using the metric tensor $G = \text{diag}(\alpha^2, 1, 1) \in \mathbb{R}^{3 \times 3}$ and body-frame screw axes $\mathcal{B}_i \in \mathbb{R}^{3 \times 1}$:

$$P_{i,\parallel} = \mathcal{B}_i (\mathcal{B}_i^\top G \mathcal{B}_i)^{-1} \mathcal{B}_i^\top G \in \mathbb{R}^{3 \times 3},$$

$$P_{i,\perp} = I_3 - P_{i,\parallel} \in \mathbb{R}^{3 \times 3}$$

where $P_{i,\parallel}$ projects onto internal motion (range of \mathcal{B}_i) and $P_{i,\perp}$ projects onto bulk motion (orthogonal complement).

Damping Matrix Construction:

$$K_{d_i} = G(P_{i,\parallel} d_{i,\parallel} + P_{i,\perp} d_{i,\perp}) \in \mathbb{R}^{3 \times 3}$$

where $d_{i,\parallel}$ and $d_{i,\perp}$ denote the per-arm damping coefficients ($d_{l,\parallel}, d_{l,\perp}$ for left arm, $d_{r,\parallel}, d_{r,\perp}$ for right arm). This allows independent damping modulation for each arm: $d_{i,\parallel}$ controls compliance along the object’s kinematic constraint (internal motion), while $d_{i,\perp}$ controls compliance orthogonal to it (bulk motion).

Commanded Wrench:

$$\mathcal{F}_{\text{cmd},i} = K_{d_i}(\mathcal{V}_i^{\text{ref}} - \mathcal{V}_i) + \mu_{b,i} \in \mathbb{R}^3$$

where $\mathcal{V}_i^{\text{ref}} = \text{Ad}_{T_{b_i d_i}} \mathcal{V}_i^{\text{des}} + k_{p_i} \mathcal{E}_i$ is the reference twist from Layer 2 (Eq. (??)), $\mu_{b,i} = C_{b,i}(q_i, \dot{q}_i) \dot{q}_i$ accounts for Coriolis/centrifugal terms, and gravity $\gamma_{b,i} = 0$ in planar settings.

Execution:

In our SE(2) simulation environment with direct body wrench control, the commanded wrenches $\mathcal{F}_{\text{cmd},i} \in \mathbb{R}^3$ are directly applied as control inputs to the end-effectors. The simulation environment integrates these wrench commands to update end-effector poses, consistent with the impedance-based control framework.

Kinematic Constraint Satisfaction:

By construction, the projection-based structure ensures the holonomic constraint is satisfied. The reference twists $\mathcal{V}_i^{\text{ref}}$ already respect the constraint through the Reference Motion Field Generator, and the damping matrix K_{d_i} preserves the constraint subspace through its construction from $P_{i,\parallel}$ and $P_{i,\perp}$.

D.5 Controller Implementation

D.5.1 Direct End-Effector Control

In the SE(2) simulation environment, we use direct end-effector wrench control without intermediate joint-space representations. The commanded body wrenches $\mathcal{F}_{\text{cmd},i} \in \mathbb{R}^3$ (computed from the impedance controller above) are directly applied as control inputs to the end-effectors. The simulation environment integrates these wrench commands through forward dynamics to update end-effector poses at each control step. This wrench-based control scheme is appropriate for the planar manipulation tasks and allows us to focus on the core challenges of force coupling and constraint satisfaction while maintaining full consistency with the impedance control framework in Method Section 3.2.4.

Control Frequency: 50 Hz (policy and controller run at the same frequency).

D.6 Reward Function in SE(2)

The reward function for SE(2) specializes the general formulation from Method Section 3.3.2, adapted for planar manipulation with the learned metric tensor $G = \text{diag}(\alpha^2, 1, 1)$:

$$r_t = r_{\text{track}} + r_{\text{safety}} + r_{\text{reg}}$$

D.6.1 Motion Tracking Reward

Following Method Eq. (??), the tracking reward uses the G-metric to measure velocity error:

$$r_{\text{track}} = -w_{\text{track}} \sum_{i \in \{l, r\}} \|\mathcal{V}_i - \mathcal{V}_i^{\text{ref}}\|_G^2 = -w_{\text{track}} \sum_{i \in \{l, r\}} (\mathcal{V}_i - \mathcal{V}_i^{\text{ref}})^T G (\mathcal{V}_i - \mathcal{V}_i^{\text{ref}})$$

Expanding with $G = \text{diag}(\alpha^2, 1, 1)$ and $\mathcal{V}_i = [\omega_{z,i}, v_{x,i}, v_{y,i}]^T \in \mathbb{R}^3$:

$$\|\mathcal{V}_i - \mathcal{V}_i^{\text{ref}}\|_G^2 = \alpha^2 (\omega_{z,i} - \omega_{z,i}^{\text{ref}})^2 + (v_{x,i} - v_{x,i}^{\text{ref}})^2 + (v_{y,i} - v_{y,i}^{\text{ref}})^2$$

This ensures tracking error is measured consistently with the impedance control framework, with adaptive weighting between rotational and translational components via the learned parameter α .

D.6.2 Safety Reward

Following Method Eq. (??), the safety reward minimizes internal wrenches—wrench components orthogonal to the object’s allowable motion direction:

$$r_{\text{safety}} = -w_{\text{int}} \sum_{i \in \{l, r\}} \|\mathcal{F}_{i,\perp}\|_2^2$$

Wrench Decomposition via Projection Operators. Consistent with Method Section 3.3.2 and the twist decomposition in Layer 4, wrenches decompose using the transpose of twist projection operators. For measured wrench $\mathcal{F}_i = [m_{z,i}, f_{x,i}, f_{y,i}]^T \in \mathbb{R}^3$:

$$\mathcal{F}_{i,\parallel} = P_{i,\parallel}^T \mathcal{F}_i, \quad \mathcal{F}_{i,\perp} = P_{i,\perp}^T \mathcal{F}_i = (I_3 - P_{i,\parallel})^T \mathcal{F}_i$$

where $P_{i,\parallel} = \mathcal{B}_i (\mathcal{B}_i^T G \mathcal{B}_i)^{-1} \mathcal{B}_i^T G$ and $P_{i,\perp} = I_3 - P_{i,\parallel}$ are the SE(2) projection operators defined in Section ??3.2.

This decomposition exploits the duality between twist and wrench spaces under the reciprocal product (virtual power). For any $\mathcal{V} \in \text{range}(P_{i,\perp})$, we have $\mathcal{V} = P_{i,\perp} \mathcal{V}'$, and:

$$\mathcal{F}_{i,\parallel}^T \mathcal{V} = (P_{i,\parallel}^T \mathcal{F}_i)^T (P_{i,\perp} \mathcal{V}') = \mathcal{F}_i^T P_{i,\parallel} P_{i,\perp} \mathcal{V}' = 0$$

where the last equality follows from $P_{i,\parallel}P_{i,\perp} = 0$ (orthogonal projections). Similarly, $\mathcal{F}_{i,\perp}^T \mathcal{V} = 0$ for all $\mathcal{V} \in \text{range}(P_{i,\parallel})$.

Physical Interpretation:

- $\mathcal{F}_{i,\parallel}$: Productive wrench that performs work along the object’s internal degree of freedom (joint articulation)
- $\mathcal{F}_{i,\perp}$: Internal wrench orthogonal to the kinematic constraint that:
 - Does not contribute to desired object motion (zero virtual power along $\text{range}(P_{i,\parallel})$)
 - Arises from coordination errors between the two arms
 - Represents constraint forces (bearing loads, friction, contact stresses) unrelated to joint actuation
 - Increases unnecessary contact stress and grasp instability
 - Wastes energy and risks hardware damage

By penalizing $\|\mathcal{F}_{i,\perp}\|_2^2$, the policy learns to minimize non-productive forces while maintaining necessary productive forces for manipulation.

D.6.3 Regularization Reward

Following Method Eq. (??), the regularization reward encourages smooth motion:

$$r_{\text{reg}} = -w_{\text{reg}} \sum_{i \in \{l, r\}} \|\dot{\mathcal{V}}_i\|_2^2$$

where $\dot{\mathcal{V}}_i = [\ddot{\theta}_i, \dot{v}_{x,i}, \dot{v}_{y,i}]^T \in \mathbb{R}^3$ is the SE(2) twist acceleration. This reduces energy consumption, joint jerkiness, and Cartesian jerkiness, promoting natural and efficient movements.

D.6.4 Termination Conditions

Following Method Section 3.3.2, grasp stability is enforced through early termination rather than reward penalties. Episodes terminate immediately (task failure) when grasp drift exceeds safety thresholds:

$$\text{Terminate if: } \exists i \in \{l, r\} \text{ such that } \left\| [\log((T_{\text{grip},i}^{\text{init}})^{-1} T_{\text{grip},i})]^\vee \right\|_2 > d_{\text{max}}$$

where $T_{\text{grip},i}^{\text{init}}$ is the initial grasp pose, $T_{\text{grip},i}$ is the current grasp pose, and d_{max} is the maximum allowable drift threshold. For SE(2), the logarithm map computes planar geodesic distance:

$$[\log(\Delta T)]^\vee = \begin{bmatrix} \Delta\theta \\ \Delta x \cos \theta_{\text{init}} + \Delta y \sin \theta_{\text{init}} \\ -\Delta x \sin \theta_{\text{init}} + \Delta y \cos \theta_{\text{init}} \end{bmatrix}$$

This ensures grasp stability throughout manipulation without explicit reward shaping.

D.7 SE(2) → SE(3) Extension Path

The SE(2) experimental validation serves as a controlled study of SWIVL’s core principles. Extension to SE(3) is straightforward:

Mathematical Framework:

- All SE(3) formulations in Section 3 directly apply
- Twist space: $\mathfrak{se}(2) \subset \mathfrak{se}(3)$ (3-dim → 6-dim)
- Metric tensor: $G = \text{diag}(\alpha^2, 1, 1) \in \mathbb{R}^{3 \times 3} \rightarrow G = \text{diag}(\alpha^2 I_3, I_3) \in \mathbb{R}^{6 \times 6}$ (scalar rotation → 3D rotation weighting)

- Action space: $(d_{l,\parallel}, d_{r,\parallel}, d_{l,\perp}, d_{r,\perp}, k_{pl}, k_{pr}, \alpha) \in \mathbb{R}^7$ (same structure for both SE(2) and SE(3))
- Object Jacobian: $\mathcal{B}_i \in \mathbb{R}^{3 \times 1} \rightarrow J_i \in \mathbb{R}^{6 \times k}$ (single screw axis \rightarrow multi-DoF Jacobian)
- Network architecture scales with input/output dimensions

Engineering Requirements:

- 6-axis F/T sensors (already available on Franka FR3)
- 7-DoF differential IK controller (standard in Franka SDK) for joint-space control
- SE(3) trajectory smoothing (geodesic interpolation, Appendix C)
- Robot proprioception (end-effector poses and twists, object tracking, gripper feedback)

Validation Strategy:

1. SE(2) experiments (current work): Isolate force coupling and constraint satisfaction with impedance-based control
2. SE(3) simulation: Validate 6-DoF extension with gravity, collisions, and per-arm impedance modulation
3. Real-world deployment: Franka FR3 dual-arm setup

The SE(2) results provide strong evidence that SWIVL’s principles—learned impedance variables, projection-based motion decomposition, FiLM-based object conditioning, and screw-decomposed control—will transfer to full SE(3) manipulation.

E Learning Settings for SE(2) Implementation

This appendix provides comprehensive implementation details for the SWIVL Low-Level Policy in the SE(2) planar manipulation setting, including network architecture, training configuration, and simulation environment specifications.

E.1 Network Architecture

The Low-Level Policy $\pi_\theta : \mathcal{O} \rightarrow \Delta(\mathcal{A})$ is implemented as a neural network with object-conditioned multi-stream architecture, employing Feature-wise Linear Modulation (FiLM) to inject object geometric structure throughout all feature processing stages.

E.1.1 Input and Output Specifications

Observation Space: $o_t \in \mathbb{R}^{30}$ (SE(2) planar setting)

- **Reference Twists** (6-dim): $\mathcal{V}_l^{\text{ref}}, \mathcal{V}_r^{\text{ref}} \in \mathbb{R}^3$
- **Object Constraints** (6-dim): Body-frame screw axes $\mathcal{B}_l, \mathcal{B}_r \in \mathbb{R}^3$
- **Wrench Feedback** (6-dim): Body wrenches $\mathcal{F}_l, \mathcal{F}_r \in \mathbb{R}^3$
- **Proprioception** (12-dim): End-effector poses $(x_i, y_i, \theta_i) \in \mathbb{R}^3 \times 2$ (6-dim), body twists $\mathcal{V}_i = (\omega_{z,i}, v_{x,i}, v_{y,i}) \in \mathbb{R}^3 \times 2$ (6-dim)

Note: This corresponds to the SE(2) observation space detailed in Method Section 3.2.3 and Appendix ??.

Input Normalization: Each modality is normalized before being fed to its respective encoder to ensure balanced gradients and stable learning:

- **Reference Twists:** Twist components clipped to $[-v_{\max}, v_{\max}]$ then scaled by s_{ref}
- **Object Constraints:** Screw axes are already unit-normalized
- **Wrench Feedback:** Running normalization with exponential moving average: $\hat{\mathcal{F}} = (\mathcal{F} - \mu_{\mathcal{F}}) / (\sigma_{\mathcal{F}} + \epsilon)$ where $\mu_{\mathcal{F}}, \sigma_{\mathcal{F}}$ are updated online with decay α_{wrench}

- **Proprioception:** Poses clipped to workspace bounds $[-p_{\max}, p_{\max}] \times [-p_{\max}, p_{\max}] \times [-\pi, \pi]$ then scaled by s_{pose} ; body twists clipped to $[-\dot{p}_{\max}, \dot{p}_{\max}]$ then scaled by s_{vel}

Action Space: $a_t \in \mathbb{R}^7$ (SE(2) planar setting)

- Per-arm damping coefficients for internal motion: $d_{l,\parallel}, d_{r,\parallel} \in \mathbb{R}$
- Per-arm damping coefficients for bulk motion: $d_{l,\perp}, d_{r,\perp} \in \mathbb{R}$
- Stiffness gains: $k_{p_l}, k_{p_r} \in \mathbb{R}$
- Characteristic length scale: $\alpha \in \mathbb{R}$

These impedance variables parameterize the SE(2) screw-decomposed controller as detailed in Appendix ??.

E.1.2 Multi-Stream Encoder Architecture

Object Structure Encoder (Conditioning Generator):

The object encoder processes kinematic constraint information and generates a shared embedding that is then projected to stream-specific FiLM parameters:

$$\begin{aligned} h_{obj}^{(1)} &= \text{SiLU}(\text{LayerNorm}(W_{obj}^{(1)}x_{obj} + b_{obj}^{(1)})) \in \mathbb{R}^{64}, \\ e_{obj} &= \text{SiLU}(\text{LayerNorm}(W_{obj}^{(2)}h_{obj}^{(1)} + b_{obj}^{(2)})) \in \mathbb{R}^{128} \end{aligned}$$

where $x_{obj} \in \mathbb{R}^6$ contains body-frame screw axes $\mathcal{B}_l, \mathcal{B}_r \in \mathbb{R}^3$. The shared object embedding e_{obj} is projected to layer-specific FiLM parameters via lightweight affine transformations:

$$[\gamma_s^{(l)}, \beta_s^{(l)}] = W_{FiLM,s}^{(l)}e_{obj} + b_{FiLM,s}^{(l)} \in \mathbb{R}^{d_s} \times \mathbb{R}^{d_s}$$

where $s \in \{\text{ref}, \text{wrench}, \text{proprio}, \text{fuse}\}$ denotes the stream, l is the layer index, and d_s is the feature dimension of that layer. This ensures dimensional compatibility between FiLM parameters and target features.

Reference Motion Encoder:

Processes reference twists with object-aware feature transformation:

$$\begin{aligned} h_{ref}^{(0)} &= \text{SiLU}(\text{LayerNorm}(W_{ref}^{(1)}x_{ref} + b_{ref}^{(1)})) \in \mathbb{R}^{128}, \\ h_{ref} &= \text{FiLM}(\text{LayerNorm}(W_{ref}^{(2)}h_{ref}^{(0)} + b_{ref}^{(2)}); \gamma_{ref}^{(1)}, \beta_{ref}^{(1)}) \in \mathbb{R}^{128} \end{aligned}$$

where $x_{ref} \in \mathbb{R}^6$ contains reference twists $\mathcal{V}_l^{\text{ref}}, \mathcal{V}_r^{\text{ref}}$. The policy network internally computes bulk-internal decomposition using projection operators.

Wrench Encoder:

Processes force-torque sensor feedback with object-aware feature transformation:

$$\begin{aligned} h_{wrench}^{(0)} &= \text{SiLU}(\text{LayerNorm}(W_{wrench}^{(1)}x_{wrench} + b_{wrench}^{(1)})) \in \mathbb{R}^{128}, \\ h_{wrench} &= \text{FiLM}(\text{LayerNorm}(W_{wrench}^{(2)}h_{wrench}^{(0)} + b_{wrench}^{(2)}); \gamma_{wrench}^{(1)}, \beta_{wrench}^{(1)}) \in \mathbb{R}^{128} \end{aligned}$$

where $x_{wrench} \in \mathbb{R}^6$ contains body wrenches $\mathcal{F}_l, \mathcal{F}_r$. The policy network internally computes productive-internal wrench decomposition using projection operators.

Proprioception Encoder:

Processes robot state information with higher capacity for rich state representation:

$$\begin{aligned} h_{proprio}^{(0)} &= \text{SiLU}(\text{LayerNorm}(W_{proprio}^{(1)}x_{proprio} + b_{proprio}^{(1)})) \in \mathbb{R}^{128}, \\ h_{proprio} &= \text{FiLM}(\text{LayerNorm}(W_{proprio}^{(2)}h_{proprio}^{(0)} + b_{proprio}^{(2)}); \gamma_{proprio}^{(1)}, \beta_{proprio}^{(1)}) \in \mathbb{R}^{128} \end{aligned}$$

where $x_{proprio} \in \mathbb{R}^{12}$ contains end-effector poses and velocities.

E.1.3 Multi-Modal Fusion and Policy Head

Feature Fusion:

Encoded features from all streams are concatenated and fused through object-conditioned layers:

$$\begin{aligned}\tilde{h} &= [h_{ref}, h_{wrench}, h_{proprio}] \in \mathbb{R}^{384}, \\ h_{fused}^{(1)} &= \text{SiLU}(\text{FiLM}(W_{fuse}^{(1)} \tilde{h} + b_{fuse}^{(1)}; \gamma_{fuse}^{(1)}, \beta_{fuse}^{(1)})) \in \mathbb{R}^{256}, \\ h_{context} &= \text{FiLM}(W_{fuse}^{(2)} h_{fused}^{(1)} + b_{fuse}^{(2)}; \gamma_{fuse}^{(2)}, \beta_{fuse}^{(2)}) \in \mathbb{R}^{256}\end{aligned}$$

Action Decoder:

The fused context is decoded into action distribution parameters:

$$\begin{aligned}h_{action} &= \text{SiLU}(W_{action}^{(1)} h_{context} + b_{action}^{(1)}) \in \mathbb{R}^{128}, \\ [\mu, \log \sigma] &= W_{action}^{(2)} h_{action} + b_{action}^{(2)} \in \mathbb{R}^{14}\end{aligned}$$

where $\mu \in \mathbb{R}^7$ and $\log \sigma \in \mathbb{R}^7$ parameterize a diagonal Gaussian action distribution $\pi_\theta(a|o) = \mathcal{N}(a; \mu(o), \text{diag}(\exp(\log \sigma(o))))$ for the 7-dimensional SE(2) impedance action space $(d_{l,\parallel}, d_{r,\parallel}, d_{l,\perp}, d_{r,\perp}, k_{p_l}, k_{p_r}, \alpha)$. The log standard deviation is clipped to $[\log(0.01), \log(10)]$ to prevent numerical instability.

Positivity Constraint: Since all impedance parameters must be strictly positive ($d_{i,\parallel}, d_{i,\perp}, k_{p_i}, \alpha \in \mathbb{R}^+$) for physical stability, the sampled actions from the Gaussian distribution are passed through a Softplus activation function:

$$a_{\text{final}} = \text{Softplus}(a_{\text{sampled}}) = \log(1 + \exp(a_{\text{sampled}}))$$

This ensures $a_{\text{final}} > 0$ for all components while maintaining differentiability for policy gradient updates. The Softplus function provides smooth gradients near zero, avoiding the non-differentiability issues of ReLU or absolute value, and naturally prevents negative damping or stiffness coefficients that would destabilize the impedance controller.

E.1.4 Architectural Components

FiLM Layer: Feature-wise Linear Modulation applies affine transformation based on object conditioning:

$$\text{FiLM}(h; \gamma^{(obj)}, \beta^{(obj)}) = \gamma^{(obj)} \odot h + \beta^{(obj)}$$

where $\gamma^{(obj)}, \beta^{(obj)} \in \mathbb{R}^d$ are stream- and layer-specific parameters projected from the shared object embedding e_{obj} and modulate features element-wise. This enables object-specific feature transformation throughout the network while maintaining dimensional compatibility.

Activation: SiLU (Swish) for smooth gradients: $\text{SiLU}(x) = x \cdot \sigma(x)$

Normalization: LayerNorm with $\epsilon = 10^{-5}$: $\text{LayerNorm}(x) = \frac{x - \mu}{\sqrt{\sigma^2 + \epsilon}} \odot \gamma_{LN} + \beta_{LN}$

Note: The learnable parameters γ_{LN} and β_{LN} in LayerNorm are distinct from the FiLM parameters $\gamma^{(obj)}$ and $\beta^{(obj)}$.

E.2 Training Configuration

E.2.1 Reinforcement Learning Algorithm

We train the Low-Level Policy using Proximal Policy Optimization (PPO) with clipped objective:

$$L^{CLIP}(\theta) = \mathbb{E}_t \left[\min \left(r_t(\theta) \hat{A}_t, \text{clip}(r_t(\theta), 1 - \epsilon, 1 + \epsilon) \hat{A}_t \right) \right]$$

where $r_t(\theta) = \frac{\pi_\theta(a_t|o_t)}{\pi_{\theta_{old}}(a_t|o_t)}$ is the probability ratio and advantages are computed via Generalized Advantage Estimation (GAE).

E.2.2 Hyperparameters

Optimization:

- **Optimizer:** Adam with $\beta_1 = 0.9$, $\beta_2 = 0.999$
- **Learning rate:** 3×10^{-4} with linear decay over training
- **Gradient clipping:** Maximum norm 0.5
- **Weight decay:** 10^{-4}

PPO Configuration:

- **Rollout horizon:** 256 steps per worker
- **Batch size:** 4096 transitions per iteration
- **Mini-batch size:** 256 transitions per update
- **Update epochs:** 10 epochs per batch
- **Clip range:** $\epsilon = 0.2$
- **Value loss coefficient:** 0.5
- **Entropy coefficient:** $0.01 \rightarrow 0.001$ (linear annealing)

GAE Configuration:

- **Discount factor:** $\gamma = 0.99$
- **GAE lambda:** $\lambda = 0.95$

Policy Distribution:

- **Type:** Diagonal Gaussian with state-dependent standard deviation
- **Initial log std:** $\log \sigma_0 = -0.5$
- **Action bounds:** $[-10, 10]$ for raw Gaussian samples before Softplus transformation (ensuring final positive actions in practical range $[\text{Softplus}(-10), \text{Softplus}(10)] \approx [4.5 \times 10^{-5}, 10.00]$)

E.2.3 Initialization Strategy

Linear Layers: Xavier initialization with fan-averaging:

$$W \sim \mathcal{U}\left(-\sqrt{\frac{6}{n_{in} + n_{out}}}, \sqrt{\frac{6}{n_{in} + n_{out}}}\right)$$

FiLM Generators: Initialize scale parameters near identity and shift parameters near zero to ensure stable initial conditioning:

$$W_\gamma \sim \mathcal{N}(0, 0.01^2), \quad b_\gamma = 1, \quad W_\beta \sim \mathcal{N}(0, 0.01^2), \quad b_\beta = 0$$

This ensures that FiLM conditioning initially approximates identity transformation, preventing disruption of gradient flow during early training.

Action Head: Small-scale initialization to encourage near-zero initial actions:

$$W_{action}^{(2)} \sim \mathcal{N}(0, 0.01^2), \quad b_{action}^{(2)} = 0$$

F SE(2) Simulation Environment Settings

This appendix specifies the simulation environment configuration, task specifications, and evaluation protocol for the SE(2) planar manipulation experiments.

F.1 Simulation Environment

Workspace Configuration:

- Total space: 512×512 pixels
- Effective space: 501×501 pixels
- Walls: Located at $(5, 5) \sim (506, 506)$ with 2-pixel thickness

Physics and Control:

- Physics timestep: 0.002s (500 Hz simulation)
- Control frequency: 50 Hz (policy and controller)
- Robot platform: Dual end-effectors with direct wrench control
- Force-torque sensing: 3-axis (m_z, f_x, f_y) per end-effector at 50 Hz

Object Dataset:

- Total objects: 9 (3 joint types \times 3 variants per type)
- Joint types: Fixed (rigid transport), Revolute (angular articulation), Prismatic (linear articulation)
- Material: Rigid body with Coulomb friction ($\mu = 0.6\text{--}0.9$)
- Contact model: Soft contact with 5000 N/m stiffness

F.2 Task Specifications

Task Objective: Manipulate an articulated object from a randomized initial configuration $({}^sT_o^{init}, q_{obj}^{init})$ to a fixed goal configuration $({}^sT_o^{goal}, q_{obj}^{goal})$ through coordinated bimanual control.

Object Configuration:

- Object frame pose: ${}^sT_o \in SE(2)$ (3 DoF: position (x_o, y_o) , orientation θ_o)
- Joint state: $q_{obj} \in \mathbb{R}$ (1 DoF: angle for revolute, extension for prismatic)

Task Structure:

- Objects: 9 total, each with one fixed goal configuration
- Episode initialization: Object spawns at random configuration within safe workspace region
- Episode duration: 8 seconds (400 timesteps at 50 Hz)
- Visual feedback: Goal configuration marked with distinct pixel color in 512×512 workspace image

Joint Type Categories:

1. **Fixed Joint:** q_{obj} constant; manipulate sT_o to ${}^sT_o^{goal}$
2. **Revolute Joint:** Manipulate both sT_o and q_{obj} (rotation) to goal
3. **Prismatic Joint:** Manipulate both sT_o and q_{obj} (extension) to goal

Success Criteria (unified):

- Object position error: $\|p_o - p_o^{goal}\|_2 < 10$ pixels
- Object orientation error: $|\theta_o - \theta_o^{goal}| < 5$
- Joint error: $|q_{obj} - q_{obj}^{goal}| < 5$ (revolute) or < 5 pixels (prismatic)
- Grasp stability: Maintained throughout episode (no drift exceeding d_{\max})

Termination Conditions:

- **Success:** All success criteria satisfied at episode end
- **Failure (grasp loss):** Grasp drift exceeds threshold: $\left\| \left[\log \left((T_{\text{grip},i}^{\text{init}})^{-1} T_{\text{grip},i} \right) \right]^{\vee} \right\|_2 > d_{\text{max}}$ for any $i \in \{l, r\}$
- **Timeout:** Maximum episode duration (8 seconds / 400 timesteps) reached without success

F.3 Training Randomization

To improve policy robustness, we apply the following randomization during training:

- Object spawn: Uniform random within safe workspace region (with wall margin)
- Object orientation: Uniform random $\theta_o \in [0, 2\pi)$
- Joint configuration: Uniform random within joint limits
- End-effector positions: Random grasp configurations on object

F.4 Evaluation Protocol

Test Configuration:

- Objects: 9 (3 per joint type)
- Goal configurations: 1 fixed goal per object
- Trials: 100 rollouts per object with randomized initial configurations

Primary Metrics:

1. **Success Rate (%)**: Binary task completion metric

$$\text{Success Rate} = \frac{1}{N} \sum_{i=1}^N \mathbb{I}[\text{task}_i \text{ succeeded}] \quad (82)$$

Confidence intervals computed via bootstrap (10,000 resamples).

2. **Constraint Violation (pixels/s)**: Time-averaged kinematic constraint violation

$$\text{CViol} = \frac{1}{T} \sum_{t=1}^T \left\| {}^s\mathcal{V}_l(t) - {}^s\mathcal{V}_r(t) - \mathcal{S}\dot{q}_{obj}(t) \right\|_2 \quad (83)$$

where \mathcal{S} is the spatial screw axis and $T = 400$ timesteps.

3. **Internal Force (N)**: Time-averaged non-productive contact forces

$$F_{\text{int}} = \frac{1}{T} \sum_{t=1}^T \sum_{i \in \{l, r\}} \left\| \mathcal{F}_{i,\perp}(t) \right\|_2 \quad (84)$$

where $\mathcal{F}_{i,\perp}$ is the wrench component orthogonal to screw axis.

Secondary Metrics:

- **Peak Contact Force (N)**: $F_{\text{peak}} = \max_{t,i} \left\| \mathcal{F}_i(t) \right\|_2$
- **Tracking RMSE (pixels)**: End-effector tracking error

$$\text{RMSE} = \sqrt{\frac{1}{2T} \sum_{t=1}^T \sum_{i \in \{l, r\}} d_{SE(2)}^2(T_{si}^{\text{actual}}(t), T_{si}^{\text{des}}(t))} \quad (85)$$

- **Motion Smoothness (pixels/s³)**: Jerk magnitude

Statistical Testing:

- Method: Welch's t-test ($p < 0.05$) with Bonferroni correction for multiple comparisons
- Effect size (Cohen's d): negligible ($|d| < 0.2$), small ($0.2 \leq |d| < 0.5$), medium ($0.5 \leq |d| < 0.8$), large ($|d| \geq 0.8$)

G Pure Imitation Learning Baseline (OWIL)

G.1 Overview

extbfOWIL (Object-Wrench conditioned Imitation Learning) is a pure imitation learning baseline designed to evaluate the necessity of SWIVL’s explicit kinematic decomposition and constraint-based action space. OWIL learns to satisfy kinematic constraints **implicitly** through behavior cloning on expert demonstrations, without explicit bulk–internal motion decomposition or kinematic-constrained action parameterization.

extbfPurpose: Answer Q1 by comparing SWIVL’s physics-aware approach against state-of-the-art imitation learning that receives the same object and wrench information but relies on implicit constraint learning.

G.2 Architecture (SE(2) Version)

G.2.1 Observation Space

OWIL receives similar object and wrench information as SWIVL’s low-level policy, but **excludes reference motion inputs**:

1. **Object Constraints** (6-dim):
 - Body-frame screw axes: $\mathcal{B}_l, \mathcal{B}_r \in \mathbb{R}^3$ (6-dim)
2. **Wrench Feedback** (6-dim):
 - Body wrenches: $\mathcal{F}_l, \mathcal{F}_r \in \mathbb{R}^3 ((m_z, f_x, f_y) \times 2 \text{ arms} = 6\text{-dim})$
 - Filtered with exponential smoothing ($\alpha=0.3$) to reduce sensor noise
3. **Proprioception** (12-dim):
 - End-effector poses: $(x_i, y_i, \theta_i) \in \mathbb{R}^3 \times 2 \text{ arms}$ (6-dim)
 - End-effector body twists: $\mathcal{V}_i = (\omega_{z,i}, v_{x,i}, v_{y,i}) \in \mathbb{R}^3 \times 2 \text{ arms}$ (6-dim)

Total observation dimension: $o_t \in \mathbb{R}^{24}$

Key Difference from SWIVL: No reference twist inputs $\{\mathcal{V}_l^{\text{ref}}, \mathcal{V}_r^{\text{ref}}\}$ (6-dim excluded). OWIL learns manipulation directly from demonstrations without explicit reference motion guidance from the vector field.

G.2.2 Action Space

OWIL uses a **direct arm twist action space** in SE(2) without kinematic constraint parameterization:

$$a_t = ([\mathcal{V}_l], [\mathcal{V}_r]) \in \mathfrak{se}(2) \times \mathfrak{se}(2)$$

Policy output: $(\mathcal{V}_l, \mathcal{V}_r) \in \mathbb{R}^3 \times \mathbb{R}^3$ (6-dim)

- Left arm twist: $(\omega_{z,l}, v_{x,l}, v_{y,l})$ (3-dim)
- Right arm twist: $(\omega_{z,r}, v_{x,r}, v_{y,r})$ (3-dim)

Constraint Handling: The kinematic constraint $\mathcal{V}_l - \mathcal{V}_r = \mathcal{S}\dot{q}_{obj}$ is **not enforced structurally**. OWIL must learn to satisfy constraints implicitly through demonstrations.

G.2.3 Network Architecture

Encoders:

- **Object encoder (FiLM conditioning):** $[64, 64] \rightarrow 128\text{-dim} \rightarrow \text{generates } (\gamma, \beta) \in \mathbb{R}^{64} \times \mathbb{R}^{64}$
- **Wrench encoder:** $[64, 64] \rightarrow 64\text{-dim}$
- **Proprioception encoder:** $[128, 128] \rightarrow 128\text{-dim}$

Policy Head:

- Fusion layer: Concatenate [wrench(64) + proprio(128)] \rightarrow 192-dim
- FiLM modulation: $\text{FiLM}(x) = \gamma \odot x + \beta$
- Residual MLP: [256, 256] \rightarrow 6-dim twist outputs (3-dim per arm)

G.3 Training

G.3.1 Data Collection

- **Source:** Human teleoperation demonstrations
- **Tasks:** All 9 objects (3 fixed, 3 revolute, 3 prismatic) with diverse initial configurations
- **Demonstrations:** 5,000 expert trajectories collected across all objects
- **Augmentation:** Random spawn within safe workspace region, uniform orientation $\theta_o \in [0, 2\pi)$, random joint configurations within limits

G.3.2 Training Protocol

- **Algorithm:** Behavior cloning with MSE loss
- **Optimizer:** Adam (lr=1e-4, $\beta=(0.9, 0.999)$)
- **Batch size:** 256 trajectories
- **Training steps:** 100k gradient updates
- **Loss function:**

$$\mathcal{L}_{\text{BC}} = \mathbb{E}_{(o,a) \sim \mathcal{D}} [\|\pi_{\theta}(o) - a^{\text{expert}}\|_2^2]$$

G.4 Key Differences from SWIVL

Component	OWIL	SWIVL
Learning Paradigm	Behavior cloning (offline)	Reinforcement learning (online)
Reference Motion	None (direct from demonstrations)	Stable imitation vector field
Action Space	Direct arm twists ($\mathcal{V}_l, \mathcal{V}_r$)	Impedance variables ($d_{l,\parallel}, d_{r,\parallel}, d_{l,\perp}, d_{r,\perp}, k_{p_l}, k_{p_r}, \alpha$)
Constraint Satisfaction	Implicit learning from data	Structural guarantee via projection operators
Observation Dimension	24-dim (no reference)	30-dim (reference included)
Action Dimension	6-dim (unconstrained twists)	7-dim (impedance modulation)

G.5 Expected Limitations

1. **Constraint Violations:** Without structural constraint enforcement, OWIL may violate kinematic constraints, especially when deviating from training distribution.
2. **Internal Forces:** Implicit coordination learning may lead to higher internal forces compared to SWIVL’s explicit force decomposition and compliance.
3. **Generalization:** Pure imitation may overfit to demonstration characteristics, limiting adaptation to novel objects or task variations.
4. **Data Efficiency:** Requires large amounts of expert data to implicitly capture constraint satisfaction patterns.
5. **Robustness:** No corrective feedback mechanism (no vector field stability) when execution deviates from demonstrated trajectories.

G.6 Evaluation Protocol

OWIL is evaluated using the same metrics as SWIVL:

Primary Metrics:

1. **Success Rate (%):** Task completion within error thresholds (position < 10 pixels, orientation $< 5^\circ$, joint $< 5^\circ$ or < 5 pixels)

2. **Constraint Violation (pixels/s):** $CViol = \frac{1}{T} \sum_t \|\mathcal{V}_l - \mathcal{V}_r - \mathcal{S}\dot{q}_{obj}\|_2$
3. **Internal Force (N):** $F_{int} = \frac{1}{T} \sum_{t,i} \|\mathcal{F}_{i,\perp}\|_2$ where $\mathcal{F}_{i,\perp}$ is the wrench component orthogonal to screw axis

Evaluation: 9 objects \times 100 trials per object = 900 rollouts per method.

extbfHypothesis: OWIL’s implicit constraint learning will show higher constraint violations and internal forces compared to SWIVL’s explicit physics-aware approach, especially on novel objects and task variations not well-represented in demonstrations.