

Energetics of the single DOF oscillator

MG3416–Advanced Structural Acoustics - Lecture #1

part B

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- Stationary forced response
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Single DOF oscillator

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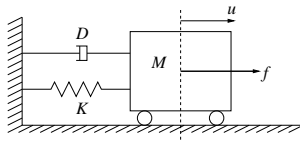
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Summary



- The single degree-of-freedom (DOF) oscillator:

$$\begin{cases} M\ddot{u}(t) + D\dot{u}(t) + Ku(t) = f(t), & t \in \mathbb{R}, \\ u(0) = u_0, \\ \dot{u}(0) = v_0; \end{cases}$$

$M > 0$: mass, $D \geq 0$: damping, and $K > 0$: stiffness.

- Fundamental parameters:

- $\omega_p = \sqrt{\frac{K}{M}}$: the undamped natural (angular) frequency,
- $\xi_p = \frac{D}{2\sqrt{KM}}$: the critical damping rate,
- $\eta_p = 2\xi_p$: the loss factor.

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Definition

The free response $t \mapsto u^\ell(t)$ is the solution of:

$$\begin{cases} M\ddot{u}(t) + D\dot{u}(t) + Ku(t) = 0, & t \geq 0, \\ u(0) = u_0, \\ \dot{u}(0) = v_0. \end{cases}$$

- Let $\omega_D = \omega_p \sqrt{1 - \xi_p^2}$, then:

$$u^\ell(t) = e^{-\xi_p \omega_p t} \left(u_0 \cos \omega_D t + \frac{\xi_p \omega_p u_0 + v_0}{\omega_D} \sin \omega_D t \right).$$

- **Property:** if $D > 0$, $\lim_{t \rightarrow +\infty} u^\ell(t) = \lim_{t \rightarrow +\infty} \dot{u}^\ell(t) = 0$.

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Definition

The forced response $t \mapsto u^f(t)$ is the solution of:

$$M\ddot{u}(t) + D\dot{u}(t) + Ku(t) = f(t), \quad t \in \mathbb{R}.$$

■ Then:

$$u^f(t) = \int_{-\infty}^t \mathfrak{h}(t - \tau) f(\tau) \, d\tau = \int_0^{+\infty} \mathfrak{h}(\tau) f(t - \tau) \, d\tau,$$

where $\mathfrak{h} : \mathbb{R} \rightarrow \mathbb{R}$ is the impulse response function of the SDOF oscillator:

$$\mathfrak{h}(t) = \mathbb{1}_{[0, +\infty[}(t) \times \frac{1}{M\omega_D} e^{-\xi_p \omega_p t} \sin \omega_D t.$$

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Frequency response function

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- $t \mapsto \mathbb{h}(t)$ is integrable and square integrable on \mathbb{R} , and its Fourier transform is:

$$\hat{\mathbb{h}}(\omega) = \int_{\mathbb{R}} e^{-i\omega t} \mathbb{h}(t) dt = \frac{1}{M(\omega_p^2 - \omega^2 + 2i\xi_p\omega_p\omega)}.$$

- $\omega \mapsto \hat{\mathbb{h}}(\omega)$ is integrable and square integrable on \mathbb{R} , and its inverse Fourier transform is:

$$\mathbb{h}(t) = \frac{1}{2\pi} \int_{\mathbb{R}} e^{i\omega t} \hat{\mathbb{h}}(\omega) d\omega.$$

- Usual quadratures:

$$\int_0^{+\infty} |\hat{\mathbb{h}}(\omega)|^2 d\omega = \frac{\pi}{2DK},$$
$$\int_0^{+\infty} \omega^2 |\hat{\mathbb{h}}(\omega)|^2 d\omega = \frac{\pi}{2DM}.$$

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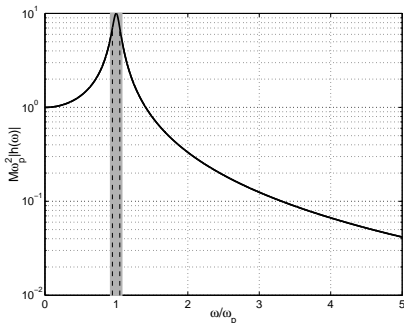
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Summary

- The **impedance** $\omega \mapsto Z(\omega)$ of the SDOF oscillator:

$$i\omega Z(\omega) = M(\omega_p^2 - \omega^2 + 2i\xi_p\omega_p\omega).$$



- $\omega'_p = \omega_p\sqrt{1 - 2\xi_p^2}$: the natural resonance frequency (for $0 < \xi_p < \frac{1}{\sqrt{2}}$);

- $b_e = \pi\xi_p\omega_p(1 - \xi_p^2)$: the **equivalent bandwidth**, s.t.

$$b_e |\hat{h}(\omega'_p)|^2 = \int_0^{+\infty} |\hat{h}(\omega)|^2 d\omega;$$

- $b_p \simeq \pi\xi_p\omega_p = \frac{\pi}{2}\Delta_p$, where $\Delta_p = \eta_p\omega_p$ is the **half-power bandwidth**.

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Definition

The evolutionary response $t \mapsto u(t)$ is the solution of:

$$\begin{cases} M\ddot{u}(t) + D\dot{u}(t) + Ku(t) = f(t), & t \geq 0, \\ u(0) = u_0, \\ \dot{u}(0) = v_0. \end{cases}$$

■ Then:

$$\begin{aligned} u(t) = u_0 e^{-\xi_p \omega_p t} & \left(\cos \omega_D t + \frac{\xi_p}{\sqrt{1 - \xi_p^2}} \sin \omega_D t \right) \\ & + \frac{v_0}{\omega_D} e^{-\xi_p \omega_p t} \sin \omega_D t + \int_0^t \mathbb{h}(t - \tau) f(\tau) d\tau. \end{aligned}$$

■ **Property:** if $D > 0$, $\lim_{t \rightarrow +\infty} |u(t) - u^f(t)| = 0$.

Energetic quantities

Definitions

Definition

- The *kinetic energy*: $\mathcal{E}_c(t) = \frac{1}{2}M\dot{u}(t)^2$,
- The *potential energy*: $\mathcal{E}_p(t) = \frac{1}{2}Ku(t)^2$,
- The *mechanical energy*: $\mathcal{E}(t) = \mathcal{E}_c(t) + \mathcal{E}_p(t)$,
- The *dissipated power*: $\Pi_d(t) = D\dot{u}(t)^2$,
- The *input power*: $\Pi_{IN}(t) = f(t)\dot{u}(t)$.

- The instantaneous power balance reads:

$$\dot{\mathcal{E}}(t) = \Pi_{IN}(t) - \Pi_d(t).$$

- It is subsequently specialized to the free, forced and evolutionary responses of the single DOF oscillator.

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- The free response mechanical energy when $\xi_p \ll 1$:

$$\mathcal{E}^\ell(t) \simeq \mathcal{E}_0 e^{-\eta_p \omega_p t},$$

where $\mathcal{E}_0 = \frac{1}{2} M v_0^2 + \frac{1}{2} K u_0^2$.

- The power balance integrated between 0 and $t > 0$:

$$\mathcal{E}_0 = \mathcal{E}_d^\ell(t) + \mathcal{E}^\ell(t),$$

hence $\mathcal{E}_d^\ell(\infty) = \mathcal{E}_0 = \int_0^{+\infty} \Pi_d^\ell(t) dt$.

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- **Data:** square integrable (finite energy) excitation with limited bandwidth,

$$|\hat{f}(\omega)| \leq C \quad \forall \omega \in \mathbb{R}; \quad \hat{f}(\omega) = 0 \quad \forall \omega \notin [-\omega_f, \omega_f].$$

- Then $\lim_{|t| \rightarrow +\infty} f(t) = 0$, and consequently **if $D > 0$:**

$$\lim_{|t| \rightarrow +\infty} u^f(t) = \lim_{|t| \rightarrow +\infty} \dot{u}^f(t) = 0.$$

- The power balance integrated between $-\infty$ and t :

$$\begin{aligned} \mathcal{E}^f(t) &= \int_{-\infty}^t \Pi_{\text{IN}}(\tau) \, d\tau - \int_{-\infty}^t \Pi_{\text{d}}^f(\tau) \, d\tau \\ &= \mathcal{E}_{\text{IN}}(t) - \mathcal{E}_{\text{d}}^f(t) \end{aligned}$$

since $\mathcal{E}^f(-\infty) = 0$. But $\mathcal{E}^f(+\infty) = 0$ as well, hence:

$$\mathcal{E}_{\text{IN}}(+\infty) = \mathcal{E}_{\text{d}}^f(+\infty).$$

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■ Hypotheses (wideband excitation):

■ i $\xi_p \ll 1;$

■ ii $\omega \mapsto \hat{f}(\omega)$ varies slowly on $[-\omega_f, \omega_f]$ and $\omega_f \gg \omega_p$.

■ Then:

$$\begin{aligned}\int_{\mathbb{R}} \mathcal{E}_c^f(t) dt &= \frac{M}{4\pi} \int_{\mathbb{R}} \omega^2 |\hat{h}(\omega)|^2 |\hat{f}(\omega)|^2 d\omega && \text{(Plancherel)} \\ &\simeq \frac{M}{4\pi} |\hat{f}(\omega_p)|^2 \int_{-\omega_f}^{\omega_f} \omega^2 |\hat{h}(\omega)|^2 d\omega && \text{(using (ii))} \\ &\simeq \frac{M}{2\pi} |\hat{f}(\omega_p)|^2 \int_0^{+\infty} \omega^2 |\hat{h}(\omega)|^2 d\omega && \text{(using (i))} \\ &= \frac{|\hat{f}(\omega_p)|^2}{4D}\end{aligned}$$

- Likewise $\int_{\mathbb{R}} \mathcal{E}_p^f(t) dt \simeq \frac{|\hat{f}(\omega_p)|^2}{4D}$, independently of the mass or the stiffness.

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Equipartition in the forced response (extended proof)

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$$\begin{aligned}\int_{\mathbb{R}} \mathcal{E}_c^f(t) dt &= \frac{M}{2} \int_{\mathbb{R}} (\dot{u}^f(t))^2 dt \\&= \frac{M}{4\pi} \int_{\mathbb{R}} |\hat{u}^f(\omega)|^2 d\omega \quad (\text{Plancherel } \int_{\mathbb{R}} |u(t)|^2 dt = \frac{1}{2\pi} \int_{\mathbb{R}} |\hat{u}(\omega)|^2 d\omega) \\&= \frac{M}{4\pi} \int_{\mathbb{R}} |i\omega \hat{u}^f(\omega)|^2 d\omega \quad (\text{Fourier transform } \hat{u}(\omega) = i\omega \hat{u}(\omega)) \\&= \frac{M}{4\pi} \int_{\mathbb{R}} \omega^2 |\hat{h}(\omega)|^2 |\hat{f}(\omega)|^2 d\omega \quad (u^f(t) = \int_{-\infty}^t h(t-\tau) f(\tau) d\tau) \\&\quad \implies \hat{u}^f(\omega) = \hat{h}(\omega) \hat{f}(\omega) \\&\simeq \frac{M}{4\pi} |\hat{f}(\omega_p)|^2 \int_{-\omega_f}^{\omega_f} \omega^2 |\hat{h}(\omega)|^2 d\omega \quad (\text{using (ii)}) \\&\simeq \frac{M}{2\pi} |\hat{f}(\omega_p)|^2 \int_0^{+\infty} \omega^2 |\hat{h}(\omega)|^2 d\omega \quad (\text{using (i)}) \\&= \frac{|\hat{f}(\omega_p)|^2}{4D} \quad (\int_0^{+\infty} \omega^2 |\hat{h}(\omega)|^2 d\omega = \frac{\pi}{2DM})\end{aligned}$$

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- Then:

$$\begin{aligned}\mathcal{E}_d^f(+\infty) &= \int_{\mathbb{R}} D(\dot{u}^f(t))^2 dt \\ &= \frac{D}{2\pi} \int_{\mathbb{R}} \omega^2 |\hat{h}(\omega)|^2 |\hat{f}(\omega)|^2 d\omega \quad (\text{Plancherel}) \\ &\simeq \frac{D}{2\pi} |\hat{f}(\omega_p)|^2 \int_{\mathbb{R}} \omega^2 |\hat{h}(\omega)|^2 d\omega \quad (\text{using (i)-(ii)}) \\ &= \frac{|\hat{f}(\omega_p)|^2}{2M},\end{aligned}$$

and the overall dissipated energy is independent of the damping.

- It is related to the overall mechanical energy by:

$$\mathcal{E}_d^f(+\infty) \simeq \int_{\mathbb{R}} \eta_p \omega_p \mathcal{E}^f(t) dt.$$

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- The power balance integrated between $t = 0$ and $t = +\infty$:

$$\mathcal{E}_d(+\infty) = \mathcal{E}_0 + \mathcal{E}_{\text{IN}}(+\infty).$$

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Summary

- **Data:** $(F_t, t \in \mathbb{R})$ is a \mathbb{R} -valued second order, centered stochastic process defined on (Ω, \mathcal{F}, P) , indexed on \mathbb{R} , and mean-square stationary.
- **Hypothesis:** $\exists \omega \mapsto S_F(\omega) : \mathbb{R} \rightarrow \mathbb{R}_+$ even, integrable.

Proposition

- $(U_t^f, t \in \mathbb{R})$ is a \mathbb{R} -valued second order, centered stochastic process defined on (Ω, \mathcal{F}, P) , indexed on \mathbb{R} , and mean-square stationary s.t. $S_U(\omega) = |\hat{h}(\omega)|^2 S_F(\omega)$.
- The same holds for its mean-square derivatives $(\dot{U}_t^f, t \in \mathbb{R})$ and $(\ddot{U}_t^f, t \in \mathbb{R})$, with $S_{\dot{U}}(\omega) = \omega^2 S_U(\omega)$ and $S_{\ddot{U}}(\omega) = \omega^4 S_U(\omega)$, respectively.

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- The instantaneous power balance reads:

$$\dot{\mathcal{E}}_t^f = \Pi_{\text{IN},t} - \Pi_{\text{d},t}^f,$$

as an equality of second-order random variables.

- Considering the mathematical expectation with:

$$\mathbb{E}\{(U_t^f)^2\} = R_U(t=0) = \text{Constant independent of } t,$$

$$\mathbb{E}\{(\dot{U}_t^f)^2\} = -\frac{d^2 R_U}{dt^2}(t=0) = \text{Constant independent of } t,$$

yields $\mathbb{E}\{\mathcal{E}_t^f\} = \text{Constant}$, and $\mathbb{E}\{\dot{\mathcal{E}}_t^f\} = 0$. Hence:

$$\mathbb{E}\{\Pi_{\text{IN},t}\} = \mathbb{E}\{\Pi_{\text{d},t}^f\}.$$

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■ Hypotheses (wideband excitation):

■ i $\xi_p \ll 1$;

■ ii $\omega \mapsto S_F(\omega)$ varies slowly on $[-\omega_p - \frac{b_e}{2}, \omega_p + \frac{b_e}{2}]$.

■ Then:

$$\begin{aligned}\mathbb{E}\{\mathcal{E}_{c,t}^f\} &= \frac{M}{2} \int_{\mathbb{R}} \omega^2 |\hat{h}(\omega)|^2 S_F(\omega) d\omega \\ &\simeq \frac{M}{2} S_F(\omega_p) \int_{\mathbb{R}} \omega^2 |\hat{h}(\omega)|^2 d\omega \quad (\text{using (i)-(ii)}) \\ &= \frac{\pi S_F(\omega_p)}{2D}\end{aligned}$$

- Likewise $\mathbb{E}\{\mathcal{E}_{p,t}^f\} \simeq \frac{\pi S_F(\omega_p)}{2D}$, independently of the mass or the stiffness, s.t.

$$\boxed{\mathbb{E}\{\mathcal{E}_t^f\} \simeq \frac{\pi S_F(\omega_p)}{D}}.$$

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Equipartition in the stationary forced response (extended proof)

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$$\begin{aligned}\mathbb{E}\{\mathcal{E}_{c,t}^f\} &= \frac{M}{2} \mathbb{E}\{(\dot{U}_t^f)^2\} \\&= \frac{M}{2} R_{\dot{U}}(0) \quad (\mathbb{E}\{\dot{U}_t \dot{U}_{t'}\} = R_{\dot{U}}(t - t') \implies \mathbb{E}\{(\dot{U}_t)^2\} = R_{\dot{U}}(t - t) = R_{\dot{U}}(0)) \\&= \frac{M}{2} \int_{\mathbb{R}} S_{\dot{U}}(\omega) d\omega \quad (R_{\dot{U}}(t) = \int_{\mathbb{R}} e^{i\omega t} S_{\dot{U}}(\omega) d\omega \implies R_{\dot{U}}(0) = \int_{\mathbb{R}} S_{\dot{U}}(\omega) d\omega) \\&= \frac{M}{2} \int_{\mathbb{R}} \omega^2 |\hat{h}(\omega)|^2 S_F(\omega) d\omega \quad (S_{\dot{U}}(\omega) = \omega^2 S_U(\omega) = \omega^2 |\hat{h}(\omega)|^2 S_F(\omega)) \\&\simeq \frac{M}{2} S_F(\omega_p) \int_{\mathbb{R}} \omega^2 |\hat{h}(\omega)|^2 d\omega \quad (\text{using (i)-(ii)}) \\&= \frac{\pi S_F(\omega_p)}{2D} \quad (\int_0^{+\infty} \omega^2 |\hat{h}(\omega)|^2 d\omega = \frac{\pi}{2DM} = \frac{1}{2} \int_{\mathbb{R}} \omega^2 |\hat{h}(\omega)|^2 d\omega)\end{aligned}$$

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- Then:

$$\begin{aligned}\mathbb{E}\{\Pi_{d,t}^f\} &= D \int_{\mathbb{R}} \omega^2 |\hat{h}(\omega)|^2 S_F(\omega) d\omega \\ &\simeq D S_F(\omega_p) \int_{\mathbb{R}} \omega^2 |\hat{h}(\omega)|^2 d\omega \quad (\text{using (i)-(ii)}) \\ &= \frac{\pi S_F(\omega_p)}{M},\end{aligned}$$

and the average dissipated power is independent of the damping.

- It is related to the average mechanical energy by:

$$\boxed{\mathbb{E}\{\Pi_{d,t}^f\} \simeq \eta_p \omega_p \mathbb{E}\{\mathcal{E}_t^f\}}.$$

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Summary

- Assuming null initial conditions to simplify, the evolutionary response $(U_t, t \geq 0)$ is a \mathbb{R} -valued second-order, centered stochastic process defined on (Ω, \mathcal{F}, P) , indexed on \mathbb{R}_+ , which is non stationary:

$$U_t = \int_0^t \mathbb{h}(t - \tau) F_\tau \, d\tau .$$

- **Property:** if $D > 0$, $\lim_{t \rightarrow +\infty} \|U_t - U_t^f\| = 0$.

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Summary

- "Equivalence" between:
 - 1 The overall energetic quantities for the forced response to a deterministic, wideband excitation;
 - 2 The average energetic quantities for the stationary forced response to a random, wideband (m.s.) stationary excitation;
 - 3 The time average energetic quantities for the forced response of the randomized SDOF oscillator to an harmonic excitation.
- These different cases are often (unduly) merged in the structural-acoustics literature.
- **Outlook:** multiple DOF systems.