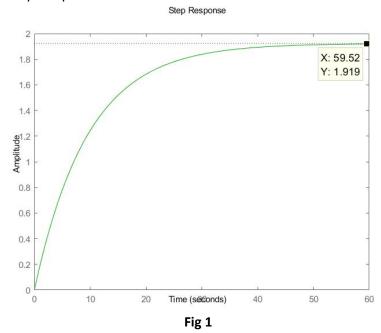
# ECSE 403 Lab 1 Report Ismail Faruk 260663521

1. Transfer function between the input voltage and the speed of the motor shaft,

$$G(s) = \frac{\omega(s)}{v_a(s)} = \frac{0.002}{0.01s + 0.00104}.$$

2. Step Response:



Steady State = 1.919

Time constant = 
$$\frac{0.01}{0.00104} = 9.615 \text{ s}$$

**3.** Step response information of G(s)

Field 📤	Value	Min	Max
RiseTime	21.1283	21.1283	21.1283
SettlingTime	37.6188	37.6188	37.6188
SettlingMin	1.7318	1.7318	1.7318
■ SettlingMax	1.9223	1.9223	1.9223
Overshoot	0	0	0
Undershoot	0	0	0
Peak	1.9223	1.9223	1.9223
PeakTime	75.1570	75.1570	75.1570

Rise time = 21.1283 s

Settling time = 37.6188 s

**4.** Steady State value can be obtained as time  $t \to \infty$ , which in Laplace domain would mean s

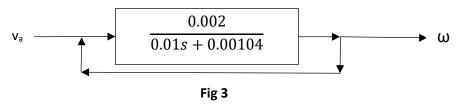
$$\rightarrow \infty$$
. Thus,  $\frac{w(s)}{v_a(s)} = \frac{0.002}{0.00104} = 1.923$ . Steady State value found in Matlab = 1.919.

This difference is because the Steady State value in Matlab is approximate and is taken at time  $t = 59.52 \, s$ , whereas the theoretical value is considered to be Steady State value as  $t \rightarrow \infty$ .

5. Transfer function between the shaft's angel and input voltage,

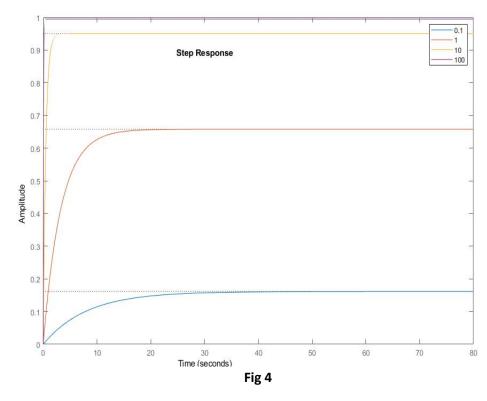
$$H(s)=rac{ heta(s)}{v_a(s)}=rac{0.002}{s(0.01s+0.00104)}$$
 This is a second order system.

**6.** Block diagram of unity feedback loop to the system:  $\omega/v_a$ 



Closed loop transfer function, 
$$\frac{\omega(s)}{v_a(s)} = \frac{0.002}{0.01s + 0.00304}$$

7. Step response of the unity feedback closed loop system KG(s)



# Step response information of KG(s)

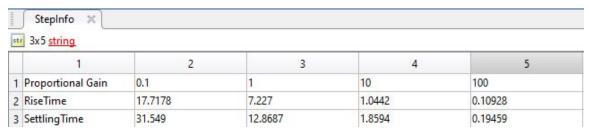
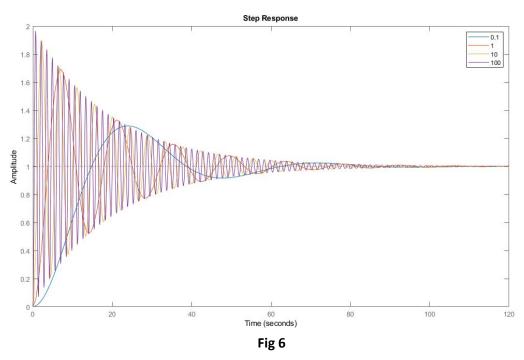


Fig 5

**Fig 4** and **Fig 5** shows the effect of proportional controller on the system. As the proportional gain increases, the rise time decreases and settling time decreases. As this is a first order system, there was no overshoot.

## **8.** Step response of the unity feedback closed loop system KH(s)



## Step response information of KH(s)

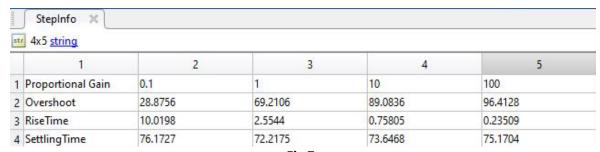


Fig 7

**Fig 6** and **Fig 7** shows the effect of proportional controller on the system. Increase in the proportional gain increases the overshoot, reduces the rise time and reduces the settling time.

**9.** For overshoot of 20%, to find the maximum value of K, we need to go through a few steps.

Step 1: Find the value of the damping ratio for 20% overshoot

$$e^{-\frac{\pi\zeta}{\sqrt{1-\zeta^2}}} = 0.2, \zeta = \sqrt{\frac{(\ln(0.2))^2}{\pi^2 + (\ln(0.2))^2}} = 0.2079$$

Step 2: Find the closed loop transfer function with proportional controller K, KH(s)

$$\frac{0.2K}{s^2 + 0.104s + 0.2K}$$

Step 3: Apply Routh-Hurwitz stability criterion

$$\begin{array}{cccc} s^2 & 1 & 0.2K \\ s^1 & 0.104 & 0 \\ s^0 & b_1 \\ b_1 = - \begin{vmatrix} 1 & 0.2K \\ 0.104 & 0 \end{vmatrix} = 0.0208K \end{array}$$

K > 0 for all times for the system to be stable, for which there will be no change in sign in the first column.

Step 4: Equate and solve with the following equation

$$\frac{A}{s^2 + 2\zeta \omega_n s + \omega_n^2}$$

Natural frequency,  $\omega_n = 0.104/(2*0.2079) = 0.25 \, s^{-1}$ 

$$K = 0.25^2/0.2 = 0.3125$$

Thus, maximum value of K = 0.3125

**10.** For rise time of 4 seconds, to find the value of K, we need to go through a few steps

Step 1: Find the natural frequency

$$\omega_n = \frac{\pi}{t_r}$$
 = 0.7854 s<sup>-1</sup>

Step 2: Find the closed loop transfer function with proportional controller K, KH(s)

$$\frac{0.2K}{s^2 + 0.104s + 0.2K}$$

Step 3: Equate and solve with the following equation

$$\frac{A}{s^2 + 2\zeta \omega_n s + \omega_n^2}$$
K = 0.7854/0.2 = 3.927

# 11. Bode plot of transfer function H

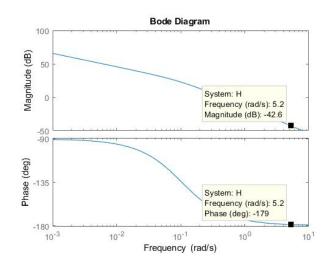


Fig 8: Gain Margin

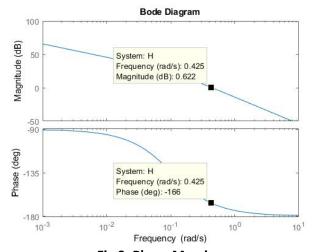


Fig 9: Phase Margin

Gain Margin = 42.6 dB, at  $\omega$  = 5.2  $s^{-1}$ 

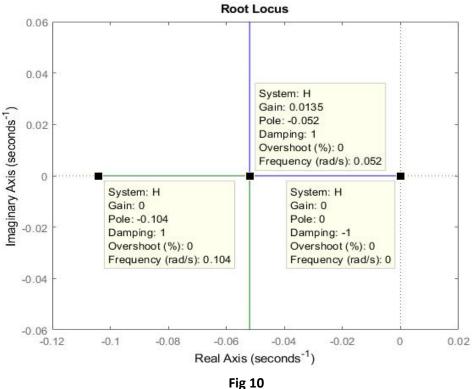
Phase Margin = -166+180 = 14 $^{\circ}$ , at  $\omega$  = 0.425  $s^{-1}$ 

### **Definitions**

Gain Margin - The gain margin is the amount of gain increase or decrease required to make the loop gain unity at the frequency, where the phase angle is -180°

Phase Margin - The phase margin is the difference between the phase of the response and -180° when the loop gain is 1.0

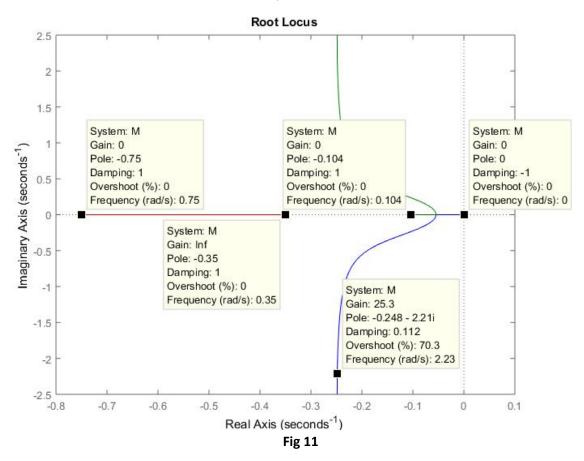
### 12. Root locus of the transfer function H



Root locus diagram is a graphical method for examining how the roots of a system change with variation of a certain system parameter.

From the **Fig 10**, we can observe that the system becomes unstable at a Gain of 0.0135.

## 13. Root locus of the transfer function H, with controller block



From the **Fig 11**, we can observe that the system is stable for all values of K because all the poles are on the left hand side plane.

### Appendix

-Units are formatted as italic

```
Source code:
s = tf('s');
% 02 to 04
G = 0.002/(0.01*s+0.00104);
figure(1)
stepplot(G);
S = stepinfo(G);
% Q5, Q11
H = 0.002/(s*(0.01*s+0.00104));
figure(2);
stepplot(H);
T = stepinfo(H);
% 07
StepInfoG = ["Proportional Gain",0,0,0,0;
             "RiseTime",0,0,0,0;
             "SettlingTime",0,0,0,0]
figure(3);
hold on;
K = [0.1 \ 1 \ 10 \ 100];
for i = 1:4
        J = 0.002*K(i)/(0.002*K(i)+0.01*s+0.00104);
        stepplot(J);
        S = stepinfo(J);
        StepInfoG(1,i+1) = K(i);
        StepInfoG(2,i+1) = S.RiseTime;
        StepInfoG(3,i+1) = S.SettlingTime;
        legendInfo{i} = [num2str(K(i))];
end
legend(legendInfo);
hold off;
% O8
StepInfoH = ["Proportional Gain",0,0,0,0;
             "Overshoot",0,0,0,0;
             "RiseTime", 0, 0, 0, 0;
```

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```
"SettlingTime",0,0,0,0]
figure(4);
hold on;
K = [0.1 \ 1 \ 10 \ 100];
for i = 1:4
        L = 0.002*K(i)/(0.002*K(i)+s*(0.01*s+0.00104));
        stepplot(L);
        S = stepinfo(L);
        StepInfoH(1,i+1) = K(i);
        StepInfoH(2,i+1) = S.Overshoot;
        StepInfoH(3,i+1) = S.RiseTime;
        StepInfoH(4,i+1) = S.SettlingTime;
        legendInfo{i} = [num2str(K(i))];
end
legend(legendInfo);
hold off;
% Q11
figure(5);
bode(H);
% Q12
figure(6);
rlocus(H);
% Q13
figure(7);
Ks = (s+0.35)/(s+0.75);
M = Ks*H;
rlocus(M);
```