ECSE 403 - Control

Laboratory 5 - Report



Mohamed Reda EL KHILI - 260678513 François-Eliott Rousseau - 260670000 Ismail Faruk - 260663521 1) Considering the position as the output and the coefficients from the previous laboratory, we obtain the transfer function: $G(s) = \frac{1200}{s*(s+8.569)}$. We obtain the bode plot shown below with a gain margin of infinity and a phase margin of 19.8 deg.

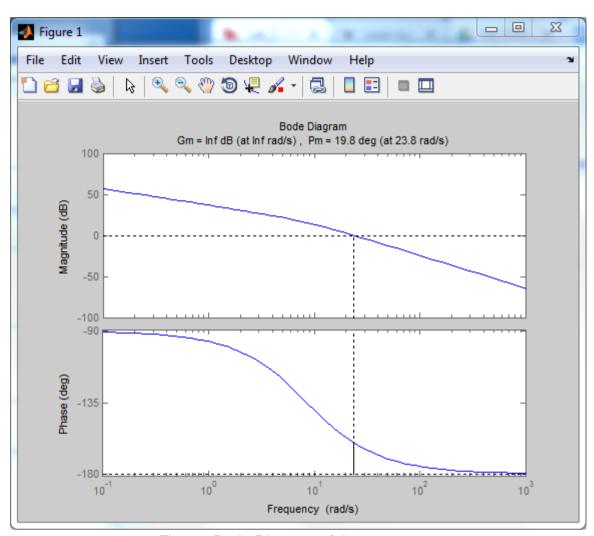


Figure: Bode Diagram of the system

2) We use Kp = 0.45 and we obtain a phase margin of 20.9 deg and a gain margin of infinity.

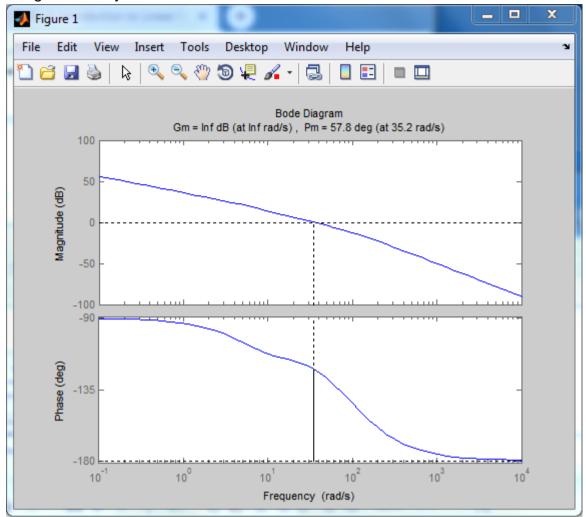


Figure: Open Loope Bode Diagram

- a) Done
- b) Done
- c) The current phase margin of the open-loop system is 20.9 deg
- d) Taking a safety margin of 5 degrees, the required phase is 44.1 deg.
- e) We have a required phase of 44.1 degrees. Solving the equation:

$$\phi = \sin^{-1}(\frac{\alpha-1}{\alpha+1})$$
 . We get $\alpha = 5.579$

- By looking at the Bode plot we find a desired frequency wm of 35.2 rad/sec
- g) We get, $T = \frac{1}{wm*\sqrt{a}} = 0.012$.

Hence, our lead controller is $C_{lead} = \frac{0.067s + 1}{0.012s + 1}$

With this lead compensator, we reach a phase margin of 57.8 deg.

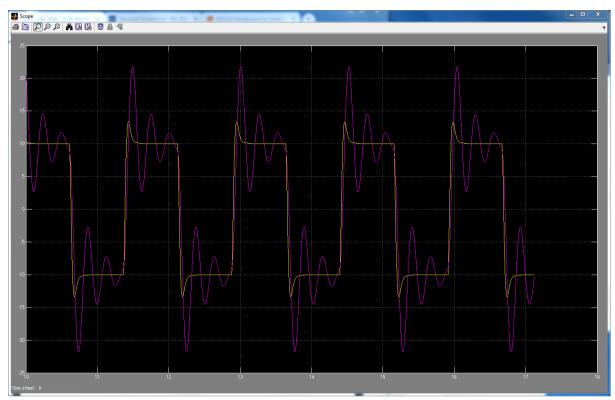


Figure: Step response of proportional feedback controller, with and without the compensator - Model

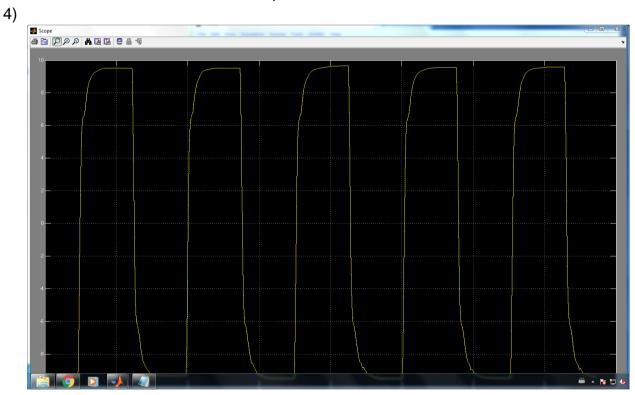


Figure: Step response of the system to proportional feedback controller with Lead compensator - Physical



Figure: Step response of the system to proportional feedback controller without Lead compensator - Physical

5) We can see that adding the lead compensator allows the system to get rid of the overshoot and to have a slower rise time by having a bigger Kp. We also have a bigger phase margin which makes the system more stable. We have a steady state error of 2.3%.

The bandwidth of the closed loop system is at 51 rad/s (frequency at which the magnitude goes to -3dB).

6)

- a) Done
- b) Taking the limit of s * H(s) * K _{lag} as s goes to 0, we find:

$$K_{v} = K_{lag} * 63.02$$

- c) We had a steady state error of 2.3% using the lead compensator. Hence, we now want a steady state error of 0.23% with the lead-lag compensator as asked. We thus get: $e_{ss} = 0.23\%$ We thus obtain $K_{lag} = 6.899$
- d) By looking at the bode plot, we find that at w = 6.92 rad/sec, we have the required phase margin of -110 deg.
- e) We find using w = 6.92 rad/sec a dB-drop 17.8 dB and we get $\alpha = 7.7625$
- f) We finally get T = 1.4451

Hence, our lead lag compensator is given by:

$$C \quad _{lag} = K \quad _{lag} \frac{Ts + 1}{\alpha Ts + 1} = 6.898 \times \frac{1.4451s + 1}{7.7625 \times 1.4451 \times s + 1}$$
 Plotting the Bode Plot, we obtain a phase margin of 57.9.

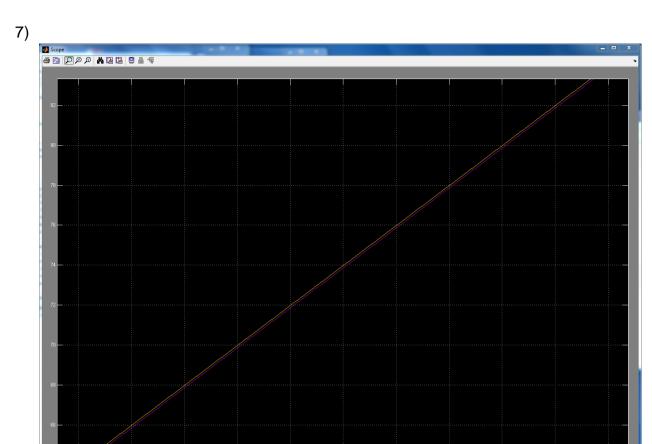


Figure: step response of proportional feedback controller with LEAD and LEAD-LAG compensators

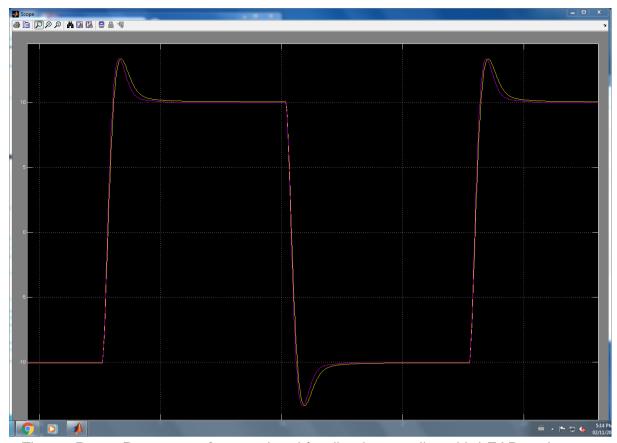


Figure: Ramp Response of proportional feedback controller with LEAD and LEAD-LAG compensators

8)

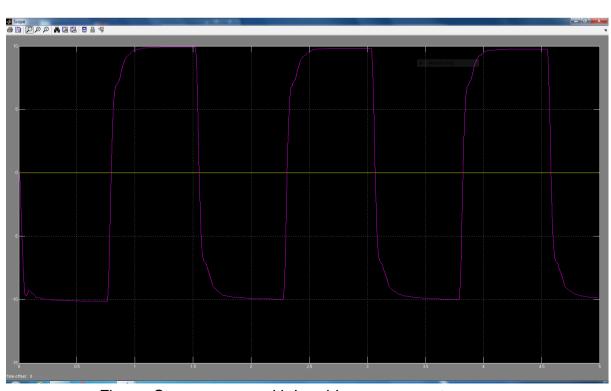


Figure: Step response with Lead-Lag compensators

9) The Lead-Lag has smaller rise but higher settling time than PID. Lead-Lag also has lower steady state error and no overshoot.

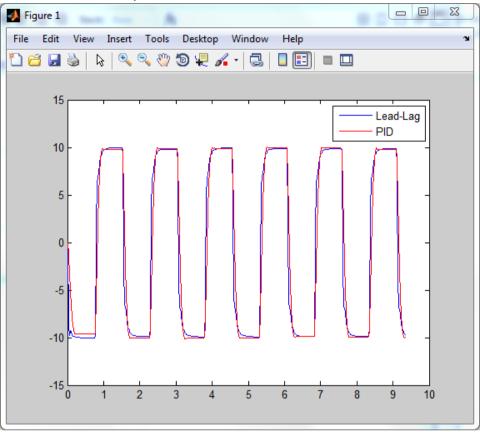


Figure: Lead-Lag compensators vs PID

Appendix

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응응 01
s = tf('s');
G = 1200/(s*(s+8.569));
응응 Q2
H = 1200*0.45/(s*(s+8.569));
bode (H)
margin(H)
%% Q5
C = (0.067*s+1)/(0.012*s+1);
Z=C*H;
margin(Z);
bode(Z);
%% Q6
X = 6.899*(1.4451*s+1)/(1.4451*7.7625*s+1);
margin(Z*X)
bode(Z*X);
%% Q9
% time = (simout.time);
% H1 = (simout.signals.values);
% H2 = (simout.signals.values); %kp = 0.178, ki = 0.092, kd = 0.0075
% used to find length
% length(H1); length 4722
% length(H2); length 4682
% length(time); length 4682
% H t = length(H2); %4682
% H3 = H1(1:H_t); %extract data of set size H_t
% H4 = H2(1:H t); %extract data of set size H t
% t = time(1:H t);
plot(t, H3, t, H2, 'r');
legend('Lead-Lag', 'PID');
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