

# EE381 Control System Engineering

## Final Exam Solution (June 15, 2023)

Total score (\_\_\_\_\_)

Student ID: \_\_\_\_\_ Name: \_\_\_\_\_

### 1. (Ans)

(a) (5 points)

$$G(j\omega) = \frac{K e^{-j\omega}}{j\omega}$$
$$\angle G(j\omega) = -\omega - \frac{\pi}{2}$$

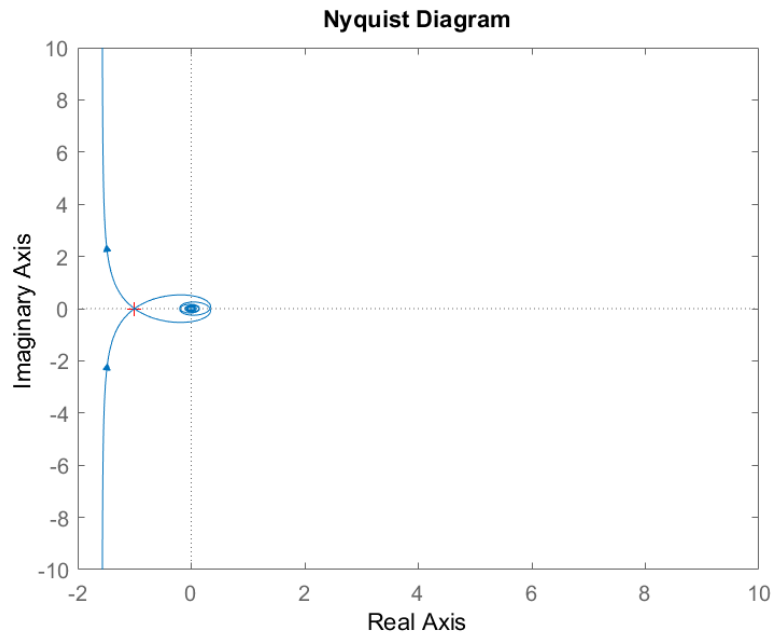
The phase angle became equal to  $-180^\circ$  at  $\omega = \pi/2$  rad/sec. For stability, the magnitude  $|G(j\omega)|$  at  $\omega = \pi/2$  must be less than unity. Hence,

$$|G(j\omega)| = \frac{K}{\omega}.$$

We require that  $K < \pi/2$  for stability.

(b) (5 points)

The Nyquist plot for  $K = \pi/2$  is shown as follows:

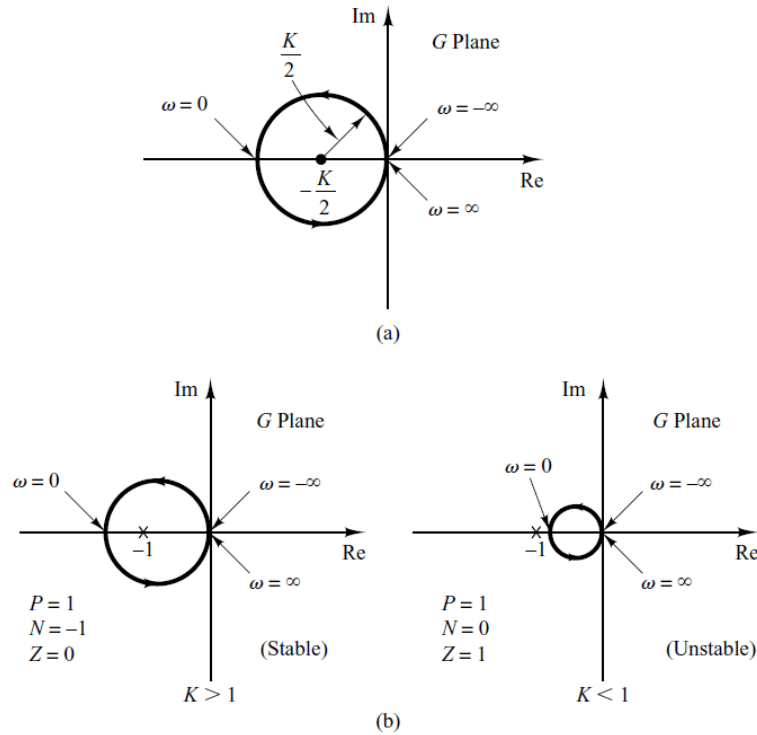


2. (Ans) (10 points)

The polar plot of

$$G(j\omega) = \frac{K}{j\omega - 1}$$

is a circle with center at  $-K/2$  on the negative real axis and radius  $K/2$ , as shown in Figure (a). As  $\omega$  is increased from  $-\infty$  to  $\infty$ , the  $G(j\omega)$  locus makes a counterclockwise rotation. In this system,  $P = 1$  because there is one pole of  $G(s)$  in the right-half  $s$  plane. For the closed-loop system to be stable,  $Z$  must be equal to zero. Therefore,  $N = Z - P$  must be equal to  $-1$ , or there must be one counterclockwise encirclement of the  $-1 + j0$  point for stability. (If there is no encirclement of the  $-1 + j0$  point, the system is unstable.) Thus, for stability,  $K$  must be greater than 1, and  $K = 1$  gives the stability limit. Figure (b) shows both stable and unstable cases of  $G(j\omega)$  plots.



3. (Ans)

(a) (5 points)

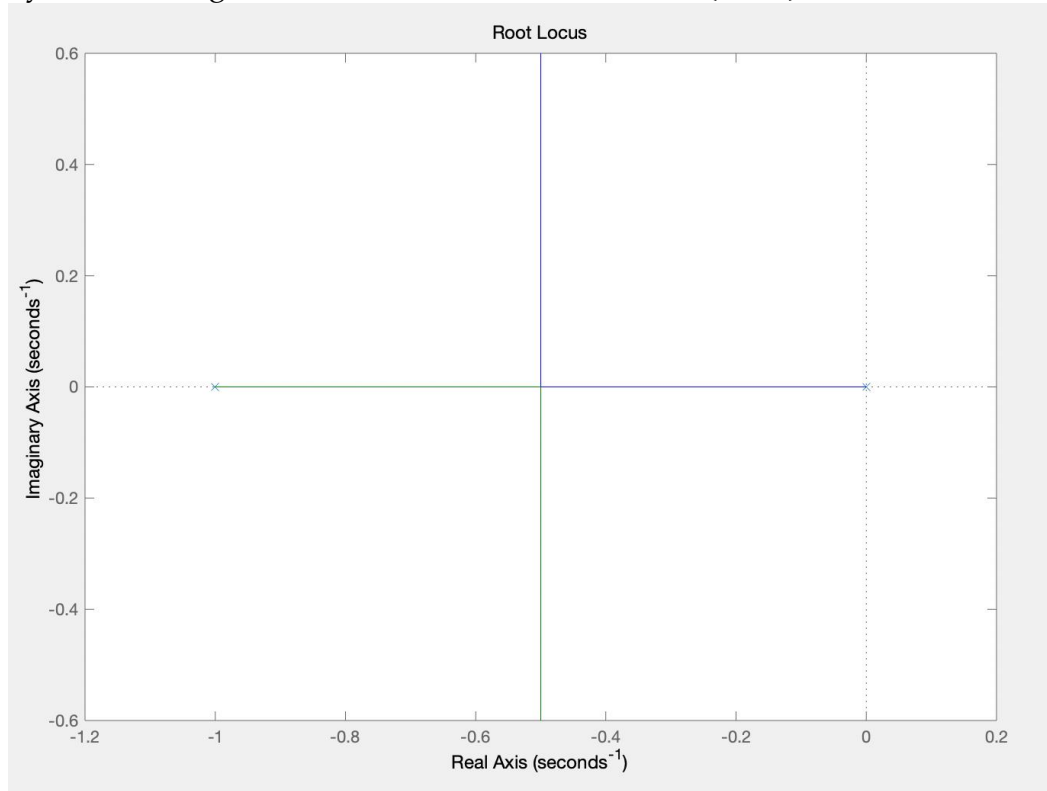
Because the response of the system is underdamped with clear evidence of overshoot and oscillation. Such phenomena cannot occur in first order system.

(b) (5 points)

Because the steady state error for step input is zero, type number of the open loop system must be 1 or higher. (i.e. type number 1 or higher) Hence the open loop transfer function must contain an integrator, or pole at the origin.

(c) (5 points)

Because the system is the second order and open loop system includes a pole at origin, let  $L(s) = K \frac{1}{s(s+2\zeta\omega_n)}$ . Then  $T_s \approx \frac{4}{\zeta\omega_n} = 8$ , hence  $\zeta\omega_n = 0.5$ , which implies that the other pole is at -1. Breakaway point at -0.5. This can be found by determining where the maximum of  $K = -s(s+1)$  occurs.



(d) (5 points)

$\phi_m = 55^\circ$ . Then  $\sin 55^\circ = \frac{9}{11} = \frac{\alpha-1}{\alpha+1}$ , hence  $\alpha = 10$ . Then  $-10 \log \alpha = -10 \text{ dB}$ . Then we can find  $\omega_n$  as follows:  $|G(j\omega)| = \left| \frac{\omega_n^2}{s(s+1)} \right| = \frac{\omega_n^2}{\omega\sqrt{1+\omega^2}} = \frac{1}{\sqrt{\alpha}} = \frac{1}{\sqrt{10}}$ , then  $\omega\sqrt{1+\omega^2} = 3\sqrt{10}$ , and it directly follows  $\omega_m = 3$ . Therefore,  $p = \omega_m\sqrt{\alpha} = 3\sqrt{10}$ , and  $z = \frac{3\sqrt{10}}{10}$ . Hence the appropriate lead compensator is given by  $G_c(s) = \frac{1 + \frac{10s}{3\sqrt{10}}}{1 + \frac{s}{3\sqrt{10}}}$ .

4. (Ans)

(a) (10 points)

Using free body diagram, we can write the equation of motion like below.

$$M\ddot{x} + b\dot{x} + kx = b\dot{r} + kr$$

Therefore, by taking Laplace transform, the transfer function  $\frac{X(s)}{R(s)}$  is

$$\frac{X(s)}{R(s)} = \frac{bs + k}{Ms^2 + bs + k}.$$

Then, using given parameters  $M = 2$  kg,  $b = 8$  Ns/m and  $k = 128$  N/m,

$$\frac{X(s)}{R(s)} = \frac{8s + 128}{2s^2 + 8s + 128} = \frac{4s + 64}{s^2 + 4s + 64}.$$

(b) (10 points)

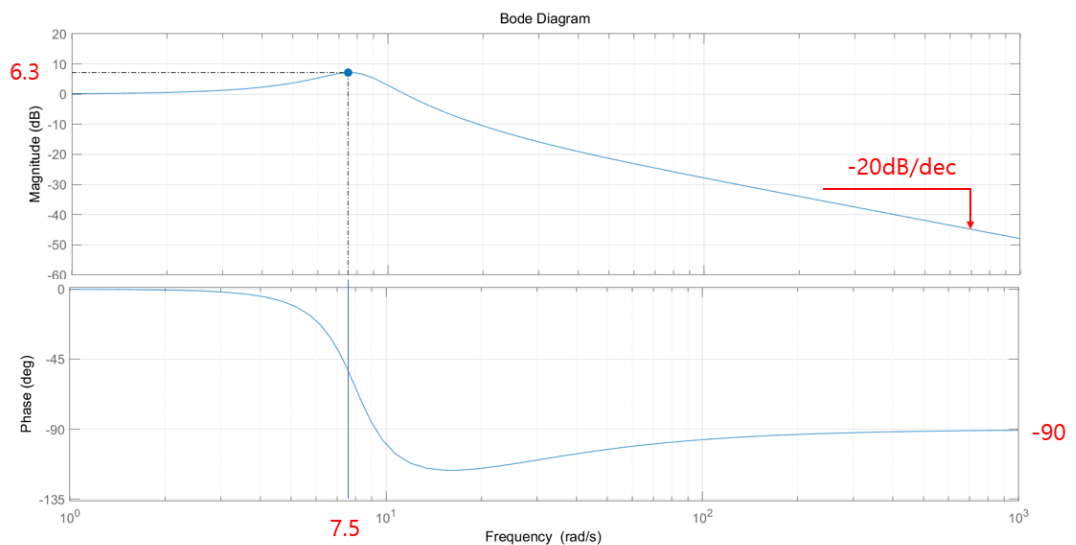
We can think about dominant complex conjugate poles.

Therefore,  $\omega_n = \sqrt{k/M} = \sqrt{64} = 8$  rad/s and the damping ratio  $\zeta = \frac{b/M}{2\omega_n} = 0.25$ .

Then,  $\omega_r = \omega_n \sqrt{1 - 2\zeta^2} = 8\sqrt{1 - 2(0.25)^2} = 2\sqrt{14} = 7.5 \frac{\text{rad}}{\text{s}}$

$$M_{p\omega} = |G_o(j\omega_r)| = \left(2\zeta\sqrt{1 - \zeta^2}\right)^{-1} = \frac{1}{2 * 0.25\sqrt{1 - (0.25)^2}} = \frac{8}{\sqrt{15}} = \frac{8\sqrt{15}}{15} = 2.07$$

$$= 20 \log\left(\frac{8\sqrt{15}}{15}\right) = 6.3 \text{ dB}$$



[Bode plot of the transfer function]

You should represent DC gain,  $M_{p\omega}$ ,  $\omega_r$  in the Bode plot.

**5. (Ans)**

**(a) (5 points)**

$P = 0, N = 1 + 1 = 2$ , so  $Z = N + P = 2$ . The system is unstable and has two roots in the right-hand  $s$ -plane.

**(b) (5 points)**

$N = -1$  (one CCW encirclement)  $+ 1 = 0$ , so  $Z = 0$ . The system is stable.

## 6. (Ans)

### (a) (5 points)

The system with initial compensation is depicted in the block diagram representation in Fig. 1(i), which can be converted to a system with cascade compensation as in Fig. 1(ii) by moving the summing point ahead of  $G_c(s)$ .

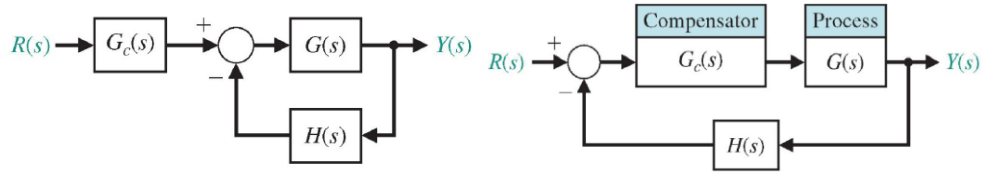


Fig.1 (i) Input compensation (ii) Cascade compensation

Since we want the converted system in Fig. 1(ii) to have unity feedback, then we need to satisfy:

$$\frac{H(s)}{G_c(s)} = 1 \rightarrow G_c(s) = H(s) = \frac{4}{s + 10}.$$

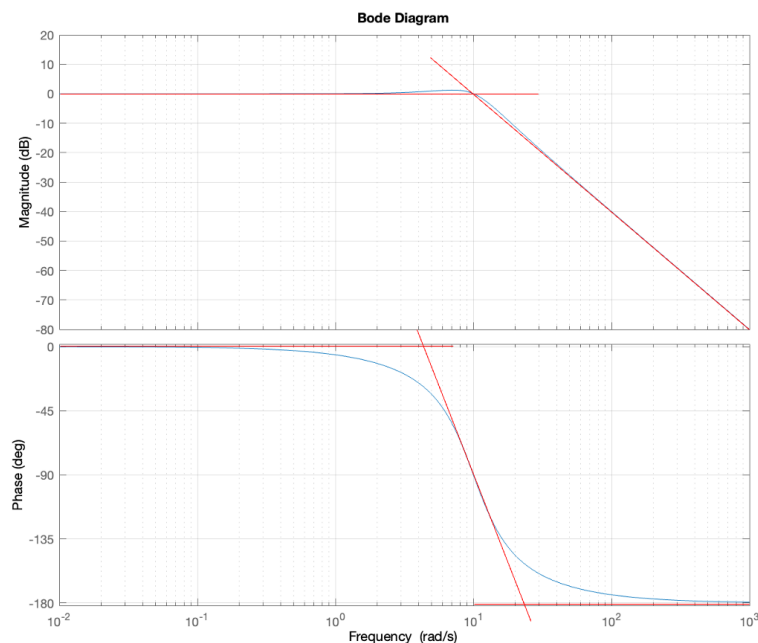
We have  $T(s)$  as in the following so it consists of a complex conjugate pole at  $\omega_n = 10$  rad/s and  $\zeta = 0.5$ .

$$T(s) = \frac{Y(s)}{R(s)} = \frac{100}{s^2 + 10s + 100} \rightarrow T(j\omega) = \frac{1}{1 + 0.1j\omega + 0.01(j\omega)^2}.$$

### (b) (5 points)

From this, the Bode plot of  $T(s)$  has the following asymptotic components as shown in the figure below.

- A magnitude plot that is 0 dB at  $\omega \ll 10$  rad/s and decreases at -40 dB/dec at  $\omega \gg 10$  rad/s, which asymptotes meet at 0 dB line at  $\omega = 10$  rad/s
- A phase angle plot that is 0 deg at  $\omega \ll 10$  rad/s, -180 deg at  $\omega \gg 10$  rad/s, and -90 deg at  $\omega = 10$  rad/s



7. (Ans)

(a) (5 points)

The loop transfer function in the frequency domain becomes:

$$L(j\omega) = \frac{K}{(j\omega t_1 + 1)(j\omega t_2 + 1)(j\omega t_3 + 1)} .$$

Taking the magnitude gives:

$$|L(j\omega)| = \frac{|K|}{\sqrt{1 + (\omega t_1)^2} \sqrt{1 + (\omega t_2)^2} \sqrt{1 + (\omega t_3)^2}} .$$

And the phase is:

$$\angle L(j\omega) = -\tan^{-1}(\omega t_1) - \tan^{-1}(\omega t_2) - \tan^{-1}(\omega t_3) .$$

As  $\omega$  approaches infinity, the phase of  $L(j\omega)$  becomes:

$$\angle L(j\omega) = -\frac{\pi}{2} - \frac{\pi}{2} - \frac{\pi}{2} = -\frac{3\pi}{2} .$$

(b) (5 points)

We first express  $L(j\omega)$  in the form of real and imaginary parts. By substituting  $s$  with  $j\omega$  and multiplying numerator and denominator by complex conjugate, we have:

$$L(j\omega) = \frac{K(1 - \omega^2 t_1)(1 - \omega^2 t_2)(1 - \omega^2 t_3) - j\omega K(t_1 + t_2 + t_3 - \omega^2 t_1 t_2 t_3)}{(1 + \omega^2 t_1^2)(1 + \omega^2 t_2^2)(1 + \omega^2 t_3^2)} .$$

Grouping real and imaginary parts yields:

$$K \cdot \frac{(1 - \omega^2(t_1 t_2 + t_2 t_3 + t_3 t_1)) - j\omega(t_1 + t_2 + t_3 - \omega^2 t_1 t_2 t_3)}{(1 + \omega^2 t_1^2)(1 + \omega^2 t_2^2)(1 + \omega^2 t_3^2)} .$$

Thus, the real part of  $L(j\omega)$  is given by:

$$\Re(L) = K \cdot \frac{1 - \omega^2(t_1 t_2 + t_2 t_3 + t_3 t_1)}{(1 + \omega^2 t_1^2)(1 + \omega^2 t_2^2)(1 + \omega^2 t_3^2)} .$$

And the imaginary part of  $L(j\omega)$  is given by:

$$\Im(L) = K \cdot \frac{-\omega(t_1 + t_2 + t_3 - \omega^2 t_1 t_2 t_3)}{(1 + \omega^2 t_1^2)(1 + \omega^2 t_2^2)(1 + \omega^2 t_3^2)} .$$

Setting the imaginary part equal to zero and solving for  $\omega$  gives us:

$$\omega = \sqrt{\frac{t_1 + t_2 + t_3}{t_1 t_2 t_3}} .$$

This is the frequency at which the imaginary part of  $L(j\omega)$  is zero.

(c) (5 points)

To find the value of  $K$  where  $L(j\omega) = -1$ , we assume that  $t = t_1 = t_2 = t_3$ .

We know from part (b) that the imaginary part is zero for  $\omega = \sqrt{\frac{3}{t^2}}$ .

Substituting  $\omega_2$  with  $\frac{3}{t^2}$  in the real part of  $L(j\omega)$ , we have:

$$\Re(L) = K \cdot \frac{1 - \left(\frac{3}{t^2}\right)(t^2 + t^2 + t^2)}{\left(1 + \left(\frac{3}{t^2}\right)t^2\right)^3}.$$

Simplifying this expression yields:

$$\begin{aligned}\Re(L) &= K \cdot \frac{1 - 9}{(1 + 3)^3} \\ \Re(L) &= K \cdot \frac{-8}{64} = -\frac{K}{8}.\end{aligned}$$

We want  $L(j\omega) = -1$ , so we set  $-\frac{K}{8} = -1$  and solve for  $K$ :

$$-\frac{K}{8} = -1 \rightarrow K = 8.$$

Thus, the value of  $K$  where  $L(j\omega) = -1$  for the system, when  $t = t_1 = t_2 = t_3$ , is  $K = 8$ .

(d) (5 points)

To draw the Nyquist plot, we need to analyze the behavior of  $L(s)$  for different frequencies. The given system has  $K = 20$ . From the results of part (a), we know that for  $\omega \rightarrow \infty$ , the magnitude part of  $L(j\omega)$  is given by  $\frac{|K|}{\sqrt{1+(\omega t)^2}^3}$ . The Nyquist plot is shown in the image below:

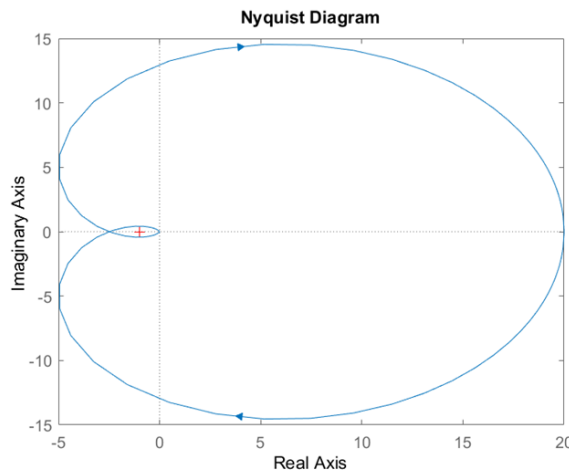


Figure 1: Nyquist Plot of  $L(s)$



As seen from the Nyquist plot, the locus of  $L(s)$  encircles the point  $(-1, 0)$  twice in the clockwise direction. According to the Nyquist criterion, each encirclement represents a right-half plane pole. Since there are two encirclements, it indicates that the system has two right-half plane poles. Therefore, the system is unstable.