# Quick Incompact3d guide and notes

This is a quick guide on to how to use **Incompact3d** and to know some details of its functioning. The present guide and the modified version of the code (baseline v4.0) are unofficial. Please refer to the official website of the code: <a href="https://www.incompact3d.com">https://www.incompact3d.com</a> and the major references [1],[2],[3],[4].

# **Compiling**

In order to use *Incompact3d*, the external Fortran library used for parallelization and FFT 2decomp-fft must be compiled. We first start by describing its installation.

#### **Compilation of 2decomp-fft**

In the folder /5-external/2decomp-fft create a build directory:

# mkdir build

Before preparing the make file with cmake (version v3.21 or higher), we need to define the variable FC of cmake (Fortran Compiler) used to compile *2decomp-fft* library:

## export FC=ifort

where in this example the Intel Fortran compiler ifort is specified. Another common possibility is the GNU compiler gfortran, that is the default option. In order to use ifort, we need to export the mkl directory path (even if a different FFT implementation is used; this is needed by Incompact3d later due to cmake):

### export MKL\_DIR=\${MKLROOT}/lib/cmake/mkl

Now, it is possible to define the make file and (optionally) to specify the specific FFT implementation (FFT\_Choice parameter). Options available are: generic, fftw, fftw\_03, mkl. Default option is generic. In the *build* directory we can run:

It is also possible to specify single precision for saving fields, through the <code>cmake</code> option <code>SINGLE\_PRECISION\_OUTPUT</code>. This allows to maintain double precision for calculations and for checkpoint files. The full <code>cmake</code> command is thus:

In closing the cmake command options, we remind that is possible to directly specify the compiler choice during cmake, through the variable CMAKE\_Fortran\_COMPILER, so for example: -DCMAKE\_Fortran\_COMPILER=ifort.

After that, the *makefile* is generated. In the same folder we can compile:

where n is the number of processors employed. Finally, it is possible to install 2decomp-fft:

# make install

For further information on 2decomp-fft please refer to the original documentation, available at: <a href="https://github.com/2decomp-fft">https://github.com/2decomp-fft</a>.

#### Compilation of Incompact3d

Similarly to the 2decomp-fft installation, we create a build directory under /2-solver:

We then define the make file through cmake inside the build directory. The default Fortran compiler is gfortran as *2decomp-fft*; the exported cmake variable FC is valid also here. Floating-point precision for outputs must be specified also in this case (if needed):

We can finally compile the binary file:

### **Basic functioning**

In this section, the main features of the code that can be useful to run simulations are reported.

1. The code is designed to solve non-dimensional Navier-Stokes equations, since the only input parameter is Re. However, consistency of other parameters (e.g. inlet velocity and domain dimensions) allow to have dimensional simulations. The Re specified in the input file is used to compute the kinematic viscosity as

$$\nu = \frac{1}{Re}$$

so we are assuming unitary reference length and velocity scales. Reference scales depend then on the specific flow case.

#### **Channel flow**

For a channel flow, the default condition of simulation is to enforce a constant flow rate (CFR), while constant pressure gradient (CPG) can be enabled if necessary. Both conditions are rescaled with the centerline Reynolds number the corresponding laminar Poiseuille flow,  $Re_0 = \frac{U_0 h}{\nu}$ . A relation is available in order to estimate the related friction Reynolds number  $Re_{\tau}$ :

$$Re_ aupprox 0.116 Re_0^{0.88}$$

With CPG option enabled, the same relation is used to impose the same non-dimensionalization, since  $Re_{\tau}$  must be specified in the input file instead of  $Re_0$ :

$$Re_0pprox \left(rac{Re_ au}{0.116}
ight)^{1/0.88}$$

In CFR conditions, being  $U_0$  constant and unitary, this means that the bulk velocity  $U_B$  must be kept to the value of 2/3. This can be demonstrated being the laminar profile parabolic.

Finally, in order to estimate the related bulk Reynolds number  $Re_B = \frac{2U_Bh}{\nu}$ , the following relation can be employed (Pope, "Turbulent Flows"):

$$Re_{ au}pprox 0.09 Re_{B}^{0.88} \Rightarrow Re_{B}pprox \left(rac{Re_{ au}}{0.09}
ight)^{1/0.88}$$

- 2. Pressure field is made non-dimensional with the reference specific kinetic energy content  $q_{ref}=\frac{1}{2}\rho U_{ref}^2$ , where  $U_{ref}$  is the reference velocity of the case. The code performs operations on the pressure field with a factor  $\Delta t$ , but it is rescaled before saving the snapshots with rescale\_pressure subroutine.
- 3. To evaluate the stretching parameter  $\beta$  for the mesh elements in y direction, some useful scripts can be found in the folder 1-pre\_processing/stretching\_mesh. The default Fortran program of Incompact3d is stretching\_parameter\_channel.f90 and it is used to setup channel flow simulations. For temporal TBL simulations, the Python script mesh\_evaluation\_ttbl.py was developed. This script can handle also preprocessing of channel flow cases. Low  $\beta$  values correspond to a strong stretching of the mesh, while high  $\beta$  values correspond to almost uniform mesh.
- 4. Variables are saved on  $y_p$  points, that are the faces of the grid elements, while  $y_{pi}$  points are the centers of the grid elements (i: internal).
- 5. Boundary values of velocity are specified through the  $b_{ijk}$  variables, where i is the wall-normal direction of the boundary, j is the direction of the specific velocity component and k specifies if we are considering the bottom or the top walls (e.g.  $b_{yx1}$  refers to the x velocity component, specified at the bottom boundary with normal direction y).
- 6. Velocity boundary conditions are specified in the input file input.i3d with the following variables: nclx1, nclxn, ncly1, nclyn, nclz1, nclzn, that specify the normal direction to the boundary and if we are considering the first or the last element along the specific direction. Values that can be adopted are: 0, for *periodic BC*, 1 for *free-slip BC* and 2 for *Dirichlet BC* (so imposed velocity value). Boundary conditions can be different along the same direction, so different combinations can be enforced. Due to the nature of the code, pressure boundary conditions are not needed.
- 7. Boundary conditions for scalar fields have the same functioning of the velocity BCs. They are called: nclxS1, nclxSn, nclyS1, nclySn, nclzS1, nclzSn.
- 8. The hyperviscous option for the second order derivative is a way to increase the numerical dissipation of the standard 6th order accurate scheme (that is sub-dissipative). In this manner, it is possible to increase the modified wavenumber at high wavenumbers, thus increasing the dissipation error  $E_{diss}$ , similarly to what is observed in high-order upwind schemes. This approach allows to prevent wiggles and thus to improve stability, even at high cell Reynolds number (or numerical Péclet)  $P\acute{e}\approx 200$ . The two parameters available can be used also to control the numerical dissipation in the context of ILES simulations.
- 9. Implicit time integration is available for the diffusive terms in y direction. From preliminary analyses, it appears that it allows to drop the restriction due to the stability parameter S < 1 of fully explicit time integration schemes (Thompson et al. (1985)). The stability parameter S can be calculated in the three directions

$$S_i = rac{u_i^2 \Delta t}{2 
u}$$

considering i=x,y,z. With implicit diffusion for y direction terms, the hyperviscous operator for the second order derivative can be used only with Dirichlet conditions on both boundaries. It is also important to notice that this approach is experimental, but channel flow simulations have been validated with data from literature, in both cases of fixed and oscillating walls.

- 10. Avoid to cancel the latest restart.info file, in order to being able to correctly calculate the current time unit (since the current time unit before restart is read from restart.info file itself, that is located in the data/restart\_info folder).
- 11. The total shear velocity is calculated as:

$$u_{ au} = \sqrt{
u rac{\partial U_{\parallel,w}}{\partial y}}$$

where  $U_{\parallel}=\sqrt{U^2+W^2}$  is the parallel velocity to the wall. Similarly, the x and z components are defined as:  $u_{ au,x}=\sqrt{
u|rac{\partial U_w}{\partial y}|}$  and  $u_{ au,z}=\sqrt{
u|rac{\partial W_w}{\partial y}|}$ .

12. The (total) friction coefficient  $c_f$  is calculated as:

$$c_f = 2igg(rac{u_ au}{U_{ref}}igg)^2$$

where  $U_{ref}$  is the reference velocity according to the specific flow case ( $U_w$  for a TTBL and  $U_B$  for a channel). The same approach is followed to calculate the streamwise and the spanwise friction coefficients,  $c_{f,x}$  and  $c_{f,z}$ .

13. The restart procedure of the code does not compromise the order of accuracy of the time-integration, since old time steps used for Adams—Bashforth schemes are stored in the checkpoint files.

#### Visualization

Visualization of the results can be performed by opening the .xdmf snapshot files of the folder data with a visualization/post-processing software (e.g. Paraview). Moreover, it is also possible to visualize 2D instantaneous z-normal planes of the scalar field (if present) and x-normal planes of streamwise vorticity. This can be done by opening .xdmf files that can be found inside the data/planes folder. This can be useful for large cases, being a single plane much less memory-demanding. The frequency of saving of the planes can be set with the parameter <code>ioutput\_plane</code> in the input file.

# **Power input for TTBLs**

The power input  $P_{in}$  for a TTBL with fixed walls is calculated in the following manner (assuming ho=1):

$$P_{in,fw} = au_{w,x} U_w = \mu rac{\partial U}{\partial y} U_w = u_{ au,x}^2 U_w$$

where  $u_{\tau,x}$  is the streamwise shear velocity. For the oscillating walls case, the additional power input required to move the wall in z direction is considered:

$$P_{in,ow} = au_{w,x} U_w + au_{w,z} W_w = \mu rac{\partial U}{\partial y} U_w + \mu rac{\partial W}{\partial y} W_w = u_{ au,x}^2 U_w + u_{ au,z}^2 W_w$$

# Spanwise wall oscillations

Spanwise wall oscillations are calculated in the following manner:

$$w_w(t) = A \sin \left(rac{2\pi}{T}t + \phi_{in}\pi
ight)$$

where A is the amplitude, T is the period and  $\phi_{in}$  is the initial phase of oscillation, given as fraction of  $\pi$ . Wall oscillations are enabled by setting <code>iswitch\_wo = 1</code>. Parameters for wall oscillations are read from the input file. If feedback control is not enabled, A and T are in external units, while in case of closed-loop control (<code>ifeedback\_control = 1</code>), the parameters are considered as rescaled in wall-units with the run-time streamwise shear velocity  $u_{\tau,x}$ .

#### References

- 1. <u>High-order compact schemes for incompressible flows, A simple and efficient method with quasi-spectral accuracy Laizet & Lamballais 2009</u>
- 2. <u>Incompact3d A powerful tool to tackle turbulence problems with up to O(10\frac{5}) computational cores Laizet & Li 2011</u>

- 3. <u>Xcompact3D An open-source framework for solving turbulence problems on a Cartesian mesh Bartholomew et al. 2020</u>
- 4. <u>Straightforward high-order numerical dissipation via the viscous term for DNS and LES Lamballais et al. 2011</u>
- 5. <u>Turbulent Flows Pope</u>
- 6. The cell Reynolds number myth Thompson et al. 1985