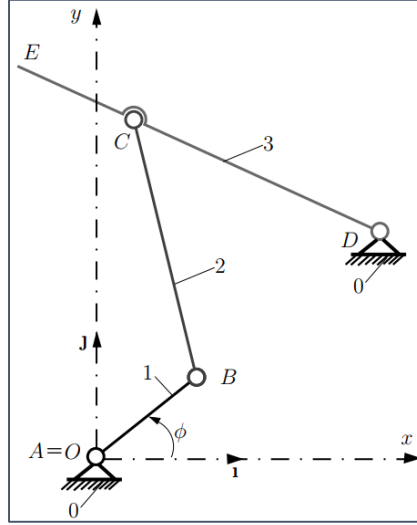


## Assignments

**GROUP 1:** The considered four-bar (R-RRR) planar mechanism is shown in Figure 1 below. The driver link is the rigid link 1 (the element AB) and the origin of the reference frame is at A. The following data are given:  $AB=0.150$  m,  $BC=0.35$  m,  $CD=0.30$  m,  $CE=0.15$  m,  $x_D=0.30$  m, and  $y_D=0.30$  m. The angle of the driver link 1 with the horizontal axis is  $\phi = \phi_1 = 45^\circ$ . Find the positions of the joints and the angles of the links with the horizontal axis.



**Figure 1: Four-bar (R-RRR) mechanism.**

**GROUP 2:** Starting from a vector-loop closure equation of a four bar linkage shown in Figure 2 below, show that;

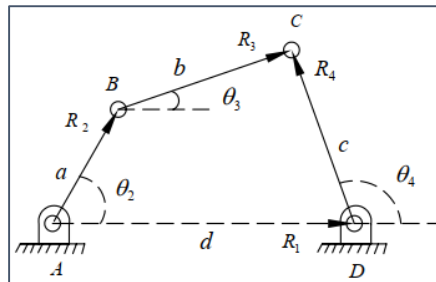
$$\mathbf{K}_1 \cos \theta_3 + \mathbf{K}_4 \cos \theta_2 + \mathbf{K}_5 = \cos \theta_2 \cos \theta_3 + \sin \theta_2 \sin \theta_3; c^2 - d^2 - a^2 - b^2 \quad \text{Eq.1}$$

Where;  $\mathbf{K}_1 = \frac{d}{a}, \quad \mathbf{K}_4 = \frac{d}{b}, \quad \mathbf{K}_5 = \frac{c^2 - d^2 - a^2 - b^2}{2ab}$

Also show that Eq.1 above reduces to a quadratic equation of form;

$$\mathbf{D} \tan^2 \frac{\theta_3}{2} + \mathbf{E} \tan \frac{\theta_3}{2} + \mathbf{F} = 0$$

Where;  $\mathbf{D} = (1 + \mathbf{K}_4) \cos \theta_2 - \mathbf{K}_1 + \mathbf{K}_2$  ;  $\mathbf{E} = -2 \sin \theta_2$  ;  $\mathbf{F} = \mathbf{K}_1 + (\mathbf{K}_4 - 1) \cos \theta_2 + \mathbf{K}_5$



**Figure 2: Position vector loop for a four-bar linkage.**

(The sum of vectors in the loop is given by;  $R_2 + R_3 - R_4 - R_1 = 0$ )