弹性力学

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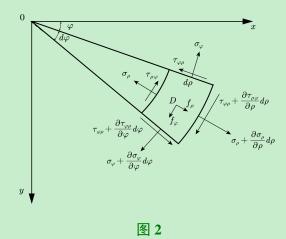
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图 1: Adhémar Jean Claude Barré de Saint-Venant

1 平面问题的极坐标解答

1.1 极坐标中的平衡微分方程



1.1.1 极坐标中的平衡微分方程

$$\begin{cases} \frac{\partial \sigma_{\rho}}{\partial \rho} + \frac{1}{\rho} \frac{\partial \tau_{\rho\varphi}}{\partial \varphi} + \frac{\sigma_{\rho} - \sigma_{\varphi}}{\rho} + f_{\rho} = 0\\ \frac{1}{\rho} \frac{\partial \sigma_{\varphi}}{\partial \varphi} + \frac{\partial \tau_{\rho\varphi}}{\partial \rho} + \frac{2\tau_{\rho\varphi}}{\rho} + f_{\varphi} = 0 \end{cases}$$
(1)

1.2 极坐标中的几何方程和物理方程

1.2.1 极坐标中的几何方程

$$\begin{cases} \varepsilon_{\rho} = \frac{\partial u_{\rho}}{\partial \rho} \\ \varepsilon_{\varphi} = \frac{u_{\rho}}{\rho} + \frac{1}{\rho} \frac{\partial u_{\rho}}{\partial \varphi} \\ \gamma_{\rho\varphi} = \frac{1}{\rho} \frac{\partial u_{\rho}}{\partial \varphi} + \frac{\partial u_{\rho}}{\partial \rho} - \frac{u_{\varphi}}{\rho} \end{cases}$$
(2)

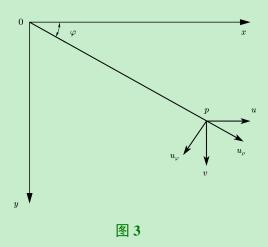
1.2.2 极坐标中的物理方程

$$\begin{cases} \varepsilon_{\rho} = \frac{1}{E} \left(\sigma_{\rho} - \mu \sigma_{\varphi} \right) \\ \varepsilon_{\varphi} = \frac{1}{E} \left(\sigma_{\varphi} - \mu \sigma_{\rho} \right) \\ \gamma_{\rho\varphi} = \frac{1}{G} \tau_{\rho\varphi} = \frac{2(1+\mu)}{E} \tau_{\rho\varphi} \end{cases}$$
(3)

平面应变的情况下,需将上式中的E换为 $\frac{E}{1-\mu^2}$, μ 换为 $\frac{\mu}{1-\mu}$ 。

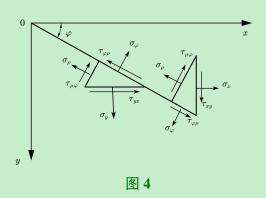
1.3 坐标变换

1.3.1 位移分量的坐标变换



$$\begin{cases}
 u_{\rho} = u\cos\varphi + v\sin\varphi \\
 u_{\varphi} = -u\sin\varphi + v\cos\varphi
\end{cases} \Longrightarrow \begin{pmatrix} u_{\rho} \\ u_{\varphi} \end{pmatrix} = \begin{bmatrix} \cos\varphi & \sin\varphi \\ -\sin\varphi & \cos\varphi \end{bmatrix}, \begin{pmatrix} u \\ v \end{pmatrix} = \begin{bmatrix} \cos\varphi & -\sin\varphi \\ \sin\varphi & \cos\varphi \end{bmatrix} \begin{pmatrix} u_{\rho} \\ u_{\varphi} \end{pmatrix} \tag{4}$$

1.3.2 应力分量的坐标变换



$$\begin{cases} \sigma_{x} = \sigma_{\rho} \cos^{2} \varphi + \sigma_{\varphi} \sin^{2} \varphi - 2\tau_{\rho\varphi} \sin \varphi \cos \varphi \\ \sigma_{y} = \sigma_{\rho} \sin^{2} \varphi + \sigma_{\varphi} \cos^{2} \varphi + 2\tau_{\varphi\rho} \sin \varphi \cos \varphi \\ \tau_{xy} = \frac{\sigma_{\rho} - \sigma_{\varphi}}{2} \sin 2\varphi + \tau_{\rho\varphi} \cos 2\varphi \end{cases}$$
 (5)

$$\begin{cases}
\sigma_{x} = \frac{\sigma_{\rho} + \sigma_{\varphi}}{2} + \frac{\sigma_{\rho} - \sigma_{\varphi}}{2} \cos 2\varphi - \tau_{\rho\varphi} \sin 2\varphi \\
\sigma_{y} = \frac{\sigma_{\rho} + \sigma_{\varphi}}{2} - \frac{\sigma_{\rho} - \sigma_{\varphi}}{2} \cos 2\varphi - \tau_{\rho\varphi} \sin 2\varphi \\
\tau_{xy} = \frac{\sigma_{\rho} - \sigma_{\varphi}}{2} \sin 2\varphi + \tau_{\rho\varphi} \cos 2\varphi
\end{cases} (6)$$

$$\begin{bmatrix} \sigma_{x} & \tau_{xy} \\ \tau_{yx} & \sigma_{y} \end{bmatrix} = \begin{bmatrix} \cos \varphi & \sin \varphi \\ -\sin \varphi & \cos \varphi \end{bmatrix} \begin{bmatrix} \sigma_{\rho} & \tau_{\varphi\rho} \\ \tau_{\rho\varphi} & \sigma_{\varphi} \end{bmatrix} \begin{bmatrix} \cos \varphi & \sin \varphi \\ -\sin \varphi & \cos \varphi \end{bmatrix}^{T}$$
(7)

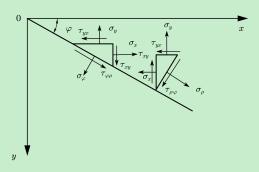


图 5

$$\begin{cases} \sigma_{\rho} = \sigma_{x} \cos^{2} \varphi + \sigma_{y} \sin^{2} \varphi + 2\tau_{xy} \sin \varphi \cos \varphi \\ \sigma_{\varphi} = \sigma_{x} \sin^{2} \varphi + \sigma_{y} \cos^{2} \varphi - 2\tau_{xy} \sin \varphi \cos \varphi \\ \tau_{\rho\varphi} = (\sigma_{y} - \sigma_{x}) \sin \varphi \cos \varphi + \tau_{xy} (\cos^{2} \varphi - \sin^{2} \varphi) \end{cases}$$
(8)

$$\begin{cases}
\sigma_{\rho} = \frac{\sigma_{x} + \sigma_{y}}{2} + \frac{\sigma_{x} - \sigma_{y}}{2} \cos 2\varphi + \tau_{xy} \sin 2\varphi \\
\sigma_{\varphi} = \frac{\sigma_{x} + \sigma_{y}}{2} - \frac{\sigma_{x} - \sigma_{y}}{2} \cos 2\varphi - \tau_{xy} \sin 2\varphi \\
\tau_{\varphi\rho} = \frac{\sigma_{x} - \sigma_{y}}{2} \sin 2\varphi + \tau_{xy} \cos 2\varphi
\end{cases} \tag{9}$$

$$\begin{bmatrix} \sigma_{\rho} & \tau_{\varphi\rho} \\ \tau_{\rho\varphi} & \sigma_{\varphi} \end{bmatrix} = \begin{bmatrix} \cos\varphi & \sin\varphi \\ -\sin\varphi & \cos\varphi \end{bmatrix}^{T} \begin{bmatrix} \sigma_{x} & \tau_{xy} \\ \tau_{yx} & \sigma_{y} \end{bmatrix} \begin{bmatrix} \cos\varphi & \sin\varphi \\ -\sin\varphi & \cos\varphi \end{bmatrix}$$
(10)

1.4 极坐标中的应力函数与相容方程

当 $f_{\rho} = f_{\varphi} = 0$ 时:

$$\begin{cases}
\sigma_{\rho} = \frac{1}{\rho} \frac{\partial \Phi}{\partial \rho} + \frac{1}{\rho^{2}} \frac{\partial^{2} \Phi}{\partial \varphi^{2}} \\
\sigma_{\varphi} = \frac{\partial^{2} \Phi}{\partial \rho^{2}} \\
\tau_{\rho \varphi} = -\frac{\partial}{\partial \rho} \left(\frac{1}{\rho} \frac{\partial \Phi}{\partial \varphi} \right)
\end{cases} \tag{11}$$

1.4.1 相容方程

$$\left(\frac{\partial^2}{\partial \rho^2} + \frac{1}{\rho} \frac{\partial}{\partial \rho} + \frac{1}{\rho^2} \frac{\partial^2}{\partial \varphi^2}\right)^2 \Phi = 0 \tag{12}$$

1.5 轴对称应力和相容的位移

所谓轴对称问题,是指物体几何形状或某物理量是绕某一轴对称的,凡通过此轴 的任何面均为对称面。如果该物体所受外部荷载也对称于该轴,那么相应所产生的应 力也必对称该轴。

 $\diamondsuit \Phi = \Phi(\rho)$

$$\sigma_{\rho} = \frac{1}{\rho} \frac{d\Phi}{d\rho} , \sigma_{\varphi} = \frac{d^2\Phi}{d\rho^2}, \tau_{\rho\varphi} = 0$$
 (13)

$$\left(\frac{\partial^2}{\partial \rho^2} + \frac{1}{\rho} \frac{\partial}{\partial \rho}\right) \Phi = 0 \tag{14}$$

即

$$\Phi = A \ln \rho + B\rho^2 \ln \rho + C\rho^2 + D \tag{15}$$

应力分量

$$\begin{cases}
\sigma_{\rho} = \frac{A}{\rho^{2}} + B(1 + 2\ln\rho) + 2C \\
\sigma_{\varphi} = -\frac{A}{\rho^{2}} + B(3 + 2 + \ln\rho) + 2C \\
\tau_{\rho\varphi} = 0
\end{cases} \tag{16}$$

代入物理方程和几何方程可得位移分量

$$\begin{cases} u_{\rho} = \frac{1}{E} \left[-(1+\mu)\frac{A}{\rho} + 2(1-\mu)B\rho(\ln\rho - 1) + (1-3\mu)B\rho + 2(1-\mu)C\rho \right] + I\cos\varphi + K\sin\varphi \\ u_{\varphi} = \frac{4B\rho\varphi}{E} + H\rho - I\sin\varphi + K\cos\varphi \end{cases}$$
(17)

- (1). 在轴对称应力条件下,应力、应变和位移的通解,适用于任何轴对称应力问题。
- (2). 在轴对称应力条件下,应变也是轴对称的,但位移不一定是轴对称的。
- (3). 实现轴对称应力的条件是: 物体形状、体力和面力应是轴对称的。
- (4). 轴对称应力及对应的位移的通解已满足相容方程,它们还需满足边界条件及多连体中的位移单值条件,并由此求出系数A、B、C。

注 欧拉方程:

$$x^{n}y^{(n)} + P_{1}x^{n-1}y^{(n-1)} + \dots + P_{n-1}xy' + P_{n}y = 0$$

特征方程为:

$$[k(k-1)\cdots(k-n+1)+P_1[k(k-1)\cdots(k-n+2)]+\cdots+P_{n-1}k+P_n]=0$$

关于K的n次代数方程,可解得n个特征根 k_1 、 k_2 、... k_n

(1). 当它们是互不相等的实根时, 通解具有幂函数的形式

$$y = C_1 x^{k_1} + C_2 x^{k_2} + \dots + C_n x^{k_n}$$

(2). 每当出现重根时,每多一重根,就多乘一个 $\ln x$ 如: $3k_1 为 m (m < n)$ 重根时,通解为

$$y = C_1 x^{k_1} + C_2 x^{k_2} \ln x + \dots + C_m x^{k_1} \ln^{m-1} x + C_{m+1} x^{k_{m+1}} + \dots + C_n x^{k_n}$$

(3). 出现共轭复根时,则虚部是三角函数因子如: 当 $k_{1,2} = \alpha \pm i\beta$,通解为

$$y = C_1 x^{\alpha} \cos(\beta \ln x) + C_2 x^{\alpha} \sin(\beta \ln x) + C_3 x^{k_3} + \dots + C_n x^{k_n}$$

(4). 当出现复重根,则实部要多乘因子 $\ln x$ 如: $k_{1,2} = \alpha \pm i\beta 为 m (m < \frac{n}{2})$ 重共轭复根时,通解为

$$y = [C_1 x^{\alpha} + C_2 x^{\alpha} \ln x + \dots + C_m x^{\alpha} \ln^{m-1} x] \cos(\beta \ln x) +$$

$$[C_{m+1} x^{\alpha} + C_{m+2} x^{\alpha} \ln x + \dots + C_{2m} x^{\alpha} \ln^{m-1} x] \sin(\beta \ln x) +$$

$$C_{2m+1} x^{k_{2m+1}} + \dots + C_n x^{k_n}$$

例 1.1 试求解平面轴对称应力问题的相容方程

$$\rho^4 \Phi^{(4)} + 2\rho^3 \Phi^{(3)} - \rho^2 \Phi'' + \rho \Phi' = 0$$

解 特征方程:

$$k(k-1)(k-2)(k-3) + 2k(k-1)(k-2) - k(k-1) + k = 0$$

解得:

$$k_{12} = 0, k_{34} = 2$$

得:

$$\Phi = A \ln \rho + B\rho^2 \ln \rho + C\rho^2 + D$$