

$$\dot{x}(t) = Ax(t) + Bu(t) + \underbrace{B_r v(t)}_{\text{DETERMINISTIC DISTURBANCE}}$$

$$y(t) = Cx(t) + \underbrace{w(t)}_{\text{MEASUREMENT NOISE}}$$

We want to establish linear feedback, a linear feedback law

$$u(t) = -Kx(t) + r(t)$$

$\underbrace{\phantom{-Kx(t)}}$      $\underbrace{\phantom{r(t)}}$  REFERENCE INPUT ( $N_u$ )

FEEDBACK OR  
GAIN MATRIX

$N_u \times N_x$

As we are assuming that all the state variables are measured the resulting feedback system is called full-state feedback

→ IT IS INTENDED THAT THE OUTPUT OBJECT SHOULD FOLLOW THE REFERENCE INPUT, IN SOME SENSE

SUBSTITUTING  $u = -Kx + r$  INTO  $\dot{x} = Ax + Bu$  WE GET

$$\dot{x} = Ax + B(-Kx + r) = \underbrace{Ax - BKx}_{\text{STATE EQUATION FOR THE}} + Br$$

CLOSED LOOP LINEAR SYS

- THE SYSTEM IS ASYMPTOTICALLY STABLE BASED ON THIS MATRIX
- UNDER SOME CONDITIONS, IT IS POSSIBLE TO ARBITRARILY ASSIGN THE EIGENVALUES OF THIS MATRIX (By setting matrix  $K$ )

ONE OF THE DRAWBACK OF STATE FEEDBACK CONTROL IS THAT THE OUTPUT OF THE SYSTEM IS NOT DIRECTLY INVOLVED IN THE CONTROL

→ we cannot assign a predetermined relationship between the inputs and the outputs

CONSIDER THE CLOSED-LOOP SYSTEM  $\dot{x} = A_k x + B r$

IT IS ASSUMED THAT THE SYSTEM IS IN STATIONARY STATE

→ THE DERIVATIVE OF THE STATE VECTOR IS ZERO

$$0 = A_k x_0 + B r_0$$

ASSUMING THAT MATRIX  $A_k$  IS NONSINGULAR, WE CAN COMPUTE THE STATIONARY STATE, SOLVING FOR  $x_0$ .

$$x_0 = -A_k^{-1} B r_0$$

THE STATIONARY OUTPUT WITH THIS STATE IS  $y_0 = C x_0$   
 $= -C A_k^{-1} B r_0$

IF THE SYSTEM HAS THE SAME NUMBER OF INPUTS AND OUTPUTS, THEN MATRIX  $-C A_k^{-1} B r_0$  WILL BE SQUARE

THE REFERENCE VECTOR FOR A GIVEN OUTPUT

$$r_0 = -(C A_k^{-1} B)^{-1} y_0$$

↑  
MUST BE NONSINGULAR

The basis for linear feedback control is a linear model, typically found by linearization of a nonlinear model about some stationary operating point

THIS MEANS THAT THIS REFERENCE SIGNAL IS USEFUL ONLY IF THE SYSTEM STATE IS EXACTLY THE STATIONARY STATE OF THE LINEARISATION

## BASIC CONCEPTS RELATED TO STABILITY OF $\dot{x} = Ax$

$$x(t) = T e^{Dt} T^{-1} x(0)$$

WITH  $e^{Dt} = \begin{bmatrix} e^{\lambda_1 t} & & 0 \\ & e^{\lambda_2 t} & \\ 0 & & e^{\lambda_n t} \end{bmatrix}$

LET  $\lambda = a + ib$  THEN  $e^{\lambda t}$  is  
 $= e^{at} [\cos(bt) + i\sin(bt)]$

CLEARLY IF ANY  $e^{\lambda_i t}$  IS UNSTABLE, THEN  $x(t)$  WILL GROW AS TIME  $\rightarrow \infty$

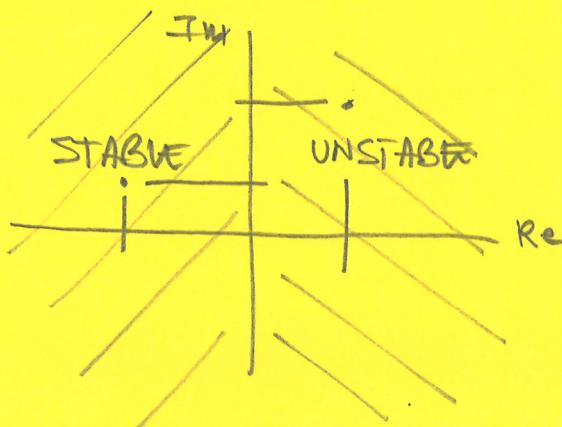
THIS IS ESPECIALLY TRUE IF WE HAVE  $x(0)$  WITH ALL COMPONENTS THAT ARE NON-ZERO

BUT WE NEED TO GET A REAL SIGNAL  $\rightarrow$  THIS GETS SORTED OUT BECAUSE FOR EACH  $\lambda_i \in \mathbb{C}$  ALSO ITS CONJUGATE COMPLEX IS A SOLUTION OF THE CHARACTERISTIC EQUATION

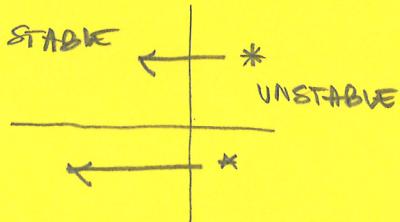
AND THE COMPLEX PART WILL CANCEL OUT

IF  $a > 0$ , THEN  $e^{at}$  WILL GROW EXPONENTIALLY (UNSTABLE)

IF  $a < 0$ , THEN  $e^{at}$  WILL DECAY EXPONENTIALLY (STABLE)



THE WHOLE IDEA OF CONTROL IS TO ACT ON THE SYSTEM IN SUCH A WAY THAT WHENEVER WE HAVE UNSTABLE EIGENVALUES WE CHANGE THEM TO BE STABLE  $\rightarrow$  THIS STABILIZING THE SYSTEM



WE THOTENTARILY CONSIDER THE DISCRETE -TIME VERSION OF  $\dot{x} = Ax$

$$x_{k+1} = \tilde{A}x_k, \text{ WITH } x_k = x(k\Delta t) \text{ FOR SOME } \Delta t$$

 THIS IS THE DISCRETE VERSION OF MATRIX A

$$\tilde{A} = e^{A\Delta t} \quad (\text{Related by the matrix exponential})$$

IN DISCRETE TIME, THE NOTION OF STABILITY

$$\text{IF GIVEN } x(0), \text{ THEN } x(1) = \tilde{A}x_0$$

$$x(2) = \tilde{A}(x_1) = \tilde{A}^2x_0$$

$$x(3) = \tilde{A}(x_2) = \tilde{A}^3x_0$$

$\vdots$

$$x(N) = \tilde{A}^N x(0)$$

ANOTHER DIFFICULTY WITH STATE FEEDBACK CONTROL ARISES IF THE SYSTEM IS SUBJECTED TO DISTURBANCES

→ THE BASIC CONTROLLER DOES NOT TAKE A NON ZERO DISTURBANCE VECTOR INTO ACCOUNT

the solution to these limitation is found in augmentation with integrators

WE CAN INCLUDE INTEGRATION IN SEVERAL WAYS

WE DEFINE THE AUGMENTED STATE  $\begin{bmatrix} x \\ \dot{x}_i \end{bmatrix} = \begin{bmatrix} x & x_i \end{bmatrix}^T$

$$\begin{aligned} \dot{x} &= Ax + Bu + B_r v \\ u &= -Kx + K_i x_i \\ \dot{x}_i &= -Cx + r \\ y &= Cx \end{aligned}$$

$$\begin{bmatrix} \dot{x} \\ \dot{x}_i \end{bmatrix} = \underbrace{\begin{bmatrix} A & 0 \\ -C & 0 \end{bmatrix}}_{A_1} \begin{bmatrix} x \\ x_i \end{bmatrix} + \underbrace{\begin{bmatrix} B \\ 0 \end{bmatrix}}_{B_u} u + \underbrace{\begin{bmatrix} 0 \\ I_r \end{bmatrix}}_{B_r} r + \underbrace{\begin{bmatrix} B_r \\ 0 \end{bmatrix}}_{B_{rv}} v$$

$$y = \underbrace{\begin{bmatrix} C & 0 \end{bmatrix}}_{C_1} \begin{bmatrix} x \\ x_i \end{bmatrix}$$

OR , IN TERMS OF THE AUGMENTED STATE VECTOR

$$\left\{ \begin{array}{l} \nu \frac{d[A]}{dt} = F([A]_f - [A]) - \nu K_1 [A]^2 \\ \nu \frac{d[B]}{dt} = F([B]_f - [B]) + \nu (K_1 [A]^2 - K_3 [B]^{1/2}) \end{array} \right.$$

$$\left\{ \begin{array}{l} \frac{d[A]}{dt} = \frac{F}{\nu} ([A]_f - [A]) - K_1 [A]^2 \\ \frac{d[B]}{dt} = \frac{F}{\nu} ([B]_f - [B]) + (K_1 [A]^2 - K_3 [B]^{1/2}) \\ \\ \dot{x}_1 = \underbrace{\frac{F}{\nu} (u_1 - x_1) - K_1 x_1^2}_{f_1(x_1, x_2, u_1, u_2)} \\ \dot{x}_2 = \underbrace{\frac{F}{\nu} (u_2 - x_2) + K_1 x_1^2 - K_3 x_2^{1/2}}_{f_2(x_1, x_2, u_1, u_2)} \end{array} \right.$$

$$\left\{ \begin{array}{l} x_1^{ss} \\ x_2^{ss} \\ u_1^{ss} \\ u_2^{ss} \end{array} \right. \quad \begin{array}{l} \text{STEADY STATE} \\ \text{VALUES} \end{array}$$

$$f_1(x_1, x_2, u_1, u_2) = f_1(x_1^{ss}, x_2^{ss}, u_1^{ss}, u_2^{ss}) + \frac{\partial f_1}{\partial x_1} \Big|_{ss} (x_1 - x_1^{ss}) + \frac{\partial f_1}{\partial x_2} \Big|_{ss} (x_2 - x_2^{ss}) \\ + \frac{\partial f_1}{\partial u_1} \Big|_{ss} (u_1 - u_1^{ss}) + \frac{\partial f_1}{\partial u_2} \Big|_{ss} (u_2 - u_2^{ss})$$

$$f_2(x_1, x_2, u_1, u_2) = f_2(x_1^{ss}, x_2^{ss}, u_1^{ss}, u_2^{ss}) + \frac{\partial f_2}{\partial x_1} \Big|_{ss} (x_1 - x_1^{ss}) + \frac{\partial f_2}{\partial x_2} \Big|_{ss} (x_2 - x_2^{ss}) \\ + \frac{\partial f_2}{\partial u_1} \Big|_{ss} (u_1 - u_1^{ss}) + \frac{\partial f_2}{\partial u_2} \Big|_{ss} (u_2 - u_2^{ss})$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} x_1 - x_1^{ss} \\ x_2 - x_2^{ss} \end{bmatrix} = \begin{bmatrix} \frac{\partial f_1}{\partial x_1} & \frac{\partial f_1}{\partial x_2} \\ \frac{\partial f_2}{\partial x_1} & \frac{\partial f_2}{\partial x_2} \end{bmatrix}_{ss} \begin{bmatrix} x_1' \\ x_2' \end{bmatrix} + \\ + \begin{bmatrix} \frac{\partial f_1}{\partial u_1} & \frac{\partial f_1}{\partial u_2} \\ \frac{\partial f_2}{\partial u_1} & \frac{\partial f_2}{\partial u_2} \end{bmatrix}_{ss} \begin{bmatrix} u_1 - u_1^{ss} \\ u_2 - u_2^{ss} \end{bmatrix}$$

$$\left\{ \begin{array}{l} \frac{\partial f_1}{\partial x_1} = -\frac{F}{V} - 2K_1 x_1 \\ \frac{\partial f_2}{\partial x_1} = 2K_1 x_1 \end{array} \right. \quad \left. \begin{array}{l} \frac{\partial f_1}{\partial x_2} = 0 \\ \frac{\partial f_2}{\partial x_2} = -F/V - 1/2 K_3 x_2^{-1/2} \end{array} \right.$$

$\rightarrow$  At SS  $\leadsto A$

$$\left\{ \begin{array}{l} \frac{\partial f_1}{\partial u_1} = F/V \\ \frac{\partial f_2}{\partial u_1} = 0 \end{array} \right. \quad \left. \begin{array}{l} \frac{\partial f_1}{\partial u_2} = 0 \\ \frac{\partial f_2}{\partial u_2} = F/V \end{array} \right.$$

$\rightarrow$  At SS  $\leadsto B$

$$\begin{bmatrix} \dot{x}_1' \\ \dot{x}_2' \end{bmatrix} = \begin{bmatrix} -F/V - 2K_1 x_1^{ss} & 0 \\ 2K_1 x_1^{ss} & -F/V - 1/2 K_3 (x_2^{ss})^{-1/2} \end{bmatrix} \begin{bmatrix} x_1' \\ x_2' \end{bmatrix} +$$

$$+ \begin{bmatrix} F/V & 0 \\ 0 & F/V \end{bmatrix} \begin{bmatrix} u_1' \\ u_2' \end{bmatrix}$$